A Generalized PSS Architecture for Balancing Transient and Small-Signal Response

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Abstract-For decades, power system stabilizers paired with high initial response automatic voltage regulators have served as an effective means of meeting sometimes conflicting system stability requirements. Driven primarily by increases in power electronically-coupled generation and load, the dynamics of largescale power systems are rapidly changing. Electric grids are losing inertia and traditional sources of voltage support and oscillation damping. The system load is becoming stiffer with respect to changes in voltage. In parallel, advancements in wide-area measurement technology have made it possible to implement control strategies that act on information transmitted over long distances in nearly real time. In this paper, we present a power system stabilizer architecture that can be viewed as a generalization of the standard $\Delta\omega$ -type stabilizer. The control strategy utilizes a realtime estimate of the center-of-inertia speed derived from wide-area measurements. This approach creates a flexible set of trade-offs between transient and small-signal response, making synchronous generators better able to adapt to changes in system dynamics. The phenomena of interest are examined using a two-area test case and a reduced-order model of the North American Western Interconnection. To validate the key findings under realistic conditions, we employ a state-of-the-art co-simulation platform to combine high-fidelity power system and communication network models. The benefits of the proposed control strategy are retained even under pessimistic assumptions of communication network performance.

Index Terms—Automatic voltage regulator, co-simulation, linear time-varying systems, phasor measurement unit, power system stabilizer, real-time control, wide-area measurement systems.

I. INTRODUCTION

HE DELICATE balance between synchronizing and damping torque components in a synchronous machine creates a conflicting set of stability-oriented exciter performance requirements [1]–[4]. Power system stabilizers (PSS) have long played a critical role in satisfying these requirements; however, changes in bulk system dynamics pose challenges to existing control strategies. As inverter-coupled variable generation displaces synchronous machines, electric grids lose inertia and traditional sources of voltage support and oscillation damping.

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Correspondingly, the rapid growth of power electronic loads may make the system load stiffer with respect to changes in voltage [5], [6]. In parallel with these changes, wide-area measurement systems (WAMS) have transformed power system monitoring. The deployment of phasor measurement units (PMUs) has made it is possible to implement control strategies that act on information transmitted over long distances in nearly real time [7]–[10]. Despite the proliferation of inverter-coupled resources, it is projected that synchronous generation will account for a significant fraction of the capacity of large-scale power systems for decades to come [11]. As the dynamics of these systems change, it may become necessary to rethink how synchronous machines are controlled.

In this paper, we derive a new PSS architecture that can be viewed as a generalization of the standard $\Delta\omega$ -type stabilizer. This control strategy stems from a time-varying linearization of the equations of motion for a synchronous machine. It utilizes a real-time estimate of the center-of-inertia speed derived from a set of wide-area measurements. The proposed strategy improves the damping of both local and inter-area modes of oscillation. The ability of the stabilizer to improve damping is decoupled from its role in shaping the system response to transient disturbances. Consequently, the interaction between the PSS and automatic voltage regulator (AVR) can be fine-tuned based on voltage requirements. This approach creates a flexible set of trade-offs between transient and small-signal response, making synchronous generators better able to adapt to changes in system dynamics. Analysis and simulation show that this strategy is tolerant of communication delay, traffic congestion, and jitter.

A. Literature Review

The role that PSSs play in shaping the dynamic system response to severe transient disturbances, such as generator trips, is explored in [12]–[15]. In [15], Dudgeon *et al.* show that the actions of PSSs and AVRs are dynamically interlinked. High initial response AVRs support transient stability but can reduce the damping of electromechanical modes of oscillation. The primary function of PSSs is to improve oscillation damping, but they can also degrade transient stability by counteracting the voltage signal sent to the exciter by the AVR. Managing these interactions through coordinated AVR and PSS design is studied in [16]–[18]. In [14], Grondin, Kamwa, *et al.* present a multiband PSS compensator aimed at improving transient stability by adding damping to the lowest natural resonant frequency. The

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objectives of this compensation approach are similar to those we outline in Section II. We present a PSS architecture that features a new type of multi-band compensator that leverages wide-area measurements to achieve amplitude response attenuation.

Employing remote, or global, input signals to improve the performance of power system damping controllers has inspired many research efforts including [19]-[24]. In [19], Aboul-Ela et al. propose a PSS architecture with two inputs, a local signal used mainly for damping the local mode and a global signal for damping inter-area modes. For the global signal, [19] considers tie-line active power flows and speed difference signals that provide observability of specific inter-area modes. As stated in [14], the ideal stabilizing signal for a PSS "should be in phase with the deviation of the generator speed from the average speed of the entire system." To approximate this ideal signal, the rotor speed is typically passed through a washout (highpass) filter, which may insufficiently attenuate steady-state changes in rotor speed and/or introduce excess phase lead into the bottom end of the control band. In contrast, we explore the implications of combining local measurements with a real-time estimate of the center-of-inertia speed.

The research community is actively working to develop simulation techniques for studying the impact of communication networks on power systems. *Federated co-simulation environments* consist of two or more independent simulation platforms combined so that they exchange data and software execution commands. In [25], Hopkinson *et al.* present EPOCHS, a co-simulation environment that combined Network Simulator 2 (ns-2) with PSLF and PSCAD/EMTDC. Many subsequent research efforts followed, including [26]–[28]. In this paper, the Hierarchical Engine for Large-scale Infrastructure Co-Simulation (HELICS) is employed [28]. We use this state-of-the-art framework to federate a communication network model developed in ns-3 with a power system model developed in the MATLAB-based Power System Toolbox (PST).

B. Paper Organization

The remainder of this paper is organized as follows. Section II derives a generalization of the standard $\Delta\omega$ -type PSS enabled by wide-area measurements. The impact of this control strategy on a two-area test system is examined in Section III. Section IV evaluates how the main results scale to large systems using a reduced-order model of the North American Western Interconnection. In Section IV-C, we study the effect of nonideal communication network performance using co-simulation. Section V summarizes and concludes.

II. PROPOSED METHOD

The proposed PSS architecture arises from a time-varying linearization of the equations of motion for a synchronous machine. Here we briefly restate some key concepts and definitions from the theory of continuous-time linear time-varying systems. In the control strategy derivation, these concepts will be applied to the nonlinear form of the swing equation.

A. Linear Time-Varying Systems

Let $f: \mathbb{R}^n \times \mathbb{R}^m \to \mathbb{R}^n$ denote a nonlinear vector field

$$\dot{x}(t) = f(x(t), u(t)),\tag{1}$$

where $x(t) \in \mathbb{R}^n$ is the system state at time t and $u(t) \in \mathbb{R}^m$ the input. Recall that a time-varying linearization of f takes the form

$$\Delta \dot{x}(t) = A(t)\Delta x(t) + B(t)\Delta u(t), \tag{2}$$

where $\Delta x(t) = x(t) - \overline{x}(t)$ and $\Delta u(t) = u(t) - \overline{u}(t)$. The time-varying trajectory about which the system is linearized is determined by $\overline{x}(t)$ and $\overline{u}(t)$.

The state-space matrices can be expressed compactly as

$$A(t) = D_x f(\overline{x}(t), \overline{u}(t)) \tag{3}$$

$$B(t) = D_u f(\overline{x}(t), \overline{u}(t)), \tag{4}$$

where the operator D_x returns the Jacobian matrix of partial derivatives with respect to x evaluated at time t, and D_u returns the analogous matrix of partials with respect to u. In general, the state-space representation is time-varying when $\overline{x}(t)$ and $\overline{u}(t)$ define a nonequilibrium trajectory.

B. Control Strategy Derivation

This derivation applies the concepts introduced in Section II-A to the equations of motion for a synchronous machine. Stating the nonlinear swing equation in terms of the per-unit accelerating power, we have

$$\dot{\omega}(t) = -\frac{D}{2H} \left[\omega(t) - \omega_0 \right] + \frac{1}{2H\omega(t)} \left[P_m(t) - P_e(t) \right], \quad (5)$$

where ω_0 is the per-unit synchronous speed, D the damping coefficient, and H the inertia constant [3], [4]. Linearizing (5) about a nonequilibrium trajectory yields

$$\Delta \dot{\omega}(t) = -\left[\frac{D}{2H} + \frac{\overline{P}_m(t) - \overline{P}_e(t)}{2H\overline{\omega}(t)^2}\right] \Delta \omega(t) + \frac{1}{2H\overline{\omega}(t)} \left[\Delta P_m(t) - \Delta P_e(t)\right], \tag{6}$$

where $\Delta\omega(t) = \omega(t) - \overline{\omega}(t)$, $\Delta P_m(t) = P_m(t) - \overline{P}_m(t)$, and $\Delta P_e(t) = P_e(t) - \overline{P}_e(t)$.

A new damping coefficient arises from analysis of (6)

$$\mathfrak{D}(t) = D + \frac{\overline{P}_m(t) - \overline{P}_e(t)}{\overline{\omega}(t)^2}.$$
 (7)

Using this coefficient, (6) can be restated as

$$\Delta \dot{\omega}(t) = -\frac{\mathfrak{D}(t)}{2H} \Delta \omega(t) + \frac{1}{2H\overline{\omega}(t)} \left[\Delta P_m(t) - \Delta P_e(t) \right]. \tag{8}$$

Hence, as with a standard $\Delta\omega$ -type stabilizer it is possible to add damping in the LTV reference frame by creating a component of electrical torque that is in phase with the rotor speed deviations. The difference is that the speed deviations are defined such that $\Delta\omega(t)=\omega(t)-\overline{\omega}(t)$, where $\overline{\omega}(t)$ is a function of time that tracks changes in the overall system operating point. The time-varying reference $\overline{\omega}(t)$ makes it possible to almost

completely wash out steady-state changes in rotor speed from the control error.

C. Nonequilibrium Speed Trajectory

In this paper, we will examine the implications of treating $\overline{\omega}(t)$ as a real-time estimate of the center-of-inertia speed

$$\overline{\omega}(t) \approx \frac{\sum_{i \in \mathcal{I}} H_i \omega_i(t)}{\sum_{i \in \mathcal{I}} H_i},$$
 (9)

where i is the unit index and \mathcal{I} the set of all online conventional generators. The right-hand side of (9) corresponds to the classical center-of-inertia definition, dating back to at least [29]. A related quantity that incorporates the machine apparent power ratings is studied in [30]. This alternative approach may be more effective than (9) in capturing the discrepancy in size between large and small machines with similar inertia constants.

The question of how to compute $\overline{\omega}(t)$ for real-time control applications is an interesting research problem in itself that is mostly outside the scope of this paper. A promising method is presented in [31]. At the time of this writing, rotor speed measurements are seldom available through wide-area measurement systems; however, a straightforward way of estimating (9) would be a weighted average of frequency measurements

$$\overline{\omega}(t) = \frac{1}{f_0} \sum_{k \in \mathcal{K}} \alpha_k f_k(t), \tag{10}$$

where k is the sensor index, and f_0 the nominal system frequency. The frequency signal reported by the kth sensor is denoted by $f_k(t)$, and the associated weight by α_k . The weights are nonnegative and sum to one, i.e., $1^T\alpha=1$. For simplicity, we will consider the arithmetic mean in which $\alpha_k=1/|\mathcal{K}|$ for all k, where $|\mathcal{K}|$ denotes the cardinality of \mathcal{K} or simply the number of available sensors. The research contributions presented in this paper do not depend strongly on this choice. There are numerous implementation-related issues posed by any wide-area control scheme, such as how to handle missing or corrupted data. For examples of how these problems may be addressed, see [10], [32].

D. A Generalization of the $\Delta\omega$ -Type PSS

The control strategy implied by (8) can be generalized to encompass the standard $\Delta\omega$ -type PSS. Splitting the linear time-invariant (LTI) control error $\Delta\omega(t)=\omega_i(t)-\omega_0$ into two constituent parts and taking the linear combination yields

$$\Delta\nu(t) \triangleq \beta_1 \left[\omega_i(t) - \overline{\omega}(t)\right] + \beta_2 \left[\overline{\omega}(t) - \omega_0\right], \quad (11)$$

where β_1 and β_2 are independent tuning parameters restricted to the unit interval. In Section IV, we show how the open-loop frequency response between the input to the exciter and the output of the PSS can be shaped by adjusting these parameters. The first term in (11) follows directly from (8), and the second makes it possible to implement a standard $\Delta\omega$ -type PSS using the same framework. As in Section II-C, $\overline{\omega}(t)$ is a real-time estimate of the center-of-inertia speed. Fig. 1 shows the block diagram corresponding to this control strategy where v_s is the

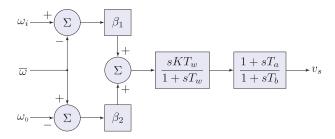


Fig. 1. Generalized $\Delta\omega$ -type PSS block diagram.

TABLE I
EFFECT OF CONTROL PARAMETERS ON PSS TUNING

Parameter Values	Tuning Description
$\beta_1 > \beta_2$ $\beta_1 < \beta_2$	Targets inter-area and local modes Targets the frequency regulation mode
$\beta_1 < \beta_2$ $\beta_1 = \beta_2 \neq 0$	Standard $\Delta\omega$ -type PSS
$\beta_1 = \beta_2 = 0$	No PSS control

output of the PSS. If necessary, more than one lead-lag compensation stage may be employed.

The frequency regulation mode of a power system is a very low-frequency mode, typically below 0.1 Hz, in which the rotor speeds of all synchronous generation units participate [14], [33]. As a consequence of synchronism, the shape of this mode is such that all conventional generators are in phase with one another. As its name implies, the frequency regulation mode is sensitive to load composition, turbine governor time constants, and droop gains. When $\beta_1 > \beta_2$, the control tuning prioritizes the damping of inter-area and local modes of oscillation while de-emphasizing the frequency regulation mode. The converse is true when $\beta_1 < \beta_2$. In the special case that $\beta_1 = \beta_2$, we have a conventional $\Delta\omega$ -type PSS. The resulting control error in this case is exactly the same as in the standard formulation presented in [3]. Table I summarizes the effect of β_1 and β_2 on the PSS tuning.

The diagram shown in Fig. 1 accurately illustrates the control strategy; however, the structure can be clarified further. Expanding the second term in (11) gives

$$\Delta \nu(t) = \beta_1 \left[\omega_i(t) - \overline{\omega}(t) \right] + \beta_2 \overline{\omega}(t) - \beta_2 \omega_0. \tag{12}$$

Thus, we can construct the control error in (11) with a constant reference and a single feedback signal

$$\Delta \nu(t) = \nu(t) - \nu_{\rm ref}, \text{ where}$$
 (13)

$$\nu_{\rm ref} = \beta_2 \omega_0$$
, and (14)

$$\nu(t) = \beta_1 \left[\omega_i(t) - \overline{\omega}(t) \right] + \beta_2 \overline{\omega}(t). \tag{15}$$

The results described in this paper are based on the strategy defined by (13)–(15) and illustrated in Fig. 1.

For the sake of completeness, we present a further refinement that permits the per-unit synchronous speed ω_0 to serve as the reference. The basic idea is to divide (12) by β_2 , taking care to account for the case where $\beta_2=0$. Beginning with the control

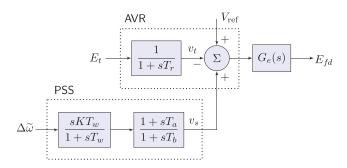


Fig. 2. Excitation system with AVR and PSS, where $G_e(s)$ represents the transfer function of the exciter.

error, we have

$$\Delta \widetilde{\omega}(t) \triangleq \widetilde{\omega}(t) - \omega_0. \tag{16}$$

The feedback signal $\widetilde{\omega}(t)$ is then given by

$$\widetilde{\omega}(t) = \begin{cases} (\beta_1/\beta_2) \left[\omega_i(t) - \overline{\omega}(t) \right] + \overline{\omega}(t), & \text{for } \beta_2 > 0\\ \beta_1 \left[\omega_i(t) - \overline{\omega}(t) \right] + \omega_0, & \text{for } \beta_2 = 0. \end{cases}$$
(17)

This construction is similar to a conventional $\Delta\omega$ -type stabilizer where the local speed measurement $\omega_i(t)$ has been replaced by $\widetilde{\omega}(t)$. Further illustrating this connection, when β_1 and β_2 are equal and nonzero the feedback signal becomes $\widetilde{\omega}(t) = \omega_i(t)$. Fig. 2 shows how the simplified PSS block diagram fits in the context of an excitation system with an AVR. This form is equivalent to the one outlined in (13)–(15) provided that the downstream gain K is scaled appropriately.

E. Comparison With Existing PSS Models

This subsection compares the generalized $\Delta\omega$ -type PSS with two industry-standard stabilizer designs: PSS2C and PSS4C. As described in the IEEE recommended practice for excitation system models [34], PSS2C represents a flexible dual-input stabilizer. This model supersedes and is backward compatible with PSS2A and PSS2B. It may be used to represent two distinct implementation types:

- 1) stabilizers that utilize two inputs to estimate the integral of accelerating power, and
- stabilizers that utilize rotor speed (or frequency) feedback and incorporate a signal proportional to the electrical power as a means of compensation.

The generalized $\Delta\omega$ -type PSS presented here bears similarities to the second of these types. The term in (11) multiplied by β_1 also represents a local rotor speed combined with an auxiliary signal. The first key difference is that $\overline{\omega}(t)$ in (11) is synthesized from from wide-area, rather than strictly local, measurements. The second is that the generalized $\Delta\omega$ PSS also provides the ability to independently adjust the amount of steady-state error included in the feedback. In contrast, PSS2C does not feature a multi-band compensation mechanism.

For multi-band compensation, we turn to PSS4C which builds upon the structure originally proposed in [14]. As discussed in Section I-A, the primary difference between the generalized $\Delta\omega$ PSS and PSS4C is the way the compensation is implemented.

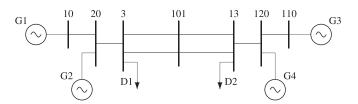


Fig. 3. Oneline diagram of the two-area test system.

The PSS4C structure uses parallel lead-lag compensators to delineate the frequency bands, whereas the generalized $\Delta\omega$ PSS uses the linear combination of steady-state and small-signal components in (11). As shown in Section IV, the latter strategy provides of a means of achieving selective attenuation with minimal impact on the phase response. For an in-depth comparison of PSS2B and PSS4B, the precursors of the models discussed here, see [35].

III. TWO-AREA SYSTEM ANALYSIS

To study the impact of the control strategy outlined in Section II, a combination of time- and frequency-domain analysis was employed. A custom dynamic model based on the block diagram shown in Fig. 1 was implemented in the MATLAB-based Power System Toolbox (PST) [36]. This application facilitates not only time-domain simulation of nonlinear systems but also linearization and modal analysis. Two test systems were studied: a small model based on the Klein-Rogers-Kundur (KRK) two-area system [37], and a reduced-order model of the Western Interconnection. This section summarizes the results of analyzing the two-area test system. It comprises 13 buses, 14 branches, and 4 synchronous generators. A oneline diagram of the system is shown in Fig. 3. In both models, the active component of the system load is modeled as constant current and the reactive component as constant impedance.

To permit study of transient disturbances, several modifications were made to the original KRK system. The synchronous machines in the standard case are representative of aggregate groups of generators concentrated in each area. Each unit has the same capacity and inertia. Hence, tripping any one generation unit offline would be equivalent to losing 25% of the rotating inertia online in the system. To facilitate the study of realistically-sized generator trips, the capacity was redistributed such that each area possessed one machine representative of a collection of generators and the other a large individual plant. Generators G1 and G3 were scaled such that they each represented 5% of the overall system capacity. The remainder was equally split between G2 and G4. Every unit in the system was then outfitted with the generalized $\Delta\omega$ PSS described in Section II.

A. Sensitivity of System Poles to the PSS Tuning Parameters

Here we examine the effects of sweeping the PSS tuning parameters β_1 and β_2 on the poles of the system. The modal analysis was performed by linearizing the system dynamics and then solving for the eigenvalues and eigenvectors of the system matrix. The main result is that the oscillatory modes effectively

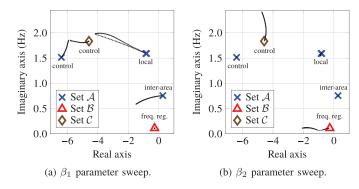


Fig. 4. Sensitivity of the system oscillatory modes to the PSS tuning parameters. The modes in \mathcal{A} are sensitive to changes in β_1 , those in \mathcal{B} to β_2 , and those in \mathcal{C} to both.

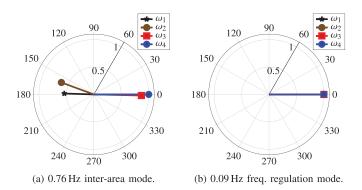


Fig. 5. Normalized mode shape plots for the two-area system.

split into two groups, one that is sensitive to changes in β_1 and the other β_2 . Let us begin by examining the effect of the tuning parameters on the inter-area and local modes. Consider the inter-area mode indicated by the blue x located at 0.76 Hz in Fig. 4. The shape of this mode observed through the machine speeds is shown in Fig. 5(a). Recall that mode shape is defined by the elements of the right eigenvector corresponding to the states of interest [38]. As demonstrated by Fig. 5(a), this mode is characterized by generators G1 and G2 oscillating against G3 and G4. The two-area system is tuned such that this inter-area mode is unstable without supplemental damping control.

The plots in Fig. 4 show the sensitivity of the system poles to the PSS tuning parameters. To generate these plots, either β_1 or β_2 was swept over an interval while the other was held at zero. The tuning parameters for all of the PSS units were swept in unison, and the gain was uniformly held fixed at K=25. Sweeping the tuning parameters for all units together facilitates study of the effect of PSSs on the frequency regulation mode. In Fig. 4(a), β_1 was swept over the interval [0, 1] while β_2 was held at zero. As β_1 increases, the inter-area mode moves to the left and decreases slightly in frequency. The local modes, indicated by the blue x's in the upper right quadrant of Fig. 4(a), move to the left and increase slightly in frequency. A well-controlled exciter mode marked by the blue x in the upper left quadrant moves up and to the right but remains comfortably in the left half of the complex plane. For all intents and purposes, the frequency

regulation mode is unaffected by changes in β_1 . Hence, β_1 dictates the extent to which the PSS damps inter-area and local modes of oscillation.

The parameter β_2 primarily influences the frequency regulation mode. This mode is indicated by the red triangle located at 0.09 Hz in Fig. 4. The shape of the frequency regulation mode observed through the machine speeds is shown in Fig. 5(b). All of the machine speeds are in phase and have nearly identical magnitudes. In Fig. 4(b), the parameter β_2 was swept over the interval [0, 1] while β_1 was held at zero. As β_2 increases, the frequency regulation mode moves to the left. The higherfrequency exciter mode marked with a diamond moves upward. This control mode exhibits some sensitivity to both β_1 and β_2 . In contrast, the inter-area and local modes are relatively unaffected by changes in β_2 . The dependence of the frequency regulation mode on β_2 indicates that PSSs, in aggregate, play an important role in shaping the system response to transient disturbances. To demonstrate this phenomenon, and the effects of the PSS tuning parameters more broadly, we present a collection of time-domain simulations.

B. Time-Domain Simulations

The two-area system was simulated in PST for a variety of PSS tunings. As in the frequency-domain analysis, all PSS units were tuned alike and used the same gain. The contingency of interest in this set of simulations is a trip of generator G3. This event was selected because it initiates a transient disturbance that excites not only the inter-area and local modes but also the frequency regulation mode. In the first set of simulations, β_1 was varied over the set $\{0.33, 0.67, 1\}$ while β_2 was held fixed at 0.33. In the second set of simulations, β_1 was held fixed at 0.33 while β_2 was varied over the set $\{0, 0.33, 0.67\}$. For all simulations, the overall PSS gain was set to K = 18. The case where $\beta_1 = \beta_2 = 0.33$ corresponds to a standard $\Delta\omega$ stabilizer with a gain of K = 6. This set of simulations assumes ideal communication in the construction of the time-varying reference $\overline{\omega}(t)$. Section IV-C addresses the effect of nonideal communication network performance.

Fig. 6 shows the key results for the case where β_1 is varied. The upper subplot shows the difference in speed between generators G2 and G4, i.e., $\omega_2(t)-\omega_4(t)$. The oscillatory content in this signal is dominated by the 0.76 Hz inter-area mode. As β_1 increases, the damping of this mode also increases. The lower subplot shows the terminal voltage of generator G4. As β_1 is varied, the large-signal trajectory of the terminal voltage and its post-disturbance value are unchanged. This reflects the fact that varying β_1 only alters the small-signal characteristics of the field current.

Fig. 7 shows the key results for the case where β_2 is varied. The upper subplot shows the system frequency response, which readily shows the behavior of the frequency regulation mode. The results show that β_2 plays a key role in determining the depth of the frequency nadir. The frequency nadir improves significantly as β_2 increases from 0 to 0.33, and modestly as it goes from 0.33 to 0.67. Effectively, β_2 determines the level of overshoot in the system step response. The lower subplot

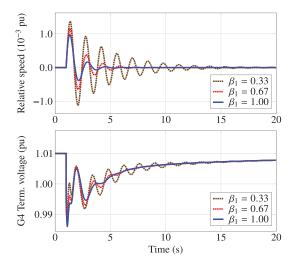


Fig. 6. Time-domain simulations of generator G3 being tripped offline for various values of β_1 . The top subplot shows the relative speed between G2 and G4.

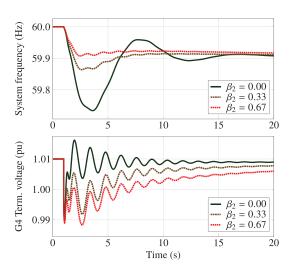


Fig. 7. Time-domain simulations of generator G3 being tripped offline for various values of β_2 .

shows the terminal voltage of generator G4. As β_2 is increased, the terminal voltage following the generator trip becomes incrementally more depressed. This can be attributed to the fact that β_2 controls the extent to which steady-state changes in rotor speed are included in the PSS control error.

The process that causes β_2 to affect the frequency nadir is indirect. Increasing β_2 amplifies the steady-state component of the control error in (11). This depresses the field current supplied by the exciter and causes the voltage induced in the stator to dip. The electrical load decreases in response to this voltage dip with the amount of relief depending on the sensitivity of the load with respect to voltage. This tends to reduce the time-varying mismatch in mechanical and electrical torque, which improves the frequency nadir. This effect depends on the load composition, and the amount of improvement in the nadir decreases as the fraction of constant power load increases. Thus, there is a trade-off between improving the frequency nadir and degrading the

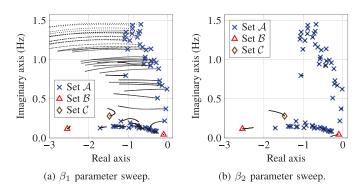


Fig. 8. Sensitivity of the miniWECC oscillatory modes to the PSS tuning parameters. The modes in \mathcal{A} are sensitive to changes in β_1 , those in \mathcal{B} to β_2 , and those in \mathcal{C} to both.

voltage response. As explained in [15], the tendency of the PSS to counteract the voltage signal sent to the exciter by the AVR can reduce synchronizing torque and degrade transient stability. The control strategy presented in this paper makes it possible to fine-tune the interaction between the PSS and AVR without affecting the damping of inter-area and local modes, and vice versa.

IV. LARGE-SCALE TEST SYSTEM ANALYSIS

For the two-area system discussed in Section III, the inter-area and local modes were influenced by β_1 , and the frequency regulation mode by β_2 . This section addresses whether this property is preserved for large-scale systems. We consider a reduced-order model of the Western Interconnection named the miniWECC, in reference to the Western Electric Coordinating Council (WECC). It comprises 122 buses, 171 ac branches, 2 HVDC lines, and 34 synchronous generators. This system spans the entirety of the interconnection including British Columbia and Alberta. Its modal properties have been extensively validated against real system data [9], [39]. The aim of this analysis is to illustrate the fundamental behavior of various aspects of the proposed architecture in a controlled setting. Prior to implementation, high-fidelity simulation studies that account for variation in PSS structure and the dynamics of inverter-coupled generation would be required.

A. Sensitivity of System Poles to the PSS Tuning Parameters

To examine the sensitivity of the oscillatory modes to the PSS tuning parameters, the method described in Section III-A was applied to the miniWECC. Every generation unit in the system was outfitted with a generalized $\Delta\omega$ PSS with the gain set to K=25. In practice, WECC policy dictates that "a PSS shall be installed on every synchronous generator that is larger than 30MVA, or is part of a complex that has an aggregate capacity larger than 75MVA, and is equipped with a suitable excitation system" [40]. Fig. 8 shows the movement of the system poles in response to changes in the tuning parameters. In each subplot, either β_1 or β_2 was swept over an interval while the other was held at zero. The main result matches the one observed for the two-area system. The inter-area and local modes are

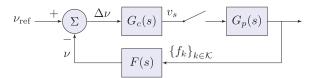


Fig. 9. Feedback loop for a single generation unit outfitted with a generalized $\Delta\omega$ PSS, where $G_c(s)$ represents the controller, $G_p(s)$ the plant, and F(s) the feedback process.

influenced by β_1 , and the frequency regulation mode by β_2 . For the miniWECC, there is one well-controlled exciter mode near 0.28 Hz that exhibits sensitivity to both parameters. This mode is marked in Fig. 8 with a diamond.

B. Open-Loop Frequency Response Analysis

The frequency-domain analysis presented in Sections III and IV-A focused on a system-wide perspective. Here we provide a unit-specific analysis of the open-loop frequency response for a single generator. Outfitting a single unit with a PSS yields the state-space representation

$$\dot{x}(t) = Ax(t) + B_n u(t) \tag{18}$$

$$y_{\nu}(t) = C_{\nu}x(t),\tag{19}$$

where B_p describes how the system states are affected by changes in the PSS control input. The closed-loop control action determined by (13) can be implemented with the input

$$u(t) = -Ky_{\nu}(t) = -KC_{\nu}x(t)$$
 (20)

$$u(t) = -K \begin{bmatrix} 0 & \gamma_1 & \gamma_2 & \dots & -\beta_1 \end{bmatrix} \begin{bmatrix} \widehat{x}(t) \\ f_1(t) \\ f_2(t) \\ \vdots \\ \omega_i(t) \end{bmatrix}, \quad (21)$$

where K is a scalar gain. The output matrix C_{ν} combines the states to form the PSS feedback signal $\nu(t)$. Note the presence of the extra negative sign to conform to the negative feedback convention. The state vector x in (21) is organized with the unused states \widehat{x} on top, followed by the frequency measurements and the local rotor speed. For the kth sensor $\gamma_k = \alpha_k \left(\beta_1 - \beta_2\right)/f_0$, where α_k stems from the linear combination in (10), and f_0 is the nominal system frequency. In this analysis, the frequencies were computed by applying a derivative-filter cascade to the bus voltage angles as described in [10]. Hence, the unity-gain open-loop transfer function between a change in the PSS reference $\nu_{\rm ref}$ and the feedback signal ν is

$$H(s) = C_{\nu}(sI - A)^{-1}B_{\nu}.$$
 (22)

Fig. 9 shows a high-level block diagram of the feedback loop for a single generation unit outfitted with a generalized $\Delta\omega$ PSS. Here $G_c(s)$ represents the PSS, $G_p(s)$ the plant, and F(s) the feedback process. The exciter dynamics are included in $G_p(s)$, and the input to the plant represents a change in the exciter

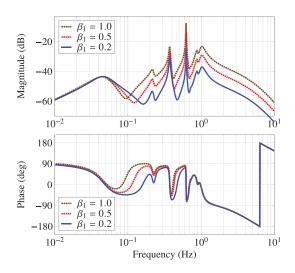


Fig. 10. The effect of β_1 on the open-loop frequency response between the input to the exciter and the output of the generalized PSS for generator G2.

voltage reference $V_{\rm ref}$. By commutativity, it holds that

$$H(s) = G_c(s)G_p(s)F(s) = G_p(s)F(s)G_c(s).$$
 (23)

Hence, the loop transfer function between $\Delta \nu$ and ν is the same as the transfer function between a change in the exciter voltage reference $V_{\rm ref}$ and the output of the PSS v_s . Using this function, we can evaluate the effect of the PSS tuning parameters on the open-loop frequency response. For this analysis, only the unit being studied was outfitted with a PSS.

Fig. 10 shows the effect of β_1 on the open-loop frequency response for generator G2, a hydroelectric unit in eastern British Columbia, where $\beta_2=1$ for all traces. The peak in the amplitude response near 0.04 Hz corresponds to the frequency regulation mode. As the plot shows, β_1 has no effect on the gain of the system at this frequency. This corroborates the system-wide modal analysis done in Sections III and IV-A at the unit level. As expected, β_1 does change the amplitude response for the inter-area and local modes of oscillation. Unlike traditional compensation methods, this approach does not degrade the phase response in the attenuation region. As β_1 is varied, the phase response at the frequencies of the dominant amplitude peaks (0.37 Hz, 0.62 Hz, and 1.0 Hz) is essentially unchanged. The observed transition in phase through 0° at the resonant frequencies is ideal for damping control.

Fig. 11 shows the effect of β_1 on the overall PSS compensation. As in [41], the uncompensated open-loop frequency response, including the washout filter dynamics, is provided for comparison. The overall compensation comprises both the lead-lag compensator and the tuning determined by β_1 , β_2 . When $\beta_1 = \beta_2 = 1$, the tuning stage has a gain of unity and imparts no phase shift. Hence, all of the compensation stems from the lead-lag compensator. This is expected because the case where $\beta_1 = \beta_2 = 1$ yields a standard $\Delta\omega$ stabilizer as shown in Table I.

Fig. 12 shows the effect of β_2 on the open-loop frequency response where $\beta_1 = 1$ for all traces. As β_2 is varied, the amplitude response at the frequencies corresponding to the local

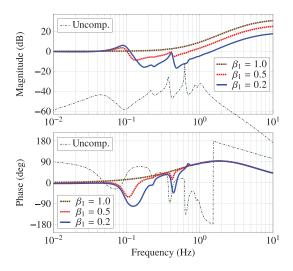


Fig. 11. The effect of β_1 on the overall PSS compensation for generator G2 with the washout filter included in the uncompensated frequency response.

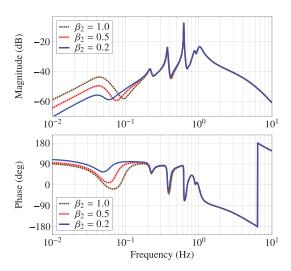


Fig. 12. The effect of β_2 on the open-loop frequency response between the input to the exciter and the output of the generalized PSS for generator G2.

and inter-area modes is effectively unchanged. In contrast, the gain at the frequency regulation mode is reduced by roughly 14 dB as β_2 goes from 1 to 0.2. For $\beta_2=0.2$, the phase response at the frequency regulation mode leads the case where $\beta_2=1$ by roughly 35°. This suggests that if a β_2 value below some nominal threshold is required for a particular application, it may be necessary to retune the lead-lag compensator and/or washout filter to ensure satisfactory low-frequency performance. The effect of β_2 on the overall PSS compensation is shown in Fig. 13.

C. Co-Simulation of Power and Communication Systems

All of the analysis presented in Sections III through IV-B was performed under the assumption of ideal communication. In this section, we analyze the effect of communication delay in the frequency domain and verify the findings in the time domain, as

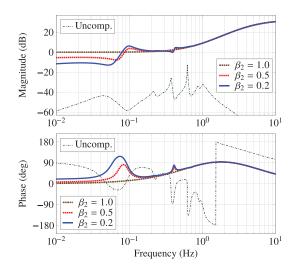


Fig. 13. The effect of β_2 on the overall PSS compensation for generator G2 with the washout filter included in the uncompensated frequency response.

in [42]. The mathematical modeling developed here represents the real-time exchange of synchronized phasor measurement data over a network. As described in the IEEE standard governing data transfer in PMU networks [43], communication delays in WAMS are typically in the range of 20–50 ms; however, the combined delay must also account for the effect of transducers, processing, concentrators, and multiplexing [44]–[46]. In [44], the delay attributed to these factors is estimated at 75 ms, which yields an approximate range of 95–125 ms for the combined delay. This range is reflective of systems that utilize fiber-optic communication. It aligns closely with the experimental results reported in [10], 69–113 ms, but may vary depending on the communication method employed, e.g., wired vs. wireless. Here we evaluate scenarios with delays that are 5 to 10 times greater than the high end of this range.

Modifying the state-space output matrix in (21) to account for delays as in [47], we have

$$\widehat{C}_{\nu}(s) = \begin{bmatrix} 0 & \gamma_1 e^{-s\tau_1} & \gamma_2 e^{-s\tau_2} & \dots & -\beta_1 \end{bmatrix}$$
 (24)

$$\hat{H}(s) = \hat{C}_{\nu}(s) [sI - A]^{-1} B_p,$$
 (25)

where τ_k is the delay of the kth sensor. Thus, the output matrix changes as a function of frequency. The open-loop transfer function with delay is given by (25). Fig. 14 shows the results of using (25) to evaluate the effect of delay on the open-loop frequency response for generator G2 with $\beta_1=1$ and $\beta_2=0.5$. For simplicity, $\tau_k=\tau$ for all k. The entries of (24) correspond to the case where the local signal is not delayed, and the local and remote measurements are not time-aligned upon arrival. As a result, $\widehat{H}(s) \neq H(s)e^{-s\tau}$. In the extreme case where $\tau=1.25\,\mathrm{s}$ shown in Fig. 14, the gain and phase are altered slightly in the neighborhood of the frequency regulation mode; however, the control performance and stability margins are essentially unchanged.

To study the impact of nonideal communication performance in the time domain, we used a co-simulation framework called

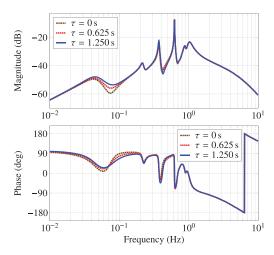


Fig. 14. The effect of the combined delay τ on the open-loop frequency response for generator G2 with $\beta_1 = 1$ and $\beta_2 = 0.5$.

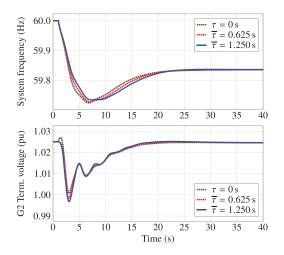


Fig. 15. Simulations of generator G26 being tripped offline for various average combined delays where $\beta_1 = 1$ and $\beta_2 = 0.5$.

HELICS [28]. A communication network model for the mini-WECC was developed in ns-3 [48]. It features PMU endpoints that communicate with the controllers via the User Datagram Protocol (UDP). This model includes transmission delay, congesting traffic, and packet-based error emulation. Each generation unit in the PST model was outfitted with a generalized $\Delta\omega$ PSS where $\beta_1=1,\ \beta_2=0.5,$ and K=9. Fig. 15 shows time-domain simulations of generator G26, a large nuclear plant in Arizona, being tripped offline for various expected delays $\overline{\tau}$. The results are in close agreement with the frequency-domain analysis shown in Fig. 14. Thus, for this example, the benefits of the control strategy are retained even under pessimistic assumptions of communication network performance.

In the miniWECC examples discussed herein, the center-of-inertia speed estimate $\overline{\omega}(t)$ was synthesized using 30 sensors geographically distributed throughout the system. The effect of delay on the open-loop frequency response is dependent on the number and placement of the frequency (or speed) sensors. These factors determine how well $\overline{\omega}(t)$ tracks the target defined

TABLE II EFFECT OF CONTROL PARAMETERS ON $\widehat{\gamma}_k$ Coefficients

Parameter Ratio	Coefficient Range
$\beta_2/\beta_1 < 1$	$0<\widehat{\gamma}_k\leq \alpha_k/f_0$
$\beta_2/\beta_1 = 1$	$0 \le \widehat{\gamma}_k \le 0$
$\beta_2/\beta_1 > 1$	$-\infty \le \widehat{\gamma}_k < 0$

in (9). By extension, they also influence its spectral content. When $\overline{\omega}(t)$ approximately tracks the true center-of-inertia speed, its amplitude spectrum is dominated by very low-frequency content, generally ≤ 0.1 Hz. If too few sensors are used to synthesize this estimate, and/or those sensors are not adequately distributed, the amplitude spectrum of $\overline{\omega}(t)$ may include significant higher-frequency content, in and above the range of the electromechanical modes. If this occurs, the delay may impart larger deviations in the phase response above the frequency regulation mode than shown in Fig. 14. For similar reasons, the coefficients of the linear combination in (10) also affect the relationship between the combined delay and the frequency response.

The other main factor influencing this relationship is the tuning determined by β_1 , β_2 . Analysis indicates that tunings where $\beta_1 < \beta_2$ may be more susceptible to the effects of delay than those where $\beta_1 \geq \beta_2$. To explore this behavior, we will analyze the entries of the output matrix $\widehat{C}_{\nu}(s)$. Let $\widehat{\gamma}_k = \gamma_k/\beta_1$. The matrix $\widehat{C}_{\nu}(s)$ may then be expressed as

$$\widehat{C}_{\nu}(s) = \beta_1 \begin{bmatrix} 0 & \widehat{\gamma}_1 e^{-s\tau_1} & \widehat{\gamma}_2 e^{-s\tau_2} & \dots & -1 \end{bmatrix}, \quad (26)$$

where

$$\widehat{\gamma}_k = \frac{\alpha_k}{f_0} \left(1 - \frac{\beta_2}{\beta_1} \right). \tag{27}$$

Recall from (10) that the weights α_k are nonnegative and sum to one. For all real $\omega \tau$, it holds that $|e^{-j\omega \tau}|=1$. Thus, the relationship between the magnitudes of the entries of $\widehat{C}_{\nu}(s)$ corresponding to the delayed and non-delayed system states is primarily determined by the ratio β_2/β_1 . Table II shows a breakdown of the possible cases. The term inside the brackets of (26) corresponding to the local rotor speed always has a magnitude of one. The magnitudes of the remaining entries may either be zero, bounded, or unbounded.

When $\beta_2/\beta_1=1$ the controller is immune to delay because $\widehat{\gamma}_k=0$ for all k. This aligns with expectations because the case where $\beta_1=\beta_2\neq 0$ corresponds to a standard $\Delta\omega$ stabilizer, as shown in Table I. When $\beta_2/\beta_1<1$, the magnitude of $\widehat{\gamma}_k$ has an upper bound of α_k/f_0 . This corresponds to the case where $\beta_1>\beta_2$, and the PSS prioritizes the damping of local and inter-area modes. When $\beta_2/\beta_1>1$, $\widehat{\gamma}_k$ is unbounded below. Thus, the magnitude of $\widehat{\gamma}_k$ may grow arbitrarily large as $\beta_1\to 0$. This does not imply that $\widehat{C}_{\nu}(s)$ may have infinite values; rather, that the steady-state component of the control error (11) may be much larger than the small-signal component. This corresponds to the case where $\beta_1<\beta_2$, and the PSS prioritizes shaping the system response to transient disturbances. It is observed that the sensitivity of the open-loop frequency response to delay

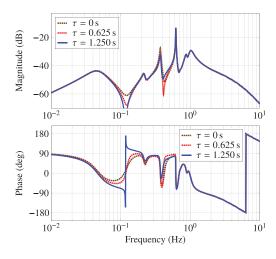


Fig. 16. The effect of the combined delay τ on the open-loop frequency response for generator G2 with $\beta_1=0.5$ and $\beta_2=1$.

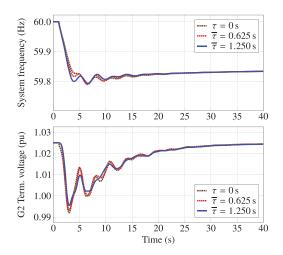


Fig. 17. Simulations of generator G26 being tripped offline for various average combined delays where $\beta_1=0.5$ and $\beta_2=1$.

increases as the ratio β_2/β_1 increases. We hypothesize that the driving factor in this relationship is that as β_2/β_1 grows, so too do the magnitudes of the entries of $\widehat{C}_{\nu}(s)$ corresponding to the delayed system states in relation to the non-delayed state(s). That is, when $\beta_1 \ll \beta_2$, it follows that $|\widehat{\gamma}_k| > 1$ for some k. This suggests that if $\beta_1 < \beta_2$, the ratio β_2/β_1 should be kept small.

To illustrate this behavior, suppose that $\beta_1=0.5$, $\beta_2=1$ for generator G2, where $\beta_2/\beta_1=2$. Fig. 16 shows the effect of delay on the open-loop frequency response for G2 in this case. As the delay increases, a key transfer function zero changes position in the complex plane. Fig. 16 indicates that this zero is pushed across the $j\omega$ -axis between 0.1–0.2 Hz and into the right half of the complex plane as the delay increases. Right-half-plane zeros, especially in the neighborhood of the electromechanical modes, may erode stability margins and are generally undesirable [21]. In this case, the system remains stable when $\tau=1.25$ s because the gain at the critical frequencies is very low.

Fig. 17 shows the system response in the time domain following a trip of G26. Each generation unit in the PST model was

outfitted with a generalized $\Delta\omega$ PSS where $\beta_1=0.5$, $\beta_2=1$, and K=9. As the state trajectories show, this tuning places more emphasis on shaping the transient response than on damping local and inter-area modes. Such parameter combinations should be used with caution. Careful stability analysis must be carried out to ensure that it is safe to employ a particular tuning given the performance characteristics of the measurement, communication, and control equipment.

V. CONCLUSION

This paper presented a generalization of the standard $\Delta\omega$ -type stabilizer. It works by incorporating local information with a real-time estimate of the center-of-inertia speed. The ability of the stabilizer to improve the damping of electromechanical modes is decoupled from its role in shaping the system response to transient disturbances. Hence, the interaction between the PSS and AVR can be fine-tuned based on voltage requirements. Future work will explore a variation of the proposed architecture that permits integral of accelerating power feedback for mitigating torsional oscillations. Finally, another interesting avenue of research will be developing online methods to optimally estimate the center-of-inertia frequency in the presence of delays, jitter, and measurement noise.

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