

- Find max stress in fuel pellet

$$\text{LHR} = 250 \frac{\text{W}}{\text{cm}} \quad E = 250 \text{ GPa} \quad \nu = 0.25$$

$$\alpha_F = 7.5 \times 10^{-6} \text{ K}^{-1} \quad K_F = 0.2 \frac{\text{W}}{\text{cm} \cdot \text{K}}$$

$$\sigma_\theta = \sigma_{\max} \quad \sigma_\theta(r = R_F) = \sigma_{\max, \theta}$$

$$\sigma_\theta = -\sigma^* (1 - 3\eta^2) \quad \eta = \frac{r}{R_F} \quad \sigma^* = \frac{\alpha E (T_o - T_s)}{4(1-\nu)}$$

$$T_o - T_s = \Delta T_F = \frac{\text{LHR}}{4\pi K_F} = \frac{250}{4\pi(0.2)} = 99.5 \text{ K}$$

$$\sigma^* = \frac{(7.5 \times 10^{-6})(250 \times 10^3 \text{ MPa})(99.5)}{4(1-0.25)} = 62.2 \text{ MPa}$$

$$\eta = \frac{r}{R_F} \rightarrow \eta = \frac{R_F}{R_F} = 1$$

$$\sigma_\theta = -\sigma^* (1 - 3\eta^2) \rightarrow -62.2(1 - 3(1)^2) = \boxed{124.4 \text{ MPa}}$$

$$\sigma_r = -\sigma^* (1 - \eta)$$

$$\sigma_\theta = -\sigma^* (1 - 3\eta^2)$$

$$\sigma_z = -2\sigma^* (1 - 2\eta^2)$$

→ min stress  $\eta = \frac{r}{R_F}$

$$\sigma_z (r=0)$$

$$\begin{aligned} r &= 0 \\ \eta &= 0 \end{aligned}$$

$$\sigma^* = 62.2 \text{ MPa}$$

$$\sigma_z = -2 \times 62.2 (1 - 2(0)^2)$$

$$\sigma_z = -124.4 \text{ MPa}$$

Z: alloy cladding w/  $P = 6 \text{ MPa}$ ,  $R_i = 5 \text{ mm}$ ,  $R_o = 5.8 \text{ mm}$   
 $E = 250 \text{ GPa}$

→ find stress assuming thin walled approx

$$\sigma_\theta = \frac{pR}{\delta} \quad \sigma_r = -\frac{p}{2} \quad \sigma_z = \frac{pR}{2\delta} \quad \begin{matrix} \delta = \text{cladding} \\ \text{thickness} \end{matrix}$$

$$\sigma_\theta = \frac{6(5.4)}{0.8} = 40.5 \text{ MPa} \quad R = \bar{R} = \frac{R_o + R_i}{2}$$

$$\sigma_r = -\frac{6}{2} = -3 \text{ MPa} \quad \sigma_z = \frac{6(5.4)}{2(0.8)} = 20.3 \text{ MPa}$$

→ thick-walled, stress across cladding →  $\sigma_\theta$

$$\sigma_\theta = p \frac{\left(\frac{R_o}{r}\right)^2 + 1}{\left(\frac{R_o}{R_i}\right)^2 - 1} \quad \begin{matrix} \sigma_\theta^i \rightarrow r = R_i \\ \sigma_\theta^o \rightarrow r = R_o \end{matrix} \quad \begin{matrix} \sigma_\theta = \sigma_{\max} \\ \frac{R_o}{R_i} = 1.16 \end{matrix}$$

$$\sigma_\theta^i = 6 \frac{(1.16^2) + 1}{1.16^2 - 1} = 40.7 \text{ MPa}$$

$$\Delta\sigma_\theta = 6 \text{ MPa}$$

$$\sigma_\theta^o = 6 \frac{(1^2) + 1}{1.16^2 - 1} = 34.7 \text{ MPa}$$

$$\frac{\Delta\sigma_\theta}{\sigma_{\max}} = 15\%$$

- assuming thin wall, strain states?  $E = 250 \text{ GPa}$

$$\epsilon_{rr} = \frac{1}{E} (\sigma_{rr} - \nu(\sigma_{\theta\theta} + \sigma_{zz})) \rightarrow \frac{1}{250 \times 10^3} (-3 - 0.25(40.5 + 20.3))$$

$$\epsilon_{\theta\theta} = \frac{1}{E} (\sigma_{\theta\theta} - \nu(\sigma_{rr} + \sigma_{zz})) \rightarrow \frac{1}{250 \times 10^3} (40.5 - 0.25(-3 + 20.3))$$

$$\epsilon_{zz} = \frac{1}{E} (\sigma_{zz} - \nu(\sigma_{rr} + \sigma_{\theta\theta})) \rightarrow \frac{1}{250 \times 10^3} (20.3 - 0.25(40.5 - 3))$$

$$\epsilon_{rr} = -7.28 \times 10^{-5} \quad \epsilon_{\theta\theta} = 1.45 \times 10^{-4} \quad \epsilon_{zz} = 4.37 \times 10^{-5}$$

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- find concentration of interstitials @ 1100K

$$S_f^i = 2K_B \quad E_f^i = 3\text{eV}$$

$$C_i = \exp\left(\frac{S_f}{K_B}\right) \exp\left(-\frac{E_f}{K_B T}\right)$$

$$C_i = \exp\left(\frac{2K_B}{K_B}\right) \exp\left(-\frac{3}{(8.617 \times 10^{-5})(1100)}\right)$$

$$C_i = 1.33 \times 10^{-13}$$

→ @ what pressure will a thick-walled cylinder w/  
 $R_o = 0.55 \text{ cm}$   $R_i = 0.5 \text{ cm}$   $\sigma_y = 400 \text{ MPa}$   
 exceed its yield stress?

$$\sigma_{\theta} = p \frac{\left(\frac{R_o}{r}\right)^2 + 1}{\left(\frac{R_o}{R_i}\right)^2 - 1}$$

$$\boxed{p = 38 \text{ MPa}}$$

$$\sigma_{\theta} = \max @ r = R_i$$

$$\sigma_y = \sigma_{\theta} = p \frac{\left(\frac{0.55}{0.5}\right)^2 + 1}{\left(\frac{0.55}{0.5}\right)^2 - 1} = 400$$

→ how far do cracks extend into the fuel?

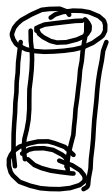
$R_F = 0.55 \text{ cm}$      $\nu = 0.25$      $E = 200 \text{ GPa}$   
 $\text{CHR} = 200 \text{ W/cm}$   
 $\alpha_F = 11 \times 10^{-6} \text{ K}^{-1}$      $\sigma_{fr} = 120 \text{ MPa}$      $\eta = \frac{r}{R_F}$   
 $K_F = 0.05 \text{ W/cm}^2\text{K}$      $\sigma^* = \frac{\alpha E}{4(1-\nu)} \Delta T$

$\sigma_\theta = \sigma^* (1 - 3\eta^2)$   
 $\sigma^* = \frac{(11 \times 10^{-6})(200 \times 10^3)(318)}{4(1 - 0.25)}$   
 $\sigma^* = 233 \text{ MPa}$

$\Delta T_F = \frac{\text{CHR}}{4\pi K_F}$   
 $\Delta T_F = \frac{200}{4\pi(0.05)} = 318 \text{ K}$

$$\sigma_\theta = \sigma^* (1 - 3\eta^2) = \sigma_{fr} \rightarrow 120 = 233 (1 - 3\eta^2)$$

$$\eta = 0.402 = \frac{r}{R_F} \quad r = 0.22 \text{ cm}$$



cylinder,  $\rho = 0$  & TE  $\Delta T = (\bar{T}_c - T_{ref})$

$$\sigma_r = \frac{\Delta T}{2} \frac{\alpha E}{1-\nu} \left( \frac{r}{R_i} - 1 \right) \left( 1 - \frac{R_i}{r} \left( \frac{r}{R_i} - 1 \right) \right)$$

$$\sigma_\theta = \sigma_z = \frac{\Delta T}{2} \frac{\alpha E}{1-\nu} \left( 1 - \frac{2R_i}{r} \left( \frac{r}{R_i} - 1 \right) \right)$$

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$$\epsilon_{rr} = \frac{1}{E} \left( \sigma_{rr} - \nu (\sigma_{\theta\theta} + \sigma_{zz}) \right) + \alpha \Delta T$$