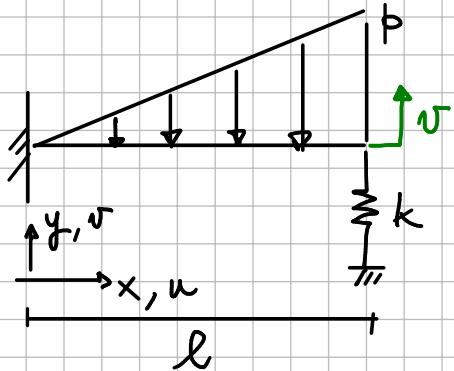


# EXERCISE SESSION 12

Ex 1 ( EXAM  
25/01/2023 )



$$p = 14 \text{ N/mm}$$

$$l = 2000 \text{ mm}$$

$$EI = 10^{10} \text{ N mm}^2$$

$$k = 100 \text{ N/mm}$$

FIND  $N$

USING 1 TERM  
Poly APPROX

SOL

$$N(x) = a_0 + a_1 x + a_2 x^2 \quad \text{By}$$

$$N(0) = 0 \quad a_0 = 0$$

$$N'(0) = 0 \quad a_1 = 0$$

$$\rightarrow N(x) = a x^2$$

$$\delta W_e = \int_0^l -\delta N q(x) dx - \delta N(l) \cdot k N(l)$$

$$\delta W_i = \int_0^l \delta N_{xx} EI N_{xx} dx$$

$$\text{WITH } \delta N = \delta a \cdot x^2$$

$$N_{xx} = a \cdot 2 \quad \delta N_{xx} = \delta a \cdot 2$$

$$q(x) = p \cdot \frac{x}{l}$$

$$\text{PvW } \delta W_i = \delta W_e$$

$$\delta a \int_0^l 2EI 2a dx = -\delta a \left( \int_0^l x^2 \cdot p \cdot \frac{x}{l} dx + l^2 \cdot k \cdot a \cdot l^2 \right)$$

$$a \cdot 4EI \cdot l = -\frac{1}{4} \frac{p l^4}{l} - a \cdot k l^4$$

$$a = -\frac{p l^2}{4EI + 4k l^3}$$

$$N(l) = a \cdot l^2 = -60.67 \text{ mm}$$

## Contents

---

- [Data](#)
- [Solution](#)

```
clear variables
close all
home
```

---

## Data

---

```
f = 14; % [N/mm]
EJ = 1E+10; % [Nm^2]
l = 2000; % [mm]
k = 100; % [N/mm]
```

---

## Solution

---

```
syms c x

q = f*x/l;

Phi = x^2;
w = c*Phi;

LHS = int(diff(Phi, x, 2)^2*EJ*c, x, 0, l);
RHS = int(-Phi*q, x, 0, l) - subs(Phi, x, l)^2*c*k;

c_d = double(solve(LHS == RHS, c));

w_a = double(subs(w, [c x], [c_d l]))
```

---

w\_a =

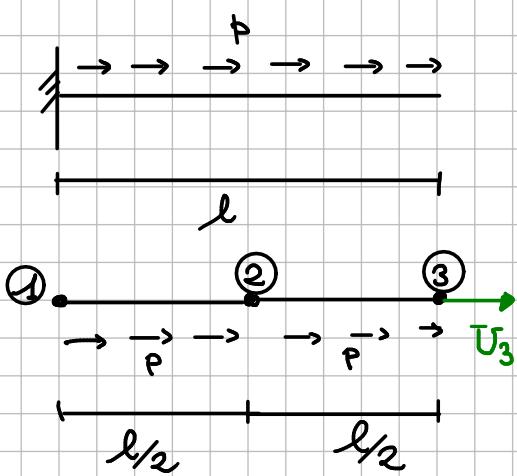
-66.66666666666671

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.....

Ex 2

( EXAM  
05/07/2023 )



$$l = 3000 \text{ mm}$$

$$EA = 7 \cdot 10^5 \text{ N}$$

$$p(x) = 10 \text{ N/mm}$$

LINEAR FE ELEMENTS

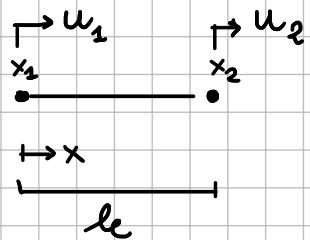
FIND  $U_3$

SOL

ELEMENT LENGTH

$$l_e = \frac{l}{2}$$

LOCAL REF SYST. FOR EACH ELEMENT



LOCAL

$$u_e(x) = N_1(x) \cdot u_1 + N_2(x) \cdot u_2 = \underline{N}_e \cdot \underline{q}_e = [N_1 \ N_2] \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$



$$N_1(x) = \frac{x - x_2}{x_1 - x_2} = \frac{x_2 - x}{l_e} \quad \bullet$$

$$N_2(x) = \frac{x - x_1}{x_2 - x_1} = \frac{x - x_1}{l_e} \quad \bullet$$

$$\varepsilon_e(x) = u_{e/x}(x) = \sum_i N_{i/x} \cdot u_i$$

$$= -\frac{1}{l_e} \cdot u_1 + \frac{1}{l_e} u_2 = \begin{bmatrix} -1/l_e & 1/l_e \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

$$= \underline{\beta}_e \cdot \underline{q}_e$$

$$\begin{aligned}
 \delta W_e &= \int_{x_1}^{x_2} \delta \varepsilon_e^T \cdot EA \varepsilon_e dx \\
 &= \int_{x_1}^{x_2} \underbrace{\delta [q_e]^T}_{1 \times 2} \underbrace{[B_e]^T}_{2 \times 1} EA \underbrace{[B_e]}_{1 \times 2} \underbrace{[q_e]}_{2 \times 1} dx \\
 &= \delta \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}^T \cdot \int_{x_1}^{x_2} \begin{bmatrix} -\frac{1}{l_e} & \frac{1}{l_e} \end{bmatrix}^T EA \cdot \begin{bmatrix} -\frac{1}{l_e} & \frac{1}{l_e} \end{bmatrix} dx \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}
 \end{aligned}$$

$$\begin{aligned}
 &\frac{1}{l_e^2} \cdot EA \cdot \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} \cdot [x_2 - x_1] \\
 &\frac{EA}{l_e} \cdot \begin{bmatrix} 1 & -1 \\ -1 & 1 \end{bmatrix} = \underline{K}_e
 \end{aligned}$$

$$\delta q_e^T \cdot \underline{K}_e \cdot q_e$$

$$\begin{aligned}
 \delta W_{ee} &= \int_{x_1}^{x_2} \delta \underline{u}_e^T \cdot p(x) dx \\
 &= \delta q_e^T \cdot \int_{x_1}^{x_2} \begin{bmatrix} N_1 & N_2 \end{bmatrix}^T \cdot p(x) dx \\
 &= \delta q_e^T \cdot \begin{bmatrix} 1 \\ 1 \end{bmatrix} \frac{p \cdot l_e}{2} = \delta q_e^T \cdot \underline{f}_e
 \end{aligned}$$

$$\begin{aligned}
 &\int_{x_1}^{x_2} \begin{bmatrix} N_1 & N_2 \end{bmatrix}^T \cdot p dx \\
 &\int_{x_1}^{x_2} \begin{bmatrix} \frac{x_2 - x}{x_2 - x_1} & \frac{x - x_1}{x_2 - x_1} \\ \frac{x_2 x - x^2 \cdot \frac{1}{2}}{x_2 - x_1} & \frac{x^2 \frac{1}{2} - x_2 x}{x_2 - x_1} \end{bmatrix}^T \cdot p \cdot dx \\
 &\left[ \begin{bmatrix} x_2^2 - \frac{1}{2} x_2^2 - x_1 x_2 + \frac{1}{2} x_1^2 & \frac{1}{2} x_2^2 - x_1 x_2 - \frac{1}{2} x_1^2 + x_2^2 \\ \frac{(x_2 - x_1)^2}{2} & \frac{(x_2 - x_1)^2}{2} \end{bmatrix} \right] \frac{p}{l_e} \\
 &\begin{bmatrix} 1 & 1 \end{bmatrix} \frac{p \cdot l_e}{2}
 \end{aligned}$$

$$\text{ASSEMBLE : } \quad \text{FOR EL #1} \quad \Rightarrow \quad u_1 = \bar{U}_1 \quad u_2 = \bar{U}_2 \\
 \text{EL #2} \quad \Rightarrow \quad u_1 = \bar{U}_2 \quad u_2 = \bar{U}_3$$

$$\begin{aligned}
 \underline{q} &= \begin{bmatrix} \bar{U}_1 \\ \bar{U}_2 \\ \bar{U}_3 \end{bmatrix} \\
 \underline{K} &= \begin{bmatrix} \underline{K}_e & 0 \\ 0 & 0 \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & \underline{K}_e \end{bmatrix} = \begin{bmatrix} 1 & -1 & 0 \\ -1 & 2 & -1 \\ 0 & -1 & 1 \end{bmatrix} \cdot \frac{EA}{l_e}
 \end{aligned}$$

$$\underline{f} = \begin{bmatrix} \frac{\underline{f}_e}{2} \\ 0 \end{bmatrix} + \begin{bmatrix} 0 \\ \underline{f}_e \end{bmatrix} = \begin{bmatrix} 1 \\ 2 \\ 1 \end{bmatrix} \frac{p \cdot l_e}{2}$$



$$\text{SOLVE } \underline{K} \cdot \underline{q} = \underline{f}$$

$$\frac{EA}{l} \begin{bmatrix} 1 & -1 & 0 \\ -1 & 2 & -1 \\ 0 & -1 & 1 \end{bmatrix} \cdot \begin{bmatrix} \bar{U}_1 \\ \bar{U}_2 \\ \bar{U}_3 \end{bmatrix} = \begin{bmatrix} 1 \\ 2 \\ 1 \end{bmatrix} \frac{P \cdot l_e}{2}$$

From BC'  $\bar{U}_1 = 0$

$$\left\{ \begin{array}{l} 2\bar{U}_2 - \bar{U}_3 = \frac{P l_e^2}{E A} \\ -\bar{U}_2 + \bar{U}_3 = \frac{P l_e^2}{2 E A} \end{array} \right. \rightarrow \left\{ \begin{array}{l} \bar{U}_2 = \frac{3}{2} \frac{P l_e^2}{E A} \\ \bar{U}_3 = 2 \frac{P l_e^2}{E A} = 64.286 \text{ mm} \end{array} \right.$$

## Contents

---

- [Data](#)
- [Sol](#)

```
close all  
clear variables  
home
```

---

## Data

---

```
f = 10; % N/mm  
l = 3000; % mm  
EA = 7E+05; % N
```

---

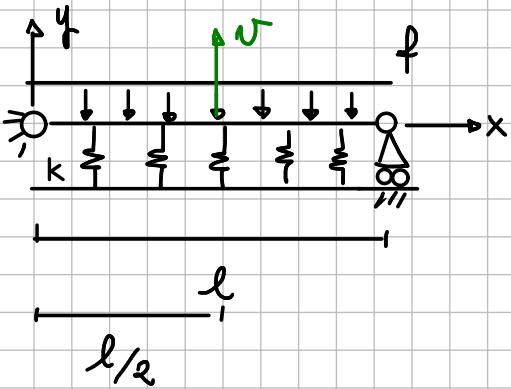
## Sol

---

```
syms x_e u_1 u_2 u_3  
  
l_e = 1/2;  
  
N_1 = (l_e-x_e)/l_e;  
N_2 = (x_e)/l_e;  
  
Ne_1 = [N_1 N_2 0];  
Ne_2 = [0 N_1 N_2];  
  
u = [u_1; u_2; u_3];  
  
Be_1 = diff(Ne_1, x_e, 1);  
Be_2 = diff(Ne_2, x_e, 1);  
  
K = (int(Be_1.*EA*Be_1, x_e, 0, l_e) + int(Be_2.*EA*Be_2, x_e, 0, l_e));  
  
dW_e = (int(Ne_1., x_e, 0, l_e) + int(Ne_2., x_e, 0, l_e))*f;  
  
sol = solve(K(2:end, 2:end)*u(2:end) == dW_e(2:end), u(2:end));  
double(sol.u_3)  
  
ans =  
64.285714285714292
```

---

Ex 3 (EXAM  
05/09/2023)



DATA

$$l = 1000 \text{ mm}$$

$$EJ = 1E+12 \text{ N mm}^2$$

$$k = 8 \text{ MPa} \left( \frac{\text{N}}{\text{mm}} \cdot \frac{1}{\text{mm}} \right)$$

$$f = 100 \text{ N/mm}$$

FIND  $N$  USING TRIG. APPROX

SOL  $N(x) = A \cdot \sin\left(\frac{\pi \cdot x}{l}\right) = A \cdot \phi(x)$  satisfies BC

$$N_{/x}(x) = A \frac{\pi}{l} \cdot \cos\left(\frac{\pi x}{l}\right) = A \phi_{/x}(x)$$

$$N_{/xx}(x) = -A \frac{\pi^2}{l^2} \cdot \sin\left(\frac{\pi x}{l}\right) = A \phi_{/xx}(x)$$

PW  $\delta W_i = \int_0^l \delta N_{/xx} E J N_{/xx} dx$

$$\delta W_e = - \int_0^l \delta N \cdot f(x) dx - \int_0^l \delta N k \cdot N dx$$

$$\delta W_i = \delta A \cdot \int_0^l -\frac{\pi^2}{l^2} \sin\left(\frac{\pi x}{l}\right) E J \cdot -\frac{\pi^2}{l^2} \cdot A \cdot \sin\left(\frac{\pi x}{l}\right) dx$$

KNOWN  $\int_0^l \sin^2\left(\frac{\pi x}{l}\right) dx = \frac{l}{2}$

$$\delta W_e = \delta A \cdot \left( -\int_0^l \sin\left(\frac{\pi x}{l}\right) \cdot f dx - \int_0^l \sin^2\left(\frac{\pi x}{l}\right) \cdot k \cdot A dx \right)$$

KNOWN  $\int_0^l \sin\left(\frac{\pi x}{l}\right) dx = \frac{2l}{\pi}$

PW  $\approx \delta W_i = \delta W_e \Rightarrow \underbrace{E J \cdot A \cdot \frac{\pi^4}{l^4} \cdot \frac{l}{2}}_{= -f \cdot \frac{2l}{\pi} - kA \cdot \frac{l}{2}}$

$$N\left(\frac{l}{2}\right) = A \cdot \sin\left(\frac{\pi \cdot l/2}{l}\right) = A = -\frac{f \frac{2l}{\pi}}{\frac{1}{2} E J \frac{\pi^4}{l^3} + k l/2} = -1.208 \text{ mm}$$

## Contents

---

- [Data](#)
- [Sol](#)

```
close all
clear variables
home
```

---

## Data

---

```
l = 1000;      % mm
EJ = 1E+12;    % Nmm^2
k = 8;         % MPa
f = 100;       % N/mm
```

---

## Sol

---

```
syms z A
phi = sin(pi*z/l);
LHS = int(diff(phi, 2, z)^2*EJ*A, z, 0, l) + ...
       int(phi*f, z, 0, l) + ...
       int(phi*k*A*phi, z, 0, l);
sol = solve(LHS == 0, A);
v = double(sol*subs(phi, z, l/2))
```

---

```
v =
-1.207902973306587
```

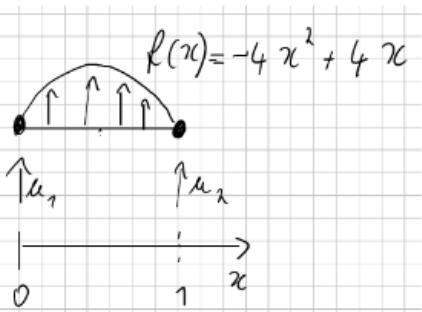
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Ex 4 ( EXAM 13/06/2023 )

The two-node finite element sketched in the figure is loaded by the distributed force  $f(x)$ . Compute the virtual work of the force for the virtual displacement of node #2. The overall length of the finite element is equal to 1 mm

(Unit for result: N mm)



Data

$$f(x) = -4x^2 + 4x \text{ N/mm}$$

FIND

$$\delta W_e = \delta u_1 \cdot \left[ \dots \right] + \delta u_2 \cdot \overbrace{\left[ \dots \right]}^{\text{TENS}}$$

SOL  $N_1(x) = (1-x)$   $u(x) = N_1 \cdot u_1 + N_2 \cdot u_2$   
 $N_2(x) = x$

$$\begin{aligned} \delta W_e &= \int_0^l \delta u \cdot f(x) dx \\ &= \delta u_1 \cdot \int_0^l N_1 \cdot f(x) dx + \delta u_2 \cdot \int_0^l N_2 \cdot f(x) dx \end{aligned}$$

$$\begin{aligned} \bullet \int_0^l N_2 \cdot f(x) dx &= \int_0^l x \cdot (-4x^2 + 4x) dx \\ &= \int_0^l -4x^3 + 4x^2 dx \\ &= \left[ -x^4 + \frac{4}{3}x^3 \right]_0^l = -1 + \frac{4}{3} = \frac{1}{3} \end{aligned}$$

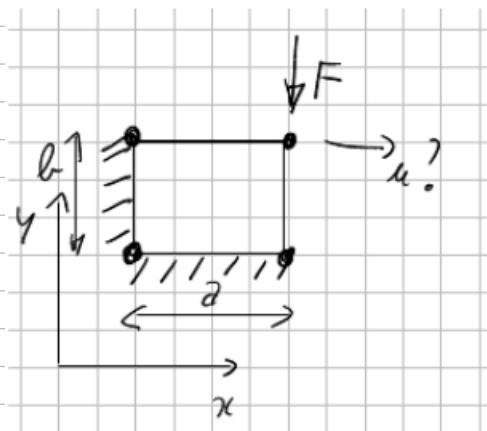
Ex 5(EXAM  
09/09/2024)

The single bilinear finite element sketched in the figure has the displacement of the top left, bottom left and bottom right nodes completely constrained. The element has unit thickness, and the material works in a state of plane stress, so that

$$\begin{Bmatrix} \sigma_{xx} \\ \sigma_{yy} \\ \tau_{xy} \end{Bmatrix} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & (1-\nu)/2 \end{bmatrix} \begin{Bmatrix} \varepsilon_{xx} \\ \varepsilon_{yy} \\ \gamma_{xy} \end{Bmatrix} \quad (1)$$

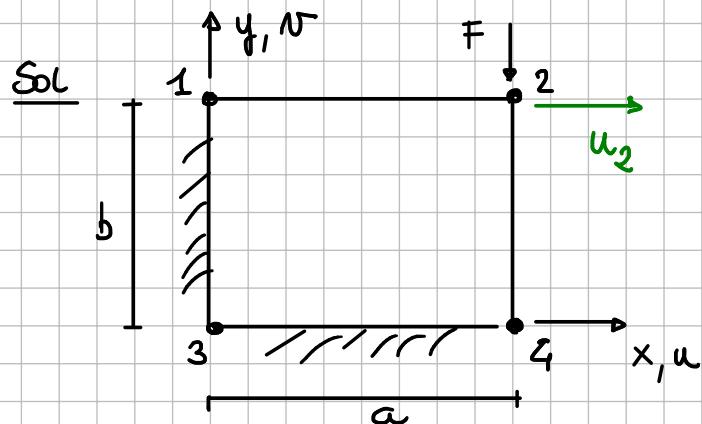
The top right node is loaded by the vertical force  $F$ , as sketched. Compute the horizontal displacement  $u$  of the top right node.

(Unit for result: mm)



*Data*  
 $t = 1 \text{ mm}$   
 $a = 4 \text{ mm}$   
 $b = 3 \text{ mm}$   
 $E = 72000 \text{ MPa}$   
 $\nu = 0.3$   
 $F = 100 \text{ N}$

BILINEAR MEANS THAT  $u(x)$  &  $N(x)$  ARE LINEAR



FROM BC

$$\begin{cases} u_1, u_3, u_4 = 0 \\ N_1, N_3, N_4 = 0 \end{cases}$$

Pw

$$\int_V \delta \underline{\varepsilon}^T \cdot \underline{\sigma} dV = \delta \underline{u}^T (a, b) \cdot \underline{F}$$

KNOW FROM PLANE STRESS EVERYTHING IS CONSTANT IN Z

$$\int_V \delta \underline{\varepsilon}^T \cdot \underline{\sigma} dV = \int_A \delta \underline{\varepsilon}^T \cdot \underline{\sigma} dA \cdot t \quad \text{WHERE } t=1$$

KNOW  $\underline{u} = \begin{Bmatrix} u(x,y) \\ N(x,y) \end{Bmatrix}$  &  $\underline{F} = \begin{Bmatrix} 0 \\ -F \end{Bmatrix}$

IF FULLY UNCONSTRAINED

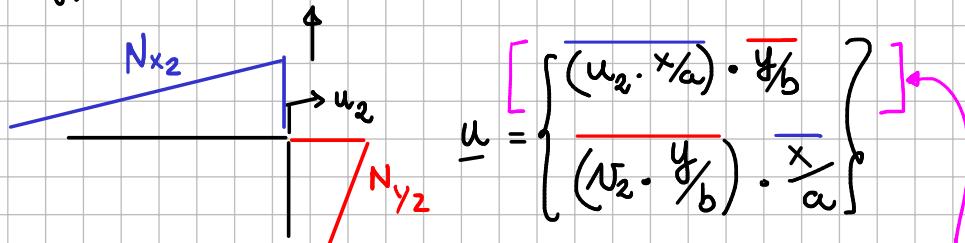
$$\underline{u}(x,y) = \underline{\underline{N}} \cdot \underline{q}$$

$$[\underline{\underline{N}}]_{2 \times 8} \cdot [\underline{q}]_{8 \times 1}$$

$\uparrow \left\{ \begin{array}{l} \{u_i\} \\ \{N_i\} \end{array} \right\}$

SINCE 1, 3, 4 ARE CONSTRAINED  $\underline{q}$  is  $2 \times 1$   $\underline{\underline{N}}$  is  $2 \times 2$

$$\underline{u}(x,y) = \begin{Bmatrix} u(x,y) \\ N(x,y) \end{Bmatrix} = \begin{bmatrix} \frac{x}{a} & 0 \\ 0 & \frac{y}{b} \end{bmatrix} \begin{Bmatrix} u_2 \\ N_2 \end{Bmatrix} = \underline{\underline{N}} \cdot \underline{q}$$



HOW TO INTERPOLATE DISPLACEMENT:

COMPUTE THE STRAIN

$$\underline{\underline{\epsilon}} = \begin{Bmatrix} \epsilon_{11} \\ \epsilon_{22} \\ \gamma_{12} \end{Bmatrix} = \begin{bmatrix} \frac{\partial}{\partial x} & 0 \\ 0 & \frac{\partial}{\partial y} \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} \end{bmatrix} \underline{u}$$

$$3 \times 2 \quad 2 \times 1$$

FROM  $u_2$  FIX  $y = b$  GET  $u(x)$

USE INTERPOLATION  $N_{x_2} = \frac{x}{a}$

$u(x) = u_2 \cdot \frac{x}{a}$ , THEN INTERP.

$u(x)$  IN  $y$  USING  $N_{y_2} = \frac{y}{b}$

$$\rightarrow u(x,y) = (u_2 \cdot \frac{x}{a}) \cdot \frac{y}{b}$$

$$= \begin{bmatrix} \frac{\partial}{\partial x} & 0 \\ 0 & \frac{\partial}{\partial y} \\ \frac{\partial}{\partial y} & \frac{\partial}{\partial x} \end{bmatrix} \begin{bmatrix} \frac{x}{a} & 0 \\ 0 & \frac{y}{b} \end{bmatrix} \cdot \begin{Bmatrix} u_2 \\ N_2 \end{Bmatrix}$$

$$3 \times 2 \quad 2 \times 2 \quad 2 \times 1$$

$\underline{\underline{B}}$

$\underline{q}$

$$\underline{\underline{\epsilon}} = \begin{bmatrix} \frac{y}{ab} & 0 \\ 0 & \frac{x}{ab} \\ \frac{x}{ab} & \frac{y}{ab} \end{bmatrix} \cdot \begin{Bmatrix} u_2 \\ N_2 \end{Bmatrix} = \underline{\underline{B}} \cdot \underline{q}$$

STRESS AS FUNCTION OF STRAIN

$$\underline{\underline{\sigma}} = \underline{\underline{D}} \cdot \underline{\underline{\epsilon}}$$

$$3 \times 1 \quad 3 \times 3 \quad 3 \times 1$$

$$\delta W_i = \int_A \underline{\delta q}^T \cdot \underline{B}^T \cdot \underline{D} \cdot \underline{B} \cdot \underline{q} \, dA$$

1x2    2x3    3x3    3x2    2x1

$$\delta W_e = \underline{\delta q}^T \cdot \underbrace{\underline{N}^T(a, b)}_{\underline{I}} \cdot \underline{F} = \underline{\delta q}^T \cdot \underline{F}_{2 \times 1} = -\delta N_2 \cdot F$$

RW

$$\underline{\delta q}^T \cdot \underbrace{\int_A \underline{B}^T \underline{D} \underline{B} \, dA \cdot \underline{q}}_{\underline{K}} = \underline{\delta q}^T \cdot \underline{F}$$

SOLVE  $\underline{K} \underline{q} = \underline{F}$  FOR  $\underline{q} = \begin{Bmatrix} u_2 \\ N_2 \end{Bmatrix} \rightarrow u_2 = 1.08 \cdot 10^{-3} \text{ mm}$

## Contents

---

- [Sol](#)
- [Data](#)

```
close all
clear variables
home
```

---

## Sol

---

```
syms x y z u_2 v_2 t a b E nu F

q = [u_2; v_2];
B = [
    y/a/b 0;
    0 x/a/b;
    x/a/b y/a/b
];
D = E/(1-nu^2)*[
    1 nu 0;
    nu 1 0;
    0 0 (1-nu)/2
];
F_v = [
    0;
    -F
];
RHS = F_v;
K = int(...  
    int( ...  
    int( ...  
        transpose(B)*D*B, x, 0, a ...  
    ), y, 0, b ...  
    ), z, 0, t);  
  
LHS = K*q;
```

---

## Data

---

```
t = 1;
a = 4;
b = 3;
E = 72000;
nu = .3;
F = 100;

sol = solve(subs(LHS) == subs(RHS), q);

u_2_d = double(sol.u_2)
```

---

```
u_2_d =
0.001084801369375
```

