

Assumption:

① incompressible flow: $\operatorname{div} V = 0$, $V = (u, v)$ velocity field.
 $\Rightarrow \frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$

② irrotational flow, no vortices: $\operatorname{rot} V = 0 \Rightarrow \frac{\partial u}{\partial y} - \frac{\partial v}{\partial x} = 0$
scalar function $\psi(x, y)$:

$$u = \frac{\partial \psi}{\partial y}, \quad v = -\frac{\partial \psi}{\partial x}$$

Then ① is identically satisfied and ② gives:

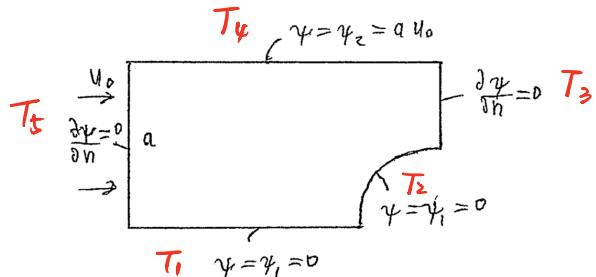
$$\frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} = 0 \quad (\text{Laplace eq.})$$

ψ is the stream function

$$\left. \begin{aligned} V = (u, v) &= \left(\frac{\partial \psi}{\partial y}, -\frac{\partial \psi}{\partial x} \right) \\ \nabla \psi &= \left(\frac{\partial \psi}{\partial x}, \frac{\partial \psi}{\partial y} \right) \end{aligned} \right\} \Rightarrow V \cdot \nabla \psi = 0 \quad \text{But } \nabla \psi \perp \psi \text{ contours} \\ V \text{ is tangential to contours,} \\ \text{i.e. } \psi \text{ contours are stream lines.} \end{math>$$

The boundary conditions are

$$\left. \begin{aligned} \frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} &= 0 & \textcircled{1} \\ \psi = \psi_1 &= 0 \text{ on } T_1 \text{ and } T_2 & \textcircled{2} \\ \psi = \psi_2 &= \text{all } 0 \text{ on } T_4 & \textcircled{3} \\ \frac{\partial \psi}{\partial n} &= 0 \text{ on } T_3 \text{ and } T_5 & \textcircled{4} \end{aligned} \right.$$



Weak formulation:

$$0 = \iint_{\Omega} \left(\frac{\partial^2 \psi}{\partial x^2} + \frac{\partial^2 \psi}{\partial y^2} \right) V \, dx \, dy \quad \text{where } V \text{ is arbitrary test function}$$

$$\text{Gauss-Green} \iint_{\Omega} \left(-\frac{\partial \psi}{\partial x} \cdot \frac{\partial V}{\partial x} - \frac{\partial \psi}{\partial y} \cdot \frac{\partial V}{\partial y} \right) \, dx \, dy + \int_T \left(\frac{\partial \psi}{\partial x} n_x + \frac{\partial \psi}{\partial y} n_y \right) V \, dS = 0$$

$$= \iint_{\Omega} \left(-\frac{\partial \psi}{\partial x} \cdot \frac{\partial V}{\partial x} - \frac{\partial \psi}{\partial y} \cdot \frac{\partial V}{\partial y} \right) \, dx \, dy + \int_T \frac{\partial \psi}{\partial n} V \, dS \quad \begin{matrix} \text{(in the outward-point} \\ \text{unit normal)} \end{matrix}$$

Apply the boundary conditions to weak formulation:

$$\left\{ \begin{aligned} \text{from ①: } & \iint_{\Omega} \left(\frac{\partial \psi}{\partial x} \cdot \frac{\partial V}{\partial x} + \frac{\partial \psi}{\partial y} \cdot \frac{\partial V}{\partial y} \right) \, dx \, dy - \int_{T_3} \frac{\partial \psi}{\partial n} V \, dS - \int_{T_5} \frac{\partial \psi}{\partial n} V \, dS = 0 \\ \text{from ②: } & \psi = \psi_1 = 0 \text{ on } T_1 \text{ and } T_2 \text{ for all } V \text{ such that } V = 0 \text{ on } T_1, T_2 \\ \text{from ④: } & \psi = \psi_2 = \text{all } 0 \text{ on } T_4 \text{ for all } V \text{ such that } V = 0 \text{ on } T_4 \end{aligned} \right.$$

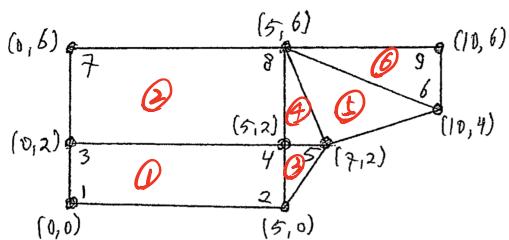
According to the FE eq.: $k_u = f_e + f_b$ where

$$K_{ij} = \iint_{\Omega} \left(\frac{\partial N_i}{\partial x} \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial N_j}{\partial y} \right) dx dy \quad (\text{stiffness matrix})$$

$$f_{li} = 0 \quad (\text{load vector})$$

$$f_{bi} = \int_{T_3} \frac{\partial \psi}{\partial n} N_i ds + \int_{T_f} \frac{\partial \psi}{\partial n} N_i ds \quad (\text{boundary vector})$$

Divide the domain of the flow into 6 parts:



From the graph, we know that
 ① ② are 4-node rectangular
 ③ ④ ⑤ ⑥ are 3-node triangular

To calculate the 4-node rectangular element

$$\text{Interpolation: } \psi = \sum_{i=1}^4 N_i \psi_i \quad \text{FE equation: } K g = f_b \quad g = [u_1, u_2, u_3]^T$$

$$\text{Galerkin: } V_i = N_i \quad (i=1..4)$$

Element ①

$$4(0,2) \quad 3(5,2)$$

$$2a=2, \quad 2b=5 \quad 4ab=10$$

Shape functions:

$$N_1(x,y) = \frac{1}{4ab} (x-x_1)(y-y_1) = \frac{1}{10} (x-0)(y-0) = \frac{xy}{10} - \frac{x}{5} - \frac{y}{2} + 1$$

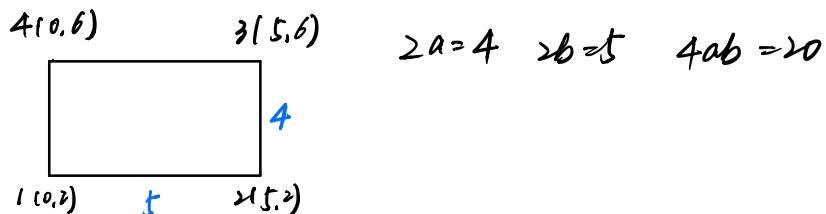
$$N_2(x,y) = -\frac{1}{4ab} (x-x_2)(y-y_2) = -\frac{1}{10} x (y-2) = -\frac{xy}{10} + \frac{x}{5}$$

$$N_3(x,y) = -\frac{1}{4ab} (x-x_3)(y-y_3) = \frac{1}{10} xy$$

$$N_4(x,y) = \frac{1}{4ab} (x-x_4)(y-y_4) = -\frac{1}{10} (x-5)(y-0) = -\frac{xy}{10} + \frac{y}{2}$$

$$K' = \begin{pmatrix} \frac{29}{30} & \frac{17}{60} & -\frac{23}{30} & -\frac{29}{60} \\ \frac{17}{60} & \frac{29}{30} & -\frac{29}{60} & -\frac{23}{20} \\ -\frac{29}{30} & -\frac{23}{60} & \frac{29}{30} & \frac{17}{60} \\ -\frac{23}{60} & -\frac{29}{30} & \frac{17}{60} & \frac{29}{30} \end{pmatrix} \quad u' = \begin{pmatrix} \psi_1 \\ \psi_2 \\ \psi_3 \\ \psi_4 \end{pmatrix}$$

Element ②:



Shape functions:

$$N_1^e(x,y) = \frac{1}{4ab} (x-x_1)(y-y_1) = \frac{1}{30} (x-5)(y-6) = \frac{1}{30} xy - \frac{3}{10} x - \frac{1}{4} y + \frac{3}{2}$$

$$N_2^e(x,y) = -\frac{1}{4ab} (x-x_1)(y-y_2) = -\frac{1}{30} xy - 6 = -\frac{1}{30} xy + \frac{3}{10} x$$

$$N_3^e(x,y) = -\frac{1}{4ab} (x-x_2)(y-y_1) = \frac{1}{30} xy - 2 = \frac{1}{30} xy - \frac{1}{10} x$$

$$N_4^e(x,y) = \frac{1}{4ab} (x-x_2)(y-y_2) = -\frac{1}{30} (x-5)(y-2) = -\frac{1}{30} xy + \frac{1}{10} x + \frac{1}{4} y - \frac{1}{2}$$

According to the stiffness matrix: $k_{ij} = \iint_R \left(\frac{\partial N_i}{\partial x} \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial N_j}{\partial y} \right) dx dy$

$$K^e = \begin{pmatrix} \frac{41}{60} & -\frac{7}{120} & -\frac{41}{120} & -\frac{17}{60} \\ -\frac{7}{120} & \frac{41}{60} & -\frac{17}{60} & -\frac{41}{120} \\ -\frac{41}{120} & -\frac{17}{60} & \frac{41}{60} & -\frac{7}{120} \\ -\frac{17}{60} & -\frac{41}{120} & -\frac{7}{120} & \frac{41}{60} \end{pmatrix} \quad u^e = \begin{pmatrix} \psi_3 \\ \psi_4 \\ \psi_1 \\ \psi_2 \end{pmatrix}$$

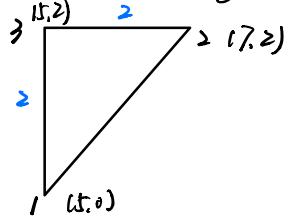
To calculate the 3-Node rectangular elements

The shape functions for a 3-Node triangular:

$$N_i^e(x,y) = \frac{1}{2A} [x_j y_k - x_k y_j + (y_j - y_k)x + (x_k - x_j)y] \quad (i,j,k \text{ cyclic})$$

where (x_i, y_i) are the coordinates of the i -th node and A is the area of the element.

Element ③



$$A = \frac{1}{2} \times 2 \times 2 = 2$$

$$\text{Shape functions: } N_1^e(x,y) = \frac{1}{2A} [x_2 y_3 - x_3 y_2 + (y_2 - y_3)x + (x_3 - x_2)y]$$

$$= \frac{1}{4} [2x7 - 2x5 + (-y)] = 1 - \frac{1}{2}y$$

$$N_2^e(x,y) = \frac{1}{2A} [x_3 y_1 - x_1 y_3 + (y_3 - y_1)x + (x_1 - x_3)y]$$

$$= \frac{1}{4} [5x0 - 5x2 + 2x] = \frac{1}{2}x - \frac{5}{2}$$

$$N_3^e(x,y) = \frac{1}{2A} [x_1 y_2 - x_2 y_1 + (y_1 - y_2)x + (x_2 - x_1)y]$$

$$= \frac{1}{4} [5x2 - 7x0 - 2x + 2y] = \frac{5}{2} - \frac{x}{2} + \frac{y}{2}$$

According to the stiffness matrix: $k_{ij} = \iint \left(\frac{\partial N_i}{\partial x} \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial N_j}{\partial y} \right) dx dy$

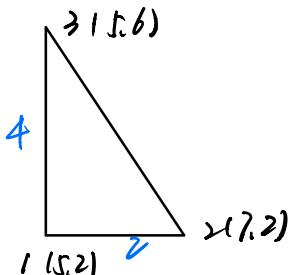
$$k_{11} = \iint \frac{1}{4} dx dy = \frac{1}{2} = k_{22}$$

$$k_{12} = k_{21} = 0$$

$$k_{13} = k_{31} = \iint (0 - \frac{1}{4}) dx dy = -\frac{1}{2} = k_{23} = k_{32}$$

$$k_{33} = \iint (\frac{1}{4} + \frac{1}{4}) dx dy = 1$$

Element ④:



Shape functions:

$$N_1^e(x,y) = \frac{1}{2A} [x_2 y_3 - x_3 y_2 + (y_2 - y_3)x + (x_3 - x_2)y]$$

$$= \frac{1}{8} [7x6 - 5x2 + (1-4x) + (-2y)] = 4 - \frac{1}{2}x - \frac{1}{4}y$$

$$N_2^e(x,y) = \frac{1}{2A} [x_3 y_1 - x_1 y_3 + (y_3 - y_1)x + (x_1 - x_3)y]$$

$$= -\frac{5}{2} + \frac{1}{2}x$$

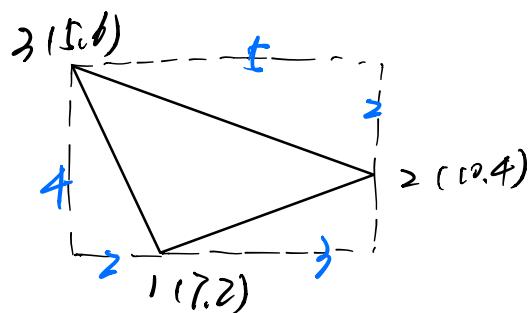
$$N_1^e(x,y) = \frac{1}{2A} [x_1y_2 - x_2y_1 + (y_1 - y_2)x + (x_2 - x_1)y]$$

$$= -\frac{1}{2} + \frac{1}{4}y$$

Similarly, we can get $N_{i,j}$

$$K^4 = \begin{pmatrix} \frac{1}{4} & -1 & -\frac{1}{4} \\ -1 & -1 & 0 \\ -\frac{1}{4} & 0 & \frac{1}{4} \end{pmatrix} \quad u^4 = \begin{pmatrix} \psi_4 \\ \psi_5 \\ \psi_8 \end{pmatrix}$$

Element ⑤



$$A = (2+3)\times 4 - \frac{1}{2} \times 2 \times 4 - \frac{1}{2} \times 2 \times 3 - \frac{1}{2} \times 2 \times 5$$

$$= 20 - 12 = 8$$

shape functions:

$$N_1^e(x,y) = \frac{5}{8} - \frac{1}{8}x - \frac{5}{16}y$$

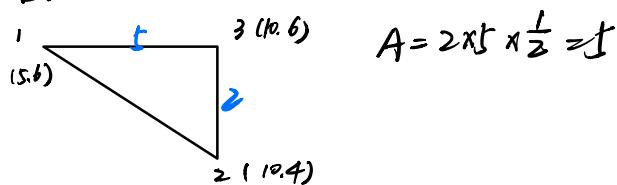
$$N_2^e(x,y) = -2 + \frac{1}{4}x + \frac{1}{8}y$$

$$N_3^e(x,y) = \frac{1}{2} - \frac{1}{8}x + \frac{3}{16}y$$

$$\text{get } K_{ij} = \iint \left(\frac{\partial N_i}{\partial x} \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial N_j}{\partial y} \right) dx dy$$

$$K^5 = \begin{pmatrix} \frac{9}{32} & -\frac{9}{16} & -\frac{11}{32} \\ -\frac{9}{16} & \frac{5}{8} & -\frac{1}{16} \\ -\frac{11}{32} & -\frac{1}{16} & \frac{13}{32} \end{pmatrix} \quad u^5 = \begin{pmatrix} \psi_5 \\ \psi_6 \\ \psi_8 \end{pmatrix}$$

Element ⑥



$$A = 2 \times 5 \times \frac{1}{2} = 5$$

Shape functions:

$$N_1^e(x,y) = \frac{1}{10}(10x6 - 10x4 + 10x8) = 2 - \frac{1}{5}x$$

$$N_2^e(x,y) = \frac{1}{10}(10x6 - 5x6 - 5y) = 3 - \frac{1}{2}y$$

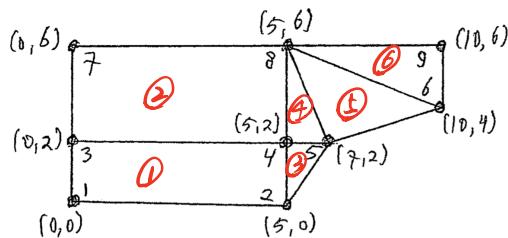
$$N_3^e(x,y) = \frac{1}{10}(5x4 - 10x6 + 20x5y) = -4 + \frac{1}{5}x + \frac{1}{2}y$$

Similarly, get stiffness matrix K_{ij} .

$$K^e = \begin{pmatrix} \frac{1}{5} & 0 & -\frac{1}{5} \\ 0 & \frac{5}{4} & -\frac{5}{4} \\ -\frac{1}{5} & -\frac{5}{4} & \frac{29}{20} \end{pmatrix} \quad u^e = \begin{pmatrix} \varphi_1 \\ \varphi_2 \\ \varphi_3 \end{pmatrix}$$

To assemble the element data into global data.

Connectivity matrix C :



$$C = \begin{pmatrix} 1 & 2 & 4 & 3 \\ 3 & 4 & 8 & 7 \\ 2 & 5 & 4 & 0 \\ 4 & 5 & 8 & 0 \\ 5 & 6 & 8 & 0 \\ 8 & 6 & 9 & 0 \end{pmatrix}$$

Sum over elements: $Ku = f_l$.

global stiffness matrix:

$$K = \begin{bmatrix} \frac{29}{30} & \frac{17}{60} & -\frac{23}{20} & -\frac{29}{60} & 0 & 0 & 0 & 0 & 0 \\ \frac{17}{60} & \frac{29}{30} & -\frac{29}{60} & -\frac{19}{15} & 0 & 0 & 0 & 0 & 0 \\ -\frac{23}{20} & -\frac{29}{60} & \frac{23}{20} & \frac{9}{40} & 0 & 0 & -\frac{17}{60} & -\frac{41}{120} & 0 \\ -\frac{29}{60} & -\frac{19}{15} & \frac{9}{40} & \frac{39}{10} & -\frac{3}{2} & 0 & -\frac{41}{120} & -\frac{9}{75} & 0 \\ 0 & 0 & 0 & -\frac{3}{2} & \frac{77}{32} & -\frac{9}{16} & 0 & -\frac{11}{32} & 0 \\ 0 & 0 & 0 & 0 & -\frac{19}{16} & \frac{15}{8} & 0 & -\frac{1}{16} & -\frac{5}{4} \\ 0 & 0 & -\frac{17}{60} & -\frac{41}{120} & 0 & 0 & \frac{41}{60} & -\frac{7}{120} & 0 \\ 0 & 0 & -\frac{41}{120} & -\frac{9}{5} & -\frac{11}{32} & -\frac{1}{16} & -\frac{7}{120} & \frac{79}{60} & -\frac{1}{5} \\ 0 & 0 & 0 & 0 & 0 & -\frac{5}{2} & 0 & -\frac{1}{5} & \frac{29}{20} \end{bmatrix}$$

by the essential boundary conditions:

$$\begin{cases} \varphi_1 = \varphi_2 = \varphi_5 = \varphi_6 = 0 \\ \varphi_7 = \varphi_8 = \varphi_9 = \text{allo} \end{cases}$$

So we need to calculate φ_3 and φ_4

And, the node 4 is an internal node so $R_4=0$

the node 3 according to the boundary conditions the line integral on the boundary is $R_3=0$

$$K \begin{pmatrix} 0 \\ 0 \\ \varphi_3 \\ \varphi_4 \\ 0 \\ 0 \\ 0 \\ \text{allo} \\ \text{allo} \\ \text{allo} \end{pmatrix} = \begin{pmatrix} R_1 \\ R_2 \\ 0 \\ 0 \\ 0 \\ 0 \\ R_7 \\ R_8 \\ R_9 \end{pmatrix}$$

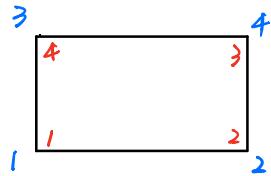
To solve φ_3, φ_4 we have

$$\begin{cases} -\frac{33}{20}\varphi_3 + \frac{9}{40}\varphi_4 - \frac{17}{60}\text{allo} - \frac{41}{120}\text{allo} = 0 \\ \frac{9}{40}\varphi_3 + \frac{17}{60}\varphi_4 - \frac{41}{120}\text{allo} - \frac{8}{15}\text{allo} = 0 \end{cases}$$

$$\begin{cases} \varphi_3 = \frac{239}{681} \text{allo} \\ \varphi_4 = \frac{139}{681} \text{allo} \end{cases}$$

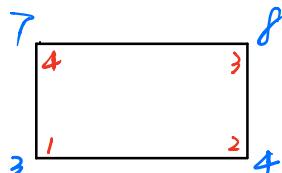
Thus:

element ① :



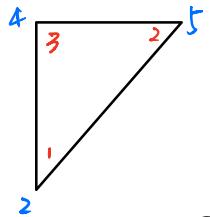
$$\begin{aligned} \varphi &= N_1 \varphi_1 + N_2 \varphi_2 + N_3 \varphi_4 + N_4 \varphi_3 \\ &= \frac{1}{20}xy \times \frac{139}{681} \text{allo} + \left(-\frac{1}{20}xy + \frac{1}{2}\right) \frac{239}{681} \text{allo} \\ &= \left(-\frac{6}{681}xy + \frac{239}{1362}y\right) \text{allo} \end{aligned}$$

element ② :



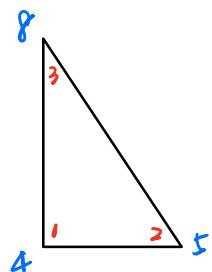
$$\begin{aligned} \varphi &= N_1 \varphi_3 + N_2 \varphi_4 + N_3 \varphi_7 + N_4 \varphi_8 \\ &= \left(\frac{1}{20}xy - \frac{3}{20}x - \frac{1}{4}y + \frac{3}{2}\right) \times \frac{239}{681} \text{allo} + \left(-\frac{1}{20}xy + \frac{3x}{20}\right) \times \frac{139}{681} \text{allo} \\ &\quad + \left(\frac{1}{20}xy - \frac{1}{20}x\right) \text{allo} + \left(-\frac{1}{20}xy + \frac{1}{10}x + \frac{1}{4}y - \frac{1}{2}\right) \text{allo} \\ &= \text{allo} \left(\frac{6}{681} - \frac{10}{681}x + \frac{221}{1362}y + \frac{5}{681}xy\right) \end{aligned}$$

element ③



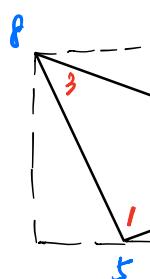
$$\begin{aligned}\varphi &= N_1 \varphi_1 + N_2 \varphi_5 + N_3 \varphi_4 \\ &= \frac{139}{681} \text{allo} \times \left(\frac{5}{2} - \frac{1}{2}x + \frac{1}{2}y \right)\end{aligned}$$

element ④



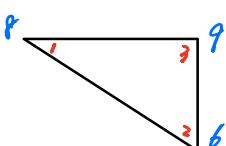
$$\begin{aligned}\varphi &= N_1 \varphi_4 + N_2 \varphi_5 + N_3 \varphi_8 \\ &= (4 - \frac{1}{2}x - \frac{1}{4}y) \times \frac{139}{681} \text{allo} + (-\frac{1}{2} + \frac{1}{4}y) \text{allo} \\ &= \text{allo} \left(\frac{431}{1362} - \frac{139}{1362}x + \frac{271}{1362}y \right)\end{aligned}$$

Element ⑤ =



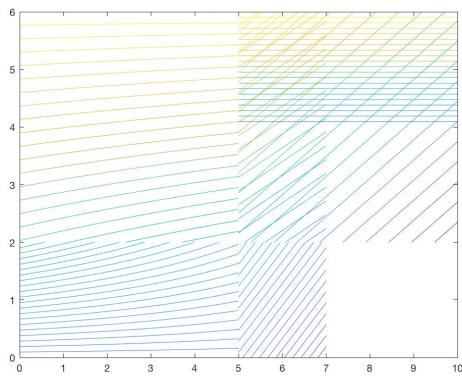
$$\begin{aligned}\varphi &= N_1 \varphi_5 + N_2 \varphi_6 + N_3 \varphi_7 \\ &= \text{allo} \left(\frac{1}{2} - \frac{1}{8}x + \frac{3}{16}y \right)\end{aligned}$$

Element ⑥



$$\begin{aligned}\varphi &= N_1 \varphi_8 + N_2 \varphi_6 + N_3 \varphi_9 \\ &= \text{allo} \left(-2 + \frac{1}{2}y \right)\end{aligned}$$

To plot contour line :



The physical sense of the solution is as following:

Unlike a real fluid, this solution indicates a net zero diag on the body.

Accuracy:

2D approximation: $T = \alpha_0 + \alpha_1 x + \alpha_2 y + \alpha_3 xy + \dots$

Since Taylor expansion:

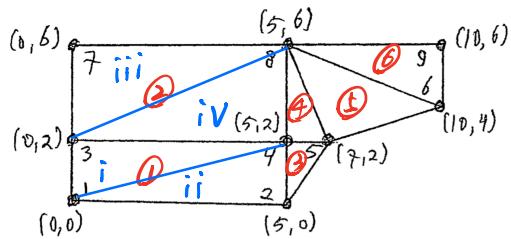
$$T = T(x_i, y_i) + \frac{\partial T}{\partial x} \Big|_{(x_i, y_i)} (x - x_i) + \frac{\partial T}{\partial y} \Big|_{(x_i, y_i)} (y - y_i) + \frac{1}{2} \frac{\partial^2 T}{\partial x^2} \Big|_{(x_i, y_i)} (x - x_i)^2 + \frac{1}{2} \frac{\partial^2 T}{\partial y^2} \Big|_{(x_i, y_i)} (y - y_i)^2 + \frac{\partial^2 T}{\partial xy} \Big|_{(x_i, y_i)} (x - x_i)(y - y_i) + \dots$$

Thus, truncation after a finite number of terms gives an error of $O(h^{p+1})$ and this estimate assumes a 'balanced mesh' in which both x and y dimensions of all elements are of order h . Here element ① and element ② are 4-node rectangular elements with error $O(h^2)$ and $O(h^2)$ separately.

elements ③ ④ ⑤ ⑥ are 3-node triangular elements, with error $O(h_3^2)$, $O(h_4^2)$, $O(h_5^2)$, $O(h_6^2)$ separately.

the error $O(h^2)_{(i=1 \dots 6)}$ has 2 ways to improve the accuracy of the solution.

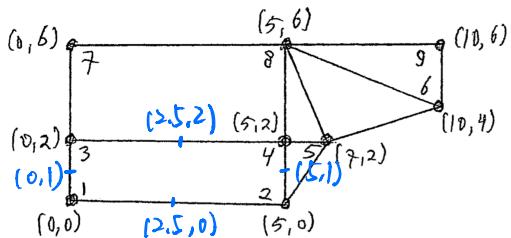
1. reduce h by refining the mesh



For example: change previous element ① into element i and element ii.

change previous element ② into element iii and element iv.

2. increase P by using higher-order shape function:



Turn element ① from 4-node rectangular elements to 8-node triangular elements.

Appendix A MATLAB Code

This is the matlab code using to plot solution.

```
a = 1;
u_0 = 1;
x = linspace(0,5);
y = linspace(0,2);
[X,Y] = meshgrid(x,y);

% figure(1)
z1 = a*u_0.*Y.*(239/1362 - 10*X/681 );
contour(X,Y,z1,20)
hold on

x = linspace(5,7);
y = linspace(0,2);
[X,Y] = meshgrid(x,y);
% figure(2)
z2 = (139/681) *a*u_0*(5/2 - 0.5*X + 0.5*Y );
contour(X,Y,z2,20)
hold on
x = linspace(0,5);
y = linspace(2,6);
[X,Y] = meshgrid(x,y);
% figure(3)
z3 = a*u_0*(6/227 - 10*X/227 + (221/1362)*Y+(5/681)*X.*Y );
contour(X,Y,z3,20)
hold on
x = linspace(5,7);
y = linspace(2,6);
[X,Y] = meshgrid(x,y);
% figure(4)
z4 = a*u_0*(432/1362 - (139/1362)*X+(271/1362)*Y );
contour(X,Y,z4,20)
hold on

x = linspace(5,10);
y = linspace(2,6);
[X,Y] = meshgrid(x,y);
% figure(5)
z5 = a*u_0*(0.5-X/8+3*Y/16 );
contour(X,Y,z5,20)
hold on
x = linspace(5,10);
y = linspace(4,6);
[X,Y] = meshgrid(x,y);
% figure(6)
z6 = a*u_0*(-2+Y/2);
contour(X,Y,z6,20)
hold off
```

Listing 1: Example code from external file.