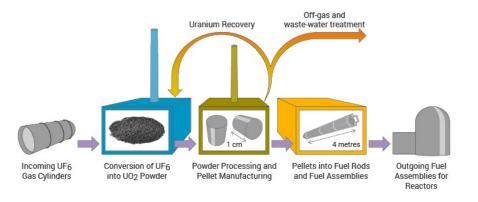
Nuclear Fuel Performance

NE 591

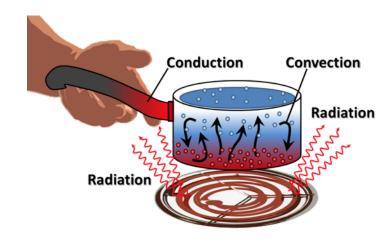
Last Time

- Mining -> Processing -> Conversion -> Enrichment -> Powder -> Compaction/Sintering -> Rod/Assembly
- U₃O₈ must be converted to UF₆ for enrichment, which is then converted to UO₂ powder for pellet manufacture
- For different fuel types, enriched UF₆ follows a different path



Last Time

- General heat transport
- Heat is produced in the fuel, transports through the gap and cladding, and into the coolant
- The geometry of our problem
- The initial condition of T
- The boundary conditions of T
- Is each parameter is a function of T
 - Thermal conductivity, heat capacity
- Function of space/time?
 - Heat generation, dependent upon flux

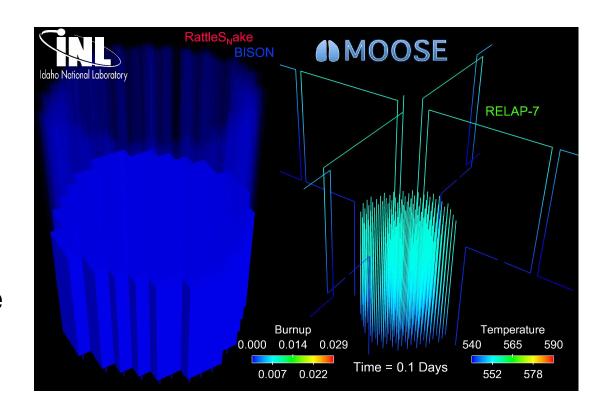


$$\rho c_p \frac{\partial T}{\partial t} - \nabla \cdot (k \nabla T)$$

ANALYTICAL SOLVE OF HEAT CONDUCTION

The most accurate solution is numerical, in 3D, requires modeling the entire core, and is multi-physics

- Solution is 3D and changes in time
- All the properties are functions of temperature
- The boundary conditions comes from information about the coolant flow
- The heat generation rate comes from information about the neutronics in the reactor
- No analytical solution is possible



In order to solve, make assumptions!

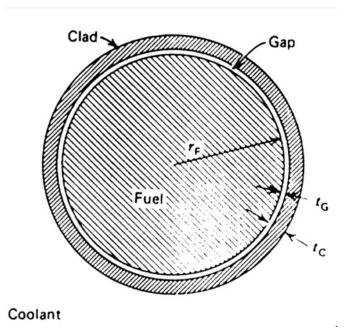
- #1: steady state -> $\nabla \cdot (k\nabla T) + Q = 0$
- #2: axisymmetric ->

$$\frac{1}{r}\frac{\partial}{\partial r}\left(rk(T)\frac{\partial T}{\partial r}\right) + \frac{\partial}{\partial z}\left(k(T)\frac{\partial T}{\partial z}\right) + Q(r,z) = 0$$

- #3: constant in z $\frac{1}{r} \frac{\partial}{\partial r} \left(rk(T) \frac{\partial T}{\partial r} \right) + Q(r) = 0$
- #4: constant thermal conductivity

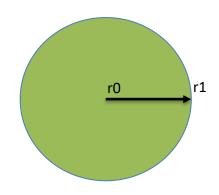
$$\frac{1}{r}\frac{\partial}{\partial r}\left(rk\frac{\partial T}{\partial r}\right) + Q = 0$$

$$\rho c_p \frac{\partial T}{\partial t} = \nabla \cdot (k \nabla T) + Q$$



Directly Solving for Temperature Profile

• Boundary conditions: $r_0 = 0$, $r_1=R$, T'(0) = 0, $T(R) = T_s$



$$\frac{1}{r}\frac{\partial}{\partial r}\left(rk\frac{\partial T}{\partial r}\right) + Q = 0$$

$$\frac{\partial}{\partial r} \left(rk \frac{\partial T}{\partial r} \right) = -Qr$$

$$rk \frac{\partial T}{\partial r} = -\frac{Qr^2}{2} + C_1 \qquad 0 = -\frac{Q0^2}{2} + C_1$$

$$\frac{\partial T}{\partial r} = -\frac{Qr}{2k}$$

$$T(r) = -\frac{Qr^2}{4k} + C_2 \qquad C_2 = \frac{QR^2}{4k} + T_s$$

$$T(r) = -\frac{Qr^2}{4k} + \frac{QR^2}{4k} + T_s \qquad T(r) = \frac{Q(R^2 - r^2)}{4k} + T_s$$

$$T_0 - T_s = \frac{QR^2}{4k}$$

Linear Heat Rate

- $LHR = \pi R^2 Q_{av}$
 - Where Q_{av} is the radially averaged heat generation rate, in units of perlength, i.e. W/cm
- Substitute LHR into previous equation on T0-Ts

$$T_0 - T_s = \frac{QR^2}{4k}$$
 $T_0 - T_s = \frac{R^2}{4k} \frac{LHR}{\pi R^2}$ $T_0 - T_s = \frac{LHR}{4\pi k}$

Alternate Geometries

 Similar derivation with appropriate boundary conditions can be applied to plate and sphere geometries

Plate

$$T(x) - T_s = \frac{LHR}{2\pi k} \left(1 - \frac{x^2}{t_f^2} \right)$$

x is the distance from the midplane of the fuel and tf is the plate fuel thickness

Sphere

$$T(r) - T_s = \frac{LHR}{6\pi k} \left(1 - \frac{r^2}{R_f^2} \right)$$

r is the distance from the sphere center and Rf is the radius of the sphere

Heat transport through the gap

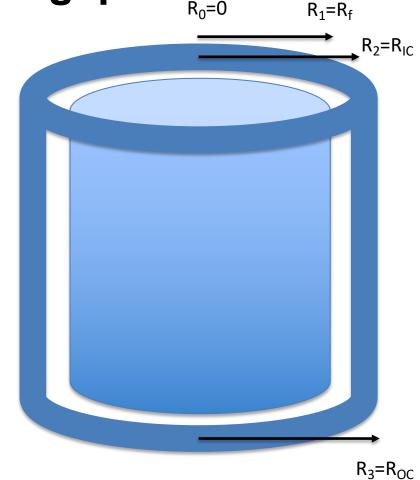
The heat flux gives the rate, per unit area, at which heat flows in a given direction

$$q = -k \frac{T_2 - T_1}{L} \qquad q_{gap} = -k_{gap} \frac{T_{IC} - T_{fuel}}{R_{IC} - R_{fuel}}$$

The heat flux from the fuel is the LHR/pellet circumference

$$q = \frac{LHR}{2-R}$$

- Heat flux from the fuel is the same as heat flux through the the gap – can make this assumption because $R_f >> t_a$
- Gap thickness = R_{IC} - R_f = t_a
- Cladding thickness = R_{OC} - R_{IC} = t_c



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Heat transport through the gap

Set heat flux fuel/gap equal

$$\frac{LHR}{2\pi R_f} = -k_{gap} \frac{T_{IC} - T_{fuel}}{t_{gap}} \qquad \frac{LHR}{2\pi R_f} = k_{gap} \frac{T_{fuel} - T_{IC}}{t_{gap}}$$

Gap conductance is defined as:

$$h_g = \frac{k_{gap}}{t_g}$$

$$h_g = \frac{k_{gap}}{t_g} \qquad T_{fuel} - T_{IC} = \frac{LHR}{2\pi R_f h_g}$$

- Gap conductance depends on the gas filling the gap
 - For pure He, $k_{qap}=16x10^{-6} * T^{0.79}$ (W/cm-K)
 - For pure Xe, $k_{qap} = 0.7x10^{-6} * T^{0.79}$ (W/cm-K)
 - Simple mixing rule: k_{gap} = k_{He}(1-y) + k_{Xe}y
 Where y is the mole/atom fraction of Xe

Heat transport through the cladding

Heat flux through the cladding

$$q = -k \frac{T_2 - T_1}{L} \qquad q_{clad} = -k_{clad} \frac{T_{CO} - T_{CI}}{R_{CO} - R_{CI}}$$

$$q = \frac{LHR}{2\pi R_f} \qquad q_{clad} = k_{clad} \frac{T_{CI} - T_{CO}}{t_{clad}}$$

Set equal your heat fluxes

$$\frac{LHR}{2\pi R_f} = k_{clad} \frac{T_{CI} - T_{CO}}{t_{clad}} \qquad T_{CI} - T_{CO} = \frac{LHR}{2\pi R_f} \frac{t_{clad}}{k_{clad}}$$

Heat transfer to the coolant

Heat is transported from the cladding to the coolant via convection

$$T_{CO} - T_{cool} = \frac{LHR}{2\pi R_F h_{cool}}$$

- T_{cool} is the coolant temperature, h_{cool} is the convective heat transfer coefficient between cladding wall and coolant
- Adding gap + cladding + coolant: $\frac{1}{h} = \frac{t_{gap}}{k_{gap}} + \frac{t_{clad}}{k_{clad}} + \frac{1}{h_{cool}}$

Summary of analytical solutions

•
$$T_0 - T_{fuel} = \frac{Q}{4k} R_{fuel}^2$$
 $T_0 - T_{fuel} = \frac{LHR}{4\pi k}$

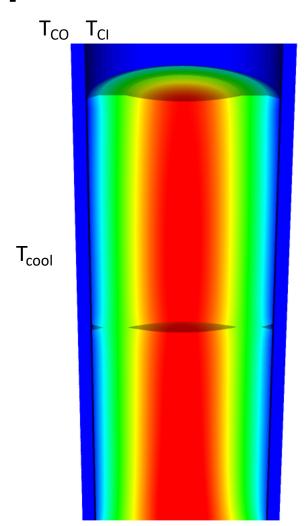
•
$$T_{fuel} - T_{CI} = \frac{Q}{2h_{gap}} R_{fuel}$$
 $T_{fuel} - T_{CI} = \frac{LHR}{2\pi R_{fuel} h_{gap}}$ $h_{gap} = \frac{k_{gap}}{t_{gap}}$

•
$$T_{CI} - T_{CO} = \frac{Qt_{clad}}{2k_{clad}}R_{fuel}$$
 $T_{CI} - T_{CO} = \frac{LHRt_{clad}}{2\pi R_{fuel}k_{clad}}$

•
$$T_{CO} - T_{cool} = \frac{Q}{2h_{cool}} R_{fuel} T_{CO} - T_{cool} = \frac{LHR}{2\pi R_{fuel} h_{cool}}$$

Solving for the temperature profile

- You first solve for the transition temperatures.
- Start from the coolant and work inward
- Have a linear profile everywhere except in the fuel
- Finally, solve for the temperature profile throughout the fuel



Fuel and Cladding Thermal Properties

Material	Density (g/cm ³)	Heat Capacity	Thermal	Thermal
		Cp (J/g-K)	Conductivity k	Expansion
			(W/cm-K)	Coefficient a (K ⁻¹)
UO ₂	10.98	0.33	0.03	1.2 x 10 ⁻⁵
Zr	6.5	0.35	0.17	1.0 x 10 ⁻⁵
Stainless steel	8.0	0.5	0.17	9.6 x 10 ⁻⁶

Example Problem

- $T_{cool} = 580 \text{ K}$; LHR = 200 W/cm; $h_{cool} = 2.65 \text{ W/cm}^2\text{-K}$
- $R_{fuel} = 0.5 \text{ cm}$; $t_{clad} = 0.06 \text{ cm}$; $t_{gap} = 0.003 \text{ cm}$; $k_f = 0.03 \text{ W/cm-K}$
- Work from outside->in, calculate cladding temperature

$$T_{co} = (200)/(2*pi*0.5*2.65) + 580$$

$$T_{co} - T_{cool} = \frac{LHR}{2\pi R_{fuel} h_{cool}}$$

 $T_{co} = 604 \text{ K}$

Calculate inner cladding temp

$$T_{ci} = (200*0.06)/(2*pi*0.5*0.17) + 604$$
 $T_{ci} - T_{clad} = \frac{LHRt_{clad}}{2\pi R_{fuel}k_{clad}}$ $T_{ci} = 626 \text{ K}$

- Calculate fuel surface temperature
- Calculate gap conductance
 - gap with He; k_{gap} =16x10⁻⁶ * T^{0.79} (W/cm-k); assume T_{ci} is appropriate for entire gap; k_{gap} = 0.0026 W/cm-K; t_{gap} = 0.003 cm
 - $h_{gap} = 0.87 \text{ W/cm}^2\text{-K}$

$$T_{\text{fuel}} = 200/(2 \text{*pi*}0.5 \text{*}0.87) + 626$$

$$T_{\text{fuel}} = 699 \text{ K}$$

$$T_{fuel} - T_{ci} = \frac{LHR}{2\pi R_{fuel} h_{gap}}$$

Calculate centerline temperature

$$T_0 = 200/(4*pi*0.03) + 699$$

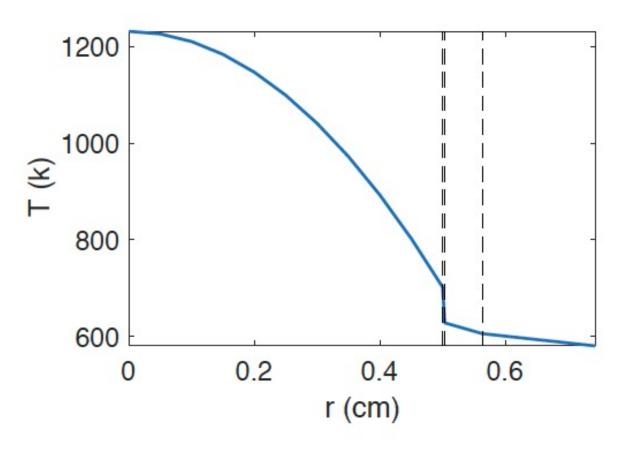
$$T_0 - T_{fuel} = \frac{LHR}{4\pi k}$$

$$T_0 = 1230 \text{ K}$$

Full temperature profile
$$T(r) = \frac{Q(R^2 - r^2)}{4k} + T_s$$

$$T(r) = \frac{LHR(R_f^2 - r^2)}{4\pi k R_f} + T_s$$
 $T(r) = \frac{LHR}{4\pi k} \left(1 - \frac{r^2}{R_f^2}\right) + T_s$

- Parabolic temperature profile in fuel
- Linear in gap and clad
- Steep temperature drop over gap, very low thermal conductivity
- Smaller temperature drop over cladding

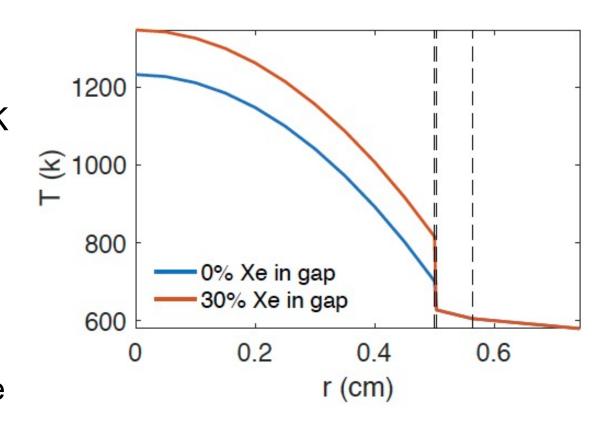


Temperature profile modification

- Instead of pure He gap, 30% of gas is Xe; how is T₀ affected?
- First, how is gap conductance affected?
- Gap conductance depends on the gas filling the gap
 - For pure He, $k_{qap}=16x10^{-6} * T^{0.79}$ (W/cm-K)
 - For pure Xe, $k_{gap}=0.7x10^{-6} * T^{0.79}$ (W/cm-K)
 - Simple mixing rule: $k_{qap} = k_{He}(1-y) + k_{Xe}y$
 - T_{CI} , T_{CO} , and T_{cool} is unchanged from previous example, also T_0 - T_{fuel} is unchanged
 - $-k_{gap} = ((16x10^{-6})*(626)^{0.79})(1-0.3) + ((0.7x10^{-6})*(626)^{0.79})(0.3) = 1.85E-3$ W/cm-K

Temperature profile modification

- $k_{qap} = 1.85E-3 \text{ W/cm-K}$
- $h_{gap} = 1.85E-3 / 0.003 = 0.62 \text{ W/cm}^2-\text{K}$
- $T_{fuel} = 200/(2*pi*0.5*0.62) + 626 = 729 K$
- $T_0 T_{fuel} = 530.5 \text{ K}$ (unchanged from before)
- $T_0 = 729 + 530.5 = 1259.5 \text{ K}$
- Increase in T₀ of 30 K
- Caveat: linear mixing of gases is not the best approach

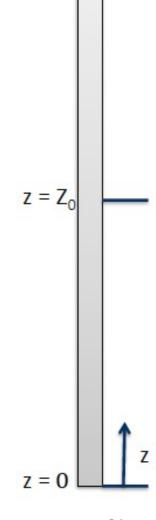


Neutron flux varies axially, so does LHR

Taking a fuel rod with length, L = 2*Z₀

$$LHR\left(\frac{z}{Z_o}\right) = LHR^o \cos\left[\frac{\pi}{2\gamma}\left(\frac{z}{Z_o} - 1\right)\right] = LHR^o F\left(\frac{z}{Z_o}\right)$$

- LHR⁰ is the midpoint linear heat rate, i.e. @ z=Z₀
- $\gamma = \frac{Z_{ex} + Z_0}{Z_0}$, where Z_{ex} is the extrapolation distance
- A typical value is $\gamma = 1.3$



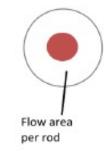
Coolant temperature varies with Z

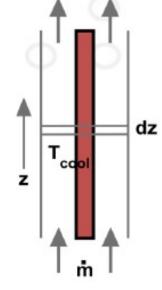
- Each rod has a given flow area
- Mass flow rate: \dot{m}
- Coolant specific heat: C_{PW}

$$\dot{m}C_{PW} \frac{dT_{cool}}{dz} = LHR\left(\frac{z}{Z_o}\right)$$

$$\dot{m}C_{PW}(T_{cool} - T_{cool}^{in}) = Z_o \int_{0}^{z/Z_o} LHR\left(\frac{z}{Z_o}\right) d\left(\frac{z}{Z_o}\right)$$

$$\dot{m}C_{PW}(T_{cool} - T_{cool}^{in}) = Z_o \times LHR^o \int_0^{z/Z_o} F\left(\frac{z}{Z_o}\right) d\left(\frac{z}{Z_o}\right)$$



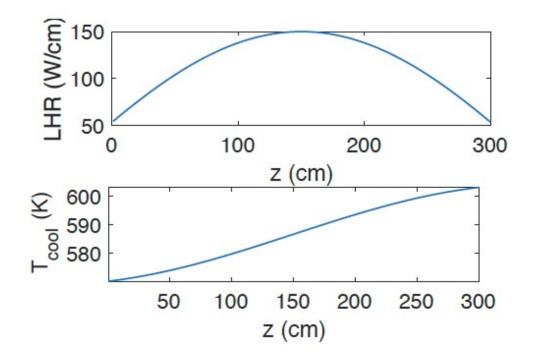


$$T_{cool} - T_{cool}^{in} = \frac{1}{1.2} \frac{Z_o \times LHR^o}{\dot{m}C_{PW}} \left\{ \sin(1.2) + \sin\left[1.2\left(\frac{z}{Z_o} - 1\right)\right] \right\}$$

Calculate LHR and T_{cool} with axial variation

• mdot = 0.25 kg/s-rod; Z_0 = 150 cm; LHR⁰ = 150 W/cm; C_{PW} = 4200 J/kg-K; T_{in} = 570 K

$$T_{cool} - T_{cool}^{in} = \frac{1}{1.2} \frac{Z_o \times LHR^o}{\dot{m}C_{PW}} \left\{ \sin(1.2) + \sin\left[1.2\left(\frac{z}{Z_o} - 1\right)\right] \right\}$$



Summary

- Developed analytical expressions for the temperature profile within a fuel rod
- Required to make four assumptions:
 - Steady-state solution
 - Temperature is axisymmetric
 - T is constant in Z
 - Thermal conductivity is independent of temperature
- Temperature profile in the fuel is parabolic, assume linear profiles in gap, clad and coolant
- Can incorporate axial variation in T_{cool} with axial variation in LHR