

# Basics of Feedback Control

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Lecture 22

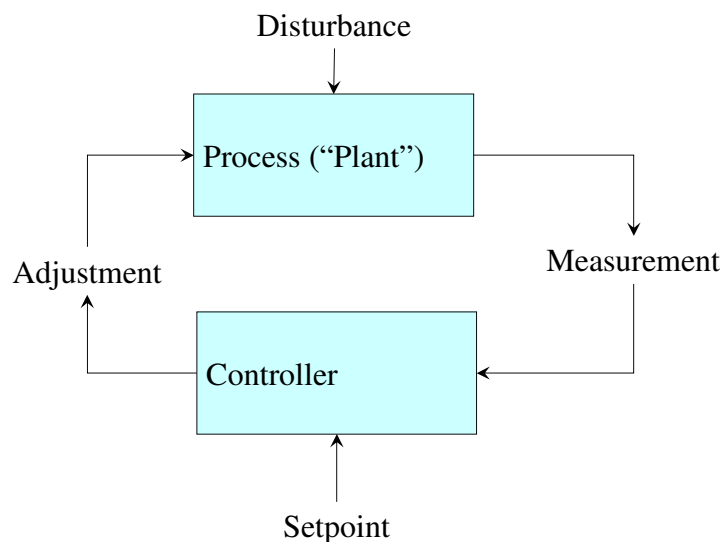
## Outline of This Lecture

- The Feedback Control Problem
- Proportional Control
- Derivative Control
- Integral Control

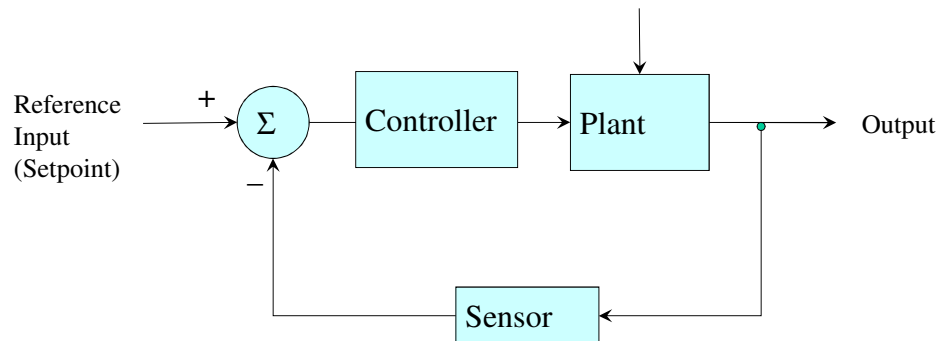
## Feedback in Embedded Systems

- Many real-time embedded systems make **control decisions**
  - E.g. Elevators, automotive cruise control
- These decisions are usually made by software and based on feedback from the hardware under its control (termed the “**plant**”)
  - Such feedback commonly takes the form of an analog sensor that can be read via an A/D converter
  - A sample from the sensor might represent position, voltage, temperature, or any other appropriate parameter
- Each sample provides the software with additional information upon which to base its control decisions

## High-Level View of Feedback Control

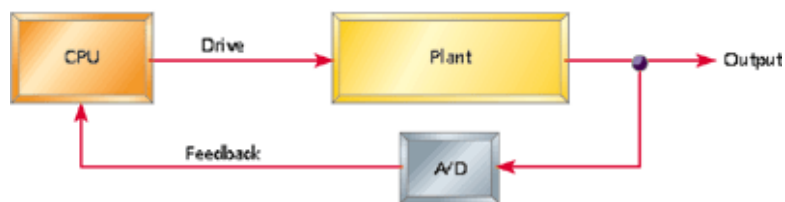


## Actual Feedback Control Loop



## Closed-Loop vs. Open-Loop

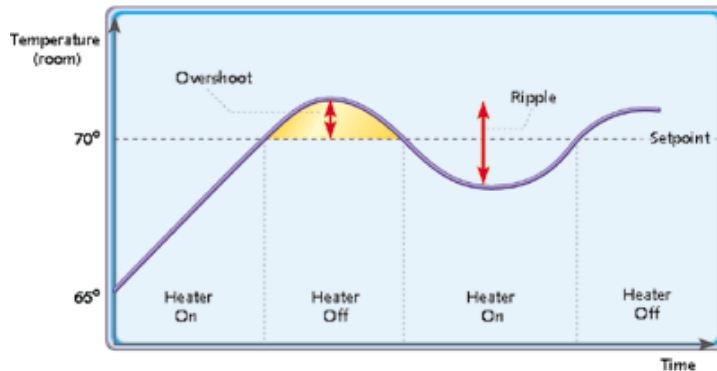
- Open-loop systems do not necessarily use feedback
- Closed-loop systems use feedback



- After reading each new sample from the sensor
  - Software reacts to the plant's changed state by recalculating and adjusting the drive signal
  - Plant responds to this change, another sample is taken, and the cycle repeats
- Eventually, the plant should reach (close to) the desired state and the software may cease making changes

## On-Off Control – Your Thermostat

- How much should the software increase or decrease the drive signal?
  - E.g. Set the drive signal to its minimum value when you want the thermostat to decrease its temperature
  - Set the drive signal to its maximum value when you want the thermostat to



## Proportional Control

- If the difference between the current output and its desired value (the current error) is large
  - Software should probably change the drive signal a lot
- If the error (difference) is small
  - Software should change the drive signal only a little
- We want the change to look like this
  - $K_p * (\text{desired} - \text{current})$
  - $K_p$  is called the **proportional gain**
- If  $K_p$  is chosen well,
  - Time taken to reach a new setpoint will be as short as possible, with overshoot (or undershoot) and oscillation minimized
- Is there a problem here?

## Proportional-Derivative (PD) Control

- Problems with Proportional Control
  - You want to reach the desired value quickly (higher  $K_p$ )
  - You want to settle down once you are near the desired value (low  $K_p$ )
- We have (or can discover) information about the rate of change of the plant's output
  - Rate of change of a signal = its derivative
  - If the output is changing rapidly, an overshoot/undershoot may lie ahead
- We can then reduce the size of the change suggested by the proportional controller
  - Subtract a derivative-based term from the proportional term
- PD Controller characteristics
  - Slower response time
  - Far less overshoot and ripple than a proportional controller

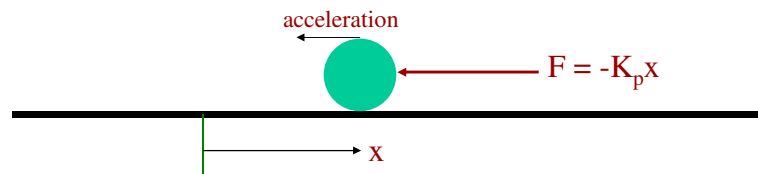
## Proportional-Integral-Derivative (PID) Control

- Problems with PD Control
  - Change in drive signal triggered by error = difference in desired and current values of output
  - Derivative only tells you how two consecutive output values differ
  - What if the error is not sufficient to trigger any action?
- Something else is needed to drive the plant toward the setpoint
- Integral term
  - Sum of all past errors in the plant output
    - Or at least a significant recent window
  - A persistent error will eventually cause the sum to grow large enough
  - The integral term then forces a change in the drive signal
- Proportional, Integral and Derivative combination = PID

## A Simple System with *Proportional Control*

Consider a marble on a flat and perfectly level table.

- Any point can be an equilibrium point (just pick one)
- Motion can be described by Newton's law  $\mathbf{F} = m\mathbf{a}$ , or  $\mathbf{x}'' = \mathbf{F}/m$
- Suppose that we want to keep the marble at  $\mathbf{x} = \mathbf{0}$ ,
  - By applying proportional control:  $\mathbf{F} = -K_p\mathbf{x}$ .
- The feedback is negative since if the marble position error is negative, it pushes with a positive force and vice versa.
- $K_p$  is a positive integer known as *proportional control constant*.



## Concepts of Stability

- The objective of controlling the marble is to put it at rest at the origin (set point). i.e.  $\mathbf{x} = \mathbf{0}$ , and  $\mathbf{x}' = \mathbf{0}$ , from any initial condition.
  - Or for that matter, after the marble was disturbed by some force.
  - This intuition is formalized as the notion of *stability*
  - Or more precisely, *asymptotic stability*, i.e. the error will converge to zero over time.
- The opposite of stability is *instability*, meaning that the error will grow larger and larger without bound.
  - That is, the marble will leave the origin for good.
- In-between is *marginal stability*, the error stays within some bound.

## Review

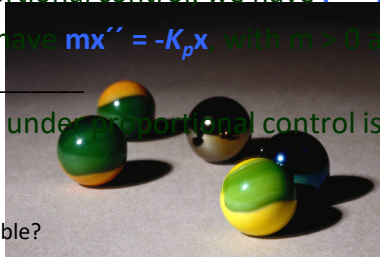
- From your calculus courses, we know that the solution of **Linear Partial Differential Equations** is a sum of exponentials:  
$$y(t) = c_1 \exp(\lambda_1 t) + c_2 \exp(\lambda_2 t) + \dots$$
  - Exponents  $\lambda_1, \lambda_2 \dots$  are known as the **eigenvalues** of the system.
  - The eigenvalues can be complex, e.g.,  $\exp(\operatorname{Re}(\lambda) t) * \exp(\operatorname{Im}(\lambda) jt)$
- Solving LPDEs using the Laplace Transform
  - $3y'' + 4y' + 5y = 0$  (replace  $n^{\text{th}}$  derivative with  $s^n$ )
  - $3s^2 + 4s + 5 = 0$
  - Solve for  $s$  and you will get the eigenvalues

## Stability and Eigenvalues

- What is the relationship between
  - The magnitude of eigenvalues and
  - The notion of (asymptotic) stability, marginal stability and instability?
- **Stability**
  - All real parts are negative
- **Instability**
  - At least one eigenvalue's real part is greater than zero.
- **Marginal stability**
  - None greater than zero but some eigenvalues equal to zero.

## Back To The Marbles

- Now, let us analyze the stability of our marble under proportional control.
- From Newton's law, we have  $F = m x''$
- From proportional control, we have  $F = -K_p x$
- Hence, we have  $m x'' = -K_p x$ , with  $m > 0$  and  $K_p > 0$ ,  
 $s = \underline{\hspace{2cm}}$
- The marble under proportional control is  
 Stable?  
 Unstable?  
 Marginally stable?
- Why?



## Marble under Proportional Control

- The system trajectories can be obtained by solving the differential equations by using Laplace transform  
 $L(x'') = s^2, L(x') = s, L(x) = 1$
- Start with  $x'' = (-K_p/m) x \Rightarrow$   
 $x'' + (K_p/m) x = 0$   
 Apply Laplace transforms  
 $s^2 + K_p/m = 0$   
 $s = \pm j \sqrt{K_p/m}$  are the eigenvalues
- As you might have guessed, the marble will oscillate with frequency  
 $\omega = \sqrt{K_p/m}$  radians/sec
- System is *marginally stable* since the real part of eigenvalue is zero



## Marble Under P + D control

$F = m x''$  // where  $m$  is the mass of the marble

$F = -K_p x$  // proportional control

$F = -K_p x - K_d x'$  // **proportional + derivative control**

$m x'' + K_d x' + K_p x = 0$  // putting it together

$m s^2 + K_d s + K_p = 0$  // applying Laplace transforms

$$s = (-K_d \pm \sqrt{K_d^2 - 4mK_p}) / 2m$$

$m > 0, K_p > 0$  and  $K_d > 0$

- Is this system stable? Why?
  - Since  $\sqrt{K_d^2 - 4mK_p} < K_d$ , the real part of  $s$  is always  $< 0$
  - It follows that the system is stable.

## Intuition

- When the position error  $x$  is positive, the **proportional control force** is negative.
  - It pushes the marble back to the origin (setpoint) at 0.
  - What will be the force and the speed when marble reaches the origin if we use only proportional control?
- When the marble is moving towards the **setpoint**, the velocity is negative. So the force due to derivative control  $-K_d x'$  is positive.
  - This counters the proportional force and slows down the marble's motion towards the origin.
- **Proportional control is like your car's gas pedal**
  - Moves the car towards the setpoint (a chosen origin)
- **Derivative control is like your brake**

## Tuning a Simple PD Controller - I

### Experimental tuning procedure

1. First set the derivative gain to zero
2. Slowly increase the proportional gain until the device oscillates around the set point with a speed that is satisfactory
  - ✓ At this point, the real part of the eigenvalue is \_\_\_\_\_
  - ✓ The imaginary part of the eigenvalue determines the \_\_\_\_\_ of oscillation
3. Slowly increase the derivative gain until the device settles down at the setpoint
  - ✓ At this point, the real part of the eigenvalue is \_\_\_\_\_

## Tuning a Simple PD Controller - II

### • Fine-tuning

- ❖ If the motion towards the setpoint is too slow, we can \_\_\_\_\_ the proportional gain **or** \_\_\_\_\_ the derivative gain
- ❖ If the motion overshoots the setpoint too much, we can \_\_\_\_\_ the derivative gain **or** \_\_\_\_\_ the proportional gain

### • Avoid using too large a proportional gain and too large a derivative gain at the same time

- This will saturate the actuator
- It is like slamming on your gas pedal and the brake together

## Integral Control

- A system often has **friction** or **changing workloads** that you may not be able to model in advance.
- In auto-cruise control, we cannot know how many passengers will be in the car or what the current slope of the road is.
- Friction may change due to machine conditions (worn-out tires, loss of engine conditioning).
- Un-modeled heavy load often results in **steady-state error**, the system will settle “near”, rather than at, the setpoint.

## What Does *Integral Control* Do?

- **Integral control** adds up (integrates) the past errors and then adds a force that is **proportional to the cumulative errors**.
- If the marble gets stuck near a setpoint due to some friction
  - The position error adds up over time
  - Use this to eventually generate a force large enough to help get the marble going.
- If the car has a heavier load and the velocity settles at a value lower than the setpoint for a while.
  - The error adds up and
  - The integral control leads to increased throttle

## Using Integral Control

- Check the eigenvalue of the system to make sure all of them are sufficiently negative.
  - $X'' = F/m$ , where  $F = -K_p x - K_d x' - K_i \int x$
  - $s^2 + (K_d/m) s + K_p/m + (K_i/m) 1/s = 0$  (because the Laplace transform of the integral is  $1/s$ )
  - $s^3 + K_d/m s^2 + K_p/m s + K_i/m = 0$
- Effect of integral is to add an **order** to the system. A large value of  $K_i$  will lead to a positive eigenvalue (bad!)
- We can solve the equation to see if the real part of the eigenvalues are still negative (best done using tools like Matlab)

## The *Dark Side* of Integral Control

- Integral control acts on *cumulative* errors
  - It takes a while to reach a large sum
  - It will also take time to reduce the sum
- Consider the following case
  - The marble gets stuck on the left side of the set-point
  - After 10 sec, the integral control is large enough to help get the marble moving
  - The integral will keep increasing until the marble crosses the origin
  - It will take a while to “wash out” the cumulative error
- An overdose of integral control is a common source of **overshoot**, **oscillation** and even **instability**.
  - What can you do to offset this?
- As a rule of thumb, start from zero and use it lightly

## How to Tune a Simple PID Controller

### Experimental tuning procedure

- First set derivative gain and integral gains to zero.
- Slowly increase the proportional gain until the device oscillates around the set point with a speed that is satisfactory.
  - At this point, the real part of the eigenvalue is \_\_\_\_\_
  - The imaginary part of the eigenvalue determines the \_\_\_\_\_ of oscillation
- Slowly increase the derivative gain until the device settles down at the setpoint
  - At this point, the real part of the eigenvalue is \_\_\_\_\_
- If there is a steady-state error, slightly increase the integral gain until the steady state error is corrected and yet does not cause serious oscillation.
  - This means that the real part of the eigenvalues are still \_\_\_\_\_

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## How to Tune a Simple PID Controller

### Fine-tuning

- If the motion towards the setpoint is too slow
  - We can \_\_\_\_\_ the proportional gain or \_\_\_\_\_ the derivative gain
  - Do *not* play with the integral gain
- If there is a steady-state error
  - We can add a little \_\_\_\_\_ gain
- If the motion overshoots the setpoint and oscillates
  - We can \_\_\_\_\_ the derivative gain or
  - We can reduce the \_\_\_\_\_ gain and the \_\_\_\_\_ gain

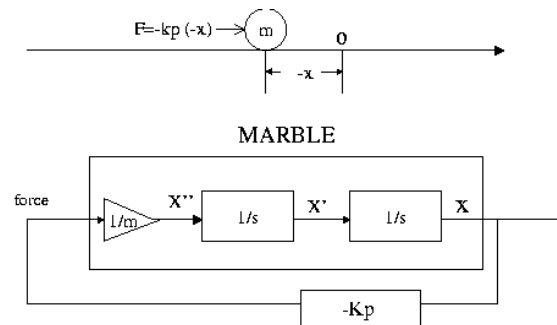
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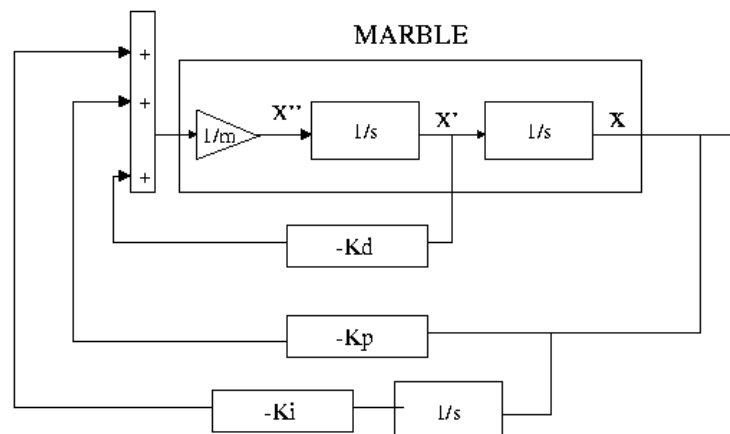
## Basic Simulation Diagram Concepts

### Proportional Control of Marble



Mass is a double-integrator with weight  $1/m$ , which transforms force to acceleration, acceleration to velocity, and then velocity to position. ( $1/s$  denotes integration)

### Marble Under PID Control



## Cruise Control – PID In Action

- Cruise control controls the speed of the car by adjusting the throttle position (throttle valve controls engine power and speed)
- A good cruise control system would accelerate aggressively to the desired speed without overshooting, and then maintain that speed with little deviation, even if you go over a hill
- **Proportional control**
  - Difference in ideal and current speeds dictates proportional control
  - Cruise control set at 60mph; car traveling at 50mph has twice the throttle opening than when it gets to 55mph
- **Derivative control** (*time derivative of speed is acceleration*)
  - While driving downhill, proportional control can lead to significant overshoot
  - If speed changes too rapidly, decrease throttle opening correspondingly
- **Integral control** (*time integral of speed is distance*)
  - Helps the car deal with hills, and settle into the correct speed
  - If car goes up a hill, proportional control helps, but may still slow the car down; the longer the car stays at a slower speed, the more the integral control, and therefore the throttle opening increases



## Summary of Lecture

- Feedback control
  - stability
  - instability
  - marginal stability
- Feedback controllers
  - proportional control
  - proportional + derivative control
  - proportional + derivative + integral control
- Simulation diagram concepts