22/30 Question 1:

U₃Si₅ is a uranium silicide fuel being considered for use in light water reactors. It has a thermal conductivity of 12.5 W/(m K) and a density of Uranium metal of 7.5 g of U/cm³. Answer the following questions

a) What is the fissile isotope in U₃Si₅? What would be the enrichment of this isotope in the natural (unenriched) form of the fuel? (7 points)

b) What enrichment would be required for U₃Si₅ to have the same energ release rate of U₃Si₂ enriched to 3% with a neutron flux of 3.2e13 n/(cm² s)? You can assume that U_{235} has a negligible impact on the total molar mass of U in the fuel (15 points)

 $Q = E_{f} N_{f} O_{f} Y_{fh}$ $O_{f} = S.S \times 10^{-12} (cm^{2})$ $V_{rh} = 3.2 \times 10^{13} \text{ (m}^{1})$ You didn't use density of U in U3Si5
Just use ratio of two U densities

My 5: 2 764. 736/mo) -8 $\frac{N_{\nu_{235}}}{Q_{35}} \approx \frac{(3)(0.03)(N_{9})(12.2)}{767} = 8.59 \times 10^{20} \left(atom \frac{\nu_{-23}}{con^{3}}\right)$

mon of U_3S_{is} = $\frac{N_{U_3S_5}(854)}{(12.2)(N_0)(3)}$ = 0.332 827 \approx 33.2% enrichment required.

c) How would you rank U₃Si₅ as a potential fuel compared to U₃Si₂? Why? (8

It would be not good. Kysis = 0.125 (m) and tysis = 0.23 (m) $\epsilon_{nr} = 3\%$

It conducts beat warse and you need higher enrichment to get the same results. Bad idea.

4. Oleo cannot enrich that high in US.

Consider a fuel rod with a pellet radius of 4.5 mm, an 80 micron gap, and a zircaloy cladding thickness of 0.6 mm. It is experiencing a linear heat rate of 250 W/cm with a coolant temperature of 580 K. The gap is filled with He and 5% Xe and the coolant conductance is 2.5 W/(cm² K).

a) What is the surface temperature of the fuel rod? (15 points)

$$V = 0.45 (m)$$

$$f_{g} = 80 \times 10^{-6} m = 0.008 (cm)$$

$$f_{c} = 0.06 (m)$$

$$LAR = 250 (m)$$

$$T_{coul} = 560 (t)$$

$$h_{coul} = 2.5 (m^{2}t)$$
Show work!

E = 2.467 x 10"Po b) Assume the pellet is made from Uranium Nitride. What is the maximum stress experienced by the pellet, given that uranium nitride has $E = 246.7 E = 246.7 \times 10^{1/N}$ GPa, v = 0.25, and $\alpha = 7.5e-6 1/K$? (10 points)

$$\mathcal{O}_{MAY} = \mathcal{O}_{\theta} \theta$$

$$\mathcal{O}_{\theta\theta} = -\mathcal{O}^{*}(l-3\eta^{2}) ; \text{ where } \mathcal{O}^{*} = \frac{\Delta E}{4(l-r)} ; \mathcal{N} = \frac{R_{f}}{R_{f}}$$

$$\mathcal{O}_{\theta\theta} = -\mathcal{O}^{*}(l-3\eta^{2}) ; \mathcal{N} = \frac{R_{f}}{R_{f}} = \frac{(7.5 \times 10^{-6} L)(2.467 \times 10^{2} N)(71.4718 H)}{4(l-0.25)}$$

$$\mathcal{O}_{\theta\theta} = -\mathcal{O}^{*}(l-3) = -\mathcal{O}^{*}(l-3) = -\mathcal{O}^{*}(l-3)$$

$$\mathcal{O}_{\theta\theta} = \mathcal{O}_{MAY} = l2, 269.846 (V)$$
-1, Use standard units (Pa)

Would you expect this stress to be higher or lower if the pellet was UO₂?

c) Would you expect this stress to be higher or lower if the pellet was UO₂?

Why? (5 points)

The pellet was
$$UO_2$$
?

Why? (5 points)

The points of the points from the pellet was UO_2 ?

Why? (5 points)

The pellet was UO_2 ?

d) What assumptions were made in your calculations for a) and b)? (5 points)

Part B arruntions, - no other strenes except for shomal (no prenume) - loop stress is maximum
- no fracture or relaxation of material (which
would relieve showal stress).

Question 3:

Consider the stress state in a zircaloy fuel rod pressurized to 6 MPa with an average radius of 5.6 mm and a cladding thickness of 0.6 mm.

a) What assumptions are made in the thin walled cylinder approximation for the stress state? (5 points)

Assume ' - small strains

- isotropio material

- "very shin walls" means that stress is constant shough

wall

b) Calculate all three components of the stress using the thin walled cylinder approximation. (10 points)

 $\overline{O_{\theta}} = \frac{\rho B}{5} ; \overline{O_{\overline{z}}} = \frac{\rho B}{25} \overline{O_{r}} = -\frac{1}{2}\rho$ $\overline{O_{\theta}} = \frac{\rho B}{5} ; \overline{O_{\overline{z}}} = \frac{\rho B}{25} \overline{O_{r}} = -\frac{1}{2}\rho$ $\overline{O_{\theta}} = \frac{1}{5} (\sqrt{6} (P_{q})) \overline{O_{\overline{z}}} = \frac{1}{2} (\sqrt{6} (P_{q})) \overline{O_{r}} = -\frac{1}{2} O(MP_{q})$ $\overline{O_{\theta}} = \frac{1}{5} (\sqrt{6} (P_{q})) \overline{O_{\overline{z}}} = \frac{1}{2} (\sqrt{6} (MP_{q})) \overline{O_{r}} = -\frac{1}{2} O(MP_{q})$ $\overline{O_{\theta}} = \frac{1}{5} (\sqrt{6} (P_{q})) \overline{O_{\overline{z}}} = \frac{1}{2} (\sqrt{6} (MP_{q})) \overline{O_{r}} = -\frac{1}{2} O(MP_{q})$

c) Quantify how accurate the thin walled cylinder approximation is for the cladding. Would the thin walled cylinder approximation be conservative if used to estimate if the cladding would fail? (10 points)

 $R_{o} \approx 0.0059 \, (m)$ $R_{o} \approx 0.0059 \, (m)$ $\frac{\left|\frac{R_{o}}{r}\right|^{2} + 1}{\left|\frac{R_{o}}{\rho_{o}}\right|^{2} - 1}$ where at multiple r

-4, Evaluate at multiple r positions to check

JAC = 56.16071429 (MPg) approximation TGE 256,2 (MPg) (No) not conservation, but very close ... off by 0.29 %

d) Write the stress and strain tensors for the stress state in the thin walled cylinder, with E = 70 GPa and v = 0.41. (10 points) E = 70,000 (mP_a)

Err = E Orr - V (OOG + OZZ) -4, No tensors En = -5.348 571 x10-4 (m) TENSOR EQUATION EBE = E[OFF - V (ON + OZZ)] (Tr) = (1+v)(1-2v) | V | V | \{ \xi_2 \} \\ \(\tau \) | \(\tau \) | \(\xi_2 \) | \($\begin{cases} \{_{\theta\theta} = 6.535 \ 714 \ \times 10^{-4} \ (m_{m}) \} \\ \{_{tz} = \frac{1}{E} \left[O_{tz} - v \left(\sigma_{Ge} + \sigma_{m} \right) \right] \end{cases}$