

1) Backwards Euler

$$y' = 4t - 3t^2 \quad y(t_0) = 6 \quad \Delta t = 0.33 \\ t_0 = 1$$

$$t_1 = 1.33 \quad y_1 = y_0 + \Delta t y_1' \\ = 6 + 0.33(4(1.33) - 3(1.33)^2) = 6.004$$

$$t_2 = 1.67 \quad y_2 = y_1 + \Delta t y_2' \\ = 6.004 + 0.33(4(1.67) - 3(1.67)^2) = 5.447$$

$$t_3 = 2.0 \quad y_3 = y_2 + \Delta t y_3' \\ = 5.447 + 0.33(4(2.0) - 3(2.0)^2) = \underline{4.107}$$

2) Thick-walled cylinder w/  $r_i = 0.5 \text{ cm}$   $r_o = 0.55 \text{ cm}$

$$p = 15 \text{ MPa}$$

- thin-walled assumption:

$$r_{avg} = 0.525 \text{ cm}$$

$$\sigma_\theta = \frac{pR}{\delta} \quad \sigma_r = -\frac{p}{2} \quad \sigma_z = \frac{pR}{2\delta}$$

$$\sigma_\theta = \frac{(15)(0.525)}{0.05} = \underline{157.5 \text{ MPa}}$$

$$\sigma_r = -\frac{15}{2} = \underline{-7.5 \text{ MPa}}$$

$$\sigma_z = \frac{p}{2} = \underline{7.5 \text{ MPa}}$$

Thick-walled @ inner wall

$$\sigma_{\theta} = \frac{\left(\frac{r_o}{r}\right)^2 + 1}{\left(\frac{r_o}{r_i}\right)^2 - 1} \rho$$

$$\sigma_r = -\rho \frac{\left(\frac{r_o}{r}\right)^2 - 1}{\left(\frac{r_o}{r_i}\right)^2 - 1}$$

$$\sigma_z = \frac{\rho}{\left(\frac{r_o}{r_i}\right)^2 - 1}$$

$$\sigma_{\theta} = 15 \frac{\left(\frac{0.55}{0.5}\right)^2 + 1}{\left(\frac{0.55}{0.5}\right)^2 - 1} = 157.7 \text{ MPa}$$

$$\sigma_r = -15 \frac{\left(\frac{0.55}{0.5}\right)^2 - 1}{\left(\frac{0.55}{0.5}\right)^2 - 1}$$

$$\sigma_r = -15 \text{ MPa}$$

$$\sigma_z = \frac{15}{\left(\frac{0.55}{0.5}\right)^2 - 1} = 71.4 \text{ MPa}$$

3) Max stress in fuel pellet due to thermal exp.?

$$\Delta T = T_o - T_i = 425 \text{ K} \quad \alpha_F = 12 \times 10^{-6} / \text{K} \quad R_F = 0.5 \text{ cm} \quad E_F = 180 \text{ GPa}$$

$$\nu = 0.28$$

$$\sigma_r = -\sigma^* (1 - \eta^2)$$

$$\sigma^* = \frac{\alpha E (T_o - T_i)}{4(1 - \nu)}$$

$$\sigma_{\theta} = -\sigma^* (1 - 3\eta^2)$$

$$\eta = \frac{r}{R_F}$$

$$\sigma_z = -2\sigma^* (1 - 2\eta^2)$$

$$\sigma^* = \frac{(12 \times 10^{-6})(180 \times 10^9)(425)}{4(1 - 0.28)} = 318.75 \text{ MPa}$$

$$\max = \sigma_{\theta} \text{ @ } r = R_F$$

$$\sigma_{\theta} = -\sigma^* (1 - 3(1)^2) = 637.5 \text{ MPa}$$

4) Cladding tube w/ thermal expansion  
- no pressure

$$R_i = 0.6 \text{ cm} \quad t_c = 0.1 \text{ cm} \quad E = 250 \text{ GPa} \quad \nu = 0.3$$

$$\alpha_c = 15 \times 10^{-6} \text{ } ^\circ\text{C}^{-1} \quad \bar{T}_c = 400 \text{ K} \quad T_c^0 = 300 \text{ K}$$

Stress @  $r = 0.62$ ?

$$\begin{aligned} \sigma_r &= \frac{\Delta T}{2} \frac{\alpha E}{1-\nu} \left( \frac{r}{R_i} - 1 \right) \left( 1 - \frac{R_i}{t_c} \left( \frac{r}{R_i} - 1 \right) \right) \\ &= \frac{300}{2} \frac{(15 \times 10^{-6})(250 \times 10^3)}{1-0.3} \left( \frac{0.62}{0.6} - 1 \right) \left( 1 - \frac{0.6}{0.1} \left( \frac{0.62}{0.6} - 1 \right) \right) \end{aligned}$$

$$\sigma_r = (803.6) (0.0267) = 21.4 \text{ MPa}$$

$$\sigma_\theta = \frac{\Delta T}{2} \frac{\alpha E}{1-\nu} \left( 1 - 2 \frac{R_i}{t_c} \left( \frac{r}{R_i} - 1 \right) \right) = \sigma_E$$

$$\sigma_\theta = (803.6) \left( 1 - 2 \frac{0.6}{0.1} \left( \frac{0.62}{0.6} - 1 \right) \right) = 492.14 \text{ MPa}$$

5) Gap Thickness change

$$\alpha_F = 12 \times 10^{-6} \quad \alpha_c = 15 \times 10^{-6}$$

$$\bar{T}_F = 925 \text{ K} \quad \bar{T}_c = 550 \text{ K}$$

$$R_i = 0.5 \text{ cm} \quad \bar{R}_c = 0.58 \text{ cm}$$

$$T_0^{CF} = 300 \text{ K}$$

$$t_f = 0.03 \text{ cm} \quad t_c = 0.1 \text{ cm}$$

$$\Delta t_g = \bar{R}_c \alpha_c (\bar{T}_c - T_0^c) - R_i \alpha_F (\bar{T}_c - T_0^F)$$

$$\Delta t_c = (0.58)(15 \times 10^{-6})(250) = 0.002175$$

$$\Delta t_F = (0.5)(12 \times 10^{-6})(625) = 0.00375$$

$$t_g = 0.03 + (0.002175 - 0.00375) = \underline{\underline{0.0284 \text{ cm}}}$$