

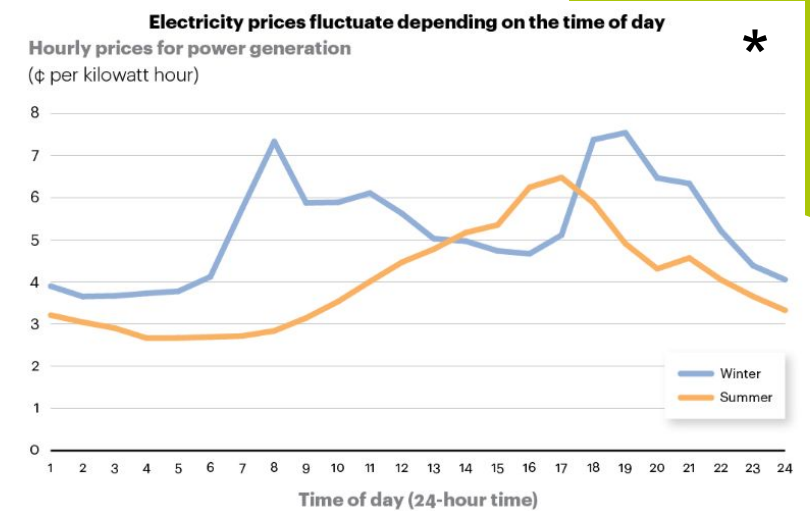
Peak load reduction using thermal energy storage in a HVAC system of a building

Student: *Aleksandr Isakov*

Research Advisor: *Elena Gryazina*

General problem

- The peak electricity demand leads to expensive energy generation;
- Increased price of energy in the market up to 4-5 times;
- High price -> high electricity bills.

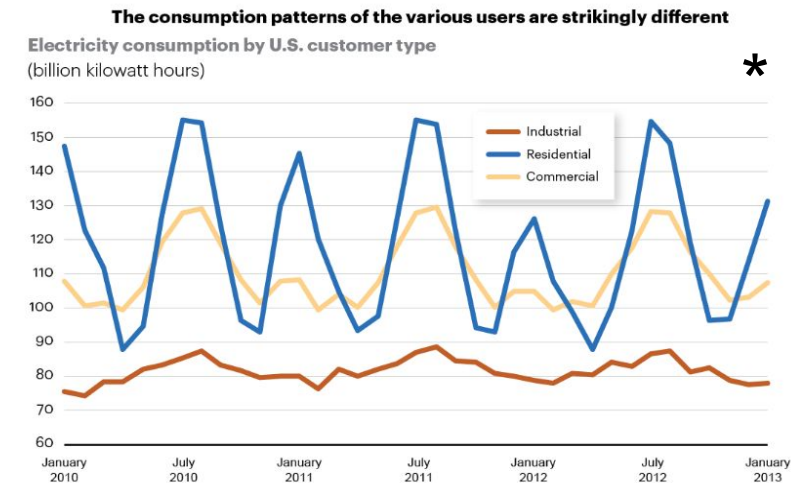
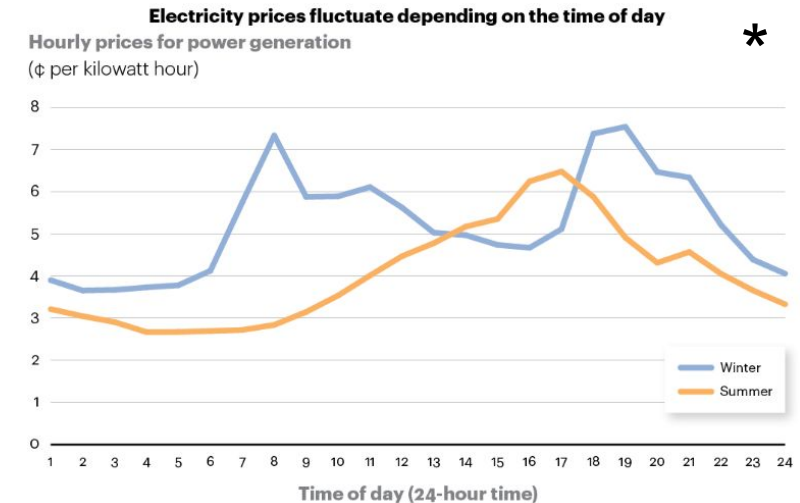


* A.T. Kearney analysis.

** Singla S, Keshav S. Demand response through a temperature setpoint market in Ontario. In: 2012 IEEE 3rd Int Conf Smart Grid Commun SmartGridComm 2012; 2012. p.103–8. doi: 10.1109/SmartGridComm.2012.6485967.

General problem

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- High price -> high electricity bills.
- The total duration of such peaks can reach up to several hours in the year(**);
- Additional equipment is idle the rest of the time -> extra expenses.



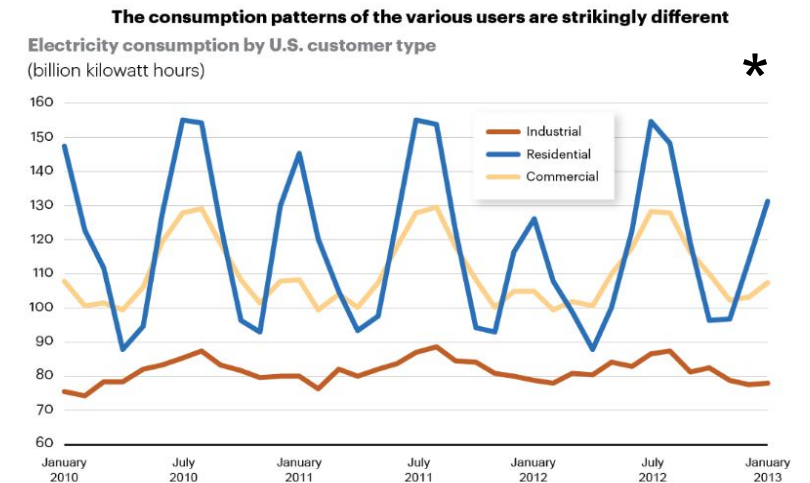
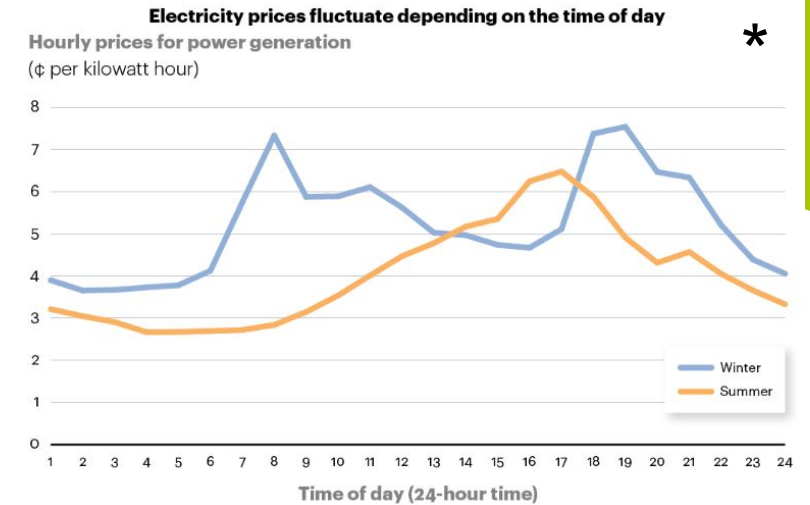
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Participants pain: Peak loads leads electricity consumers and generation companies spend a lot of money



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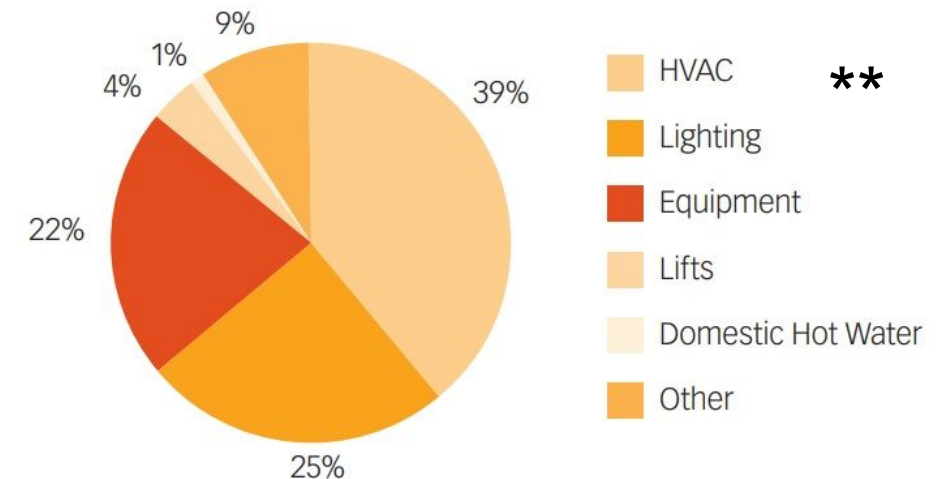
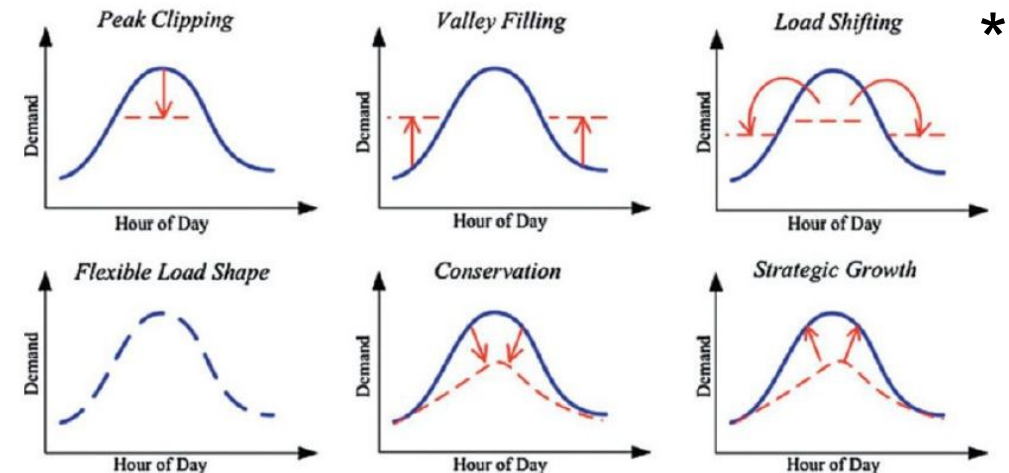
** Singla S, Keshav S. Demand response through a temperature setpoint market in Ontario. In: 2012 IEEE 3rd Int Conf Smart Grid Commun SmartGridComm 2012;

Task formulation

Among different approaches to peak load reduction we focus on peak shifting.

The thesis aims to numerically evaluate the possibility of reducing the peak for commercial buildings.

Since the major consumer is the air conditioning system, we are reducing the peak load modifying HVAC with low CAPEX.

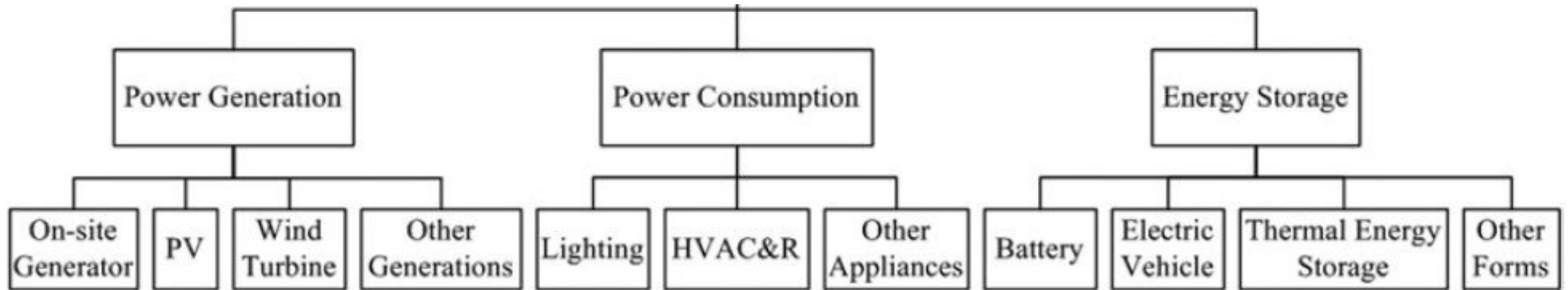


* Wang, Shengwei & Xue, Xue & Yan, Chengchu. (2014).

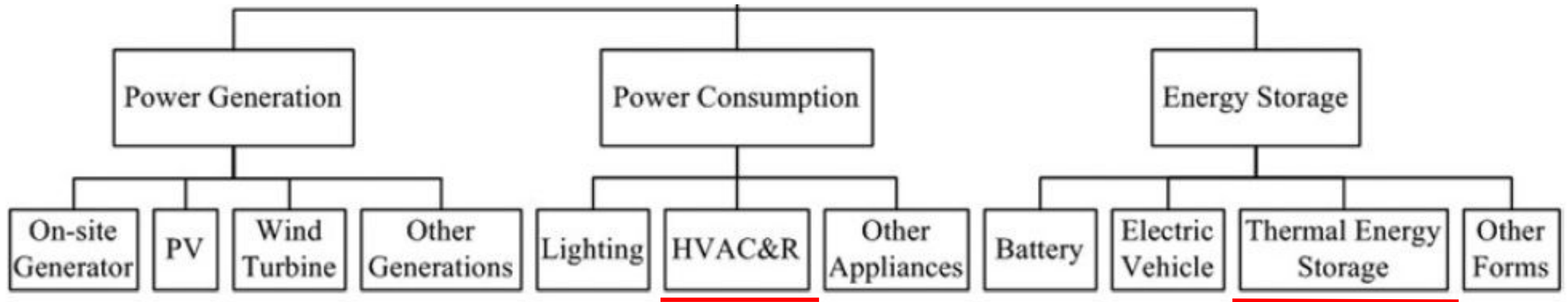
Building power demand response methods toward smart grid. HVAC&R Research

** Guide to Best Practice Maintenance and Operation of HVAC Systems for Energy Efficiency (January 2012), p. 36-37

Peak reduction from consumer side



Peak reduction from consumer side



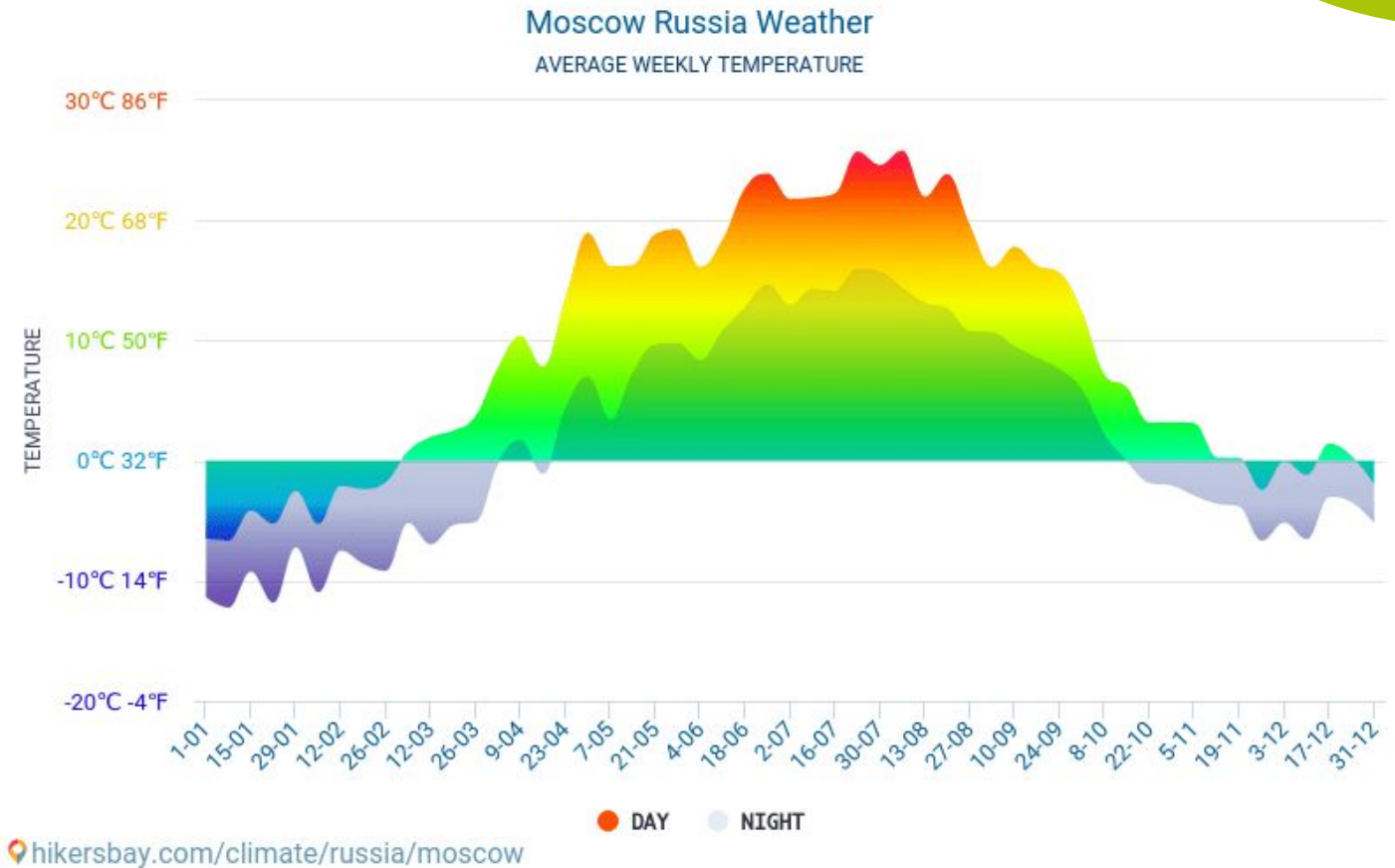


Main focus on the task:

Numerically evaluate peak load reduction
using different types of thermal energy
storage in a HVAC system of a building

The hypothesis to test:

- ★ Check the ability to store the energy at the night time from ambient cold air



Possible approaches for buildings

Control	Peak Load Reduction, %
Rule-based control [2]	up to 15%
Model predictive control [3]	up to 20%
AI based control [2]	up to 20%

Technology	Peak Load Reduction, %
Energy-efficient equipment [1]	up to 15%
Ground heat exchanger [4]	up to 20%
Electrochemical storage [5]	up to 60%
Thermal energy storage [2]	up to 60%

[1] Lee et al. (2018) Improvements to the customer baseline load

(CBL) using standard energy consumption considering energy efficiency and demand response.

[2] Wang, Shengwei & Xue, Xue & Yan, Chengchu. (2014). Building power demand response methods toward smart grid. HVAC&R Research

[3] A. Ryzhov, H. Ouerdane, E. Gryazina, A. Bischi, K. Turitsyn. (2019). Model predictive control of indoor microclimate: Existing building stock comfort improvement.

[4] B. Akhmetov, A. G. Georgiev, A. Kaltayev, A. A. Dzhomartov, R. Popov M. S. Tungatarova. (2016). Thermal energy storage systems – review.

[5] He, G., Chen, Q., Kang, C., Pinson, P., & Xia, Q. (2016). Optimal bidding strategy of battery storage in power markets considering performance based regulation and battery cycle life.

Thermal energy storages

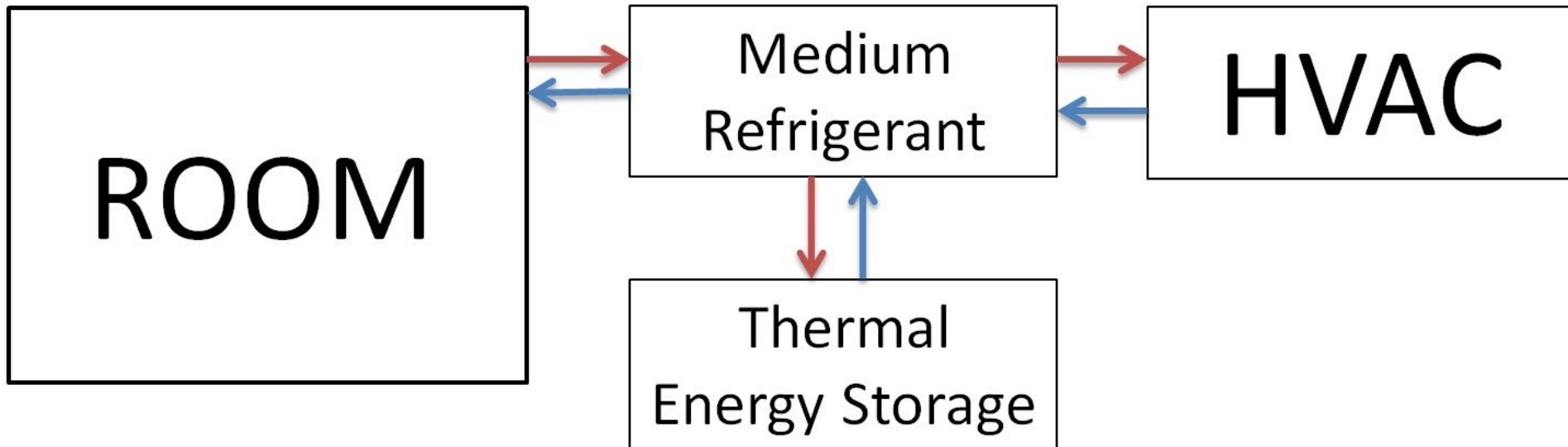
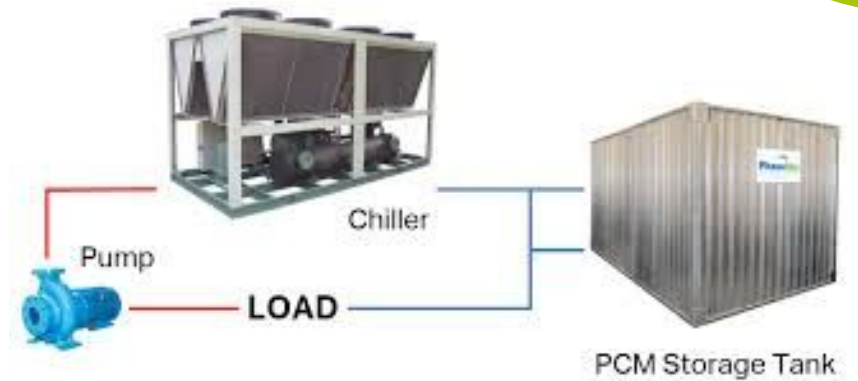
Water storage



Ice storage



PCM storage



Microclimate model

Heat balance equation:

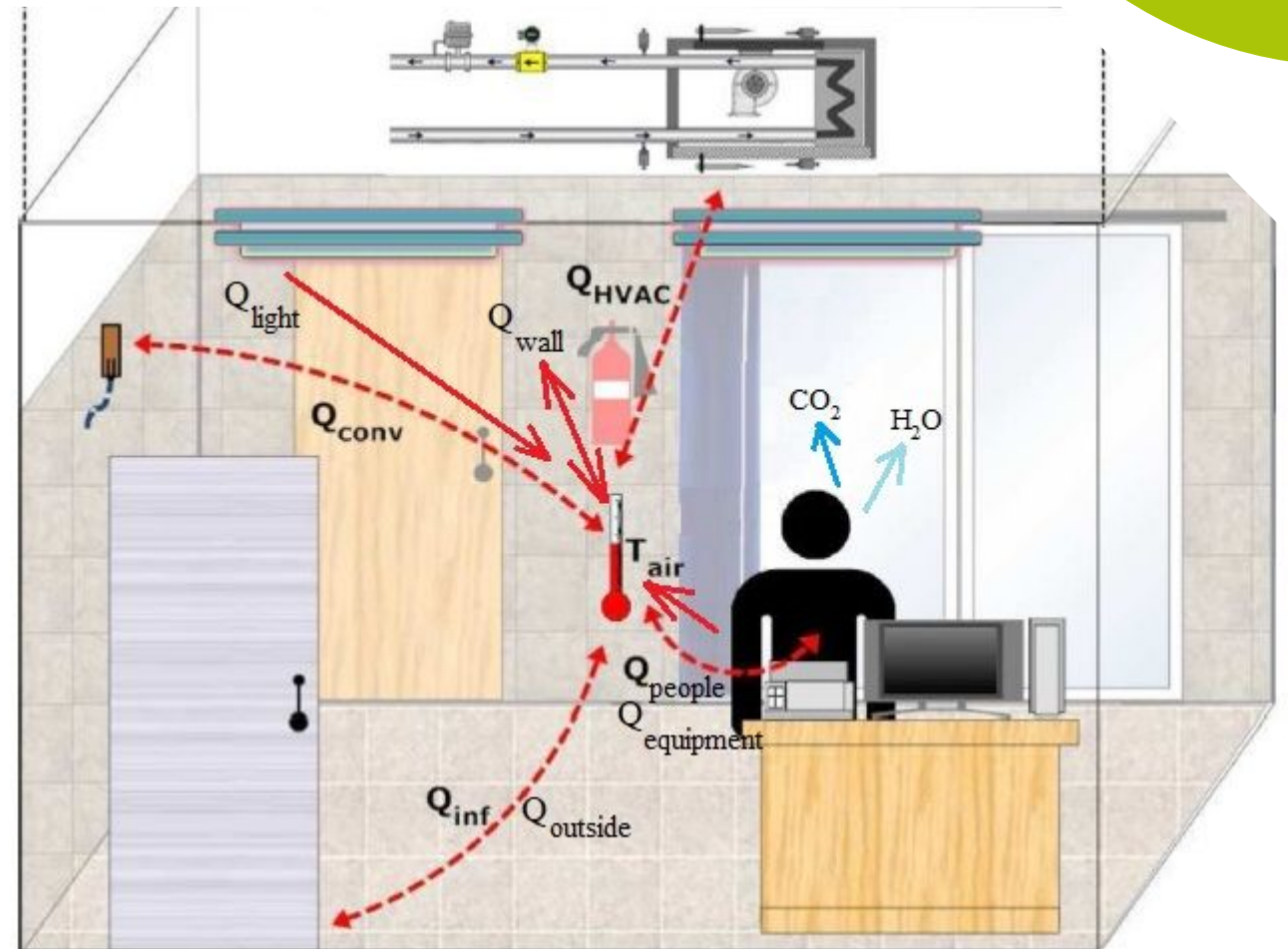
$$-Q_{\text{HVAC}} - Q_{\text{wall}} - Q_{\text{inf}} + Q_{\text{light}} + Q_{\text{people}} + Q_{\text{equip}} + Q_{\text{vent}} + \Delta Q_{\text{air}} = 0;$$

Heat flux:

$$Q_{\text{flux}} = k \cdot \Delta T = (1/R)(T_{\text{in}} - T_{\text{out}})$$

Human activity:

$$Q_{\text{people}} = M_{\text{met}} \cdot N_{\text{oc}}$$



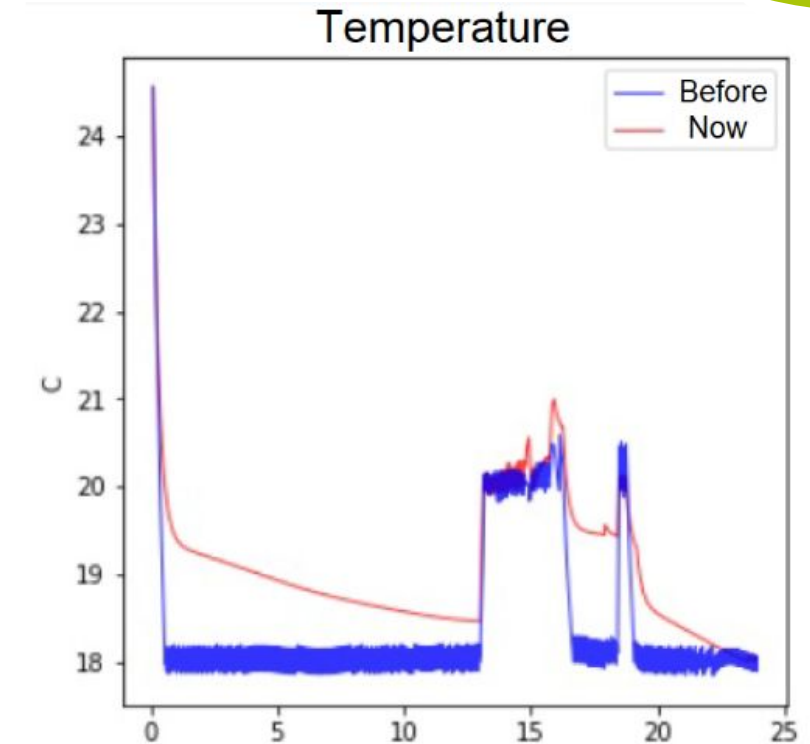
* A. Ryzhov, H. Ouerdane, E. Gryazina, A. Bisch, K. Turitsyn. (2018). Model predictive control of indoor microclimate: Existing building stock comfort improvement.

Microclimate model modification

- Before(*):
 - comfort parameters: T, CO₂;
 - constant heat transfer coefficients;
 - wall temperature is constant;
 - people are heat and CO₂ sources;

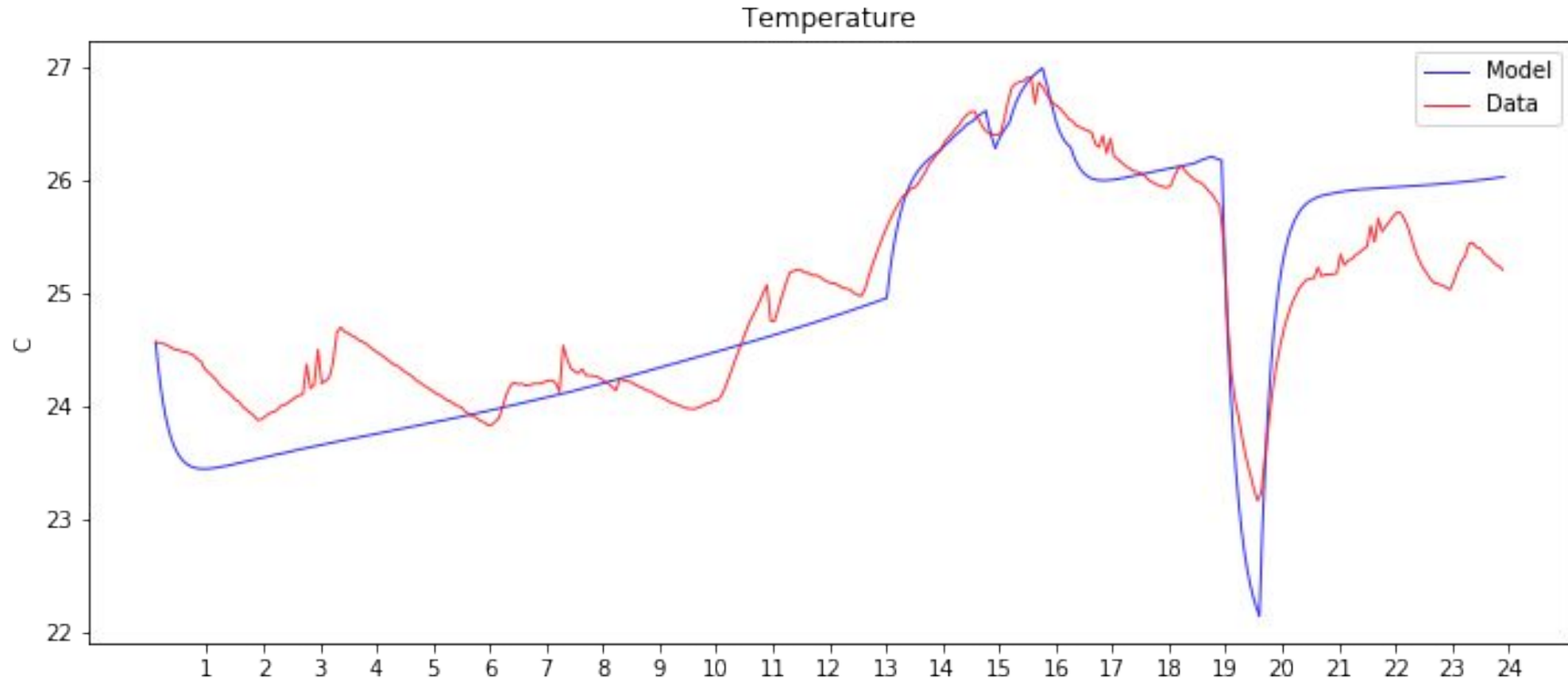
Microclimate model modification

- Before(*):
 - comfort parameters: T, CO2;
 - constant heat transfer coefficients;
 - wall temperature is constant;
 - people are heat and CO2 sources;
- Now:
 - comfort parameters: T, CO2, humidity;
 - heat transfer coefficients vary;
 - wall temperature is introduced;
 - working equipment and lighting are assumed to emit heat
 - people are also moisture sources.
 - added thermal energy storage to HVAC



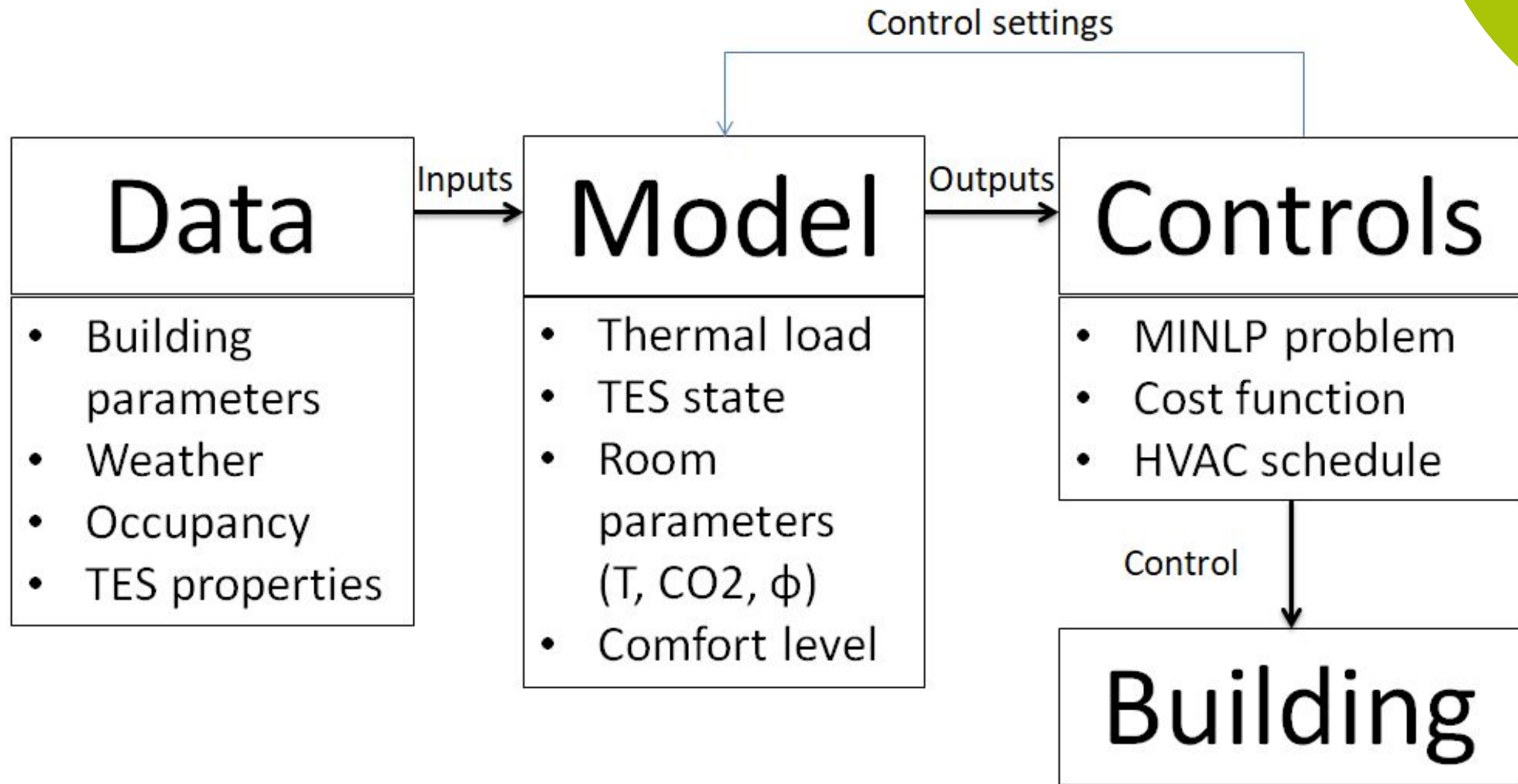
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Microclimate model validation



Maximal average error in the worst case is equal to 4.23% (~1.15 °C)

Control scheme



Optimization problem

$$\min \sum_t c_t(P_t^c + P_t^f), \forall t \in \mathbb{T}$$

objective function - bills

Optimization problem

$$\min \sum_t c_t(P_t^c + P_t^f), \forall t \in \mathbb{T}$$

$$\text{s.t. } W_t^{HVAC} = COP \cdot P_t^c + W_t^{dis} - W_t^{ch}$$

$$Q_t = 0.55 \frac{P_t^f}{100}$$

objective function - bills

HVAC system cooling function

Optimization problem

$$\min \sum_t c_t(P_t^c + P_t^f), \forall t \in \mathbb{T}$$

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$$Q_t = 0.55 \frac{P_t^f}{100}$$

$$E_{t+1} = E_t - \tau(W_t^{dis} + W_t^{ch})$$

$$W_t^{ch} = W_t^{HVAC}, t \in [00.00 - 07.00]$$

$$W_t^{dis} = C_{out} Q_t (T_{input} - T_{output}), t \in [9.00 - 18.00]$$

objective function - bills

HVAC system cooling function

TES functions

Optimization problem

$$\min \sum_t c_t(P_t^c + P_t^f), \forall t \in \mathbb{T}$$

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$$T_{t+1}^r = T_t^r + p_{a1}((p_{a2} + p_{a4})(T_t^{out} - T_t^r) + p_{a3}(T_t^w - T_t^r) + p_{a5}N_t^{oc} - W_t^{HVAC} + C_{out}T_{vent}Q_t - C_{in}T_t^rQ_t + W_{light})$$

$$T_{t+1}^w = T_t^w + p_{w1}p_{w2}(T_t^r - T_t^w)$$

$$n_{t+1}^{CO_2} = n_t^{CO_2} + p_{n1}((Q_t + p_{n2})(300 - n_t^{CO_2}) + k_{CO_2}N_t^{oc})$$

objective function - bills

HVAC system cooling function

TES functions

Room parameters functions

Optimization problem

$$\min \sum_t c_t(P_t^c + P_t^f), \forall t \in \mathbb{T}$$

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$$n_{t+1}^{CO_2} = n_t^{CO_2} + p_{n1}((Q_t + p_{n2})(300 - n_t^{CO_2}) + k_{CO_2}N_t^{oc})$$

$$W_t^{HVAC} \geq 0$$

$$0 \leq E_t \leq E_{max}$$

$$P_t^c = \{0, 500, 1000, 1500\}$$

$$P_t^f = \{0, 10, 20, 30, 40\}$$

objective function - bills

HVAC system cooling function

TES functions

Room parameters functions

HVAC and TES constraints

Discrete regulation constraint

Optimization problem

$$\min \sum_t c_t(P_t^c + P_t^f), \forall t \in \mathbb{T}$$

$$\text{s.t. } W_t^{HVAC} = COP \cdot P_t^c + W_t^{dis} - W_t^{ch}$$

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$$W_t^{HVAC} \geq 0$$

$$0 \leq E_t \leq E_{max}$$

$$P_t^c = \{0, 500, 1000, 1500\}$$

$$P_t^f = \{0, 10, 20, 30, 40\}$$

$$18^\circ C \leq T_t^r \leq 24^\circ C$$

$$0 \leq n_t^{CO_2} \leq 800$$

objective function - bills

HVAC system cooling function

TES functions

Room parameters functions

HVAC and TES constraints

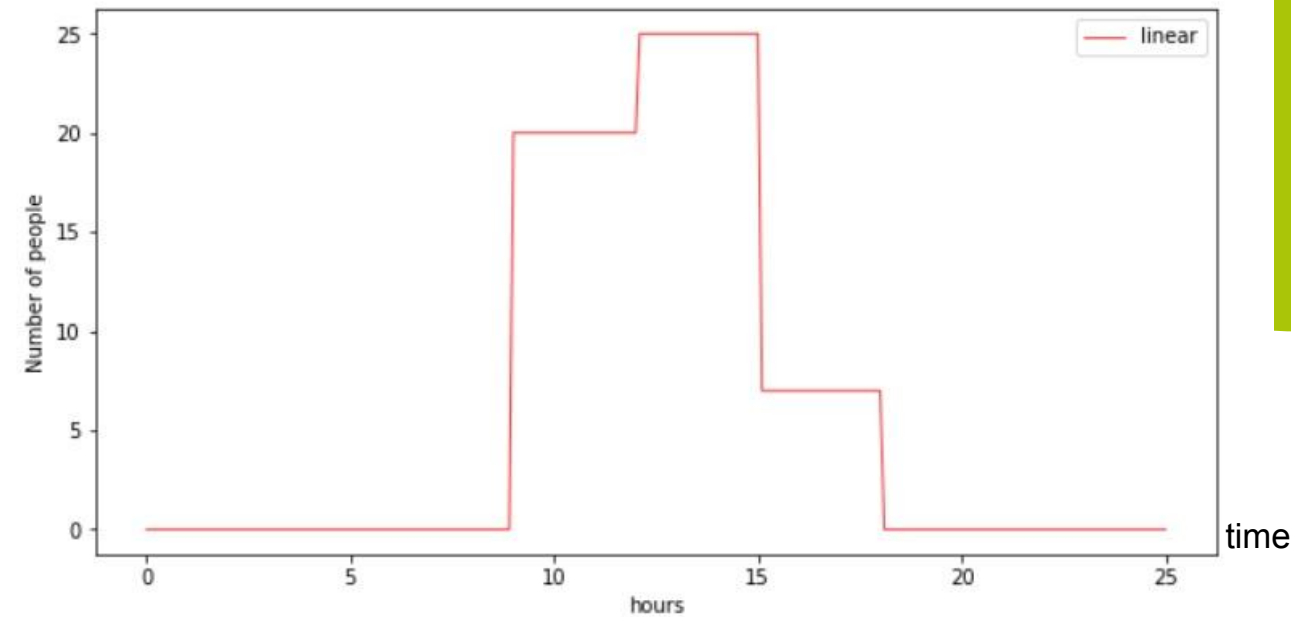
Discrete regulation constraint

Comfort conditions constraint

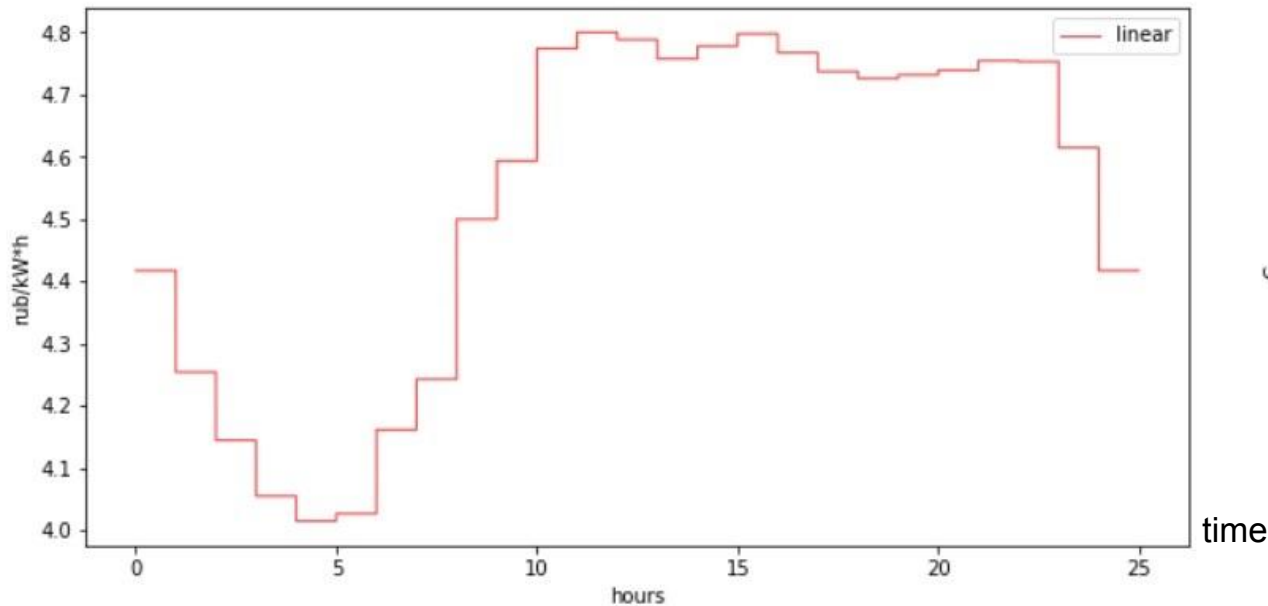
Results - Input Data

- Occupancy
- Price
- Ambient temperature

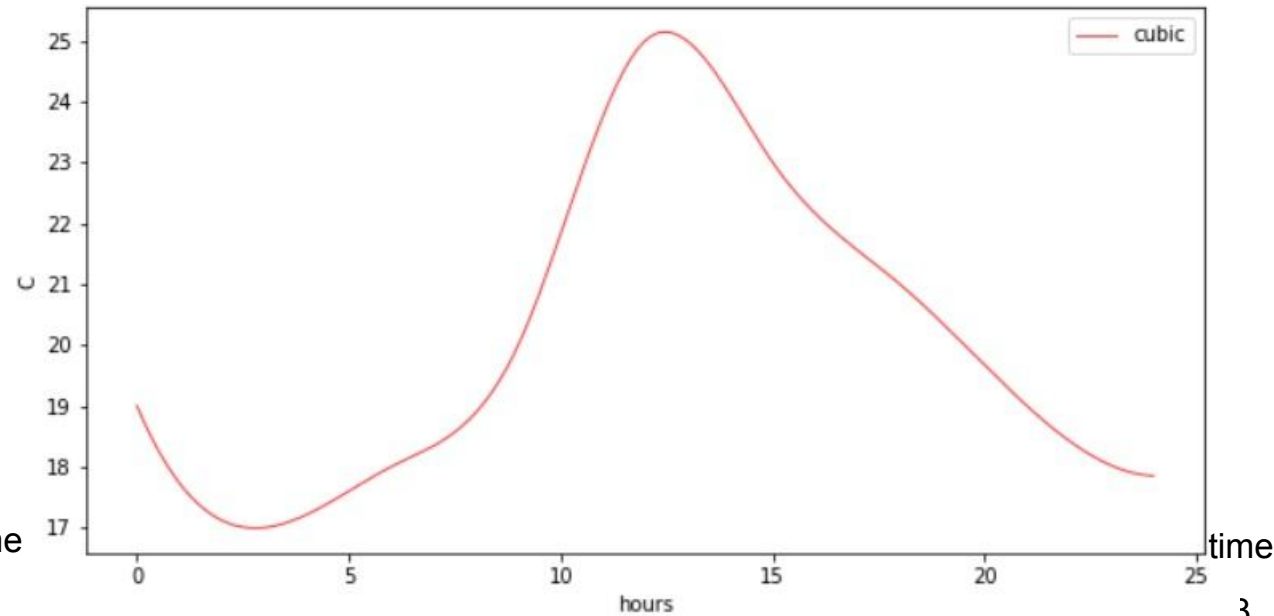
Occupancy



Price



Ambient temperature

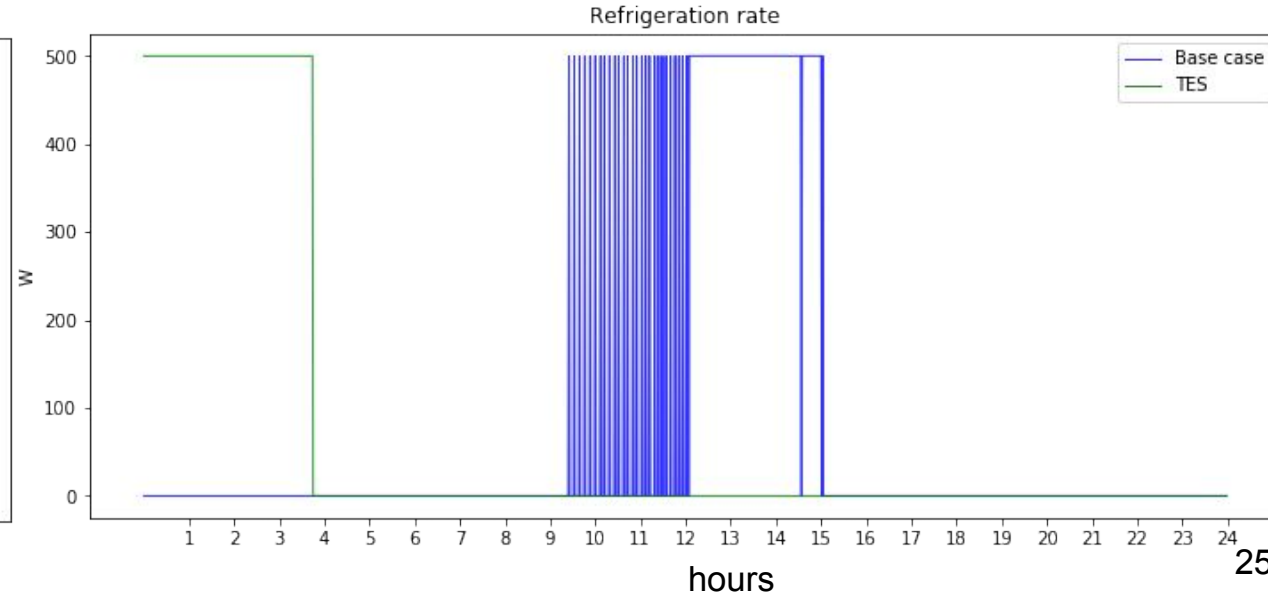
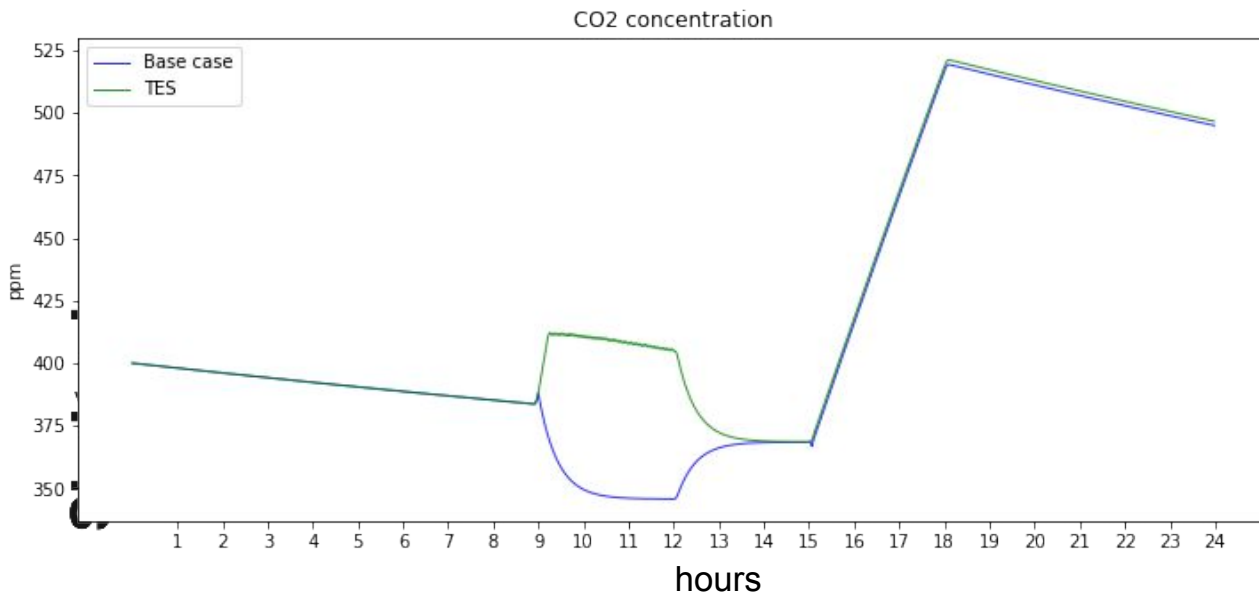
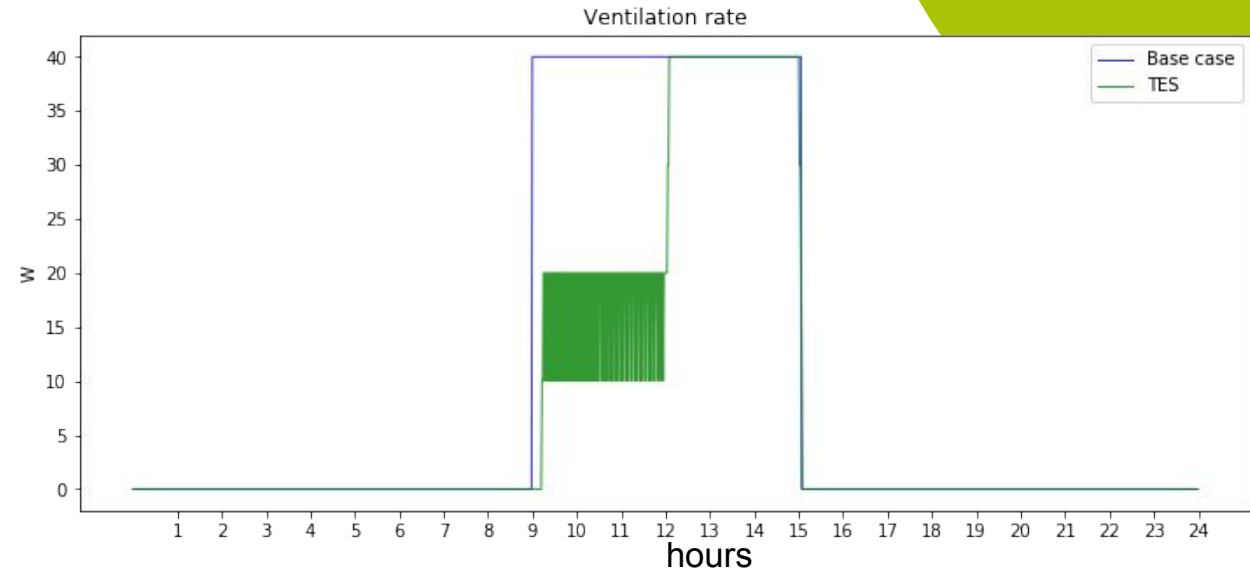
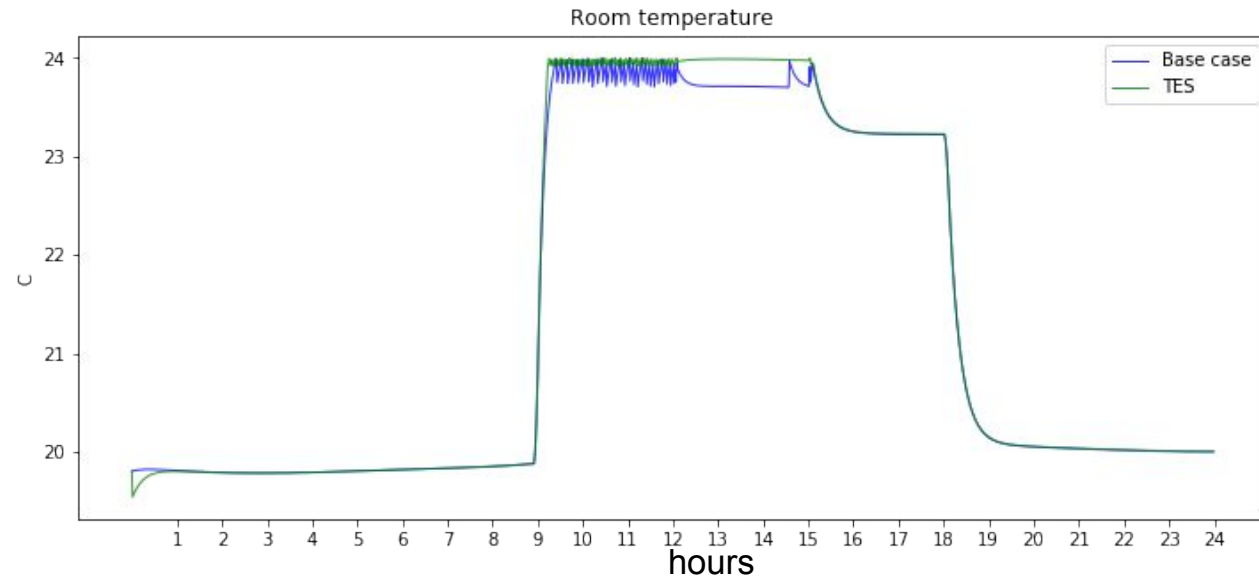


Results - comparison of Base case and TES case

- Base case - Opt. problem for usual HVAC system
- TES case - Opt. problem for modified HVAC system (TES included)
 - In the TES case, 100% of the compressor's power consumption has been transferred from the peak time to the night time.
 - In comparison with Base case, there is a reduction in electricity costs in the amount of 6.6%

Type of run	Bills, rub	Consumption, kW*h	Time, sec
Base MPC	9.3	1.9	76
MPC using TES	8.7	2.0	638

Results - comparison of Base case and TES case



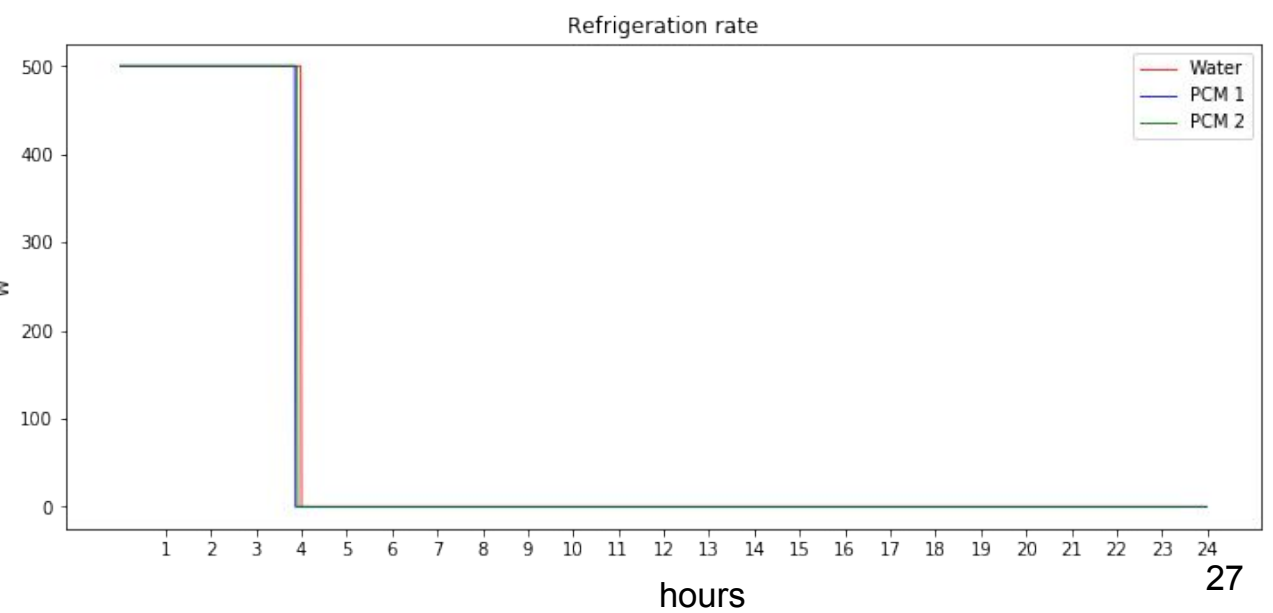
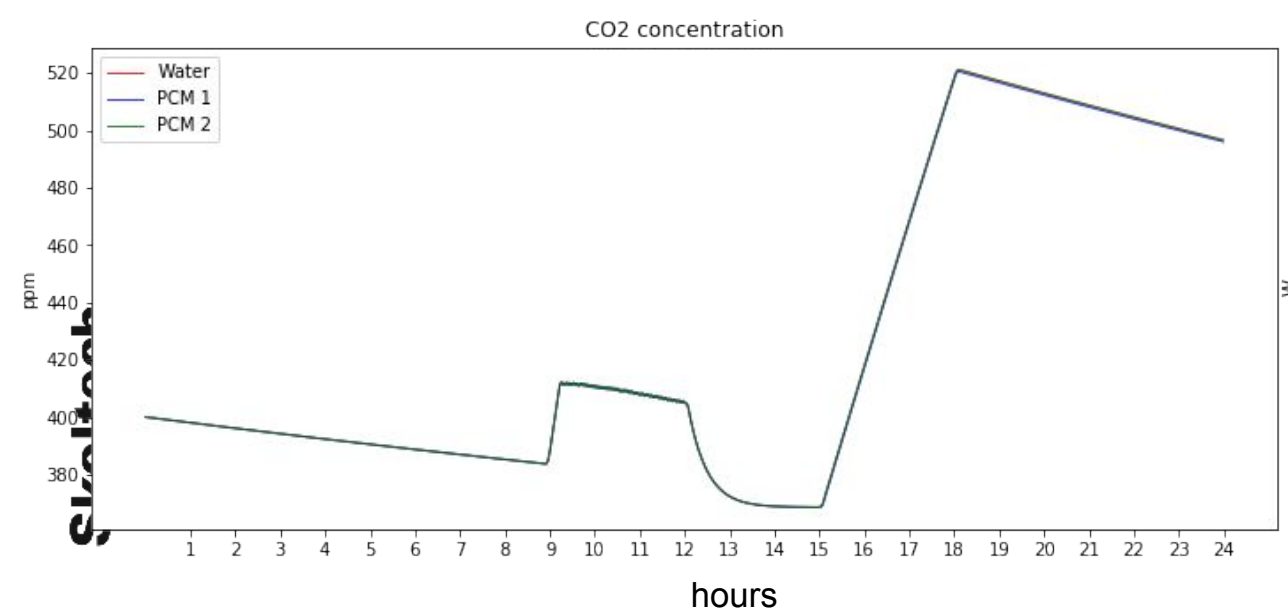
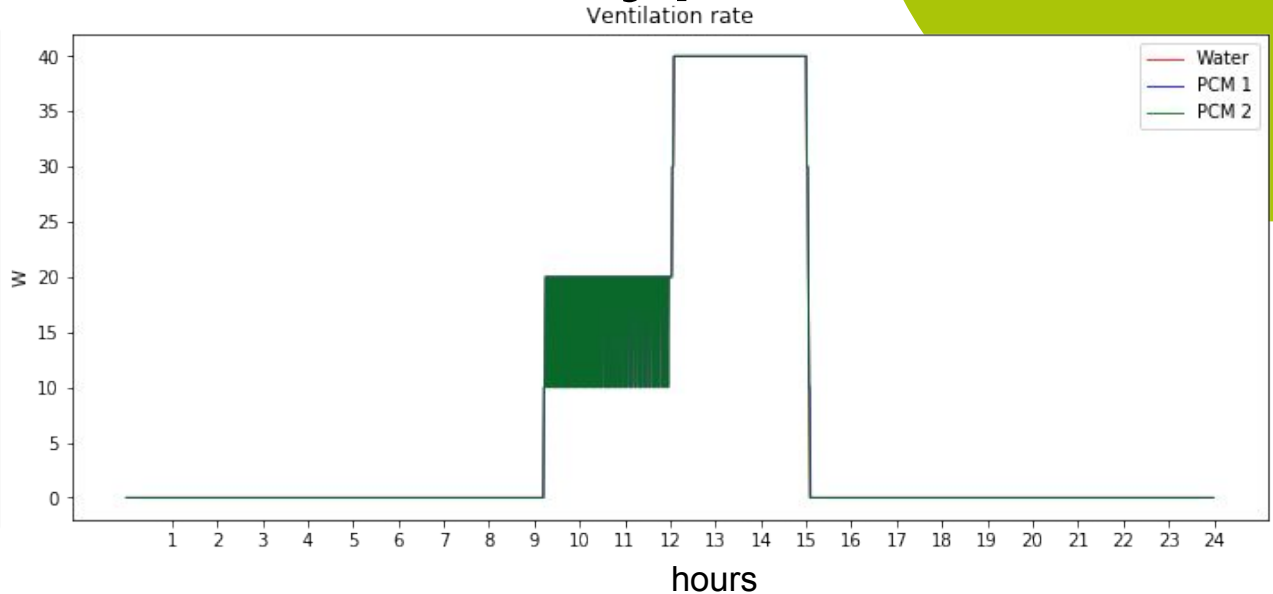
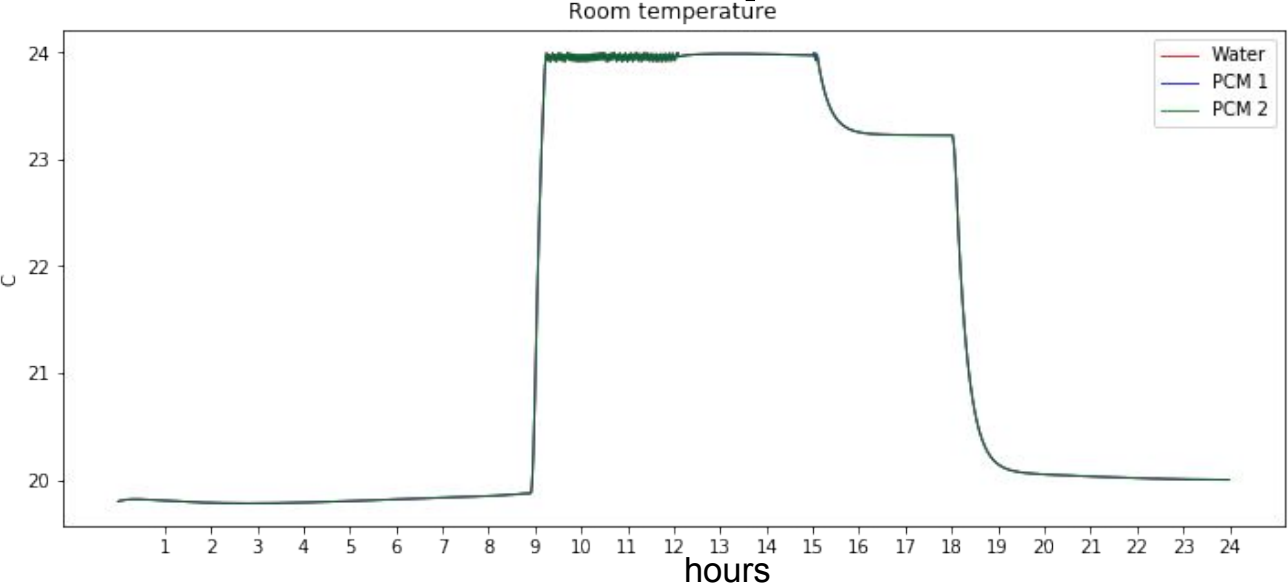
Results - comparison of different fluid types

Opt. problem is formulated for usual HVAC system including TES

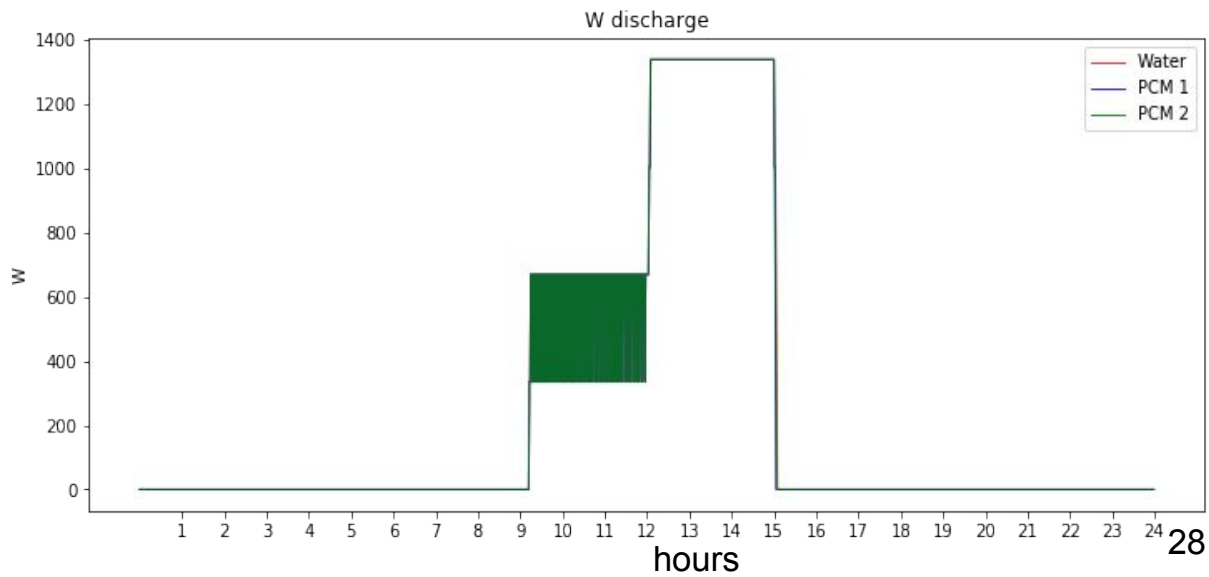
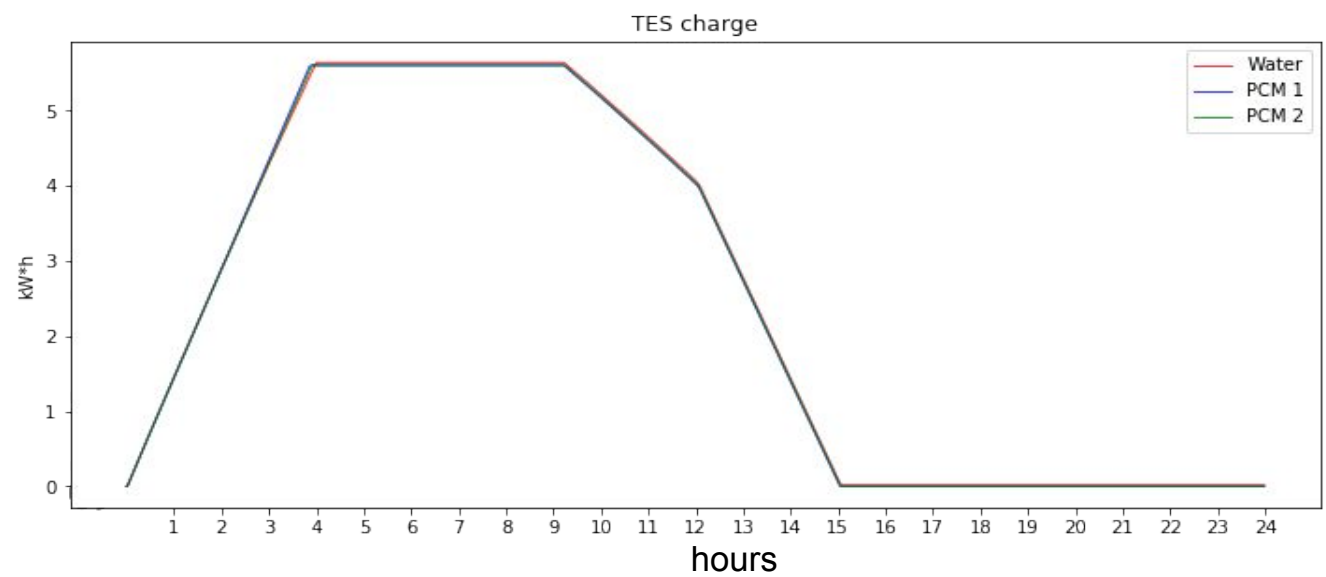
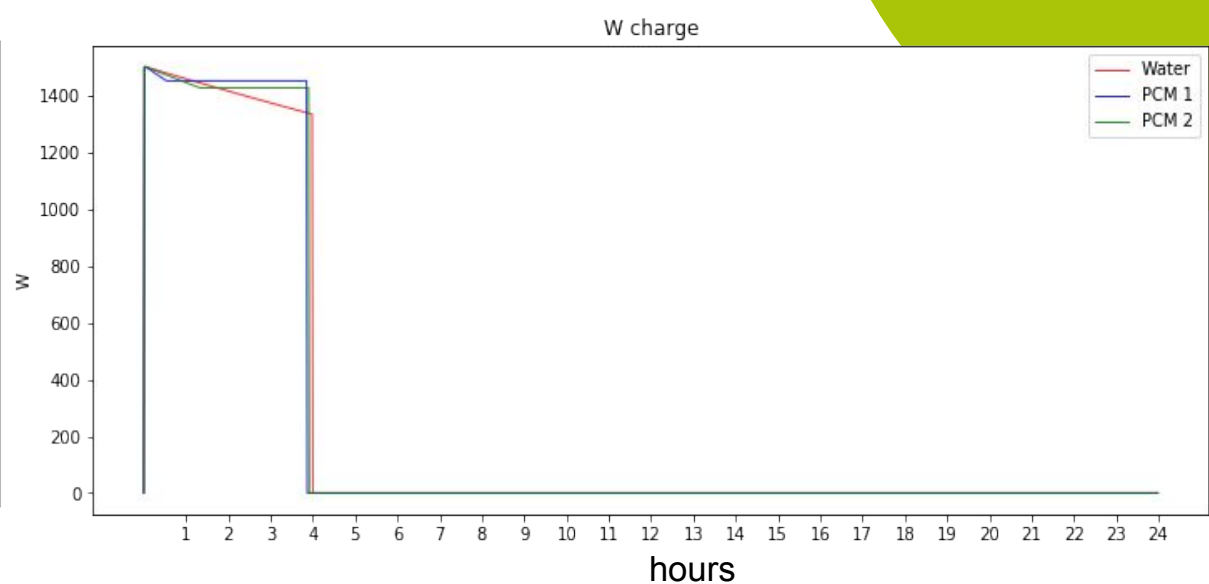
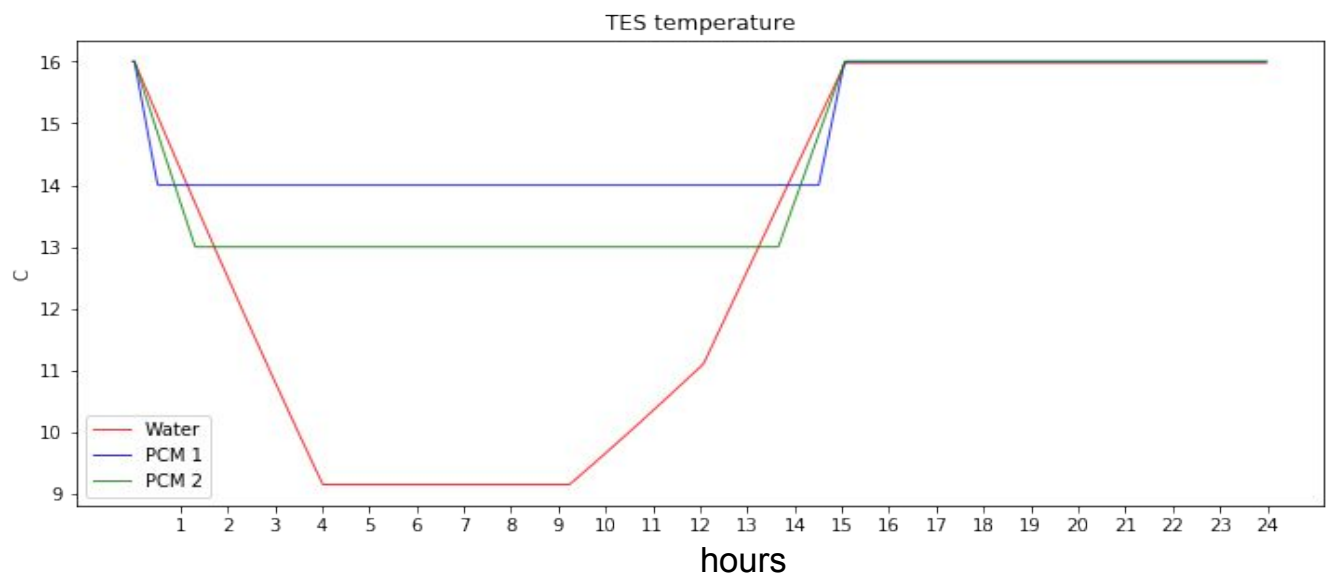
- Water - Distilled water, $T_m = 0\text{ }^{\circ}\text{C}$;
 - PCM1 - Dipotassium phosphate hexahydrate, $T_m = 14\text{ }^{\circ}\text{C}$;
 - PCM2 - Salt hydrate S13 ("PCM Products"), $T_m = 13\text{ }^{\circ}\text{C}$.
- There is low difference in costs
 - Visible differences in the graphs are only in the temperature of the working substances

Type of run	Bills, rub	Consumption, kW*h	Solve time, sec
TES water/ice	9.0	2.2	687
TES PCM 1	8.9	2.1	726
TES PCM 2	8.9	2.1	689

Results - comparison of different fluid types



Results - comparison of different fluid types



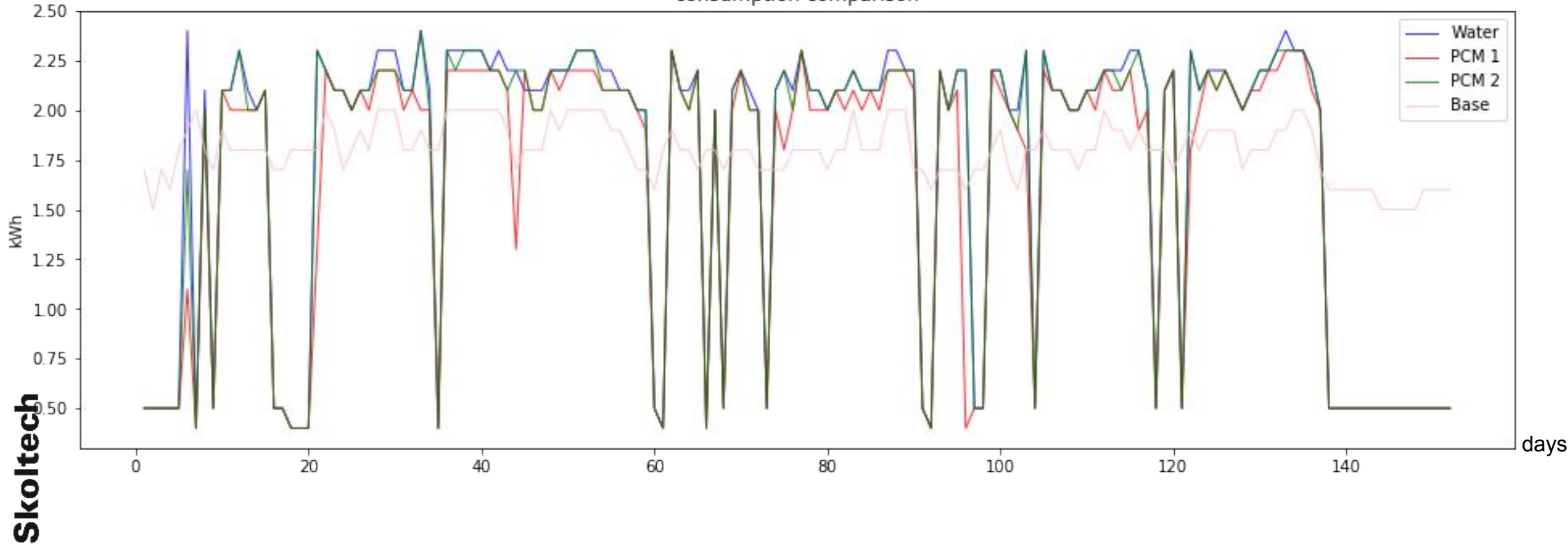
Results - comparison of different fluid types in the long-term during 153 days

- Period - May 1, 2019, to September 30, 2019
- Place - Moscow
- There is significant difference in total costs
- The hypothesis of charging TES from night cold air works
- Melting temperature affects seasonal reductions in air conditioning costs

Type of run	Bills, rub	Consumption, kW*h
Base MPC	1293	271.6
TES water/ice	1108	260.7
TES PCM 1	1072	251.2
TES PCM 2	1085	258.8

Results - comparison of different fluid types in the long-term during 153 days

consumption



Conclusions

- Technology review demonstrates that PCMs is the most widely spread fluid for TES used in HVAC systems;
- Complex thermodynamic room model was created;
- TES implemented to the MPC optimization problem formulation;
- Hypothesis with charging thermal energy storage at night from ambient is working and useful;
- TES usage allow us to reduce bills up to 6 % at constant COP;
- PCM1 ($K_2HPO_4(H_2O)_6$) with the melting temperature $T_m = 14\text{ }^{\circ}\text{C}$ shown the best efficiency in the long-term period during 153 days;

Research plans

- ✓ Analyse the international experience in peak load reduction;
- ✓ Estimate the potential of existing technologies;
- ✓ Explore real TES implementations;
- ✓ Modificate the room model and validate it;
- ✓ Integrate thermal energy storage (TES) to the model;
- ✓ Formulate the Optimization Problem for current task;
- ✓ Make simulations and analysis of results.

Acknowledgements

- ★ Prof. Elena Gryazina, Skoltech
- ★ Aleksandr Baluev, TION



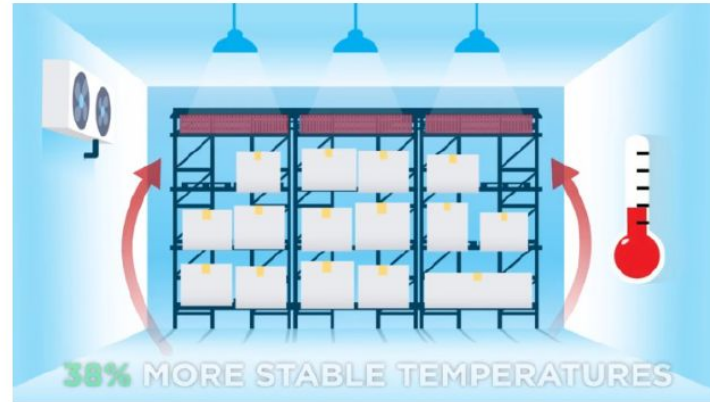
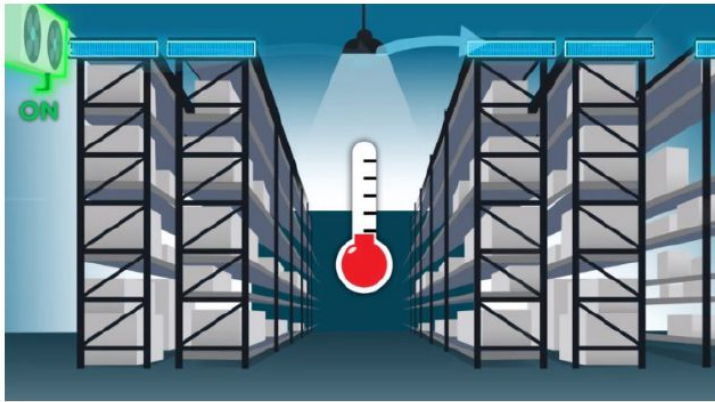
THANK YOU FOR
YOUR ATTENTION!

TES applications in building scale

- Thermal comfort maintaining with lower fluctuations;
- Emergency water supply in case with water TES;
- Possibility in Demand Response programs participation;
- Increase of building demand flexibility;
- Reduction of peak load consumption -> decreased bills;
- Possibility to charge TES in mid-season by ambient temperature.

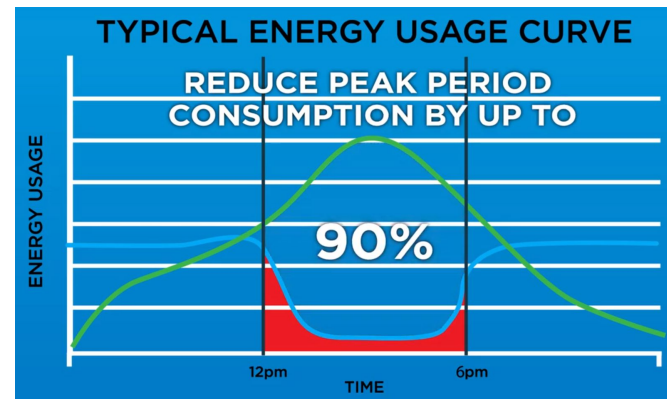
PCM usage in industry

VIKING COLD Solutions

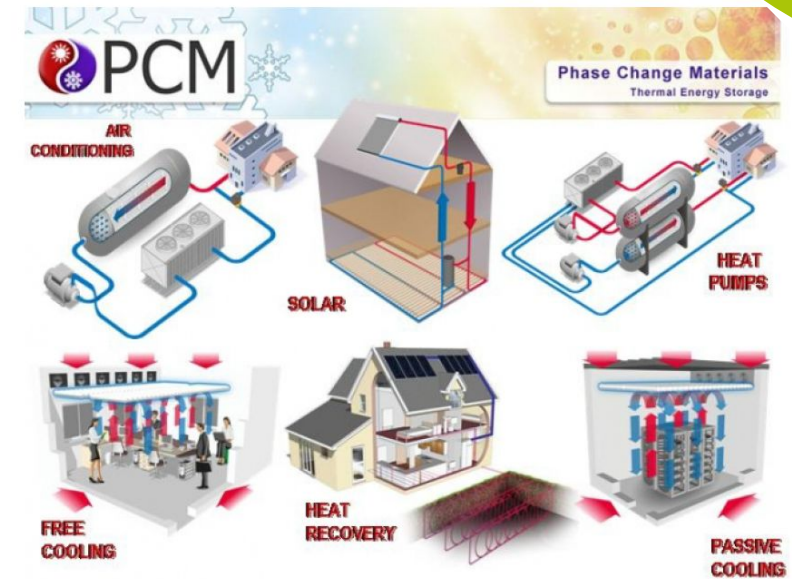


This company mainly operates in the market for food storage. Using of PCM materials allows to customers to reduce peak loads up to 90%.

Skoltech



PCM Products Limited

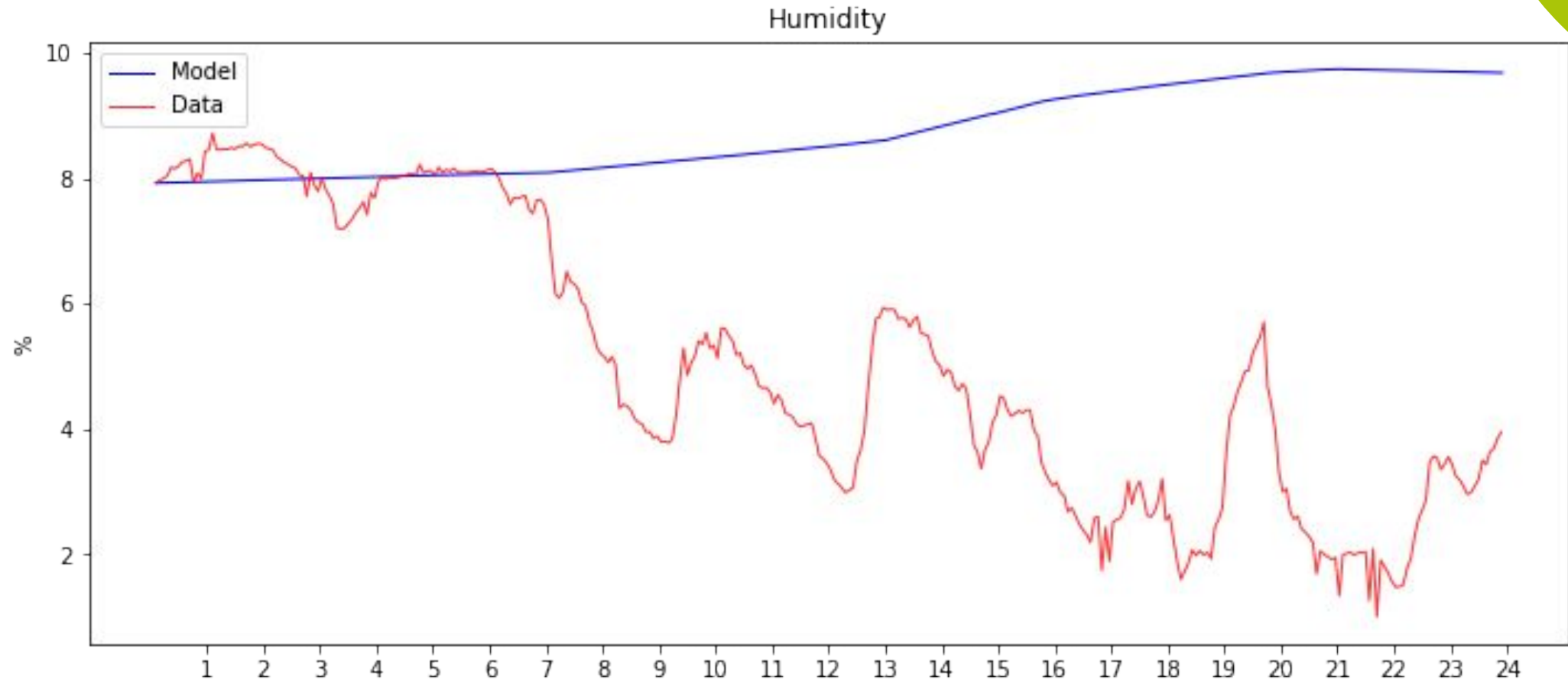


The company is a leader in the application of PCM in HVAC systems. Examples include University of Bergen, Norway; Australian parliament house, Canberra, Australia and other.

Sensitivity analysis

Input data	Input data change, %	Bills shift, %	Consumption shift, %
Occupancy	+ 5	+ 7	+ 9.5
Occupancy	- 5	- 17	- 16
Out temperature	+ 5	+ 1	+ 5
Out temperature	- 5	+ 1	0
Price	+ 5	0	0
Price	- 5	0	0
M_{met}	+ 5	+ 8	+ 9.5
M_{met}	- 5	- 18	- 15.5

Microclimate model validation



Unusual sensor readings forced us not to use humidity as a parameter

Solving algorithm

- MINLP problem solver (Juniper) did not converge for 24 hours;
- P^f variable specification as input gives us NLP problem simplification to simple LP, but there was permissible error equal to 0.3%;
- MILP problem solvers (Cbc, GLPK) did not converge for 6 hours;
- This forced us to write our own algorithm with assumptions for solving this particular problem.

Solver	Problem	Bills, rub	Consumption	Error, %
JuMP	NLP	8.14	1.7	base
CVXPY (P^f specified)	LP	8.16	1.71	0.3
A-solver (P^f specified)	MILP	8.18	1.7	0.5

What if all the customers will use TES?

- The total consumption will remain the same, since using TES we primarily reduce the load of the air conditioning system;
- The peak load will shift to the evening hours, which will reduce the daily load on generators and avoid short-term use of expensive generators - baseline takes the form of a plateau (ideal);
- Consumers will get reduced payments and lower peak prices, and generation companies will rid from the need to maintain expensive equipment;

TES functions

$$W_t^{ch} = \begin{cases} W, & \text{if } T_t^{out} \geq T_{melt} - 5^\circ C \\ C_{out}Q(T_t^{out} - T_{melt}), & \text{if } T_t^{out} < T_{melt} - 5^\circ C \end{cases}$$

$$W_t^{dis} = C_{out}Q(T_{input} - T_{output})$$

$$T_t^{TES} = \begin{cases} T_{TES_0} - E_t / (m_{TES} C_{TES_l}), & \text{if } E_t < m_{TES} C_{TES_l} (T_{TES_0} - T_{melt})^{(*)} \\ T_{melt}, & \text{if } (*) \leq E_t \leq (*) + m_{TES} L_{TES} \\ T_{melt} - (E_t - (*) - m_{TES} L_{TES}) / (m_{TES} C_{TES_s}), & \text{if } E_t \geq (*) + m_{TES} L_{TES} \end{cases}$$

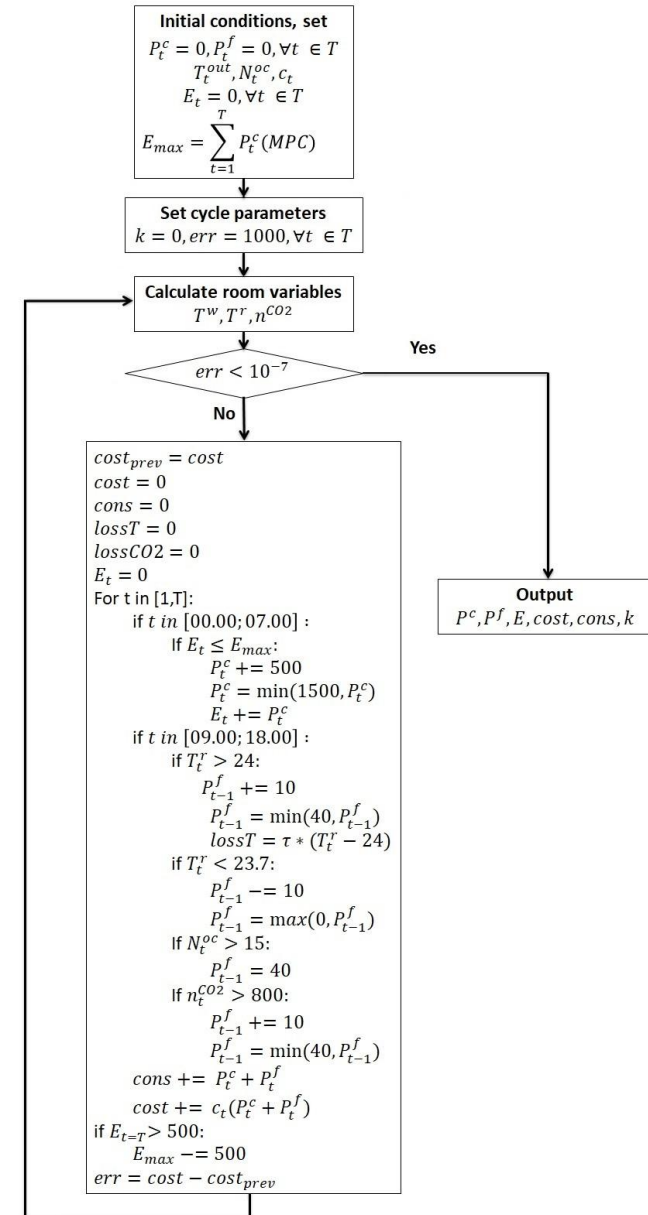
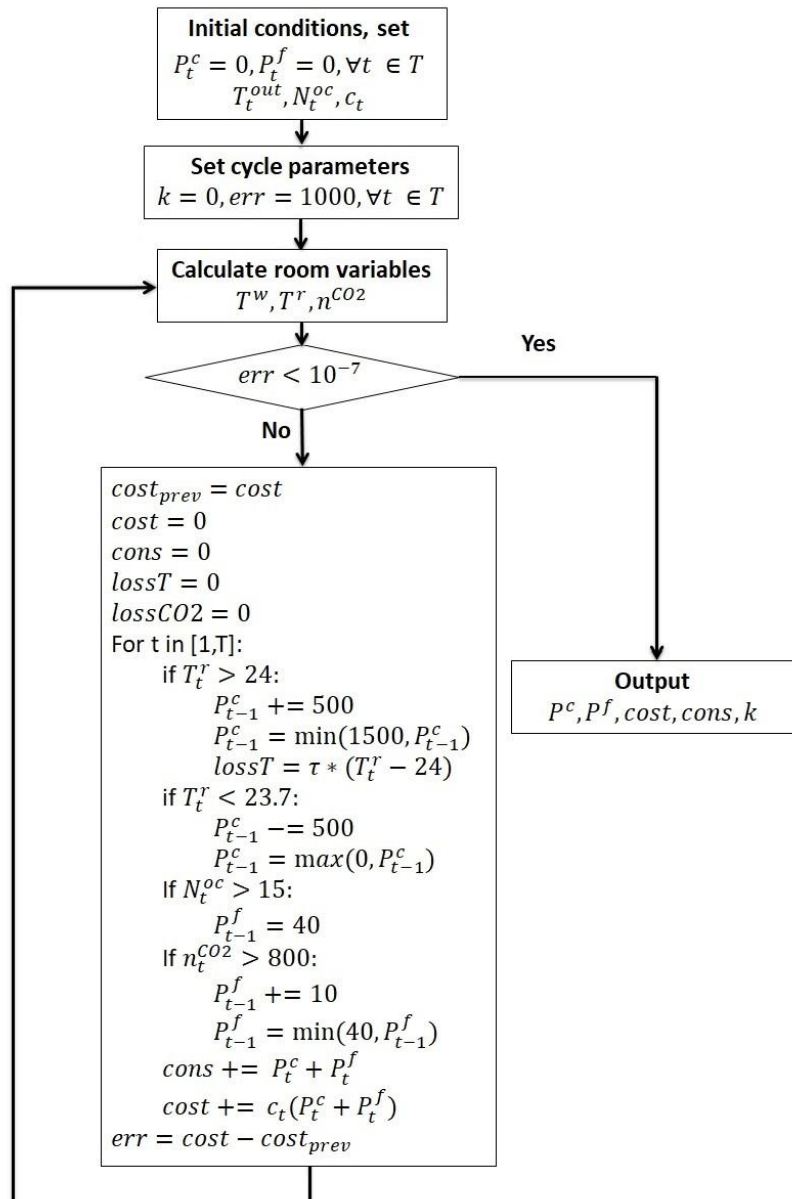
Room functions

$$\begin{aligned}
 T_{t+1}^r = & T_t^r + \frac{\tau}{m_{air}C_{in}} \cdot (h \cdot b \cdot k(T_t^{out} - T_t^r) + \\
 & + 0.5\alpha_{int}h(S/b + b) \cdot (T_t^w - T_t^r) + M_{met}N_t^{oc} - (COP P_t^c + W_t^{dis}) + \\
 & + C_{out}Q_tT_{vent} - C_{in}Q_tT_t^r + a_{inf}m_{air}C_{out} \cdot (T_t^{out} - T_t^r) + \\
 & + N_{light}M_{light} - L_{evap}M_{moisture}N_t^{oc}), t \in \mathbb{T}
 \end{aligned}$$

$$T_{t+1}^w = T_t^w + 0.5 \frac{\tau}{m_{wall}C_{wall}} \alpha_{int}h(S/b + b) \cdot (T_t^r - T_t^w), t \in \mathbb{T}$$

$$n_{t+1}^{CO2} = n_t^{CO2} + \frac{\tau}{m_{air}} (Q_t(300 - n_t^{CO2}) + a_{inf}m_{air}(300 - n_t^{CO2}) + k_{CO2}N_t^{oc}), t \in \mathbb{T} \quad (3.19)$$

Proposed algorithms



Methods used in Thesis

- Power Markets theory in demand-supply
- Mathematical modeling
- Application of heat-transfer theory
- Formulation of Optimization Problem
- Numerical calculations in Python

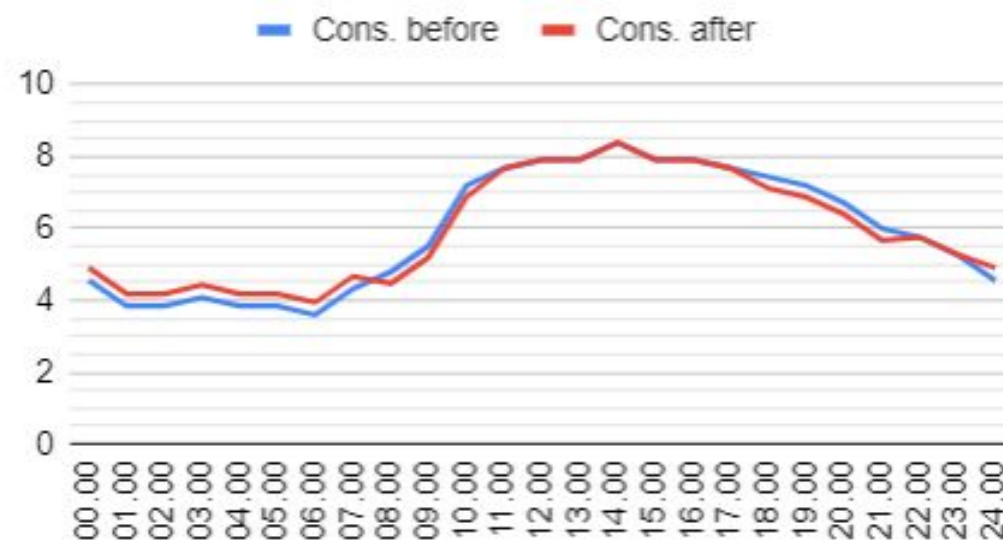
Benefits of using electrochemical energy storage devices

Commercial usage

Capacity = 0,7 MWh
Price = 25 mln. rub
Month save = - 8k rub (-0,04%)
ROI = No pay off

Aggregation usage

Daily consumption



Capacity = 0,7 MWh
Price = 25 mln. rub
Month save = 417k rub (2,1%)
ROI = 5 years
(stated payout = 600k rub/MW)