

NE 591 Problem Session

1) Max stress in a fuel pellet due to thermal exp.

$$\Delta T = T_0 - T_S = 428 \text{ K} \quad \alpha_F = 12 \times 10^{-6} \text{ /K}$$

$$R_F: 0.5 \text{ cm} \quad E_F: 180 \text{ GPa} \quad \nu = 0.28$$

$$\sigma_r = -\sigma^* (1 - \eta^*)$$

$$\sigma^* = \frac{\alpha E (T_0 - T_S)}{4(1-\nu)}$$

$$\sigma_\theta = -\sigma^* (1 - 3\eta^2)$$

$$\eta = \frac{r}{R_F}$$

$$\sigma_z = -2\sigma^* (1 - 2\eta^2)$$

m^{-x}

$$\sigma_0$$

$$J^* = \frac{(12 \times 10^{-6})(180 \times 10^3)(425)}{4(1 - 0.28)} = 159.4 \text{ m}^2$$

$$\sigma_0 (m^{-x}) \rightarrow r = R_p \rightarrow n = 1$$

$$\sigma_0 = -159.4 (1 - 3(1)^2) = \underline{\underline{319 \text{ m}^2}}$$

2) Zircaloy cladding w/ $r_o = 0.5 \text{ cm}$ $r_i = 0.6 \text{ cm}$
 $\rho = 25 \text{ MPa}$

Is stresses in thin-walled assumption?



$$r_{avg} = 0.55 \text{ cm}$$

$$\sigma_\theta = \frac{\rho R}{\delta}$$

$$\sigma_r = -\frac{\rho}{2}$$

$$\sigma_z = \frac{\rho R}{2\delta}$$

$$\sigma_\theta = \underbrace{\frac{(\rho)(R)}{\delta}}_{0.1} = 137.5 \text{ MPa}$$

$$\sigma_z = \frac{\sigma_\theta}{2} = \underbrace{68.75 \text{ MPa}}$$

$$\sigma_r = -\frac{(\rho)}{2} = \underbrace{-12.5 \text{ MPa}}$$

→ thick-walled assumption

→ stress states @ inner wall?

$$\sigma_r = -p \frac{\left(\frac{R_o}{r}\right)^2 - 1}{\left(\frac{R_o}{R_i}\right)^2 - 1}$$

$$\sigma_\theta = p \frac{\left(\frac{R_o}{r}\right)^2 + 1}{\left(\frac{R_o}{R_i}\right)^2 - 1}$$

$$\sigma_z = \cancel{p} \frac{\left(\frac{R_o}{R_i}\right)^2 - 1}{\left(\frac{R_o}{R_i}\right)^2 - 1}$$

$$\frac{R_o}{R_i} = \frac{0.6}{0.5} = 1.2 \quad r = R_i$$

$$\sigma_r = -(\alpha \delta) \underbrace{\frac{1.2^2 - 1}{1.2^2 - 1}}_{= -25 \text{ MPa}} = -25 \text{ MPa}$$

$$\sigma_0 = 25 \frac{1.2^2 + 1}{1.2^2 - 1} = 138.4 \text{ MPa}$$

$$\sigma_2 = \frac{25}{1.2^2 - 1} \rightarrow 56.8 \text{ MPa}$$

thick vs thin

$$\sigma_0(R_i) = \sigma_0(\text{thin}) \quad \sigma_r(R_i) = 2\sigma_r(\text{thin})$$

$$\sigma_2(R_i) \approx 1.2 \sigma_2(\text{thin})$$

3) Gap thickness change

$$\alpha_F = 12 \times 10^{-6} \frac{1}{K} \quad \alpha_c = 5 \times 10^{-6} \frac{1}{K} \quad T_F^{avg} = 925 K$$

$$T_0(\text{fuel}) = T_0(\text{clad}) = 300 K$$

$$T_c^{avg} = 550 K$$

$$R_F = 0.5 \text{ cm} \quad R_c^{avg} = 0.58 \text{ cm} \quad t_c = 0.1 \text{ cm}$$

$$t_g = 6.03 \text{ cm}$$

$$\Delta t_{gap} = R_c \alpha_c (T_c^{avg} - T_0^{clad}) = R_F T_F^{avg} / (T_F^{avg} - T_0)$$

$$\Delta t_c = (0.58) (5 \times 10^{-6}) (550 - 300) = 7.25 \times 10^{-4}$$

$$\Delta t_f = (0.8) \left(1.2 \times 10^{-6} \right) (925 - 300) = -3.75 \times 10^{-3}$$

$$t_{\gamma^{\mu}} = 0.03 \text{ cm} + 7.25 \times 10^{-4} - 3.75 \times 10^{-3}$$
$$= 0.0297 \text{ cm}$$



$$T_s^{av\gamma} \rightarrow \Delta t_\gamma \rightarrow \Delta T_s^{av\gamma}$$

4) Cladding tube w/ no internal pressure but w/ thermal expansion

$$\Rightarrow \frac{r}{R_c} = 1.1, \sigma_\theta, \sigma_r, \sigma_z?$$

$$R_c = 0.6 \text{ cm} \quad t_{cl} = 0.1 \text{ cm} \quad E = 250 \text{ GPa} \quad v = 0.3$$

$$\alpha_c = 15 \times 10^{-4} \text{ } ^\circ\text{K} \quad T_c^{\text{avg}} = 600 \text{ K} \quad T_c^0 = 300 \text{ K}$$

$$\sigma_r = \frac{\partial T}{\partial} \frac{\alpha E}{1-v} \left(\frac{r}{R_c} - 1 \right) \left(1 - \frac{R_c}{t_{cl}} \left(\frac{r}{R_c} - 1 \right) \right)$$

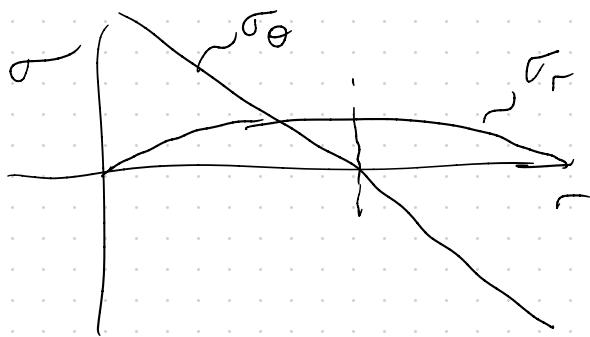
$$\sigma_0 = \frac{\partial T}{\partial} \frac{\alpha E}{1-v} \left(1 - \frac{R_c}{t_{cl}} \left(\frac{r}{R_c} - 1 \right) \right) = \sigma_z$$

$$\sigma_r = \frac{300}{2} \frac{(15 \times 10^{-6})(250 \times 10^3)}{1 - 0.3} (1.1 - 1) \left(1 - \frac{0.6}{0.1} (1.1 - 1) \right)$$

$$\sigma_r = 64 \text{ MPa}$$

$$\sigma_\theta = \frac{300}{2} \frac{(15 \times 10^{-6})(250 \times 10^3)}{1 - 0.3} \left(1 - 2 \frac{0.6}{0.1} (1.1 - 1) \right)$$

$$\sigma_\theta = \sigma_z = -161 \text{ MPa}$$



5) Densification ^{strain?} @ $B = 2 \text{ mW/kg u}$ $T = 900 \text{ K}$

max densification = 2% $B_0 = 0.005 \text{ FIMA}$

$$\epsilon_d = \Delta \rho_0 \left(e^{\frac{B_0 \cdot 0.01}{C_0 B_0}} - 1 \right)$$

$$B \text{ convert to FIMA} \rightarrow \frac{2 \text{ mW/kg u}}{950} \rightarrow 0.002 \text{ FIMA}$$

$$C_0 \Rightarrow T < 750 \text{ C} \rightarrow 7.235 - 0.0086 \left(\frac{T}{\text{C}} - 25 \right)$$

$$C_0 = 2.058$$

$$\epsilon_0 = 0.02 \left(e^{\frac{(0.002)(\ln 0.01)}{6.058}(0.005)} - 1 \right)$$

$$\epsilon_y = -0.012$$

$$\frac{0.012}{0.02} \rightarrow \underline{602}$$

6) Given only strains, calculate each stress ...

$$\epsilon_{rr} = \frac{1}{E} (\sigma_r - \nu(\sigma_0 + \sigma_z))$$

$$\epsilon_\theta = \frac{1}{E} (\sigma_\theta - \nu(\sigma_r + \sigma_z))$$

$$\epsilon_z = \frac{1}{E} (\sigma_z - \nu(\sigma_r + \sigma_\theta))$$

$$\epsilon_E = \sigma_z - \nu(\sigma_r + \sigma_\theta)$$

$$\sigma_z = \epsilon_z E + \nu \sigma_r + \nu \sigma_\theta$$

$$\epsilon_0 E = \sigma_0 - v\sigma_r - v\sigma_z$$

$$\epsilon_0 E = \sigma_0 - v\sigma_r - v(\epsilon_2 E + v\sigma_r + v\sigma_0)$$

$$\epsilon_0 E = \left(\sigma_0 \right) - v\sigma_r - v\epsilon_2 E - v^2 \sigma_r (-v^2 \sigma_0)$$

$$\sigma_0 = \frac{v^2 \sigma_r + v\sigma_r + vE \epsilon_2 + \epsilon_0 E}{1 - v^2}$$

$$E\sigma_r = \sigma_r - v\sigma_0 - v\sigma_z \quad \text{can continue...}$$