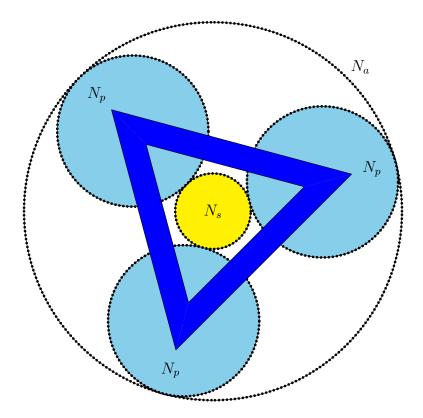
The system

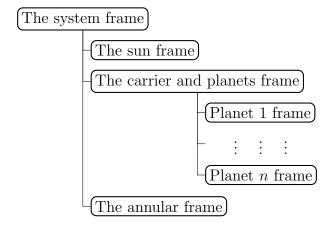
The system consists of the following components:

- \bullet "Sun gear" in centre, with N_s teeth and angular velocity $\omega_s.$
- "Planet gears" surrounding sun gear, with N_p teeth and angular velocity ω_p .
- "Annular gear" surrounding planet gears, with N_a teeth and angular velocity ω_a .
- "Planet carrier" which is fixed to each planet's axle with angular velocity ω_c .



Input and output can be connected to any of the gears, or to the carrier. If a planet gear is directly used for input or output, the system reduces to a simple 2-gear system.

In terms of reference frames, we have:



Solving

Fixed carrier

In the case where the carrier is fixed, gear ratios and thus rotation rates are trivial to obtain:

$$\omega_s : \omega_p = N_p : -N_s$$

$$\omega_p : \omega_a = N_a : N_p$$

$$\omega_s : \omega_a = N_a : -N_s$$

$$\therefore \ \omega_s : \omega_p : \omega_a : \omega_c = 1 : -\frac{N_s}{N_p} : -\frac{N_s}{N_a} : 0$$

Note: all rotation rates are relative to orientation of parent frame.

Fixed annular gear

To fix the annular gear instead of the carrier, we require that ω_c become free and that $\omega_a = 0$. This is achieved by applying a rotation to the top-level frames to cancel out the rotation of the annular gear's frame. We calculate the new rotation rates ω_x for all child frames of the system frame:

$$\omega_x o \omega_x' - \left(-\frac{N_s}{N_a}
ight)$$
 only for ω_x that are relative to the system frame

Note that ω_p is not changed since it is measured relative to the carrier frame, not the system frame.

$$\omega_s : \omega_p : \omega_a : \omega_c = 1 + \frac{N_s}{N_a} : -\frac{N_s}{N_p} : 0 : \frac{N_s}{N_a}$$
$$= \frac{N_a + N_s}{N_a} : -\frac{N_s}{N_p} : 0 : \frac{N_s}{N_a}$$

For convenience, normalize the sun gear:

$$\omega_s : \omega_p : \omega_a : \omega_c = 1 : -\frac{N_s N_a}{N_n (N_a + N_s)} : 0 : \frac{N_s}{N_a + N_s}$$