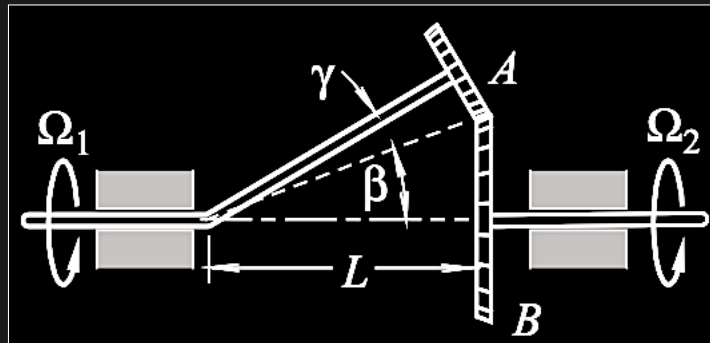


September 24, 2024

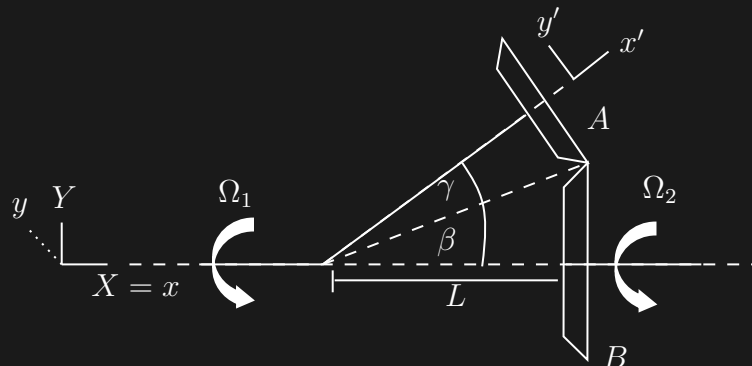
MODULE 4 — Assignment

EXERCISE 4.39

Gear A spins relative to its shaft, which rotates at variable rate Ω_1 about the horizontal axis. Gear B rotates at the variable rate Ω_2 . Determine the angular velocity and angular acceleration of gear A.



In the following sketch, we will define two coordinate frames, $\{XYZ\}$, $\{xyz\}$ which rotates with Ω_1 , and $\{x'y'z'\}$ which is rotated to align with the tilted disk from $\{xyz\}$ and rotates about the x -axis with Ω_1 :



We can write the angular velocity $\bar{\omega}$ as:

$$\bar{\omega} = \Omega_1 \bar{i} + \Omega_A \bar{i}'$$

Where Ω_A is the rotation about the horizontal bar.

We will confront this problem in the rotating $\{x'y'z'\}$ frame. The rotation from $\{xyz\}$ to $\{x'y'z'\}$ is defined by the z axis rotation as:

$$R = \begin{bmatrix} \cos(\beta + \gamma) & \sin(\beta + \gamma) & 0 \\ -\sin(\beta + \gamma) & \cos(\beta + \gamma) & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

Which we can use to define:

$$\begin{aligned} \bar{i} = R \bar{i}' &= \begin{bmatrix} \cos(\beta + \gamma) \bar{i}' \\ -\sin(\beta + \gamma) \bar{j}' \end{bmatrix} \\ \bar{j} = R \bar{j}' &= \begin{bmatrix} \sin(\beta + \gamma) \bar{i}' \\ \cos(\beta + \gamma) \bar{j}' \end{bmatrix} \end{aligned}$$

This makes:

$$\bar{\omega} = \begin{bmatrix} \{\Omega_A + \cos(\beta + \gamma) \Omega_1\} \bar{i}' \\ \{-\sin(\beta + \gamma) \Omega_1\} \bar{j}' \end{bmatrix}$$

We can solve for Ω_A , by first solving for the velocity of $v_{B\{ijk\}}$ relative to the rotation rate Ω_1 . The rotation of Ω_2 relative to Ω_1 is :

$$\Omega_{2/1} = \Omega_2 - \Omega_1$$

Therefore:

$$\begin{aligned} v_{B\{ijk\}} &= (\Omega_{2/1} \bar{i} \times L \tan(\beta) \bar{j}) \\ &= \left\{ -L \tan(\beta) (\Omega_1 - \Omega_2) \cos(\beta + \gamma)^2 - L \tan(\beta) (\Omega_1 - \Omega_2) \sin(\beta + \gamma)^2 \right\} \bar{k}' \end{aligned}$$

We can also define $v_{A\{ijk\}}$ relative to the rotation rate Ω_1

$$\begin{aligned}
v_{A_{\{ijk\}}} &= \Omega_A \bar{i}' \times L \frac{\sin(\gamma)}{\cos(\beta)} (-\bar{j}') \\
&= \left\{ -\frac{L \Omega_A \sin(\gamma)}{\cos(\beta)} \right\} \bar{k}'
\end{aligned}$$

If we equate these two velocities as they are equal due to the no slip condition, we can solve for Ω_A :

$$v_{A_{\{ijk\}}} = v_{B_{\{ijk\}}} \rightarrow \Omega_A = \frac{\sin(\beta)}{\sin(\gamma)} (\Omega_1 - \Omega_2)$$

This makes the angular velocity $\bar{\omega}$:

$$\bar{\omega} = \begin{bmatrix} \left\{ \frac{\sin(\beta)}{\sin(\gamma)} (\Omega_1 - \Omega_2) + \cos(\beta + \gamma) \Omega_1 \right\} \bar{i}' \\ \left\{ -\sin(\beta + \gamma) \Omega_1 \right\} \bar{j}' \end{bmatrix}$$

→ Answer

From this, the angular acceleration $\bar{\alpha}$ is solved for as:

$$\begin{aligned}
\bar{\alpha} &= \sum_2 (\dot{\omega}_n \bar{e}_n + \bar{\Omega}_n \times \omega_n \bar{e}_n) \\
&= \dot{\Omega}_1 \bar{i} + \cancel{\Omega_1 \bar{i} \times \Omega_1 \bar{i}} + \overset{0}{\dot{\Omega}_A \bar{i}'} + \bar{\omega} \times \Omega_A \bar{j} \\
&= \dot{\Omega}_1 \bar{i} + \frac{\sin(\beta)}{\sin(\gamma)} (\dot{\Omega}_1 - \dot{\Omega}_2) \bar{i}' + \bar{\omega} \times \frac{\sin(\beta)}{\sin(\gamma)} (\Omega_1 - \Omega_2) \bar{j}
\end{aligned}$$

This evaluated to:

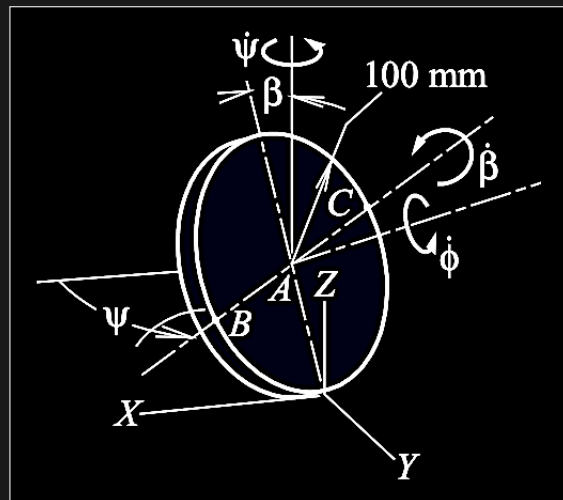
$$\bar{\alpha} = \begin{bmatrix} \left\{ \cos(\beta + \gamma) \dot{\Omega}_1 + (\dot{\Omega}_1 - \dot{\Omega}_2) \frac{\sin(\beta)}{\sin(\gamma)} \right\} \bar{i} \\ \left\{ -\sin(\beta + \gamma) \dot{\Omega}_1 \right\} \bar{j} \\ \left\{ \sin(\beta + \gamma) (\Omega_1 - \Omega_2) \Omega_1 \frac{\sin(\beta)}{\sin(\gamma)} \right\} \bar{k} \end{bmatrix}$$

→ Answer

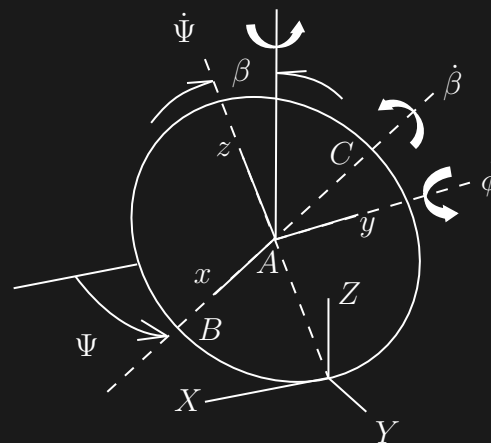
MATLAB's symbolic toolbox was leveraged in solving this problem. The script used to solve it is provided with the assignment submission as Q4_39.m.

EXERCISE 4.44

The disk rolls without slipping over the horizontal XY plane. At the instant when $\beta = 36.87^\circ$, the X and Y components of the velocity of point B on the horizontal diameter of the disk are 8 m/s and -4 m/s, respectively, and the corresponding velocity components of center A at this instant are 4 m/s and 2 m/s. Determine the precession angle Ψ between the horizontal diameter BAC and the X axis, and also evaluate the precession, nutation, and spin rates.



In the following sketch, we will define two coordinate frames, $\{XYZ\}$ and $\{xyz\}$:



We can write the angular velocity $\bar{\omega}$ as:

$$\bar{\omega} = \dot{\Psi} \bar{K} + \dot{\beta} \bar{i} + \dot{\phi} \bar{j}$$

The transformation to convert $\{XYZ\}$ to $\{xyz\}$ is:

$$\begin{aligned} R &= \begin{bmatrix} \cos(\Psi) & -\sin(\Psi) & 0 \\ \sin(\Psi) & \cos(\Psi) & 0 \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} 1 & 0 & 0 \\ 0 & \cos(\beta) & -\sin(\beta) \\ 0 & \sin(\beta) & \cos(\beta) \end{bmatrix} \\ &= \begin{bmatrix} \cos(\Psi) & -\sin(\Psi) \cos(\beta) & \sin(\Psi) \sin(\beta) \\ \sin(\Psi) & \cos(\Psi) \cos(\beta) & -\cos(\Psi) \sin(\beta) \\ 0 & \sin(\beta) & \cos(\beta) \end{bmatrix} \end{aligned}$$

From this we see that:

$$\begin{aligned} \bar{i} &= \begin{bmatrix} \cos(\Psi) \bar{I} \\ \sin(\Psi) \bar{J} \end{bmatrix} \\ \bar{j} &= \begin{bmatrix} -\sin(\Psi) \cos(\beta) \bar{I} \\ \cos(\Psi) \cos(\beta) \bar{J} \\ \sin(\beta) \bar{K} \end{bmatrix} \\ \bar{k} &= \begin{bmatrix} \sin(\Psi) \sin(\beta) \bar{I} \\ -\cos(\Psi) \sin(\beta) \bar{J} \\ \cos(\beta) \bar{K} \end{bmatrix} \end{aligned}$$

Using these, we can write the angular velocity in terms of $\{XYZ\}$:

$$\bar{\omega} = \begin{bmatrix} \left\{ \cos(\Psi) \dot{\beta} - \sin(\Psi) \cos(\beta) \dot{\phi} \right\} \bar{I} \\ \left\{ \sin(\Psi) \dot{\beta} + \cos(\Psi) \cos(\beta) \dot{\phi} \right\} \bar{J} \\ \left\{ \sin(\beta) \dot{\phi} + \dot{\Psi} \right\} \bar{K} \end{bmatrix}$$

From inspection of the sketch, we can infer that:

$$\begin{aligned}
r_{A/D} &= 0.1\bar{k} \\
&= \begin{bmatrix} \{0.1 \sin(\Psi) \sin(\beta)\} \bar{I} \\ \{-0.1 \cos(\Psi) \sin(\beta)\} \bar{J} \\ \{0.1 \cos(\beta)\} \bar{K} \end{bmatrix} \\
r_{B/D} &= 0.1(\bar{k} + \bar{i}) \\
&= \begin{bmatrix} \{0.1 \cos(\Psi) + 0.1 \sin(\Psi) \sin(\beta)\} \bar{I} \\ \{0.1 \sin(\Psi) - 0.1 \cos(\Psi) \sin(\beta)\} \bar{J} \\ \{0.1 \cos(\beta)\} \bar{K} \end{bmatrix}
\end{aligned}$$

Solving for the velocities at points A and B:

$$\begin{aligned}
\bar{v}_A &= \bar{v}_D^0 + \bar{\omega} \times \bar{r}_{A/D} \\
&= \begin{bmatrix} \left\{ 0.1 \cos(\beta) \left(\dot{\beta} \sin(\Psi) + \dot{\phi} \cos(\Psi) \cos(\beta) \right) + 0.1 \cos(\Psi) \sin(\beta) \left(\dot{\Psi} + \dot{\phi} \sin(\beta) \right) \right\} \bar{I} \\ \left\{ 0.1 \sin(\Psi) \sin(\beta) \left(\dot{\Psi} + \dot{\phi} \sin(\beta) \right) - 0.1 \cos(\beta) \left(\dot{\beta} \cos(\Psi) - \dot{\phi} \sin(\Psi) \cos(\beta) \right) \right\} \bar{J} \\ \left\{ -0.1 \dot{\beta} \sin(\beta) \right\} \bar{K} \end{bmatrix} \\
\bar{v}_B &= \bar{v}_D^0 + \bar{\omega} \times \bar{r}_{B/D} \\
&= \begin{bmatrix} \left\{ 0.1 \cos(\beta) \left(\dot{\beta} \sin(\Psi) + \dot{\phi} \cos(\Psi) \cos(\beta) \right) - (0.1 \sin(\Psi) - 0.1 \cos(\Psi) \sin(\beta)) \left(\dot{\Psi} + \dot{\phi} \sin(\beta) \right) \right\} \bar{I} \\ \left\{ (0.1 \cos(\Psi) + 0.1 \sin(\Psi) \sin(\beta)) \left(\dot{\Psi} + \dot{\phi} \sin(\beta) \right) - 0.1 \cos(\beta) \left(\dot{\beta} \cos(\Psi) - \dot{\phi} \sin(\Psi) \cos(\beta) \right) \right\} \bar{J} \\ \left\{ -0.1 \dot{\phi} \cos(\beta) - 0.1 \dot{\beta} \sin(\beta) \right\} \bar{K} \end{bmatrix}
\end{aligned}$$

Given this we can set the \bar{I} and \bar{J} components of both velocities base on the information from the problem, as well as $\beta = 36.87^\circ$:

$$\begin{aligned}
0.4 &= \left\{ 0.0640 \dot{\phi} \cos(\Psi) + 0.08 \dot{\beta} \sin(\Psi) + 0.06 \cos(\Psi) \left(\dot{\Psi} + 0.6 \dot{\phi} \right) \right\} \bar{I} \\
0.2 &= \left\{ 0.0640 \dot{\phi} \sin(\Psi) - 0.08 \dot{\beta} \cos(\Psi) + 0.06 \sin(\Psi) \left(\dot{\Psi} + 0.6 \dot{\phi} \right) \right\} \bar{J} \\
0.8 &= \left\{ 0.0640 \dot{\phi} \cos(\Psi) + 0.08 \dot{\beta} \sin(\Psi) + \left(0.06 \cos(\Psi) - 0.1 \sin(\Psi) \right) \left(\dot{\Psi} + 0.6 \dot{\phi} \right) \right\} \bar{I} \\
-0.4 &= \left\{ 0.0640 \dot{\phi} \sin(\Psi) - 0.08 \dot{\beta} \cos(\Psi) + \left(0.1 \cos(\Psi) + 0.06 \sin(\Psi) \right) \left(\dot{\Psi} + 0.6 \dot{\phi} \right) \right\} \bar{J}
\end{aligned}$$

Solving these simultaneous equations in MATLAB, the system resolves to two different answers, one of which matches the answer from the textbook:

$$\begin{aligned}
\Psi &= \mathbf{33.6901^\circ} \\
\dot{\Psi} &= \mathbf{-15.4277 \text{ rad/s}} \\
\dot{\beta} &= \mathbf{0.6934 \text{ rad/s}} \\
\dot{\phi} &= \mathbf{13.6942 \text{ rad/s}}
\end{aligned}$$

→ Answer

MATLAB's symbolic toolbox was leveraged heavily in solving this problem. The script used to solve it is provided with the assignment submission as Q4_44.m.

Submitted by Austin Barrilleaux on September 24, 2024.