

**Question 1:**

$U_3Si_5$  is a uranium silicide fuel being considered for use in light water reactors. It has a thermal conductivity of  $12.5 \text{ W/(m K)}$  and a density of Uranium metal of  $7.5 \text{ g/cm}^3$ . Answer the following questions

- a) What is the fissile isotope in  $U_3Si_5$ ? What would be the enrichment of this isotope in the natural (unenriched) form of the fuel? (7 points)

$U-235$

0.75%

- b) What enrichment would be required for  $U_3Si_5$  to have the same energy release rate of  $U_3Si_2$  enriched to 3% with a neutron flux of  $3.2 \times 10^{13} \text{ n/(cm}^2 \text{ s)}$ ? You can assume that  $U_{235}$  has a negligible impact on the total molar mass of U in the fuel (15 points)

$$Q = E_f N_f \sigma_f \phi_{th}$$

Assume  $Q_{U_3Si_5} = Q_{U_3Si_2}$

$$E_1 N_1 \sigma_1 \phi_{th} = E_2 N_2 \sigma_2 \phi_{th}$$

$$N_1 = N_2$$

$$N_f = \frac{\rho_{Na} \rho}{M_f}$$

$$\frac{\rho_1 \rho_{U_3Si_5}}{M_{U_3Si_5}} = \frac{\rho_2 \rho_{U_3Si_2}}{M_{U_3Si_2}}$$

$$\rho_{U_3Si_5} = 7.5 \text{ g/cm}^3$$

$$\rho_{U_3Si_2} = 11.31 \text{ g/cm}^3$$

$$M_{U_3Si_5} = 854 \text{ g/mole}$$

$$M_{U_3Si_2} = 770 \text{ g/mole}$$

$$\rho_1 = \frac{\rho_2 \rho_{U_3Si_2} M_{U_3Si_5}}{M_{U_3Si_2} \rho_{U_3Si_5}}$$

-5, the density of U was provided, not the density

$$\rho_1 = \frac{0.03(11.31) 854}{(7.5) 770} = \boxed{5.02\% \text{ enriched}}$$

- c) How would you rank  $U_3Si_5$  as a potential fuel compared to  $U_3Si_2$ ? Why? (8 points)

$U_3Si_5$  is a worse ~~fuel~~ fuel than  $U_3Si_2$  because it requires a higher enrichment than  $U_3Si_2$  to have the same energy release rate.

-3, thermal conductivity?

**Question 2:**

Consider a fuel rod with a pellet radius of 4.5 mm, an 80 micron gap, and a zircaloy cladding thickness of 0.6 mm. It is experiencing a linear heat rate of 250 W/cm with a coolant temperature of 580 K. The gap is filled with He and 5% Xe and the coolant conductance is 2.5 W/(cm<sup>2</sup> K).

a) What is the surface temperature of the fuel rod? (15 points)

$$T_{co} = \frac{LHR}{2\pi R_f h_{cool}} + T_{cool}$$

$$T_{co} = \frac{250}{2\pi (0.45)(2.5)} + 580 = 615.37 \text{ K}$$

$$T_c = \frac{LHR \cdot t_c}{2\pi R_f k_c} + T_{co}$$

$$T_c = \frac{250(0.06)}{2\pi (0.45)(0.17)} + 615.37 = 646.57 \text{ K}$$

$$h_{gap} = \frac{k_{gap}}{1 - y_{He} - y_{Xe}}$$

$$k_{gap} = (16e-6(646.57)^{0.74})^{0.95} (0.7e-6(646.57)^{0.74})^{0.05}$$

$$h_{gap} = \frac{0.0023}{80e-4 \text{ cm}} = 0.2875$$

$$T_s = \frac{LHR}{2\pi R_f h_{gap}} + T_c = \frac{250}{2\pi (0.45)(0.2875)} + 646.57 = 954.11 \text{ K}$$

-1, Don't round until the end. You are off by 4 K

b) Assume the pellet is made from Uranium Nitride. What is the maximum stress experienced by the pellet, given that uranium nitride has  $E = 246.7$  GPa,  $\nu = 0.25$ , and  $\alpha = 7.5e-6$  1/K? (10 points)

$$T_0 = \frac{LHR}{4\pi k} + T_s$$

$$T_0 = \frac{250}{4\pi (0.2)} + 954.11$$

$$T_0 = 1053.58 \text{ K}$$

$$\sigma^* = \frac{\alpha E (T_0 - T_s)}{4(1-\nu)}$$

$$\sigma^* = \frac{(7.5e-6 \text{ 1/K})(246.7 \text{ GPa})(1053.58 - 954.11) \text{ K}}{4(1 - 0.25)}$$

$$\sigma^* = 0.0613$$

Maximum Stress will be hoop stress  $r = 1$  at max

$$\sigma_{\theta\theta} = -0.0613(1 - 3(1)^2) = 0.1226 \text{ GPa}$$

c) Would you expect this stress to be higher or lower if the pellet was  $\text{UO}_2$ ? Why? (5 points)

The stress would be higher for  $\text{UO}_2$  because it has a lower thermal conductivity which would make the centerline temperature much hotter than the surface temperature and cause greater stress.

d) What assumptions were made in your calculations for a) and b)? (5 points)

- linear temperature profile across the gap
- Steady state
- axisymmetric
- $T$  is constant in  $z$
- Thermal conductivity is independent of temperature

**Question 3:**

Consider the stress state in a zircaloy fuel rod pressurized to 6 MPa with an average radius of 5.6 mm and a cladding thickness of 0.6 mm.

- a) What assumptions are made in the thin walled cylinder approximation for the stress state? (5 points)

$$\overline{\sigma_\theta} = \frac{pR}{\delta} \quad \overline{\sigma_z} = \frac{pR}{2\delta} \quad \overline{\sigma_r} = -\frac{1}{2}p$$

-5, isotropic, small strain, stress constant through thickness

- b) Calculate all three components of the stress using the thin walled cylinder approximation. (10 points)

$$\begin{aligned} \overline{\sigma_\theta} &= \frac{6 \text{ MPa} (5.6 \text{ mm})}{0.6 \text{ mm}} = 56 \text{ MPa} \\ \overline{\sigma_z} &= \frac{6 \text{ MPa} (5.6 \text{ mm})}{2(0.6 \text{ mm})} = 28 \text{ MPa} \\ \overline{\sigma_r} &= -\frac{1}{2} (6 \text{ MPa}) = -3 \text{ MPa} \end{aligned}$$

- c) Quantify how accurate the thin walled cylinder approximation is for the cladding. Would the thin walled cylinder approximation be conservative if used to estimate if the cladding would fail? (10 points)

Using thick wall to calculate at max stress

$$\sigma_{\theta\theta}(r) = p \frac{(R_o/r)^2 + 1}{(R_o/R_i)^2 - 1} = 6 \text{ MPa} \frac{(5.9 \text{ mm})^2 + 1}{(5.3 \text{ mm})^2 - 1} = 56.16 \text{ MPa}$$

$$\% \text{ error} = \frac{56.16 - 56}{56} = 2.85\%$$

-4, Calculate stress at TWO radii and compare to see if stress is constant

Not conservative because thin walled is smaller than the thick wall calculations, so you may predict that the cladding will not fail even though it will.

- d) Write the stress and strain tensors for the stress state in the thin walled cylinder, with  $E = 70 \text{ GPa}$  and  $\nu = 0.41$ . (10 points)

$$\begin{aligned} \sigma_{rr} &= -p \frac{(R_o/r)^2 - 1}{(R_o/R_i)^2 - 1} = -6 \text{ MPa} \\ \sigma_{zz} &= p \frac{1}{(R_o/R_i)^2 - 1} = 25.08 \text{ MPa} \\ \epsilon_{rr} &= \frac{1}{E} (\sigma_{rr} - \nu(\sigma_{\theta\theta} + \sigma_{zz})) = -0.268 \\ \epsilon_{\theta\theta} &= \frac{1}{E} (\sigma_{\theta\theta} - \nu(\sigma_{rr} + \sigma_{zz})) = 0.691 \\ \epsilon_{zz} &= \frac{1}{E} (\sigma_{zz} - \nu(\sigma_{\theta\theta} + \sigma_{rr})) = 0.0645 \end{aligned}$$

$$\begin{aligned} \sigma &= \begin{bmatrix} \sigma_{\theta\theta} \\ \sigma_{rr} \\ \sigma_{zz} \end{bmatrix} = \begin{bmatrix} 56.16 \\ -6 \\ 25.08 \end{bmatrix} \text{ MPa} \\ \epsilon &= \begin{bmatrix} \epsilon_{\theta\theta} \\ \epsilon_{rr} \\ \epsilon_{zz} \end{bmatrix} = \begin{bmatrix} 0.691 \\ -0.268 \\ 0.0645 \end{bmatrix} \end{aligned}$$

-4, Show stress and strain in tensor form  
-1, math error in strain calculation