

## 5 Performance Indicators

Performance indicators are used to judge the quality of a control system.

### 5.1 1<sup>st</sup>-order System Generic Performance Indicators

Generic performance indicators are:

- steady-state value  $x_{ss} = \lim_{t \rightarrow \infty} x(t)$
- steady-state error  $e_{ss} = \lim_{t \rightarrow \infty} e(t)$  where  $e(t) = f(t) - x(t)$

To expand on the error definition,  $e(t) = f(t) - x(t)$ , consider a step response where  $x(\infty) = 1$  and  $f(\infty) = 1$ , therefore  $e(\infty) = 0$ .

For a 1<sup>st</sup>-order system,  $x_{ss}$  and  $e_{ss}$  can be found using the final value theorem and exist in both the time domain and the s-domain. Recall the final value theorem for  $x_{ss}$  and  $e_{ss}$  leads to

$$x_{ss} = \lim_{s \rightarrow 0} sX(s) \quad (5.1)$$

$$e_{ss} = \lim_{s \rightarrow 0} sE(s) \quad (5.2)$$

Therefore, starting at the transfer function of a 1<sup>st</sup>-order system

$$G(s) = \frac{1}{Ts + 1} \quad (5.3)$$

we can expand on this to show

$$\begin{aligned} X(s) &= G(s)F(s) \\ &= \frac{1}{Ts + 1} F(s) \end{aligned} \quad (5.4)$$

next, the error in s-domain is shown to be

$$E(s) = F(s) - X(s) \quad (5.5)$$

$$= F(s) - G(s)F(s) \quad (5.6)$$

$$= (1 - G(s))F(s)$$

$$= \frac{Ts}{Ts + 1} F(s)$$

Again, these are general terms for a 1<sup>st</sup>-order system.

5.1.1 Step response 1<sup>st</sup>-order system performance indicators

For a 1<sup>st</sup>-order system subjected to a step response, we want to find  $x_{ss}$  and  $e_{ss}$  and we know the transfer function is defined as

$$G(s) = \frac{1}{Ts + 1} \quad (5.7)$$

while the equation of motion is

$$T\dot{x} + x = F(s) \quad (5.8)$$

Where the step function is defined as

$$f(t) = 1, t > 0 \quad (5.9)$$

therefore, the system response is

$$x(t) = 1 - e^{-t/T} \quad (5.10)$$

while the steady-state response is

$$\begin{aligned} x_{ss} &= \lim_{t \rightarrow \infty} \\ &= 1 \end{aligned} \quad (5.11)$$

Next, the error as a function of time is

$$\begin{aligned} e(t) &= 1 - (1 - e^{-t/T}) \\ &= e^{-t/T} \end{aligned} \quad (5.12)$$

while the steady-state error is

$$\begin{aligned} e_{ss} &= \lim_{t \rightarrow \infty} e(t) \\ &= \lim_{t \rightarrow \infty} e^{-t/T} \\ &= 0 \end{aligned} \quad (5.13)$$

These same solutions can be found in the s-domain where the step function is  $F(s) = \frac{1}{s}$ . Consider the s-domain expression solved for  $X(s)$ ,

$$X(s) = \frac{1}{Ts + 1} \cdot \frac{1}{s} \quad (5.14)$$

the steady-state error is shown to be

$$\begin{aligned} x_{ss} &= \lim_{s \rightarrow 0} s \frac{1}{Ts + 1} \cdot \frac{1}{s} \\ &= \lim_{s \rightarrow 0} \frac{1}{Ts + 1} \\ &= 1 \end{aligned} \quad (5.15)$$

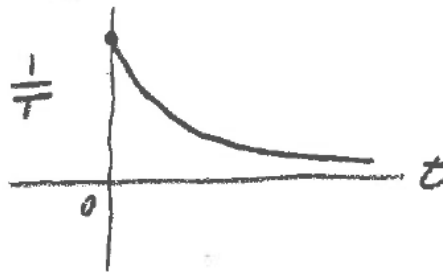
Next, we can build the s-domain representation of the error as

$$\begin{aligned} E(s) &= \frac{Ts}{Ts + 1} \cdot \frac{1}{s} \\ &= \frac{T}{Ts + 1} \end{aligned} \quad (5.16)$$

solving for the steady-state error results in

$$\begin{aligned} e_{ss} &= \lim_{s \rightarrow 0} s \frac{T}{Ts + 1} \\ &= 0 \end{aligned} \quad (5.17)$$

### 5.1.2 Impulse response 1<sup>st</sup>-order system performance indicators



For a 1<sup>st</sup>-order system subjected to a impulse response, we want to find  $x_{ss}$  and  $e_{ss}$ . The impulse function is defined as

$$f(t) = \delta(t) \quad (5.18)$$

therefore, the response is

$$x(t) = \frac{1}{T} e^{-t/T} \quad (5.19)$$

where the steady-state response is

$$x_{ss} = 0 \quad (5.20)$$

The error is

$$e(t) = \delta(t) - \frac{1}{T} e^{-t/T} \quad (5.21)$$

Lastly, the steady-state error is

$$\begin{aligned} e_{ss} &= \lim_{t \rightarrow \infty} e(t) \\ &= 0 \end{aligned} \quad (5.22)$$

These same solutions can be found in the s-domain where the impulse function is  $F(s) = 1$ . Consider the s-domain expression solved for  $X(s)$ ,

$$X(s) = \frac{1}{Ts + 1} \quad (5.23)$$

the steady-state error is shown to be

$$\begin{aligned} x_{ss} &= \lim_{s \rightarrow 0} s \frac{1}{Ts + 1} \\ &= 0 \end{aligned} \quad (5.24)$$

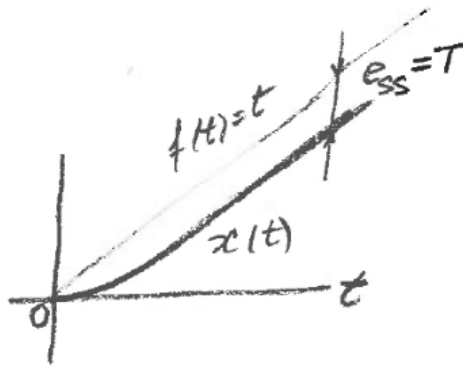
The s-domain error is expressed as

$$E(s) = \frac{T}{Ts + 1} \quad (5.25)$$

which leads to the steady-state error value

$$\begin{aligned} e_{ss} &= \lim_{s \rightarrow 0} s \frac{T}{Ts + 1} \\ &= 0 \end{aligned} \quad (5.26)$$

### 5.1.3 Ramp response 1<sup>st</sup>-order system performance indicators



For a 1<sup>st</sup>-order system subjected to a ramp response, we want to find  $x_{ss}$  and  $e_{ss}$ . The impulse function is defined as

$$f(t) = t \quad (5.27)$$

therefore, the response is

$$x(t) = t - T(1 - e^{-t/T}) \quad (5.28)$$

which leads to the steady-state response

$$x_{ss} = \infty \quad (5.29)$$

which means that there is no steady-state value. The error is

$$e(t) = T(1 - e^{-t/T}) \quad (5.30)$$

Lastly, the steady-state error is

$$\begin{aligned} e_{ss} &= \lim_{t \rightarrow \infty} e(t) \\ &= T - \lim_{t \rightarrow \infty} e^{-t/T} \\ &= T \end{aligned} \quad (5.31)$$

as  $\lim_{t \rightarrow \infty} e^{-t/T} = 0$ .

These same solutions can be found in the s-domain where the ramp function is  $F(s) = \frac{1}{s^2}$ . Consider the s-domain expression solved for  $X(s)$ ,

$$X(s) = \frac{1}{Ts + 1} \cdot \frac{1}{s^2} \quad (5.32)$$

the steady-state error is shown to be

$$\begin{aligned} x_{ss} &= \lim_{s \rightarrow 0} s \frac{1}{Ts + 1} \cdot \frac{1}{s^2} \\ &= \lim_{s \rightarrow 0} \frac{1}{Ts + 1} \cdot \frac{1}{s} \\ &= \infty \end{aligned} \quad (5.33)$$

therefore, there is no steady-state value. The s-domain error is expressed as

$$\begin{aligned} E(s) &= \frac{Ts}{Ts + 1} \cdot \frac{1}{s^2} \\ &= \frac{T}{Ts + 1} \cdot \frac{1}{s} \end{aligned} \quad (5.34)$$

which leads to the steady-state error value

$$\begin{aligned} e_{ss} &= \lim_{s \rightarrow 0} s \frac{T}{Ts + 1} \cdot \frac{1}{s} \\ &= \lim_{s \rightarrow 0} \frac{T}{Ts + 1} \\ &= T \end{aligned} \quad (5.35)$$

## 5.2 1<sup>st</sup>-order System Specific Performance Indicators

Specific performance indicators exist. They depend on system order and excitation type. Examples are:

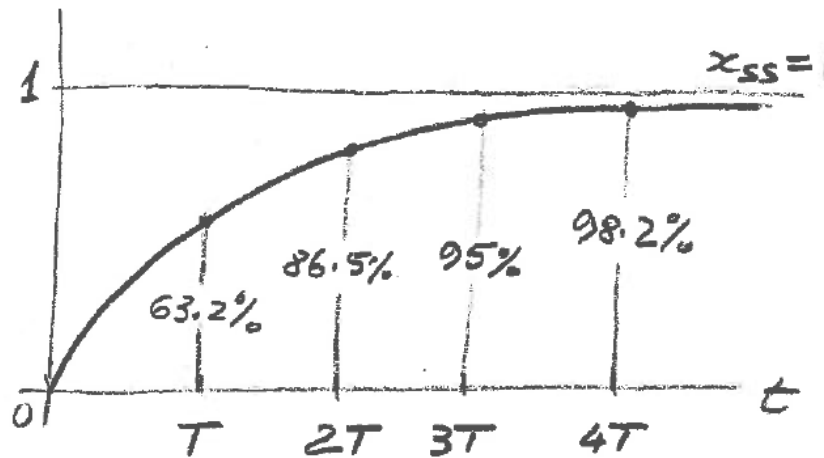
- rise-time  $\rightarrow t_r$
- delay time  $\rightarrow t_d$
- settling time  $\rightarrow t_s$

- decay time / half-time  $\rightarrow t_{1/2}$

For a step response, consider the system displacement

$$x(t) = 1 - e^{-t/T} \quad (5.36)$$

that is plotted as



The rise-time ( $t_r$ ) of a system

- to rise 63.2% of  $x_{ss}$  is  $t_{r,63.2\%} = T$
- to rise 86.5% of  $x_{ss}$  is  $t_{r,86.5\%} = 2T$
- to rise 95% of  $x_{ss}$  is  $t_{r,95\%} = 3T$
- to rise 98.2% of  $x_{ss}$  is  $t_{r,98.2\%} = 4T$

in general,  $1 - e^{-t/T} = x \rightarrow t = -T \ln(1 - x)$

The delay time ( $t_d$ ) is the time it takes to rise to 50% of  $x_{ss}$ ,

$$x(t) = 1 - e^{-t/T} = 0.5 \quad (5.37)$$

therefore, solving for  $t = t_d$ ,

$$e^{-t_d/T} = 0.5 \quad (5.38)$$

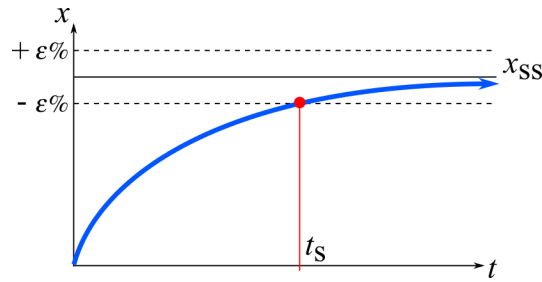
$$\frac{-t_d}{T} = \ln(0.5)$$

$$t_d = -T \ln(0.5)$$

$$t_d = 0.693T$$

$$t_d \approx 0.7T$$

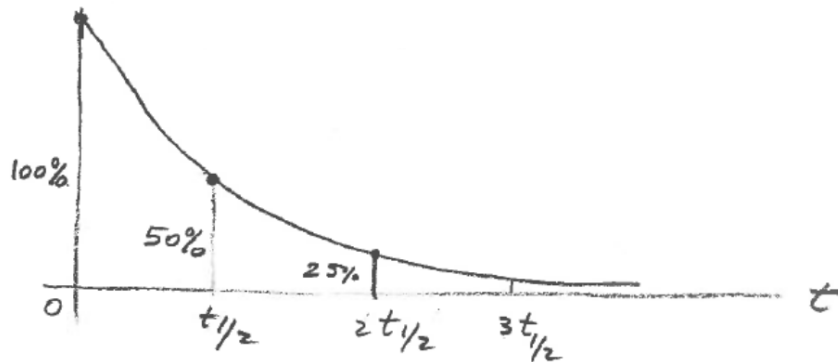
The settling time ( $t_s$ ) is the time it takes  $x$  to get within  $\varepsilon\%$  of  $x_{ss}$ .



A more precise estimator is the rise time to any selected  $x$  which can be defined as

$$\begin{aligned} x_{ss}(1 - e^{-t/T}) &= x \\ e^{(-t/T)} &= 1 - \frac{x}{x_{ss}} \\ -t/T &= \log \left[ 1 - \frac{x}{x_{ss}} \right] \\ t &= -T \log \left[ 1 - \frac{x}{x_{ss}} \right] \end{aligned} \quad (5.39)$$

### 5.2.1 Impulse Response Specific Performance Indicators



The half-cycle decay time ( $t_{1/2}$ ) is an impulse response specific performance indicator. Given that the system response to an impulse is

$$x(t) = \frac{1}{T} e^{-t/T} \quad (5.40)$$

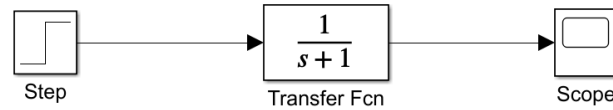
the half-life decay time is where  $e^{-t/T} = \frac{1}{2}$ , therefore

$$\begin{aligned} t_{1/2} &= -\ln \left( \frac{1}{2} \right) T \\ &\approx 0.693T \end{aligned} \quad (5.41)$$

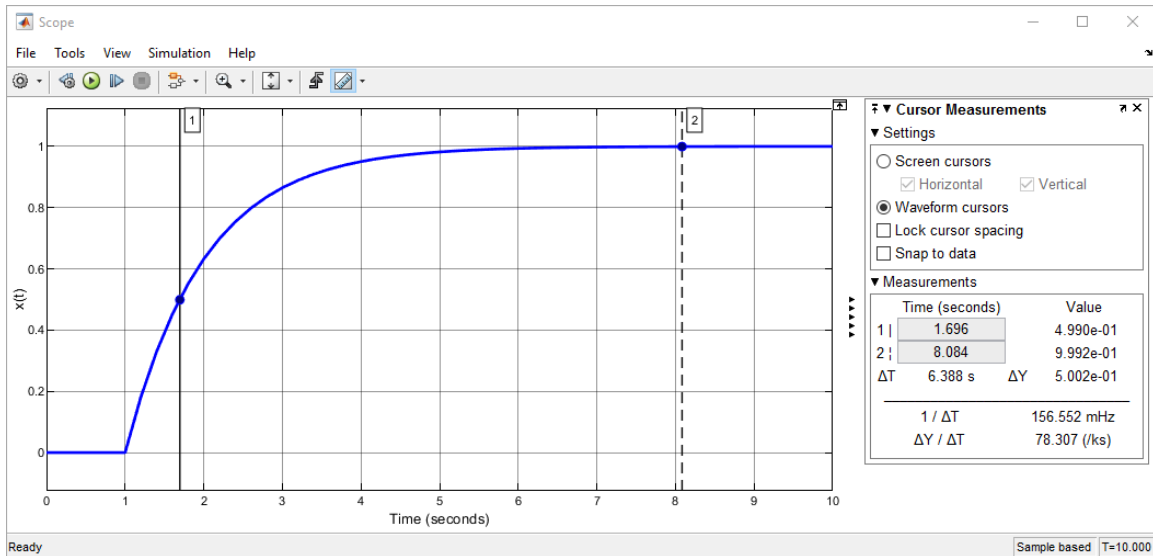
Note that the signal continues to decay by half of its value after each additional  $t_{1/2}$ .

**Example 5.1 SIMULINK Tutorial on Measuring Performance Indicators**

Build the simple 1<sup>st</sup>-order system as shown below



Open the scope and press the ‘Cursor Measurements’ button to activate the cursors.



To measure the delay time  $t_d$ , place the first cursor around the point where ‘Value’ measurement is closest to 0.5.  $x(t) = 0.5$ . Read the ‘Time’ value. This is estimate for  $t_d$ . It gives the value  $t_d = 0.70$  sec, as the step function happens at 1 sec.

The second cursor can be used to get the settling time  $t_s$ . We are going to use the 1% definition of  $t_d$ . This means that the response should be around 0.99, or 9.9e-1. Reading the corresponding time value, we get  $t_s \approx 7.0$  sec.

**5.3 2<sup>nd</sup>-order System Generic Performance Indicators**

Again, starting at the equation of motion

$$\ddot{x} + 2\zeta\omega_n\dot{x} + \omega_n^2x = \omega_n^2f(t) \quad (5.42)$$

and transfer function of a 2<sup>nd</sup>-order system

$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_ns + \omega_n^2} \quad (5.43)$$



we can expand on this to show

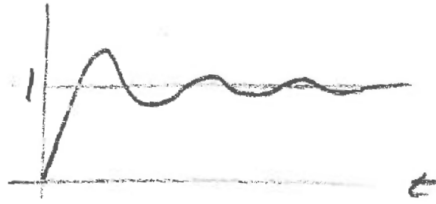
$$\begin{aligned} X(s) &= G(s)F(s) \\ &= \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} F(s) \end{aligned} \quad (5.44)$$

next, the error in the s-domain is shown to be

$$\begin{aligned} E(s) &= F(s) - X(s) \\ &= F(s) - G(s)F(s) \\ &= (1 - G(s))F(s) \\ &= \frac{s^2 + 2\zeta\omega_n s}{s^2 + 2\zeta\omega_n s + \omega_n^2} F(s) \end{aligned} \quad (5.45)$$

Again, these are general terms for a 2<sup>nd</sup>-order system. Note that the definitions for the steady-state value  $x_{ss}$  and steady-state error  $e_{ss}$  remain largely unchanged from the 1<sup>st</sup>-order system.

### 5.3.1 Step response 2<sup>nd</sup>-order system performance indicators



For a 2<sup>nd</sup>-order system subjected to a step response, we want to find  $x_{ss}$  and  $e_{ss}$ . Using the transfer function method to solve for the response, we know the system response is

$$x(t) = 1 - \frac{1}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_d t + \phi) \quad (5.46)$$

where

$$\omega_d = \omega_n \sqrt{1-\zeta^2} \quad (5.47)$$

and

$$\phi = \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta} = \sin^{-1} \sqrt{1-\zeta^2} \quad (5.48)$$

next, the steady-state displacement can be solved for as

$$\begin{aligned} x_{ss} &= \lim_{t \rightarrow \infty} x(t) \\ &= 1 - \frac{\sqrt{1-\zeta^2}}{\zeta} \lim_{t \rightarrow \infty} e^{-\zeta\omega_n t} \sin(\omega_d t + \phi) \\ &= 1 \end{aligned} \quad (5.49)$$

as  $e^{-\zeta \omega_n t}$  goes to 0. Next, the error as a function of time is

$$\begin{aligned} e(t) &= f(t) - x(t) \\ &= \frac{1}{\sqrt{1-\zeta^2}} e^{-\zeta \omega_n t} \sin(\omega_d t + \phi) \end{aligned} \quad (5.50)$$

while the steady-state error is

$$\begin{aligned} e_{ss} &= \lim_{t \rightarrow \infty} e(t) \\ &= \frac{\sqrt{1-\zeta^2}}{\zeta} \lim_{t \rightarrow \infty} e^{-\zeta \omega_n t} \sin(\omega_d t + \phi) \\ &= 0 \end{aligned} \quad (5.51)$$

as  $e^{-\zeta \omega_n t}$  goes to 0.

These same solutions can be found in the s-domain where the step function is  $F(s) = \frac{1}{s}$ . Consider the s-domain expression solved for  $X(s)$ ,

$$X(s) = \frac{\omega_n^2}{s^2 + 2\zeta \omega_n s + \omega_n^2} \cdot \frac{1}{s} \quad (5.52)$$

the steady-state displacement is shown to be

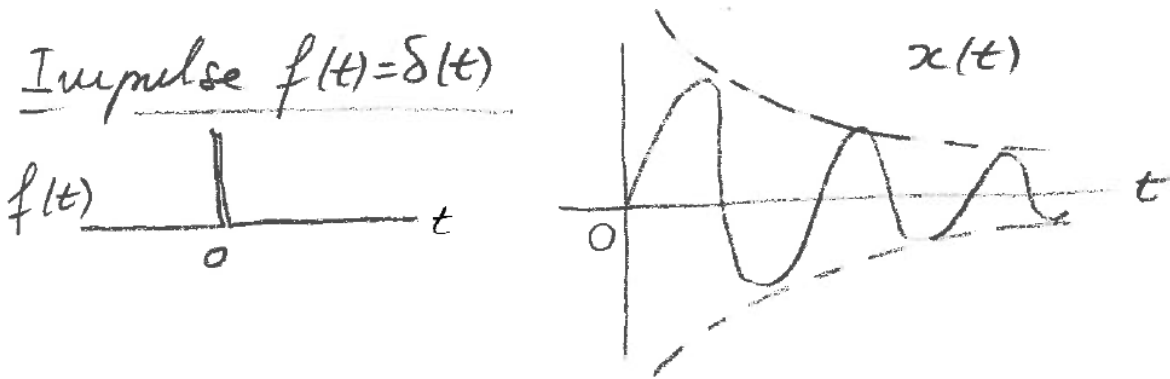
$$\begin{aligned} x_{ss} &= \lim_{s \rightarrow 0} s \frac{\omega_n^2}{s^2 + 2\zeta \omega_n s + \omega_n^2} \cdot \frac{1}{s} \\ &= \frac{\omega_n^2}{\omega_n^2} \\ &= 1 \end{aligned} \quad (5.53)$$

Next, we can build the s-domain representation of the error as

$$E(s) = \frac{s^2 + 2\zeta \omega_n s}{s^2 + 2\zeta \omega_n s + \omega_n^2} \cdot \frac{1}{s} \quad (5.54)$$

solving for the steady-state error results in

$$\begin{aligned} e_{ss} &= \lim_{s \rightarrow 0} s \frac{s^2 + 2\zeta \omega_n s}{s^2 + 2\zeta \omega_n s + \omega_n^2} \cdot \frac{1}{s} \\ &= \frac{0}{\omega_n} \\ &= 0 \end{aligned} \quad (5.55)$$

5.3.2 Impulse response 2<sup>nd</sup>-order system performance indicators

For a 2<sup>nd</sup>-order system subjected to a impulse response, we want to find  $x_{ss}$  and  $e_{ss}$ . The impulse function is defined as

$$f(t) = \delta(t) \quad (5.56)$$

therefore, the response is

$$x(t) = \frac{\omega_n}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_d t) \quad (5.57)$$

where the steady-state response is

$$\begin{aligned} x_{ss} &= \lim_{t \rightarrow \infty} x(t) \\ &= \frac{\omega_n}{\sqrt{1-\zeta^2}} \lim_{t \rightarrow \infty} e^{-\zeta\omega_n t} \sin(\omega_d t + \phi) \\ &= 0 \end{aligned} \quad (5.58)$$

as  $e^{-\zeta\omega_n t}$  goes to 0. Next, the error as a function of time is

$$\begin{aligned} e(t) &= f(t) - x(t) \\ &= \delta(t) - \frac{\omega_n}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_d t + \phi) \end{aligned} \quad (5.59)$$

Lastly, the steady-state error is

$$\begin{aligned} e_{ss} &= \lim_{t \rightarrow \infty} e(t) \\ &= \lim_{t \rightarrow \infty} \delta(t) - \frac{\omega_n}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_d t + \phi) \\ &= 0 \end{aligned} \quad (5.60)$$

as  $\delta(t)$  and  $e^{-\zeta\omega_n t}$  both go to 0.

These same solutions can be found in the s-domain where the impulse function is  $F(s) = 1$ . Consider the s-domain expression solved for  $X(s)$ ,

$$X(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \quad (5.61)$$

the steady-state error is shown to be

$$\begin{aligned} x_{ss} &= \lim_{s \rightarrow 0} s \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \\ &= 0 \end{aligned} \quad (5.62)$$

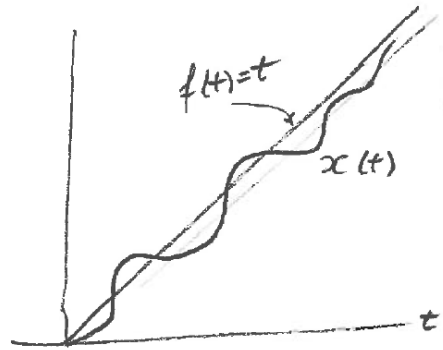
The s-domain error is expressed as

$$E(s) = \frac{s^2 + 2\zeta\omega_n s}{s^2 + 2\zeta\omega_n s + \omega_n^2} \quad (5.63)$$

which leads to the steady-state error value

$$\begin{aligned} e_{ss} &= \lim_{s \rightarrow 0} s \frac{s^2 + 2\zeta\omega_n s}{s^2 + 2\zeta\omega_n s + \omega_n^2} \\ &= 0 \end{aligned} \quad (5.64)$$

### 5.3.3 Ramp response 2<sup>nd</sup>-order system performance indicators



For a 2<sup>nd</sup>-order system subjected to a ramp response, we want to find  $x_{ss}$  and  $e_{ss}$ . The impulse function is defined as

$$f(t) = t \quad (5.65)$$

therefore, the response can be found through the transfer function approach as

$$x(t) = t - \frac{2\zeta}{\omega_n} \left[ 1 + \frac{1}{\sin \phi_1} e^{-\zeta\omega_n t} \sin(\omega_d t + \phi_1) \right] \quad (5.66)$$

where

$$\phi_1 = \tan^{-1} \frac{2\zeta\sqrt{1-\zeta^2}}{1-2\zeta^2} \quad (5.67)$$

which leads to the response

$$\begin{aligned} x_{ss} &= \lim_{t \rightarrow \infty} x(t) \\ &= t - \frac{2\zeta}{\omega_n} \\ &= \infty \end{aligned} \quad (5.68)$$

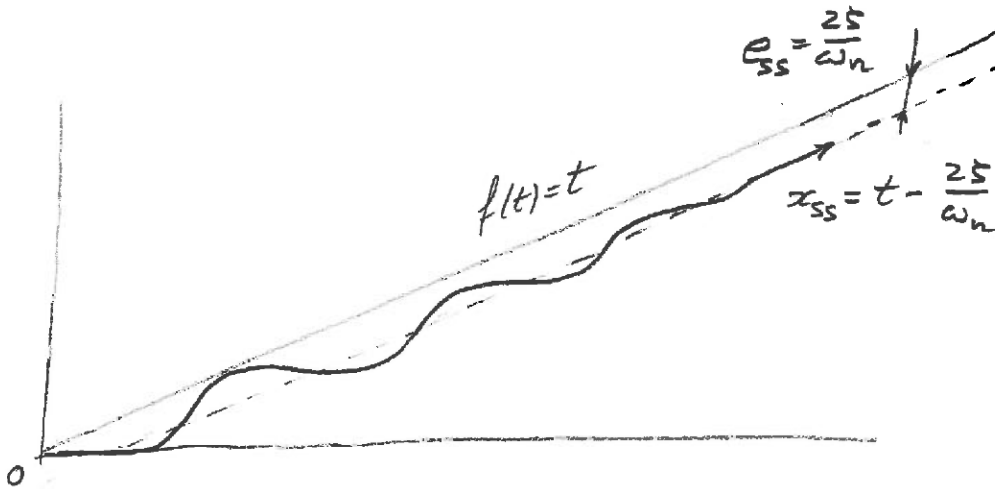
which means that there is no steady-state value. Next, the error as a function of time is

$$\begin{aligned} e(t) &= f(t) - x(t) \\ &= t - \left[ t - \frac{2\zeta}{\omega_n} \left[ 1 + \frac{1}{\sin \phi_1} e^{-\zeta \omega_n t} \sin(\omega_d t + \phi_1) \right] \right] \\ &= \frac{2\zeta}{\omega_n} \left[ 1 + \frac{1}{\sin \phi_1} e^{-\zeta \omega_n t} \sin(\omega_d t + \phi_1) \right] \end{aligned} \quad (5.69)$$

Lastly, the steady-state error is

$$\begin{aligned} e_{ss} &= \lim_{t \rightarrow \infty} e(t) \\ &= \frac{2\zeta}{\omega_n} \lim_{t \rightarrow \infty} \left[ 1 + \frac{1}{\sin \phi_1} e^{-\zeta \omega_n t} \sin(\omega_d t + \phi_1) \right] \\ &= \frac{2\zeta}{\omega_n} \end{aligned} \quad (5.70)$$

as the term in the brackets goes to 1.



These same solutions can be found in the s-domain where the ramp function is  $F(s) = \frac{1}{s^2}$ . Consider the s-domain expression solved for  $X(s)$ ,

$$X(s) = \frac{\omega_n^2}{s^2 + 2\zeta \omega_n s + \omega_n^2} \cdot \frac{1}{s^2} \quad (5.71)$$

the steady-state error is shown to be

$$\begin{aligned}
 x_{ss} &= \lim_{s \rightarrow 0} s \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot \frac{1}{s^2} \\
 &= \lim_{s \rightarrow 0} \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot \frac{1}{s} \\
 &= \lim_{s \rightarrow 0} \frac{1}{s} \\
 &= \infty
 \end{aligned} \tag{5.72}$$

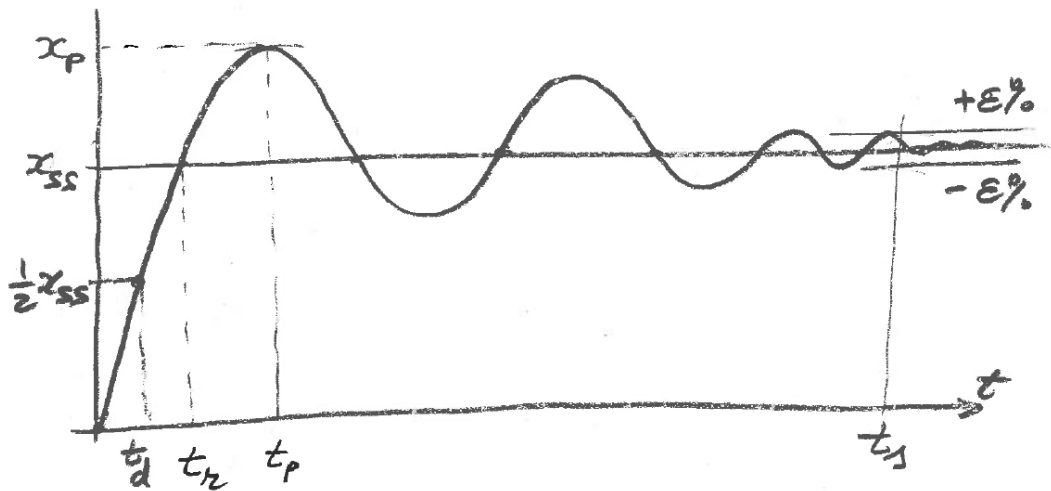
therefore, there is no steady-state value. The s-domain error is expressed as

$$E(s) = \frac{s^2 + 2\zeta\omega_n s}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot \frac{1}{s^2} \tag{5.73}$$

which leads to the steady-state error value

$$\begin{aligned}
 e_{ss} &= \lim_{s \rightarrow 0} s \frac{s^2 + 2\zeta\omega_n s}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot \frac{1}{s^2} \\
 &= \lim_{s \rightarrow 0} \frac{s^2(s + 2\zeta\omega_n)}{s^2 + 2\zeta\omega_n s + \omega_n^2} \cdot \frac{1}{s^2} \\
 &= \frac{2\zeta\omega_n}{\omega_n^2} \\
 &= \frac{2\zeta}{\omega_n}
 \end{aligned} \tag{5.74}$$

## 5.4 2<sup>nd</sup>-order System Specific Performance Indicators



Specific performance indicators exist. They depend on system order and excitation type. Examples are:

- rise time  $\rightarrow t_r$
- peak time  $\rightarrow t_p$
- peak value  $\rightarrow x_p$
- settling time  $\rightarrow t_s$
- delay time  $\rightarrow t_d$
- max percentage overshoot  $\rightarrow M_p$

Many of these are the same as a 1<sup>st</sup>-order system or taken directly from the system response. The max percentage overshoot ( $M_p$ ) is defined as

$$M_p = \left( \frac{x_p}{x_{ss}} - 1 \right) \cdot 100 \quad (5.75)$$

### 5.4.1 Procedure for a Step Response

Given the 2<sup>nd</sup>-order system, with the transfer function

$$G(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \quad (5.76)$$

where  $\omega_n$  is the natural frequency and  $\zeta$  is the critical damping ratio. Considering that the system is subjected to a step response, we know the system response is

$$x(t) = 1 - \frac{1}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_d t + \phi) \quad (5.77)$$

where we can calculate:

$$\omega_d = \omega_n \sqrt{1-\zeta^2} \quad (5.78)$$

and

$$\phi = \tan^{-1} \frac{\sqrt{1-\zeta^2}}{\zeta} = \sin^{-1} \sqrt{1-\zeta^2} \quad (5.79)$$

Next, compute the values for the performance indicators.

First the rise time is calculated by the fact that  $x(0) = 0$  and  $x(t_r) = 1$ . From equation 5.77, the mean height of the system is shown to be when  $\sin(\omega_d t + \phi) = 0$ , therefore, as  $\sin(\pi) = 0$ , we can show that  $\omega_d t + \phi = \pi$ , rearranging this yields

$$t_r = \frac{\pi - \phi}{\omega_d} \quad (5.80)$$

To find the peak time ( $t_p$ ), we need to find the peak, which is defined as  $\frac{dx}{dt} \big|_{t=t_p} = 0$ , or drawn as



where we consider only the decaying part of the signal caused by the step function, or  $x = e^{-\zeta \omega_n t} \sin(\omega_d t + \phi)$ . Therefore

$$\begin{aligned}
 0 &= \frac{dx}{dt} \\
 &= \frac{d}{dt} \left[ e^{-\zeta \omega_n t} \sin(\omega_d t + \phi) \right] \\
 &= -\zeta \omega_n e^{-\zeta \omega_n t} \sin(\omega_d t + \phi) + \omega_d e^{-\zeta \omega_n t} \cos(\omega_d t + \phi) \\
 \zeta \omega_n \sin(\omega_d t + \phi) &= \omega_d \cos(\omega_d t + \phi) \\
 \frac{\sin(\omega_d t + \phi)}{\cos(\omega_d t + \phi)} &= \frac{\omega_d}{\zeta \omega_n}
 \end{aligned} \tag{5.81}$$

By converting the left hand side of this equation to  $\tan(\omega_d t_p + \phi)$  yields

$$\begin{aligned}
 \tan(\omega_d t_p + \phi) &= \frac{\omega_d}{\zeta \omega_n} \\
 &= \frac{\sqrt{1 - \zeta^2}}{\zeta} \\
 &= \tan(\phi)
 \end{aligned} \tag{5.82}$$

when considering the definition of  $\tan$  provided by equation 5.79. We must find  $t_p$  such that  $\tan(\omega_d t_p + \phi) = \tan(\phi)$ . Given that the function  $\tan(\alpha)$  repeats itself after  $\pi$ ,  $2\pi$ ,  $3\pi$ ,  $\dots$  as shown in figure 5.1, we know  $\tan(\alpha) = \tan(\alpha + \pi) = \tan(\alpha + 2\pi), \dots$ .

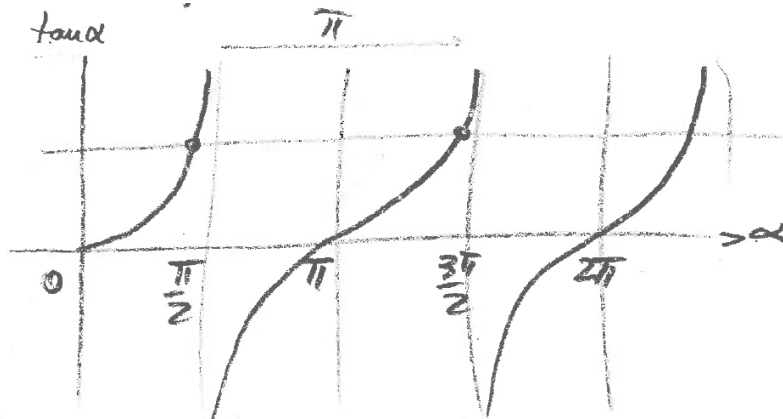


Figure 5.1: Plotting the  $\tan(\alpha)$

Hence, picking back up from equation 5.82 lead to

$$\begin{aligned}
 \omega_d t_p + \phi &= \phi + \pi \\
 \omega_d t_p &= \pi
 \end{aligned} \tag{5.83}$$



which results in

$$t_p = \frac{\pi}{\omega_d} \quad (5.84)$$

The peak value ( $x_p$ ) can be found as

$$\begin{aligned} x_p &= x(t_p) \\ &= 1 - \frac{1}{\sqrt{1-\zeta^2}} e^{-\zeta \omega_n \frac{\pi}{\omega_d}} \sin(\omega_d(\pi/\omega_d) + \phi) \\ &= 1 + \frac{1}{\sqrt{1-\zeta^2}} e^{-\zeta \omega_n \frac{\pi}{\omega_d}} \sin(\phi) \\ &= 1 + \frac{1}{\sqrt{1-\zeta^2}} \left( e^{-\frac{\zeta}{\sqrt{1-\zeta^2}} \pi} \right) \sqrt{1-\zeta^2} \\ &= 1 + e^{-\frac{\zeta}{\sqrt{1-\zeta^2}} \pi} \end{aligned} \quad (5.85)$$

when considering that  $\sin(\phi + \pi) = -\sin(\phi)$  and  $\sin(\phi) = \sqrt{1-\zeta^2}$ .

The max overshoot ( $M_p$ ) can be found as

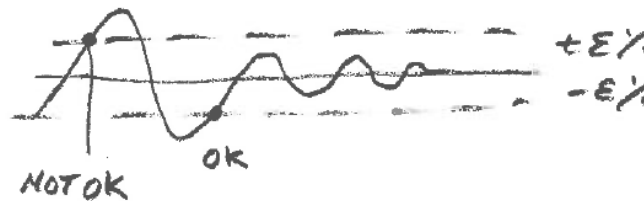
$$\begin{aligned} M_p &= \frac{x_p - x_{ss}}{x_{ss}} \\ &= \frac{1 + e^{-\frac{\zeta}{\sqrt{1-\zeta^2}} \pi} - 1}{1} \\ &= e^{-\frac{\zeta}{\sqrt{1-\zeta^2}} \pi} \end{aligned} \quad (5.86)$$

for a few select damping ratios, the overshoot percentages are shown in Table 1. However, the typical range of damping is  $0.4 < \zeta < 0.8$  and therefore the typical max overshoot is  $0.25\% < M_p < 1.5\%$ .

Table 1: Overshoot percentages for select damping ratios.

$\zeta$	0	0.2	0.4	0.6	0.8	1
$M_p$	100.00%	52.68%	25.40%	9.49%	1.52%	0.00%

The definition of settling time ( $t_s$ ) is to “get withing  $\pm\epsilon\%$  of  $x_{ss}$  and stay so”.



The settling time is defined as the time when both

$$|x_{ss} - x(t_s)| < \Delta \quad (5.87)$$

and

$$|x_{ss} - x(t > t_s)| < \Delta \quad (5.88)$$

are true where  $\Delta = \varepsilon \cdot x_{ss}$ . This is shown in figure 5.2.

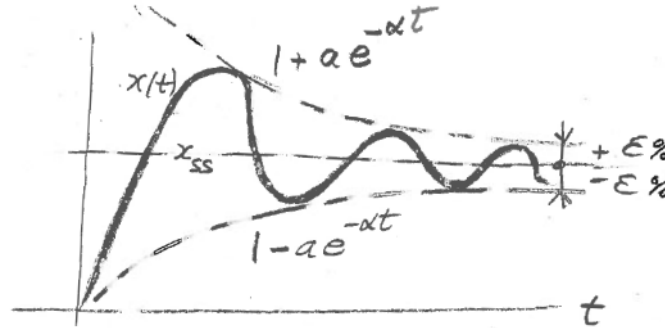


Figure 5.2: Settling time for a 2<sup>nd</sup>-order system.

We can find a generalized expression for  $t_s$  if we consider the system response for  $x_{ss} = 1$  as

$$x(t) = 1 - \left[ \frac{1}{\sqrt{1 - \zeta^2}} \right] e^{-[\zeta \omega_n]t} \sin(\omega_d t + \phi) \quad (5.89)$$

redefining the items in the brackets as  $a$  and  $\alpha$ , respectively, the system response can be simplified to

$$x(t) = 1 - ae^{-\alpha t} \sin(\omega_d t + \phi) \quad (5.90)$$

Therefore, the envelop of the system response is  $1 \pm ae^{-\alpha t}$  while the settling condition is  $ae^{-\alpha t} = \Delta$ . Considering that the final peak above the error range will happen when  $a \cong 1$ , we make the approximate calculation

$$a = \frac{1}{\sqrt{1 - \zeta^2}} \Big|_{\zeta < 1} \cong 1 \quad (5.91)$$

and considering that  $t_s$  is when the system decays under the value  $\varepsilon$ , we need to find the  $t$  value when

$$e^{-\alpha t} = \varepsilon \quad (5.92)$$

therefore, setting  $\varepsilon = 2\%$ , we can find

$$e^{-\alpha t} = 0.02 \quad (5.93)$$

$$-\alpha t = \log(0.02) \quad (5.94)$$

$$= -3.9$$

$$\cong -4$$

Therefore, knowing that  $\alpha = \zeta \omega_n$ , we can deduce

$$-\alpha t_s \cong -4 \quad (5.95)$$

$$-\zeta \omega_n t_s \cong -4$$

$$t_s \cong \frac{4}{\zeta \omega_n}$$

when  $\zeta \ll 1$  this simplified expression is within  $\pm 2\%$ .

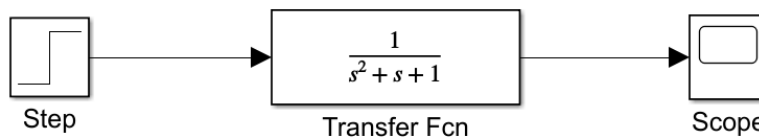
Importantly, one should always consider the effects of  $\zeta$  and  $\omega_n$  on performance.

- An increase in  $\omega_n$  shortens rise time ( $t_r$ ), peak time ( $t_p$ ), and settling time ( $t_s$ ).
- An increase in  $\zeta$  reduces max overshoot ( $M_p$ ) and shortens settling time ( $t_s$ ).

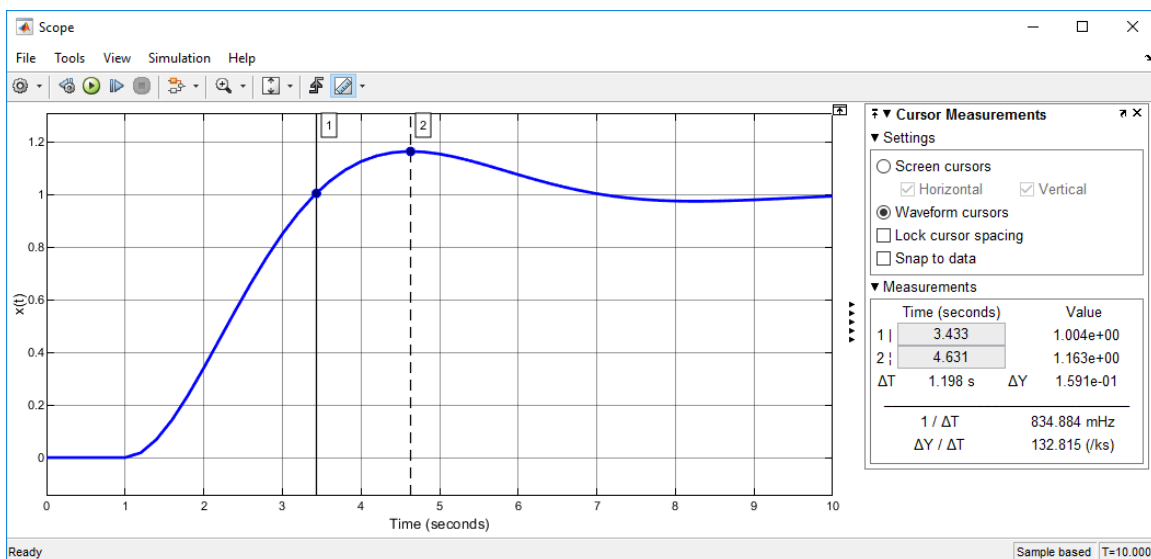
As before, the definition of time delay ( $t_d$ ) for a 2<sup>nd</sup>-order system is the time it takes to rise to 50% of  $x_{ss}$  the first time. Similarly, the max percentage overshoot ( $M_p$ ) for a 2<sup>nd</sup>-order system is calculated in the same way as a 1<sup>st</sup>-order system.

### Example 5.2 SIMULINK Tutorial on Measuring Performance Indicators

Build the simple 2<sup>nd</sup>-order system as shown below



Open the scope and press the 'Cursor Measurements' button to activate the cursors.



Use the first cursor to find the first crossing of  $x_{ss} = 1$ . Read the time as  $t_r = 2.433$  sec as the step function starts at 1 sec. Place the second cursor at peak value. Read the peak time  $t_p = 3.631$  sec and peak amplitude  $x_p = 1.163$ . Calculate  $M_p = 16.3\%$ .