

→ "Attitude determination"

**ALGEBRAIC-METHOD**

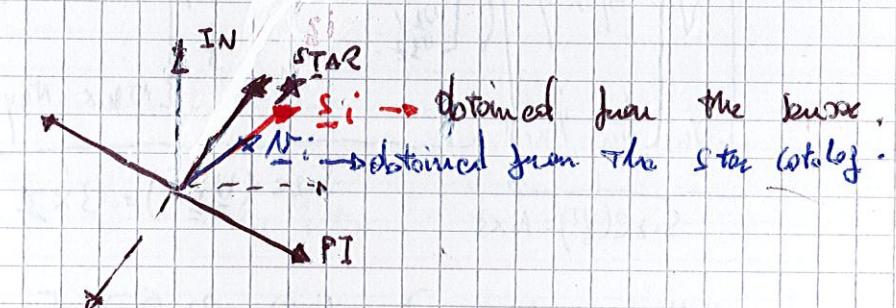
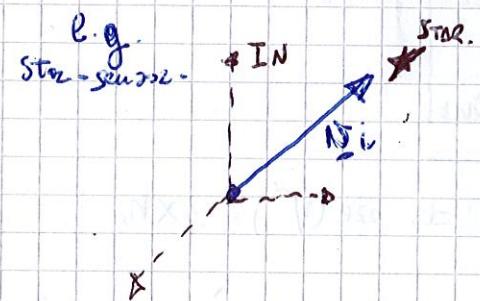
\* Having obtained  $N$  measurements of  $N$  celestial objects:  
 $\left\{ \begin{array}{l} \underline{s}_1 \\ \underline{s}_2 \\ \vdots \\ \underline{s}_N \end{array} \right\}$   
 !! in the P.I. r.f. !!

\* Known the position of such  $N$  objects in the P.J.N. r.f.:  
 $\left\{ \begin{array}{l} \underline{n}_1 \\ \underline{n}_2 \\ \vdots \\ \underline{n}_N \end{array} \right\}$   
 !!

is then possible to determine (fully determine)  
 the attitude of the spacecraft.

$N=3$  Measurements

$$A_i : \underline{s}_i = \underline{A} \cdot \underline{n}_i$$



⇒ Having 3 measurements available is possible to build  
 the 2 matrices:

$$\underline{S} = [ \underline{s}_1 | \underline{s}_2 | \underline{s}_3 ] ; \underline{V} = [ \underline{n}_1 | \underline{n}_2 | \underline{n}_3 ]$$

$$\underline{A} : \underline{s} = \underline{A} \cdot \underline{V} \Rightarrow \text{if } \underline{V} \text{ is NOT SINGULAR} \\ \text{THEN } \exists \underline{V}^{-1}$$

$$\Rightarrow \Gamma^{N=3} \\ \underline{A} = \underline{S} \cdot \underline{V}^{-1}$$

### N > 3 measurements.

The problem will be:  $\underline{S} = \underline{A} \underline{N}$  [  $s_1, s_2, s_3, s_4, \dots$  ]

$$\underline{S} = \underline{A} \underline{N} \quad \text{Size } (\underline{S}) = 3 \times n \quad \text{Size } (\underline{A}) = 3 \times 3 \quad \text{Size } (\underline{N}) = 3 \times n$$

!! Such problem is redundant therefore:

is NECESSARY to use  $\underline{V}^*$  ( $\underline{V}$ -pseudo inverse)  
INSTEAD OF  $\underline{V}^{-1}$  ( $\underline{V}$ -inverse) !!

→ Pseudo inverse matrix.

$$\underline{V}^* \triangleq (\underline{V}^T) (\underline{V} \underline{V}^T)^{-1}$$

$$\begin{bmatrix} N_{1x}, N_{1y}, N_{1z} \\ \vdots \\ N_{nx}, N_{ny}, N_{nz} \end{bmatrix} \left( \begin{bmatrix} N_{1x} & \dots & N_{nx} \\ N_{1y} & \dots & N_{ny} \\ N_{1z} & \dots & N_{nz} \end{bmatrix} \cdot \begin{bmatrix} V_{1x}, V_{2y}, V_{3z} \\ \vdots \\ V_{nx}, V_{ny}, V_{nz} \end{bmatrix} \right)^{-1} = \underline{V}^*$$

Size  $(\underline{V} \underline{V}^T) = 3 \times 3$

$$\begin{bmatrix} N_{1x} & N_{1y} & N_{1z} \\ \vdots \\ N_{nx} & N_{ny} & N_{nz} \end{bmatrix} \begin{bmatrix} ① & ② & ③ \\ \vdots & \vdots & \vdots \\ 1 & 1 & 1 \end{bmatrix} = \begin{bmatrix} \cdot & \cdot & \cdot \\ \cdot & \cdot & \cdot \\ \cdot & \cdot & \cdot \end{bmatrix} \Rightarrow \text{Size } (\underline{V}^*) = 3 \times n$$

in fact.

$$\underline{S} = \underline{A} \cdot \underline{N} \Rightarrow \underline{S} = \underline{A} \cdot \underline{V}$$

$$\underline{S} \underline{V}^T = \underline{A} (\underline{V} \underline{V}^T)$$

$$\underline{S} \underline{V}^T (\underline{V} \underline{V}^T)^{-1} = \underline{A} (\underline{V} \underline{V}^T) (\underline{V} \underline{V}^T)^{-1}$$

$$\underline{S} \underline{V}^T (\underline{V} \underline{V}^T)^{-1} = \underline{A}$$

$\underline{V}^*$

⇒  $\Gamma_{N>3}$

$$\underline{A} = \underline{S} \cdot \underline{V}^* \\ \underline{V}^* = \underline{V}^T (\underline{V} \cdot \underline{V}^T)^{-1}$$

### N=2 measurements.

The strategy is about to build a 3rd vector orthogonal to the other 2

$$s_3 \perp (s_1, s_2)$$

$$[s_1, s_2] \rightarrow [N_1, N_2]$$

↓

$$p_s = s_1$$

$$p_N = N_1$$

$$q_s = \frac{s_1 \times s_2}{\|s_1 \times s_2\|}$$

$$q_N = \frac{N_1 \times N_2}{\|N_1 \times N_2\|}$$

(!! this is possible if and only if  $s_1$  is not parallel to  $s_2$  !!)

$$m_s = p_s \times q_s \quad m_N = p_N \times q_N$$

(!! this procedure works only if  $s_1$  is NOT ORTHOGONAL TO  $s_2$  in fact the 3 vectors  $[p_s, q_s, q_N]$  must not be parallel to determine the attitude. !! )

↓

$$[p_s, q_s, m_s] = \underline{A} [p_N, q_N, m_N]$$

Up to now the procedure is the same used for the attitude determination used with 3 measurement.

SINCE  $\{p_s, q_s, m_s\}$  AND  $\{p_N, q_N, m_N\}$  are orthogonal.

$$(p_s \perp (q_s, m_s); q_s \perp (m_s, p_s) \dots \dots )$$

THEN  $\underline{S}$  matrix and  $\underline{V}$  matrix are both orthogonal matrices.

⇒  $\Gamma_{N=2}$

$$\underline{A} = \underline{S} \underline{V}^T$$

$$\underline{V}^T = \underline{V}^{-1}$$

obs: matrix  $\underline{V}$  is transposed (inverted) instead of  $\underline{S}$  because is not affected by zeros.

obs: MEASUREMENTS:  $\{\underline{s}_1, \underline{s}_2 \dots, \underline{s}_M\}$  MAY come from

DIFFERENT SENSORS  $\Rightarrow$  MAY BRING DIFFERENT ERRORS TO THE COMPLEX ERROR ON ATTITUDE DETERMINATION.

$\hookrightarrow$  Therefore optimizing the situation:

\*  $N=2 \Rightarrow \underline{P}_s = \underline{s}_1 + \underline{s}_2 \Rightarrow$  choose  $\underline{s}_1$  as the most precise measurement since  $\underline{P}_s$  determine both  $\underline{q}_s$  and  $\underline{r}_s$

$$\underline{q}_s = \frac{\underline{P}_s \times \underline{s}_2}{\|\underline{P}_s \times \underline{s}_2\|}$$

$$\underline{r}_s = \underline{P}_s \times \underline{q}_s$$

\*  $N=3 \Rightarrow \underline{S} \stackrel{V^{-1}}{=} \underline{A} \Rightarrow$  I'm homogeneously distributing the error on matrix  $\underline{A}$  of the 3 measurements.

in fact:

$$\underline{s}_1 = \underline{A} \cdot \underline{N}_1 + \underline{\varepsilon}_1$$

$$\underline{s}_2 = \underline{A} \cdot \underline{N}_2 + \underline{\varepsilon}_2$$

$$\underline{s}_3 = \underline{A} \cdot \underline{N}_3 + \underline{\varepsilon}_3$$

\*  $N \geq 3 \Rightarrow \underline{S} \cdot \underline{V}^* = \underline{A} \Rightarrow$  I'm making a sort of: Mean square root of the error.

!! I'm looking the most precise measurement !!

STATISTICAL METHOD (NOTATION:  $N \geq 3$ )

L. The global determination of the attitude is the results of many different measurements affected by different errors  $\neq$  sensors

$\Rightarrow$  is possible to "weight" each measurement and find the optimal attitude determination.

$$\underline{s}_i = \underline{A} \cdot \underline{N}_i + \underline{\varepsilon}_i \Rightarrow \underline{\varepsilon}_i = \underline{s}_i - \underline{A} \cdot \underline{N}_i$$

$$\Rightarrow J = \frac{1}{2} \sum_{i=1}^N \lambda_i (\underline{\varepsilon}_i^T \cdot \underline{\varepsilon}_i)$$

$\lambda_i: \sum_i \lambda_i = 1$  measurement  $\Rightarrow$  to determine through optimization.

$\hookrightarrow$  optimize FUNCTIONAL:  $J = \frac{1}{2} \sum_{i=1}^N \lambda_i \|\underline{s}_i - (\underline{A} \cdot \underline{N}_i)\|^2$

$$\left\{ \begin{array}{l} \sum_{i=1}^N \lambda_i = 1 \\ \underline{\varepsilon}_i = \underline{s}_i - \underline{A} \cdot \underline{N}_i \end{array} \right.$$

therefore,

$$J = \frac{1}{2} \sum_{i=1}^N \lambda_i (\underline{s}_i^T \cdot \underline{s}_i + \underline{s}_i^T \cdot (\underline{A} \cdot \underline{N}_i) + \underline{N}_i^T \underline{A}^T \underline{s}_i + \underline{N}_i^T \underline{A}^T \underline{A} \cdot \underline{N}_i)$$

$$\underline{A} = \underline{A}^T \Rightarrow \underline{s}_i^T \underline{A} \cdot \underline{N}_i = \underline{s}_i^T (\underline{A}^T \cdot \underline{N}_i) \text{ - scalar} \Rightarrow \underline{s}_i^T (\underline{A} \cdot \underline{N}_i) = \underline{N}_i^T \underline{A}^T (\underline{A} \cdot \underline{N}_i)$$

$$J = \frac{1}{2} \sum_{i=1}^N \lambda_i (\underline{s}_i^T \cdot \underline{s}_i + \underline{N}_i^T \underline{A}^T \underline{A} \cdot \underline{N}_i - 2 \underline{s}_i^T \underline{A} \cdot \underline{N}_i)$$

$$= \frac{1}{2} \sum_{i=1}^N \lambda_i (\underline{s}_i^T \cdot \underline{s}_i + \underline{N}_i^T \cdot \underline{N}_i - 2 \underline{s}_i^T \underline{A} \cdot \underline{N}_i)$$

$\hookrightarrow$  Supporting of working with NORMALIZED MEASUREMENTS / INERTIAL REFERENCES

$$\|\underline{s}_i\|^2 = \underline{s}_i^T \cdot \underline{s}_i = 1 \quad ; \quad \|\underline{N}_i\|^2 = \underline{N}_i^T \cdot \underline{N}_i = 1$$

$$\Rightarrow J = \sum_{i=1}^N \lambda_i (1 - \underline{s}_i^T \underline{A} \cdot \underline{N}_i) = 1 - \sum_{i=1}^N \underline{s}_i^T \underline{A} \cdot \underline{N}_i$$

↳ As commonly done: is possible to optimize  
also for optimal control.

$$J = 1 - \sum_{i=1}^N \alpha_i (\underline{S}_i^T \underline{A} \underline{N}_i) \rightarrow \text{since } J \text{ is a scalar}$$

↓  
I can optimize the two instead of  $J$  itself.

$$\Rightarrow \tilde{J} = \sum_{i=1}^N \alpha_i (\underline{S}_i^T \underline{A} \underline{N}_i) = \text{Tr}(\underline{A} \cdot \underline{B})$$

$$J = 1 - \sum_{i=1}^N \alpha_i (\underline{S}_i^T \underline{A} \underline{N}_i)$$

↓  
!! Minimizing  $J$  is equivalent with maximizing  $\tilde{J}$  !!

$$\left\{ \begin{array}{l} \tilde{J} = \text{Tr} \left( \sum_{i=1}^N \alpha_i (\underline{S}_i^T \underline{A} \underline{N}_i) \right) = \text{Tr}(\underline{A} \cdot \underline{B}^T) \\ \underline{B} = \sum_{i=1}^N \alpha_i \underline{S}_i \underline{N}_i^T. \end{array} \right.$$

→ trace property

$$(i) \text{Tr}(\underline{A} \underline{B}) = \text{Tr}(\underline{B} \cdot \underline{A}) \quad (ii) \text{Tr}(\underline{A}^T) = \text{Tr}(\underline{A})$$

↓  
 $\text{Tr}(\underline{A} + \underline{B}) = \text{Tr}(\underline{A}) + \text{Tr}(\underline{B}) \quad (iii)$

$$\tilde{J} = \text{Tr} \left( \sum_{i=1}^N \alpha_i (\underline{S}_i^T \underline{A} \underline{N}_i) \right) \quad (i)$$

$$= \text{Tr} \left( \sum_{i=1}^N \alpha_i (\underline{A} \underline{N}_i \underline{S}_i^T) \right) \quad (ii)$$

$$= \text{Tr} \left( \sum_{i=1}^N \alpha_i (\underline{S}_i \underline{N}_i^T \underline{A}^T) \right) \quad (iii)$$

$$= \text{Tr} \left( \left[ \sum_{i=1}^N \alpha_i \underline{S}_i \underline{N}_i^T \right] \underline{A}^T \right)$$

$$= \text{Tr} \left( \underline{A} \cdot \underline{B}^T \right) \quad (iii)$$

!!  $J = J(\underline{A})$  must find  
 $\underline{A}$  that minimizes  $J$  !!

" A direct solution of such optimization problem is not possible in a direct way  $\Rightarrow$  decomposition procedure.  
but:

$\underline{B}$  must not be singular  $\Rightarrow$  AT LEAST 3 MEASUREMENTS ARE REQUIRED

⇒ The solution deserved is the following:

$$\underline{A} = (\underline{B}^T)^{-1} \cdot (\underline{B}^T \underline{B})^{1/2} = \underline{B} \cdot (\underline{B}^T \underline{B})^{-1/2}$$

detailed minimization of  $J$

\* COST FUNCTION:  $J = \text{Tr}(\underline{A} \cdot \underline{B}^T)$

\* RESTRAINT:  $\underline{A}^T \underline{A} - \underline{I} = \underline{0}$

orthogonality of the  $\underline{A}$  matrix.

→ TRACE PROPERTY DERIVATION

$$\frac{\partial J}{\partial \underline{B}} = \underline{A} = \underline{0}$$

$$J^* = \text{Tr}(\underline{A} \cdot \underline{B}^T) + \lambda \text{Tr}(\underline{A} \cdot \underline{A}^T - \underline{I}) \quad (J^* = J^*(\underline{A}, \lambda))$$

$$\rightarrow J^* = \text{Tr}(\underline{B}^T \underline{A}) + \lambda \text{Tr}(\underline{A} \cdot \underline{A}^T - \underline{I}) \underset{(i)}{=} \text{Tr}(\underline{B} \cdot \underline{A}^T) - \lambda \text{Tr}(\underline{A} \cdot \underline{A}^T - \underline{I})$$

$$\frac{\partial J^*}{\partial \underline{A}} = \underline{B} + \lambda \underline{A} = \underline{0} \quad (a)$$

$$\begin{aligned} \text{Tr}(\underline{B}^T \underline{A}) &= \text{Tr}([\underline{B}^T \cdot \underline{A}]^T) \\ &= \text{Tr}(\underline{A}^T \underline{B}) = \text{Tr}(\underline{B} \cdot \underline{A}^T) \end{aligned} \quad (i)$$

$$\frac{\partial J^*}{\partial \lambda} = \underline{A} \cdot \underline{A}^T - \underline{I} = \underline{0} \quad (b)$$

$$(a) \quad \underline{A} = -\frac{1}{\lambda} \underline{B}$$

$$(b) \quad \frac{1}{\lambda^2} \underline{B} \cdot \underline{B}^T = \underline{I} \quad \Rightarrow \quad \frac{1}{\lambda} = (\underline{B} \cdot \underline{B}^T)^{1/2}$$

⇒ CVD

$$\underline{A} = (\underline{B} \cdot \underline{B}^T)^{1/2} \cdot \underline{B}$$

!! This implies that

$\underline{B}$  is a non singular matrix  $\Rightarrow$  3 independent measurements

⇒ CVD

$$\underline{A} = \underline{V} \begin{bmatrix} 1 & & \\ & 1 & \\ & & 1 \end{bmatrix}$$

$$\begin{aligned} \underline{V} &= \text{diag}(\underline{B} \cdot \underline{B}^T) \\ V &= \text{diag}(\underline{B} \cdot \underline{B}^T) \end{aligned}$$

quaternion  $\rightarrow$  replacing  $\mathbb{H}$  expression with quaternions:

$$\tilde{\underline{q}} = \begin{bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \end{bmatrix}, \quad q^T \cdot \tilde{\underline{q}}$$

$$\Rightarrow A = (q_1^2 - q^T \cdot \tilde{\underline{q}}) \mathbb{I} + 2\tilde{\underline{q}} \cdot \tilde{\underline{q}}^T - 2q_4 [q^T]$$

$$[q^T] = \begin{bmatrix} 0 & -q_3 & q_2 \\ q_3 & 0 & -q_1 \\ -q_2 & q_1 & 0 \end{bmatrix}$$

$$\text{constraint: } \tilde{\underline{q}}^T \cdot \tilde{\underline{q}} = 1.$$

Avoiding the notable algebra required:

$$\underline{J}_q = q^T K q + \lambda (q^T \cdot q - 1)$$

where:

$$K = \begin{bmatrix} \frac{s}{2} - \frac{B^T}{2} & \frac{z}{2} \\ \frac{z}{2} & B^T \end{bmatrix}$$

$$B^T = T^2 (\underline{B}) \quad (\underline{B} = \sum_{i=1}^n k_i S_i N_i T)$$

$$S = \underline{B}^T + \underline{B}$$

$$z = [B_{3,2} - B_{2,3}; B_{1,3} - B_{3,1}; B_{1,2} - B_{2,1}]^T$$

the general derivation of a quadratic form:

$$J = \{x_1, x_2\} \begin{bmatrix} a_1 & a_2 \\ a_3 & a_4 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} = \{x_1, x_2\} \begin{bmatrix} a_1 x_1 + a_2 x_2 \\ a_3 x_1 + a_4 x_2 \end{bmatrix}$$

$$= a_1 x_1^2 + a_2 x_1 x_2 + a_3 x_1 x_2 + a_4 x_2^2 (= \underline{x}^T \underline{A} \underline{x})$$

$$\Rightarrow \frac{\partial J}{\partial x_1} = 2a_1 x_1 + 2a_3 x_2 = 2a_1 x_1 + (a_2 + a_3) x_2$$

$$\frac{\partial J}{\partial x_2} = 2a_2 x_1 + (a_2 + a_3) x_2$$

$$\Rightarrow \nabla(J) = \begin{bmatrix} 2a_1 & (a_2 + a_3) \\ (a_2 + a_3) & 2a_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix}$$

$\frac{1}{2}$  if it's symmetric  $\Rightarrow a_2 = a_3 \Rightarrow \nabla(J) = \nabla(\underline{x}^T \underline{A} \underline{x}) = 2\underline{A} \underline{x}$

~~$K$  is symmetric~~:  ~~$\underline{x}^T \underline{A} \underline{x}$~~

$$\underline{B} + \underline{B}^T = \begin{bmatrix} 1 & 2 \\ 3 & 4 \end{bmatrix} + \begin{bmatrix} 1 & 3 \\ 2 & 4 \end{bmatrix} = \begin{bmatrix} 1 & 5 \\ 5 & 8 \end{bmatrix} \text{ is not symmetric.}$$

$\Rightarrow$  The result of such differentiation:

$$\nabla(J_q) = 2K q + 2\lambda q = 0 \Rightarrow K q = -\lambda q$$

"this means the the statistical optimale determination problem can be reduced to an eigen-values / eigen-vector problem.

$\rightarrow$  Solution is not unique  $\rightarrow$  the given vector with the maximum associated eigenvalues  $\lambda_{\max}$  will be the final quaternion  $q_{sol}$ .

$\hookrightarrow$  rewriting the eigen-values / eigen-vector problem in the partitioned form:

$$K q = \lambda_{\max} q \rightarrow \begin{cases} \frac{s}{2} - \frac{B^T}{2} \tilde{q} + \frac{z}{2} q_4 = \lambda_{\max} \tilde{q} \\ \frac{z}{2} q + \frac{B^T}{2} q_4 = \lambda_{\max} q_4 \end{cases} \quad q_1, \dots, q_3$$

Approximation

$\hookrightarrow$  Assuming  $\lambda_{\max} = 1$  (normalized eigen-values)

$$(\frac{s}{2} - \frac{B^T}{2}) \tilde{q} + \frac{z}{2} q_4 = \tilde{q} \rightarrow (\frac{s}{2} - \frac{B^T}{2}) \tilde{q} / q_4 + \frac{z}{2} = \tilde{q} / q_4$$

$$(\frac{s}{2} - \frac{B^T}{2} - \frac{z}{2}) \tilde{q} / q_4 + \frac{z}{2} = 0$$

$\rightarrow$  Gibbs Vector.

$$g = \begin{bmatrix} e_1 \tan \frac{\theta}{2} \\ e_2 \tan \frac{\theta}{2} \\ e_3 \tan \frac{\theta}{2} \end{bmatrix} \Rightarrow \tilde{q} / q_4 = \begin{bmatrix} e_1 \sin \frac{\theta}{2} \\ e_2 \sin \frac{\theta}{2} \\ e_3 \sin \frac{\theta}{2} \end{bmatrix} / \cos \frac{\theta}{2}$$

$$[\frac{s}{2} - (\frac{B^T}{2} - \frac{z}{2})] g = -\frac{z}{2}$$

Final system is  
a  $3 \times 3$  linear system.

→ "control problem → INTERNAL TORQUES"

⇒ Considering both environment and disturbance affecting the dynamic of the system:

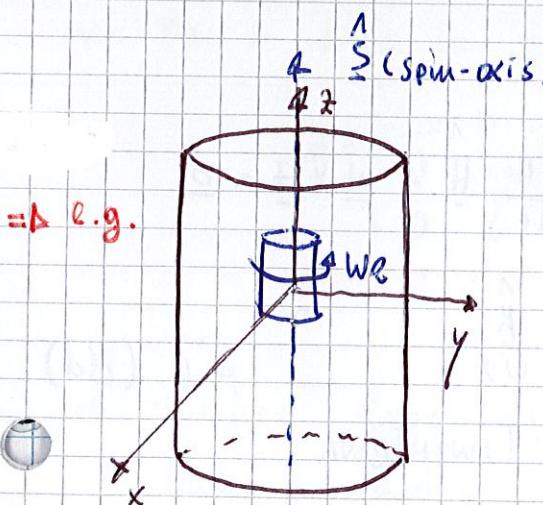
$$\dot{\underline{h}} + \underline{W} \times \underline{h} = \underline{M}_{\text{ext}} + \underline{M}_c$$

disturbance

↳ EXCHANGED WITH ENVIRONMENT:  
 • Thrusters  
 • Magnetic actuators

#### GYROSCOPIC ACTUATOR.

⇒ require that Euler's equation is modified for the contribution of the R.W.  $\underline{h} \rightarrow \underline{H} = \underline{h}_{\text{sys}} + \underline{h}_{\text{R.W.}}$



⇒ e.g.

$$\underline{H} = \underline{I} \underline{w} + \underline{A} \underline{M}_n$$

↳ define the orientation of the actuator in the PI Reference frame ( $\equiv$  spin axis direction)

In this condition  $\underline{S} = \underline{Z}$  ⇒  $\underline{M}_n = I_z \cdot w_z \Rightarrow \underline{A} \cdot \underline{M}_n = \begin{bmatrix} 0 \\ 0 \\ I_z w_z \end{bmatrix}$

In general:  $\begin{bmatrix} 0 & \frac{1}{2} & 0 & \frac{1}{2} \\ 0 & 0 & 1 & \frac{1}{2} \\ 0 & 0 & 0 & 0 \end{bmatrix} \cdot \begin{Bmatrix} \text{actuator 1} \\ \text{actuator 2} \\ \vdots \end{Bmatrix} \rightarrow N = N^0 \text{ of actuator}$

↳  $N = \text{actuators with a different spin axis of rotation.}$

$$\begin{bmatrix} 0 & \frac{1}{2} & 0 & \frac{1}{2} \\ 0 & 0 & \frac{1}{2} & \frac{1}{2} \\ 1 & 0 & 0 & 0 \end{bmatrix} \cdot \begin{Bmatrix} I_{z1} w_{z1} \\ I_{z2} w_{z2} \\ \vdots \end{Bmatrix} =$$

Therefore the correct Euler's equation should be written considering both the actuators and the system itself ]

$$\dot{\underline{H}} + \underline{W} \times \underline{H} = \underline{M}_d$$

$$\frac{d\underline{H}}{dt}|_W = \underline{M}_{ext}|_{in} \rightarrow \frac{d\underline{H}}{dt}|_{PI} + \underline{W}|_{PI} \times \underline{H}|_{PI} - \underline{M}_d|_{PI}$$

$\Rightarrow$  terms  $\begin{cases} \underline{W} \\ \underline{I} \end{cases}$  are relative to the whole system.

Review:

↳ For the generic dual spin satellite:



$$\underline{H} = \underline{h}_B + \underline{h}_R = \begin{pmatrix} I_x w_x \\ I_y w_y \\ I_z w_z + I_R w_R \end{pmatrix}$$

$$\frac{d\underline{H}}{dt} = 0 \rightarrow \frac{d\underline{H}}{dt} \cdot \dot{\underline{H}} + \underline{H} \cdot \frac{d\underline{H}}{dt} = \frac{d\underline{H}}{dt} \cdot \dot{\underline{H}} + \underline{W} \times \underline{H} = 0$$

$$\begin{pmatrix} I_x w_x \\ I_y w_y \\ I_z w_z + I_R w_R \end{pmatrix} + \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix} \begin{pmatrix} w_x \\ w_y \\ w_z \end{pmatrix} = 0 \quad (\text{Md})$$

$$\begin{cases} I_x w_x + I_z w_z w_y + I_R w_R w_y - I_y w_z w_y = 0 \\ I_y w_y - I_z w_z w_x - I_R w_R w_x + I_x w_x w_z = 0 \\ I_z w_z + I_R w_R + I_y w_y w_x - I_x w_x w_y = 0 \end{cases}$$

+  $I_R w_R = M_R \approx$  electric engine (gives  $M_R$ )

(TIA) equation is needed since the  $\underline{w}_R$  degrees of freedom has been added)

$$\rightarrow \begin{cases} I_x w_x + (I_z - I_y) w_z w_y + I_R w_R w_y = 0 \\ I_y w_y + (I_x - I_z) w_x w_z - I_R w_R w_x = 0 \\ I_z w_z + (I_y - I_x) w_x w_y + I_R w_R = 0 \end{cases}$$

!! even if  $w_R = 0$  the inertia moments of the spacecraft must be computed by considering the inertia of the motor.

$$\begin{matrix} I_{xS} \\ I_{yS} \\ I_{zS} \end{matrix}$$

$$\begin{matrix} I_{ex} \\ I_{ey} \\ I_{ez} \end{matrix}$$

$$\begin{matrix} I_{xR} \\ I_{yR} \\ I_{zR} \end{matrix}$$

$$\begin{matrix} I_{xS} + I_{ex} = I_x^* \\ I_{yS} + I_{ey} = I_y^* \\ I_{zS} + I_{ez} = I_z^* \end{matrix}$$

$$\begin{matrix} I_{xR} = I_x^* \\ I_{yR} = I_y^* \\ I_{zR} = I_z^* \end{matrix}$$

$$\Rightarrow \underline{H} = \begin{bmatrix} I_x & 0 & 0 \\ 0 & I_y & 0 \\ 0 & 0 & I_z \end{bmatrix} \cdot \underline{W} + \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \cdot I_R w_R$$

↳ more than 1 body in relative motion: (1 for each axis)

$$\underline{H} = \begin{bmatrix} I_x & 0 & 0 \\ 0 & I_y & 0 \\ 0 & 0 & I_z \end{bmatrix} \cdot \underline{W} + \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \cdot \begin{bmatrix} I_{Rx} w_x \\ I_{Ry} w_y \\ I_{Rz} w_z \end{bmatrix}$$

$\Rightarrow$  therefore (agreed with this notation) we can write Euler's equation:

$$\begin{cases} \underline{H} = \underline{I} \cdot \underline{W} + \underline{A} \cdot \underline{h}_R \\ \frac{d\underline{H}}{dt} + \underline{W} \times \underline{H} = \underline{M}_d \end{cases} \rightarrow \frac{d}{dt} [\underline{I} \cdot \underline{W} + \underline{A} \cdot \underline{h}_R] + \underline{W} \times [\underline{I} \cdot \underline{W} + \underline{A} \cdot \underline{h}_R] = \underline{M}_d$$

Proceeding with the computation:

$$\underline{I} = \frac{d\underline{W}}{dt} + \frac{d\underline{I}}{dt}, \underline{W} + \frac{d\underline{A}}{dt} \cdot \underline{h}_R + \underline{A} \cdot \frac{d\underline{h}_R}{dt} + \underline{W} \times (\underline{I} \cdot \underline{W}) + \underline{A} \cdot \frac{d\underline{A}}{dt} \cdot \underline{h}_R = 0$$

!! Since  $\frac{d\underline{A}}{dt} \neq 0 \Rightarrow$  also the orientation of the rates may change modifying the inertia.  $\Rightarrow \frac{d\underline{I}}{dt} \neq 0$ .

(Orientation)

$$(i) \underline{\underline{I}} \dot{\underline{\underline{W}}} + \underline{\underline{W}} \cdot \dot{\underline{\underline{I}}} + \underline{\underline{W}} \times (\underline{\underline{I}} \cdot \underline{\underline{W}}) + \underline{\underline{A}} \dot{\underline{\underline{h}}}_R + \underline{\underline{A}} \cdot \dot{\underline{\underline{h}}}_R + \underline{\underline{W}} \times (\underline{\underline{A}} \cdot \underline{\underline{h}}_R) = \underline{\underline{M}}_d$$

$\downarrow$   
The only gyroscopic actuator that works  
modifying the position of the wheel is the

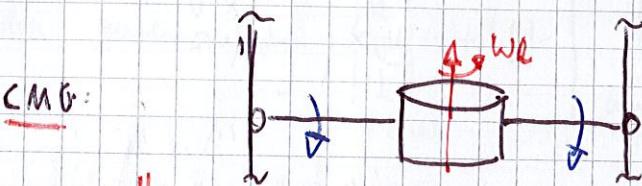
CMG  $\rightarrow$  control moment gyro

while:

RW  $\rightarrow$  reaction wheels  $\rightarrow \underline{\underline{w}}_R(t=0) = 0 \rightarrow \underline{\underline{h}}_R \uparrow$

IW  $\rightarrow$  inertia wheels  $\rightarrow \underline{\underline{w}}_R(t=0) = \underline{\underline{w}}_R \rightarrow \underline{\underline{h}}_R \uparrow$

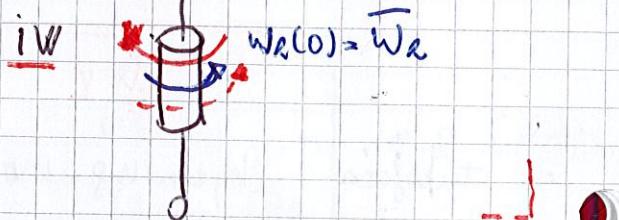
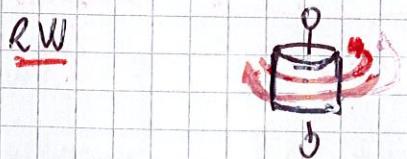
by increasing  
the  $\underline{\underline{h}}_R$  terms  
with



(mechanism that can be tilted.)

!! anyway the term  $\|\underline{\underline{A}} \cdot \underline{\underline{h}}_R\| \gg \|\underline{\underline{I}} \cdot \underline{\underline{W}}\|$

since the inertia of the rotor is small if compared to the payload!!



Working on equation (i) is then possible to put in evidence  
the contribution due to control.

$$(i) \underline{\underline{I}} \dot{\underline{\underline{W}}} + \underline{\underline{W}} \times (\underline{\underline{I}} \cdot \underline{\underline{W}}) = \underline{\underline{M}}_d - \underline{\underline{I}} \cdot \dot{\underline{\underline{W}}} - \underline{\underline{A}} \cdot \dot{\underline{\underline{h}}}_R + \underline{\underline{A}} \cdot \underline{\underline{h}}_R - \underline{\underline{W}} \times (\underline{\underline{A}} \cdot \underline{\underline{h}}_R)$$

$M_c$ .

## 1. REACTION WHEELS

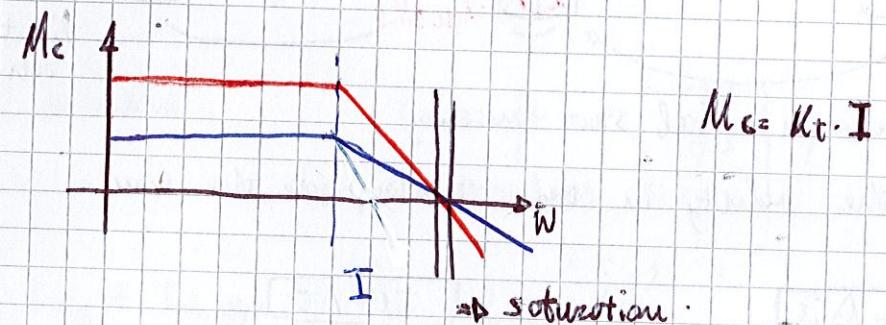
With only modifying the term  $\underline{\underline{h}}_R \Rightarrow \underline{\underline{I}} = 0, \dot{\underline{\underline{I}}} = 0$

therefore equation (i) becomes:

$$(i) \underline{\underline{I}} \dot{\underline{\underline{W}}} + \underline{\underline{W}} \times (\underline{\underline{I}} \cdot \underline{\underline{W}}) = \underline{\underline{M}}_d - \underline{\underline{A}} \cdot \dot{\underline{\underline{h}}}_R - \underline{\underline{W}} \times (\underline{\underline{A}} \cdot \underline{\underline{h}}_R)$$

... In order to increase the speed of reaction wheels (and so home ( $\dot{\underline{\underline{h}}}_R \neq 0$ ) electric (brushless) engines are required).

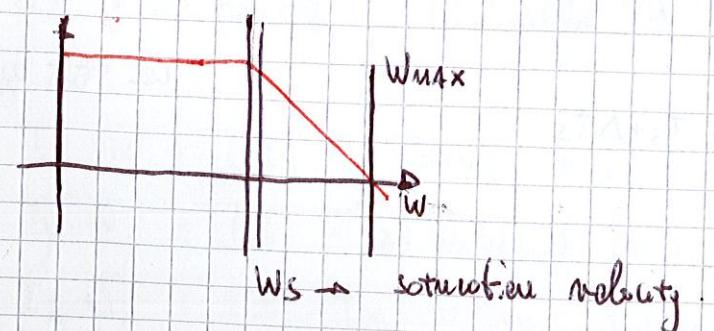
$\Rightarrow$  The typical operational curve for a DC motor would be the following:



$\rightarrow$  power balance:

$$\begin{cases} M_c = \underline{\underline{h}}_R \\ M_c = k_t \cdot I_R \end{cases} \Rightarrow P_{motor} = P_{el} \Rightarrow \Delta V I_R = M_c \cdot \underline{\underline{w}}_R \quad \left\{ \Delta V \approx \text{const.} \right.$$

$$\Rightarrow \Delta V \frac{1}{k_t} = k_t \frac{1}{k_a} w_R \rightarrow w_R |_{\max} = \frac{\Delta V}{k_t}$$



Assumption:  $I_{zi} \ll I_{is}$

it means that we can neglect into (i) equation the "gyroscopic term" of the R.W.

$$(i): \underline{\underline{I}} \dot{\underline{\underline{W}}} + \underline{\underline{W}} \times (\underline{\underline{I}} \underline{\underline{W}}) = -\underline{\underline{A}} \dot{\underline{\underline{h}}}_r - \underline{\underline{W}} \times \underline{\underline{M}}_r$$

and in this example  $\underline{\underline{A}} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$

so equation will become

$$\left\{ \begin{array}{l} I_x \dot{w}_x + (I_z - I_y) \dot{w}_z w_y + I_R \dot{w}_{Rx} = 0 \\ I_y \dot{w}_y + (I_x - I_z) \dot{w}_z w_x + I_R \dot{w}_{Ry} = 0 \\ I_z \dot{w}_z + (I_y - I_x) w_x w_y + I_R \dot{w}_{Rz} = 0 \end{array} \right.$$

$$M_{Rx} = I_R \dot{w}_{Rx}$$

Assumption 2: No torque

$$M_{Ry} = -I_R \dot{w}_{Ry}$$

transmitted between the rotor and the spacecraft

$$M_{Rz} = -I_R \dot{w}_{Rz}$$

provided by the electric bypass.

(IDEAL TRANSMISSION)

$\Rightarrow$  Putting in evidence the control action:

$$\left\{ \begin{array}{l} I_x \dot{w}_x + (I_z - I_y) \dot{w}_z w_y = -I_R \dot{w}_{Rx} = M_{Rx} \\ I_y \dot{w}_y + (I_x - I_z) \dot{w}_z w_x = -I_R \dot{w}_{Ry} = M_{Ry} \\ I_z \dot{w}_z + (I_y - I_x) w_x w_y = -I_R \dot{w}_{Rz} = M_{Rz} \end{array} \right.$$

In designing the control torque value many strategy can be used:

e.g.  $\rightarrow$  FULL-STATE FEEDBACK (Proportional)

$$M_{Rx} = K_{xx} w_x + K_{xy} w_y + K_{xz} (w_z - \bar{w}_z)$$

or PD

$$M_{Rx} = K_x w_x + K_{x,z} \dot{w}_z$$

$\Rightarrow$  Considering also the gyroscopic effect of the reaction wheels  
(relaxing Assumption 1)

$$\underline{\underline{I}} \dot{\underline{\underline{W}}} + \underline{\underline{W}} \times (\underline{\underline{I}} \underline{\underline{W}}) = -\underline{\underline{A}} \dot{\underline{\underline{h}}}_r - \underline{\underline{W}} \times (\underline{\underline{A}} \dot{\underline{\underline{h}}}_r) = M_c$$

$$\Rightarrow M_c = -\underline{\underline{A}} \dot{\underline{\underline{h}}}_r - \underline{\underline{W}} \times (\underline{\underline{A}} \dot{\underline{\underline{h}}}_r)$$

$$M_c = f(w_x, w_y, w_z, \dots) \text{ from PID, LQR, } \dots + I_R \dot{w}_R = M_R$$

--> PROCEDURE (general)

(1) Compute  $M_c$  thanks to the control law:

$$(*) M_c = K_D (\dot{\alpha} - \dot{\alpha}_{sp}) + K_P (\alpha - \alpha_{sp}) \text{ e.g. PD; } \alpha \text{ attitude parameters.}$$

(2) Determine  $\dot{\underline{\underline{W}}}$  (by solving Euler's equation)

$$\underline{\underline{I}} \dot{\underline{\underline{W}}} + \underline{\underline{W}} \times (\underline{\underline{I}} \underline{\underline{W}}) = M_c \rightarrow \dot{\underline{\underline{W}}}$$

(3) Determine  $\dot{\underline{\underline{h}}}_r$

$$\dot{\underline{\underline{h}}}_r = \underline{\underline{A}}^{-1} M_c - \underline{\underline{A}}^{-1} \underline{\underline{W}} \times (\underline{\underline{A}} \dot{\underline{\underline{h}}}_r)$$

(4) Determine the power required to increase the angular momentum of the reaction wheels:

$$M_R = I_R \dot{w}_R$$

$$\Rightarrow M_R = K_T \cdot i_i$$

current in the  $i$ -th RW  
feeding the DC motor

(\*) in the initial equation [control-LAW]

$$\underline{M}_c = K_D \cdot (\dot{\underline{x}} - \dot{\underline{x}}_{sp}) - K_p (\underline{x} - \underline{x}_{sp})$$

$$\text{generic PD} \rightarrow \underline{M}_c = K_D \dot{\underline{x}}(t) + K_p \underline{x}(t)$$

the attitude parameter is expressed generally while:

(for small deviations from target conditions) is possible to

express it in a very definite form:

$\underline{A}_T \rightarrow$  "Target-direction cosine matrix"

$\underline{A}_m \rightarrow$  "Direction cosine matrix measured by the sensors"

$$\Rightarrow \underline{A}_e = \underline{A}_T^{-1} \underline{A}_m$$

BUT since both  $\underline{A}_T$  and  $\underline{A}_m$  are supposed to be orthogonal  
and equals.

THEN  $\underline{A}_e$  (representing the non orthogonality) can be computed

as product:

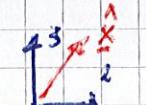
$$\underline{A}_e = \underline{A}_m \cdot \underline{A}_T^T$$

in fact:

$$\underline{A}_m = \begin{bmatrix} a_{11m} & a_{12m} & a_{13m} \\ a_{21m} & a_{22m} & a_{23m} \\ a_{31m} & a_{32m} & a_{33m} \end{bmatrix} = \begin{bmatrix} \underline{x}_m \\ \underline{y}_m \\ \underline{z}_m \end{bmatrix}$$

where:  $a_{11} = \underline{x} \cdot \hat{\underline{1}}$  as initial  $\underline{1}$ ;  $a_{12} = \underline{x} \cdot \hat{\underline{2}}$  ...  
& P.I. r.g.

is the first row of the direction cosine matrix represent  
the projection of the " $\hat{\underline{x}}$ " axis in P.I. on the  
I.N. reference frame



completely analogous:

$$\underline{A}_T^T = \begin{bmatrix} a_{11T} & a_{12T} & a_{13T} \\ a_{21T} & a_{22T} & a_{23T} \\ a_{31T} & a_{32T} & a_{33T} \end{bmatrix} = \begin{bmatrix} \underline{x}_T \\ \underline{y}_T \\ \underline{z}_T \end{bmatrix}$$

therefore:

$$\underline{A}_e = \underline{A}_m \cdot \underline{A}_T^T = \begin{bmatrix} \underline{x}_m \\ \underline{y}_m \\ \underline{z}_m \end{bmatrix} \cdot \begin{bmatrix} \underline{x}_T \\ \underline{y}_T \\ \underline{z}_T \end{bmatrix}^T$$

$$= \begin{bmatrix} \underline{x}_m \\ \underline{y}_m \\ \underline{z}_m \end{bmatrix} \cdot \begin{bmatrix} \underline{x}_T^T & \underline{y}_T^T & \underline{z}_T^T \end{bmatrix} = \begin{bmatrix} \underline{x}_m \cdot \underline{x}_T^T & \underline{x}_m \cdot \underline{y}_T^T & \underline{x}_m \cdot \underline{z}_T^T \\ \underline{y}_m \cdot \underline{x}_T^T & \underline{y}_m \cdot \underline{y}_T^T & \underline{y}_m \cdot \underline{z}_T^T \\ \underline{z}_m \cdot \underline{x}_T^T & \underline{z}_m \cdot \underline{y}_T^T & \underline{z}_m \cdot \underline{z}_T^T \end{bmatrix}$$

$\Rightarrow$  each element of matrix  $\underline{A}_e$  represent the scalar product  
between 2 axis done in the I.N. reference frame.

$$\text{e.g. } \underline{i}_m = [\underline{i}_m \cdot \hat{\underline{1}}; \underline{i}_m \cdot \hat{\underline{2}}; \underline{i}_m \cdot \hat{\underline{3}}] \quad \underline{i}_m \leftrightarrow \underline{x}_m$$

$$\underline{i}_T = [\underline{i}_T \cdot \hat{\underline{1}}; \underline{i}_T \cdot \hat{\underline{2}}; \underline{i}_T \cdot \hat{\underline{3}}] \quad \underline{i}_T \leftrightarrow \underline{x}_T$$

$$\hat{\underline{1}}_T = [\hat{\underline{1}}_T \cdot \hat{\underline{1}}; \hat{\underline{1}}_T \cdot \hat{\underline{2}}; \hat{\underline{1}}_T \cdot \hat{\underline{3}}]$$

for small deviation from the target condition:

$$\cos(\underline{i}_m \cdot \underline{i}_T) \approx 0[\text{rad}] \Rightarrow \underline{i}_m \cdot \underline{i}_T \approx 1$$

$$\cos(\underline{i}_m \cdot \hat{\underline{1}}_T) \approx \pi/2[\text{rad}] \Rightarrow \underline{i}_m \cdot \hat{\underline{1}}_T \approx$$

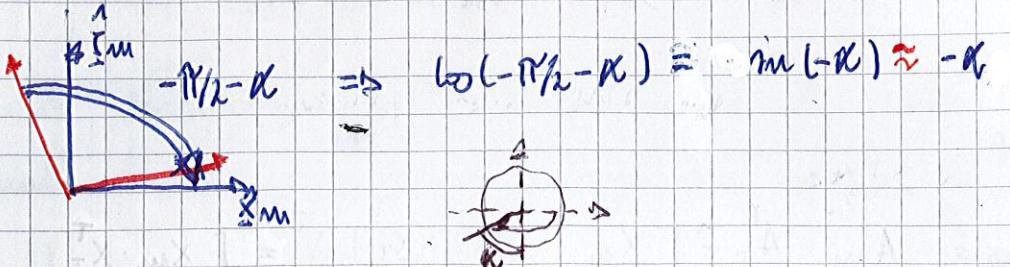


$\Rightarrow$  in truth is equal to  $\pi/2 + \alpha$   
that is a small deviation angle  
 $\Rightarrow \cos(\pi/2 + \alpha) = \sin(\alpha) \approx \alpha$   
(considering small angles)

Since this angle " $\alpha$ " represent the rotation around  $\underline{z}_m$  axis  
is called  $\alpha_2$ .

WHILE

considering the opposite scalar-product.



$\Rightarrow$  Considering small rotation angles between the 3 PI inertia axis

$$\underline{A}_e = \begin{bmatrix} \underline{x}_m \cdot \underline{x}_T^T & \underline{x}_m \cdot \underline{y}_T^T & \underline{x}_m \cdot \underline{z}_T^T \\ \underline{y}_m \cdot \underline{x}_T^T & \underline{y}_m \cdot \underline{y}_T^T & \underline{y}_m \cdot \underline{z}_T^T \\ \underline{z}_m \cdot \underline{x}_T^T & \underline{z}_m \cdot \underline{y}_T^T & \underline{z}_m \cdot \underline{z}_T^T \end{bmatrix} \approx \begin{bmatrix} 1 & \alpha_2 & -\alpha_y \\ -\alpha_z & 1 & \alpha_x \\ +\alpha_y & -\alpha_x & 1 \end{bmatrix}$$

!! This is a good approximation of the error (small variations from nominal/set-point condition)

As seen before.

$$[\underline{A}_e]_{1,2} = \alpha_{1,2} = K_2 \quad \begin{array}{l} \text{drives rotation} \\ \text{around} \end{array} \quad \begin{array}{l} \text{"z-axis"} \\ \rightarrow \end{array}$$

$$[\underline{A}_e]_{2,3} = \alpha_{2,3} = -\alpha_y \quad \begin{array}{l} \text{drives rotation} \\ \text{around} \end{array} \quad \begin{array}{l} \text{"y-axis"} \\ \rightarrow \end{array}$$

$$[\underline{A}_e]_{1,3} = \alpha_{1,3} = K_x \quad \begin{array}{l} \text{drives rotation} \\ \text{around} \end{array} \quad \begin{array}{l} \text{"x-axis"} \\ \rightarrow \end{array}$$

$$\alpha_{1,2} = \underline{x}_m \cdot \underline{y}_T^T \quad ; \quad \alpha_{2,3} = \underline{x}_m \cdot \underline{z}_T^T \quad ; \quad \alpha_{1,3} = \underline{y}_m \cdot \underline{x}_T^T$$

$\Rightarrow$  PD, control law [SMALL ROTATION]

$$\begin{cases} M_x = K_p x \alpha_x + K_d x \dot{\alpha}_x \\ M_y = K_p y \alpha_y + K_d y \dot{\alpha}_y \\ M_z = K_p z \alpha_z + K_d z \dot{\alpha}_z \end{cases}$$

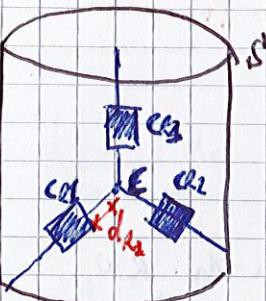
$$\begin{cases} M_{x,s} = K_p x \alpha_{23,e} + K_d x w_x \\ M_{y,s} = K_p y \alpha_{31,e} + K_d y w_y \\ M_{z,s} = -K_p z \alpha_{12,e} + K_d z w_z \end{cases}$$

MENGLI: Control system with 3 RW aligned with the 3 inertia axis.

equilibrium condition:

$$\begin{bmatrix} 0 \\ 0 \\ n \end{bmatrix}$$

L rotation per orbit (circular)  $h = \frac{2\pi}{T_{\text{orbit}}}$



$c \rightarrow$  overall center of mass.

$$\underline{I}_c = \underline{I}_{s,c} + \underline{I}_{r1,c} + \underline{I}_{r2,c} + \underline{I}_{e,c}$$

Defining inertia moments matrices:

$$\underline{I}_{s,c} = \begin{bmatrix} A_s & 0 & 0 \\ 0 & B_s & 0 \\ 0 & 0 & C_s \end{bmatrix}$$

$$\underline{I}_{r1,c} = \begin{bmatrix} I_{R1} & 0 & 0 \\ 0 & B_{R1} & 0 \\ 0 & 0 & C_{R1} \end{bmatrix} ; \quad \underline{I}_{r2,c} = \begin{bmatrix} A_{R2} & 0 & 0 \\ 0 & I_{R2} & 0 \\ 0 & 0 & C_{R2} \end{bmatrix}$$

matrice di inerzia  
rotore 1 rispetto a  
CoM (Centro del primo rotore)

!! via nella TEOREMA FONDAMENTALE DI INERZIA !!

$\Rightarrow$  Huygen's theorem

$$I_A = I_B + M \parallel(B-A)\parallel^2$$

( $I_A, I_B$  inertia moment

along the same axis  $\Rightarrow$  ( $A, B$ ) contained in the same plane)



$$\underline{I}_c^{\text{tot}} = \underline{I}_{s,c} + \underline{I}_{r1,c} + \underline{I}_{r2,c} + \underline{I}_{e,c}$$

$$= \begin{bmatrix} A_s & 0 & 0 \\ 0 & B_s & 0 \\ 0 & 0 & C_s \end{bmatrix} + \begin{bmatrix} I_{R1} & 0 & 0 \\ 0 & B_{R1} + M_{R1} d_{R1}^2 & 0 \\ 0 & 0 & C_{R1} + M_{R1} d_{R1}^2 \end{bmatrix} + \dots$$

therefore the 3 inertia moments (With respect to the center of mass of the overall system ( $S + R_1 + R_2 + R_3$ ) are the following:

$$I_x = A_s + I_{R1} + A_{R2} d_{R1}^2 + A_{R3} d_{R3}^2$$

$$I_y = B_s + I_{R2} + B_{R1} d_{R1}^2 + B_{R3} d_{R3}^2$$

$$I_z = C_s + I_{R3} + C_{R1} d_{R1}^2 + C_{R2} d_{R2}^2$$

(i)

**Assumption 1**

second spin axis of the i-th RW.

$$\{\underline{w}_{ri}\}_{CB} = \begin{bmatrix} w_x \\ w_y \\ w_z \end{bmatrix} + \begin{bmatrix} \alpha_1 \\ 0 \\ 0 \end{bmatrix} \approx \begin{bmatrix} w_x \\ w_y \\ w_z \end{bmatrix}$$

f  
body centered term.

$$\Rightarrow \underline{w}_{R2} \approx \begin{bmatrix} w_x \\ \alpha_2 \\ w_z \end{bmatrix}; \quad \underline{w}_{R3} \approx \begin{bmatrix} w_x \\ w_y \\ \alpha_3 \end{bmatrix} \quad (\text{ii})$$

(i) + (ii)  $\Rightarrow \underline{w} +$

$$\underline{H}_c^{tot} = \begin{bmatrix} A_s & 0 & 0 \\ 0 & B_s & 0 \\ 0 & 0 & C_s \end{bmatrix} \cdot \begin{bmatrix} w_x \\ w_y \\ w_z \end{bmatrix} + \begin{bmatrix} I_{R1} & 0 & 0 \\ 0 & B_{R1} + m d^2 & 0 \\ 0 & 0 & C_{R1} + m d^2 \end{bmatrix} \cdot \begin{bmatrix} \alpha_1 \\ w_y \\ w_z \end{bmatrix} +$$

$$+ \begin{bmatrix} A_{R2} + m d^2 & 0 & 0 \\ 0 & I_{R2} & 0 \\ 0 & 0 & C_{R2} + m d^2 \end{bmatrix} \cdot \begin{bmatrix} \alpha_2 \\ w_z \\ w_x \end{bmatrix}$$

$$+ \begin{bmatrix} A_{R3} + m d^2 & 0 & 0 \\ 0 & B_{R3} + m d^2 & 0 \\ 0 & 0 & I_{R3} \end{bmatrix} \cdot \begin{bmatrix} w_x \\ w_z \\ \alpha_3 \end{bmatrix}$$

$\Rightarrow$  enclosing terms in  $\{w_x, w_y, w_z\}$  (even neglecting Assumption 1) separately from the ones in  $\{I_{R1}, I_{R2}, I_{R3}\}$ .

$$\underline{H}_c = \begin{bmatrix} I_x & & \\ & I_y & \\ & & I_z \end{bmatrix} \cdot \begin{bmatrix} w_x \\ w_y \\ w_z \end{bmatrix} + \begin{bmatrix} I_{R1} \alpha_1 \\ I_{R2} \alpha_2 \\ I_{R3} \alpha_3 \end{bmatrix}$$

$$\frac{d \underline{H}_c}{dt} = \underline{M}_d$$

;  $\{I_{R1} \alpha_1 \triangleq h_1; I_{R2} \alpha_2 = h_2; I_{R3} \alpha_3 = h_3\}$ .

$\Rightarrow$  By defining:

$$\underline{H}_{c,s} = \begin{bmatrix} I_x & & \\ & I_y & \\ & & I_z \end{bmatrix} \cdot \begin{bmatrix} w_x \\ w_y \\ w_z \end{bmatrix}$$

↑ capital "H" since includes also inertia contribution due to the reaction wheels.

$$\underline{H}_{c,R} = \begin{bmatrix} I_{R1} \alpha_1 \\ I_{R2} \alpha_2 \\ I_{R3} \alpha_3 \end{bmatrix} = \begin{bmatrix} h_1 \\ h_2 \\ h_3 \end{bmatrix}$$

$$\underline{H}_c = \underline{H}_{c,s} + \underline{H}_{c,R}$$

$\Rightarrow$  Euler's equation will be in such condition:

$$\frac{d \underline{H}_{c,s}}{dt} + \frac{d \underline{H}_{c,R}}{dt} + \underline{w} \times (\underline{H}_{c,s} + \underline{H}_{c,R}) = \underline{M}_d$$

$\Rightarrow$  Putting again in order the contribution due to control:

$$\underline{H}_{c,s} + \underline{w} \times (\underline{H}_{c,s} + \underline{H}_{c,R}) = \underline{M}_d - \underline{H}_{c,R}$$

$$\Leftrightarrow M_c \triangleq -\underline{H}_{c,R}$$

" As observed previously the torque provided by the gravity gradient. can give a sort of stabilization (or it's anyway possible) to open otherwise new stability regions "

\* Equilibrium condition:  $\begin{pmatrix} 0 \\ 0 \\ h \end{pmatrix}_{\gamma_B}$

\*  $\dot{\phi}, \dot{\theta}, \dot{\psi} \rightarrow$  Euler's equation determining the orientation with respect to the LVLH reference frame of the P.I. reference frame.

Sequence:  $x \rightarrow y \rightarrow z$   
 $\Rightarrow \underline{\underline{A}}_{\text{PI2LHLV}} = \begin{bmatrix} c\alpha_z c\alpha_y & c\alpha_z s\alpha_y s\alpha_x & \dots \\ \dots & \dots & \dots \\ \dots & \dots & \dots \end{bmatrix}$

If  $\underline{\underline{\alpha}} = \begin{pmatrix} 0 \\ 0 \\ h \end{pmatrix}; h = \frac{2\pi}{T_{\text{orb}}}$  ;  $(\alpha_x = \dot{\phi}; \alpha_y = \dot{\theta}; \alpha_z = \dot{\psi})$   
 $\text{!! DIFFERENTLY FROM BEFORE !!}$

Then  $\underline{\underline{A}}_{\text{PI}} \equiv \underline{\underline{LHLV}} \Rightarrow$  I can linearize  $\underline{\underline{A}}_{\text{PI2LHLV}}$  around equilibrium condition  $\begin{pmatrix} \alpha_x \\ \alpha_y \\ \alpha_z \end{pmatrix} = \underline{\underline{\alpha}}$ .

$\underline{\underline{A}}_{\text{PI2LHW}} = \begin{bmatrix} c\alpha_z c\alpha_y & c\alpha_z s\alpha_y s\alpha_x + s\alpha_z c\alpha_x & -c\alpha_z s\alpha_y c\alpha_x + c\alpha_z s\alpha_x \\ -s\alpha_z c\alpha_y & -s\alpha_z s\alpha_y s\alpha_x + c\alpha_z c\alpha_x & s\alpha_z s\alpha_y s\alpha_x + c\alpha_z s\alpha_x \\ s\alpha_y & -c\alpha_y s\alpha_x & c\alpha_y c\alpha_x \end{bmatrix}$

$\left\{ \begin{array}{l} c(\alpha_i) \approx 1 \\ s(\alpha_i) \approx \alpha_i \\ s(\alpha_i) \cdot s(\alpha_j) \approx 0 \end{array} \right.$

$\lim_{\underline{\underline{A}}_{\text{PI2LHLV}}} = \begin{bmatrix} 1 & \alpha_z & -\alpha_y \\ -\alpha_z & 1 & \alpha_x \\ \alpha_y & \alpha_x & 1 \end{bmatrix}$

\*  $\underline{\underline{W}}_{B_B} = \underline{\underline{W}}_0 + \underline{\underline{W}}_{B0}$

→ previous computations.

[previous exercise book]

$\begin{cases} W_x = \alpha_x + \alpha_y \cdot h \\ W_y = \alpha_y - \alpha_x \cdot h \\ W_z = \alpha_z + h \end{cases}$

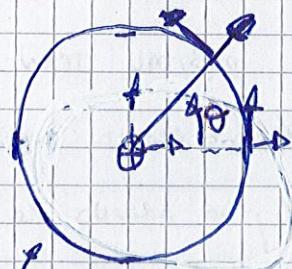
Therefore:

$$\underline{\underline{W}} = \begin{bmatrix} \alpha_x - \alpha_y \cdot h \\ \alpha_y + \alpha_x \cdot h \\ \alpha_z + h \end{bmatrix}$$

[HAVING CONSIDERED SMALL ROTATIONS, THE ORDER IN WHICH THEY ARE OBTAINED IS NOT RELEVANT.]

$\{\underline{\underline{W}}\}_{\gamma_B} = \{\underline{\underline{W}}_{B0}\}_{\gamma_B} + \{\underline{\underline{W}}_0\}_{\gamma_B}$

\*  $\{\underline{\underline{W}}_0\}_{\gamma_B} = \{\underline{\underline{A}}_{\text{LHLV}}\}_{\text{PI}} \cdot \underline{\underline{A}}_{\text{in2LHLV}} \begin{pmatrix} 0 \\ 0 \\ h \end{pmatrix} \quad (h = \sqrt{\frac{\mu \Omega}{r^3}} \rightarrow \text{circular orbit})$



equatorial + circular orbit.

$\underline{\underline{I}}_0 = \underline{\underline{I}} \Rightarrow \{\underline{\underline{W}}_0\}_{\gamma_I} = \begin{pmatrix} 0 \\ 0 \\ h \end{pmatrix}$

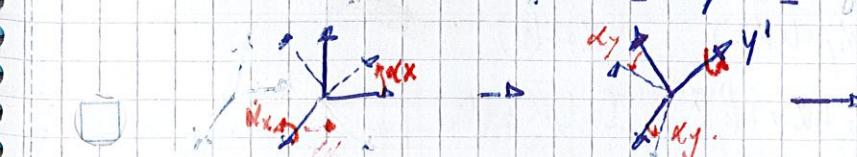
$\underline{\underline{A}}_{\text{in2LHLV}} = \begin{bmatrix} \cos \theta & \sin \theta & 0 \\ -\sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix}$

$\{\underline{\underline{W}}_0\}_{\gamma_0} = \begin{bmatrix} \cos \theta & \sin \theta & 0 \\ -\sin \theta & \cos \theta & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{pmatrix} 0 \\ 0 \\ h \end{pmatrix} = \begin{pmatrix} 0 \\ 0 \\ h \end{pmatrix}$

\*  $\{\underline{\underline{W}}_{B0}\}_{\gamma_B} = \begin{bmatrix} 1 & \alpha_z & -\alpha_y \\ -\alpha_z & 1 & \alpha_x \\ \alpha_y & \alpha_x & 1 \end{bmatrix} \begin{pmatrix} \alpha_x \\ \alpha_y \\ \alpha_z \end{pmatrix}$

law of independence of rotations.  $\alpha_x \rightarrow \alpha_y \rightarrow \alpha_z$

$\underline{\underline{W}}_{B0} = T(\alpha_x) \underline{\underline{\alpha}}_x + T(\alpha_y) T(\alpha_x) \underline{\underline{\alpha}}_y + T(\alpha_z) T(\alpha_x) T(\alpha_y) \underline{\underline{\alpha}}_z$



$T(\alpha_x) = \begin{bmatrix} 1 & 0 & 0 \\ 0 & c\alpha_x & s\alpha_x \\ 0 & -s\alpha_x & c\alpha_x \end{bmatrix}$

$T(\alpha_y) = \begin{bmatrix} c\alpha_y & 0 & s\alpha_y \\ 0 & 1 & 0 \\ -s\alpha_y & 0 & c\alpha_y \end{bmatrix}$

$\Rightarrow$  for small rotations ( $\alpha$  is previously demonstrated) it is possible to obtain  $\dot{\alpha}$  such that:

$$\Delta H_{L,V2,PL} = \begin{bmatrix} \dots \\ \alpha_1 \\ \alpha_2 \end{bmatrix} \cdot \begin{bmatrix} \dot{\alpha}_x \\ \alpha_1 \\ \alpha_2 \end{bmatrix}$$

$$*\quad \dot{\alpha}_{WS} \gamma_B = \dot{\alpha}_{BS0} \gamma_B + \dot{\alpha}_{BS} \gamma_B = \begin{bmatrix} \dot{\alpha}_x - h \dot{\alpha}_y \\ \dot{\alpha}_1 + h \dot{\alpha}_x \\ \dot{\alpha}_2 + h \end{bmatrix}$$

$\rightarrow$  EULER'S EQUATION:  $\dot{H}_{c,S} + \underline{W} \times (\dot{H}_{c,S} + \dot{H}_{c,R}) = \underline{M}_{dL} - \dot{H}_{c,R}$

$\rightarrow \underline{W}$  as function of small rotations from orbital term:

$$\underline{W} = \begin{bmatrix} \dot{\alpha}_x + h \dot{\alpha}_y \\ \dot{\alpha}_y + h \dot{\alpha}_x \\ \dot{\alpha}_2 + h \end{bmatrix} \quad \begin{array}{l} \dot{\alpha}_x \rightarrow \text{AROUND YAW AXIS} \\ \dot{\alpha}_y \rightarrow \text{AROUND ROLL AXIS} \\ \dot{\alpha}_2 \rightarrow \text{AROUND PITCH AXIS} \end{array}$$

$$\cancel{\frac{I_s}{2} \frac{d}{dt} \left( \begin{bmatrix} \dot{\alpha}_x - h \dot{\alpha}_y \\ \dot{\alpha}_y + h \dot{\alpha}_x \\ \dot{\alpha}_2 + h \end{bmatrix} \right)} + \begin{bmatrix} \dot{\alpha}_x - h \dot{\alpha}_y \\ \dot{\alpha}_y + h \dot{\alpha}_x \\ \dot{\alpha}_2 + h \end{bmatrix} \times (I_s \begin{bmatrix} \dot{\alpha}_x - h \dot{\alpha}_y \\ \dot{\alpha}_1 + h \dot{\alpha}_x \\ \dot{\alpha}_2 + h \end{bmatrix}) +$$

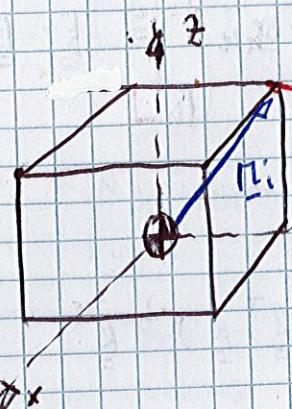
$$+ \begin{bmatrix} \dot{\alpha}_x - h \dot{\alpha}_y \\ \dot{\alpha}_y + h \dot{\alpha}_x \\ \dot{\alpha}_2 + h \end{bmatrix} \times \begin{bmatrix} I_{x1} \cdot \underline{R}_1 \\ I_{x2} \cdot \underline{R}_2 \\ I_{x3} \cdot \underline{R}_3 \end{bmatrix} = \underline{M}_{dL} + \underline{M}_c.$$

$\rightarrow$  computing separately the 2 cross products

$$1) \quad \underline{W} \times \dot{H}_{c,S} = \begin{bmatrix} 1 & 3 & 2 \\ (\dot{\alpha}_x - h \dot{\alpha}_y) & (\dot{\alpha}_y + h \dot{\alpha}_x) & (\dot{\alpha}_2 + h) \\ I_x(\dot{\alpha}_x - h \dot{\alpha}_y) & I_y(\dot{\alpha}_y + h \dot{\alpha}_x) & I_z(\dot{\alpha}_2 + h) \end{bmatrix}$$

Thrusters: for start option

$\Rightarrow$  NON LINEAR CONTROL.



on thrusters are put in the "center" of the body in order to maximize the Torque.

!! Usually thrusters have a fixed direction

$$\rightarrow \underline{M}_i = \underline{\alpha}_i \times \underline{F}_i = \underline{\zeta}_i \times \underline{\xi}_i \underline{F}_i$$

$\left\{ \begin{array}{l} \underline{\zeta}_i = \begin{bmatrix} \zeta_{x,i} \\ \zeta_{y,i} \\ \zeta_{z,i} \end{bmatrix} \\ \underline{\xi}_i = \begin{bmatrix} \xi_{x,i} \\ \xi_{y,i} \\ \xi_{z,i} \end{bmatrix} \end{array} \right.$  projection of each i-thruster along the poi. axis.

$\rightarrow$  Considering many actuators

$$\underline{M} = \underline{\alpha}_i \times \underline{\xi}_i \Rightarrow \underline{M} = \frac{1}{2} \underline{\zeta}_i \times \underline{\xi}_i = \frac{1}{2} \underline{\alpha}_i \times \underline{\zeta}_i \underline{F}_i$$

That in matrix form becomes:

$$\underline{M} = \left[ \begin{array}{ccc} \underline{\zeta}_1 \times \underline{\xi}_1 & \dots & \underline{\zeta}_n \times \underline{\xi}_n & \underline{F}_1 \\ & \vdots & \vdots & \vdots \\ & & & \underline{F}_n \end{array} \right]$$

2-Thrusters

$$\underline{M}_1 = \underline{\alpha}_1 \times \underline{\xi}_1 \underline{F}_1 = \left[ \begin{array}{c} 1 \\ 0 \\ 0 \end{array} \right] \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \underline{F}_1 \\ \underline{F}_2 \\ \underline{F}_3 \end{bmatrix} = \begin{bmatrix} \underline{F}_1 \\ \underline{F}_2 \\ \underline{F}_3 \end{bmatrix}$$

$$\underline{M}_2 = \underline{\alpha}_2 \times \underline{\xi}_2 \underline{F}_2 = \begin{bmatrix} 0 & 1 & 0 \\ 0 & 0 & 1 \\ 1 & 0 & 0 \end{bmatrix} \begin{bmatrix} \underline{F}_1 \\ \underline{F}_2 \\ \underline{F}_3 \end{bmatrix} = \begin{bmatrix} \underline{F}_1 \\ \underline{F}_2 \\ \underline{F}_3 \end{bmatrix}$$

$$\underline{M} = \left[ \begin{array}{cc} \underline{\alpha}_1 \times \underline{\xi}_1 & \underline{\alpha}_2 \times \underline{\xi}_2 \end{array} \right] \begin{bmatrix} \underline{F}_1 \\ \underline{F}_2 \end{bmatrix} =$$

$\Rightarrow$  for small rotations (as previously demonstrated)

is possible to obtain  $\underline{\omega}$  such that:

$$\underline{\omega}_{HUV2,PL} = \begin{bmatrix} \cdot & \cdot & \cdot \\ \end{bmatrix} \cdot \begin{bmatrix} \dot{\alpha}_x \\ \dot{\alpha}_y \\ \dot{\alpha}_z \end{bmatrix}$$

$$*\underline{\omega}_{BS}^T \underline{\tau}_B = \underline{\omega}_{BS}^T \underline{\tau}_B + \underline{\omega}_0^T \underline{\tau}_B = \begin{bmatrix} \dot{\alpha}_x - h\dot{\alpha}_y \\ \dot{\alpha}_y + h\dot{\alpha}_x \\ \dot{\alpha}_z + h \end{bmatrix}$$

→ EULER's equation:  $\underline{\dot{H}}_{c,s} + \underline{\omega} \times (\underline{\dot{H}}_{c,s} + \underline{\dot{H}}_{c,R}) = \underline{M}_d - \underline{\dot{H}}_{c,R}$

→  $\underline{\omega}$  as function of small rotations from orbital term:

$$\underline{\omega} = \begin{bmatrix} \dot{\alpha}_x + h\dot{\alpha}_y \\ \dot{\alpha}_y + h\dot{\alpha}_x \\ \dot{\alpha}_z + h \end{bmatrix} \quad \begin{array}{l} \dot{\alpha}_x \rightarrow \text{AROUND YAW AXIS} \\ \dot{\alpha}_y \rightarrow \text{AROUND ROLL AXIS} \\ \dot{\alpha}_z \rightarrow \text{AROUND PITCH AXIS} \end{array}$$

$$\frac{d}{dt} \left( \begin{bmatrix} \dot{\alpha}_x + h\dot{\alpha}_y \\ \dot{\alpha}_y + h\dot{\alpha}_x \\ \dot{\alpha}_z + h \end{bmatrix} \right) + \begin{bmatrix} \dot{\alpha}_x - h\dot{\alpha}_y \\ \dot{\alpha}_y + h\dot{\alpha}_x \\ \dot{\alpha}_z + h \end{bmatrix} \times \left( I_s \begin{bmatrix} K_x + h\alpha_x \\ K_y + h\alpha_x \\ K_z + h \end{bmatrix} \right) +$$

$$+ \begin{bmatrix} \dot{\alpha}_x - h\dot{\alpha}_y \\ \dot{\alpha}_y + h\dot{\alpha}_x \\ \dot{\alpha}_z + h \end{bmatrix} \times \begin{bmatrix} I_{zL} - R_1 \\ I_{z2} - R_2 \\ I_{z3} - R_3 \end{bmatrix} = \underline{M}_d + \underline{M}_c.$$

→ Computing separately the 2 mass products

$$1) \underline{\omega} \times \underline{H}_{c,s} = \begin{bmatrix} 1 & 3 & K \\ (Kx - h\dot{\alpha}_y) & (Ky + h\dot{\alpha}_x) & (\dot{\alpha}_z + h) \\ I_x(\dot{\alpha}_x - h\dot{\alpha}_y) & I_y(\dot{\alpha}_y + h\dot{\alpha}_x) & I_z(\dot{\alpha}_z + h) \end{bmatrix}$$

$$\underline{\omega} \times \underline{H}_{c,s} = \begin{bmatrix} 1 & [I_z(\dot{\alpha}_z \dot{\alpha}_y + h\dot{\alpha}_y + hK_x \dot{\alpha}_z + h^2 \dot{\alpha}_x) - I_y(h\dot{\alpha}_y + h^2 \dot{\alpha}_x)] \\ 3 & [I_x(h\dot{\alpha}_x - h^2 \dot{\alpha}_y) - I_z(h\dot{\alpha}_x - h^2 \dot{\alpha}_y)] \\ K & [I_y(\dot{\alpha}_x \dot{\alpha}_y + h\dot{\alpha}_x K_x - h\dot{\alpha}_y \dot{\alpha}_x - h^2 \dot{\alpha}_y K_x) - I_x(0)] \end{bmatrix}$$

$$\underline{\omega} \times \underline{H}_{c,s} = \begin{bmatrix} (I_z - I_y) (h\dot{\alpha}_y + h^2 \dot{\alpha}_x) \\ (I_x - I_z) (h\dot{\alpha}_x - h^2 \dot{\alpha}_y) \\ 0 \end{bmatrix}$$

$$2) \underline{\omega} \times \underline{H}_{c,R} = \begin{bmatrix} 1 & 3 & K \\ (Kx - h\dot{\alpha}_y) & (Ky + h\dot{\alpha}_x) & (\dot{\alpha}_z + h) \\ h\dot{\alpha}_x & h\dot{\alpha}_y & h\dot{\alpha}_z \\ (h\alpha_x = I_{zL} R_1) & (h\alpha_y = I_{z2} R_2) & (h\alpha_z = I_{z3} R_3) \end{bmatrix}$$

$$\underline{\omega} \times \underline{H}_{c,R} = \begin{bmatrix} h\dot{\alpha}_x (Ky + h\dot{\alpha}_x) - h\dot{\alpha}_y (\dot{\alpha}_z + h) \\ h\dot{\alpha}_x (\dot{\alpha}_z + h) - h\dot{\alpha}_z (\dot{\alpha}_x - h\dot{\alpha}_y) \\ h\dot{\alpha}_y (\dot{\alpha}_x - h\dot{\alpha}_y) - h\dot{\alpha}_x (\dot{\alpha}_y + h\dot{\alpha}_x) \end{bmatrix}$$

→ Euler's equation then becomes:

F Euler's equation (3 rot.) specialized around 10° condition.

$$I_x(\ddot{\alpha}_x - h\ddot{\alpha}_y) + (I_z - I_y)(h\dot{\alpha}_y + h^2 \dot{\alpha}_x) + h\dot{\alpha}_z(\dot{\alpha}_y + h\dot{\alpha}_x) - h\dot{\alpha}_y(\dot{\alpha}_z + h) = M_x$$

$$I_y(\ddot{\alpha}_y + h\ddot{\alpha}_x) + (I_x - I_z)(h\dot{\alpha}_x - h^2 \dot{\alpha}_y) + h\dot{\alpha}_x(\dot{\alpha}_z + h) - h\dot{\alpha}_z(\dot{\alpha}_x - h\dot{\alpha}_y) = M_y$$

$$I_z \ddot{\alpha}_z + h\dot{\alpha}_y(\dot{\alpha}_x - h\dot{\alpha}_y) - h\dot{\alpha}_x(\dot{\alpha}_y + h\dot{\alpha}_x) = M_z.$$

$$\underline{M} = \underline{M}_d + \begin{bmatrix} I_{zL} & I_1 \\ I_{z2} & I_2 \\ I_{z3} & I_3 \end{bmatrix}$$

$$\Rightarrow \underline{M}_c = \begin{bmatrix} I_{zL} R_1 \\ I_{z2} R_2 \\ I_{z3} R_3 \end{bmatrix}$$

$$I_x(\ddot{\alpha}_x - h\ddot{\alpha}_y) + (I_z - I_y)(h\dot{\alpha}_y + h^2\alpha_x) + h\alpha_z(\dot{\alpha}_y + h\alpha_x) - h\alpha_y(\dot{\alpha}_z + h) = M_x$$

$$I_y(\ddot{\alpha}_y + h\dot{\alpha}_x) + (I_x - I_z)(h\dot{\alpha}_x - h^2\alpha_y) + h\alpha_x(\dot{\alpha}_z + h) - h\alpha_z(\dot{\alpha}_x - h\dot{\alpha}_y) = M_y$$

$$I_z(\dot{\alpha}_z + h\alpha_y(\dot{\alpha}_x - h\dot{\alpha}_y) - h\alpha_x(\dot{\alpha}_y + h\dot{\alpha}_x) = M_z$$

$M = M_d + M_c$ , where (for now) the gravity gradient torque is included inside the disturbances, and contributes to the dynamic of the system (even stabilizing it)

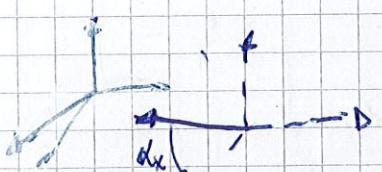
$\rightarrow M_g$

$$M_g = 3 \frac{\mu}{R^3} \begin{vmatrix} (I_z - I_y) c_2 c_3 \\ (I_x - I_y) c_1 c_3 \\ (I_y - I_x) c_1 c_2 \end{vmatrix}$$

$$c_1 = \frac{R \cdot h}{L} = \cos(\alpha_x) \approx 1$$

$$c_2 = \frac{R \cdot L}{h} \approx \sin(\alpha_y) \approx K_y$$

$$c_3 = \frac{L \cdot h}{R} \approx K_z.$$



$$+ \quad h = \sqrt{\frac{\mu}{R^3}} \Rightarrow \frac{\mu}{R^3} = h^2$$

$$\sqrt{\frac{\mu}{R^3}} \Delta t = R - e^{-\text{init.}}$$

Pitch control ( $\alpha_y$ -axis)  $\rightarrow$  set-point:  $\hat{\alpha}_y$  aligned with  $\alpha_0$ .

Assumptions: Rotors' angular moments are hereby neglected.

(supposed to be really less massive than the spacecraft)

$$h\alpha_y \approx h\alpha_x \approx 0.$$

obs: Pitch Euler's equation is decoupled from the other 2.

Under Assumption 2 validity

Model to design control law be slightly different from reality

but must be enough robust to this uncertainty

$$I_z \ddot{\alpha}_z = (3h^2(I_y - I_x))K_z \xrightarrow{\mathcal{L}} I_z s^2 K_z(s) = 3h^2(I_y - I_x)K_z(s)$$

$$I_z s^2 K_z(s) + 3h^2(I_x - I_y)K_z(s) = 0$$

System has 2 complex-conjugated poles stable if and only if  $I_x > I_y$

$\hookrightarrow$  Neglecting at this stage the gravity-gradient.

(still considering it a general disturb.)

$$G(s) = \frac{K_z(s)}{M_d(s)} = \frac{1}{I_z s^2}$$

$$\frac{M_d(s)}{\longrightarrow} \boxed{\frac{1}{I_z s^2}} \xrightarrow{K_z(s)}$$

$\hookrightarrow$  Choosing of implement on PD controller

$$M_c = -h_z = K_p z K_z + K_d z \dot{K}_z$$

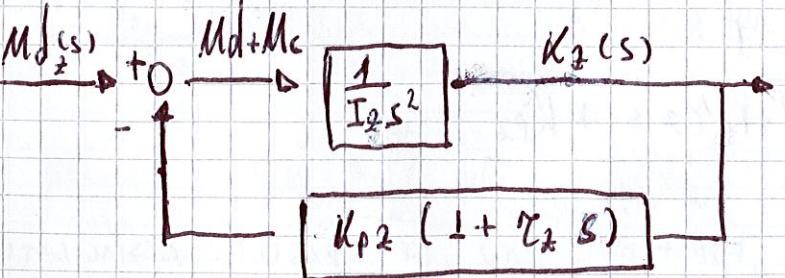
$$\rightarrow h_z = -K_p z K_z - K_d z \dot{K}_z.$$

By exploiting derivative time:

$$\rightarrow h_z = -K_p z (K_z + \gamma_z \dot{K}_z)$$

$$\gamma_z \triangleq \frac{K_d z}{K_p z}.$$

$\hookrightarrow$  feed-back (cc system) is then defined.



$\uparrow$  feedback torque function would be:

$$h_z = -K_p z (K_z + \gamma_z \dot{K}_z)$$

$\downarrow$

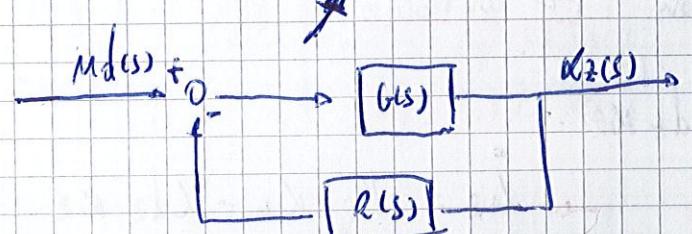
$$h_z(s) = K_p z (1 + \gamma_z s) K_z(s)$$

Obs: since the set-point condition are  $\begin{cases} \alpha_x = 0 \\ \alpha_y = 0 \\ K_2 = 0 \end{cases}$

then implementing the controller for  $\alpha_z$  instead than for the force the effect is the same.

Closed loop function will then be:

$$\frac{\alpha_z(s)}{M_d(s)} = \frac{1/(I_2 s^2)}{1 + \left(\frac{1}{I_2 s^2}\right) [K_{p2}(1 + \gamma_2 s)]}$$



$$\alpha_z(s) = G(s) \cdot (M_d(s) - R(s) \cdot \alpha_z(s))$$

$$\alpha_z(1 + R(s)G(s)) = G(s) \cdot M_d(s) \Rightarrow \frac{\alpha_z(s)}{M_d(s)} = \frac{G(s)}{1 + R(s)G(s)}$$

$$\frac{\alpha_z(s)}{M_d(s)} = \frac{1}{I_2 s^2 + K_{p2} \gamma_2 s + K_{p2}}$$

$\sqrt{I_2}$ ,  $\sqrt{W_h}$ ,  $\sqrt{K_{p2}}$

[ THIS TRANSFER FUNCTION CAN BE EASILY ASSIMILATED TO A SPRING-MASS-DAMPER SYSTEM:

$$I_2 s^2 + \gamma_2 s + K_{p2} \leftrightarrow m \ddot{x} + c\dot{x} + kx = f(t)$$

$$\left( \frac{x(s)}{R(s)} = \frac{1}{m s^2 + c s + k} \right)$$

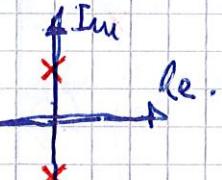
[ THE POLES OF THIS CLOSED LOOP SYSTEM ARE ASINTOTICALLY STABLE ]

$$\times OL: \frac{\alpha_z(s)}{M_d(s)} = \frac{1}{I_2 s^2} \rightarrow p_{1,2} = 0$$

$$\times OL: \frac{\alpha_z(s)}{M_d(s)} = \frac{1}{I_2 s^2 + 3h^2(I_x - I_y)} \rightarrow p_{1,2} = \pm j \sqrt{\frac{3h^2(I_x - I_y)}{I_2}}$$

(with  $M_d(s)$ )

!! gravity gradient torque add some stiffness to the system. !!



( $\Leftrightarrow I_x > I_y \parallel I_{yaw} > I_{roll}$ )

$$I_2 s^2 + 3h^2 (I_x - I_y) = 0 \rightarrow$$

$$s^2 = -\frac{3h^2 (I_x - I_y)}{I_2} \rightarrow s_{1,2} = \pm j \sqrt{\frac{3h^2 (I_x - I_y)}{I_2}}$$

$$\times CL: \frac{\alpha_z(s)}{M_d(s)} = \frac{1}{I_2 s^2 + K_{p2} \gamma_2 s + K_{p2}} = \frac{1/I_2}{s^2 + \frac{K_{p2} \gamma_2}{I_2} s + \frac{K_{p2}}{I_2}}$$

[ SUCH TRANSFER FUNCTION IS COMPLETELY EQUIVALENT TO A SPRING-MASS-DAMPER'S TRANSFER FUNCTION:

$$s^2 + \frac{K_{p2} \gamma_2}{I_2} s + \frac{K_{p2}}{I_2} \leftrightarrow s^2 + 2\xi W_h + W_h^2$$

$$\text{where: } W_h = \sqrt{\frac{K_{p2}}{I_2}} ; \quad \xi = \frac{1}{2} \gamma_2 \sqrt{\frac{K_{p2}}{I_2}}$$

$$p_{1,2} = -\xi W_h \pm j \sqrt{1 - \xi^2} W_h \quad \left( \xi = \frac{1}{2} \frac{K_{p2}}{\sqrt{K_{p2} I_2}} \right)$$

in fact:

$$2\xi W_h = \frac{K_{p2} \gamma_2}{I_2} \rightarrow \sqrt{\frac{K_{p2}}{I_2}} \cdot \sqrt{\frac{K_{p2}}{I_2}} \cdot \gamma_2 = W_h \gamma_2 \cdot \sqrt{\frac{K_{p2}}{I_2}} = 2\xi W_h$$

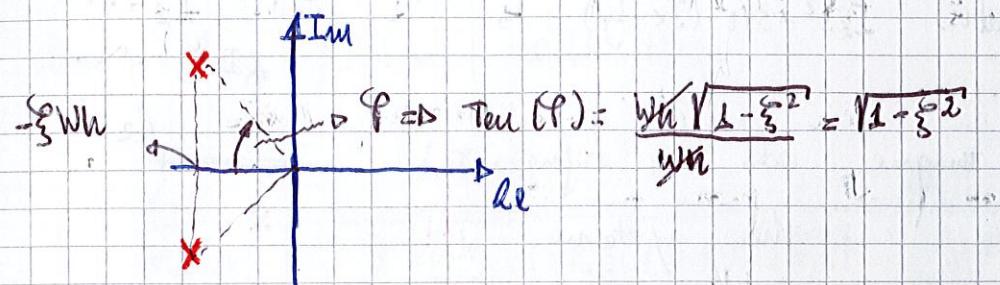
$$\Rightarrow \begin{cases} 2\xi = \gamma_2 \cdot \sqrt{\frac{K_{p2}}{I_2}} \\ \gamma_2 = \frac{K_{p2}}{K_{p2}} \end{cases} \rightarrow \begin{cases} \xi = \frac{1}{2} \gamma_2 \sqrt{\frac{K_{p2}}{I_2}} \\ \xi = \frac{1}{2} \frac{K_{p2}}{\sqrt{K_{p2} I_2}} \end{cases}$$

Poles in such condition will be:

$$s^2 + 2\xi\omega_n s + \omega_n^2 \rightarrow -\xi\omega_n \pm \sqrt{\xi^2\omega_n^2 - \omega_n^2}$$

( $\xi < 1 \Rightarrow$  the  $\sqrt{\cdot}$  term will be imaginary)

$$s^2 + 2\xi\omega_n s + \omega_n^2 \rightarrow p_{1,2} = -\xi\omega_n \pm j\omega_n\sqrt{1-\xi^2}$$



**obs1:** Since in the situation of inertial pointing, the linearized equations are the following:

$$\begin{cases} I_x \ddot{\alpha}_x = M_d x + M_a x \\ I_y \ddot{\alpha}_y = M_d y + M_a y \\ I_z \ddot{\alpha}_z = M_d z + M_a z \end{cases} \quad \begin{cases} I_x s^2 \alpha_x(s) = M_d(s) \\ I_y s^2 \alpha_y(s) = M_d(s) \\ I_z s^2 \alpha_z(s) = M_d(s) \end{cases}$$

THEN (Only introducing linearized conditions)  
the 3 equations are fully decoupled.

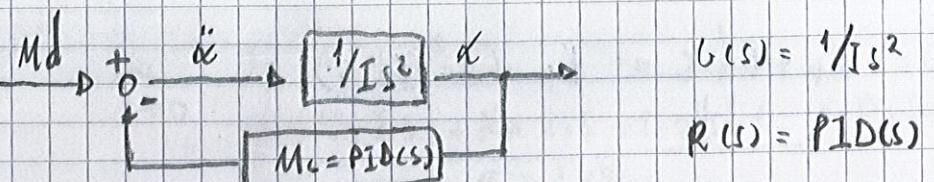
$\Rightarrow$  are coincident with the one seen just now for the control on the pitch axis: A double integrator.

TO BE CONTROLLED WITH A PID  
CONTROL LAW:

$i = x, y, z$

$$M_{ci} = K_{pi} e_i(t) + K_{di} \dot{e}_i(t) + K_{ii} \int_0^t e_i(\tau) d\tau$$

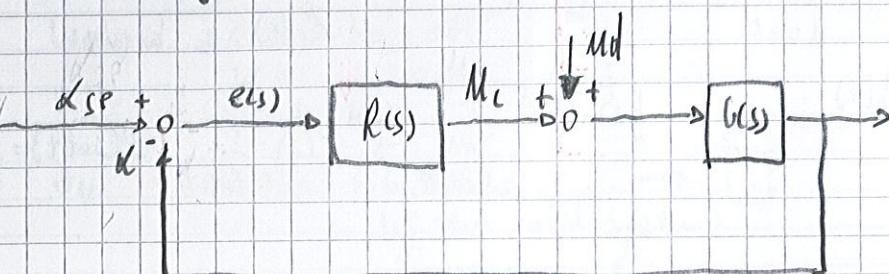
**obs2:** Only about how to design the loop (closed loop)



$$[Md - R(s) \cdot \alpha(s)] \cdot G(s) = K(s)$$

$$G(s) \cdot Md = [1 + L(s) \cdot G(s)] \alpha(s) \Rightarrow \frac{K(s)}{Md(s)} = \frac{G(s)}{[1 + L(s)G(s)]}$$

Result is the same even considering the following representation  
possible also of show derivation from a SP



$$(Mc + Md) \cdot G(s) = \alpha(s) \Rightarrow [R(s)(\alpha_{sp} - \alpha) + Md] \cdot G(s) = K(s)$$

$$\Rightarrow Md \cdot G(s) = -R(s)(\alpha_{sp} - \alpha) + K \Rightarrow \frac{K(s)}{Md} = \frac{G(s)}{1 + L(s)G(s)} \quad (K_{sp}(s) = 0)$$

The transfer function  $Md \rightarrow \alpha$  //  $Md \rightarrow \alpha - \alpha_{sp}$  is the same for both the configurations. !! THIS NEW DRAW IS ALSO CAPABLE OF SHOWING THE TRANSFER FUNCTION:  $\alpha_{sp} \rightarrow \alpha$

(VARIATION IN THE ACCORDANCE)

$$(\alpha_{sp} - \alpha) \cdot R(s) \cdot G(s) = K(s) \Rightarrow R(s) \cdot G(s) \cdot K_{sp} = \alpha(s)(1 + R(s)G(s))$$

$$\Rightarrow \frac{K(s)}{K_{sp}} = \frac{R(s)G(s)}{1 + R(s)G(s)}$$

DISTURBANCE SENSITIVITY FUNCTION  $\rightarrow \frac{d(s)}{Md(s)} = \frac{G(s)}{1 + R(s)G(s)}$

TRACKING SENSITIVITY FUNCTION  $\rightarrow \frac{K(s)}{K_{sp}(s)} = \frac{R(s)G(s)}{1 + R(s)G(s)}$

Poles

$$R(s) = K_p + K_d s = K_p (1 + \gamma_d s)$$

$$\text{PID: } \frac{R(s)}{E(s)} = K_p + K_d s + \frac{K_i}{s} \rightarrow R(s) = K_p (1 + \gamma_d s + \frac{1}{\gamma_i s})$$

$$\gamma_d \stackrel{!}{=} \frac{K_d}{K_p} \quad \& \quad K \gamma_i \stackrel{!}{=} \frac{K_p}{K_i} \Rightarrow R(s) = K_p \left( \frac{\gamma_i s + \gamma_i \gamma_d s^2 + 1}{\gamma_i s} \right)$$

→ final value theorem

$$\lim_{t \rightarrow \infty} f(t) = \lim_{s \rightarrow 0} s \cdot F(s)$$

to be used with

the  $\sin(\omega t)$  as input

$$F(s) = \mathcal{L}(f(t))$$

$$\lim_{s \rightarrow 0} s \cdot F(s) \cdot \frac{1}{s} \quad (\delta(\omega t) = \frac{1}{s})$$

\* REFERENCE TRACKING

$$\text{PD: } \lim_{s \rightarrow 0} s \cdot \frac{K_p(1 + \gamma_d s) \cdot \frac{1}{s^2}}{1 + \frac{K_p(1 + \gamma_d s)}{s^2}} = \lim_{s \rightarrow 0} s \cdot \frac{K_p + K_d s}{K_p + K_d s + I_s^2} \cdot \frac{1}{s} = 1 \quad (\text{OK})$$

$$\text{PID: } \lim_{s \rightarrow 0} s \cdot \frac{K_p \left( \frac{\gamma_i s + \gamma_i \gamma_d s^2 + 1}{\gamma_i s} \right) \cdot \frac{1}{s^2}}{1 + \frac{K_p}{I_s^2} \left( \frac{\gamma_i s + \gamma_i \gamma_d s^2 + 1}{\gamma_i s} \right)} = 1$$

$$= \lim_{s \rightarrow 0} s \cdot \frac{K_p \gamma_i s + K_p \gamma_i \gamma_d s^2 + K_p}{\gamma_i I_s^3 + K_p \gamma_i s + K_p \gamma_i \gamma_d s^3 + K_p} \cdot \frac{1}{s} = 1 \quad (\text{OK})$$

\* DISTURBANCE REJECTION

$$\text{PD: } \lim_{s \rightarrow 0} s \cdot \frac{\frac{1}{s^2}}{1 + K_p (1 + \gamma_d s) \cdot \frac{1}{s^2}} = s \cdot \frac{\frac{1}{s^2}}{\frac{1}{s^2} + K_p + K_p \gamma_d s} \cdot \frac{1}{s}$$

$$\approx \lim_{s \rightarrow 0} \frac{1}{\frac{1}{s^2} + K_p + K_p \gamma_d s} = \frac{1}{K_p} \quad \text{NOT OK}$$

[ At steady state conditions the PD controller HAVE AN ERROR (FOR A STEP INPUT) EQUAL TO:  $1/K_p$  ]

$$\text{PID: } \lim_{s \rightarrow 0} s \cdot \frac{\frac{1}{s^2}}{1 + \frac{K_p}{I_s^2} \left( \frac{\gamma_i s + \gamma_i \gamma_d s^2 + 1}{\gamma_i s} \right)} = \frac{\frac{1}{s^2}}{\frac{1}{s^2} + \frac{K_p}{I_s^2} \left( \frac{\gamma_i s + \gamma_i \gamma_d s^2 + 1}{\gamma_i s} \right)}$$

$$\frac{\frac{1}{s^2}}{\frac{\gamma_i I_s^3 + K_p \gamma_i s + \gamma_i \gamma_d s^2 + 1}{\gamma_i I_s^2}} = \frac{\gamma_i s}{\gamma_i I_s^3 + K_p \gamma_i s + \gamma_i \gamma_d s^2 + 1} = 0 \quad (\text{OK})$$

[ therefore in order to have a error for a step disturbance

A PID controller must be improved

To tune a PID requirements of the control system must be converted into  $\{\omega_n, \zeta\}$  specifications for the controller.

\* \*

$\omega_n \leftrightarrow$  rising time  $\rightarrow T_r$   
 $\zeta \leftrightarrow$  settling time  $\rightarrow T_p$

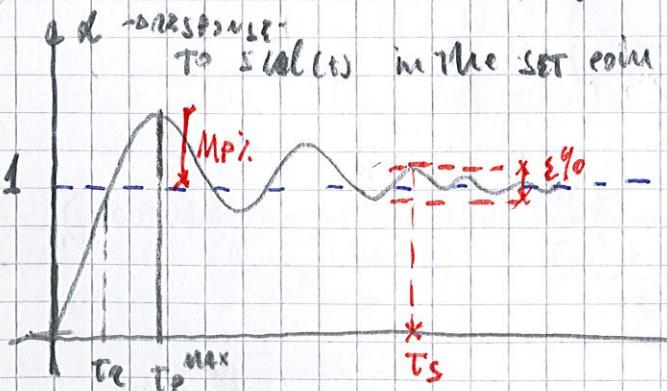
$\phi_m \leftrightarrow$  damping (introduced by the control system)  $\rightarrow \zeta$

linked also to the maximum % overshoot permissible.

Always looking at the response of the system for a step disturbance.

$$\phi_m = 130^\circ + \angle L(j\omega) = \tan^{-1} \left( \frac{2\zeta}{\sqrt{1 - \zeta^2}} \right)$$

$$\omega_n = \frac{\pi}{T_p \max \sqrt{1 - \zeta^2}}, \quad \omega_n = -\ln \left( \frac{\epsilon \% \sqrt{1 - \zeta^2}}{T_s \max \cdot \zeta} \right)$$



$$M_p = \theta^{\max}$$

admissible deviation  
from the nominal position.

$$T_p \cdot M_p = e^{-\frac{\pi}{T_p \cdot \zeta}} \times 100\%$$

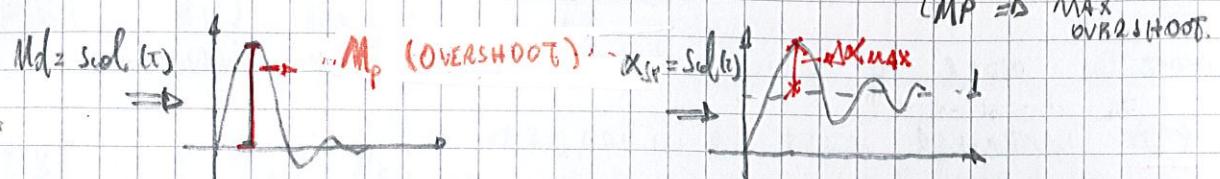
This condition DETERMINE THE DAMPING OF THE SYSTEM.

$$\frac{\Delta \theta^{\max}}{\theta^{\max}} = \zeta \approx 0.3 \dots$$

- procedure my.

(1) determine  $M_d - \text{max}$  (perturbation analysis by look axis)

(2) choose the maximum deviation from the  $K - s\omega = \Delta K$



$$\frac{M_p}{M_{d\max}} \parallel \frac{\Delta K_{\max} \cdot \tau_0}{I} \rightarrow \text{choose the most short equipment.}$$

(3) Determine the damping

$$\frac{M_p}{M_{d\max}} \parallel \frac{\Delta K_{\max} \cdot \tau_0}{I} = e^{-\frac{1}{\zeta^2}} \rightarrow \boxed{\zeta}$$

(4) choose the values for  $\left\{ \begin{array}{l} T_e(\tau_p) \\ \tau_s^{\max} \end{array} \right\}$  and solve for  $v_m$

$$w_h = \frac{1}{\tau_p^{\max} \sqrt{1-\zeta^2}} \parallel w_h = -\frac{\ln(s)}{\tau_s^{\max} \zeta} \rightarrow \boxed{w_h}$$

(5) Recording of the various positions:

$$\phi_m = \tan^{-1} \left( \frac{2\zeta}{\sqrt{1-\zeta^2}} \right), w_c = w_h$$

(6) Design the PID controller to respect  $\left\{ \begin{array}{l} b_m \\ w_c \end{array} \right\}$  (SISO TOOL)

MENGALI: Reaction wheels sizing (and selection between manufactured model)

Control relation between the actuators and the nominal torque is the following:

$$\begin{cases} M_c = -h \dot{\tau} \\ h = I_a \cdot \dot{\tau} \end{cases}$$

e.g. (PD initial law)

$$\Rightarrow I_a \cdot \dot{\tau} = -K_p (\alpha + \gamma \dot{\alpha})$$

Going into Laplace domain is possible to obtain a relation linking  $\underline{R}(s)$  to the disturbance torque:

$$\mathcal{L}[j(\tau)] = s \cdot F(s) - f(0)$$

$$\Rightarrow I_a [s \cdot \underline{R}(s) - \underline{R}(0)] = -K_p (\gamma s + 1) \cdot \alpha(s) \quad \leftarrow$$

$$I_a \cdot \dot{\tau} = -K_p (\alpha + \gamma \dot{\alpha}) \xrightarrow{\mathcal{L}} I_a (\underline{R}(s) - \underline{R}(0)) = -K_p (\alpha(s) + \gamma s \alpha(s))$$

Therefore:  $s \cdot \underline{R}(s) = \underline{R}(0) - \frac{K_p (\gamma s + 1)}{I_a} \cdot \alpha(s)$

$$\underline{R}(s) = \frac{\underline{R}(0)}{s} - \frac{K_p (\gamma s + 1)}{I_a} \cdot \frac{\alpha(s)}{s}$$

Using the closed loop transfer function:  $\frac{\alpha(s)}{M_d(s)} = \frac{b(s)}{1 + R(s) \cdot G(s)}$

$$\frac{\alpha(s)}{M_d(s)} = \frac{1/I_s^2}{1 + \frac{K_p}{I} \frac{1}{s^2} (1 + \gamma s)} = \frac{1/I_s^2}{s^2 + K_p \gamma s + K_p} = \frac{1/I}{s^2 + K_p \frac{\gamma}{I} s + \frac{K_p}{I}}$$

$$W \triangleq \sqrt{\frac{K_p}{I}}$$

$$\Rightarrow \frac{\alpha(s)}{M_d(s)} = \frac{1/I}{s^2 + W^2 \gamma s + W^2}$$

Assumption (DESIGN PD): In order to have UNITARY DAMPING,

it's necessary to select  $\gamma_p \triangleq \frac{K_d}{K_p} = 2 \sqrt{\frac{I}{K_p}}$

in fact:

$$\frac{d(s)}{M_d(s)} = \frac{1/I}{s^2 + w^2 \gamma s + w^2} \leftrightarrow \frac{1/I}{s^2 + 2\zeta w s + w^2}$$

$$\Rightarrow \zeta \neq 1 = \frac{\zeta}{w} \Rightarrow \zeta \leq 1$$

$$\Rightarrow 2\zeta w = w^2 \gamma \xrightarrow{\zeta=1} \gamma w^2 = 2w \rightarrow \gamma = 2/w = 2\sqrt{\frac{I}{K_p}}$$

Therefore:

$$\frac{d(s)}{M_d(s)} = \frac{1/I}{s^2 + K_p \frac{\gamma}{I} + \frac{K_p}{I}} = \frac{1/I}{s^2 + \gamma w^2 s + w^2} = \frac{1/I}{s^2 + 2\zeta w s + w^2}$$

$$\Rightarrow \frac{K(s)}{M_d(s)} = \frac{1/I}{s^2 + 2ws + w^2} = \frac{1/I}{(s+w)^2}$$

### PS2 - complement

By selecting in this way the PD controller (such that

$$\zeta=1 \rightarrow \frac{K_d}{K_p} = 2\sqrt{\frac{I}{K_p}}$$

Using the final value theorem for switch in this condition the answer to a scal(t) disturbance

$\lim_{s \rightarrow 0} s \cdot \frac{1/I}{(s+w)^2} \cdot \frac{1}{s} = \frac{1/I}{w^2}$  but if  $w$  is high enough it is possible to reduce the error approximately to zero.

↳ Substituting this expression into the  $R(s)$  transfer function,

$$R(s) = \frac{R(0)}{s} - \frac{K_p(\gamma s + 1)}{IR} \cdot \frac{K(s)}{s}$$

$$\frac{d(s)}{M_d(s)} = \frac{1/I}{(s+w)^2} \quad (w = \sqrt{\frac{K_p}{I}})$$

$$\Rightarrow R(s) = \frac{R(0)}{s} - \frac{K_p(\gamma s + 1)}{IR} \cdot \frac{1}{s} \cdot \frac{1/I}{(s+w)^2}$$

$$\Rightarrow R(s) = \frac{R(0)}{s} - \frac{2\gamma s + 1}{IR s (\gamma s + 1)^2} M_d(s)$$

$$R(s) = \frac{R(0)}{s} - \frac{K_p}{I} \cdot \frac{(\gamma s + 1)}{IR} \cdot \frac{1}{s} \cdot \frac{1}{w^2 (\frac{1}{w} s + 1)^2}$$

$$= \frac{R(0)}{s} - w^2 \cdot \frac{(\gamma s + 1)}{IR} \cdot \frac{1}{s} \cdot \frac{1}{w^2 (\frac{1}{w} s + 1)^2} \quad || \quad \gamma = \frac{2}{w}$$

\* Response to imp(t) ( $\mathcal{L}[imp(t)] = 1$ )

$$\lim_{s \rightarrow 0} s \cdot R(s) = \left( \frac{R(0)}{s} - \frac{(2\gamma s + 1) \cdot M_d}{IR s (\gamma s + 1)} \right) \rightarrow = R(t \rightarrow 0)$$

$$R(t \rightarrow 0) = R(0) - \frac{M_d}{IR}$$

$$\underbrace{R(t \rightarrow 0) - R(0)}_{\Delta R_{max}} = \frac{M_d}{IR}$$

\* Response to scal(t)

$$\lim_{s \rightarrow 0} s \cdot R(s) \cdot \frac{1}{s} = \frac{R(0)}{s} - \frac{2\gamma s + 1 \cdot M_d}{IR (\gamma s + 1)}$$

theoretically  $\infty$  but practically the result is the same obtained for the impulse response.

\* Periodic disturb ( $\mathcal{L}[sin(w^* t)] = \frac{w^*}{s^2 + w^{*2}}$ )

$$\lim_{s \rightarrow 0} \frac{w}{s^2 + w^2} \cdot R(s) \cdot s = R(0) \cdot \frac{w^*}{s^2 + w^{*2}} - \frac{2\gamma s + 1}{IR (\gamma s + 1)} M_d \cdot \frac{w^*}{s^2 + w^{*2}}$$

$$\Rightarrow R(t \rightarrow 0) - R(0) = \frac{M_d}{w^* IR_2}$$

If  $w^* < w$  this disturbance can beat the impulse one.

→ Reaction wheels disposition:

ONCE  $\underline{M}_c$  has been computed (thanks to  $\underline{\underline{A}}_{\underline{\underline{z}}}$ )

THEN  $\underline{M}_r$  can be computed

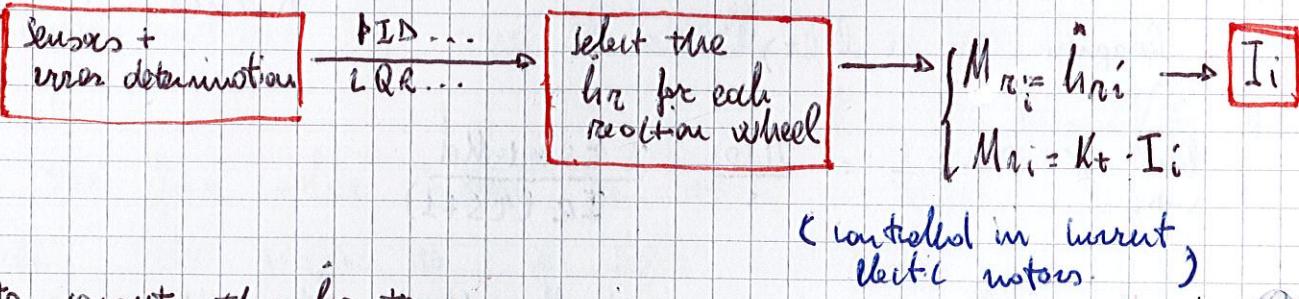
$$\text{Ende with initial: } \rightarrow \underline{\underline{H}} + \underline{W} \times \underline{\underline{H}} = -\underline{\underline{A}}_{\underline{\underline{z}}} \underline{\underline{h}}_r - \underline{W} \times (\underline{\underline{A}}_{\underline{\underline{z}}} \underline{\underline{h}}_r) + \underline{M}_d$$

(capital  $H$  to remember that it's also including rotors' inertia)

$$\Rightarrow \underline{M}_r = -\underline{\underline{A}}_{\underline{\underline{z}}} \underline{\underline{h}}_r - \underline{W} \times (\underline{\underline{A}}_{\underline{\underline{z}}} \underline{\underline{h}}_r)$$

$$\rightarrow \underline{\underline{A}}_{\underline{\underline{z}}} \underline{\underline{h}}_r = -\underline{M}_r - \underline{W} \times (\underline{\underline{A}}_{\underline{\underline{z}}} \underline{\underline{h}}_r)$$

In order to compute the term  $\underline{\underline{h}}_r$  needed to compute the torque that the electric engine have to provide:



⇒ to compute the  $\underline{\underline{h}}_r$  term

$$\text{must solve the equation: } \underline{\underline{A}}_{\underline{\underline{z}}} \underline{\underline{h}}_r = -\underline{M}_r - \underline{W} \times (\underline{\underline{A}}_{\underline{\underline{z}}} \underline{\underline{M}}_r)$$

BUT  $\therefore$  in general → there are more than 3 actuators  
 → actuators are not along the 3 PI axis.

THEN →  $\underline{\underline{A}}$  is not a  $3 \times 3$  matrix

→  $\underline{\underline{A}}$  is not a square matrix  $\Rightarrow \underline{\underline{A}}^{-1}$

IN FACT 3 rotors are enough to solve exactly the initial problem, but in practice a 4th is added for redundancy

→ Pseudo-inverse matrix.

size  $(\underline{\underline{A}}) = 3 \times n$   
 n = no of actuators.

$$\underline{\underline{A}}_{3,n} \cdot \underline{x}_{n,1} = \underline{y}_{3,1} \quad \Rightarrow \quad \underline{\underline{A}}_{3,n}^T \underline{\underline{A}}_{3,n} \underline{x}_{n,1} = \underline{\underline{A}}_{3,n}^T \underline{y}_{3,1}$$

$$(\underline{\underline{A}}_{3,n} \cdot \underline{h}_r = \underline{M}_c)$$

if  $\exists$  the inverse of  $(\underline{\underline{A}}_{3,n}^T \underline{\underline{A}}_{3,n})$

$$\text{then } (\underline{\underline{A}}_{3,n}^T \underline{\underline{A}}_{3,n})^{-1} (\underline{\underline{A}}_{3,n}^T \underline{\underline{A}}_{3,n}) \cdot \underline{x}_{n,1} = (\underline{\underline{A}}_{3,n}^T \underline{\underline{A}}_{3,n})^{-1} \underline{\underline{A}}_{3,n}^T \underline{y}_{3,1}$$

$$\underline{\underline{A}} \cdot \underline{\underline{A}}^T = \begin{bmatrix} 0_{11} & \cdots & 0_{1n} \\ 0_{21} & \cdots & 0_{2n} \\ 0_{31} & \cdots & 0_{3n} \end{bmatrix} \begin{bmatrix} 0_{11} & 0_{21} & 0_{31} \\ 0_{21} & 0_{22} & 0_{23} \\ 0_{31} & 0_{32} & 0_{33} \end{bmatrix} = \begin{bmatrix} 3 \times 3 \\ \vdots \\ 3 \times 3 \end{bmatrix}$$

$$\underline{\underline{A}} \cdot \underline{\underline{h}}_r = \begin{bmatrix} 0_{11} & \cdots & 0_{1n} \\ 0_{21} & \cdots & 0_{2n} \\ 0_{31} & \cdots & 0_{3n} \end{bmatrix} \begin{bmatrix} I_{R1,WRL} \\ I_{R2,WRL} \\ I_{R3,WRL} \end{bmatrix} = \begin{bmatrix} 3 \times 1 \\ \vdots \\ 3 \times 1 \end{bmatrix}$$

$$\underline{\underline{A}}^T \cdot \underline{\underline{A}} = \begin{bmatrix} 0_{11} & 0_{12} & 0_{13} \\ 0_{21} & 0_{22} & 0_{23} \\ 0_{31} & 0_{32} & 0_{33} \end{bmatrix} \begin{bmatrix} 0_{11} & \cdots & 0_{1n} \\ 0_{21} & \cdots & 0_{2n} \\ 0_{31} & \cdots & 0_{3n} \end{bmatrix} = \begin{bmatrix} n \times n \\ \vdots \\ n \times n \end{bmatrix}$$

∴ true size for  $(\underline{\underline{A}}^T \underline{\underline{A}})^{-1}$

therfore:

$$(\underline{\underline{A}}_{3,n}^T \underline{\underline{A}}_{3,n})^{-1} (\underline{\underline{A}}_{3,n}^T \underline{\underline{A}}_{3,n}) \cdot \underline{x}_{n,1} = (\underline{\underline{A}}_{3,n}^T \underline{\underline{A}}_{3,n})^{-1} \underline{\underline{A}}_{3,n}^T \underline{y}_{3,1}$$

$\underline{\underline{x}}_{n,1}$

$$\begin{bmatrix} 0_{11} & \cdots & 0_{1n} \\ 0_{21} & \cdots & 0_{2n} \\ 0_{31} & \cdots & 0_{3n} \end{bmatrix} \begin{bmatrix} 0_{11} & 0_{12} & 0_{13} \\ 0_{21} & 0_{22} & 0_{23} \\ 0_{31} & 0_{32} & 0_{33} \end{bmatrix} \begin{bmatrix} I_{R1,WRL} \\ I_{R2,WRL} \\ I_{R3,WRL} \end{bmatrix}$$

$$= \begin{bmatrix} n \times 3 \\ \vdots \\ n \times 3 \end{bmatrix} \Rightarrow \begin{bmatrix} n \times 3 \\ \vdots \\ n \times 3 \end{bmatrix} \cdot \begin{bmatrix} y_1 \\ y_2 \\ y_3 \end{bmatrix}$$

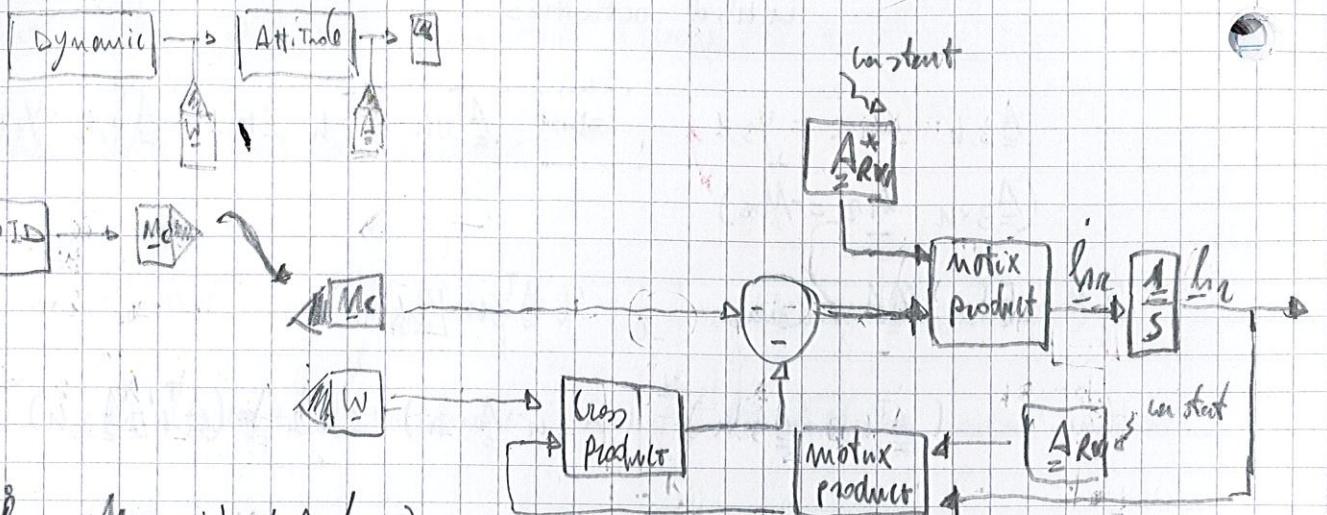
$$\Rightarrow \underline{x}_{n,1} = \underline{\underline{A}}_{3,n}^* \underline{y}_{3,1}$$

$$\underline{\underline{A}}^* = (\underline{\underline{A}}_{3,n}^T \underline{\underline{A}}_{3,n})^{-1} \underline{\underline{A}}_{3,n}^T$$

(size  $(\underline{\underline{A}}^*) = n \times 3$ )

$$\underline{y}_{3,1} \Rightarrow M_c = M_c (\text{PID})$$

A simulation function for this might be the following:

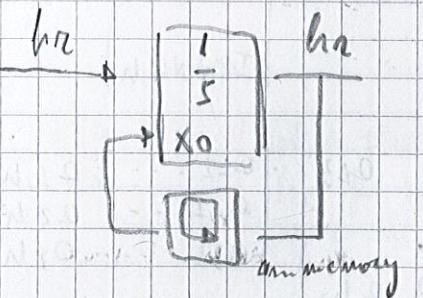


$$\underline{A}_c \underline{I}_r = \underline{M}_c - \underline{W} \times (\underline{A}_c \underline{I}_r)$$

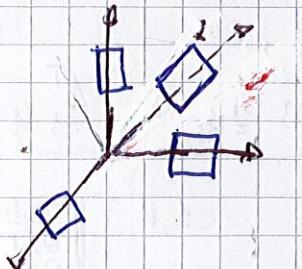
$$\underline{I}_r = -\underline{A}_{rw}^* \underline{M}_c - \underline{A}_{rw}^* \underline{W} \times (\underline{A}_c \underline{I}_r)$$

$$\underline{A}_{rw}^* = \underline{A}_{rw}^{-1} (\underline{A}_{rw}^* \underline{A}_{rw} + \underline{A}_{rw} \underline{A}_{rw}^*)^{-1} \cdot \underline{A}_{rw}$$

To avoid heavy computation  $\Rightarrow$  select interval



$\Rightarrow$  4 rotors common configurations are:



$$\Rightarrow \underline{A} = \begin{bmatrix} 1 & 0 & 0 & 1/\sqrt{3} \\ 0 & 1 & 0 & 1/\sqrt{3} \\ 0 & 0 & 1 & 1/\sqrt{3} \end{bmatrix}$$

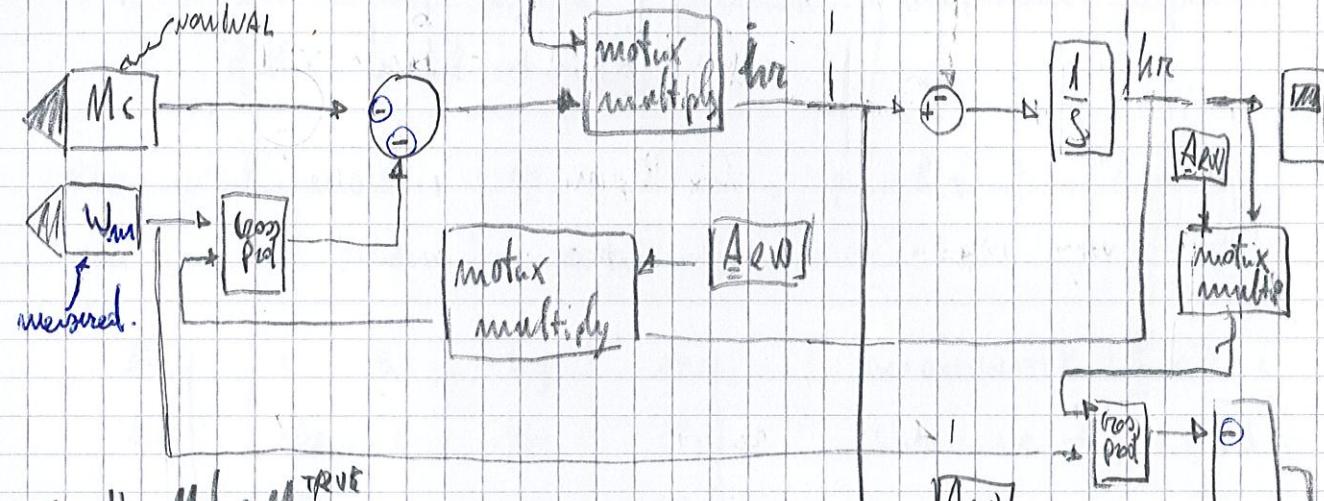
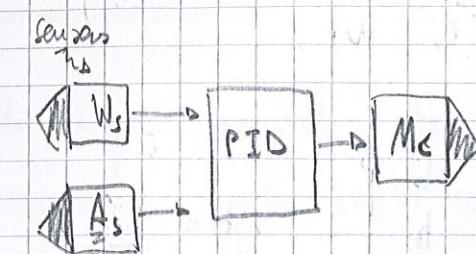
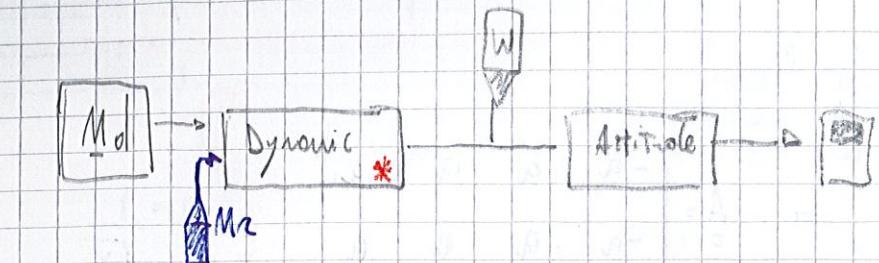
3 R.W. aligned with the 3 axes

4<sup>th</sup> R.W. equal component along the 3 axes

$$\sqrt{\left(\frac{1}{\sqrt{3}}\right)^2 + \left(\frac{1}{\sqrt{3}}\right)^2 + \left(\frac{1}{\sqrt{3}}\right)^2} = \left(\frac{3}{\sqrt{3}}\right)^{1/2} = 1$$

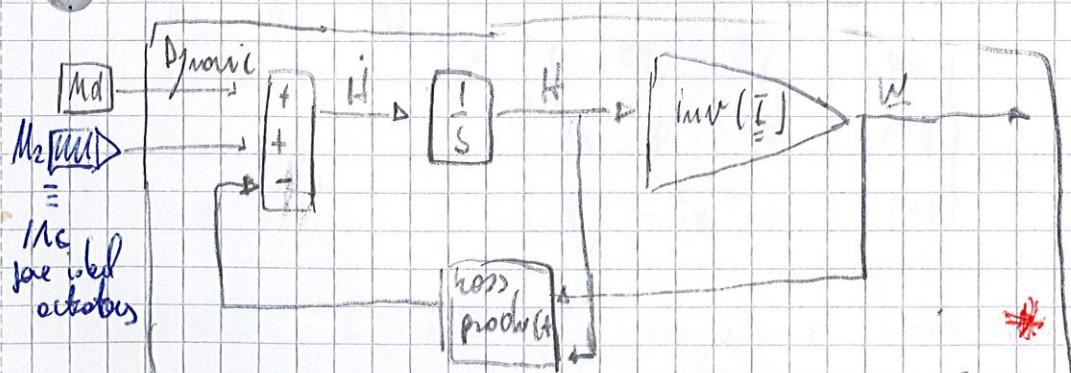
$$\Rightarrow \underline{A}^* = \begin{bmatrix} 5/6 & -1/6 & -1/6 \\ -1/6 & 3/6 & -1/6 \\ -1/6 & -1/6 & 3/6 \\ 1/2\sqrt{3} & 1/2\sqrt{3} & 1/2\sqrt{3} \end{bmatrix}$$

For more ideal actuators } introduction ENERGY LOSSES  
problems in various form



$$\underline{H} + \underline{W} \times \underline{H} = \underline{M}_d + \underline{M}_c^{TRUE}$$

$\underline{M}_c^{TRUE}$  same sign of  $\underline{M}_c$  AND  $\sim$  equal.



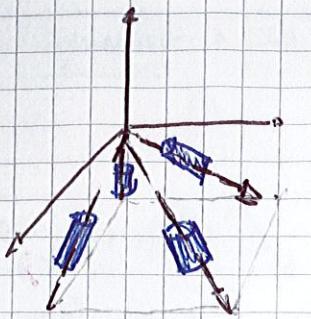
Obs: With this 4 rotors configuration a control to be applied along a single axis will generate an control action along each rotor

$$\underline{M}_c = \begin{bmatrix} 1/6 \\ 0 \\ 0 \end{bmatrix} \Rightarrow \underline{M}_c = \underline{A}^* (-\underline{M}_c - \underline{W} \times \underline{I}_r) = \begin{bmatrix} 5/6 \\ -1/6 \\ -1/6 \end{bmatrix} \Rightarrow \|\underline{M}_c\| = \sqrt{\frac{30}{36}}$$

supposed to be zero.

(ELECTRIC ENGINE)  
 $\propto$  non linearities due to saturation  $\Rightarrow$  losses

b) The other possible configuration is the quadrilateral pyramid



$$\Rightarrow A_z = \begin{bmatrix} -a & a & a & -a \\ -a & -a & a & a \\ a & a & a & a \end{bmatrix}; a = \frac{1}{\sqrt{3}}$$

$$\Rightarrow A_z^* = \begin{bmatrix} -b & -b & b \\ b & -b & b \\ b & b & b \\ -b & b & b \end{bmatrix}; b = \frac{\sqrt{3}}{4}$$

→ "Magnetic actuators"

→ Dipole moment induced in a coil.



$$\Rightarrow M = \mu \cdot N \cdot S \cdot i$$

$$\Rightarrow M = \underline{m} \times \underline{B} \quad \text{order: } 10^{-3} \div 10^{-6} \text{ NM}$$

[the torque generated by such device is perpendicular to the magnetic dipole ( $\underline{m}$ ) and to the extend magnetic field ( $\underline{B}$ )

⇒ It's never possible to generate 3 independent component of the control torque

↳ The control command is  $\underline{M}$  (i) ⇒ is possible to formulate the control problem in matrix form as:

$$\begin{pmatrix} M_{x1} \\ M_{y1} \\ M_{z1} \end{pmatrix} = \begin{pmatrix} 0 & B_2 & -B_y \\ -B_2 & 0 & B_x \\ B_y & -B_x & 0 \end{pmatrix} \begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix} \quad !! \text{UNFORTUNATELY } [\underline{B} \times] \text{ IS SINGULAR OR } [\underline{B} \times]^{-1} \text{!!}$$

$$\begin{pmatrix} M_x & M_y & M_z \\ B_x & B_y & B_z \end{pmatrix} = (M_y B_z - M_z B_y) \hat{i} + (M_z B_x - M_x B_z) \hat{j} + (M_x B_y - M_y B_x) \hat{k}$$

$$= \begin{pmatrix} 0 & B_2 & -B_y \\ B_x & 0 & -B_z \\ B_y & -B_x & 0 \end{pmatrix} \begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix}$$

SINCE,  $[\underline{B} \times]$  is singular (THIS MEANS THAT IT'S IMPOSSIBLE TO GENERATE 3 INDEPENDENT MOMENT ALONG THE 3 P.I. axis)

$$\begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix} = \begin{pmatrix} B_x & 0 & 0 \\ 0 & B_y & 0 \\ 0 & 0 & B_z \end{pmatrix} \begin{pmatrix} M_{x1} \\ M_{y1} \\ M_{z1} \end{pmatrix}$$

THEN A solution for coupled dynamic is required.

↳ consider a solution where the control action is required only along 2 axes.

e.g. control action required only along  $(y, z)$  components

$$\begin{pmatrix} 0 \\ M_y \\ M_z \end{pmatrix} = \begin{pmatrix} 0 & B_2 & -B_y \\ -B_2 & 0 & B_x \\ B_y & -B_x & 0 \end{pmatrix} \cdot \begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix}$$

$\hookrightarrow$  imposing  $M_x = 0$

(no current circulating inside the "x" coil)

Then the problem

is fully decoupled:

$$\begin{cases} M_y = B_x M_z \\ M_z = -B_x M_y \end{cases} \quad \begin{cases} M_z = \frac{M_y}{B_x} \\ M_y = -\frac{M_z}{B_x} \end{cases}$$

BUT

$\hookrightarrow$  A residual torque along "x" will be generated.

$$M_{x,R} = +B_2 M_y - B_y M_z$$

Residual not avoided  
BY CONTROL ACTION

$$\rightarrow M_{x,R} = -M_z \frac{B_2}{B_x} - M_y \frac{B_y}{B_x}$$

THEN

$\hookrightarrow$  Another actuator is required to provide control along null axis

$\hookrightarrow$  a torque must counter-balance this residual moment.

NOW

$\hookrightarrow$  supposing that the control is required along  $(x, y) \Rightarrow$  the "perp" torque will be generated along "z".

$$\begin{pmatrix} M_x \\ M_y \\ 0 \end{pmatrix} = \begin{pmatrix} 0 & B_2 & -B_y \\ -B_2 & 0 & B_x \\ B_y & -B_x & 0 \end{pmatrix} \cdot \begin{pmatrix} M_x \\ M_y \\ 0 \end{pmatrix} \rightarrow \begin{cases} M_x = M_y B_2 \rightarrow M_y = \frac{M_x}{B_2} \\ M_y = -B_2 M_x \rightarrow M_x = -\frac{M_y}{B_2} \\ M_{z,R} = -\frac{B_y}{B_2} M_y + \frac{B_x}{B_2} M_x \end{cases}$$

THEFORE

$\hookrightarrow$  Ensuring stability along  $z$  axis tends to a RW the control problem must be re-formulated including the torque provided by the RW

$$\Rightarrow \begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix} = \begin{pmatrix} 0 & B_2 & -B_y \\ -B_2 & 0 & B_x \\ B_y & -B_x & 0 \end{pmatrix} \cdot \begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix} + \begin{pmatrix} 0 \\ 0 \\ -h_2 \end{pmatrix}$$

$$= \begin{pmatrix} 0 & B_2 & 0 \\ -B_2 & 0 & 0 \\ B_y & -B_x & 1 \end{pmatrix} \cdot \begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix}$$

$\hookrightarrow$  this time such matrix is non-singular.

and the control problem can be solved directly

$$\rightarrow \begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix} = \frac{1}{B_2} \cdot \begin{pmatrix} 0 & -1 & 0 \\ 1 & 0 & 0 \\ B_x & B_y & B_2 \end{pmatrix} \cdot \begin{pmatrix} M_x \\ M_y \\ M_z \end{pmatrix}$$

obs: such control problem is solvable if and only if the "z" component of the magnetic field is different from zero (in PI reference frame)

Alternative procedure. ( $\neq$  control strategy)

$$\underline{M}_c = \underline{M} \times \underline{B} \rightarrow \underline{B} \times \underline{M}_c = \underline{B} \times (\underline{M} \times \underline{B}) \parallel \underline{B} (\underline{C} \cdot \underline{A}) - \underline{C} (\underline{A} \cdot \underline{B})$$

$$\text{therefore: } \underline{B} \times \underline{M}_c = \underline{M} \cdot (\underline{B} \cdot \underline{B}) - \underline{B} (\underline{M} \cdot \underline{B})$$

$$\hookrightarrow \text{GENERATING A DIPOLE MOMENT EXACTLY PERPENDICULAR TO } \underline{B} \parallel \underline{M} \perp \underline{B}$$

$$\underline{B} \times \underline{M}_c = \|\underline{B}\|^2 \cdot \underline{M} \rightarrow \underline{M} = \frac{\underline{B} \times \underline{M}_c}{\|\underline{B}\|^2} \quad (\text{100% orthogonal to } \underline{B})$$

$\hookrightarrow$  THEREFORE THE EFFECTIVE CONTROL TORQUE GENERATED WILL BE.

$$\underline{M}_{eff} = \underline{M} \times \underline{B} \rightarrow \underline{M}_{eff} = \frac{1}{B^2} (\underline{B} \times \underline{M}_c) \times \underline{B}$$

$$\text{!! } \underline{M}_{eff} = \underline{M}_c \Leftrightarrow \underline{M} \perp \underline{B} \text{ !!}$$

not is the desired control torque.

$$\text{in fact: } \begin{cases} M_{\text{eff}} = \underline{M} \times \underline{B} \\ \underline{M} = \frac{\underline{M}_c \times \underline{B}}{B^2} \end{cases} \Rightarrow M_{\text{eff}} = \frac{1}{B^2} (\underline{B} \times \underline{M}_c) \times \underline{B}$$

$$\Rightarrow M_{\text{eff}} = \frac{1}{3^2} [\underline{M}_c (\underline{B} \cdot \underline{B}) - \underline{B} \cdot (\underline{M}_c \cdot \underline{B})] \parallel M_c = \underline{M} \times \underline{B}$$

$$= \frac{1}{B^2} [\underline{M}_c (\underline{B} \cdot \underline{B}) - \underline{B} \cdot (\underline{M} \times \underline{B}) \cdot \underline{B}]$$

$$= \underline{M}_c$$

therefore  $\begin{cases} \underline{M}_c = \underline{M} \times \underline{B} \\ \underline{M} \perp \underline{B} \end{cases}$  will generate a  $\underline{M}_c$  torque orthogonal to  $\underline{B}$

!! this control method really works when the control torque required are (for their nature) about  $\perp$  to  $\underline{B}$ .

→ "Minimum time moreover!"

$$M_{\text{min}} J^* = \int_{t_0}^{t_f} dt + \lambda \cdot \int_{t_0}^{t_f} (\dot{x} + Ax - Bu)$$

$\downarrow$   
cost function  
(J)

constraint: dynamic of the system.  
(expressed in the 1<sup>st</sup> order)

↳ rotation along a single axis  $\Rightarrow M_i = I_i \ddot{\alpha}_i$  represent the linearized equation of dynamic for multi system. !! rotations around 1 axis is one of the small number of cases with analytical solution!

$$\begin{aligned} u = \ddot{\alpha} &= \frac{M}{I} \quad \int dt \\ \ddot{\alpha} &= \dot{\alpha}_0 + Mt \quad \int dt \\ M &= \frac{M}{I} \quad \ddot{\alpha} = \dot{\alpha}_0 + \dot{\alpha}_0 t + \frac{1}{2} Mt^2 \\ M &= \frac{M}{I} \end{aligned}$$

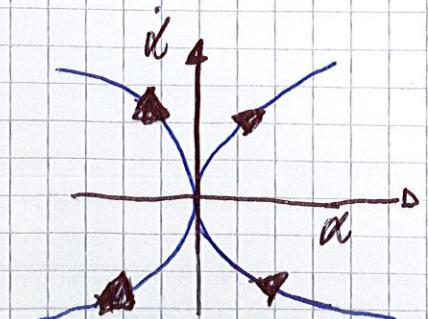
therefore 2 equations are available:

$$\begin{cases} \ddot{\alpha} = \dot{\alpha}_0 + Mt \\ \alpha = \dot{\alpha}_0 t + \frac{1}{2} Mt^2 \end{cases} \quad \begin{array}{l} \xrightarrow{\text{compute time from 1<sup>st</sup> equation}} t = \frac{\ddot{\alpha} - \dot{\alpha}_0}{M} \\ \xrightarrow{\text{substitute}} \end{array}$$

$$\Rightarrow \begin{cases} \alpha = \dot{\alpha}_0 + \dot{\alpha}_0 \cdot \frac{(\ddot{\alpha} - \dot{\alpha}_0)}{M} + \frac{1}{2} \cdot \left( \frac{(\ddot{\alpha} - \dot{\alpha}_0)}{M} \right)^2 \\ M = \frac{M}{I} \quad (M: control torque along the station axis) \end{cases}$$

IF  $M = \text{const.} \rightarrow M = M^*$

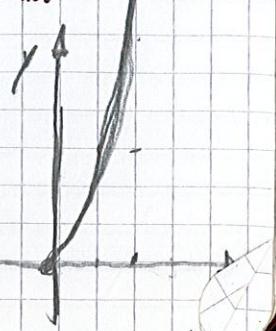
THEN the phase-space representation of such curve is a parabola whose concavity depends on the torque limit ( $M^*$ )



$$(\ddot{\alpha} - \dot{\alpha}_0) = \frac{1}{2M} (\dot{\alpha} - \dot{\alpha}_0)^2 + \frac{1}{M} \dot{\alpha}_0 (\ddot{\alpha} - \dot{\alpha}_0)$$

parabola is a "cano" diagram  
 $y = ax^2 + bx$

$$[a=1, b=2] \quad \begin{array}{|c|c|c|c|} \hline y & 8 & 4 & 0 \\ \hline 0 & 0 & 0 & 0 \\ \hline 8 & 2 & 3 & 1 \\ \hline \end{array}$$



I<sup>st</sup> design concept:

- $\ddot{\alpha}_i = 0 \quad \& \quad \dot{\alpha}_i = 0$
- bounded maximum torque
- $\ddot{\alpha}^*$  is maximum  
 $t$  is minimum.

therefore having chosen  $\ddot{\alpha}_i = 0$  the parabola reduces to:

$$\alpha = \alpha_0 + \frac{1}{2u} (\dot{\alpha} - \dot{\alpha}_0)^2 + \overset{\dot{\alpha}=0}{\underset{\text{not still constraint equal to } \frac{M^*}{I}}{\alpha_0}} (t - t_0)$$

therefore the dependence of the angle from the angular velocity will be:

$$\alpha = \alpha_0 + \frac{1}{2u} \dot{\alpha}^2$$

e.g. X-axis:  $\alpha_x = [\dot{\alpha}_e]_{2,3} \rightarrow \dot{\alpha}_x = [\dot{\alpha}_e]_{2,3} \rightarrow 1$

POSITION SET-POINT  $\rightarrow [\dot{\alpha}_e]_{2,3}(+) = [\dot{\alpha}_e]_{2,3}\Big|_{t=0} + \frac{1}{2u} \left(\frac{M^*}{I_x}\right) t^2$

VELOCITY SET-POINT  $\rightarrow w_x \approx [\dot{\alpha}_{e,3}] = [\dot{\alpha}_{2,3}] \Big|_{t=0} + M \cdot t = \frac{M^*}{I_x} \cdot t$

$$\Rightarrow [\dot{\alpha}_e]_{2,3}(+) = \frac{1}{2} \left(\frac{M^*}{I_x}\right) t^2$$

$$w_{x,sp} = \frac{M^*}{I_x} t$$

the problem is at this point to determine when (for which angle) the torque ( $M = \text{Max-torque}$ ) must be changed in sign.

and according with the control logic.

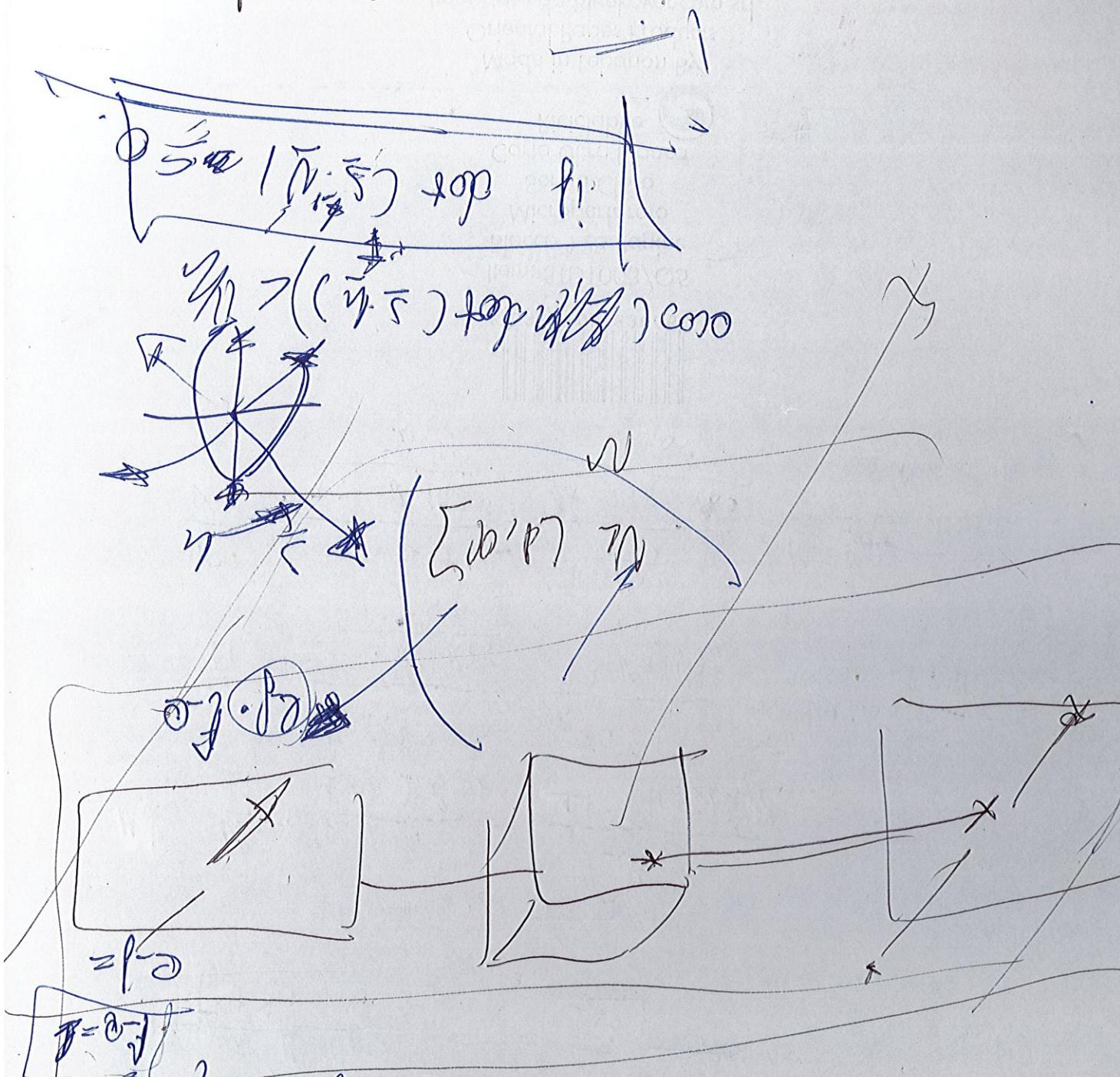
$$M = -M_{\text{max}} \operatorname{sign}(\alpha)$$

and in order to stabilize the system:

$$M = -M_{\text{max}} \operatorname{sign}(\alpha + K \dot{\alpha})$$

$\theta_{\text{eo}} \rightarrow \text{R.A.}$

$\phi_{\text{eo}} \rightarrow \text{CO-DEC.}$



$\phi_{\text{eo}} = (B \cdot j) \text{ top } \phi_{\text{eo}} = (B \cdot j) \cos(\omega t)$

Shuttle coil method of finding angle of rotation - (q3)

Plot of angle of rotation vs time  $\rightarrow$  elliptical plot.

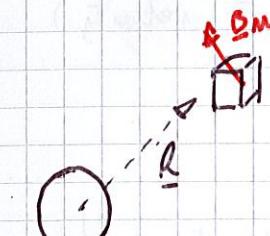
[001] [001]

Winding before pole

### Sensor 1) magnetic field sensor.

This sensor provide the measurement of one vector. ( $\underline{\underline{B}}_{\text{m}}$ )

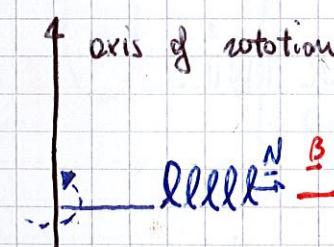
$\Rightarrow$  By coupling such measurement with the magnetic field's mathematical model is possible to get the attitude.



$$\Rightarrow \underline{\underline{B}}(R) = \underline{\underline{A}} \underline{\underline{B}}_m$$

$$-\frac{d\Phi(B)}{dt} = \text{FEM} \Rightarrow i = \frac{\text{F.E.M.}}{R} = -\frac{1}{R} \frac{d\Phi_s(B)}{dt}$$

e.g.



$$\Phi(B) = \int_{\text{coil}} \underline{\underline{B}} \cdot d\underline{\underline{A}} = B \cdot N \cdot A_{\text{coil}} \cdot \cos(\omega t)$$

$$A = A_{\text{coil}} \quad (B \cdot N = \underline{\underline{B}} \cdot \underline{\underline{A}})$$

Therefore for a constant magnetic field the recorded current will be a sinusoidal form:

$$i(t) = -\frac{1}{R} \frac{d\Phi(B)}{dt} = W \frac{B \cdot N \cdot A_{\text{coil}}}{R} \sin(\omega t) \rightarrow \text{measured.}$$

$$\frac{di(t)}{dt} = W \frac{B \cdot N \cdot A}{R} \cdot \cos(\omega t) \Rightarrow B \cos(\omega t) \propto \frac{di(t)}{dt}$$

$(B \cdot \cos(\omega t)) \rightarrow$  represent the projection of the magnetic field on the coil axis.

!! this means that the derivative of the signal is in fact proportional to the projection of the magnetic field on the coil's axis. !!

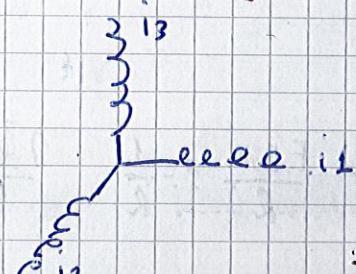
this means that:

if The spacecraft is rotating (at constant velocity)

AND if Three coils are used

THEN it is possible to determine the magnetic field

by measuring 3 currents



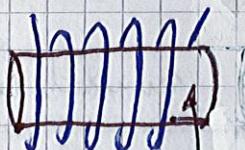
$$\Rightarrow i_1 \propto B_x \cos(\omega_x t) \cdot w_x$$

$$i_2 \propto B_y \cos(\omega_y t) \cdot w_y$$

$$i_3 \propto B_z \cos(\omega_z t) \cdot w_z$$

$$I = i_1 + i_2 + i_3 \propto \| \underline{w} \|^2$$

↳ Adding a ferromagnetic material inside a coil.



FERRO-MAGNETIC MATERIAL.

this is done in order

to increase the magnetic field.

In general:

• INDUCTOR: Device capable of immagininating electric energy into a magnetic field

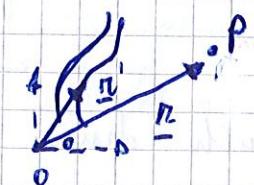
• CIRCUITS:  $i$   $\rightarrow$   $\frac{dN}{dt} = L \cdot \frac{di}{dt}$   $\Rightarrow$   $\Delta N(t) = L \cdot \frac{di(t)}{dt}$

In order to compute  $L$ :

→ Ampere's law / Laplace-Joumula.

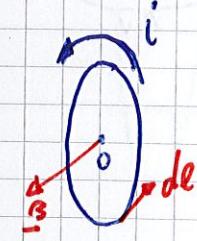
$$\text{Ampere's law: } \oint_S \underline{B} \cdot d\underline{n} = \mu_0 \cdot \frac{1}{i} \text{ [we]}$$

$$\text{Laplace's Joumula: } d\underline{B}(\underline{r}) = \frac{\mu_0}{4\pi} i \cdot \frac{d\underline{l} \times (\underline{r} - \underline{r}')}{\| \underline{r} - \underline{r}' \|^3}$$



$$\Rightarrow \underline{B}(\underline{r}) = \frac{\mu_0 i}{4\pi} \int_P \frac{d\underline{l} \times (\underline{r} - \underline{r}')}{\| \underline{r} - \underline{r}' \|^3}$$

\* CIRCULAR SINGLE COIL → Magnetic field in the origin.

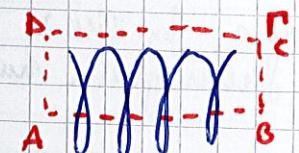


$$\underline{B}(0) = \frac{\mu_0 i}{4\pi} \int_P - \frac{d\underline{l} \times \underline{r}'}{\| \underline{r}' \|^3}$$

$$\| d\underline{l} \perp \underline{r}' \}; \quad d\underline{l} = R \sin(\theta) d\theta \approx R d\theta.$$

$$\Rightarrow \underline{B}(0) = \frac{\mu_0 i}{4\pi} \cdot \frac{1}{R^3} R^2 \int_0^{2\pi} d\theta = \frac{\mu_0 i}{2R}$$

\*  $N$  COILS →  $N$  singular circular coils constituting a winding.



Hp\*: Solenoid →  $\underline{B} \neq 0 \Rightarrow \underline{B}$  external to the winding is null.

$$\Rightarrow \oint_S \underline{B} \cdot d\underline{s} = \int_A^B \underline{B} \cdot d\underline{r} + \int_B^C \underline{B} \cdot d\underline{r} + \int_C^D \underline{B} \cdot d\underline{r} + \int_D^A \underline{B} \cdot d\underline{r}$$

$$\Rightarrow \underline{B} \cdot \underline{L} = N \cdot i \cdot \mu_0 \quad \star$$

conservation of current.

$$\begin{cases} B = N \cdot i / L \\ N = N/L \end{cases}$$

\* N coils winding a ferro magnetic material  $\rightarrow \mu = \mu_r \cdot \mu_0$

$$B = \mu \cdot H \cdot i$$

\* CONSTITUTIVE LAW FOR INDUCTORS  $\rightarrow$  Circuit element.

$\rightarrow$  Faraday's law

$$\frac{d\Phi(B)}{dt} = -\text{F.E.M.}$$



Therefore having such device acting on a  $i(t)$

$$\begin{cases} B = \mu \cdot H \cdot i(t) \\ \Phi(B) = (NA) \cdot B \end{cases}$$

$$\Phi_{\text{sd}}(B) = \sum_{i=1}^N A_i \cdot \Phi_{A_i}(B) = N \cdot A \cdot B.$$

$$\Rightarrow \text{F.E.M.} = \Delta V(t) = \mu \cdot H \cdot N \cdot A \cdot \frac{di(t)}{dt}$$

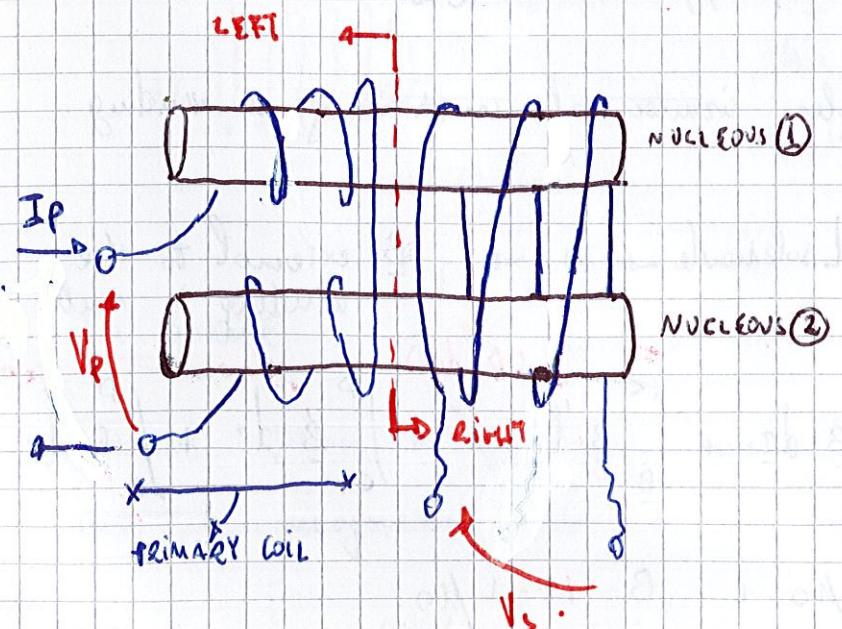
$$(H = \frac{N}{l})$$



$$\Delta V(t) = \frac{\mu N^2}{l} A \cdot \frac{di(t)}{dt}$$

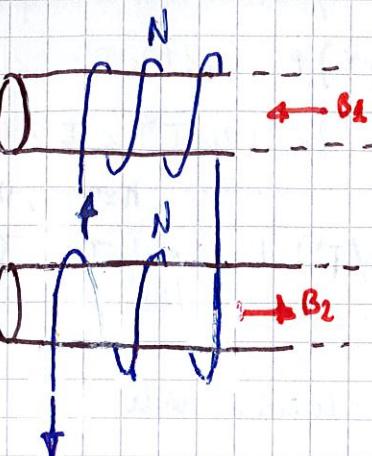
" Moving  $\alpha$  constant  
current  $\Rightarrow$   $\phi$  potential  
losses will be detected "

To get an attitude determination:



$\rightarrow$  sectional view.  
(Same for the 2 ferromagnetic nucleus)

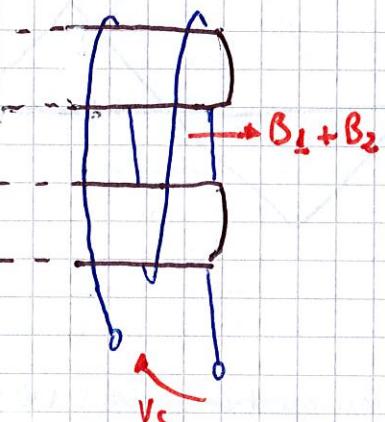
### LEFT SIDE



$$B_2 = -B_1 = \mu \cdot H \cdot i(t)$$

This magnetic field depending by time is capable to induce a F.E.M. in the right side of the sensor.

### RIGHT SIDE



Thanks to Faraday's law:

$$V_s = - \frac{d\Phi(B_1 + B_2)}{dt} = 2 \mu \frac{N^2}{l} A_{\text{sd}} \frac{di}{dt} \stackrel{!}{=} L \frac{di}{dt}$$

$\rightarrow$  length of this segment (right)  
Winding around the 2 nucleus.

A  $\rightarrow$  area of the winding on the right side

If a EXTERNAL MAGNETIC FIELD is taken into account.

$$\text{THEN } B_1 = -B_2 \rightarrow \Phi(B) = -\Phi(B_2) \rightarrow V_s = \frac{d}{dt} \Phi(B_1 + B_2) = 0$$

WHILE considering the WORLD MAGNETIC FIELD

the true magnetic flow across the right winding will be:

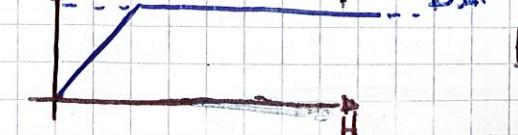
$$V_s = \frac{d}{dt} \Phi(B_1 + B_2) + \frac{d}{dt} \Phi_{\text{B.W.F.}}$$

$\rightarrow$  NEGLIGIBLE.

$\rightarrow$  Magnetic induction.

!! USUALLY IT'S NOT LINEAR  $\Rightarrow$  curve with AN HYSTERESIS !!

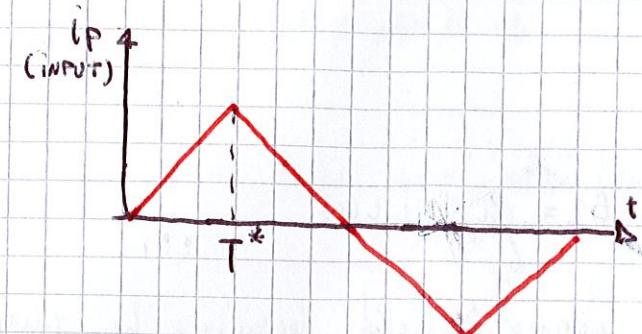
$\rightarrow$  There's a limit in the energy stored into the ferromagnetic nucleus.



$$H = \frac{B}{\mu_0} - M$$

$\rightarrow$  Long magnetic softening due to the molecule

↳ Setting  $i_p$  as a triangular signal:

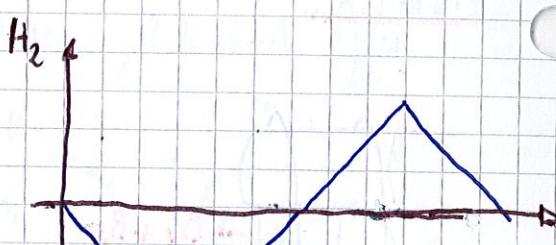
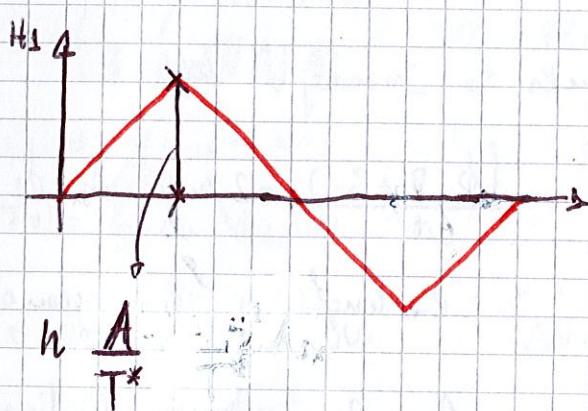


$$A \rightarrow \text{amplitude of the current.}$$

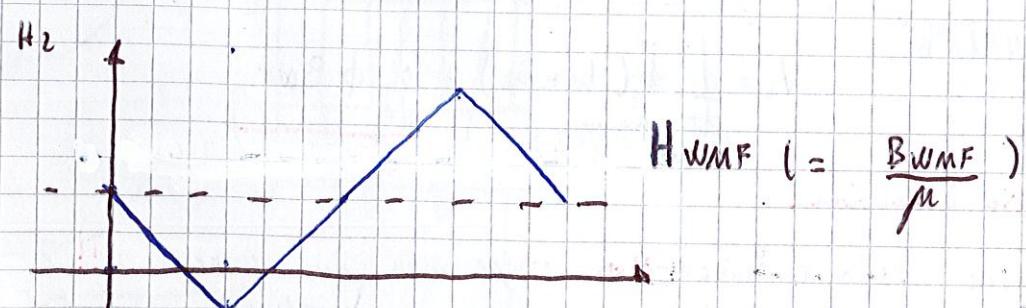
$$i_p = \begin{cases} (A/T^*)t & \text{for } 2nT^* < t < (2n+1)T^* \\ 0, \dots, N \end{cases}$$

$$-(A/T^*)t ; \quad T^* = T_{IP}/4$$

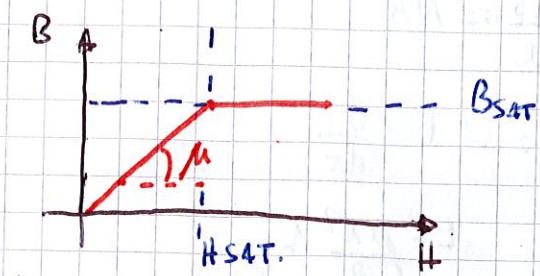
↳ The current circulating in the 2 nuclei will be opposite in "direction"  $\Rightarrow$  able to induce to induce opposite magnetic induction.



↳ Adding the effect of a constant magnetic field (W.M.F.)



↳ Considering the non-linearity of the ferromagnetic material: saturation

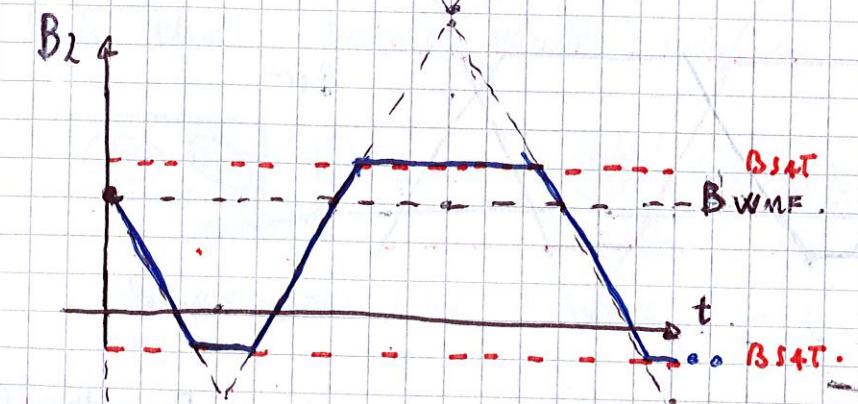
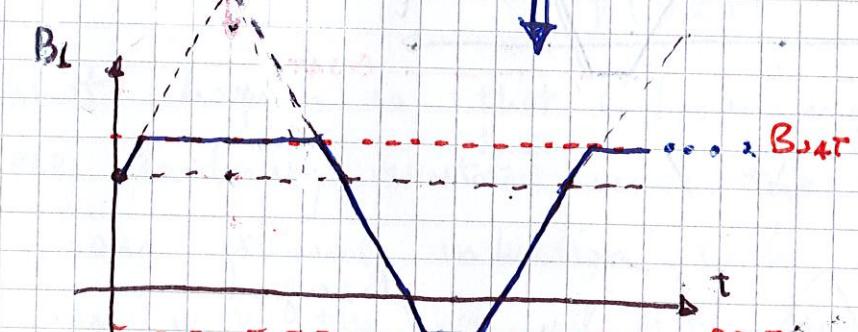
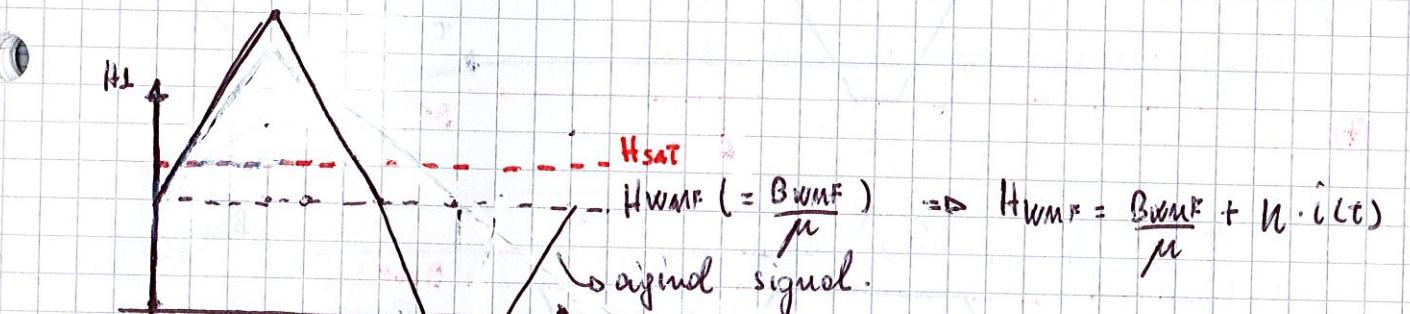


$$\begin{cases} B = \mu \cdot h \cdot i(t) (\rightarrow \text{saturation}) \\ B = \mu H \end{cases} \Rightarrow H = h \cdot i(t)$$

!! There's a limit in the stable energy inside the ferromagnetic nuclei !!

This means that if:  $|B_{1/2} + B_{ext}| > B_{SAT} \Rightarrow B_{1/2} = B_{SAT}$

$|B_{1/2} + B_{ext}| < B_{SAT} \Rightarrow B_{1/2} = B_{ext}$



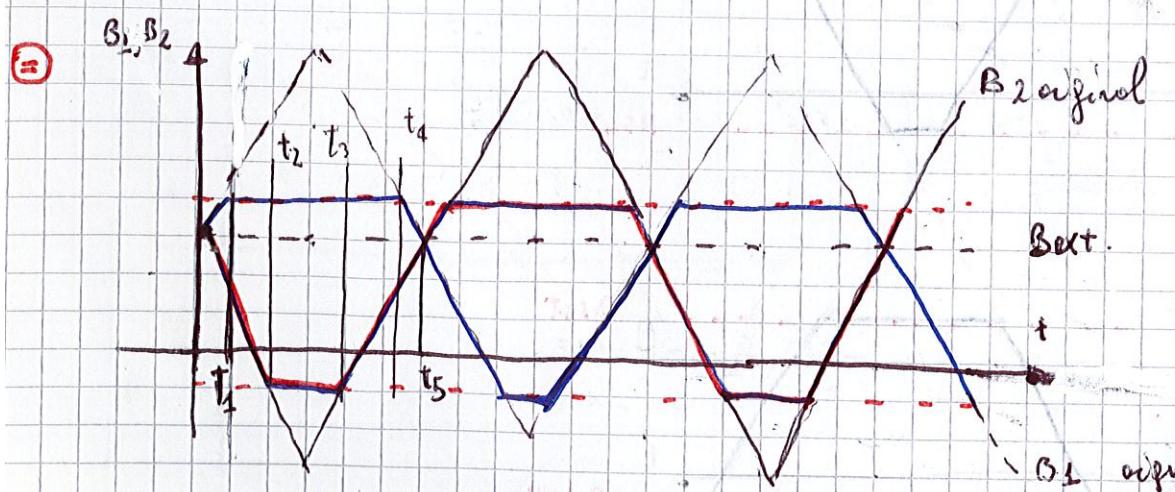
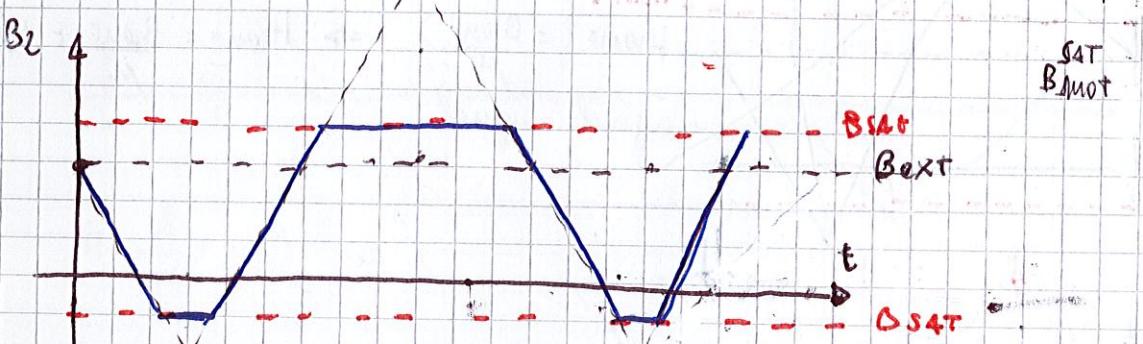
↳ To evaluate the Vs the derivative of this signal must be taken into account.

according with Faraday's law:

$$V_s = - \frac{d\Phi(B_1 + B_2)}{dt} = \mu \cdot h \cdot N \cdot A \cdot \frac{d\Phi}{dt}$$

$$= \mu \cdot (N \cdot A) \frac{dH}{dt} (= L \cdot \frac{di}{dt})$$

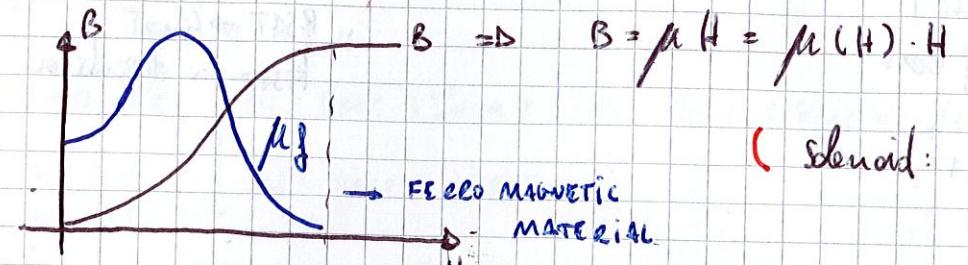
$$L = \frac{\mu N^2 A}{2}$$



- a)  $B_1 \vee B_2$  SATURATED (one saturated  $\oplus$  the other  $\ominus$ )  $\Rightarrow B_{TOT} = B_{ext} \Rightarrow V_s = 0$
- b)  $B_1 \vee B_2$  NOT SATURATED  $\Rightarrow B_{TOT} = B_{ext} + \text{slope} \cdot T + B_{ext} - \text{slope} \cdot T$   
 $= 2B_{ext} \Rightarrow V_s = 0$

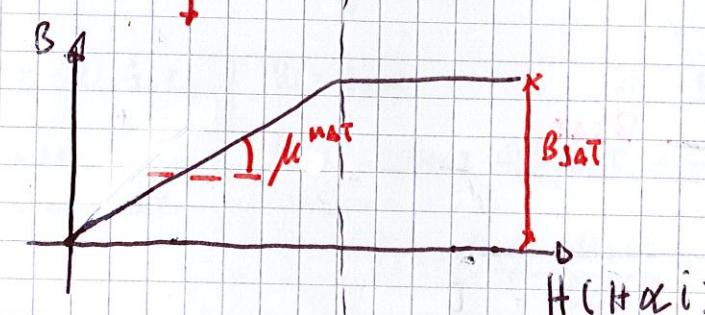
c)  $B_1 \vee B_2$  NOT SATURATED  $\left\{ \begin{array}{l} B_1 = \\ B_2 = \end{array} \right.$

Details about saturation



$$( \text{Solenoid: } H = n \cdot i )$$

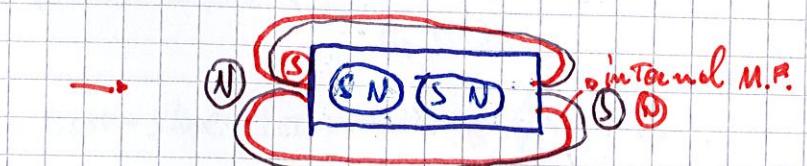
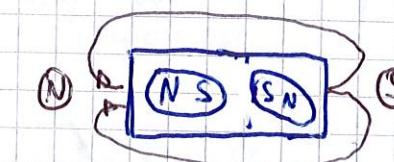
Magnetic permeability is not constant but depends NOT LINEARLY by  $H$ .



$$\text{e.g. Fe-C: } \mu_{Fe-C} = 3,75 \times 10^{-4}$$

$$B_{sat} \approx 2T$$

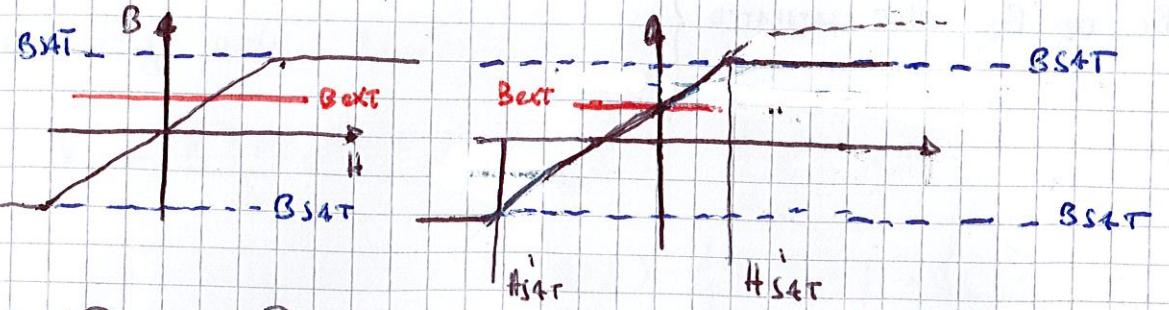
What happens is that dipoles (magnetic dip) originally are randomly oriented inside the material. cause of an external induction such dipoles tend to align with the external M.F. creating a "cored-M.F." such that further magnetic induction inside the material is ENABLED



IF An external (NOT inductive) magnetic field is present.

THEN SATURATION WILL OCCUR AT LOWER VALUES OF  $H$  IN THE DIRECTION OF THE EXTERNAL M.F. AND AT HIGHER VALUES OF  $H$  IN THE OPPOSITE DIRECTION

$\Rightarrow$  THIS CORRESPONDS TO A TRANSLATION OF THE PREVIOUS GRAPH.

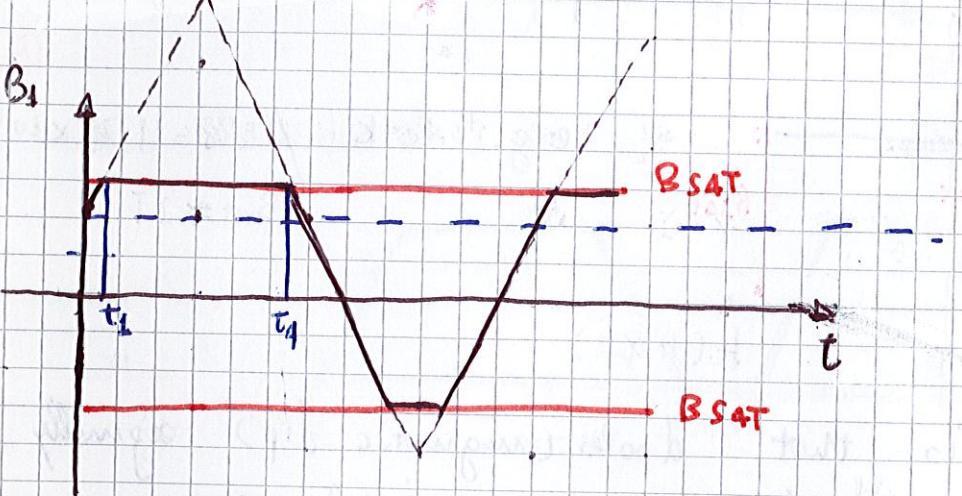


$$|H_{SAT}| > |H_{SAT}|$$

opposite direction of  $B_{ext}$

$$H_{SAT}^+ < H_{SAT}^-$$

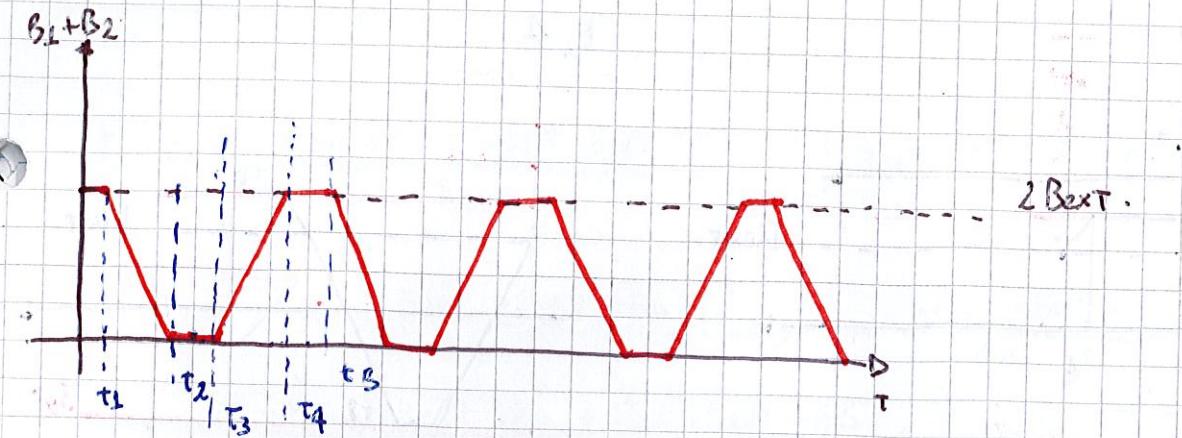
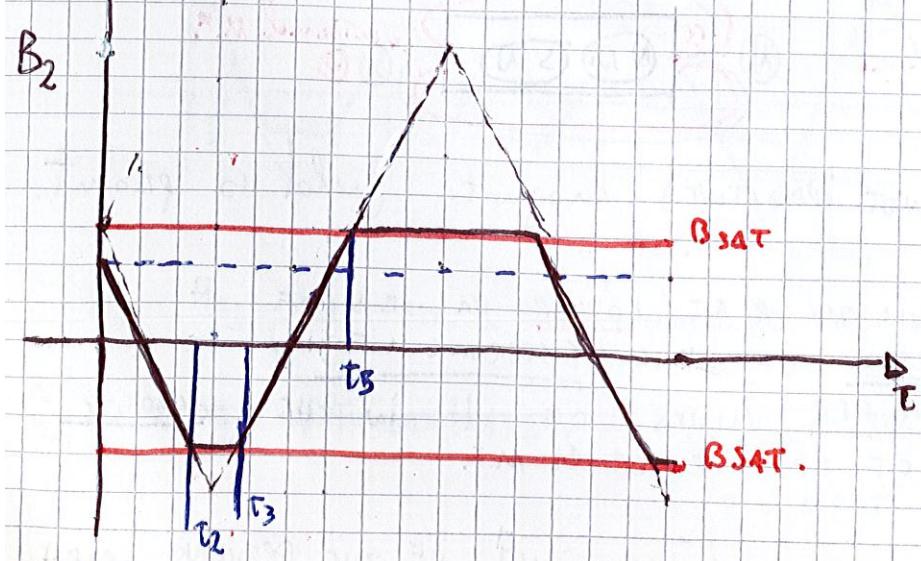
$B_{SAT} \rightarrow \text{const}$   
 $H_{SAT} \rightarrow \text{depends on } B_{ext}$



!! In order to have an interception with  $B_{SAT}$  must be that

$$I_p \cdot \mu \cdot n > (B_{SAT} + B_{ext})$$

$$(i = I_p \text{ triangle}(t)) \Rightarrow \mu H^{\max} > (B_{SAT} + B_{ext})_{\max} !!$$



$$\begin{aligned} * t = 0 \div t_1 & \left\{ \begin{array}{l} B_1 = B_{ext} + \text{slope} \cdot t \\ B_2 = B_{ext} - \text{slope} \cdot t \end{array} \right. \Rightarrow B_{tot} = 2 B_{ext} \Rightarrow V_s = -\frac{d B(B)}{dt} = 0 \\ 1 \rightarrow \text{NOT SAT.} & \\ 2 \rightarrow \text{NOT SAT.} & \end{aligned}$$

$$t_1: \frac{B_{sat} - B_{ext}}{T^*} = \text{slope} = \frac{B_D}{T^*} \rightarrow t_1 = T^* \frac{(B_{sat} - B_{ext})}{B_D}$$

$$(T^* = \frac{T_{ip}}{4})$$

$$\begin{aligned} * t = t_1 \div t_2 & \left\{ \begin{array}{l} B_1 = B_{sat} \\ B_2 = B_{ext} - \text{slope} \cdot t_2 = -B_{sat} \end{array} \right. \Rightarrow B_{tot} = B_{sat} + B_2(t_2) - \text{slope} \cdot t \\ 1 \rightarrow \text{SAT.} & \\ 2 \rightarrow \text{NOT SAT.} & \end{aligned}$$

$$t_2 = \frac{B_{ext} + B_{sat}}{\text{slope}} = \frac{B_{ext} + B_{sat}}{B_D} \cdot T^*$$

$$V_s = -\frac{d B}{dt} = \text{slope}$$

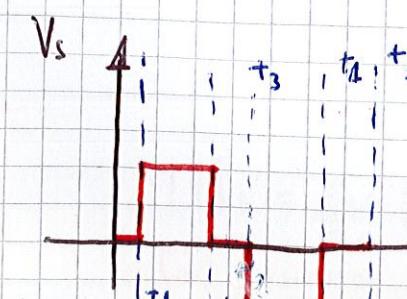
$$V_s = \frac{B_D}{T^*}$$

$$\begin{aligned} * t = t_2 \div t_3 & \left\{ \begin{array}{l} B_1 = B_{sat} \\ B_2 = -B_{sat} \end{array} \right. \Rightarrow B_{tot} = 0 \Rightarrow V_s = 0. \\ 1 \rightarrow \text{SAT.} & \\ 2 \rightarrow \text{SAT.} & \end{aligned}$$

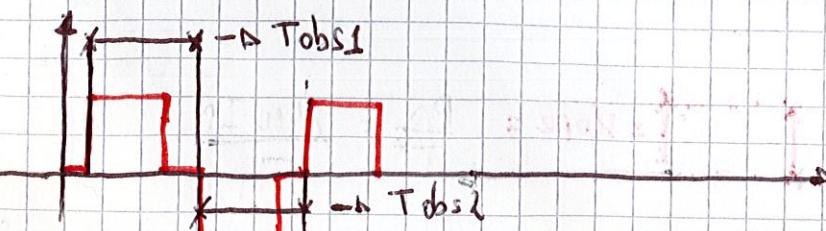
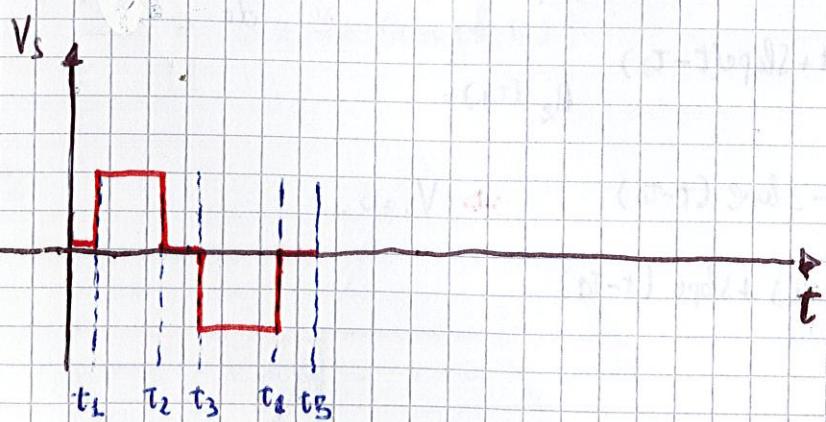
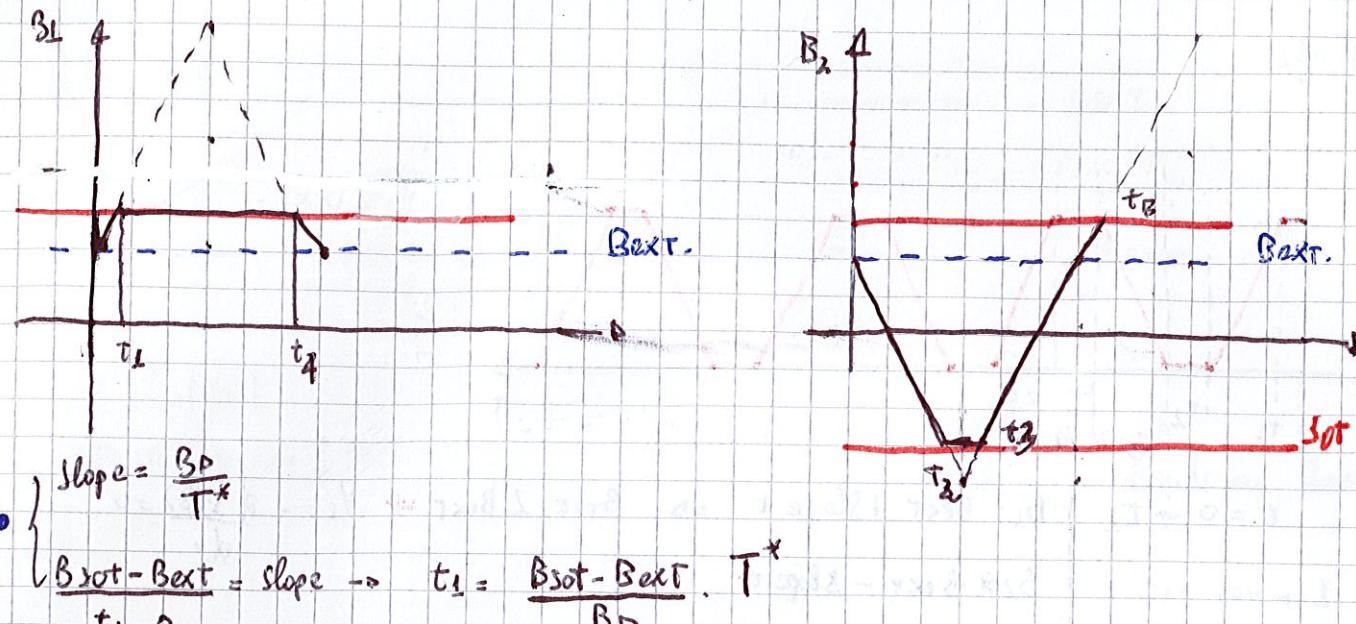
$$\begin{aligned} * t = t_3 \div t_4 & \left\{ \begin{array}{l} B_1 = B_{sat} \\ B_2 = -B_{sat} + \text{slope} \cdot (t - t_3) \end{array} \right. \Rightarrow B_{tot} = \text{slope} \cdot t \Rightarrow V_s = -\frac{d B}{dt} = -\text{slope}. \\ 1 \rightarrow \text{SAT.} & \\ 2 \rightarrow \text{NOT SAT.} & \end{aligned}$$

$$B_2(t_4) =$$

$$\begin{aligned} * t = t_4 \div t_5 & \left\{ \begin{array}{l} B_1 = B_{sat} - \text{slope} \cdot (t - t_4) \\ B_2 = B_2(t_4) + \text{slope} \cdot (t - t_4) \end{array} \right. \Rightarrow V_s = 0. \\ 1 \rightarrow \text{NOT SAT.} & \\ 2 \rightarrow \text{NOT SAT.} & \end{aligned}$$



$$\text{slope} = \frac{B_D}{T^*} = \frac{\mu n I_p}{T^* \cdot T_{ip}/4}$$



$$t_2 - t_1 = \frac{B_{SOT} + B_{ext} - B_{SOT} + B_{ext}}{B_D} \cdot T^* = \frac{2 B_{ext}}{B_D} T^* \quad (T^* = T_{IP}/4)$$

$$\begin{aligned} t_4 - t_3 &= T^* + \frac{-B_{SOT} + B_{ext} + B_D}{B_D} T^* - T^* - \frac{-B_{SOT} - B_{ext} + B_D}{B_D} T^* \\ &= \left[ \frac{-B_{SOT} + B_{ext} + B_D + B_{SOT} + B_{ext} - B_D}{B_D} \right] = \left[ \frac{2 B_{ext}}{B_D} \right] T^* \end{aligned} \quad \text{c.v.d.} \quad t_2 - t_1 = t_4 - t_3$$

$$\begin{aligned} t_3 - t_2 &= T^* + \frac{-B_{SOT} - B_{ext} + B_D}{B_D} T^* - \frac{B_{SOT} + B_{ext}}{B_D} T^* \\ &= T^* + \frac{-2 B_{SAT} - 2 B_{ext}}{B_D} T^* + \frac{B_D}{B_D} T^* = 2 T^* \left( 1 - \frac{B_{SAT} + B_{ext}}{B_D} \right) \end{aligned}$$

$$\begin{aligned} t_5 - t_4 &= T^* + \frac{-B_{ext} + B_D + B_{SOT}}{B_D} T^* - T^* - \frac{-B_{SOT} + B_{ext} + B_D}{B_D} T^* \\ &= \left[ \frac{-B_{ext} + B_D + B_{SOT} + B_{SOT} - B_{ext} - B_D}{B_D} \right] T^* = \frac{2(B_{SOT} - B_{ext})}{B_D} T^* \end{aligned}$$

$$\begin{aligned} T_{ob1} &= (t_2 - t_1) + (t_3 - t_2) = 2 T^* \left[ 1 - \frac{B_{SAT} + B_{ext}}{B_D} + \frac{-B_{ext}}{B_D} \right] \\ &= 2 T^* \left[ 1 - \frac{2 B_{ext}}{B_D} \right] \end{aligned}$$

$$\begin{aligned} T_{ob2} &= (t_4 - t_3) + (t_5 - t_4) = \frac{2 B_{ext}}{B_D} T^* + \frac{2 B_{SOT}}{B_D} T^* - \frac{2 B_{ext}}{B_D} T^* \\ &= 2 T^* \left[ \frac{B_{SOT}}{B_D} \right] \end{aligned}$$

↳ Defining  $K \triangleq \frac{B_{SOT}}{B_D}$

$$* T_{ob1} = 2 T^* (1 - K)$$

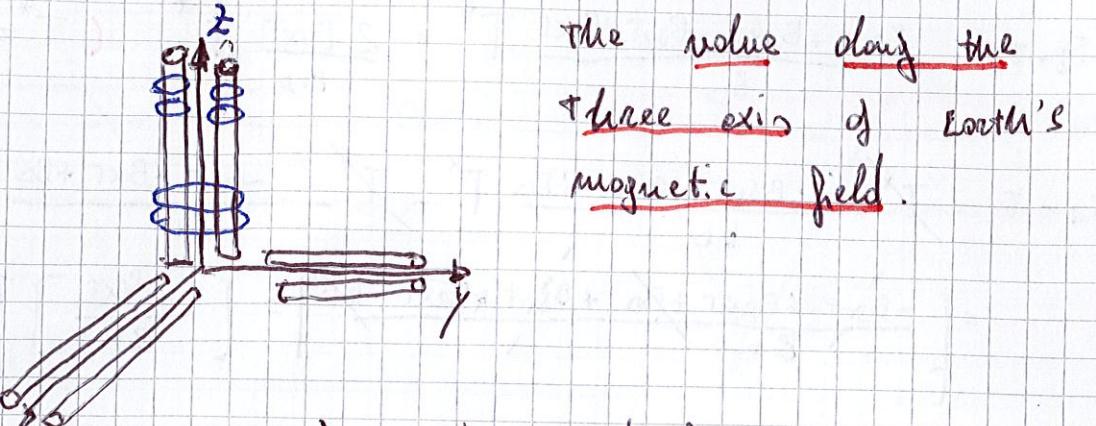
$$* T_{ob2} = 2 T^* K$$

F

$$\Delta T_{obs} = \frac{B_{ext}(2 T^*)}{B_D} \rightarrow B_{ext} = \frac{B_D \Delta T_{obs}}{2 T^*} \quad \text{length of the signal for non } \emptyset \text{ values}$$

$$B_D = \frac{\mu N^2 A}{l} I_P \quad T^* = T_{IP}/4$$

Using a tuple setup:  $\Rightarrow$  it's possible to know



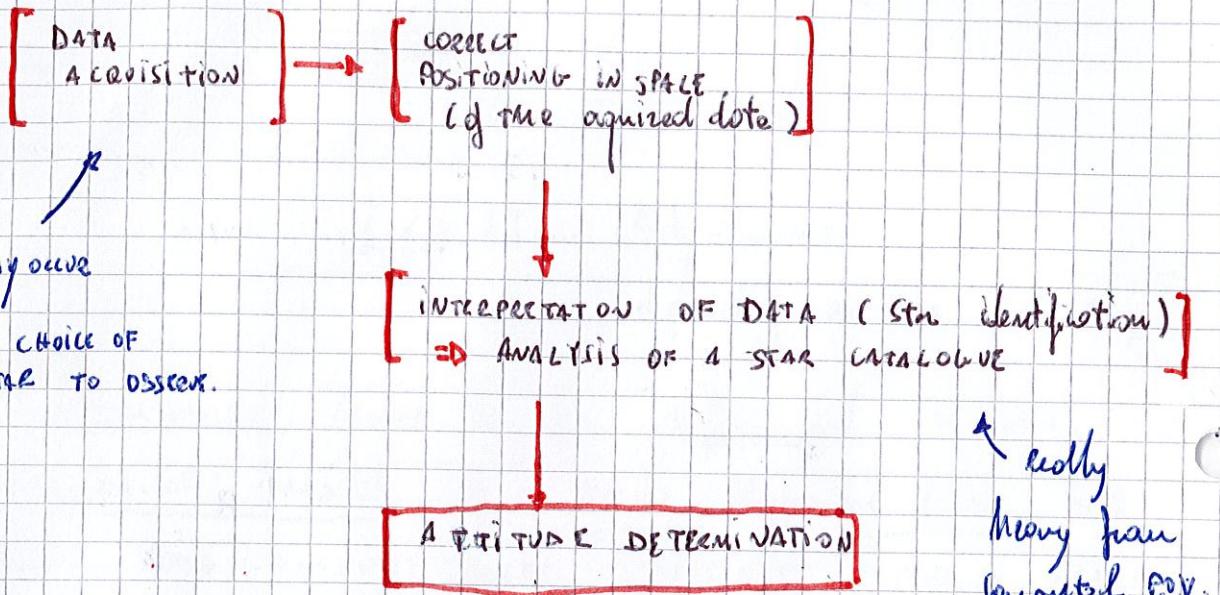
the value along the  
three axis of Earth's  
magnetic field.

$\{x, y, z\} \rightarrow$  Body-frame  
(depending on the mounting)

!! this measurement have to be referenced to the  
Mothermotional model of World's Magnetic field but  
KEEPING INTO ACCOUNT THE DIFFERENT REFERENCE FRAMES !!

## Sensor 2) Star Sensors (Star tracker)

The sensor follows the procedure:



Eclipse may occur

• correct choice of the star to observe.



→ " Matrix sensor (CCD)

→ Image is projected in a plane.



→ " In the FOV. (field of view) of the sensor even if it's really narrow, the stars captured might be really numerous

FOV.

Star tracker

CCD Sensor. (STAR MAPPER SENSORS)

\* FIXED ON THE SPACECRAFT.

RECONSTRUCT THE MAP OF THE OBSERVED SKY.

\* TILTING CAPABILITY

=> detects always the same

stars(\*) in such a way

that  $s^*$  is always in the middle of the FOV.

Assumption:  
[REASONABLE]

Star sensor catalog is given in the EARTH-CENTERED - INERTIAL - REFERENCE FRAME.

But:

SINE stars are theoretically at  $\infty$  distance from Earth.

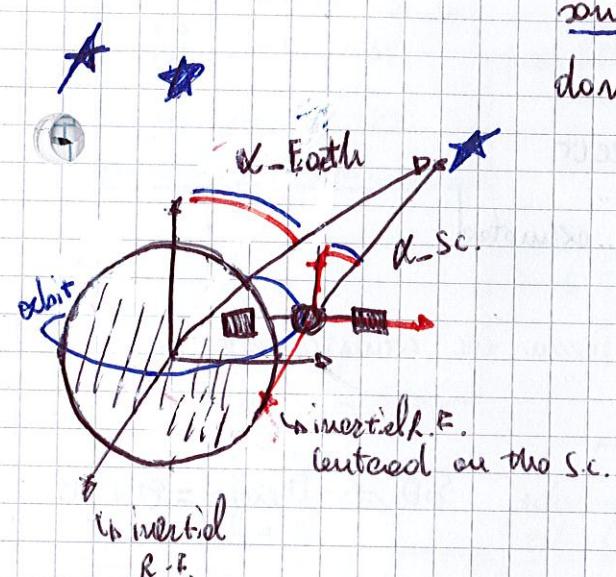
THEN each observation (data acquisition) mode from the spacecraft will give the same results (angles) as if it was done from Earth

!! Where the star catalogue has been build !!

=>  $X\text{-Earth} \approx X\text{-Spacecraft}$ .

e.g. catalogue:

NAM	SAO	RA	DEC	M
ALDEBARAN	1234	10°	20°	6



↳  $\{s_1, \dots, s_N\}$  → stars in the catalogue.

$\{\hat{o}_1, \dots, \hat{o}_N\}$  → Observation available.

$N \ll N$  → depending on the FOV of the sensor.

≈ obvious: At this stage  $A_{in 2pi}$  is not available.  
( $A$  = normally used)

AIM: is to associate each observation (in the sensor reference frame) with one star of the catalogue.

$$O_i \xrightarrow{A^T} O \rightarrow s_i$$

$$\hat{o}_n \xrightarrow{A^T} O_n \rightarrow s_i$$

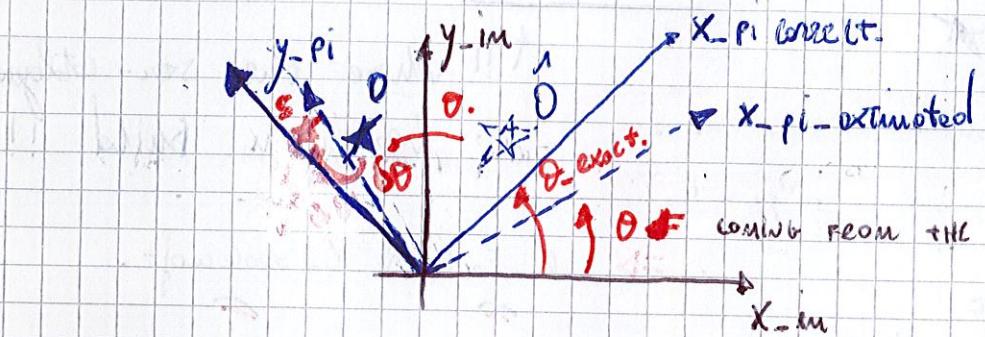
strictly necessary.

!! THE PROCESS REQUIRES AS FIRST GUESS THE EXTIMATION OF MATRIX  $A$  !!

$$\begin{aligned} O_1 &\xrightarrow{A^T} O_1 \dashrightarrow S_1 \\ O_2 &\xrightarrow{A^T} O_2 \dashrightarrow S_2 \\ O_3 &\xrightarrow{A^T} O_3 \dashrightarrow S_3 \\ \vdots & \\ O_n &\xrightarrow{A^T} O_n \dashrightarrow S_n. \end{aligned}$$

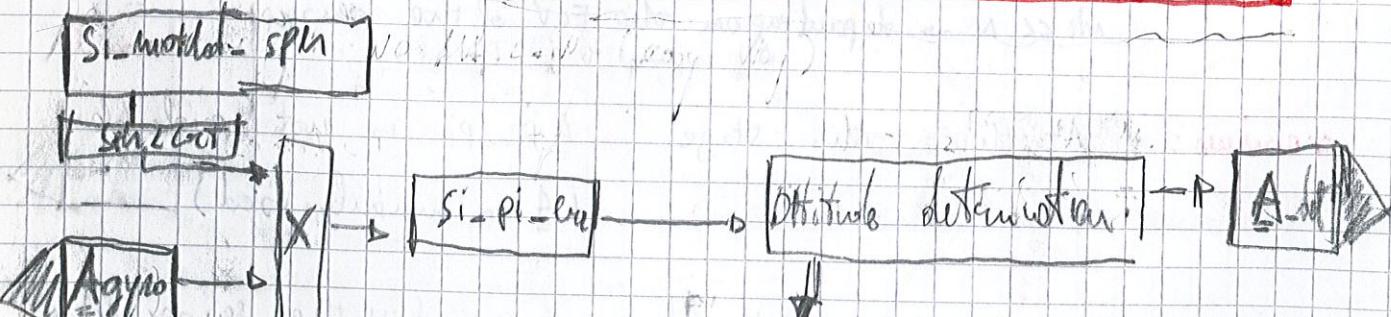
\*

STAR POSITION  
IN THE PI. r.f.



$\Rightarrow$  the angle will be corrected for  $\delta\theta \Rightarrow \theta_{\text{exact}} = \theta + \delta\theta$

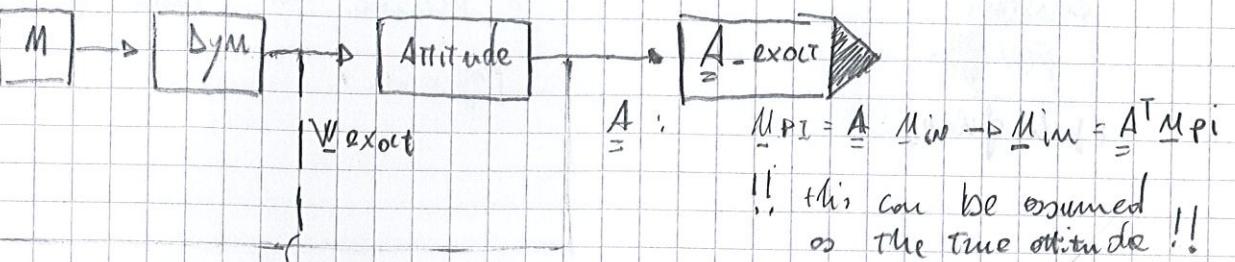
SUCH OPERATION IS POSSIBLE  
IF AND ONLY IF THE ASSOCIATION  
BETWEEN  $\hat{\theta}$  AND  $S$  IS CORRECT.



Based on the hypothesis that  
taking many measurements,  
no error will be taken

PROJECT\_MY

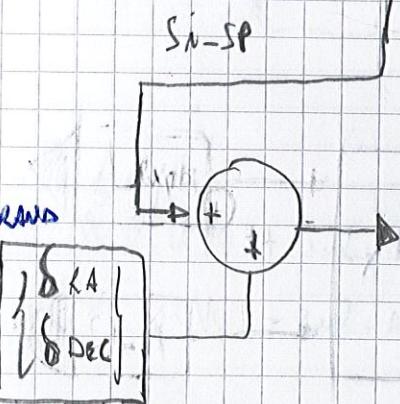
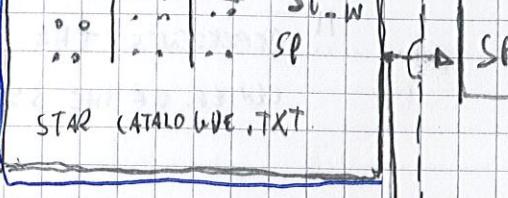
A reasonable simulink procedure is the following:



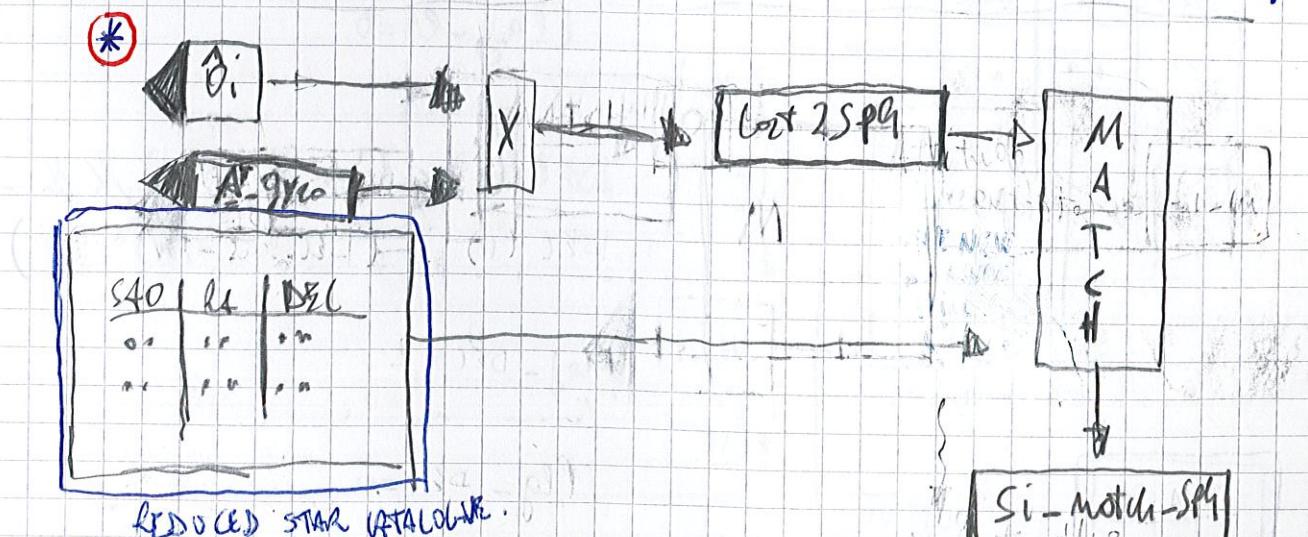
REDUCED STAR CATALOGUE

S40	Re	dec	Si-W
00	00	00	Si-W
00	00	00	SP
00	00	00	

STAR CATALOGUE.TXT



internal error  
in the star source.



Si-motion-SP4

insert store

True position  
of star i  
in the PI. r.f.

contains  
an error.  
linked to  
gyro's error.

observation done  
by the SS.  
in the PI. r.f.

⇒ To select the correct star [portion of celestial sphere]



!! To increase the speed of the projector !!

