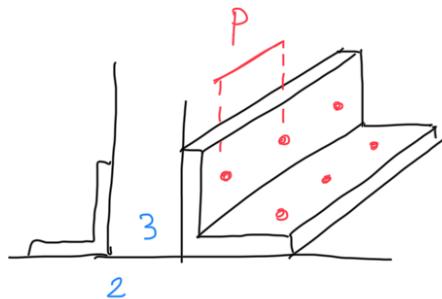
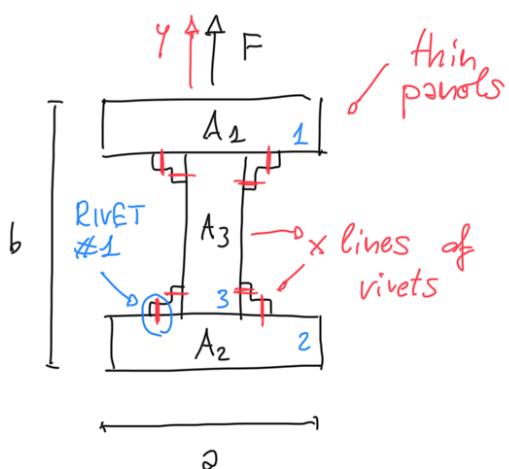


EX 10 - Ribs and Junctions

1) EXAM 05/07/2023



let's find the shear stress in rivet #1

DATA

$$J_{xx} = 15 \times 10^4 \text{ Nmm}$$

$$A_1 = A_2 = A = 5000 \text{ mm}^2$$

$$A_3 = 3500 \text{ mm}^2$$

$$P = 100 \text{ mm}$$

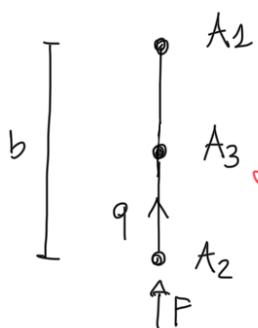
$$\phi = 7 \text{ mm} \quad \text{rivet diameter}$$

$$d = 500 \text{ mm}$$

$$b = 260 \text{ mm}$$

$$F = 1000 \text{ N}$$

- Shear Flakes in the panels

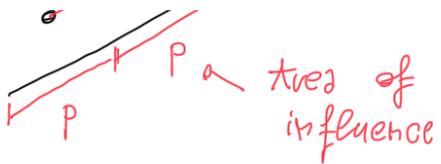
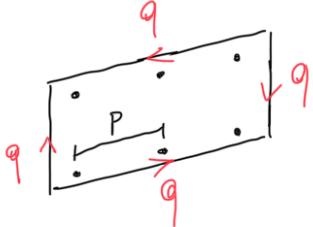


$$q = -F \frac{S_{x1}}{J_{xx}} = -F \cdot \left(-\frac{A \frac{b}{2}}{J_{xx}} \right)$$

here there is not a stringer (i.e. axial load), thus q is constant along all the panel side.

- Plate 3



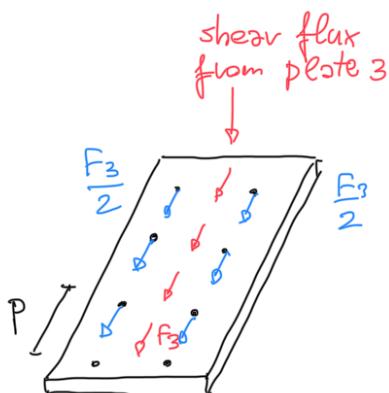


$$F_3 = q \cdot P \quad \tau_3 = \frac{F_3}{A_{\text{rivet}}} = \frac{q \cdot P}{\pi \left(\frac{\phi}{2}\right)^2}$$

this is

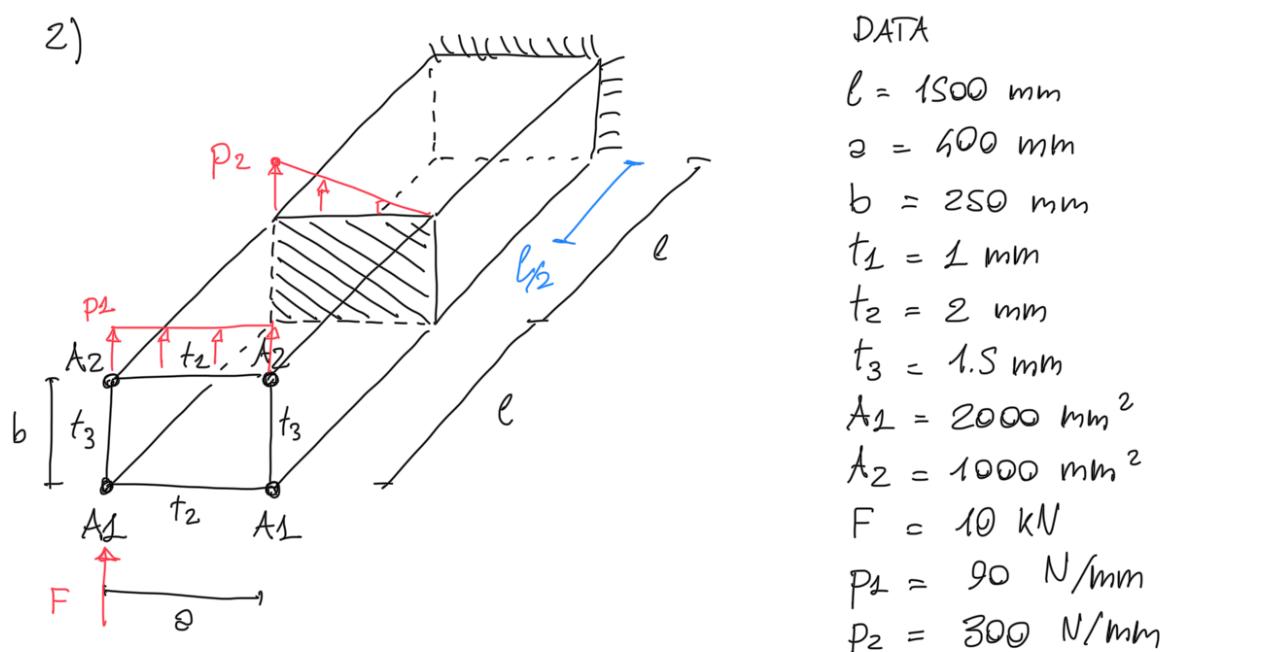
- Plate 2

$$\int q \cdot P$$



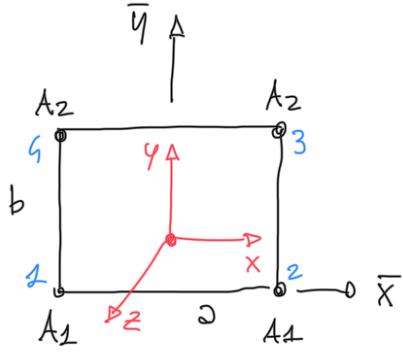
$$F_2 = \frac{F_3}{2} = \frac{q \cdot P}{2}$$

$$\tau_1 = \frac{F_2}{A_{\text{rivet}}} = \frac{\frac{1}{2} q \cdot P}{\pi \left(\frac{\phi}{2}\right)^2}$$



- q_i and σ_{zz} at $l/2$ from the clamp?
- Plot the internal actions in the rib

- Section Properties



$$\bar{y}_c = \frac{2A_2 b}{2A_1 + 2A_2} = \frac{1}{3}b$$

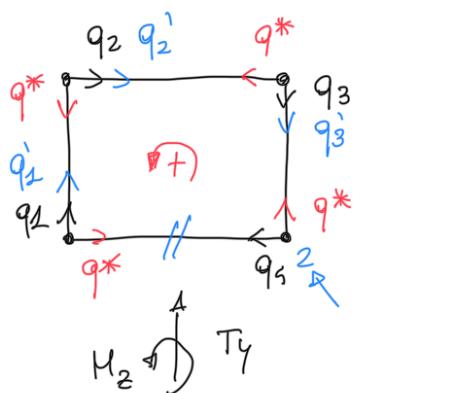
$$J_{xx} = 2A_2 \left(\frac{2}{3}b\right)^2 + 2A_1 \left(-\frac{1}{3}b\right)^2 = \frac{4}{3}A_2 b^2$$

$$S_{x1} = S_{x2} = -A_1 \cdot \frac{b}{3} = -\frac{2}{3}A_2 b$$

$$S_{x3} = S_{x6} = \frac{2}{3}A_2$$

- Open Cell Fluxes

Let's compute them for a generic T_y and M_z



$$q_1' = -T_y \frac{S_{x1}}{J_{xx}} = \frac{T_y}{2b} = -q_3'$$

$$q_2' = \emptyset$$

- Moment Equilibrium wrt ②

$$LHS = M_z - T_y \cdot \frac{2}{2}$$

$$RHS = 2q^* \Omega_{cell} - 2q_1' \Omega_1$$

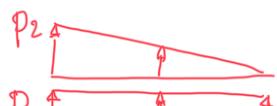
$$\Omega_{cell} = 2 \cdot b$$

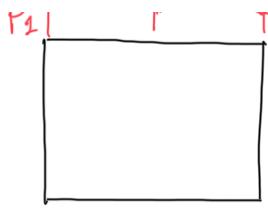
$$\Omega_1 = \frac{2 \cdot b}{2}$$

$$M_z - T_y \frac{2}{2} = 2q^* b - 2 \cancel{\frac{T_y}{2b} \cdot \frac{2}{2}}$$

$$q^* = \frac{M_z}{2ab}$$

- Lumped Forces

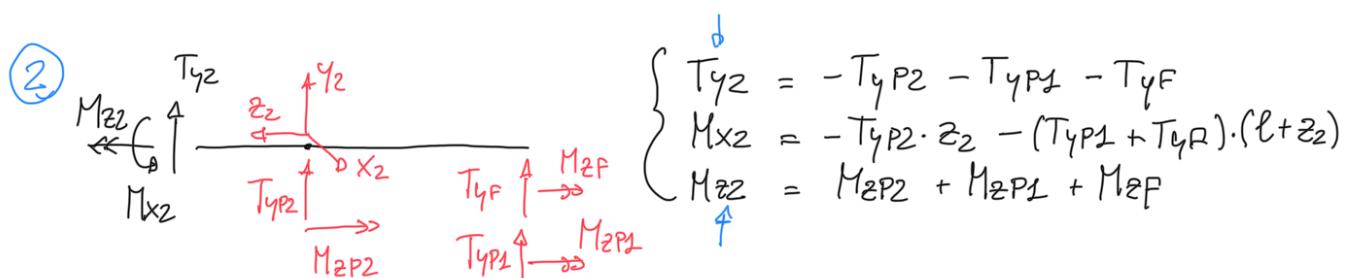
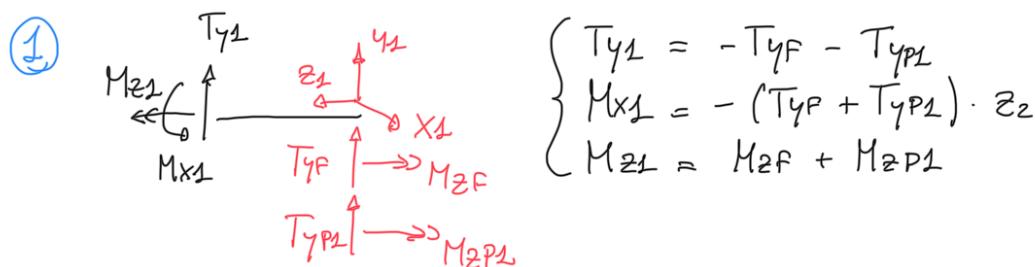
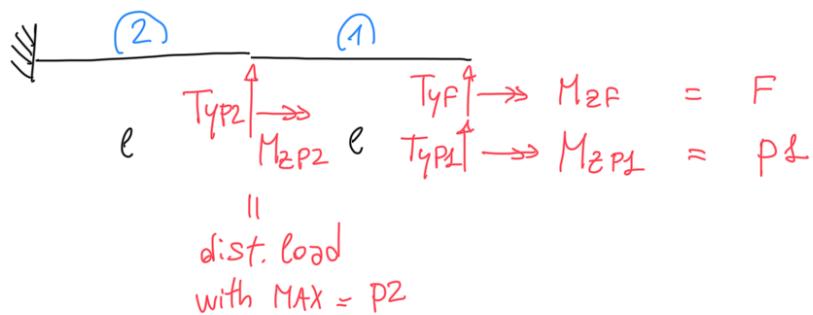




$$T_{yF} = T \\ M_{zF} = -F \cdot \frac{d}{2} \\ I_{yP1} = p_1 \cdot d \\ M_{zP1} = \cancel{d}$$

$T_{yP2} = \frac{1}{2} p_2 \cdot d$ $M_{zP2} = -\left(\frac{1}{2} p_2 \cdot d\right) \cdot \frac{1}{6} d$

- Internal Actions in the beam



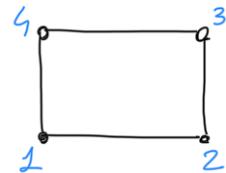
- Shear fluxes in $z_2 = \frac{l}{2}$

$$q_2 = q_2' - q^* = \frac{T_{y2}}{2b} - \frac{M_{z2}}{2zb} = -262 \text{ N/mm}$$

$$q_2 = q_2' - q^* = -30 \text{ N/mm}$$

$$q_3 = q_3' - q^* = 182 \text{ N/mm}$$

$$q_4 = -q^* = -30 \text{ N/mm}$$

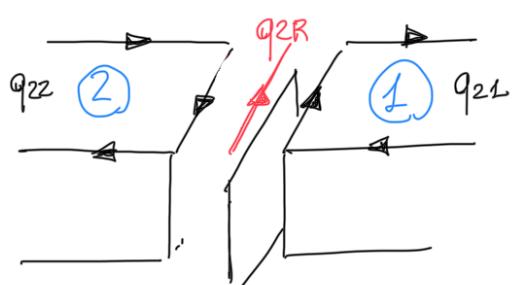


- Axial Stress in stringers at $z_2 = l/2$

$$\sigma_{221} = -\frac{M_{x2}}{J_{xx}} \cdot \frac{1}{3} b = 168.5 \text{ MPa} = \sigma_{221}$$

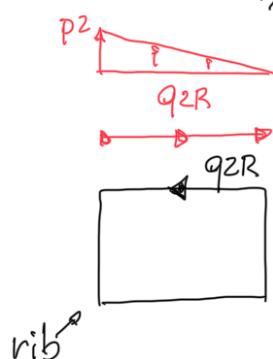
$$\sigma_{223} = \sigma_{224} = \frac{M_{x2}}{J_{xx}} \cdot \frac{2}{3} b = -297 \text{ MPa}$$

- Internal Actions in the rib



$$q_{22} = q_{21}$$

$$q_{2R} = q_{22} - q_{21}$$



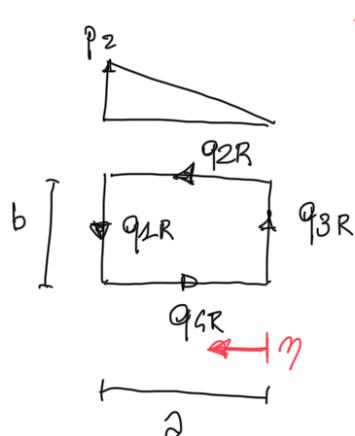
the rib equilibrates :

- the external force P_2
- the shear fluxes jump in the panel

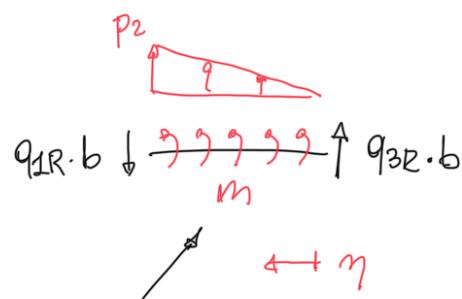
the rib must be in equilibrium

- Beam model of the rib

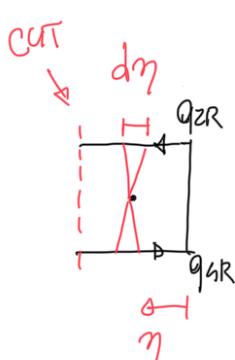
This makes it easier to estimate the internal actions in the rib, along one of its sides



$$TBN: q_{2R} = q_{3R} = q^*$$



distributed bending moment given by q_{2R} and q_{GR}



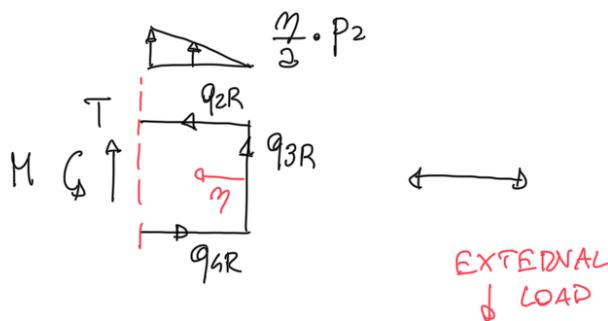
$$m = 2 q_{2R} \cdot \frac{h}{2} + 2 q_{GR} \cdot \frac{h}{2} =$$

$$= 2 (q_{2R} + q_{GR}) \cdot \left(\frac{1}{2} \frac{b}{2} d\eta \right)$$

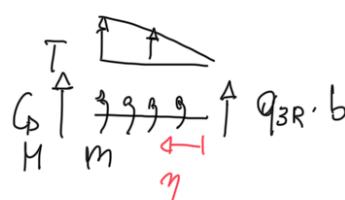
area of the triangle

$d\eta = 1 \rightarrow m = \text{bending moment} \times \text{unit length}$ given by q_{2R} and q_{GR}

- Internal Actions in the rib



$$\frac{m}{2} \cdot P_2$$

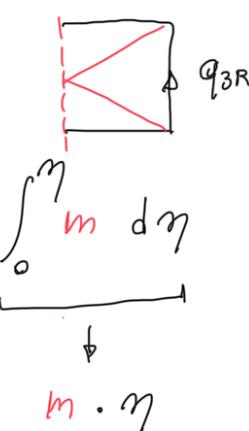


EXTERNAL LOAD

$$T = -q_{3R} \cdot b - \frac{1}{2} \left(\frac{m}{2} \cdot P_2 \right) \cdot \eta$$

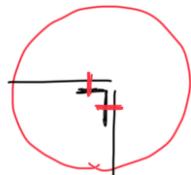
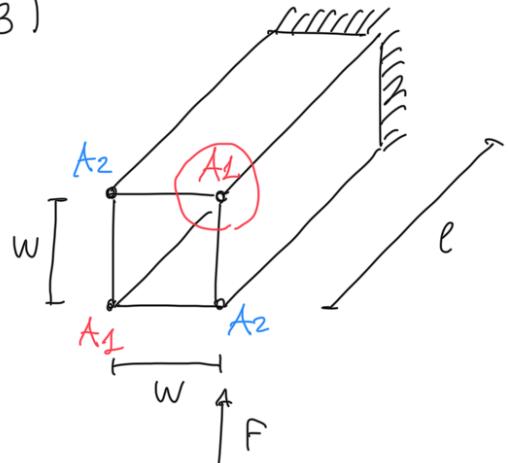
$$M = -2 q_{3R} \left(\frac{1}{2} b \eta \right) - \left(\frac{1}{2} \left(\frac{m}{2} \cdot P_2 \right) \eta \right) \cdot \left(\frac{1}{3} \eta \right) - \int_0^\eta m d\eta$$

EXTERNAL LOAD



$$m \cdot \eta$$

3)



DATA

$$A_2 = 300 \text{ mm}^2 = 2A$$

$$A_2 = 150 \text{ mm}^2 = A$$

$t = 1.5 \text{ mm}$ for all the panels

$$w = 100 \text{ mm}$$

$$E = 72000 \text{ MPa}$$

$$\nu = 0.3$$

$$l = 1500 \text{ mm}$$

$$d_1 = 2 \text{ mm}, p_1 = 10 \text{ mm}$$

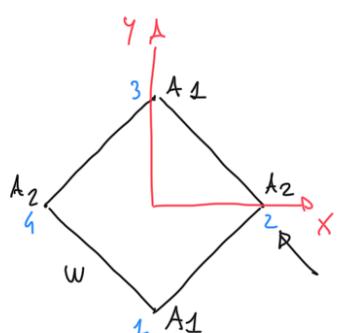
$$d_2 = 3 \text{ mm}, p_2 = 12 \text{ mm}$$

$$F = 15 \text{ kN}$$

$$\sigma_{zz} \text{ at } \frac{l}{2} ?$$

shear stress in the rivets?

- Section properties



if we rotate the section, it becomes symmetric wrt x and y $\rightarrow J_{xy} = \phi$

$$J_{xx} = 2A_1 \left(\frac{\sqrt{2}}{2}w\right)^2 = A_1 w^2 = 2Aw^2$$

$$J_{yy} = 2A_2 \left(\frac{\sqrt{2}}{2}w\right)^2 = A_2 w^2 = Aw^2$$

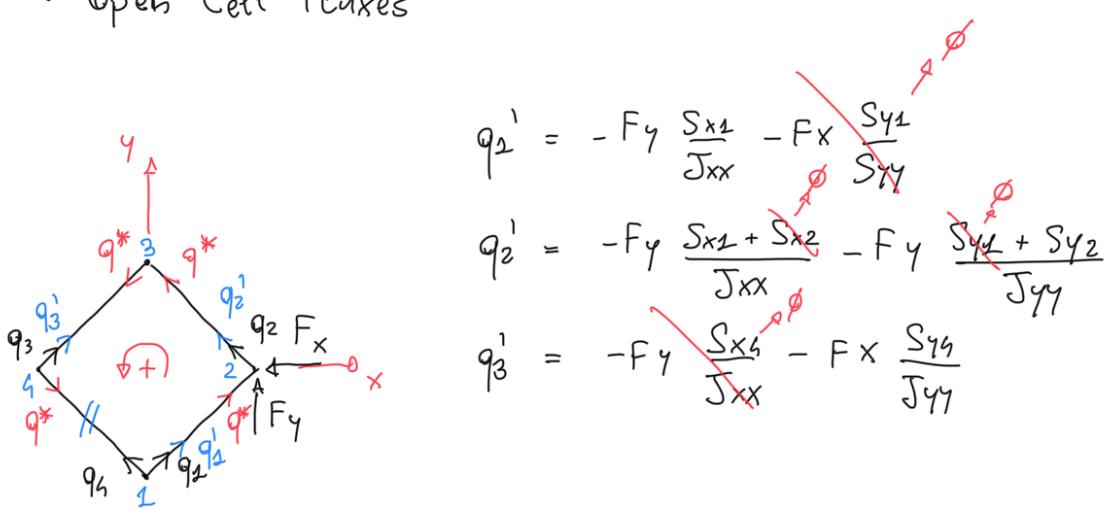
$$S_{x1} = -S_{x3} = -A_1 \frac{\sqrt{2}}{2}w = -\sqrt{2}Aw$$

$$S_{x2} = S_{x4} = \phi$$

$$S_{y1} = S_{y3} = \phi$$

$$S_{y2} = -S_{y4} = A_2 \frac{\sqrt{2}}{2}w = \frac{\sqrt{2}}{2}Aw$$

- Open Cell Fluxes



- Moment Equivalence wrt ②

$$LHS = \phi$$

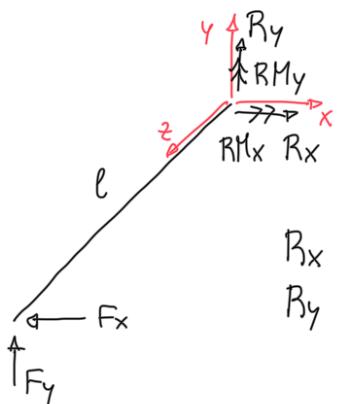
$$RHS = 2q^* w^2 - 2q_3^1 \frac{w^2}{2}$$

$$q^* = \frac{q_3^1}{2} = \frac{1}{2} \left(F_x \cdot \frac{\sqrt{2}}{2} w \right) = \frac{\sqrt{2}}{2} F_x$$

- Closed Cell

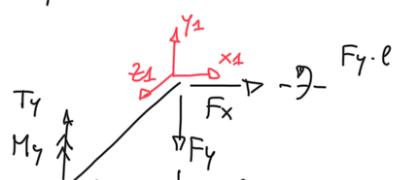
$$\begin{aligned} q_L &= q_2^1 + q^* \\ q_2 &= q_2^1 + q^* \\ q_3 &= q_3^1 - q^* \\ q_4 &= -q^* \end{aligned}$$

- Reaction Forces and Internal Actions in the beam



we don't care about RM_z because it doesn't act on stringers

$$\begin{aligned} R_x &= F_x & RM_x &= F_y \cdot l \\ R_y &= -F_y & RM_y &= F_x \cdot l \end{aligned}$$



$$\begin{aligned} T_x &= -F_x \\ T_y &= F_y \\ M_x &= F_y (z - l) \end{aligned}$$

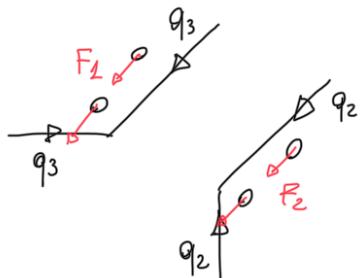
$$\text{M}_x \text{ } T_x \iff \text{F}_x \cdot l \quad \text{M}_y = \text{F}_x (z - l)$$

- Stress in stringers at $z = \frac{l}{2}$

$$\sigma_{zzL} = + \frac{M_x(\ell)}{J_{xx}} \cdot \gamma = + \frac{F_y \cdot \frac{\ell}{2}}{J_{xx}} \left(+ \frac{\sqrt{2}}{2} w \right) = 187.5 \text{ MPa} = - \sigma_{zz3}$$

$$\sigma_{222} = - \frac{M_y}{J_{yy}} \cdot x = + \frac{(+F_x \cdot z_2)}{J_{yy}} \cdot \left(\frac{\sqrt{2}}{2} w \right) = 375.0 \text{ MPa} = - \sigma_{224}$$

- ## • Shear Stress in Rivets



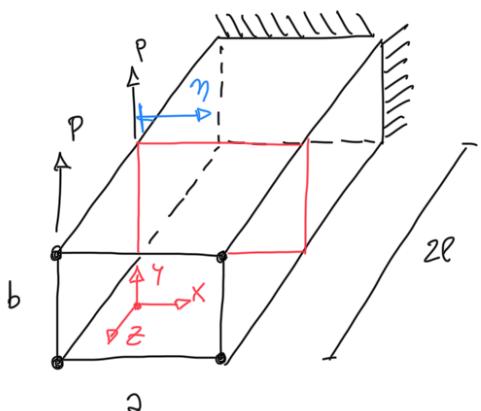
$$F_2 = q_3 \cdot p_2$$

$$F_2 = q_2 \cdot d_2$$

$$\sigma_2 = \frac{F_1}{\pi \left(\frac{d_1}{2}\right)^2} = \frac{93 P_1}{\pi \left(\frac{d_1}{2}\right)^2} = 119.37 \text{ MPa}$$

$$\tilde{\gamma}_2 = \frac{F_2}{\pi \left(\frac{d_2}{2} \right)^2} = \frac{q_2 \cdot P_2}{\pi \left(\frac{d_2}{2} \right)^2} = 190,99 \text{ MPa}$$

4)



DATA

$z = 500$ mm

$$b = 250 \text{ m}$$

$$t = 0.6 \text{ mm}$$

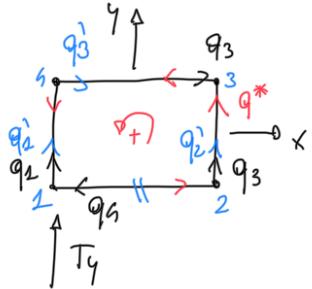
$$A = 500 \text{ mm}^2$$

$$l = 2000 \text{ mm}$$

$$P = 1000 \text{ N}$$

Let's find the moment in the rib
at $\eta = \frac{2}{3}$

- Open Cell Fluxes



$$q_1^1 = q_2^1 = \frac{T_y}{2b} \quad q_3^1 = \phi$$

- Moment Equivalence wrt ①

$$\text{LHS} = \phi$$

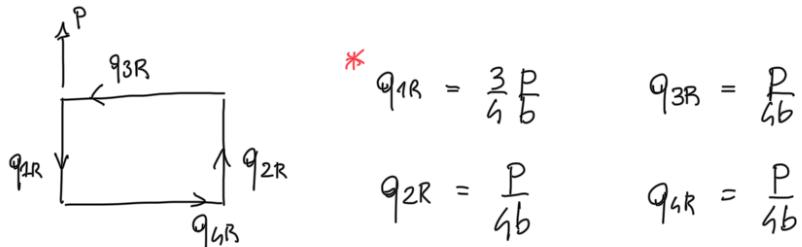
$$\text{RHS} = 2q^* ab + 2 \frac{T_y}{2b} \cdot \frac{ab}{2} \quad q^* = -\frac{T_y}{4b}$$

- Total Fluxes

$$q_1 = q_1^1 - q^* = \frac{T_y}{2b} + \frac{T_y}{4b} = \frac{3}{4} \frac{T_y}{b} \quad q_3^* = -q^* = \frac{T_y}{4b}$$

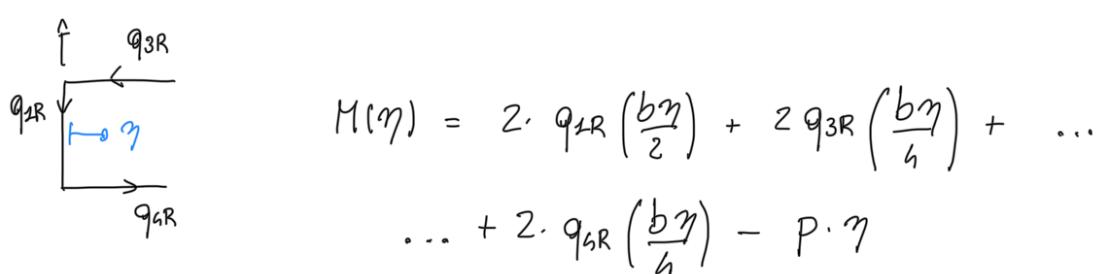
$$q_2 = q_2^1 + q^* = \frac{T_y}{4b} \quad q_4 = -q^* = \frac{T_y}{4b}$$

- Internal Action in the vib



$$q_{1R} = \frac{3}{4} \frac{P}{b} \quad q_{3R} = \frac{P}{4b}$$

$$q_{2R} = \frac{P}{4b} \quad q_{4R} = \frac{P}{4b}$$



$$M(\eta) = 2 \cdot q_{1R} \left(\frac{b\eta}{2} \right) + 2 q_{3R} \left(\frac{b\eta}{4} \right) + \dots$$

$$\dots + 2 \cdot q_{4R} \left(\frac{b\eta}{4} \right) - P \cdot \eta$$

$$M\left(\frac{\eta}{3}\right) = \phi$$