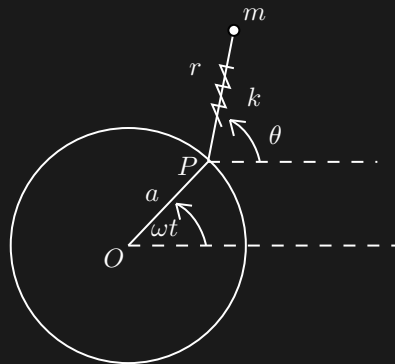


October 29, 2024

## MODULE 9 — Assignment

### Problem 1

Derive the equations of motion for a rotating spring-pendulum shown below. The spring-pendulum is attached at point  $P$ .



For this problem, the kinetic energy of the system,  $T$ , is defined as:

$$T = \frac{1}{2}mv^2$$

Therefore:

$$T = \frac{1}{2}m (\dot{x}^2 + \dot{y}^2)$$

By inspection of the sketch:

$$\begin{aligned}x &= a \cos(\omega t) + r \cos(\theta) \\y &= a \sin(\omega t) + r \sin(\theta)\end{aligned}$$

Taking the time derivative of both:

$$\begin{aligned}\dot{x} &= \dot{r} \cos(\theta) - r \dot{\theta} \sin(\theta) - a \omega \sin(\omega t) \\ \dot{y} &= \dot{r} \sin(\theta) + r \dot{\theta} \cos(\theta) + a \omega \cos(\omega t)\end{aligned}$$

This gives:

$$T = \frac{1}{2} m \left( \dot{r}^2 + r^2 \dot{\theta}^2 + a^2 \omega^2 + 2 a \omega \sin(\theta - \omega t) \dot{r} + 2 a \omega r \cos(\theta - \omega t) \dot{\theta} \right)$$

The potential energy of the system,  $V$  is defined by:

$$V = \frac{1}{2} k (r - r_0)^2$$

Given that the Lagrange is  $L = T - V$ :

$$L = \frac{1}{2} m \left( \dot{r}^2 + r^2 \dot{\theta}^2 + a^2 \omega^2 + 2 a \omega \sin(\theta - \omega t) \dot{r} + 2 a \omega r \cos(\theta - \omega t) \dot{\theta} \right) - \frac{1}{2} k (r - r_0)^2$$

We will solve for the equations of motion along the two generalized coordinates,  $\theta$  and  $r$ . Starting with  $r$ , we will solve:

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{r}} - \frac{\partial L}{\partial r} = 0$$

The component parts of this equation are:

$$\frac{\partial L}{\partial r} = m \left( r \dot{\theta}^2 + a \omega \cos(\theta - \omega t) \dot{\theta} \right) - k (r - r_0)$$

$$\frac{\partial L}{\partial \dot{r}} = m (\dot{r} + a \omega \sin(\theta - \omega t))$$

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{r}} = m \left( \ddot{r} - a \omega \cos(\theta - \omega t) (\omega - \dot{\theta}) \right)$$

This results in the equation of motion:

$$\ddot{r} - r \dot{\theta}^2 - a \omega^2 \cos(\theta - \omega t) + \frac{k}{m} (r - r_0) = 0$$

Solving along  $\theta$ :

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = 0$$

The component parts of this equation are:

$$\frac{\partial L}{\partial \theta} = m \left( a \omega \cos(\omega t - \theta) \dot{r} + a \omega \sin(\omega t - \theta) r \dot{\theta} \right)$$

$$\frac{\partial L}{\partial \dot{\theta}} = m \left( r^2 \dot{\theta} + a \omega \cos(\theta - \omega t) r \right)$$

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} = m \left( r^2 \ddot{\theta} + 2 r \dot{\theta} \dot{r} + a \omega \cos(\theta - \omega t) \dot{r} + a \omega r \sin(\theta - \omega t) (\omega - \dot{\theta}) \right)$$

This results in the equation of motion:

$$r \ddot{\theta} + 2 \dot{\theta} \dot{r} + a \omega^2 \sin(\theta - \omega t) = 0$$

The equations of motion for the system are:

$$\begin{aligned} \ddot{r} - r \dot{\theta}^2 - a \omega^2 \cos(\theta - \omega t) + \frac{k}{m} (r - r_0) &= 0 \\ r \ddot{\theta} + 2 \dot{\theta} \dot{r} + a \omega^2 \sin(\theta - \omega t) &= 0 \end{aligned}$$

→ Answer

The following MATLAB script was used to solve this problem:

```
clc,clear
syms t m a k omega r_0
syms theta(t)
syms r(t)

% Position
x = a*cos(omega*t)+r*cos(theta);
y = a*sin(omega*t)+r*sin(theta);
% velocity
x_dot = diff(x,t);
y_dot = diff(y,t);
% Kinetic Energy
T = 0.5*m*(simplify(expand(x_dot^2 + y_dot^2)));
% Kinetic energy from spring
V = 0.5*k*(r-r_0)^2;
% Lagrange Function
```

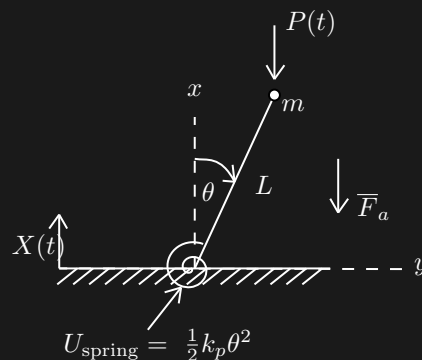
```

L = T - V;
% Coordinate derivatives
r_dot = diff(r,t);
theta_dot = diff(theta,t);
% r coordinate derivatives
dL_dr = diff(L,r);
dL_dr_dot = diff(L,r_dot);
dL_dr_dot_dt = diff(dL_dr_dot,t);
% theta coordinate derivatives
dL_dtheta = diff(L,theta);
dL_dtheta_dot = diff(L,theta_dot);
dL_dtheta_dot_dt = diff(dL_dtheta_dot,t);
% Resulting equations of motion
EOM1 = simplify(dL_dr_dot_dt-dL_dr);
EOM2 = simplify(dL_dtheta_dot_dt-dL_dtheta);

```

## Problem 2

Derive the equations of motion for the base-excited structure subject to an axial load  $P(t)$ .



For this problem, the kinetic energy of the system,  $T$ , is defined as:

$$T = \frac{1}{2} m v^2$$

Therefore:

$$T = \frac{1}{2} m (\dot{x}^2 + \dot{y}^2)$$

By inspection of the sketch:

$$\begin{aligned}x &= L \cos(\theta) + X(t) \\y &= L \sin(\theta)\end{aligned}$$

Taking the time derivative of both:

$$\begin{aligned}\dot{x} &= X'(t) - L \sin(\theta) \dot{\theta} \\ \dot{y} &= L \cos(\theta) \dot{\theta}\end{aligned}$$

This gives:

$$T = \frac{1}{2}m \left( X'(t)^2 + L^2 \dot{\theta}^2 - 2L \sin(\theta) X'(t) \dot{\theta} \right)$$

The potential energy of the system,  $V$  is defined by:

$$V = \frac{1}{2}k_p \theta^2 + m(-a)x$$

Given that the Lagrange is  $L = T - V$ :

$$L = \frac{1}{2}m \left( X'(t)^2 + L^2 \dot{\theta}^2 - 2L \sin(\theta) X'(t) \dot{\theta} \right) - \left( \frac{1}{2}k_p \theta^2 - ma (L \cos(\theta) + X(t)) \right)$$

We will solve for the equation of motion along the generalized coordinate,  $\theta$ :

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = Q_\theta$$

The component parts of this equation are:

$$\frac{\partial L}{\partial \theta} = -L m a \sin(\theta) - L m \cos(\theta) \dot{\theta} X'(t) - k_p \theta$$

$$\frac{\partial L}{\partial \dot{\theta}} = m L^2 \dot{\theta} - m L \sin(\theta) X'(t)$$

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} = m L^2 \ddot{\theta} - m \sin(\theta) L X''(t) - m \cos(\theta) \dot{\theta} L X'(t)$$

Where  $r = [x \ y]^T$ , the external force is computed as:

$$\begin{aligned}
 Q_\theta &= \sum_{k=1}^{N=3} F_k \frac{\partial x_k}{\partial \theta} \\
 &= \begin{bmatrix} 0 \\ -P(t) \end{bmatrix} \cdot \begin{bmatrix} L \cos(\theta) \\ -L \sin(\theta) \end{bmatrix} \\
 &= L \sin(\theta(t)) P(t)
 \end{aligned}$$

This results in the system equation of motion:

$$m L^2 \ddot{\theta} - m L \sin(\theta) X''(t) + k_p \theta + m a L \sin(\theta) = P(t) L \sin(\theta)$$

Or:

$$m L^2 \ddot{\theta} + [a - X''(t)] m L \sin(\theta) + k_p \theta = P(t) L \sin(\theta)$$

→ Answer

The following MATLAB script was used to solve this problem:

```

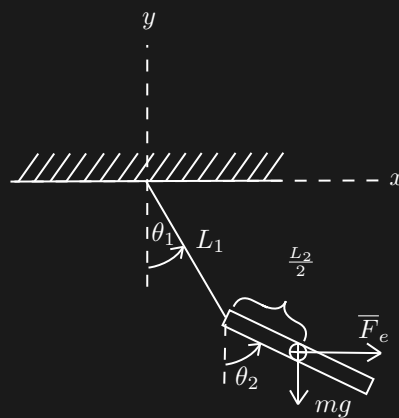
clc,clear
syms t m L k_p a X(t)
syms theta(t)
% Position
y = L*sin(theta);
x = L*cos(theta)+X;
% velocity
x_dot = diff(x,t);
y_dot = diff(y,t);
% Kinetic Energy
T = 0.5*m*(simplify(expand(x_dot^2 + y_dot^2)));
% Kinetic energy from spring
V = 0.5*k_p*theta^2 + m*(-a)*x;
% Lagrange Function
L = T - V;
% Coordinate derivatives
theta_dot = diff(theta,t);
% theta coordinate derivatives
dL_dtheta = diff(L,theta);
dL_dtheta_dot = diff(L,theta_dot);
dL_dtheta_dot_dt = simplify(diff(dL_dtheta_dot,t));

```

```
% Resulting equations of motion excluding external forces
EOM = simplify(dL_dtheta_dot_dt-dL_dtheta);
% External force:
syms P(t); assume(P(t),'real')
r = [x;y];
dr_dtheta = diff(r,theta);
F = (P*[-1;0])' * dr_dtheta; % dot product
```

### Problem 3

The double pendulum. Find the equations governing a rigid bar attached to a string as shown in the figure.



a. This system starts with 9 degrees of freedom. What constraints exist that reduce this system to only two degrees of freedom?

The system is composed of two objects; a mass particle and a rigid body. The mass,  $m_1$ , particle has three translational dimensions,  $x_1$ ,  $y_1$  and  $z_1$ . The rigid body,  $m_2$ , particle has three translational dimensions,  $x_2$ ,  $y_2$  and  $z_2$ , and three rotational dimensions,  $R_{x_2}$ ,  $R_{y_2}$  and  $R_{z_2}$ .

Because this is a 2-D problem, the following four constraints are inherent:

$$z_1 = z_2 = R_{x_2} = R_{y_2} = C = 0$$

Via inspection of the diagram we can see the following 3 constraints:

$$L_1^2 = x_1^2 + y_1^2$$

$$\left(\frac{L_1}{2}\right)^2 = (x_2 - x_1)^2 + (y_2 - y_1)^2$$

$$R_{z_2} = \theta_2 = \tan^{-1} \left( \frac{x_2 - x_1}{y_2 - y_1} \right)$$

With **9 degrees of freedom** for the **unconstrained system** and **7 constraints**, the system has **2 degrees of freedom**.

→ Answer

**b. Show that the Lagrangian is:**

$$L = \frac{1}{2}m \left[ L_1^2 \dot{\theta}_1^2 + L_1 L_2 \dot{\theta}_1 \dot{\theta}_2 \cos(\theta_2 - \theta_1) + \frac{1}{3} L_2^2 \dot{\theta}_2^2 \right] - mg \left[ L_1 (1 - \cos \theta_1) + \frac{L_2}{2} (1 - \cos \theta_2) \right]$$

For this problem, the kinetic energy of the system,  $T$ , is defined as:

$$T = \frac{1}{2}mv^2 + \frac{1}{2}I\dot{\theta}_2^2$$

Therefore:

$$T = \frac{1}{2}m (\dot{x}^2 + \dot{y}^2) + \frac{1}{2} \left( \frac{1}{12} \right) L_2^2 \dot{\theta}_2^2$$

By inspection of the sketch:

$$\begin{aligned} x &= L_1 \sin(\theta_1) + \frac{L_2}{2} \sin(\theta_2) \\ y &= \left( L_1 + \frac{L_2}{2} \right) - L_1 \cos(\theta_1) - \frac{L_2}{2} \cos(\theta_2) \end{aligned}$$

Note that these equations define  $(x, y)_0$  at the location of the c.g. of the bar when  $\theta_1 = \theta_2 = 0$ .

Taking the time derivative of both:



$$\begin{aligned}\dot{x} &= L_1 \cos(\theta_1) \dot{\theta}_1 + \frac{1}{2} L_2 \cos(\theta_2) \dot{\theta}_2 \\ \dot{y} &= L_1 \sin(\theta_1) \dot{\theta}_1 + \frac{1}{2} L_2 \sin(\theta_2) \dot{\theta}_2\end{aligned}$$

This gives:

$$T = \frac{1}{2} m \left[ L_1^2 \dot{\theta}_1^2 + L_1 L_2 \dot{\theta}_2 \dot{\theta}_1 \cos(\theta_2 - \theta_1) + \frac{1}{3} L_2^2 \dot{\theta}_2^2 \right]$$

The potential energy of the system,  $V$  is defined by:

$$V = mgh$$

Therefore:

$$\begin{aligned}V &= mg \left( L_1 + \frac{1}{2} L_2 - L_1 \cos(\theta_1) - \frac{1}{2} L_2 \cos(\theta_2) \right) \\ &= mg \left[ L_1 (1 - \cos(\theta_1)) + \frac{1}{2} L_2 (1 - \cos(\theta_2)) \right]\end{aligned}$$

Given that the Lagrange is  $L = T - V$ :

$$\begin{aligned}L &= \frac{1}{2} m \left[ L_1^2 \dot{\theta}_1^2 + L_1 L_2 \dot{\theta}_2 \dot{\theta}_1 \cos(\theta_2 - \theta_1) + \frac{1}{3} L_2^2 \dot{\theta}_2^2 \right] \\ &\quad - mg \left[ L_1 (1 - \cos(\theta_1)) + \frac{1}{2} L_2 (1 - \cos(\theta_2)) \right]\end{aligned}$$

→ Answer

This matches the Lagrangian in the question prompt.

**c. Use Lagrange's equations to derive the equations of motion (don't forget about the external force!)**

We will solve for the equations of motion along the two generalized coordinates,  $\theta_1$  and  $\theta_2$ . Starting with  $\theta_1$ , we will solve:

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}_1} - \frac{\partial L}{\partial \theta_1} = Q_{\theta_1}$$

The component parts of this equation are:

$$\frac{\partial L}{\partial \theta_1} = \frac{1}{2} L_1 L_2 m \sin(\theta_2 - \theta_1) \dot{\theta}_2 \dot{\theta}_1 - L_1 g m \sin(\theta_1)$$

$$\frac{\partial L}{\partial \dot{\theta}_1} = \frac{1}{2} m \left( 2 L_1^2 \dot{\theta}_1 + L_2 \cos(\theta_1 - \theta_2) L_1 \dot{\theta}_2 \right)$$

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}_1} = \frac{1}{2} m \left( 2 L_1^2 \ddot{\theta}_1 + L_1 L_2 \cos(\theta_1 - \theta_2) \ddot{\theta}_2 - L_1 L_2 \sin(\theta_1 - \theta_2) (\dot{\theta}_1 - \dot{\theta}_2) \dot{\theta}_2 \right)$$

Where  $r = [x \ y]^T$ , solving for the external force:

$$\begin{aligned} Q_{\theta_1} &= \sum_{k=1}^{N=3} F_{e_k} \frac{\partial x_k}{\partial \theta_1} \\ &= \begin{bmatrix} F_e \\ 0 \end{bmatrix} \cdot \begin{bmatrix} L_1 \cos(\theta_1) \\ L_1 \sin(\theta_1) \end{bmatrix} \\ &= F_e L_1 \cos(\theta_1) \end{aligned}$$

This results in the equation of motion:

$$m L_1^2 \ddot{\theta}_1 + \frac{1}{2} L_2 m \cos(\theta_1 - \theta_2) L_1 \ddot{\theta}_2 + \frac{1}{2} L_2 m \sin(\theta_1 - \theta_2) L_1 \dot{\theta}_2^2 + g m \sin(\theta_1) L_1 = F_e L_1 \cos(\theta_1)$$

Solving along  $\theta_2$ , we will solve:

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}_2} - \frac{\partial L}{\partial \theta_2} = Q_{\theta_2}$$

The component parts of this equation are:

$$\frac{\partial L}{\partial \theta_2} = \frac{1}{2} L_1 L_2 m \sin(\theta_1 - \theta_2) \dot{\theta}_2 \dot{\theta}_1 - \frac{1}{2} L_2 g m \sin(\theta_2)$$

$$\frac{\partial L}{\partial \dot{\theta}_2} = \frac{1}{2} m \left( \frac{2}{3} L_2^2 \dot{\theta}_2 + L_1 \cos(\theta_1 - \theta_2) L_2 \dot{\theta}_1 \right)$$

$$\frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}_2} = \frac{1}{2} m \left( \frac{2}{3} L_2^2 \ddot{\theta}_2 + L_1 L_2 \cos(\theta_1 - \theta_2) \ddot{\theta}_1 - L_1 L_2 \sin(\theta_1 - \theta_2) (\dot{\theta}_1 - \dot{\theta}_2) \dot{\theta}_1 \right)$$

Solving for the external force:

$$\begin{aligned}
 Q_{\theta_2} &= \sum_{k=1}^{N=3} F_{e_k} \frac{\partial x_k}{\partial \theta_2} \\
 &= \begin{bmatrix} F_e \\ 0 \end{bmatrix} \cdot \begin{bmatrix} \frac{1}{2} L_2 \cos(\theta_2) \\ \frac{1}{2} L_2 \sin(\theta_2) \end{bmatrix} \\
 &= \frac{1}{2} F_e L_2 \cos(\theta_2)
 \end{aligned}$$

This results in the equation of motion:

$$\begin{aligned}
 \frac{1}{3} m L_2^2 \ddot{\theta}_2 + \frac{1}{2} L_1 m \cos(\theta_1 - \theta_2) L_2 \ddot{\theta}_1 - \frac{1}{2} L_1 m \sin(\theta_1 - \theta_2) L_2 \dot{\theta}_1^2 + \frac{1}{2} g m \sin(\theta_2) L_2 \\
 = \frac{1}{2} F_e L_2 \cos(\theta_2)
 \end{aligned}$$

All together, the equations of motion for the system are:

$$\begin{aligned}
 m L_1^2 \ddot{\theta}_1 + \frac{1}{2} L_2 m \cos(\theta_1 - \theta_2) L_1 \ddot{\theta}_2 + \frac{1}{2} L_2 m \sin(\theta_1 - \theta_2) L_1 \dot{\theta}_2^2 + g m \sin(\theta_1) L_1 \\
 = F_e L_1 \cos(\theta_1) \\
 \frac{2}{3} m L_2^2 \ddot{\theta}_2 + L_1 m \cos(\theta_1 - \theta_2) L_2 \ddot{\theta}_1 - L_1 m \sin(\theta_1 - \theta_2) L_2 \dot{\theta}_1^2 + g m \sin(\theta_2) L_2 \\
 = F_e L_2 \cos(\theta_2) \\
 \longrightarrow \text{Answer}
 \end{aligned}$$

The following MATLAB script was used to solve this problem:

```

clc,clear
syms t m g
syms L_1 L_2 theta_1(t) theta_2(t)
% Position
x = L_1*sin(theta_1) + L_2/2*sin(theta_2);
y = (L_1 + L_2/2) - L_1*cos(theta_1) - L_2/2*cos(theta_2);
% Potential Energy
V = m*g*(y);

```

```

V = collect(V,[L_1,L_2]);
% Velocity
x_dot = diff(x,t);
y_dot = diff(y,t);
% Kinetic Energy
T = 0.5*m*(simplify(expand(x_dot^2 + y_dot^2)) +...
    (1/12)*L_2^2*diff(theta_2,t)^2);
% Lagrange Function
L = T-V;
% Coordinate derivatives
theta_1_dot = diff(theta_1,t);
theta_2_dot = diff(theta_2,t);
% theta_1 coordinate derivatives
dL_dtheta_1 = diff(L,theta_1);
dL_dtheta_1_dot = diff(L,theta_1_dot);
dL_dtheta_1_dot_dt = simplify(diff(dL_dtheta_1_dot,t));
% theta_2 coordinate derivatives
dL_dtheta_2 = diff(L,theta_2);
dL_dtheta_2_dot = diff(L,theta_2_dot);
dL_dtheta_2_dot_dt = simplify(diff(dL_dtheta_2_dot,t));
% Resulting equations of motion excluding external forces
EOM_1 = simplify(dL_dtheta_1_dot_dt-dL_dtheta_1);
EOM_2 = simplify(dL_dtheta_2_dot_dt-dL_dtheta_2);
% Compute external forces:
syms F_e; assume(F_e,'real')
r = [x;y];
% Partial With respect to theta_1
dr_dtheta_1 = diff(r,theta_1);
Q_theta_1 = (F_e*[1;0])' * dr_dtheta_1; % dot product
% Partial With respect to theta_2
dr_dtheta_2 = diff(r,theta_2);
Q_theta_2 = (F_e*[1;0])' * dr_dtheta_2; % dot product

```

*Submitted by Austin Barrilleaux on October 29, 2024.*