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# 1 Introduction

Robustness of a system is defined by its ability to be insensitive to parameter variations and unwanted exogenous signals. For a control system, these variations could either be plant disturbances and parameter variations within the plant or these could be due to the sensor noises in measurement. Also, at times it is not possible to model the system dynamics accurately. These unmodeled plant dynamics may also affect the system in undersired manner.(See the block diagram in Figure 1).

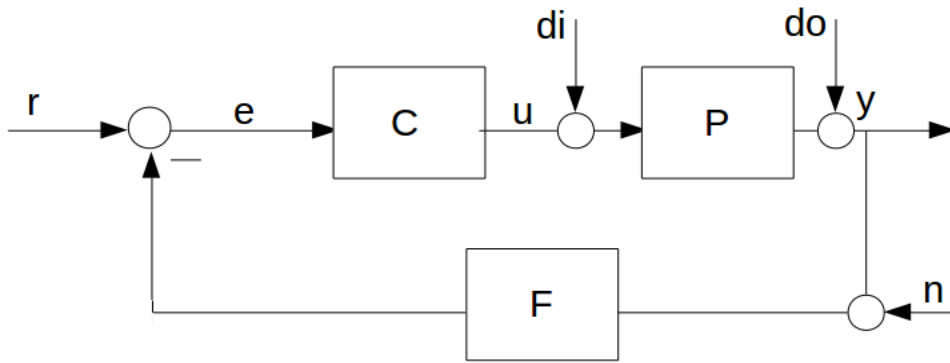


Figure 1: Control System Block Diagram

n-Sensor Noise, r-Reference Input, y-Outpt, P-Plant, C-Controller, F-Feedback, di-Input Disturbance, do-Output Disturbance, e-Error, u-Control Input

Hence, the aim of a control engineer is to design a controller in such a way that it deals with these uncertainties. The classical control theories, though, provide good performance for SISO systems but they are, in general, not replicable to multivariable systems. To overcome these difficulties and to achieve a robust performance in MIMO systems several successful design methods have been developed in the past three decades. Among them, the  $H_\infty$  control is a popular and effective robust control design technique.

The theory of robust control was first proposed in the late 1970s and early 1980s and since then there has been a huge effort in various methods to achieve specified performance of a system robustly. The most widespread and important of these methods is the  $H_\infty$  loop-shaping technique which was proposed by McFarlane and Glover of Cambridge University. The  $H_\infty$  loop-shaping technique combines classical loop-shaping concepts with the  $H_\infty$  controller design techniques.

For application of a robust controller to a particular application the constraints of the system need to be modeled as well and the controller design

has to be changed accordingly. This project aims to study in detail the  $H_\infty$  loop-shaping robust controller design technique and then model a mobile robot system in such a way that a robust controller for the mobile robot's locomotion is designed. Some challenges that might be faced in this process are those concerning the complexity of the very theory of  $H_\infty$  loop-shaping technique. Moreover, designing the controller for a mobile robot would be faced with many challenges with respect to actuator saturation and other modeling constraints.

## 2 Background and Motivation

The robust control theory is covered in great detail in the book by authors Kemin Zhou, Keith Glover and J C Doyle in "Robust and Optimal Control" [1]. The book was published in 1995 and hence it covers most of the research and contributions by various pioneers to modern control theory.  $H_\infty$  methods in control theory are used to design controllers which guarantee specific performance with robust stability. The  $H_\infty$  control problem is expressed as an optimization problem. Loop shaping control technique is a part of classical control that has been existent in control theory since a long time. Hence, the amalgamation of the two, known as  $H_\infty$  loop-shaping design method combines the two methods and achieves a good robust performance and stability.

The progress in robust control design techniques has been enormous and has even been applied to the industry to an aircraft. The 1995 publication breifs about the same [2]. Despite the popularity, the advantages of a robust controller are such that in future most of the controllers would be preferred to be designed using a robust algorithm and hence there remains a great scope of learning and research in this field. With regards to application of robust control theory, Gu, Petkov and Konstantinov published a book titled "Robust Control Design with MATLAB" [3]. The book elucidates the robust control design procedure using MATLAB and considers various multivariable systems and their controller designs as case studies. The second edition of this book demonstrates the use of the latest Robust Control Toolbox v3.0.

The motivation for this project comes from the fact that usually great amount of experimental, hit and trial based methods are employed for controlling a mobile robot. In this sense, if a robust control system can be designed it would provide great ease for further development of technology on robots. Robotics is finding it's way into our lives at a pace which was never expected and is more than ever before. The applications range from big industries to small household chores and beyond. In this changing

world, the control of such a mobile robot is essential and important to be achieved in a very robust manner so that more and more complex applications can be designed on top of it.

### 3 Preliminaries

As explained in Section 2, the robust controller design revolves around solving an optimization problem and hence it is obvious that various mathematical techniques need to be learnt and applied to successfully be able to learn the theory of  $H_\infty$  control technique. The following subsections go over the basics required as prerequisites for all the study on robust control that shall follow. The mathematical tools and concepts given in this section have been limited to an introduction only and proper references are given for the readers who wish to delve more into a particular concept.

#### 3.1 Vector Norms and Normed Spaces

In linear algebra, norm is a function that is defined for a vector which gives a positive scalar quantity representing the vector's size. It can be understood as an analogy to the absolute value function that we have for scalar quantities. A vector space for which a particular norm is defined is known as a normed space.

##### 3.1.1 Types of norms

**p-Norm** The p-norm is the most commonly used form of norm used in linear algebra. For a vector  $x \in \mathbb{R}$ , the p-norm is given by

$$\|x\|_p := \left( \sum_{i=1}^n (x_i)^p \right)^{\frac{1}{p}} \quad (1)$$

p can take any value in  $[1, \infty)$  which would give different norms such as the 2-norm and so on. The two norm is also known as the Euclidean norm or the energy norm.

#### 3.2 Banach Spaces

A complete normed space is called a Banach Space. For a norm space to be complete, the following two properties should be satisfied for any vector  $x, x_n$  and  $x_m$  in the general vector space  $V$ .

1. Convergence Property : When  $\|(x_n - x)\| \rightarrow 0$ , then  $x_n \rightarrow x$ .

2. Cauchy Sequence Property: For  $n$  and  $m \rightarrow \infty$ ,  $\|x_n - x_m\| < \epsilon$ , for small  $\epsilon > 0$ .

### 3.3 Inner product and Inner Product Spaces

The inner product between two vectors  $x$  and  $y$  is denoted by  $\langle x, y \rangle$  and is given by  $x^*y$  where the  $*$  operator is the adjoint operator which can be calculated by taking the transpose and then the conjugate of the complex vector.

$$\langle x, y \rangle = x^*y = \sum_{i=1}^n (\bar{x}_i y_i)$$

If for a vector space  $V$ , the inner product is defined in the manner as in eq. 3.3 and it exists. Then the vector space  $V$  is called an inner product space.

#### 3.3.1 Significance of inner product

The inner product operation gives a scalar output related to the two vectors. This scalar is significant in vector and geometrical analysis in many ways. The inner product is used to determine the length of a vector, angle between two vectors and it is also used to define the important property of orthogonality. The following equations have been mentioned without proof to summarize the results:

For length of the vector,

$$\langle x, x \rangle = \|x\|^2$$

Angle between two vectors  $x$  and  $y$  can be given as

$$\cos(\theta) = \frac{\langle x, y \rangle}{\|x\| \|y\|}$$

where  $\theta$  is the angle between  $x$  and  $y$ . Finally, the two vectors will be orthogonal to each other when -

$$\theta = \frac{\pi}{2}$$

that is,

$$\langle x, y \rangle = 0$$

### 3.4 Hilbert Spaces

A Hilbert space is a vector space which is a complete inner product space with the norm induced by its inner product.

#### 3.4.1 Examples

There can be many common examples for a Hilbert Space. Some of them are

1. The space  $\mathbb{R}^n$ , with the usual inner product defined is a finite dimensional Hilbert Space.
2. The space  $\mathbb{R}^{n \times m}$  of matrix valued functions is a Hilbert Space with the inner product defined as:

$$\langle A, B \rangle := \text{Trace} A^* B = \sum_{i=1}^n \sum_{j=1}^m \bar{a}_{ij} b_{ij} \quad \forall A, B \in \mathbb{C}^{n \times m}$$

3.  $l_2(-\infty, \infty)$  is an infinite dimensional Hilbert Space which consists of sequences  $\dots, x_{-2}, x_{-1}, x_0, x_1, \dots$  (real or complex) which are square summable. The inner product is given by:

$$\langle x, y \rangle := \sum_{i=-\infty}^{\infty} \bar{x}_i y_i$$

If each component is a vector or a matrix, then the inner product shall be given by

$$\langle x, y \rangle := \sum_{i=-\infty}^{\infty} \text{Trace} (x_i^* y_i)$$

4. Subspaces of  $l_2$  viz.  $l_2[0, \infty], l_2(-\infty, 0]$  are defined similarly.

### 3.5 Orthogonality

The  $\mathbb{L}_{2+}$  space has all matrix or vector valued functions which are zero for  $t < 0$  and  $\mathbb{L}_{2-}$  has all the functions with  $t > 0$  equal to zero. This implies that inner product between any two functions, one from  $\mathbb{L}_{2+}$  and other from  $\mathbb{L}_{2-}$  will always be equal to zero. Hence, the subspaces  $\mathbb{L}_{2+}$  and  $\mathbb{L}_{2-}$  are orthogonal to each other. It is also easy to see from here that  $\mathbb{L}_2$  is actually an orthogonal direct sum of its subspaces  $\mathbb{L}_{2+}$  and  $\mathbb{L}_{2-}$ .

### 3.6 Hardy Spaces

A function  $f(t)$  is said to be analytic at a point  $t_0$  if it is continuous at that point and in the neighbourhood along with being differentiable at  $t_0$ . For analytic functions over a set  $S$ , the derivatives of all orders at all points in  $S$  of the function  $f(t)$  exist. This also implies that the function will have a power series at all those points where it is analytic. The opposite of this is also true, that is, when the power series exists for a function at a point then it is differentiable and analytic at the given point as well. If a function is a matrix valued function then all elements of the matrix should be analytic at the considered point or at all points in the considered set, whatever may be the case.

$\mathbb{L}_2(j\mathbb{R})$  **space** can also be written simply as  $\mathbb{L}_2$  space is a Hilbert space of matrix valued functions on  $j\mathbb{R}$  and consists of all matrix valued functions for which the inner product is defined as follows

$$\langle F, G \rangle := \frac{1}{2\pi} \int_{-\infty}^{\infty} \text{Trace} [F^*(j\omega) G(j\omega)] d\omega \quad (2)$$

and the inner product induced norm is given by

$$\|F\|_2 := \sqrt{\langle F, F \rangle} \quad (3)$$

Now, if for a given transfer matrix  $F$  (real, rational, strictly proper), if there are no poles on the  $j\omega$  axis, then we have a subspace of  $\mathbb{L}_2$  space known as  $\mathbb{RL}_2$ .

**Hardy Spaces** are subspaces of  $\mathbb{L}_2(j\mathbb{R})$  space but with an additional property of analyticity of the functions. Formally, all matrix valued functions analytic in  $\text{Re}(s) > 0$  form the  $\mathbb{H}_2$  space. The inner product can be shown to be equal to the one shown in eq. 2. On similar lines as to  $\mathbb{RL}_2$ , we define  $\mathbb{RH}_2$  which is a subspace of  $\mathbb{H}_2$ . The  $\mathbb{RH}_2$  consists of all real rational and stable (all poles in LHP) transfer matrices. This is because if there were poles in the RHP then the functions would not be analytic anymore. The space  $\mathbb{H}_2^\perp$  is the subspace of  $\mathbb{L}_2$  having analytic functions in LHP, which implies  $\mathbb{RH}_2^\perp$  would contain all proper rational transfer matrices having all poles in the RHP.

Next, we have  $\mathbb{H}_\infty$  spaces. The above definitions follow in the same way for  $\mathbb{L}_\infty(j\mathbb{R})$ ,  $\mathbb{H}_\infty$  and  $\mathbb{H}_\infty^-$  spaces and the  $\mathbb{RL}_\infty$ ,  $\mathbb{RH}_\infty$ ,  $\mathbb{RH}_\infty^-$  subspaces respectively except that now the norms are given by the following set of equations,

For  $\mathbb{L}_\infty(j\mathbb{R})$  space

$$\|F\|_\infty := \operatorname{ess\,sup}_{\omega \in \mathbb{R}} \bar{\sigma}[F(j\omega)]$$

For  $\mathbb{H}_\infty$  space

$$\|F\|_\infty := \sup_{\operatorname{Re}(s) > 0} \bar{\sigma}[F(s)] = \operatorname{ess\,sup}_{\omega \in \mathbb{R}} \bar{\sigma}[F(j\omega)]$$

The second equality can be regarded as the generalization of maximum modulus theorem for matrix functions. Now for  $\mathbb{H}_\infty^-$  space

$$\|F\|_\infty := \sup_{\operatorname{Re}(s) < 0} \bar{\sigma}[F(s)] = \operatorname{ess\,sup}_{\omega \in \mathbb{R}} \bar{\sigma}[F(j\omega)]$$

where  $\bar{\sigma}$  is equal to maximum singular value of the matrix, where a singular value of a matrix  $T$  is given by  $\sqrt{\lambda_i(T^*T)}$  for  $i$ th eigen value of the matrix.

### 3.7 Well Posedness and Internal Stability

A feedback system is said to be well posed if all closed-loop transfer function matrices are well-defined and proper.

For a linear time invariant system given by a transfer function  $G(s)$ , stability of the system is assured if and only if all poles of the the transfer function lie in the open left half complex plane. This is also often referred to as the BIBO stability condition, i.e. when the input is bounded the output will be bounded. For a system described by state space representation, the concept of asymptotic stability is defined. A system is asymptotically stable if, for an identically zero input, the system states will converge to zero from any initial states. Hence, an interconnected system is internally stable if the subsystems of all input–output pairs are asymptotically stable.

### 3.8 Coprime factorization over $\mathbb{RH}_\infty$

For a SISO linear system whose transfer function is given  $b(s)/a(s)$ ,  $b(s)$  and  $a(s)$  are said to be coprime if they do not have any common zeros. Such a representation of the system is often referred to as the minimal realization of the system. For SISO systems, it is always possible to write a transfer function as a ratio of two stable transfer function with no common divisors (i.e. their greatest common divisor (gcd) is 1). This is called coprime factorization of the transfer function as the two factored transfer functions have no common divisors. Mathematically, using Euclid's algorithm we can write transfer functions  $X(s)$  and  $Y(s)$  in  $\mathbb{RH}_\infty$  such that for a given



transfer function  $G(s)$ , transfer functions  $M(s)$  and  $N(s)$  in  $\mathbb{RH}_\infty$  are it's coprime factors such that:

$$G(s) = N(s)M(s)^{-1} \quad (4)$$

where  $X$  and  $Y$  exists such that

$$N(s)X(s) + M(s)Y(s) = 1 \quad (5)$$

(5) is known as the Bezout's Identity. With some variations, the above is valid for MIMO systems as well. But, for MIMO systems both left and right coprime factorization need to be defined appropriately. In Chapter 5 of [1], detailed explanations and equations for the same have been given.

### 3.9 Feedback control interpretation of coprime factorization of a transfer matrix

On changing the control variable by a state feedback as shown in Fig.(2), the right coprime factorization occurs naturally as we will see in this section. For a given system, the state equations can be written as:

$$\dot{\mathbf{x}} = \mathbf{A}\mathbf{x} + \mathbf{B}\mathbf{u} \quad (6)$$

$$\mathbf{y} = \mathbf{C}\mathbf{x} + \mathbf{D}\mathbf{u} \quad (7)$$

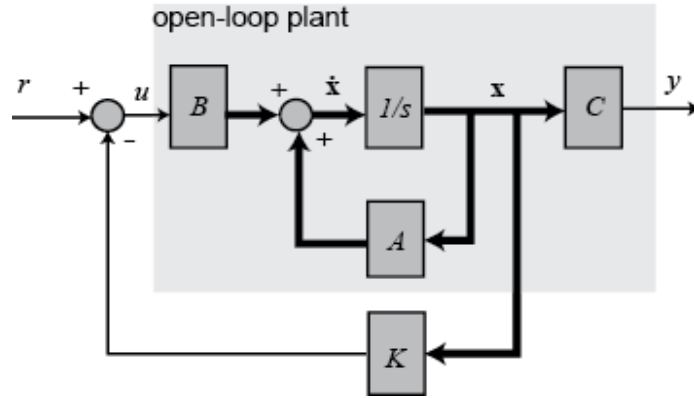


Figure 2: A Control System with State Feedback

When a state feedback is used such that  $u = r - Fx$ , where  $F$  is such that  $A+BF$  is stable. The state space representation reads:

$$\dot{\mathbf{x}} = (\mathbf{A}-\mathbf{BF})\mathbf{x} + \mathbf{B}\mathbf{r} \quad (8)$$

$$\mathbf{y} = (\mathbf{C}-\mathbf{DF})\mathbf{x} + \mathbf{D}\mathbf{r} \quad (9)$$

Now, on calculating the closed loop transfer matrix  $P(s) = y(s)/r(s)$  (see Figure (2)), we see that it's coprime factors are the transfer matrices  $r(s)/u(s)$  and  $y(s)/u(s)$ .

## 4 Linear Fractional Transformation

Linear Fractional Transformation or LFT is a unified theory of transfer function and state space representations including uncertainty. All  $H_\infty$  control problems use LFT representation of systems. An LFT representation could be of two types, upper and lower LFTs. Both of these structures are shown in the Figures (3) and (4).

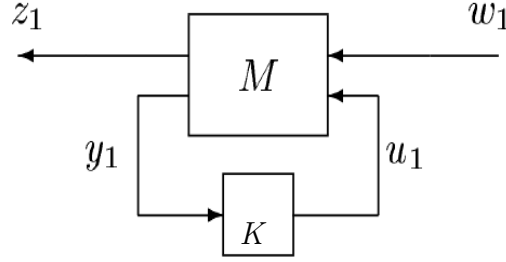


Figure 3: Lower LFT Structure

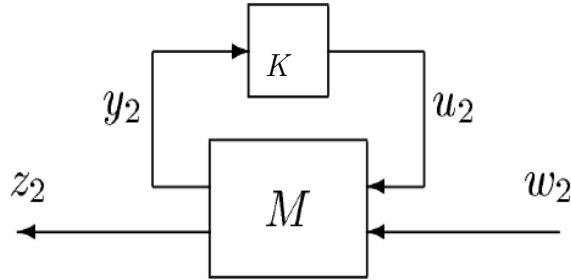


Figure 4: Upper LFT Structure

For the lower LFT structure from Figure(3) the transfer matrix from  $w$  to  $z$  is denoted by  $\mathcal{F}_l(G, K)$ . Now for this structure since we have,

$$\begin{bmatrix} z \\ y \end{bmatrix} = \begin{bmatrix} G_{11} & G_{12} \\ G_{21} & G_{22} \end{bmatrix} \begin{bmatrix} w \\ u \end{bmatrix}$$

we can easily work out (using  $u = Ky$ ) that the transfer matrix  $T_{zw}$  is given by  $G_{11} + G_{12}K(I - G_{22}K)^{-1}G_{21}$  which is denoted by  $\mathcal{F}_l(G, K)$ . Similarly for upper LFT one can write from Figure(4) the following equations:

$$\begin{bmatrix} y \\ z \end{bmatrix} = \begin{bmatrix} G_{11} & G_{12} \\ G_{21} & G_{22} \end{bmatrix} \begin{bmatrix} u \\ w \end{bmatrix}$$

The transfer function comes out as follows (the notation changes as well):  $T_{zw} = \mathcal{F}_u(G, K) = G_{22} + G_{21}K(I - G_{11}K)^{-1}G_{12}$ . Here the vectors  $z, w, u, y$  have special meaning as follows:

1.  $z$ : The objective signal vector. All signals which are to be penalized (or are of interest, performance wise) are kept in this vector.
2.  $w$ : The exogenous input vector. As the name suggests, all input exogenous signals (be it reference signal or disturbance) are included in this vector.
3.  $u$ : This is the signal which is given as output by the controller, called the “control signal”.
4.  $y$ : The measured variables are a part of this vector.

#### 4.1 Basic Principle

LFT framework is widely used for  $H_\infty$  control because it provides a general representation for all kinds of systems and systems with different uncertainties as well. This is one of the reasons why LFT forms the basic working framework for robust controller synthesis and analysis. The uncertainties in a system in an LFT framework are separated out and the resulting framework, called as an M- $\Delta$  structure consists of the nominal plant  $M$  (without any uncertainties) and  $\Delta$  matrix has all uncertain parameters. A closed loop system with uncertainties can hence be reduced in LFT framework as shown in Figure 5.

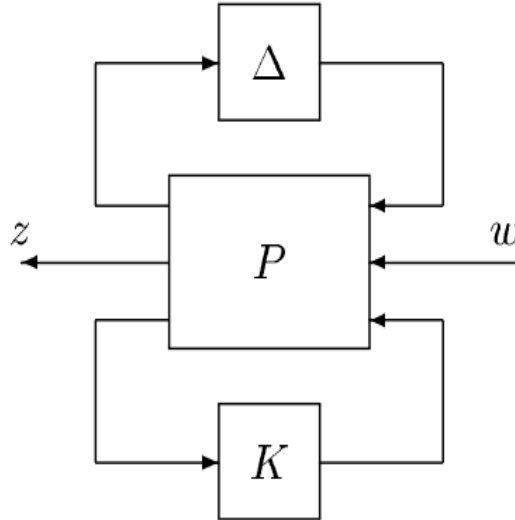


Figure 5: Closed Loop System in LFT Structure

The input output relation for the system can easily be worked out by considering the upper and lower LFT structures in the given system. For any given system, to reduce in this M- $\Delta$  format, all uncertain parameters need to be “pulled out” of the given plant. This can be achieved after drawing the block diagram of the system such that all uncertain parameters are individual blocks in the block diagram. This technique is illustrated in detail in the following example of a spring mass damper system.

## 4.2 Spring Mass Damper System : LFT Configuration

The Figure(6) shows a simple spring mass damper system with external force  $F$  being applied on the mass.

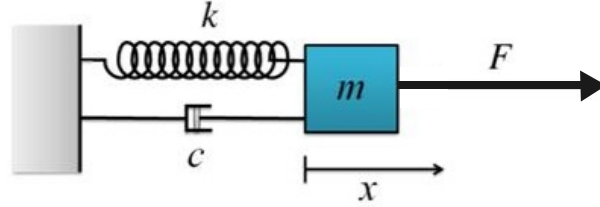


Figure 6: Spring Mass Damper System

The system parameters  $m$  (mass of the block),  $c$  (the damping constant) and  $k$  (spring constant) are uncertain parameters with given upper and lower bound on uncertainties. The equation of motion is given by :

$$\ddot{x} + \frac{c}{m}\dot{x} + \frac{k}{m}x = \frac{F}{m} \quad (10)$$

A block diagram with two integrators and parametric uncertainties can be easily drawn, as shown in Figure(7), from the above equation using the uncertainty values within 10% of mass  $\tilde{m}$ , within 20% of  $\tilde{c}$  and within 30% for  $\tilde{k}$ . The corresponding uncertain parameters are given by  $\delta_m, \delta_c$  and  $\delta_k$  respectively. Note here that  $\tilde{x}$  represents the nominal value of the parameter, where  $x$  is the uncertain parameter.

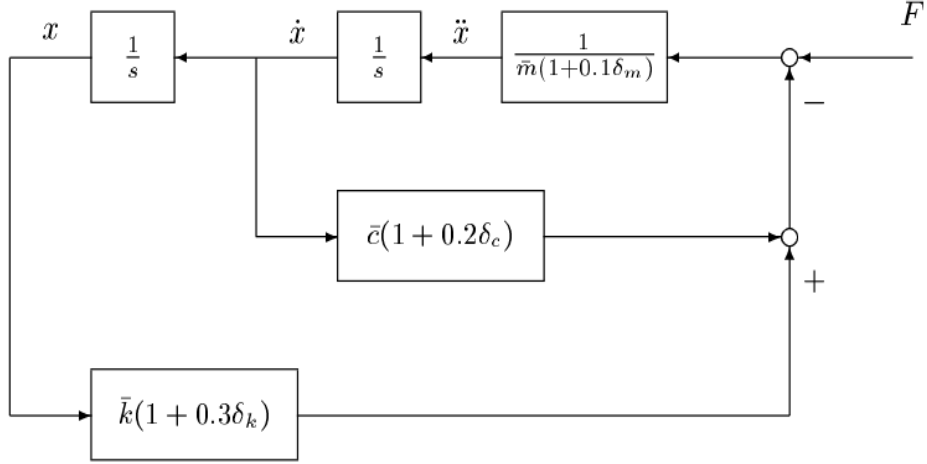


Figure 7: Spring Mass Damper Block Diagram with Uncertainties

The three uncertain parameters now need to be represented in LFT framework and for that we need to separate out the  $\Delta$ s. To achieve this, we first evaluate the  $\frac{1}{m}$  block in the given block diagram. The  $\frac{1}{m}$  block can be represented as LFT by the following manipulation:

$$\frac{1}{m} = \frac{1}{\tilde{m}(1 + 0.1\delta_m)} \quad (11)$$

on evaluating, we get

$$\frac{1}{m} = \frac{1}{\tilde{m}} - \frac{0.1\delta_m}{\tilde{m}} \cdot (1 + 0.1\delta_m)^{-1} \quad (12)$$

The above equation can be represented in lower LFT framework where, the nominal plant  $M$  is given by  $\mathcal{F}_l(M, \Delta_m)$  and  $\Delta = \delta_m$ .

$$M = \begin{bmatrix} \frac{1}{\tilde{m}} & \frac{-0.1}{\tilde{m}} \\ 1 & -0.1 \end{bmatrix}$$

Next, to separate out  $\delta_c$  and  $\delta_k$ , simple block diagram manipulations lead us to the following block diagram shown in Figure(8).

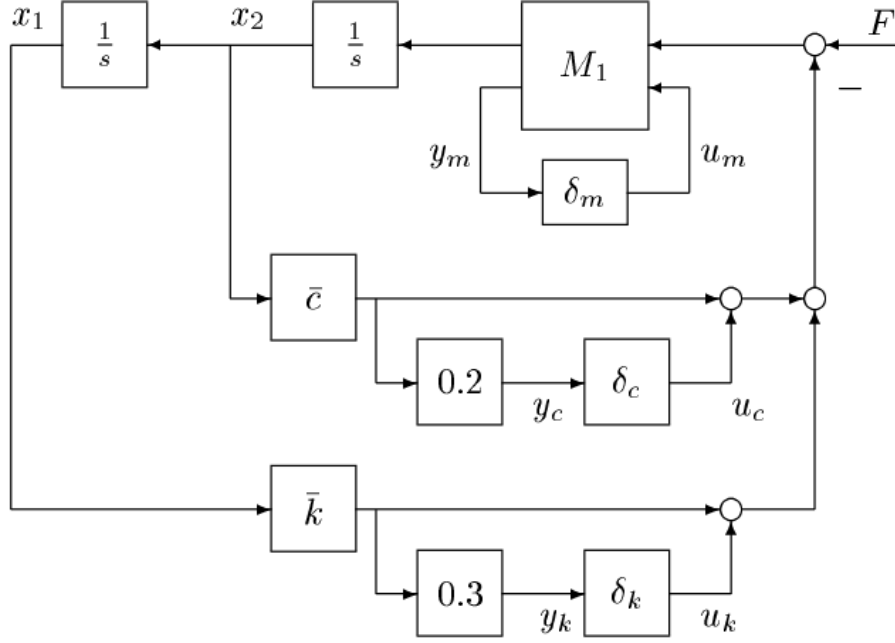


Figure 8: Spring Mass Damper Block Diagram with Uncertainties Pulled Out

Now, we can mark the inputs and outputs to these  $\delta$ s appropriately.  $y$  is the input to the delta and  $u$  is its output. Using these inputs and outputs to  $\delta$ s along with exogenous input vector and the objective signal vector we can represent the whole spring mass damper system into M- $\Delta$  structure. The values of  $M$  and  $\Delta$  can be found out by actually writing out the state space representation using the block diagram. For this system the exogenous signal vector can be chosen to comprise of  $F$ ,  $x_1$  and  $x_2$  and the objective signal vector as  $\dot{x}_2$  and  $\dot{x}_1$ , where  $x_1$  and  $x_2$  are the states of the system. The final M- $\Delta$  structure in LFT form is given as follows:

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ y_k \\ y_c \\ y_m \end{bmatrix} = \left[ \begin{array}{ccc|ccc} 0 & 1 & 0 & 0 & 0 & 0 \\ \frac{-\tilde{k}}{\tilde{m}} & \frac{-\tilde{c}}{\tilde{m}} & \frac{1}{\tilde{m}} & \frac{-1}{\tilde{m}} & \frac{-1}{\tilde{m}} & \frac{-0.1}{\tilde{m}} \\ \hline 0.3\tilde{k} & 0 & 0 & 0 & 0 & 0 \\ 0 & 0.2\tilde{c} & 0 & 0 & 0 & 0 \\ -\tilde{k} & -\tilde{c} & 1 & -1 & -1 & -0.1 \end{array} \right] \begin{bmatrix} x_1 \\ x_2 \\ F \\ u_k \\ u_c \\ u_m \end{bmatrix}$$

,

$$\begin{bmatrix} u_k \\ u_c \\ u_m \end{bmatrix} = \Delta \begin{bmatrix} y_k \\ y_c \\ y_m \end{bmatrix}$$

. Hence, the uncertainty in parameters is all put together in the matrix  $\Delta$  which is given by:

$$\Delta = \begin{bmatrix} \delta_k & 0 & 0 \\ 0 & \delta_c & 0 \\ 0 & 0 & \delta_m \end{bmatrix}$$

### 4.3 Generalized SISO Control System Block Diagram

A simple SISO system consisting of a reference signal, controller, disturbance and a plant. The block diagram is shown in Figure (9).

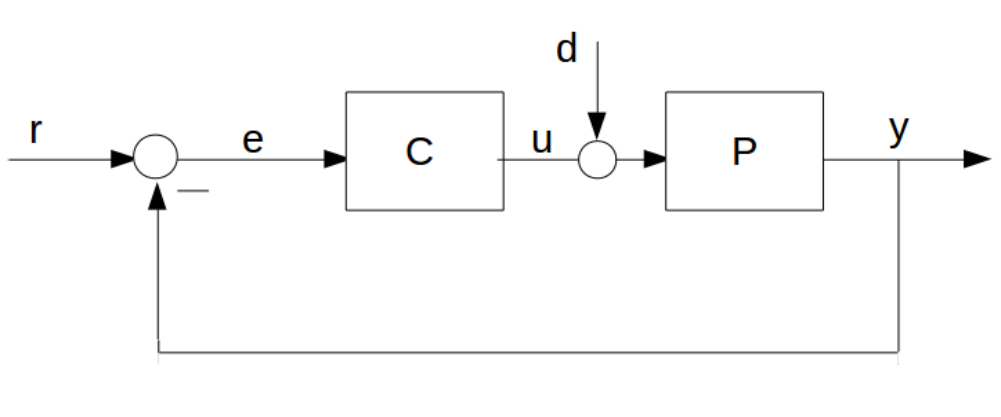


Figure 9: A SISO Control System Block Diagram

The exogenous signal vector ( $w$ ) for this system will have  $r$  and  $d$  as its elements. The objective signal (if good tracking of reference is desired) can be chosen as  $e$ . The input to the controller  $C$  is equal to  $e$ , hence  $y = e$ . The control input in this case is given as  $u$ . Now, we can write the transfer function matrix form of the system in LFT framework.

$$e = r - y \quad (13)$$

$$e = r - Pd - Pu \quad (14)$$

We can write,

$$\begin{bmatrix} e \\ e \end{bmatrix} = \left[ \begin{array}{cc|c} I & -P & -P \\ I & -P & -P \end{array} \right] \begin{bmatrix} r \\ d \\ u \end{bmatrix}$$

Hence, we can write  $z = \mathcal{F}_l(G, C)w$  where

$$G_{11} = [I \quad -P], G_{12} = [-P]$$

$$G_{21} = [I \quad -P], G_{22} = [-P]$$

The state space representation can also be written by following a similar procedure. The following equations describe the state variable model. It is assumed that the state space representation of the plant P and the controller C are known and their states are denoted as  $x_p$  and  $x_c$ .

$$\dot{x}_p = A_p x_p + B_p d + B_p u \quad (15)$$

$$y = C_p x_p + D_p d + D_p u \quad (16)$$

$$\dot{x}_c = A_c x_c + B_c r - B_c y \quad (17)$$

$$u = C_c x_c + D_c r - D_c y \quad (18)$$

On solving using the block diagram, we finally get,

$$\begin{bmatrix} \dot{x}_p \\ \dot{x}_c \\ e \\ e \end{bmatrix} = \left[ \begin{array}{cc|cc|c} A_p & 0 & 0 & B_p & B_p \\ -B_c C_p & A_c & B_c & -B_c D_p & -B_c D_p \\ \hline -C_p & 0 & I & -D_p & -D_p \\ -C_p & 0 & I & -D_p & -D_p \end{array} \right] \begin{bmatrix} x_p \\ x_c \\ r \\ d \\ u \end{bmatrix}$$

Hence, the generalised system matrices are

$$A = \begin{bmatrix} A_p & 0 \\ -B_c C_p & A_c \end{bmatrix}, B_1 = \begin{bmatrix} 0 & B_p \\ B_c & B_c D_p \end{bmatrix}, B_2 = \begin{bmatrix} B_p \\ -B_c D_p \end{bmatrix}$$

$$C_1 = [-C_p \ 0], C_2 = [-C_p \ 0]$$

$$D_{11} = [I \ -D_p], D_{12} = [-D_p], D_{21} = [I \ -D_p], D_{22} = [-D_p]$$

#### 4.4 Other Worked Out Examples

Since LFT plays such an important role in  $H_\infty$  control design and analysis, some more examples were worked out as a part of this project. The following examples deal with different situations such as presence of sensor noise in measurement in a SISO control system, presence of exogenous output and input disturbances in the system simultaneously along with sensor noise (this is the most general case for SISO systems) and assigning weights to objective signals in multivariable systems (in presence of disturbance and noise signals).

The block diagram shown in Figure (10) can be reduced in LFT framework in both state space and transfer function forms.



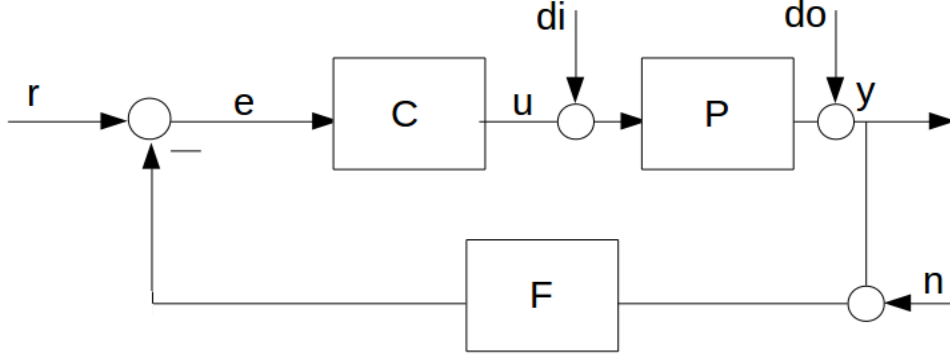


Figure 10: A SISO Control System Block Diagram with Noise and Disturbances

The exogenous input signal vector ( $w$ ) for this case is given by:

$$w = \begin{bmatrix} r \\ d_i \\ d_o \\ n \end{bmatrix}$$

The objective signal vector  $z$  can be taken as equal to  $e$  for robust tracking requirement. The input to the controller  $C$  is taken as  $e$  and the control input is  $u$ . The transfer function form is given by:

$$\begin{bmatrix} e \\ e \end{bmatrix} = \left[ \begin{array}{cccc|c} I & -FP & -F & -F & -FP \\ I & -FP & -F & -F & -FP \end{array} \right] \begin{bmatrix} r \\ d_i \\ d_o \\ n \\ u \end{bmatrix}$$

Hence,  $z = \mathcal{F}_l(G, C)w$  is the lower LFT representation of the system in transfer function form. For state space form, similar to the previous example we can write the state space representation of the system. The final result is given below:

$$\begin{bmatrix} \dot{x}_p \\ \dot{x}_c \\ \dot{x}_f \\ e \\ e \end{bmatrix} = \left[ \begin{array}{ccc|cccc|c} A_p & 0 & 0 & 0 & B_p & 0 & 0 & B_p \\ -B_c B_f C_p & A_c & -B_c C_f & B_c & -B_c D_f D_p & -B_c D_f & -B_c D_f & -B_c D_f D_p \\ B_f C_p & 0 & A_f & 0 & B_f D_p & B_f & B_f & B_f D_p \\ \hline -D_f C_p & 0 & -C_f & I & -D_f D_p & -D_f & -D_f & -D_f D_p \\ -D_f C_p & 0 & -C_f & I & -D_f D_p & -D_f & -D_f & -D_f D_p \end{array} \right] \begin{bmatrix} x_p \\ x_e \\ x_f \\ r \\ d_i \\ d_o \\ n \\ u \end{bmatrix}$$

Hence, the final system matrices are

$$A = \begin{bmatrix} A_p & 0 & 0 \\ -B_c B_f C_p & A_c & -B_c C_f \\ B_f C_p & 0 & A_f \end{bmatrix}$$

$$B_1 = \begin{bmatrix} 0 & B_p & 0 & 0 \\ B_c & -B_c D_f D_p & -B_c D_f & -B_c D_f \\ 0 & B_f D_p & B_f & B_f \end{bmatrix}, B_2 = \begin{bmatrix} B_p \\ -B_c D_f D_p \\ B_f D_p \end{bmatrix}$$

$$C_1 = [-D_f C_p \quad 0 \quad -C_f], C_2 = [-D_f C_p \quad 0 \quad -C_f]$$

$$D_{11} = [I \quad -D_f D_p \quad -D_f \quad -D_f], D_{12} = [-D_f D_p]$$

$$D_{21} = [I \quad -D_f D_p \quad -D_f \quad -D_f], D_{22} = [-D_f D_p]$$

As one last example, we take a multivariable system with weights  $W_1$  and  $W_2$  assigned to the control input and the objective signal vectors. These weights help in better formulation of cost function in the  $H_\infty$  optimization problem. Assigning weight to control input  $u$  helps in achieving the performance criteria of having less control energy along with achieving the required performance by assigning weight to the objective signal. The block diagram for the system is shown in Figure(11).

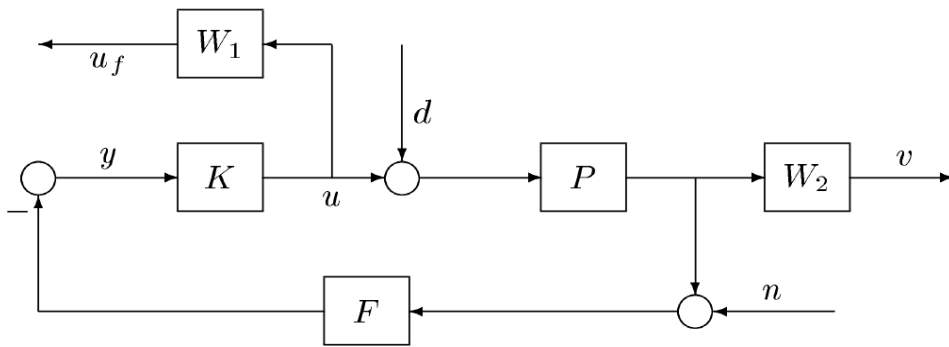


Figure 11: A MIMO Control System Block Diagram with Weights Assigned

The objective signal vector here will have both  $u_f$  and  $v$  (See that  $u_f = W_1 u$ , where  $u$  is the control input signal). Other vectors are chosen

in similar fashion as previous systems.

$$\begin{bmatrix} v \\ u_f \\ y \end{bmatrix} \left[ \begin{array}{cc|c} W_2P & 0 & W_2P \\ 0 & 0 & W_1 \\ -FP & -F & -FP \end{array} \right] \begin{bmatrix} d \\ n \\ u \end{bmatrix}$$

The LFT structure is  $z = \mathcal{F}_l(G, K)w$ . The elements of  $G$  are clear from the partition shown above.

## 5 Uncertainties

The previous section dealt with uncertainties in various systems in detail and it was shown that using LFT framework these uncertainties can be handled very efficiently. However, we did not look much into the modeling of  $\Delta$ , the uncertainty. This section deals briefly on the types of uncertainties that exist in a system and gives a short introduction on modeling two such kinds of uncertainties.

Broadly, there may be three kinds of uncertainties in a system:

1. Unstructured Uncertainty
2. Parametric Uncertainty
3. Structured Uncertainty

At the moment, the structured uncertainties are out of scope of this project and hence the other two kinds of uncertainties were studied and are briefly described below:

### 5.1 Unstructured Uncertainties

Many dynamic perturbations occurring throughout the system may be lumped into one single block,  $\Delta$ . This is referred to as unstructured uncertainty. To model these uncertainties in a system either transfer function matrix approach or LFT framework could be used. Some examples of unstructured uncertainties are additive uncertainty, multiplicative uncertainty etc. These and many others have been elaborately presented in [1] and [3]. For a system with unstructured uncertainties simulation was performed in MATLAB using the Robust Control Toolbox. The system shown in Figure (10), with  $G(s)$  and  $C(s)$  transfer function matrices as follows, was chosen. The feedback transfer function matrix  $H(s) = 1$ .

$$G(s) = \begin{bmatrix} \frac{1}{s+1} & \frac{0.5s+0.2}{(s+0.3)(s+1)} \\ \frac{-0.7}{0.2s+1} & \frac{1}{s+0.3} \end{bmatrix} K(s) = \begin{bmatrix} \frac{7(s+1)}{0.3s+1} & 0 \\ 0 & \frac{18(s+2)}{s+1} \end{bmatrix}$$

The Figures (12), (13),(14),(15) show the transient responses of the two input two output system.

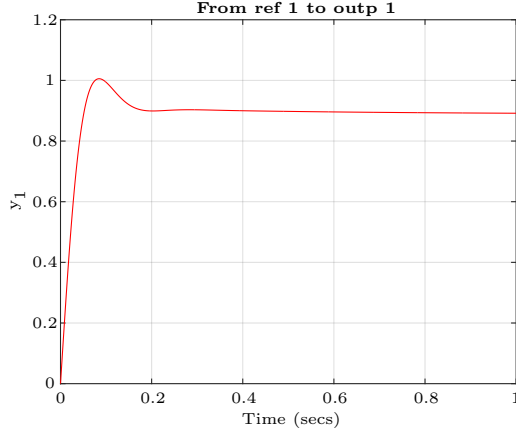


Figure 12: Ref1 to Out1

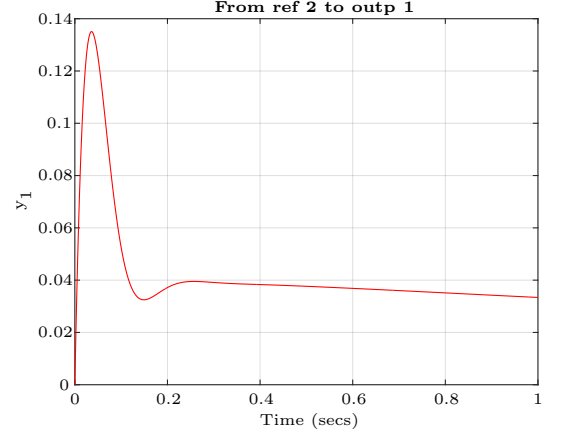


Figure 13: Ref2 to Out1

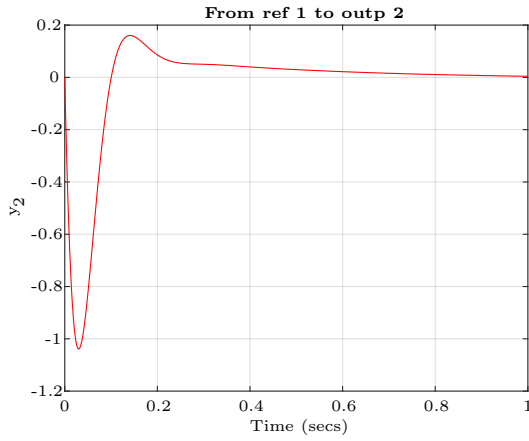


Figure 14: Ref1 to Out2

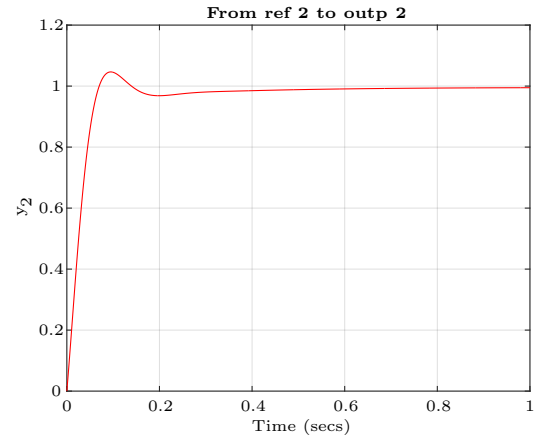


Figure 15: Ref2 to Out2

Also, the Figures (16),(17), (18), (19) show the singular value plots of input and output sensitivity and complimentary sensitivity functions plotted using MATLAB. Along with this, many other simulations were done such as responses to disturbance signal etc.

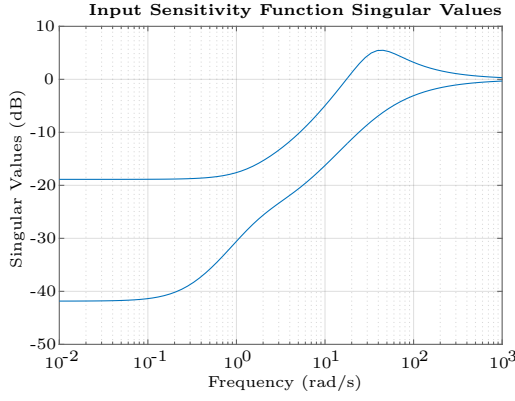


Figure 16: Input Sensitivity Singular Value

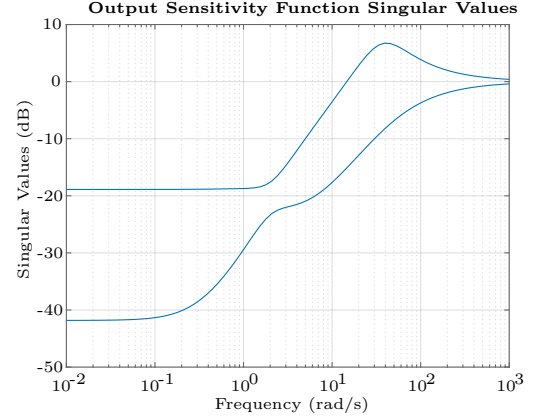


Figure 17: Output Sensitivity Singular Value

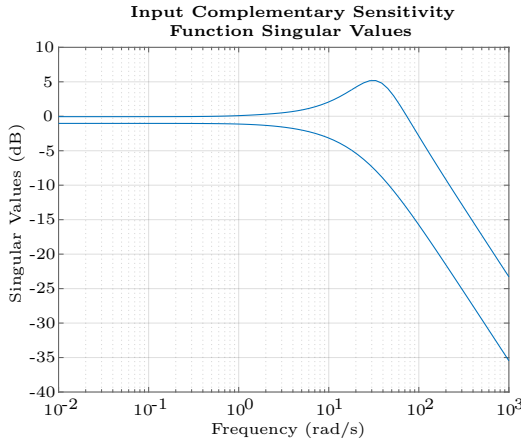


Figure 18: Input Complementary Sensitivity Singular Value

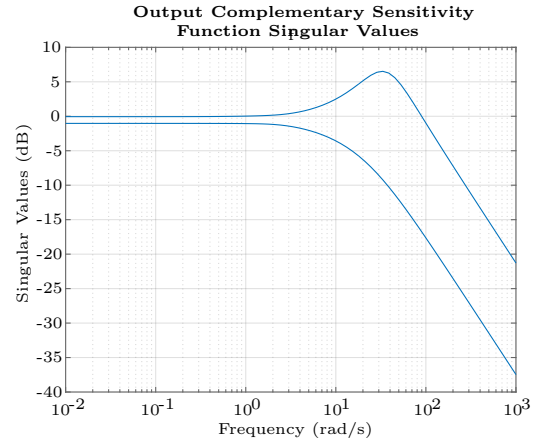


Figure 19: Output Complementary Sensitivity Singular Value

## 5.2 Parametric Uncertainties

The example of spring mass damper was a perfect example to demonstrate parametric uncertainties in a physical system. These uncertainties are due to unknown exact values of certain parameters within the plant. These are often useful in defining unmodeled or neglected system dynamics.

For spring mass damper system, the parametric uncertainties were modeled in MATLAB and the responses of the system in presence of these uncertainties was studied in detail. The bode plot of the uncertain system is shown in 20. Also, the step response of the system in presence of these parametric uncertainties of given level is shown in Figure (21).

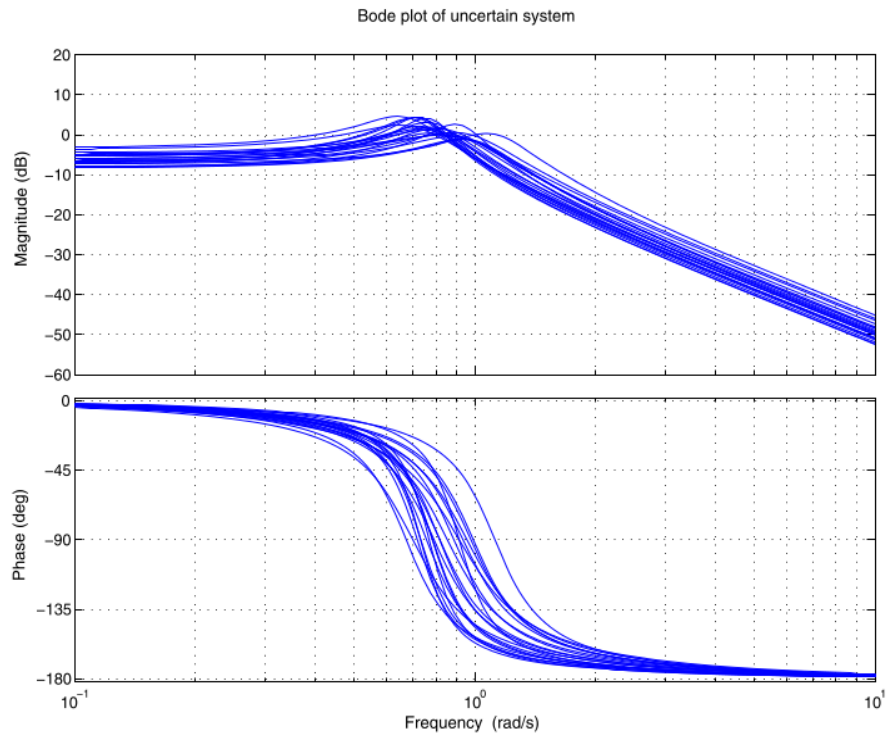


Figure 20: Bode Plot for Spring Mass Damper Uncertain System

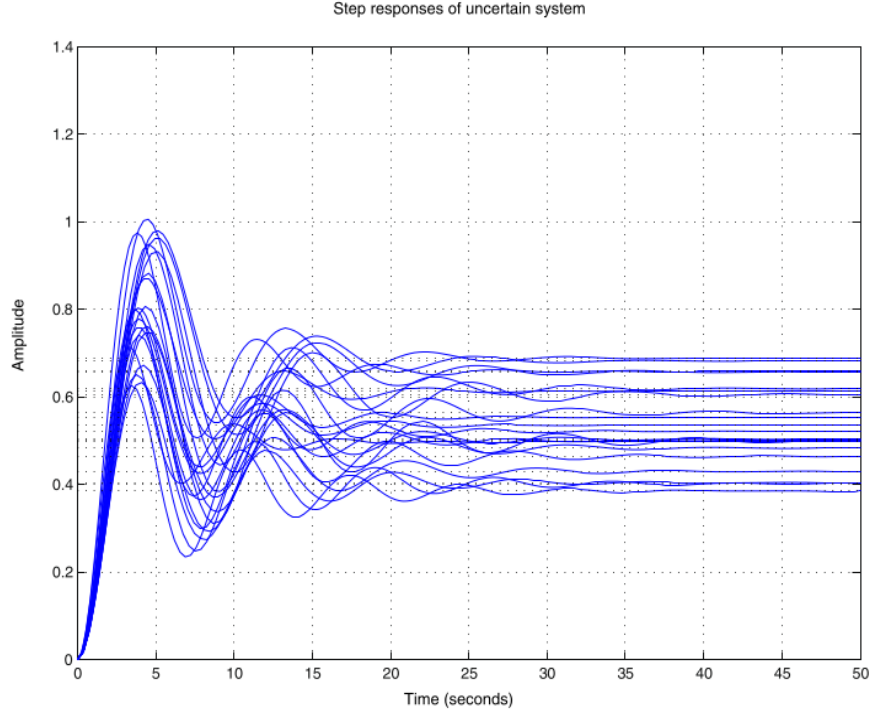


Figure 21: Step Response for Spring Mass Damper Uncertain System

Apart from the spring mass damper system, two other systems with parametric uncertainties was simulated on MATLAB.

## 6 Small-Gain Theorem

In general, the small-gain theorem provides a sufficient condition for stability of a given system. To ensure internal stability, it is widely used in robust control system designs. For a given  $M$ - $\Delta$  structure, the theorem states that the system is internally stable if,

$$\|M\Delta\|_{\infty} < 1 \quad (19)$$

The infinity norm can be calculated as shown in Section 3.6. The small-gain theorem can then be formulated as follows:

$$\sup_{\omega}(\Delta(j\omega)) \sup_{\omega}(M(j\omega)) = \|\Delta\|_{\infty} \|M\|_{\infty} < 1 \quad \forall \omega \in \mathbb{R} \quad (20)$$

where  $\Delta, M \in \mathbb{RH}_{\infty}$ .

The small-gain theorem is used in analysis problem of a given closed loop control system. The analysis of a given system involves checking for robust

stability of a system given a controller for the nominal plant. To analyse a closed loop system, first, it needs to be reduced to LFT framework. This can be done by methods described in the previous section. Next, to comment about the internal stability of the closed loop system the small-gain theorem statement above needs to be verified. From Equation 20, we see that first we need to find out the norm of the  $M$  block. A later section in this report gives the details on calculation of the norm and its simulations on MATLAB. However, to analyse systems where the controller is dependent on a few unknown parameters, we need to calculate the norm using other methods which involve the use of Bounded Real Lemma and the Linear Matrix Inequalities.

## 7 $H_\infty$ Control

Aim of any robust controller synthesis would be to allow for as much uncertainties in the system as possible. Using the small-gain theorem, this controller design criteria can be fulfilled by minimizing the infinity norm of nominal plant model ( $M$ , see section 6). This is the very idea that forms the basis of  $H_\infty$  control.

### 7.1 Optimization Problem and Cost Functions

The objective of  $H_\infty$  controller design is to find a stabilizing controller  $K$  to minimize the output  $z$ , in the sense of energy (i.e. the induced 2-norm or the transfer matrix matrix operator norm), over all  $w$  with energy less than or equal to 1. This is equivalent to minimizing the  $H_\infty$  norm of the transfer function from  $w$  to  $z$ . This transfer function has been shown to be represented in terms of LFT structure and hence, the design objective is stated as follows:

$$\min_{K \text{ stabilizing}} \|\mathcal{F}_l(P, K)\|_\infty \quad (21)$$

where the generalized plant is given by  $P$ .

### 7.2 $H_\infty$ Suboptimal Solutions

At times, in practical designs it suffices to achieve the above minimization upto a certain extent, i.e. the  $H_\infty$  norm above can be minimized such that it is less than a given  $\gamma$  where  $\gamma > \gamma_{min}$ . The solution so obtained is called a suboptimal solution.



### 7.3 Calculation of $H_\infty$ norm

As explained in Section 3.6, the  $H_\infty$  norm for a transfer matrix  $G(s) \in \mathbb{RH}_\infty$  is given by maximum singular value of the matrix. The  $H_\infty$  norm is defined as follows:

$$\|G(s)\|_\infty = \sup_w \bar{\sigma}(G(s)) \quad (22)$$

which implies the following should be satisfied for all frequencies

$$\underline{\sigma}(G) \leq \|G\|_\infty \leq \bar{\sigma}(G) \quad (23)$$

Some examples were worked out regarding the above. For a constant matrix  $G$  given as,

$$G = \begin{bmatrix} 5 & 4 \\ 3 & 2 \end{bmatrix}$$

Using,

$$\sigma_i = \sqrt{\lambda_i(G^*G)} \quad (24)$$

The singular values were calculated and maximum of the values was found. This was verified using the MATLAB function *hinfnorm* of the Robust Control Toolbox. Also, the singular values of this matrix were plotted on logarithmic scale. The plot is shown in Figure (22).

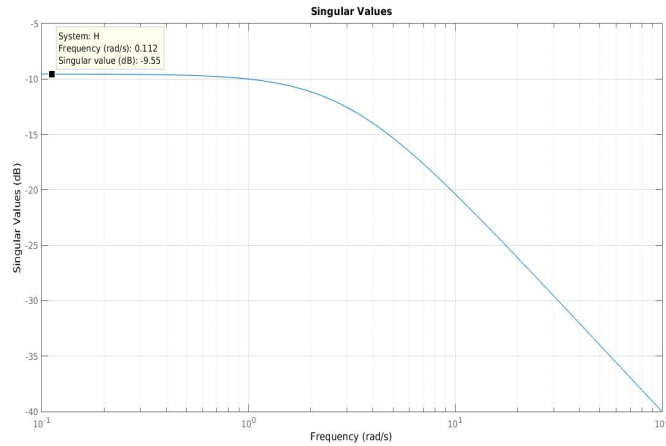


Figure 22: Singular Value Plot for SISO system

The singular value plot shows the system gain and the peak of the plot corresponds to the  $H_\infty$  norm of the given matrix. This peak value represents the maximum gain the system can produce to a given input.

For SISO systems, this is equal to the peak value of the frequency response plot. This fact was verified by calculating the  $H_\infty$  norm of a given SISO system. The result was verified using MATLAB code as well. The system  $H(s) = \frac{1}{s+3}$  was chosen and the peak value of the magnitude frequency response was obtained to be equal to 0.33. The MATLAB plot displayed the corresponding value in dB to be equal to -9.54, hence verifying the stated fact.

Extending the above to multivariable system where the  $H_\infty$  norm is equal to the maximum singular value of the system, the following system was evaluated:

$$A = \begin{bmatrix} -1 & 0 \\ 0 & -3 \end{bmatrix} B = \begin{bmatrix} 0 & 1 \\ 2 & 1 \end{bmatrix}$$

$$C = \begin{bmatrix} 1 & 2 \\ 1 & 0 \end{bmatrix} D = \begin{bmatrix} 0 & 0 \\ 0 & 0 \end{bmatrix}$$

The maximum singular value for this system is equal to 2.2836. The MATLAB simulation of this system was used to verify the result. The concerned plot is shown in Figure(23).

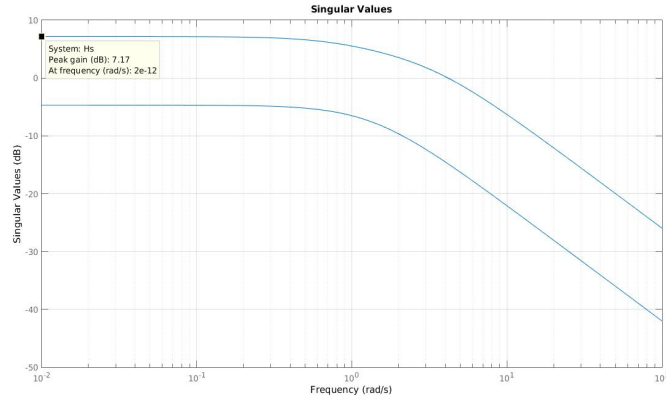


Figure 23: Singular Value Plot for the MIMO system

## 8 Conclusion and Future Work

The project looked into the basic preliminaries needed for robust control system synthesis and analysis. The linear fractional transformation theory which is widely used to represent all systems with uncertainties was studied. Using the LFT framework, many systems were modeled. Some examples from generalized SISO systems were also taken to represent them in LFT structure. Also, a common physical system, the spring mass damper,

with parametric uncertainties was expressed as an LFT. Various other multivariable examples were taken to illustrate the LFT framework which leads to the  $M$ - $\Delta$  structure, i.e. the generalized plant  $M$  and the uncertainties  $\Delta$  are separated out. The uncertainty modeling for various physical systems was done using MATLAB and responses of the plants were plotted in presence of uncertainty. The spring mass damper system uncertainties were modeled as well. Along with this, two other systems were modeled in MATLAB with their uncertainties to illustrate the procedure. To analyze a given physical system with uncertainties, the small-gain theorem was studied. In the end, the  $H_\infty$  control problem statement was formulated as well, using the developed theories of LFT and small-gain theorem. Future work comprises of rigorous analysis of different physical systems for robust stability and then synthesis of controller using the McFarlane-Glover  $H_\infty$  loop-shaping technique.

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- [1] Kemin Zhou, John C. Doyle, and Keith Glover. *Robust and Optimal Control*. Englewood Cliffs, New Jersey: Prentice Hall, 1995.
- [2] R. Hyde. “VSTOL first flight on an  $H_\infty$  control law”. In: (1995).
- [3] Da-Wei Gu, Petko H. Petkov, and Mihail M. Konstantinov. *Robust Control Design with MATLAB Second Edition*. London: Springer, 2012.