

Mechanical Control Systems (ME 4473)

Recitation - 5

GTA: Shishir Khanal, khanshis@isu.edu

1. Agenda:

- Revision(System Representation, Block Diagram, State Space)

2. Things we learnt so far is representation of system using:

a. Differential Equations

- i. Higher order coupled differential equations
- ii. First order system of differential equations

b. Block Diagram representation

c. Transfer Function representation

Modeling using Differential Equations	
Pro	Cons

Analogous Systems:

Modeling using Block Diagrams	
Pro	Cons

Modeling using Transfer Functions	
Pro	Cons

3. Write the matlab keywords to evaluate:

- Transfer function:
- Step Response:
- Closed Loop Transfer function:

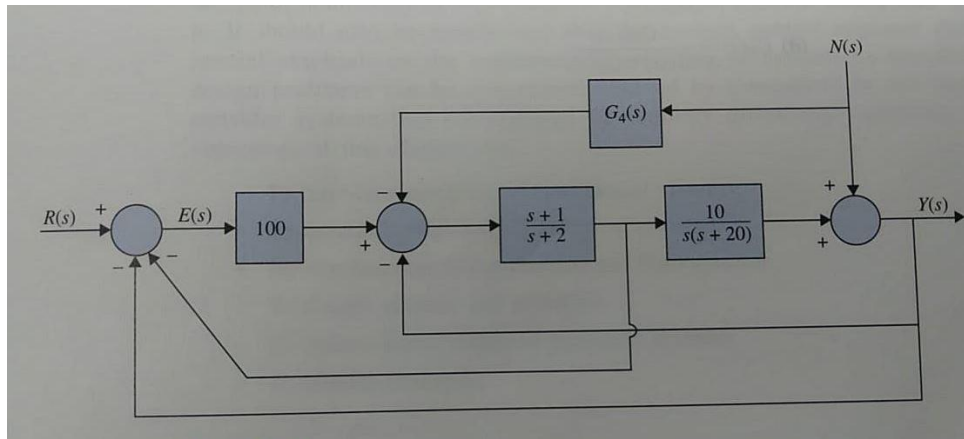
Problem-1: For a given transfer function:

$$G(s) = \frac{s(s-a)(s+b)}{(s+c)(s+d- ej)}$$

where, a, b, c, d, e <0

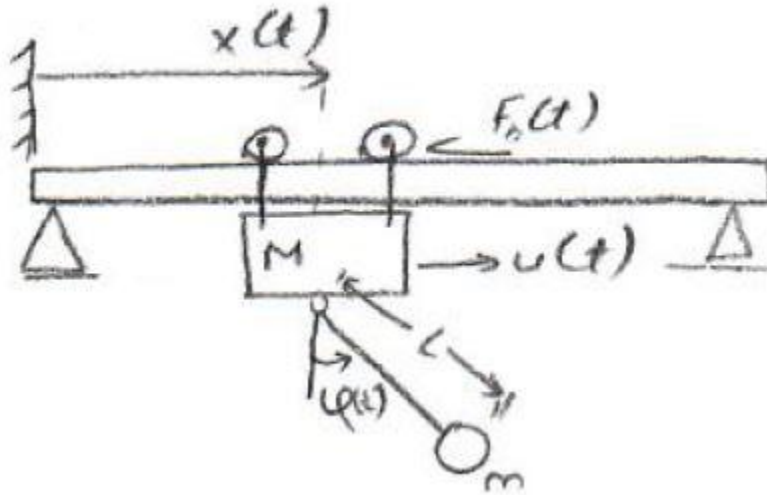
- What are the poles of the transfer function?
- What are the zeros of the transfer function?
- Represent the poles and zeros in the σ vs. $j\omega$ graph.
- Is the system stable?

Problem 2: The block diagram below represents a Feedback system with noise. Transfer function $G_4(s)$ is for the reduction of effect of Noise.

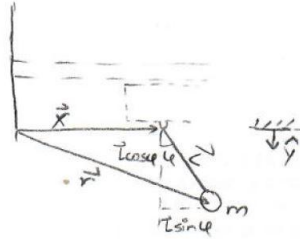
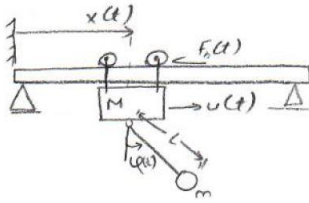


a. Evaluate the transfer function $Y(s)/R(s)$ when $N(s) = 0$

Problem-3: Consider the hanging crane structure in the fig below. The mass of the cart is M , the mass of the payload is m , massless rigid connector has length ' L ' and the friction is modeled as $F_b(t) = b\dot{x}$ where $x(t)$ is the distance travelled by the cart.

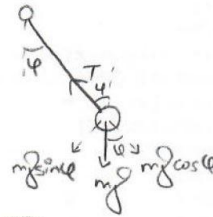
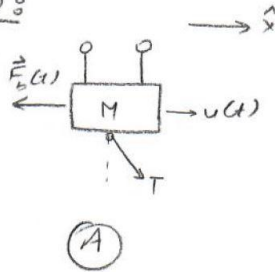


- a. Write the equations of the motion describing the motion of the cart and the payload.



(1)

FBD:



$$\vec{x}_p = \vec{x} + l \sin \phi$$

$$y_p = -l \cos \phi$$

$$\dot{x}_p = \dot{x} + l \dot{\phi} \cos \phi$$

$$\dot{y}_p = -l \dot{\phi} \sin \phi$$

$$\ddot{x}_p = \ddot{x} + l (\ddot{\phi} \cos \phi - (\dot{\phi})^2 \sin \phi)$$

$$\ddot{y}_p = -l (\ddot{\phi} \sin \phi + (\dot{\phi})^2 \cos \phi)$$

(A) $\sum F_x = m \ddot{x}$

$$\dot{u}(t) - T \sin \phi - F_b(t) = M \ddot{x}$$

$$T \sin \phi = \dot{u}(t) - M \ddot{x} - F_b(t) \quad (1)$$

(B) $\sum F_{x_p} = m \ddot{x}_p$

$$-T \sin \phi = m (\ddot{x} + l (\ddot{\phi} \cos \phi - (\dot{\phi})^2 \sin \phi)) \quad (2)$$

$$\sum F_{y_p} = m \ddot{y}_p$$

$$-T \cos \phi + mg = m l (\ddot{\phi} \sin \phi + (\dot{\phi})^2 \cos \phi) \quad (3)$$

Now, Eliminate \vec{T} from eq's,

I. Substitute $\vec{T} \sin \varphi$ in ② from ①,

$$\vec{u}(t) - M\ddot{\vec{x}} - \vec{F}_b(t) = m\ddot{\vec{x}} + mL\ddot{\varphi} \cos \varphi - mL(\dot{\varphi})^2 \sin \varphi$$

$$(M+m)\ddot{\vec{x}} + mL\ddot{\varphi} \cos \varphi - mL(\dot{\varphi})^2 \sin \varphi + \vec{F}_b(t) = \vec{u}(t) \quad \text{--- ④}$$

II Multiply ② by $\cos \varphi$ & ③ by $\sin \varphi$ & Add ②+③,

$$-T \cos \varphi \sin \varphi + m g \sin \varphi = mL\ddot{\varphi} \sin^2 \varphi + mL(\dot{\varphi})^2 \cos \varphi \sin \varphi$$

$$(+) \quad T \sin \varphi \cos \varphi = m\ddot{x} \cos \varphi + mL\ddot{\varphi} \cos^2 \varphi - mL(\dot{\varphi})^2 \sin \varphi \cos \varphi$$

$$m g \sin \varphi = mL\ddot{\varphi} \sin^2 \varphi + mL\ddot{\varphi} \cos^2 \varphi + m\ddot{x} \cos \varphi$$

$$m g \sin \varphi = mL\ddot{\varphi} + m\ddot{x} \cos \varphi$$

$$\therefore mL\ddot{\varphi} + m g \sin \varphi + m\ddot{x} \cos \varphi = 0 \quad \text{--- ⑤}$$

$$(M+m)\ddot{\vec{x}} + mL\ddot{\varphi} \cos \varphi - mL(\dot{\varphi})^2 \sin \varphi + \vec{b}\dot{\vec{x}}(t) = \vec{u}(t)$$

$$mL\ddot{\varphi} + m g \sin \varphi + m\ddot{x} \cos \varphi = 0$$

b. Linearize the system about small angles around stable equilibrium of the pendulum.

c. Decouple the linearized coupled differential equation.

d. Write the state space representation of the system.

Bibliography:

- Modern Control Systems, 13th ed
- Automatic Control Systems, 9th ed