

Kinematic design of a humanoid robotic shoulder complex

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Abstract

In this paper, we present a mechanical design for a humanoid robotic shoulder complex. The aim of the mechanism is to support the load of the arm and to copy four principal motions of the human shoulder in order to furnish the desired arm mobility and reachability. We utilize a fully parallel mechanism with four driven legs that provide to the movable platform (on which the arm is to be attached) three rotations and one translation. These three rotations represent the shoulder's flexion-extension (about the vertical axis), the shoulder's abduction-adduction (about the anterior-posterior axis), and the shoulder's rotation (about the medial-lateral axis). The fourth degree of freedom is controlled depending on the angles of the first two rotations and represents the shoulder contraction and enlargement along the medial-lateral axis.

1 Introduction

Today's robotic shoulders incorporate different types of spherical or universal joints representing the function of glenohumeral articulation [1,2]. In humans, however, the shoulder complex is an assemblage of three articulations, which are the sternoclavicular joint, the scapulothoracic joint, and the glenohumeral joint. The first two articulations connect the shoulder to the body, while the third one connects the upper arm to the shoulder. They are driven by a number of coupled actuators. Even though the first two joints possess

relatively small ranges of motion if compared to the glenohumeral joint and other joints of the arm, their contribution to the arm performances is extremely important. The shoulder complex radically increases the arm's mobility and reachability by the use of the sternoclavicular and scapulothoracic degrees of freedom. This role of the sternoclavicular and the scapulothoracic joint in the human arm movement has been demonstrated in our previous works [3,4]. Recent investigations showed that they also enlarge the self-motion ability and the amount of positioning redundancy of the arm [5]. We can speculate that they additionally contribute to the arm's adaptability to adjust the manipulability, isotropy, compliance and other performance indices respecting the task requirements.

The objective of this article is to describe a design of a humanoid robotic shoulder mechanism simulating the function of the sternoclavicular and the scapulothoracic joint. In this work, we first mathematically formulate the kinematical properties of the human shoulder when modeled as a serial mechanism. In our design, however, we utilize a fully parallel mechanism with four driven extendable legs connecting a fixed base and a movable platform. It is understood that the base is attached to the body, while the upper arm is connected to the movable platform by a glenohumeral joint. The platform has four degrees of freedom; three are rotational and represent the shoulder's flexion-extension, the shoulder's abduction-adduction, and the shoulder's rotation. The fourth is a translation that can be associated with the shoulder's contraction along the clavicular axis that occurs during the arm elevation.

2 Principal kinematic aspects of the human shoulder complex

The human shoulder complex is made of a clavicle and scapula [1]. The clavicle is on one side attached to the sterna (anterior part of the body) by a spherical-type sternoclavicular joint, and on the other side to the scapula. The planar-type scapulothoracic joint enables the motion of scapula. The upper arm (humerus) is attached at the lateral end of the scapula by the spherical-type glenohumeral joint. In a first approximation [3], we can represent the shoulder complex as shown in Fig. 1. In this approximation, the inner articulation is concentrated in point S and simulates the sternoclavicular joint, the outer articulation is concentrated in point G and associated with the glenohumeral joint. The translation along the shoulder link represents the shoulder's enlargement. This is dependent on the inclination in the articulation about point S. The shoulder link between points S and G simulates the clavicle. The upper arm link is between points G and the elbow center E.

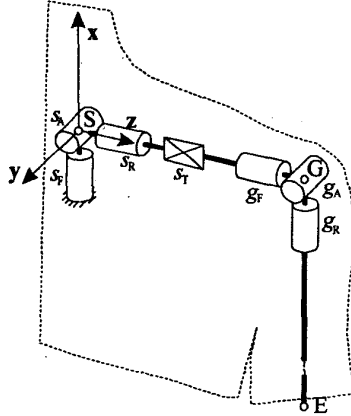


Fig. 1: A serial model of the human shoulder complex

In Fig 1, angles s_F, s_A, s_R represent the shoulder flexion-extension, the shoulder abduction-adduction, and the shoulder rotation, while angles g_F, g_A, g_R are the upper arm flexion-extension, the upper arm abduction-adduction, and the upper arm rotation, respectively. Linear coordinate s_T is the shoulder translation. We assume that all coordinates are zero, if the mechanism's position is given as that in Fig. 1. Note that the listed joint coordinates and their upper and lower limits are coupled. Their interdependency is very complex and it is difficult

to formulate it with precise mathematical functions. Some attempts, however, can be found in the literature. The dependency of the clavicular abduction as function of the humeral elevation in the frontal plane was reported in [6]. The following simplified relationships were synthesized based on optical measurements of the human arm in the frontal (plane x-z), horizontal (plane y-z), and sagittal (plane x-y). They represent an idealistic status and serve as inputs for the mechanical design.

The frontal plane: We assumed that all rotations in the mechanism are zero except the abduction-adduction angles s_A and g_A . We observed that the lower limit of angle s_A depends on the angle of the humeral elevation e_A as shown in Fig.2. Its minimum value is zero when the humeral elevation is zero.

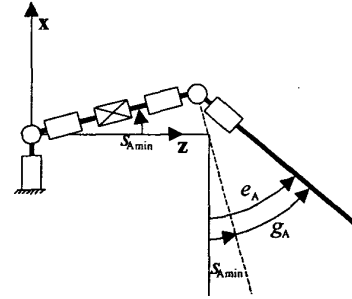


Fig. 2: The shoulder motion in the frontal plane

The lower limit of angle g_A is dependent on the current value of angle s_A , while the shoulder expansion depends on the shoulder abduction angle as follows

$$s_T = \left(\frac{4}{3} + \frac{6}{3} \frac{s_A}{\pi} \right) s_{T0} . \quad (1)$$

Note that all angles are measured in radians. Here, s_{T0} is the length of the shoulder link when the shoulder abduction is zero. The shoulder link shortens by increasing s_A and is about 80% of the initial value when s_A reaches its upper limit. The lower and the upper limits of the moving coordinates are listed in Tb. 1.

Coordinate	Lower limit	Upper limit
e_A	0	π
s_A	$\frac{1}{6} e_A$	$\frac{1}{6} \pi$
g_A	$-s_A$	$\frac{5}{6} \pi$
$s_T = f(s_A)$	$\frac{4}{3} s_{T0}$	s_{T0}

Tb. 1: The joint limits corresponding to the shoulder movements in the frontal plane

The horizontal plane: We assumed that the changing rotations are the shoulder flexion-extension s_F and the upper arm rotation g_R . The lower and the upper limit of both angles are independent, but the shoulder translation s_T is a function of the rotation angle s_F

$$s_T = \left(\frac{4}{3} + \frac{6}{3} \frac{s_F}{\pi}\right) s_{T0} . \quad (2)$$

The lower and the upper joint limits of the measured coordinates are listed in Tb. 2.

Coordinate	Lower limit	Upper limit
s_F	$-\frac{1}{6}\pi$	0
g_R	$-\frac{1}{2}\pi$	$\frac{1}{6}\pi$
$s_T = f(s_F)$	$\frac{4}{3}s_{T0}$	s_{T0}

Tb. 2: The joint limits corresponding to the shoulder movements in the horizontal plane

The sagittal plane: The changing rotations were the shoulder rotation s_R and the upper arm flexion-extension g_F . Their relationship appeared to be linear

$$s_R = \frac{1}{9} g_F . \quad (3)$$

The joint limits are shown in Tb. 3.

Coordinate	Lower limit	Upper limit
s_R	$-\frac{1}{10}\pi$	$\frac{1}{30}\pi$
g_F	$-\frac{9}{10}\pi$	$\frac{9}{30}\pi$

Tb. 3: The joint limits corresponding to the shoulder movements in the sagittal plane

The glenohumeral joint provides a great majority of the arm's motion ability. The range of motion in the sternoclavicular joint is only a few degrees. Nevertheless, the sternoclavicular joint plays an important role in the reachability of the arm. It serves for collision avoidance between the arm links and the body. If we fix the sternoclavicular joint, the motion of the arm will be troubled by collisions with the torso.

3 The humanoid shoulder designed by a fully parallel mechanism

The shoulder mechanism (the sternoclavicular joint and the shoulder link) has to support and orient the arm.

Therefore, a serial mechanism in Fig. 1 may not be the best choice. Instead, we propose in this section a fully parallel mechanism, which can provide higher stiffness and more precise orientation [7]. It is understood that the base is fixed and corresponds to the body, while the movable platform supports the glenohumeral joint. The main kinematic parameters of this mechanism are illustrated in Fig. 3.

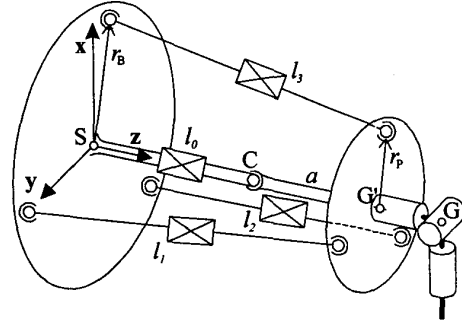


Fig. 3: The shoulder modeled with a parallel mechanism

The mechanism possesses four extendable legs. The translational joints are driven and the lengths l_1, l_2, l_3, l_0 are the joint coordinates. The three outer legs l_1, l_2, l_3 are attached to the base plane and to the platform plane via passive spherical joints. There is also a passive spherical joint C located in the central leg, which lies at a distance a from the platform plane.

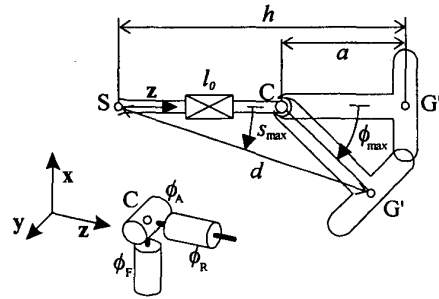


Fig. 4: A side view of the platform's inclination

The basic structure of the mechanism is symmetrical. The attachments of the outer legs are positioned on a circle of radius r_B on the base side and on a circle of a radius r_p on the platform side. The platform has four degrees of freedom. It can be placed in a desired orientation and expanded. The last, however, is intended to be dependent (via an imposed control law) in the sense

of coordinate s_T in the serial mechanism (1). The spatial orientating of the platform is realized about the central spherical joint C.

Fig. 4 shows a longitudinal section of the mechanism in its maximum inclination. Here, h is the size of the shoulder and will be used from now on as a unit to represent all distances. It is assumed that the maximum inclination angle s_{\max} is symmetric about axis z . Its value results from the ranges of angles s_A and s_F . According to the range of translation s_T , we can also deduce length d . Ratio ϕ_{\max}/s_{\max} does not have a straightforward anatomical explanation. A higher ϕ_{\max} introduces better orientation capabilities for the arm, a higher s_{\max} assures avoidance with the body. Tb. 4 lists the chosen values.

s_{\max}	ϕ_{\max}	d	r_B	r_P
$\frac{1}{12}\pi$	$\frac{21}{120}\pi$	$\frac{4}{5}h$	$\frac{8}{17}h$	$\frac{4}{17}h$

Tb. 4: Input parameters of the parallel mechanism

When $\phi = 0 \Rightarrow l_0 = l_{0\max}$ and $\phi = \phi_{\max} \Rightarrow l_0 = l_{0\min}$, the following is valid

$$a = \frac{4}{5}h \frac{\sin s_{\max}}{\sin \phi_{\max}}, \quad (4)$$

$$l_{0\min} = \frac{4}{5}h \frac{\sin(\phi_{\max} - s_{\max})}{\sin \phi_{\max}}, \quad (5)$$

$$l_{0\max} = h - a. \quad (6)$$

From a mechanical viewpoint, it is desirable to implement a consistent overlapping in the actuator of the central leg (minimize $l_{0\max} - l_{0\min}$).

4 Kinematic analysis and sensitivity

Let vector \mathbf{l}_0 be associated with the central leg. We can compute the vectors of outer legs by

$$\mathbf{l}_i = \mathbf{l}_0 + \mathbf{T}_F \mathbf{T}_A \mathbf{T}_R \mathbf{p}_i - \mathbf{b}_i, \text{ where } \mathbf{l}_0 = (0, 0, l_0)^T \quad (7)$$

and $i = 1, 2, 3$. Here, the following attachment points are introduced

$$\mathbf{b}_1 = (0, r_B, 0)^T, \quad \mathbf{p}_1 = (0, r_P, a)^T, \quad (8)$$

$$\mathbf{b}_2 = (-\frac{\sqrt{3}}{2}r_B, -\frac{1}{2}r_B, 0)^T, \quad \mathbf{p}_2 = (-\frac{\sqrt{3}}{2}r_P, -\frac{1}{2}r_P, a)^T, \quad (9)$$

$$\mathbf{b}_3 = (\frac{\sqrt{3}}{2}r_B, -\frac{1}{2}r_B, 0)^T, \quad \mathbf{p}_3 = (\frac{\sqrt{3}}{2}r_P, -\frac{1}{2}r_P, a)^T. \quad (10)$$

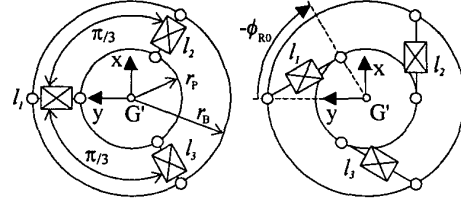


Fig. 5: The upper view of the leg attachment points

Transformation matrices $\mathbf{T}_F, \mathbf{T}_A, \mathbf{T}_R$ are associated with rotation angles ϕ_F, ϕ_A, ϕ_R about point C as shown in Fig. 4). These angles are the mechanism's external (Cartesian) coordinates

$$\mathbf{T}_F = \begin{bmatrix} 1 & 0 & 0 \\ 0 & c_F & -s_F \\ 0 & s_F & c_F \end{bmatrix}, \quad \mathbf{T}_A = \begin{bmatrix} c_A & 0 & s_A \\ 0 & 1 & 0 \\ -s_A & 0 & c_A \end{bmatrix},$$

$$\mathbf{T}_R = \begin{bmatrix} c_R & -s_R & 0 \\ s_R & c_R & 0 \\ 0 & 0 & 1 \end{bmatrix}, \quad (11)$$

where we mark $s_x = \sin \phi_x$ and $c_x = \cos \phi_x$. The lengths of legs $i = 1, 2, 3$ are the independent internal (joint) coordinates

$$l_i = \sqrt{\mathbf{l}_i^T \mathbf{l}_i}. \quad (12)$$

The length of the central leg depends on the inclination angle ϕ . In order to achieve a smooth motion through the initial position we impose the following dependency

$$l_0 = l_{0\min} + (l_{0\max} - l_{0\min}) \cos\left(\frac{1}{2}\pi \frac{\phi}{\phi_{\max}}\right), \quad (13)$$

$$\phi = \arccos(c_F c_A). \quad (14)$$

The differential relationship of (7) can be expressed by

$$d\mathbf{l} = d \begin{bmatrix} l_1 \\ l_2 \\ l_3 \end{bmatrix} = \begin{bmatrix} \frac{\partial l_1}{\partial \phi_F} & \frac{\partial l_1}{\partial \phi_A} & \frac{\partial l_1}{\partial \phi_R} \\ \frac{\partial l_2}{\partial \phi_F} & \frac{\partial l_2}{\partial \phi_A} & \frac{\partial l_2}{\partial \phi_R} \\ \frac{\partial l_3}{\partial \phi_F} & \frac{\partial l_3}{\partial \phi_A} & \frac{\partial l_3}{\partial \phi_R} \end{bmatrix} d \begin{bmatrix} \phi_F \\ \phi_A \\ \phi_R \end{bmatrix} = \mathbf{J} d \begin{bmatrix} \phi_F \\ \phi_A \\ \phi_R \end{bmatrix}, \quad (15)$$

$$\frac{\partial l_i}{\partial \phi_F} = \frac{\mathbf{l}_i^T}{\sqrt{\mathbf{l}_i^T \mathbf{l}_i}} \frac{\partial \mathbf{l}_i}{\partial \phi_F}, \quad \frac{\partial l_i}{\partial \phi_F} = \frac{\partial \mathbf{T}_F}{\partial \phi_F} \mathbf{T}_A \mathbf{T}_R \mathbf{p}_i, \quad (16)$$

$$\frac{\partial l_i}{\partial \phi_A} = \frac{\mathbf{l}_i^T}{\sqrt{\mathbf{l}_i^T \mathbf{l}_i}} \frac{\partial \mathbf{l}_i}{\partial \phi_A}, \quad \frac{\partial l_i}{\partial \phi_A} = \mathbf{T}_F \frac{\partial \mathbf{T}_A}{\partial \phi_A} \mathbf{T}_R \mathbf{p}_i, \quad (17)$$

$$\frac{\partial l_i}{\partial \phi_R} = \frac{\mathbf{l}_i^T \frac{\partial \mathbf{l}_i}{\partial \phi_R}}{\sqrt{\mathbf{l}_i^T \mathbf{l}_i}}, \quad \frac{\partial l_i}{\partial \phi_R} = \mathbf{T}_F \mathbf{T}_A \frac{\partial \mathbf{T}_R}{\partial \phi_R} \mathbf{p}_i, \quad (18)$$

$i = 1, 2, 3$. The angular velocity of the platform and the linear velocities in the legs are related by

$$\omega = \begin{bmatrix} 1 & 0 & s_A \\ 0 & c_F & -s_F c_A \\ 0 & s_F & c_F c_A \end{bmatrix} \mathbf{J}^{-1} \frac{d\mathbf{l}}{dt}. \quad (19)$$

Matrix \mathbf{J} is singular when $\phi_F, \phi_A, \phi_R = 0$. In this singularity, the mechanism cannot produce any resistance to a moment about axis ϕ_R . To avoid the singularity, we introduced a starting angle ϕ_{R0} and analyzed throughout the whole workspace the following norms

$$\delta_F = \sqrt{\frac{d\mathbf{l}^T}{d\phi_F} \frac{d\mathbf{l}}{d\phi_F}}, \quad d\phi_A, d\phi_R = 0, \quad (20)$$

$$\delta_A = \sqrt{\frac{d\mathbf{l}^T}{d\phi_A} \frac{d\mathbf{l}}{d\phi_A}}, \quad d\phi_F, d\phi_R = 0, \quad (21)$$

$$\delta_R = \sqrt{\frac{d\mathbf{l}^T}{d\phi_R} \frac{d\mathbf{l}}{d\phi_R}}, \quad d\phi_F, d\phi_A = 0. \quad (22)$$

The smallest values of these norms for all orientations ϕ_F, ϕ_A, ϕ_R in the workspace expressed in terms of $[h/\text{rad}]$, where h is the size of the mechanism, obtained for different starting angles $\phi_{R0} = (-\frac{1}{2}\pi, 0)$ (given in [rad]) are plotted in Fig. 6. It is evident that norms δ_F and δ_A decrease by decreasing angle ϕ_{R0} , while norm δ_R increases (the maximum is at about $\phi_{R0} = -\frac{1}{8}\pi$).

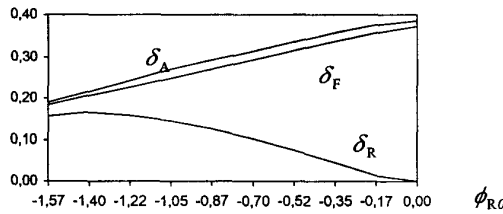


Fig. 6: Norms (20-22) depending on the starting angle

5 Leg lengths and interference

Leg lengths l_i , $i = 1, 2, 3$ were computed for all platform's orientations ϕ_F, ϕ_A, ϕ_R inside the

workspace by the use of Eq. (7). Since the mechanism is symmetric, the minimum value l_{imin} and the maximum value l_{imax} are identical for all legs. They vary as functions of the starting angle ϕ_{R0} . Lengths l_{0min} and l_{0max} are constant (see Eqs. 5-6). Once these values are known, we can estimate the overlapping of leg translations by

$$o_i = 2l_{imin} - l_{imax}, \quad i = 0, \dots, 3. \quad (23)$$

Fig. 7 shows the obtained l_{imin} , l_{imax} , and o_i , $i = 1, 2, 3$ in terms of the mechanism's size h .

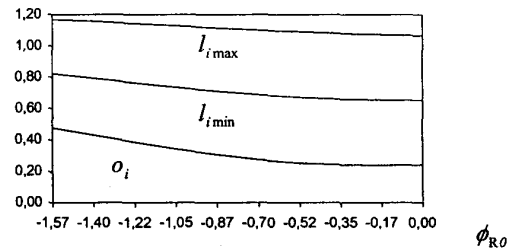


Fig. 7: Minimum/maximum leg lengths and overlapping

Possible interference (minimal distances) between legs were analyzed based on equations that formulate distance L between two lines, first with end points $\mathbf{m}_1, \mathbf{m}_2$ and second with end points $\mathbf{n}_1, \mathbf{n}_2$

$$L = \sqrt{(\mathbf{m} - \mathbf{n})^T (\mathbf{m} - \mathbf{n})}, \quad (25)$$

$$\mathbf{m} = \mathbf{m}_1 + k_m (\mathbf{m}_2 - \mathbf{m}_1), \quad (26)$$

$$\mathbf{n} = \mathbf{n}_1 + k_n (\mathbf{n}_2 - \mathbf{n}_1), \quad (27)$$

$$\begin{bmatrix} k_m \\ k_n \end{bmatrix} = \mathbf{Q}^{-1} \frac{(\mathbf{m}_1 - \mathbf{n}_1)^T (\mathbf{m}_2 - \mathbf{m}_1)}{(\mathbf{m}_1 - \mathbf{n}_1)^T (\mathbf{n}_2 - \mathbf{n}_1)}, \quad (28, 29)$$

$$\mathbf{Q} = \begin{bmatrix} -(\mathbf{m}_2 - \mathbf{m}_1)^T (\mathbf{m}_2 - \mathbf{m}_1) & (\mathbf{m}_2 - \mathbf{m}_1)^T (\mathbf{n}_2 - \mathbf{n}_1) \\ -(\mathbf{m}_2 - \mathbf{m}_1)^T (\mathbf{n}_2 - \mathbf{n}_1) & (\mathbf{n}_2 - \mathbf{n}_1)^T (\mathbf{n}_2 - \mathbf{n}_1) \end{bmatrix}.$$

To compute the distance between a point \mathbf{n} and a line with end points $\mathbf{m}_1, \mathbf{m}_2$ Eqs. (25,26) were used, where

$$k_m = \frac{(\mathbf{m}_1 - \mathbf{n})^T (\mathbf{m}_2 - \mathbf{m}_1)}{(\mathbf{m}_2 - \mathbf{m}_1)^T (\mathbf{m}_2 - \mathbf{m}_1)}. \quad (30)$$

Note that the end points of legs $i = 1, 2, 3$ are $\mathbf{b}_i, \mathbf{b}_i + \mathbf{l}_i$, of the lower part of leg 0 are 0, \mathbf{l}_0 , while of the upper part of leg 0 are \mathbf{l}_0 and $\mathbf{l}_0 + \mathbf{T}_F \mathbf{T}_A \mathbf{T}_R (0, 0, a)^T$. The minimum distances for all possible orientations inside the workspace between the outer legs L_{ij} , and between the outer legs and the central leg L_{i0} , in terms of mechanism's size h are shown in Fig. 8.

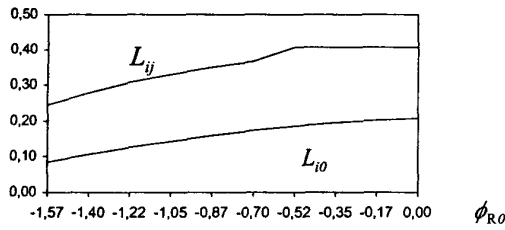


Fig. 8: Minimum distances between mechanism's legs

With respect to the dependencies in Figs. 6-8, one can select an ideal value ϕ_{R0} with the aim to avoid singularity and to simultaneously fulfill other mechanical restrictions in the shoulder's design. Our choice was $\phi_{R0} = -\frac{1}{3}\pi$. In the expected implementation of this parallel mechanism, the base will be attached to the body with an additional initial inclination (of axis z) upward and forward of about $\frac{1}{12}\pi$ radians.

Given the leg lengths l_0 and l_i , $i = 1, 2, 3$, we can compute for this mechanism the orientation angles ϕ_F , ϕ_A , ϕ_R in a closed-form [7,8]. Effective numerical solutions can also be found. Note that the direct kinematics problem is far more complex than the inverse kinematics. This mechanism possesses eight assemblies of legs of the same lengths. It means that for a given combination of the leg lengths the platform can accomplish eight different orientations ϕ_F , ϕ_A , ϕ_R . Fortunately, within the imposed orientation limits and for the selected starting angle $\phi_{R0} = -\frac{1}{3}\pi$, we can find only one solution.

A physical model of the proposed parallel shoulder mechanism was built. The size corresponds to an average size of the human shoulder ($h = 17\text{cm}$). The picture with the indicated characteristic points is shown in Fig. 9.

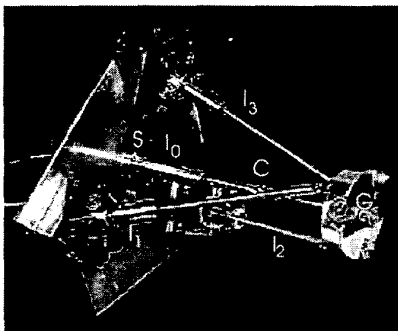


Fig. 9: Physical model of the parallel shoulder mechanism

6 Conclusions

Basic kinematic parameters of a fully parallel robotic mechanism that copies the human shoulder functions are reported. In the mechanism, four principal motions of the human shoulder were incorporated that provide the desired arm mobility and reachability. These are three rotations and one translation. The rotations represent the shoulder's flexion-extension, the shoulder's abduction-adduction, and the shoulder's rotation. The fourth degree of freedom is a translation associated with the shoulder enlargement-contraction along the medial-lateral axis. The mechanism is symmetric and doesn't suffer from singularities or leg collisions inside the imposed workspace.

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