

October 4, 2024

MODULE 6 — Assignment

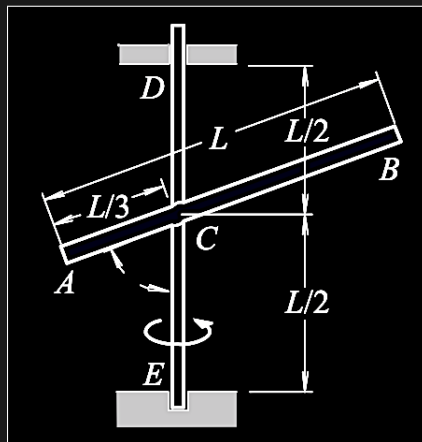
EXERCISE 5.26

Thin bar ACB is welded to a shaft that rotates at the constant angular speed Ω , so the angle θ between the bar and the shaft is constant.

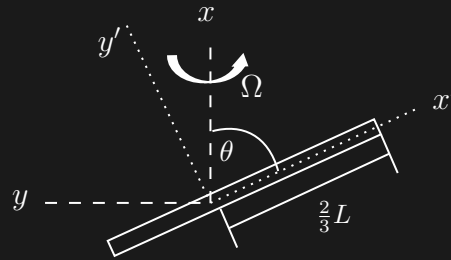
(a) Derive expressions for the angular momentum \bar{H}_C and the kinetic energy of the bar. Draw a sketch of \bar{H}_C .

(b) Based on an analysis of the manner in which \bar{H}_C in Part (a) rotates, derive an expression for $\frac{\partial}{\partial t} \bar{H}_C$.

(c) Use Eq. (5.3.4) to evaluate $\frac{\partial}{\partial t} \bar{H}_C$, and compare it with the result of Part (b).



The following sketch shows the two frames of the system:



The rotation matrix that converts the $\{xyz\}$ frame to the $\{x'y'z'\}$ frame is:

$$R = \begin{bmatrix} \cos(-\theta) & \sin(-\theta) & 0 \\ -\sin(-\theta) & \cos(-\theta) & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

Constructing the angular velocity vector:

$$\bar{\omega} = \Omega \, i$$

In the body frame is:

$$\bar{\omega} = R \, \Omega \, i = \begin{bmatrix} \Omega \cos(\theta) \, i' \\ \Omega \sin(\theta) \, j' \\ 0 \, k' \end{bmatrix}$$

From textbook Appendix, the centroidal inertia mass properties of the shaft are:

$$\begin{aligned} I_{xx} &= 0 \\ I_{yy} &= \frac{1}{12} m L^2 \\ I_{zz} &= \frac{1}{12} m L^2 \end{aligned}$$

This expressed as the inertia tensor is:

$$I_{x'y'z'} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{12} m L^2 & 0 \\ 0 & 0 & \frac{1}{12} m L^2 \end{bmatrix}$$

The distance from the body frame is:

$$d_{x'y'z'} = \begin{bmatrix} \frac{1}{6} L & 0 & 0 \end{bmatrix}$$

Using the parallel axis theorem to get the parallel axis transformation of inertia matrix relative to the center of the frame of reference in question:

$$I_{pat} = m \begin{bmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{36} L^2 & 0 \\ 0 & 0 & \frac{1}{36} L^2 \end{bmatrix}$$

This makes the inertia tensor at point C:

$$I_C = I_{x'y'z'} + I_{pat}m \begin{pmatrix} 0 & 0 & 0 \\ 0 & \frac{1}{9} L^2 & 0 \\ 0 & 0 & \frac{1}{9} L^2 \end{pmatrix}$$

From this we can compute the angular momentum as:

$$\bar{H}_C = I_C \bar{\omega} = \frac{1}{9} m L^2 \Omega, \sin(\theta) \ j'$$

→ Answer

Since the bar rotates about the x axis, the rotation component in the terminal frame that is orthogonal to the j' -axis is $\Omega \cos(\theta) \ i'$, therefore:

$$\dot{\bar{H}}_C = \frac{1}{9} m L^2 \Omega^2 \cos(\theta) \sin(\theta) \ k'$$

→ Answer

If we use Eq. (5.3.4) to evaluate this, we get the same answer:

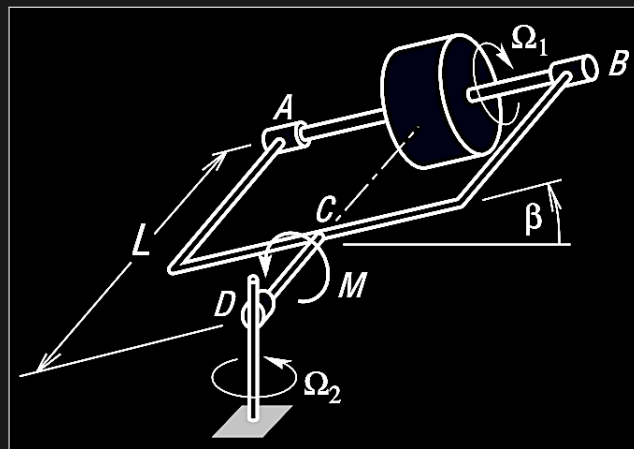
$$\dot{\bar{H}}_C = \frac{\partial}{\partial t} \bar{H}_C + \bar{\omega} \times \bar{H}_C$$

$$\begin{aligned} \dot{\bar{H}}_C &= \begin{bmatrix} 0 & i' \\ 0 & j' \\ \frac{1}{9} m L^2 \Omega^2 \cos(\theta) \sin(\theta) & k' \end{bmatrix} \\ &= \frac{1}{9} m L^2 \Omega^2 \cos(\theta) \sin(\theta) \ k' \end{aligned}$$

→ Answer

EXERCISE 6.8

The torque M acting on the gimbal of the gyroscopic turn indicator is exerted by a torsional spring, so $M = -k\beta$. The precession rate Ω_2 is a specified function of time, and the spin rate Ω_1 is held constant by a servomotor. Let I_1 denote the moment of inertia of the flywheel about axis AB , and let I_2 be the centroidal moment of inertia perpendicular to axis AB . Derive the differential equation of motion for β .



Submitted by Austin Barrilleaux on October 4, 2024.