

NUCE 497 Midterm
Resubmit

Bryan Evers

* Corrections noted w/ yellow highlighter

* Note: The answer key was consulted in making these corrections.

Balayev
Engels

1.a) U-235

Sii: 28.1 u

Ui: 238 u

e_n: 0.72%

$$\cancel{U_3Si_5 = 5(28.1u) + 3(238u) = 854.5 \text{ au}}$$

② wt % 235:

$$\frac{3 \cdot 238u \cdot 72\%}{854.5u} = 0.59\% \text{ wt}$$

1.b) $Q = E_F \cdot N_F \cdot \sigma_F \cdot \phi_{th}$

$$Q_1 = Q_2$$

$$\cancel{E_F \cdot N_{U_3Si_5}(e_{U_3Si_5}) \cdot \phi_F \cdot \phi_{th}} = \cancel{E_F \cdot N_{U_3Si_2}(e_{U_3Si_2}) \cdot \cancel{\phi_F \cdot \phi_{th}}}$$

$$e_{U_3Si_5} = e_{U_3Si_2} \cdot \frac{N_{U_3Si_2}}{N_{U_3Si_5}}$$

$$= 3\% \cdot \frac{769.9}{854.5} = 2.7\%$$

(lab)

$$Q = E_F N_F \sigma_F \phi_{th}$$

$$Q_1 = Q_2$$

$$N_{F_1} = N_{F_2}$$

$$N_F = 3 \cdot g^* N_a \cdot \delta^* \mu M$$

$$MM_{U_3Si_2} = (238)3 + (28)2 = \cancel{770 \text{ g/mol}}$$

$$MM_{U_3Si_5} = (238)3 + (28)5 = \cancel{854 \text{ g/mol}}$$

$$\cancel{\frac{7 \cdot (0.03) \cdot N_a \cdot (12.2 \text{ g/cm}^3)^3}{11.31}} / 11.31$$

$$= 7(7) N_a (7.5 \text{ g/cm}^3) / 854 \text{ g/mol}$$

$$\frac{0.03 \cdot \cancel{12.2} / 770}{7.5 / 854} = \cancel{5.0\%} / \cancel{5.11\%}$$

* But since δ_u given instead of ~~δ_d~~ ,

use MM of U fraction only!?

$$\frac{0.03 \cdot \cancel{12.2} / \cancel{770}}{7.5 / (238.3)} = \boxed{\cancel{4.5\%}} \rightarrow \boxed{4.2\%}$$

(b) ***Resubmit ***

This is just a question of $\frac{N_{us,iz}}{N_{us,ss}}$

as I realized early. I just got flustered
or something?

$$e_{us,ss} = e_{us,iz} \cdot \frac{\cancel{\Delta_a \cdot \delta u_{us,iz} \cdot \mu_3}}{\cancel{\Delta_a \cdot \delta u_{us,ss} \cdot \mu_3}}$$

$$= 0.3 \cdot \frac{11.31}{7.5}$$

$$= \boxed{4.52 \text{ g/cc}}$$

	Thermal Cond (W/cm/K)	Density (g U/cm³)	Comparable Enrichment
U ₃ Si ₅	0.125	7.5	4.2%
U ₃ Si ₂	0.23	11.31	3%

~~U₂₃₅~~ Fuel efficiency of U₃Si₅ is higher!

$$4.2\% \cdot 7.5 \text{ g U/cm}^3 = \cancel{0.315} \text{ g U}_{235}/\text{cm}^3$$

$$11.31 \text{ g U}_{235} \cdot \cancel{0.3\%} = 0.339 \text{ g U}_{235}/\text{cm}^3$$

$$\Rightarrow \frac{0.339 - 0.315}{0.339}$$

7% less U₂₃₅

Used by switching
to U₃Si₅

U₃Si₅ thermal conductivity
and power density are
lower - see above table.

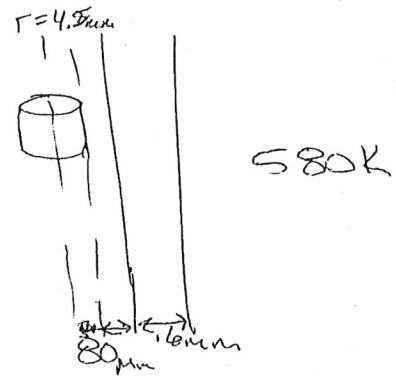
U₃Si₅ is a poorer fuel.

2.a)

$$T_{co} = T_{cool} + \frac{LHR}{2\pi R_F h_{cool}}$$

$$= 580K + \frac{2500W/cm}{2\pi(4.5mm) 2.5W/cm^2 \cdot K}$$

$$= \underline{615.3 \text{ K}}$$



* Assumed K_C
for 2π

$$T_{cs} = T_{ic} + \frac{LHR \cdot t_c}{2\pi R_F \cdot K_c}$$

$$= 615.3 \text{ K} + \frac{250 \text{ W/cm} \cdot 0.2 \text{ cm}}{2\pi(4.5 \text{ mm}) \cdot 0.17 \text{ W/cm/K}}$$

$$= \underline{646.5 \text{ K}}$$

$$T_s = T_{ic} + \frac{LHR}{2\pi R_F \cdot \left(\frac{K_{gap}}{t_g}\right)}$$

$$= 646.5 \text{ K} + \frac{250 \text{ W/cm}}{2\pi(4.5 \text{ mm}) \cdot \frac{0.027 \text{ W/cm/K}}{80 \text{ mm}}}$$

$$= \boxed{\underline{912.7 \text{ K}}}$$

$K_{gap} = 1.6 \times 10^{-6} \cdot T_K^{0.79}$

$= \underline{0.027 \text{ W/cm/K}}$

* Wrong Formula

2.b)

$$\Delta P = -E \cdot \Delta T \cdot \alpha$$

$$= -246.76 \text{ Pa} (-912.7 \text{ K} + 300 \text{ K}) 7.5 \times 10^{-6} \frac{1}{\text{K}}$$

Assume fuel
dimensions measured
at this temp

2.a) # Resubmit #

$$K_{gap} = \left(16E-6 \cdot T_{ic}^{0.79} \right)^{1-x_e} \cdot \left(0.7E-6 T_{ic}^{0.79} \right)^{x_e}$$

$$T_{ic} = 646.6 \text{ K}$$

$$x_e = 0.05$$

$$\Rightarrow K_{gap} = \left(16E-6 (646.6)^{0.79} \right)^{0.95} \cdot \left(0.7E-6 (646.6)^{0.79} \right)^{0.05}$$
$$= \underline{\underline{0.00227 \text{ W/cm-K}}}$$

$$T_s = 646.6 \text{ K} + \frac{250 \text{ W/cm}}{2\pi (4.5 \text{ mm}) 0.00227 \text{ W/cm-K} / 80 \mu\text{m}}$$
$$= \boxed{958.2 \text{ K}}$$

2. b) Assume the pellet is constrained:

$$\sigma = -E(\Delta T)\alpha$$

Hoop stress is largest:



$$\sigma_{\text{hoop}} = \frac{1}{2} \Delta T \frac{\alpha E}{1-\nu} \left(1 - 2 \frac{R_i}{\delta} \left(\frac{r}{R_i} - 1 \right) \right)$$

Max

stress

$$= \frac{1}{2} (912.7K - 300K) \frac{7.5 \times 10^6 \text{ K} + 246.76 \text{ Pa}}{1 - .25}$$

$$\cdot \left(1 - 2 \frac{4.5 \text{ mm}}{1.6 \text{ mm}} \left(1 - 1 \right) \right)$$

$$= \boxed{0.756 \text{ GPa @ surface}}$$

2.b)

* Resubmit *

I used the equation for a thin constrained rod instead of for a fuel pellet. I should have used:

$$\textcircled{1} \quad T(r) = \frac{LHR}{4\pi K} \left(1 - \frac{r^2}{R_F^2} \right) + T_s$$

$\underbrace{}$
 ΔT_{fuel}

$$\Rightarrow \Delta T = \frac{250 \text{ W/cm}}{4\pi \cdot (0.2 \text{ W/cm}\cdot\text{K})} \quad \leftarrow (\text{For U-N})$$

$$= \underline{99.47 \text{ K}}$$

$$\textcircled{2} \quad \sigma^* = \frac{\alpha E (T_b - T_s)}{4(1-\nu)}$$
$$= \frac{(7.5 \times 10^{-6} \text{ K}) \cdot (246.7 \text{ GPa}) \cdot (99.47 \text{ K})}{4(1-0.28)}$$

$$= \underline{61.4 \text{ } \frac{\text{M}}{\text{Pa}}}$$

$$\textcircled{3} \quad \sigma_{\text{res}}(r) = -\sigma^* (1 - 3r^2)$$

$$= -61.4 \text{ } \frac{\text{M}}{\text{Pa}} \cdot (1 - 3(1)^2)$$

$$= \boxed{123 \text{ MPa}}$$

2.c) U-N has a much \uparrow conductivity than $\text{UO}_2 \Rightarrow$ ~~low~~ lower T_s .

$\text{UO}_2 \sigma_{\infty}$ will be higher for this

~~*Resubmit*~~ Reason.

~~* Right answer, wrong reason. I understood it in my head. T_s only determined by T_{cool}, T_e lower for U-N due to $\uparrow K$ in the fuel.~~

2.d) Axisymmetry, isotropic material,

K independent of T , no geometry

change (such as gap size)

~~* Resubmit~~

Also, thin wall cyl w/ therm. expansion \propto to ϵ but not σ . (linear)

Also, small ϵ approximation.

Also, no internal pressure.

Also, σ constant through the wall.

Also, Young's Modulus: Assume $\frac{\sigma}{\epsilon}$ is linear, New steady state.

- long rod
- + No shear stress

3. a) Force is constant over length,

Axymmetric,

Isotropic material response,

$$\frac{\partial x}{\partial t} = 0, \text{ gravity is negligible}$$

b) ~~P = 6 MPa~~, $r = 5.6 \text{ mm}$, $R_o = 0.6 \text{ mm}$

(*) $\bar{\sigma}_0 = \frac{P \cdot r}{8} = \frac{(6 \text{ MPa}) \cdot 5.6 \text{ mm}}{8 \cdot 0.6 \text{ mm}} = 56 \text{ MPa}$

$$\bar{\sigma}_z = \frac{P \cdot r}{28} = \frac{(6 \text{ MPa}) \cdot 5.6 \text{ mm}}{28 \cdot (0.6 \text{ mm})} = 28 \text{ MPa}$$

$$\bar{\sigma}_r = -\frac{1}{2} P = -\frac{1}{2} \cdot 6 \text{ MPa} = -3 \text{ MPa}$$

c) Compare to thick wall solns & $r = R_o$

~~$\sigma_{rr} = -P = -6 \text{ MPa}$~~

$R_o = 5 \text{ mm}$

Wrong $R_o = 0.2 \text{ mm}$

~~$\sigma_{00} = P \cdot \frac{(R_o/R_F)^2 + 1}{(R_o/R_F)^2 - 1} = -59 \text{ MPa}$~~

~~59~~ MPa

~~$\sigma_{zz} = P \frac{1}{(R_o/R_F)^2 - 1} = 26.6 \text{ MPa}$~~

Wrong

Hoop stress most important, $\frac{56}{59} = 95\%$

σ_r off by factor of 2, σ_z is close.

correct

~~Thin wall should be conservative (but I find it is not in this case)~~

3.c)

* Resubmit *

Only need to look at $\sigma_{\theta\theta}$ b/c this will have the \gg stress. Also, must compare 2 points since thin wall is a single-point approx.

$$\sigma_{\theta\theta}(r) = P \cdot \frac{(R_o/r)^2 + 1}{(R_o/R_i)^2 - 1}$$

$$\sigma_{\theta\theta}(R_o) = \frac{(1)^2 + 1}{(\cancel{5.9}/\cancel{5.3}) - 1} = \boxed{50.1 \text{ MPa}}$$

$$\sigma_{\theta\theta}(R_i) = \frac{(5.9/5.3)^2 + 1}{(5.9/5.3)^2 - 1} = \boxed{56.1 \text{ MPa}}$$

$$\sigma_{\theta\theta}(R_o) \ll \overline{\sigma_\theta} < \sigma_{\theta\theta}(R_i)$$

$$\frac{56 - 50.1}{50.1} = 0.7\% \text{ too low @ outer radius}$$

$$\frac{56 - 50.1}{50.1} = 12\% \text{ too high @ inner radius}$$

* As an average value, thin-wall is conservative b/c it approximately matches the max stress value.

3.2) ~~$\sigma_{\theta}(R) = \frac{P \cdot R}{\delta}$~~

$$\epsilon = \begin{bmatrix} \frac{P \cdot R}{\delta} & 0 \\ 0 & \frac{P}{\delta} \end{bmatrix} =$$

3.3) What is the displacement??

$$\begin{bmatrix} \sigma_{rr} \\ \sigma_{\theta\theta} \end{bmatrix} = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu \\ \nu & 1-\nu \end{bmatrix} \begin{bmatrix} u_r, r \\ u_r/r \end{bmatrix}$$

$$= \frac{70 \text{ GPa}}{(1.41)(0.18)} \begin{bmatrix} -59 & 41 \\ 41 & -59 \end{bmatrix} \begin{bmatrix} u_r, r \\ u_r/r \end{bmatrix}$$

~~$= 200 \text{ GPa} \begin{bmatrix} 162.7 & 113.1 \\ 113.1 & 162.7 \end{bmatrix} \begin{bmatrix} u_r, r \\ u_r/r \end{bmatrix}$~~

Stress tensor

Strain tensor is

$\begin{bmatrix} u_r, r & 0 \\ 0 & u_r/r \end{bmatrix}$ but what
is displacement?

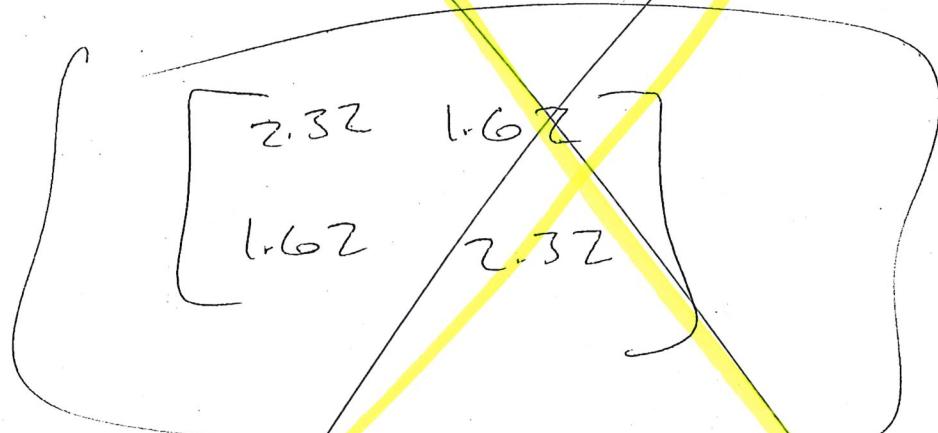
u for
this??

next page

3) Just looking at hoop stress:
 $\mu(r)$ isn't

~~Given~~ elastic expansion,

$$\epsilon = \frac{\delta}{E} \Rightarrow$$



3. d)

* Resubmit *

This problem was just wrong. I didn't really understand this section and I'm still not sure I do. Nevertheless, the

formulae for isotropic materials are on lecture 12, slide 19 + somehow they result in a diagonal matrix.

Pretty sure this implies there are no

σ_{rz} , σ_{xz} , σ_{xy} etc interactions. -?

Anyhow:

$$\epsilon_{rr} = \frac{1}{E} (\sigma_{rr} - \nu(\sigma_{xx} + \sigma_{zz})) = \frac{1}{70GPa} (-3MPa - .41(28+28)) \\ = \boxed{-5.35E-4}$$

$$\epsilon_{xx} = \frac{1}{E} (\sigma_{xx} - \nu(\sigma_{rr} + \sigma_{zz})) = \frac{1}{70E3} (56 - .41(-3+28)) \\ = \boxed{6.84E-4}$$

$$\epsilon_{zz} = \frac{1}{E} (\sigma_{zz} - \nu(\sigma_{rr} + \sigma_{xx})) = \frac{1}{70E3} (28 - .41(56+28))$$

$$\Rightarrow E = \begin{bmatrix} -53 & 0 & 0 \\ 0 & 9 & 0 \\ 0 & 0 & 65 \end{bmatrix} \times 10^{-5} = \boxed{8.96E-5}$$