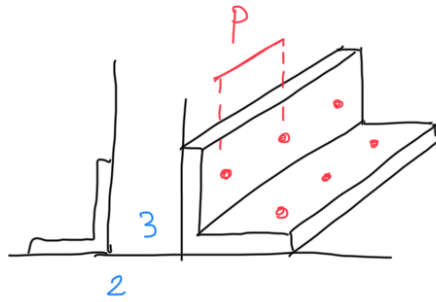
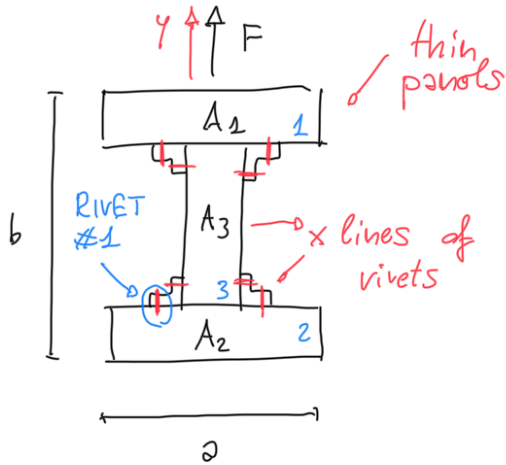


# EX 10 - Ribs and Junctions

1) EXAM 05/07/2023



Let's find the shear stress in rivet #1

DATA

$$J_{xx} = 15 \times 10^4 \text{ Nmm}$$

$$A_1 = A_2 = A = 5000 \text{ mm}^2$$

$$A_3 = 3500 \text{ mm}^2$$

$$p = 100 \text{ mm}$$

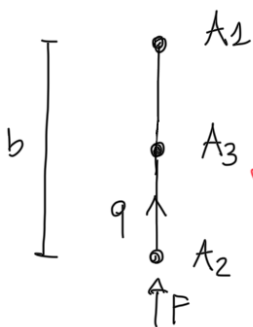
$$\phi = 7 \text{ mm} \text{ rivet diameter}$$

$$a = 500 \text{ mm}$$

$$b = 260 \text{ mm}$$

$$F = 1000 \text{ N}$$

- Shear Fluxes in the panels

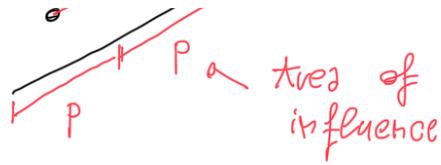


$$q = -F \frac{S_{x1}}{J_{xx}} = -F \cdot \left( -\frac{A \frac{b}{2}}{J_{xx}} \right)$$

here there is not a stringer (i.e. axial load), thus  $q$  is constant along all the panel side.

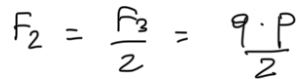
- Plate 3





this is

59 p



$$r_1 = \frac{F_2}{A_{\text{rivet}}} = \frac{\frac{1}{2} q \cdot p}{\pi \left(\frac{\phi}{2}\right)^2}$$

[illegible]
$$l = 1500 \text{ mm}$$

$$a = 400 \text{ mm}$$

$$b = 250 \text{ mm}$$

$$t_1 = 1 \text{ mm}$$

$$t_2 = 2 \text{ mm}$$

$$t_3 = 1.5 \text{ mm}$$

$$A_1 = 2000 \text{ mm}^2$$

$$A_2 = 1000 \text{ mm}^2$$

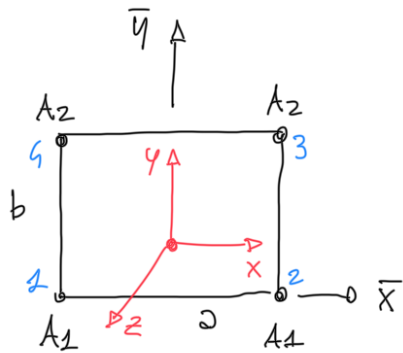
$$F = 10 \text{ kN}$$

$$p_1 = 90 \text{ N/mm}$$

$$p_2 = 300 \text{ N/mm}$$

- $q_i$  and  $\sigma_{zz}$  at  $l/2$  from the clamp?
- Plot the internal actions in the vib

- Section Properties



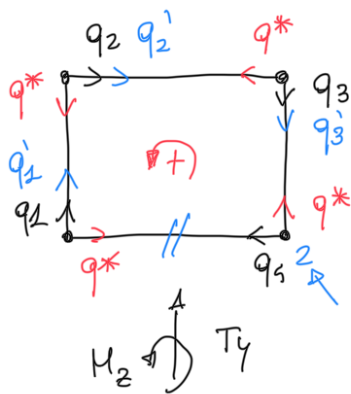
$$\bar{y}_c = \frac{2A_2 b}{2A_1 + 2A_2} = \frac{1}{3}b$$

$$J_{xx} = 2A_2 \left(\frac{2}{3}b\right)^2 + 2A_1 \left(-\frac{1}{3}b\right)^2 = \frac{4}{3}A_2 b^2$$

$$S_{x1} = S_{x2} = -A_1 \cdot \frac{b}{3} = -\frac{2}{3}A_2 b$$

$$S_{x3} = S_{x4} = \frac{2}{3}A_2 b$$

- Open Cell Fluxes



Let's compute them for a generic  $T_y$  and  $M_z$

$$q_1' = -T_y \frac{S_{x1}}{J_{xx}} = \frac{T_y}{2b} = -q_3'$$

$$q_2' = 0$$

- Moment Equilibrium wrt ②

$$LHS = M_z - T_y \cdot \frac{2}{2}$$

$$RHS = 2q^* \Omega_{acc} - 2q_1' \Omega_1$$

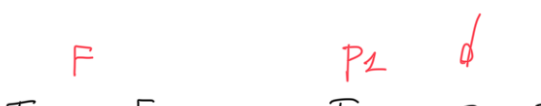
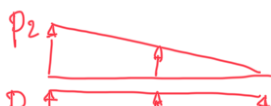
$$\Omega_{acc} = 2 \cdot b$$

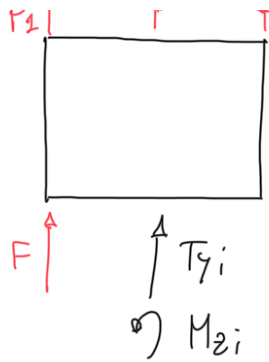
$$\Omega_1 = 2 \cdot \frac{b}{2}$$

$$M_z - T_y \frac{2}{2} = 2q^* 2b - 2 \frac{T_y \cdot \frac{2b}{2}}{2}$$

$$q^* = \frac{M_z}{2 \cdot 2b}$$

- Lumped Forces





$$T_{yF} = F$$

$$M_{zF} = -F \cdot \frac{a}{2}$$

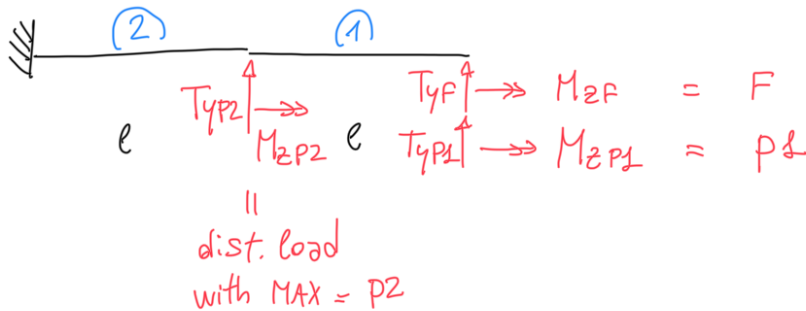
$$T_{yP1} = P1 \cdot a$$

$$M_{zP1} = 0$$

$$T_{yP2} = \frac{1}{2} P2 \cdot a$$

$$M_{zP2} = -\left(\frac{1}{2} P2 \cdot a\right) \cdot \frac{1}{6} a$$

- Internal Actions in the beam



①

$$\begin{cases} T_{y1} = -T_{yF} - T_{yP1} \\ M_{x1} = -(T_{yF} + T_{yP1}) \cdot z_2 \\ M_{z1} = M_{zF} + M_{zP1} \end{cases}$$

②

$$\begin{cases} T_{y2} = -T_{yP2} - T_{yP1} - T_{yF} \\ M_{x2} = -T_{yP2} \cdot z_2 - (T_{yP1} + T_{yF}) \cdot (l + z_2) \\ M_{z2} = M_{zP2} + M_{zP1} + M_{zF} \end{cases}$$

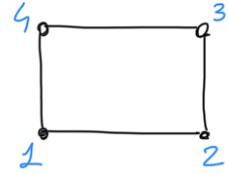
- Shear fluxes in  $z_2 = \frac{a}{2}$

$$q_1 = q_1' - q^* = \frac{T_{y2}}{2b} - \frac{M_{z2}}{2ab} = -242 \text{ N/mm}$$

$$q_2 = q_2' - q^* = -30 \text{ N/mm}$$

$$q_3 = q_3' - q^* = 182 \text{ N/mm}$$

$$q_4 = -q^* = -30 \text{ N/mm}$$

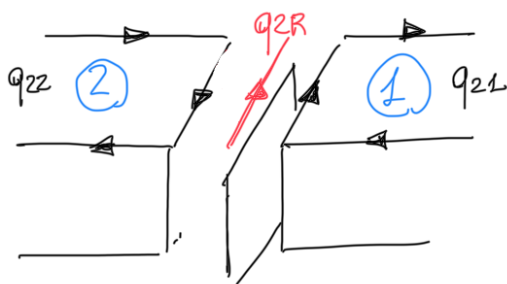
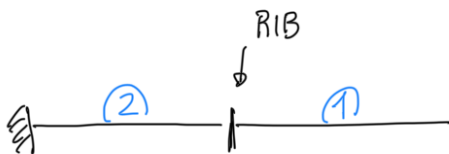


- Axial stress in stringers at  $z_2 = l/2$

$$\sigma_{zz1} = - \frac{M_{x2}}{J_{xx}} \cdot \frac{1}{3}b = 148.5 \text{ MPa} = \sigma_{zz4}$$

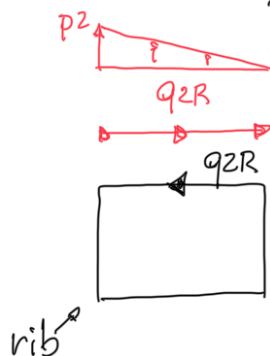
$$\sigma_{zz3} = \sigma_{zz4} = \frac{M_{x2}}{J_{xx}} \cdot \frac{2}{3}b = -297 \text{ MPa}$$

- Internal Actions in the rib



$$q_{z2} = q_{z1}$$

$$q_{zR} = q_{z2} - q_{z1}$$



the rib equilibrates :

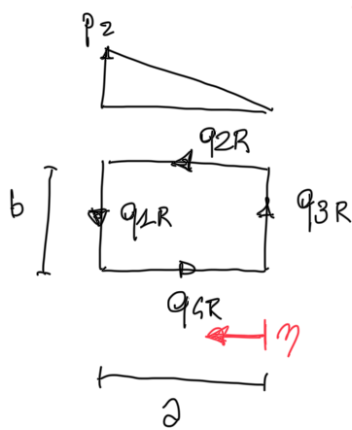
- the external force  $p_z$
- the shear fluxes jump in the panel

the rib must be in equilibrium

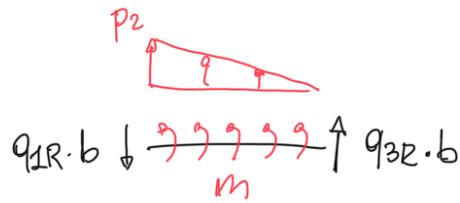
- Beam model of the rib

11 . 10 00 . 1 . + . - 0 11 . + .

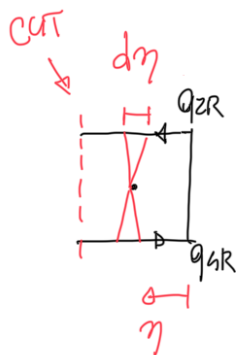
THIS model allows an estimation of the internal actions in the rib, along one of its sides



TBN:  $q_{2R} = q_{3R} = q^*$



distributed bending moment given by  $q_{2R}$  and  $q_{4R}$

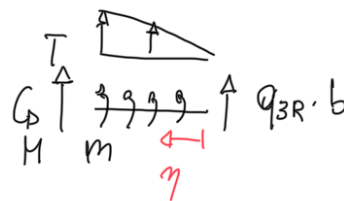
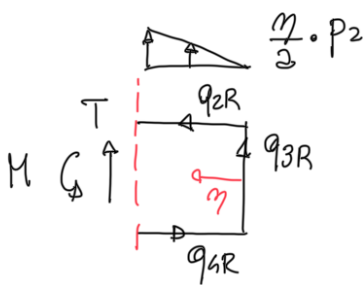


$$m = 2 q_{2R} \cdot \Omega + 2 q_{4R} \cdot \Omega =$$

$$= 2 (q_{2R} + q_{4R}) \cdot \left( \frac{1}{2} \frac{b}{2} d\eta \right) \quad \text{area of the triangle}$$

$d\eta = 1 \rightarrow m = \text{bending moment} \times \text{unit length given by } q_{2R} \text{ and } q_{4R}$

### • Internal Actions in the rib



EXTERNAL  
LOAD

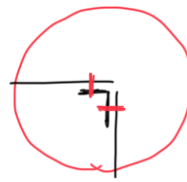
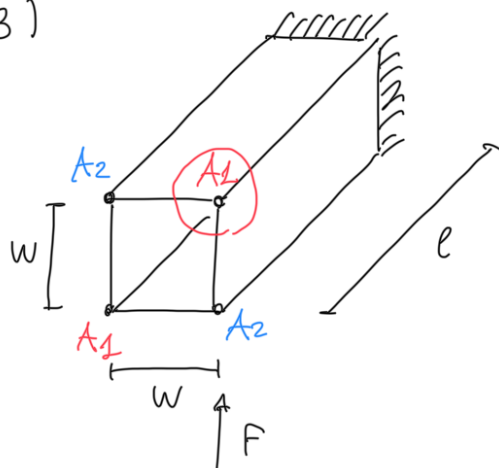
$$T = -q_{3R} \cdot b - \frac{1}{2} \left( \frac{\eta}{2} p_2 \right) \cdot \eta$$

$$M = -2 q_{3R} \left( \frac{1}{2} b \eta \right) - \left( \frac{1}{2} \left( \frac{\eta}{2} p_2 \right) \eta \right) \cdot \left( \frac{1}{3} \eta \right) - \int_0^\eta m d\eta$$

EXTERNAL  
LOAD



3)



DATA

$$A_1 = 300 \text{ mm}^2 = 2A$$

$$A_2 = 150 \text{ mm}^2 = A$$

$t = 1.5 \text{ mm}$  for all the panels

$$w = 100 \text{ mm}$$

$$E = 72000 \text{ MPa}$$

$$\nu = 0.3$$

$$l = 1500 \text{ mm}$$

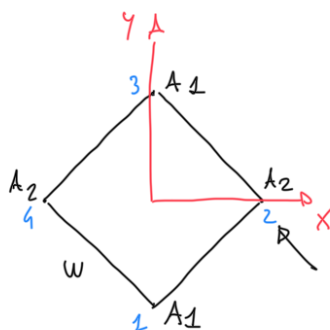
$$d_1 = 2 \text{ mm}, p_1 = 10 \text{ mm}$$

$$d_2 = 3 \text{ mm}, p_2 = 12 \text{ mm}$$

$$F = 15 \text{ kN}$$

$\sigma_{zz}$  at  $\frac{l}{2}$ ?  
shear stress in the rivets?

• Section properties



if we rotate the section, it becomes symmetric wrt x and y  $\rightarrow J_{xy} = 0$

$$J_{xx} = 2A_1 \left( \frac{\sqrt{2}}{2} w \right)^2 = A_1 w^2 = 2Aw^2$$

$$J_{yy} = 2A_2 \left( \frac{\sqrt{2}}{2} w \right)^2 = A_2 w^2 = Aw^2$$

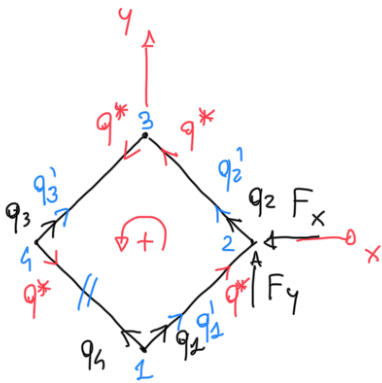
$$S_{x1} = -S_{x3} = -A_1 \frac{\sqrt{2}}{2} w = -\sqrt{2} Aw$$

$$S_{x2} = S_{x4} = 0$$

$$S_{y1} = S_{y3} = 0$$

$$S_{y2} = -S_{y4} = A_2 \frac{\sqrt{2}}{2} w = \frac{\sqrt{2}}{2} Aw$$

- Open Cell Fluxes



$$q_1' = -F_y \frac{S_{x1}}{J_{xx}} - F_x \frac{S_{y1}}{J_{yy}}$$

$$q_2' = -F_y \frac{S_{x1} + S_{x2}}{J_{xx}} - F_x \frac{S_{y1} + S_{y2}}{J_{yy}}$$

$$q_3' = -F_y \frac{S_{x1}}{J_{xx}} - F_x \frac{S_{y1}}{J_{yy}}$$

- Moment Equivalence wrt ②

$$LHS = \phi$$

$$RHS = 2q^* w^2 - 2q_3' \frac{w^2}{2}$$

$$q^* = \frac{q_3'}{2} = \frac{1}{2} \left( F_x \cdot \frac{\sqrt{2}}{2} w \right) = \frac{\sqrt{2}}{4} F_x$$

- Closed Cell

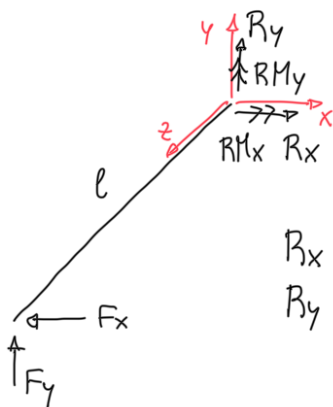
$$q_1 = q_1' + q^*$$

$$q_2 = q_2' + q^*$$

$$q_3 = q_3' - q^*$$

$$q_4 = -q^*$$

- Reaction Forces and Internal Actions in the beam



we don't care about  $RM_z$  because it doesn't act on stringers

$$\begin{aligned} R_x &= F_x & RM_x &= F_y \cdot l \\ R_y &= -F_y & RM_y &= F_x \cdot l \end{aligned}$$



$$\begin{aligned} T_x &= -F_x \\ T_y &= F_y \\ M_x &= F_y (z - l) \end{aligned}$$



$$\Rightarrow \begin{matrix} \curvearrowright F_x \cdot l \\ M_x T_x \end{matrix}$$

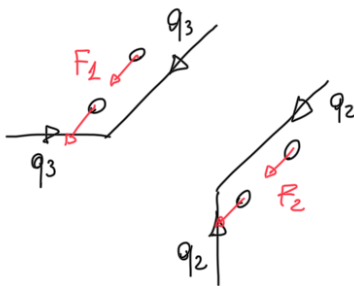
$$M_y = F_x (z - e)$$

- Stress in stringers at  $z = \frac{l}{2}$

$$\sigma_{zz1} = + \frac{M_x(\frac{l}{2})}{J_{xx}} \cdot y = + \frac{F_y \cdot \frac{l}{2}}{J_{xx}} \left( + \frac{\sqrt{2}}{2} w \right) = 187.5 \text{ MPa} = -\sigma_{zz3}$$

$$\sigma_{zz2} = - \frac{M_y}{J_{yy}} \cdot x = + \frac{(+F_x \cdot z_1)}{J_{yy}} \cdot \left( \frac{\sqrt{2}}{2} w \right) = 375.0 \text{ MPa} = -\sigma_{zz4}$$

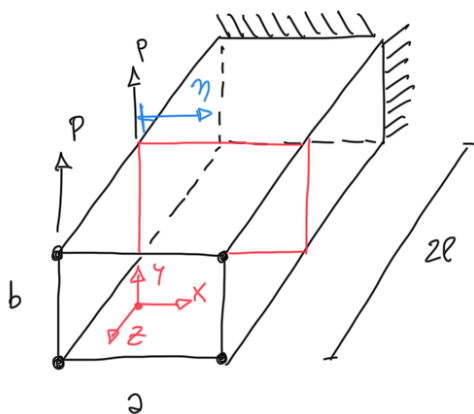
- Shear Stress in Rivets



$$F_1 = q_3 \cdot p_1 \quad \tau_1 = \frac{F_1}{\pi \left( \frac{d_1}{2} \right)^2} = \frac{q_3 p_1}{\pi \left( \frac{d_1}{2} \right)^2} = 119.37 \text{ MPa}$$

$$F_2 = q_2 \cdot p_2 \quad \tau_2 = \frac{F_2}{\pi \left( \frac{d_2}{2} \right)^2} = \frac{q_2 p_2}{\pi \left( \frac{d_2}{2} \right)^2} = 190.99 \text{ MPa}$$

4)



DATA

$$a = 500 \text{ mm}$$

$$b = 250 \text{ mm}$$

$$t = 0.6 \text{ mm}$$

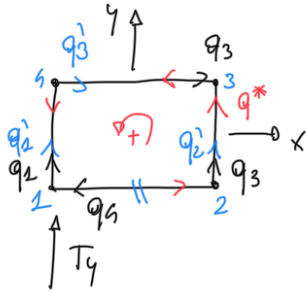
$$A = 500 \text{ mm}^2$$

$$l = 2000 \text{ mm}$$

$$P = 1000 \text{ N}$$

Let's find the moment in the riv  
at  $\eta = \frac{a}{3}$

- Open Cell Fluxes



$$q_1' = q_2' = \frac{T_y}{2b} \quad q_3' = 0$$

- Moment Equivalence wrt  $(\frac{1}{2})$

$$LHS = 0$$

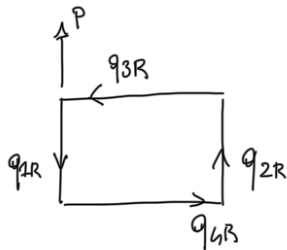
$$RHS = 2q^* \cdot 2b + 2 \cdot \frac{T_y}{2b} \cdot \frac{2b}{2} \quad q^* = -\frac{T_y}{4b}$$

- Total Fluxes

$$q_1 = q_1' - q^* = \frac{T_y}{2b} + \frac{T_y}{4b} = \frac{3}{4} \frac{T_y}{b} \quad q_3^* = -q^* = \frac{T_y}{4b}$$

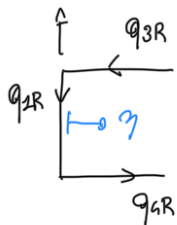
$$q_2 = q_2' + q^* = \frac{T_y}{4b} \quad q_4 = -q^* = \frac{T_y}{4b}$$

- Internal Action in the rib



$$q_{1R} = \frac{3}{4} \frac{P}{b} \quad q_{3R} = \frac{P}{4b}$$

$$q_{2R} = \frac{P}{4b} \quad q_{4R} = \frac{P}{4b}$$



$$M(\gamma) = 2 \cdot q_{1R} \left( \frac{b\gamma}{2} \right) + 2 \cdot q_{3R} \left( \frac{b\gamma}{4} \right) + \dots$$

$$\dots + 2 \cdot q_{4R} \left( \frac{b\gamma}{4} \right) - P \cdot \gamma$$

$$M\left(\frac{2}{3}\right) = 0$$