

Exam 2 NE 591

1/ True Stress Strain +
Stress is measured according to the
current area of applied force
and same for the strain so the
initial area A_0 and measurement of elongation —
change through time meanwhile, in
engineering Stress Strain the area
is constant as it's the initial area
prior to any applied force same for
the strain the elongation it's the
same prior to any load and

$$\sigma_{\text{engineering}} = \frac{F}{A_0} \quad ; \quad \epsilon_{\text{true}} = \frac{\Delta L}{L_0}$$

$$E_{\text{engineering}} = \frac{\Delta L}{L_0} \quad E_{\text{true}} = \ln\left(\frac{L}{L_0}\right)$$

2) elastic deformation The material deforms and after removing the applied force it returns to its original status

plastic Deformation. is a permanent Deformation so the material won't be able to return to its original shape after Deformation

3) 0 D: Vacancies

3 D: Voids

4) - melting temperature

- thermal conductivity

- Grain Growth

5) Grain size effect on the S.G.

of the grains we have effect
~~not~~ on chemical properties

If we have bigger grains we have more density of dislocation inside the grain. So more ductile material.

Smaller grains means less density of defect,

which means harder material and all

of that depends on the processing etc.

6.1 Strain hardening after plastic deformation

If we unload the sample it still has

the permanent strain, so now a

material becomes harder but less

ductile what causes strain hardening

is the evolution of the microstructure

as our ~~one~~ grain will become thinner

after strain so more plastic

so it's harder but less ductile

7) all fuel performance codes must
be able to predict:

The mechanical interaction

- between fuel and cladding
- the temperature profile
- the stress and volumetric change

8) for fuel densification is burnup
"fission"

9) Temperature ~~reduction~~ can
affect grain growth more
temperature more grain
grain faster grain growth

10]

a) Thin-walled

$$\bar{\sigma}_0 = \frac{pR}{\delta}$$

$$\bar{\sigma}_3 = \frac{pR}{2\delta}$$

$$\bar{\sigma}_r = -\frac{1}{2} p$$

$$\bar{\sigma}_0 = 135 \text{ MPa}$$

$$\bar{\sigma}_3 = \frac{\bar{\sigma}_0}{2} = 67.5 \text{ MPa}$$

$$\bar{\sigma}_r = -\frac{1}{2} p = -10 \text{ MPa}$$

b) Thick-walled assumption

$$\sigma_{rr}(r) = -p \frac{((R_o/r)^2 - 1)}{((R_o/R_i)^2 - 1)}$$

$$\sigma_{00} = p \frac{(R_o/r)^2 + 1}{(R_o/R_i)^2 - 1}$$

$$\sigma_{33} = p \frac{1}{(R_o/R)^2 - 1}$$

at the midpoint

$$\Rightarrow r = 5,4 + \frac{0,8}{2} = 5,8 \text{ m}$$

~~$$6_{rr} = -20 \left(\frac{6,2}{5,8} \right)^2 - 1$$~~

$$6_{rr} = -20 \frac{\left(\frac{6,2}{5,8} \right)^2 - 1}{\left(\frac{6,2}{5,8} \right)^2 - 1}$$

$$6_{rr} = -6,77 \text{ MPa}$$

$$6_{\theta\theta} = 20 \frac{\left(\frac{6,2}{5,8} \right)^2 + 1}{\left(\frac{6,2}{5,8} \right)^2 - 1}$$

$$6_{\theta\theta} = 106,64 \text{ MPa}$$

$$6_{zz} = 20 \frac{\cancel{\left(\frac{6,2}{5,8} \right)^2 - 1}}{\cancel{\left(\frac{6,2}{5,8} \right)^2 - 1}}$$

$$6_{zz} = 140,96 \text{ MPa}$$

CJ :

$$G_{rr} = \overline{G_{rr}}$$

$$G_{33} = \overline{G_{33}}$$

$$\cancel{G_{00}} = \overline{G_{00}}$$

11]

$$\sigma_{00}(\eta) = -\sigma^*(1-3\eta^2)$$

$$\sigma^* = \alpha \frac{F(T_0 - T_S)}{4(1-\gamma)}$$

~~$$\sigma^*$$~~
$$T_0 - T_S = \frac{LHR}{4\pi K}$$

$$T_0 - T_S = \frac{250}{4\pi \times 0,1} \quad T_0 - T_S = 198,94 \text{ K}$$

$$\sigma^* = \alpha \frac{8,8 \times 10^{-6} \times 290 \times 198,94}{4(1-0,5)}$$

$$\sigma = 168,95 \text{ MPa}$$

The maximum stress is when $\gamma = 1$

So

$$\sigma_{00} = -168,95 \times (-2) \\ = 337,91 \text{ MPa}$$

191

$$\Delta \delta_{S^{\circ}P} = R_C \times C (T_C - T_{f,a}) \\ - R_f \text{ad} f (T_B - T_{f,a})$$

~~T_C?~~ ~~T_F?~~ + T_C? T_F?

~~$T_C - T_{f,a} = LHR$~~

~~$T_B - T_{f,a} = LHR$~~

$$T_{Fuel} - T_{S^{\circ}P} = \frac{LHR}{2\pi R_f h_{g^{\circ}P}}$$

$$T_B - T_{S^{\circ}P} = \frac{32F}{2\pi \times 0,5 \times \frac{0,01}{0,02}}$$

$$= 51,72$$

$$T_{fuel} = 51,72 + 6,50$$

$$= 58,22$$

$$T_{gap} - T_{cold} = \frac{LHR + \text{load}}{2\pi R_f K_{load}}$$

=

$$\cancel{T_{gap} - T_{cold}} = T_{ic}$$

$$\text{So } S_{gap} = \bar{R}_c \times c (T_c - T) \\ - \bar{R}_f \times b (T_f - T_{ext})$$

$$S_{gap} = R_f(0,6 + 0,02) \{ \times 4,5 \times 10^{-6} \\ \times (150 - 300) - 0,1 \times 10 \times 10^{-8} \\ \times (501,72 - 300) \}$$

$$S_{gap} = 0,0011$$

The new temperature

$$T_f - T_{gap} = \frac{LHR}{2\pi R_f h_g} = \frac{325}{2\pi \times 0,001,001,001} \\ = 1,8,88$$

$$\text{so } T_{\text{fuel}} = 498,88 \text{ K}$$

-6
10²⁸

5
 $\frac{105}{0,01} \frac{0,01}{0,02-0,01}$

1311

$$G_{\text{oo}}(\eta) = -G(1 - 3\eta^2)$$

$$G = \frac{\alpha E(T_0 - T_S)}{4(1-\nu)}$$

$$T_0 - T_S = \frac{LHR}{4\pi \times K_B} = \frac{200}{4\pi \times 0,05}$$

$$T_0 - T_S = 636,61 \text{ K}$$

~~$$G = 10,1 \times 10^{-6} \times 210 \times 10^3$$~~

$$G = \frac{10,1 \times 10^{-6} \times 210 \times 10^3 \times 636,61}{4 \times 0,75}$$

$$G = 1,67,52 \text{ MPa}$$

$$-6F_r/10^9 = 1 - 3n^2$$

$$3n^2 = 1 + 6F_r/10^9$$

$$\bar{n} = \sqrt{\frac{1 + 6F_r/10^9}{3}}$$

$$m = \sqrt{\frac{1 + 120/462,57}{3}}$$

$$n = 0,6472$$