



## Fission induced swelling and creep of U–Mo alloy fuel<sup>☆</sup>

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### ABSTRACT

Tapering of U–Mo alloy fuel at the end of plates is attributed to lateral mass transfer by fission induced creep, by which fuel mass is relocated away from the fuel end region where fission product induced fuel swelling is in fact the highest. This mechanism permits U–Mo fuel to achieve high burnup by effectively relieving stresses at the fuel end region, where peak stresses are otherwise expected because peak fission product induced fuel swelling occurs there. ABAQUS FEA was employed to examine whether the observed phenomenon can be simulated using physical–mechanical data available in the literature. The simulation results obtained for several plates with different fuel fabrication and loading scheme showed that the measured data were able to be simulated with a reasonable creep rate coefficient. The obtained creep rate constant lies between values for pure uranium and MOX, and is greater than all other ceramic uranium fuels.

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### 1. Introduction

Two forms of U–Mo alloy fuel have been irradiation-tested in the US GTRI (Global Threat Reduction Initiative) in order to qualify this fuel for the conversion of research and test reactors from highly enriched uranium (HEU) to low enriched uranium (LEU). One form, dispersion fuel, consists of fuel particles dispersed in an aluminum matrix and the other, monolithic fuel, is of solid alloy fuel foil directly bonded to aluminum cladding. The geometry for both fuel forms is a thin plate. Fig. 1 schematically illustrates the cross section of a monolithic mini-fuel plate.

Swelling of U–Mo fuel only increases the plate thickness because the other dimensions are constrained. Consequently, fuel swelling can be estimated with fair accuracy by measuring the fuel plate thickness during post-irradiation analyses. U–Mo fuel swelling kinetics as a function of fission density is previously documented [1]. The U–Mo fuel swelling we are dealing with occurs at irradiation temperatures typically lower than ~250 °C in research and test reactors. In addition, because the fission gas bubbles are small, the fuel swelling part that is affected by temperature is also small.

In the postirradiation examination (PIE) of fuel plates, it is a common observation that the fuel meat or foil swelling close to the plate width end is significantly diminished. This phenomenon is clearly observable in some RERTR plates where one of the plate width ends faces the ATR core, resulting in a large power peaking at the foil end and subsequently a peak in fission density. Because fuel swelling increases with fission density, this is an obvious contradiction accordingly (see Fig. 2).

The most plausible causes for this phenomenon are lateral fuel transport toward the fuel center in the width direction and accumulation of fuel mass at regions away from the foil end. The driving force for the fuel volume transport is the shear stress that builds up because of fission product induced fuel swelling while the Al cladding volume is preserved. Because the fuel foil end is blocked by the cladding, the resulting stress directs the fuel volume transport away from the plate to a lower stress region toward the fuel width center (Fig. 2). Fission induced creep of U–Mo is the underlying mechanism that controls the rate of the fuel volume transport [2].

Although similar observations have been made for dispersion fuel plates, the phenomenon is more difficult to quantify in this type fuel because of its complex microstructure that include variations in fuel volume fraction in the meat, the presence of variable fractions of fuel–Al matrix interdiffusion products, and pore formation in matrix in some cases. Monolithic fuel plates do not have these complications, so this paper focuses on analyzing monolithic fuel plates, leaving dispersion fuel plate analysis for future work.

Finite element analysis (FEA) using a commercial code ABAQUS [3] was employed to examine whether the observed phenomenon is physically realistic by applying materials physical–mechanical

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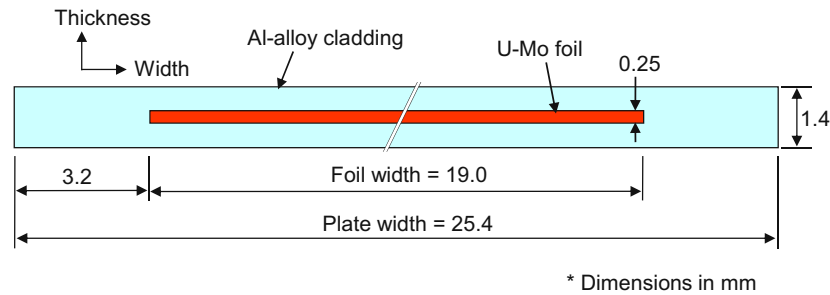


Fig. 1. Schematic of as-fabricated cross section at axial mid-plane of a monolithic miniplate.

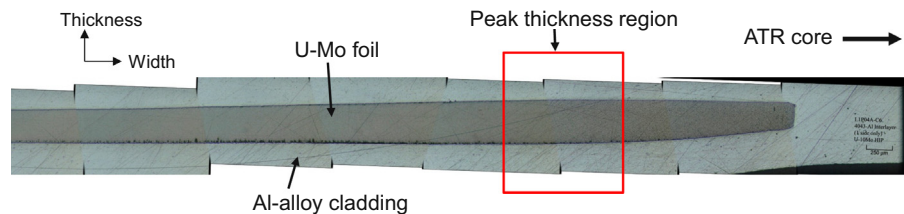


Fig. 2. Fuel plate cross section in the width direction at near the ATR-core side end of L1P04A.

parameters in reasonable ranges. Simultaneously, another objective of the FEA simulation was to obtain a creep rate coefficient that enables the extent of the fuel mass transport observed at the end of life (EOL) of the plates caused by creep.

## 2. Experimental

The plates used for this paper were from irradiation tests RERTR-6, -7, -8, -9A, -9B, -10, and -12. The plate fabrication and irradiation properties are summarized in Table 1. The plate dimensions are 100 mm in length, 25 mm in width, and 1.40 mm in thickness, which are a miniature version, except for the thickness, of full-size fuel plates used in research reactors and test reactors. The fuel foil dimensions are 82.6 mm in length and 19.0 mm in width. Foil thickness is typically 0.25 mm, and in some cases 0.50 mm is used to examine the performance of thinner cladding.

The fuel foil was metallurgically bonded to Al 6061 cladding. Two kinds of bonding method were applied. One was friction stir welding (FSW) and – as the name implies – it achieved a metallurgical bonding between the fuel foil and cladding by welding, during which the cladding was instantaneously melted at the foil–cladding interface. The other method was hot isostatic pressing (HIP), in which an isostatic pressure of  $\sim 103$  MPa was applied for 90 min while the plate was heated at 560–580 °C. The details of the bonding methods can be found, for example, in Ref. [4]. It is worth noting that both bonding methods produce uniform bonding and, more importantly, do not alter foil thickness during plate fabrication.

All the test plates were irradiated in the Advanced Test Reactor (ATR) at INL. The fission rate histories of one representative plate from each test are compared in Fig. 3.

The plates were loaded in the ATR in such a configuration that either a flat plate surface or a narrow edge faced the ATR core cen-

**Table 1**  
Description of irradiated mini fuel plates used in the analysis.

Test	Plate ID	U–Mo foil property and plate fabrication method <sup>a</sup>	Enrichment (%U–235)	U–235 burnup, U–235-fissioned/U–235-initial (%)	Total duration (EFPD)	Fission density <sup>b</sup> ( $10^{21}$ f/cm <sup>3</sup> )	BOL Fuel Temp <sup>c</sup> (°C)
RERTR-6	L1F040	U–10Mo(f,n)	19.7	46	135	3.0	113
RERTR-6	L1F100	U–10Mo(f,n)	19.7	46	135	3.0	124
RERTR-6	L2F030	U–10Mo(f,t)	19.7	40	135	2.6	145
RERTR-7	L1F140	U–10Mo(f,n)	58.2	27	90	4.4	177
RERTR-7	L2F040	U–10Mo(f,t)	58.3	17	90	2.7	199
RERTR-8	H1P010	U–12Mo(p,n)	57.5	31	105	5.7	164
RERTR-9A	L1P04A	U–10Mo(p,n)	58.3	28	98	4.5	152
RERTR-9A	L1F26C	U–10Mo(f,n)	57.5	33	98	5.5	181
RERTR-9A	L1F32C	U–10Mo(f,n)	57.8	32	98	5.4	181
RERTR-9B	L1F34T	U–10Mo(f,n)	58.8	37	115	6.4	186
RERTR-9B	L1P05A	U–10Mo(p,n)	58.3	34	115	5.5	170
RERTR-9B	L1P09T	U–10Mo(p,n)	58.8	39	115	6.8	189
RERTR-10	L1P12Z	U–10Mo(p,n)	67.0	22	75	4.9	167
RERTR-12	L1P755 <sup>d</sup>	U–10Mo(p,n)	70.0	25	90	5.9	156

f: Friction bonding.

p: Hot isostatic pressing bonding.

n: As-fabricated foil thickness = 0.25 mm, plate thickness = 1.4 mm.

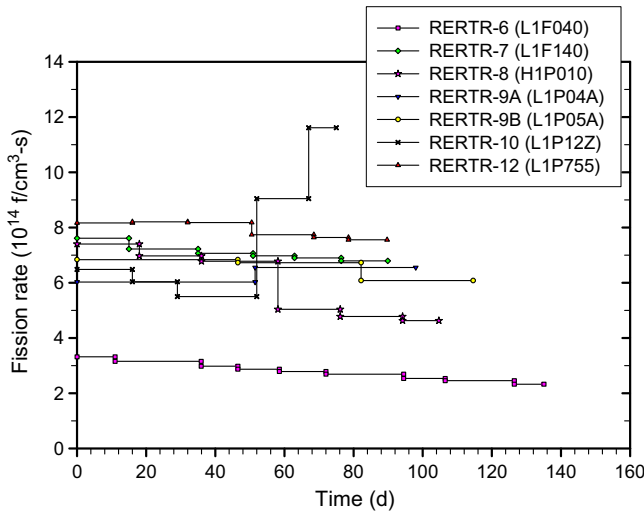
t: As-fabricated foil thickness = 0.50 mm, plate thickness = 1.4 mm.

<sup>a</sup> Number in front of Mo stands for Mo alloying content in weight%.

<sup>b</sup> Fuel volume average at EOL including fissions by Pu produced during irradiation.

<sup>c</sup> At fuel width center.

<sup>d</sup> Face-on loading. All other plates were edge-on loaded to the center of the ATR (Fig. 4).



**Fig. 3.** Foil-averaged fission rate histories of fuel plates. A representative plate from each test is shown. An axial power peaking of 0.95 was multiplied to obtain the real value at the location where the PIE was performed except for RERTR-6. For RERTR-6, the axial power factor is about 1.

ter. In the former case, called face-on loading, the power distribution of the plate in the width direction is approximately symmetric and more uniform. Slight power peakings at the ends still exist due to the self-shielding effect. In the latter case, edge-on loading, the fission density of the edge of the plate that was closer to the ATR core is higher, as is illustrated in Fig. 4. The power ratio of the high power edge to the low power edge is about 2.5 for HEU plates and about 2 for LEU plates of RERTR-6, as determined in neutron physics analyses [5,6].

After irradiation, the plates were sectioned at the axial mid-plane. The fuel cross section was metallographically examined as shown in Fig. 4.

### 3. Post-irradiation analysis

As mentioned earlier, fuel swelling in a plate results only in a plate thickness increase because the cladding is relatively unrestrained in the thickness direction. Therefore, once the post-irradiation foil thickness is known, total fuel swelling can be quantified using the following equation:

$$\left(\frac{\Delta V}{V_0}\right)_f = \frac{\Delta t^f}{t_0^f} \quad (1)$$

where  $\Delta t^f$  is foil thickness change after irradiation, and  $t_0^f$  is the as-fabricated foil thickness.

For plates from RERTR-6, -7, -8, -9A, 9B, -10, and -12, foil thicknesses were measured using post-irradiation micrographs across the foil width with an interval typically of 0.5 mm. The measured fuel swelling data are given in Table 2. The uncertainty in the measured fuel swelling is  $\pm 2\%$ , mainly due to the fabrication variability in foil thickness.

Fuel swelling was also calculated by using the correlation available in the literature [1]. The correlation includes two parts: swelling by solid fission products and swelling by fission gas bubbles. It is expressed as a function of fission density. Therefore, once the fission density at the desired point is known, fuel swelling can be calculated. However, the calculated fuel swelling is purely by fission products; mass transfer by creep is not included. The calculated fuel swelling values are also included in Table 2.

The measured fuel swelling data of L1P04A are plotted with the calculated in Fig. 5. The measured swelling at the tapered end of the foil, marked by A in Fig. 5, is substantially lower than the calculated value. This and the evidence that the fission gas porosity does not vary in size and number density appreciably along several millimeters from the foil end (see Fig. 6) rule out, at least to the first order, the effect of fission gas induced fuel swelling. The calculated areas A and B, the areas enclosed by the measured fuel swelling curve and the calculated swelling curve, were close to each other. The same calculation showed that the areas C and D at the opposite end of the foil were also similar. This salient phenomenon is commonly observed for all plates, although area comparisons are not perfectly consistent in some plates. The conclusion is that a mass transfer has occurred from A to B, facilitated by fission induced creep in the fuel as a response to an applied stress.

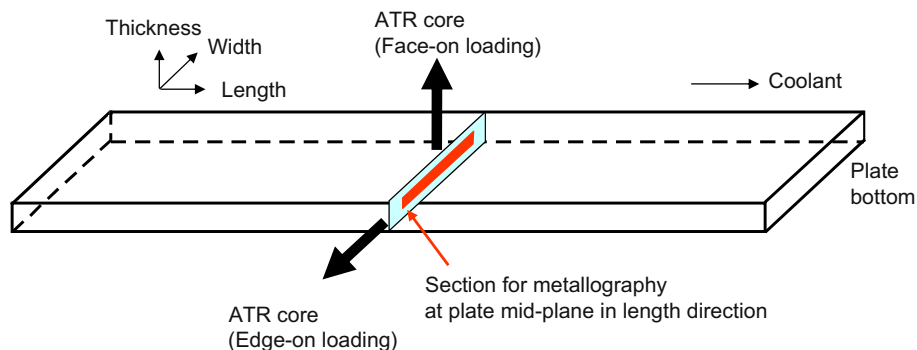
## 4. ABAQUS simulation

### 4.1. Input data

A Young's modulus of 66 GPa and a Poisson's ratio of 0.34 for AA6061 cladding were taken from [7]. Yield strength of the tempered Al alloys increases to  $\sim 280$  MPa due to irradiation hardening [1]. This datum was used for the cladding considering the heat processes during and post fabrication: hot rolling process for plate fabrication and autoclaving for oxide pre-filming. Strain hardening was taken into account and a strain hardening exponent of 0.13 for AA6061-T6 [8] was applied to the cladding with consideration of a decreased potential of the strain hardening by neutron irradiation.

A Young's modulus of U–Mo fuel of 85 GPa and Poisson's ratio of 0.34 were used for U–Mo alloy [9]. Yield strength of U–Mo was not needed in the simulation.

Fission-induced creep of U–Mo is dependent on the applied stress and fission rate. Thermal creep was not considered because the temperature regime of interest is so low that this phenomenon



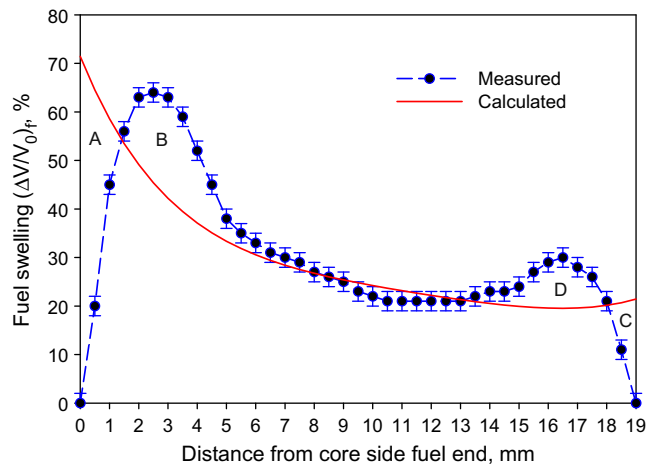
**Fig. 4.** Schematic showing plate loading directions and PIE location.

**Table 2**Fuel swelling comparison between measured and calculated,  $\left(\frac{\Delta V}{V_0}\right)_f$  (%).

Distance from foil end <sup>a</sup> (mm)		0	0.5	1	1.5	2	2.5	3	3.5	4	4.5	5	5.5	6	6.5	7	7.5	8	8.5	9	9.5
L1F040	M	0	7	17	22	28	29	28	25	23	21	21	19	19	19	19	17	17	16	16	15
	C	21	19	19	18	17	16	16	15	15	14	14	14	13	13	13	12	12	12	12	12
L1F100	M	0	10	17	26	31	31	31	29	27	21	25	23	23	22	17	19	19	18	18	16
	C	26	25	24	23	22	21	21	20	19	19	18	18	18	17	17	17	17	16	16	15
L2F030	M	0	8	14	20	24	24	22	20	18	17	16	15	14	13	13	13	12	12	12	11
	C	17	16	16	15	14	14	14	13	13	12	12	12	12	11	11	11	11	10	10	10
L1F140	M	0	28	45	55	59	60	56	48	42	38	35	33	30	30	29	29	27	27	25	24
	C	65	58	52	48	45	42	40	38	36	34	33	32	30	29	28	27	26	26	25	24
L2F040	M	0	9	18	25	29	30	29	26	24	22	21	20	20	19	19	19	17	17	17	16
	C	35	31	28	26	25	23	22	21	20	19	18	18	17	16	16	15	15	15	14	14
H1P010	M	0	23	45	60	70	70	70	60	60	53	50	48	45	43	40	40	38	38	38	38
	C	75	67	61	55	52	49	47	44	42	40	39	39	37	35	34	32	32	31	30	29
L1P04A	M	0	20	45	56	63	64	63	59	52	45	38	35	33	31	30	29	27	26	25	23
	M <sup>b</sup>	0	11	21	26	28	30	29	27	24	23	23	22	21	21	21	21	21	21	22	23
	C	71	65	59	54	49	45	42	39	37	35	33	32	31	29	28	28	27	26	25	25
	C <sup>b</sup>	21	21	20	20	20	20	20	20	20	20	21	21	21	22	22	23	23	24	24	25
L1F26C	M	2	18	37	57	70	74	76	75	71	65	57	53	49	45	41	37	36	35	34	33
	C	76	68	63	59	56	53	50	48	46	44	42	40	39	38	37	36	34	34	33	32
L1F32C	M	0	21	42	58	70	75	77	75	71	62	55	50	45	42	39	37	35	33	32	32
	C	72	67	62	58	54	51	48	46	44	42	40	39	38	36	35	35	34	33	32	32
L1F34T	M	0	25	46	64	77	80	80	76	73	68	63	57	52	50	47	45	42	40	40	39
	C	98	89	82	76	70	66	62	58	55	53	51	49	47	46	44	43	42	41	40	39
L1P05A	M	0	21	42	56	67	71	73	73	69	63	58	52	48	44	42	39	37	35	33	32
	M <sup>b</sup>	0	13	25	31	32	33	32	31	30	29	29	28	28	28	29	30	30	31	31	32
	C	91	81	73	67	62	58	55	52	49	47	45	43	41	40	39	37	36	35	34	33
	C <sup>b</sup>	28	27	27	27	27	27	27	27	27	27	27	28	28	28	29	30	30	31	34	33
L1P09T	M	5	37	64	76	88	90	88	86	80	74	70	66	62	60	56	49	47	45	43	41
	C	104	95	87	81	75	70	66	63	60	57	55	53	51	50	48	47	46	45	44	43
L1P12Z	M	8	30	52	61	68	62	58	53	47	43	37	37	33	31	28	25	25	22	22	20
	M <sup>b</sup>	0	7	16	21	21	21	20	18	18	18	19	18	19	19	20	21	21	21	21	20
	C	72	65	59	54	49	46	42	40	37	35	33	32	31	29	28	28	27	26	25	25
	C <sup>b</sup>	21	21	20	20	20	20	20	20	20	20	21	21	21	21	22	22	23	24	24	25
L1P755	M	15		45		52		43		37		32		30		27		26		25	
	M <sup>c</sup>	12		38		42		38		28		27		25		25		25		24	
	C	50		43		38		35		32		30		29		28		27		27	
	C <sup>c</sup>	42		36		31		29		27		26		26		26		26		26	

M: Measured.

C: Calculated.

<sup>a</sup> Distance from higher power side of foil end.<sup>b</sup> Distance from lower power side of foil end.<sup>c</sup> Distance from the opposite end of the foil in the face-on loaded plate.**Fig. 5.** Comparison between measured and calculated fuel swelling for L1P04A plate.

is inactive. It is accepted in the literature that the fission-induced creep rate is linearly proportional to the fission rate. In some case, a stronger stress dependence was employed such as  $\dot{\epsilon}_c \sim \sigma^{1.5}$  for  $\text{UO}_2$  [10]. However, as Dienst noted [11], a linear stress dependence is most frequently used because it is easy to standardize experimental data with good accuracy. Hence, the following equation has been adopted for the U–Mo creep rate:

$$\dot{\epsilon}_c = A\sigma\dot{f} \quad (2)$$

where  $\dot{\epsilon}_c$  is the equivalent creep strain rate ( $\text{s}^{-1}$ ),  $A$  the creep rate coefficient ( $\text{cm}^3/\text{MPa}$ ),  $\sigma$  the equivalent stress (MPa), and  $\dot{f}$  the fission rate (fissions/ $\text{cm}^3 \text{ s}$ ).

The total fuel swelling (i.e., swelling by fission gas bubble and swelling by solid fission products) correlation is available from [1]:

$$\left(\frac{\Delta V}{V_0}\right)_f = 5.0f_d, \quad \text{for } f_d \leq 3 \times 10^{21} \text{ fissions/cm}^3 \quad (3)$$

$$\left(\frac{\Delta V}{V_0}\right)_f = 15 + 6.3(f_d - 3) + 0.33(f_d - 3)^2, \quad \text{for } 3 \times 10^{21} < f_d \text{ fissions/cm}^3 \quad (4)$$

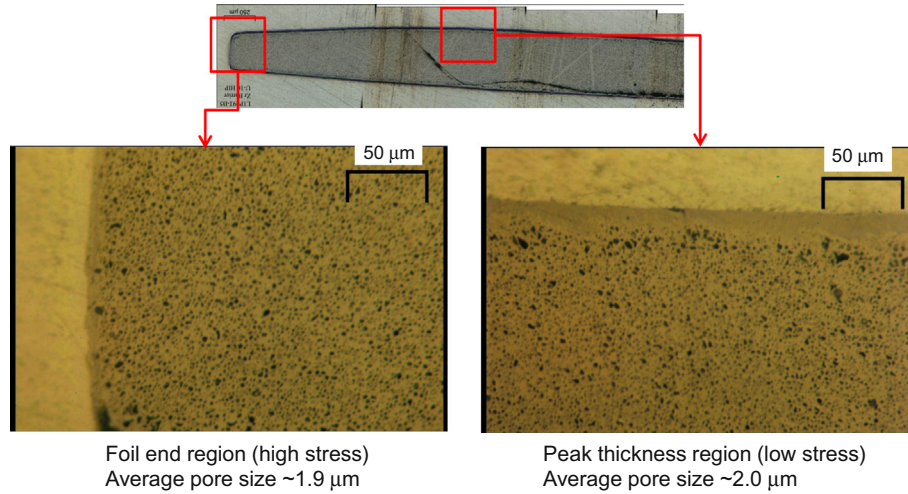


Fig. 6. Optical micrographs showing fission gas pore morphology of L1P09T.

where fuel swelling is in percent and  $f_d$  is in  $10^{21}$  f/cm<sup>3</sup>. Note that the temperature effect is neglected in these equations for the following reasons. At the low temperature regime ( $< \sim 250$  °C), fission gas diffusion is predominantly initiated by fission enhanced diffusion (FED) and the amount of thermally activated gas diffusion is small. Also at these low temperatures, the decomposition of the as-fabricated meta-stable  $\gamma$  phase to the  $\alpha + \gamma'$  phases is very slow and fission induced diffusion effectively reverse the  $\alpha + \gamma'$  phases to the  $\gamma$  phase [12,13]. Because of the  $\gamma$  phase that has small gas bubble size, temperature effect on fuel swelling is small to the extent difficult to discern masked by measurement uncertainties [1].

Fuel true strain is derived from fuel swelling given in Eqs. (3) and (4) as follows:

$$\varepsilon_{f,true} = \ln \left[ 1 + \left( \frac{\Delta V}{V_0} \right)_f \right] \quad (5)$$

#### 4.2. Simulation for creep rate coefficient

The creep rate coefficient  $A$  in Eq. (2) was obtained by simulation of the U–Mo creep using ABAQUS FEA simulation.

The EOL condition of the plate L1P04A, which was irradiated with edge-on loading and had the same cladding thickness on both faces, was simulated. For this case, only the upper half of the fuel plate was modeled by symmetry. A schematic of fuel plate cross-section and the FEA modeling scheme are shown in Fig. 7.

CPEG8R, generalized plane strain 8-node biquadratic quadrilateral with reduced integration, was used. A geometrically nonlinear analysis was applied to make more precise analysis based on the geometry in the most recently completed increment. Again, bonding between fuel and cladding was assumed intact throughout life, which is inaccurate in some cases where debonding occurs.

FEA simulation was performed for fission product induced fuel swelling together with creep in L1P04A using several  $A$  values, and the best simulation was found when  $A = 500 \times 10^{-25}$  cm<sup>3</sup>/MPa, as shown in Fig. 8. The fuel swelling simulation plotted in Fig. 8a shows fair simulation of the swelling peaks at both fuel ends. As the creep coefficient increases, the peaks move toward the foil width center, and their apices become closer to the measured. Fitting becomes worse again when  $A$  is increased above  $500 \times 10^{-25}$  cm<sup>3</sup>/MPa. However, the uncertainties involved in the fuel swelling and the small difference from the case with  $A = 250 \times 10^{-25}$  cm<sup>3</sup>/MPa suggest that

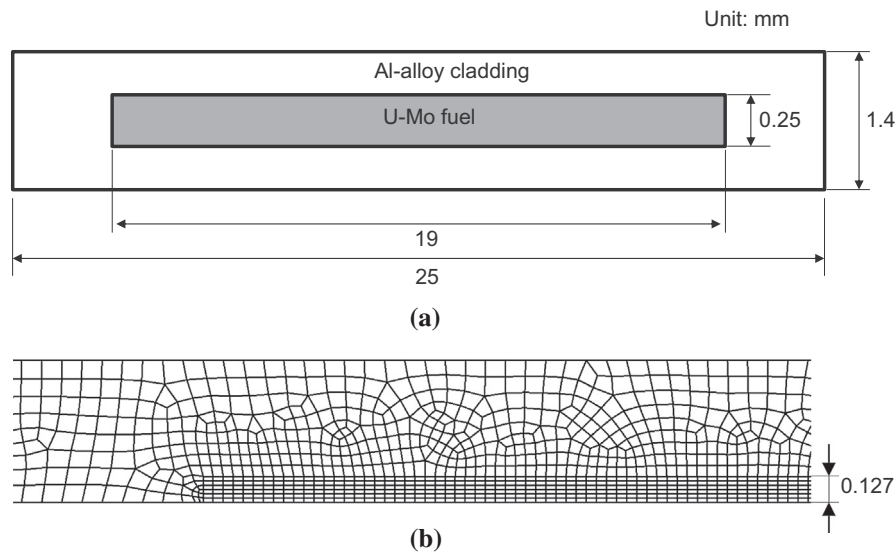
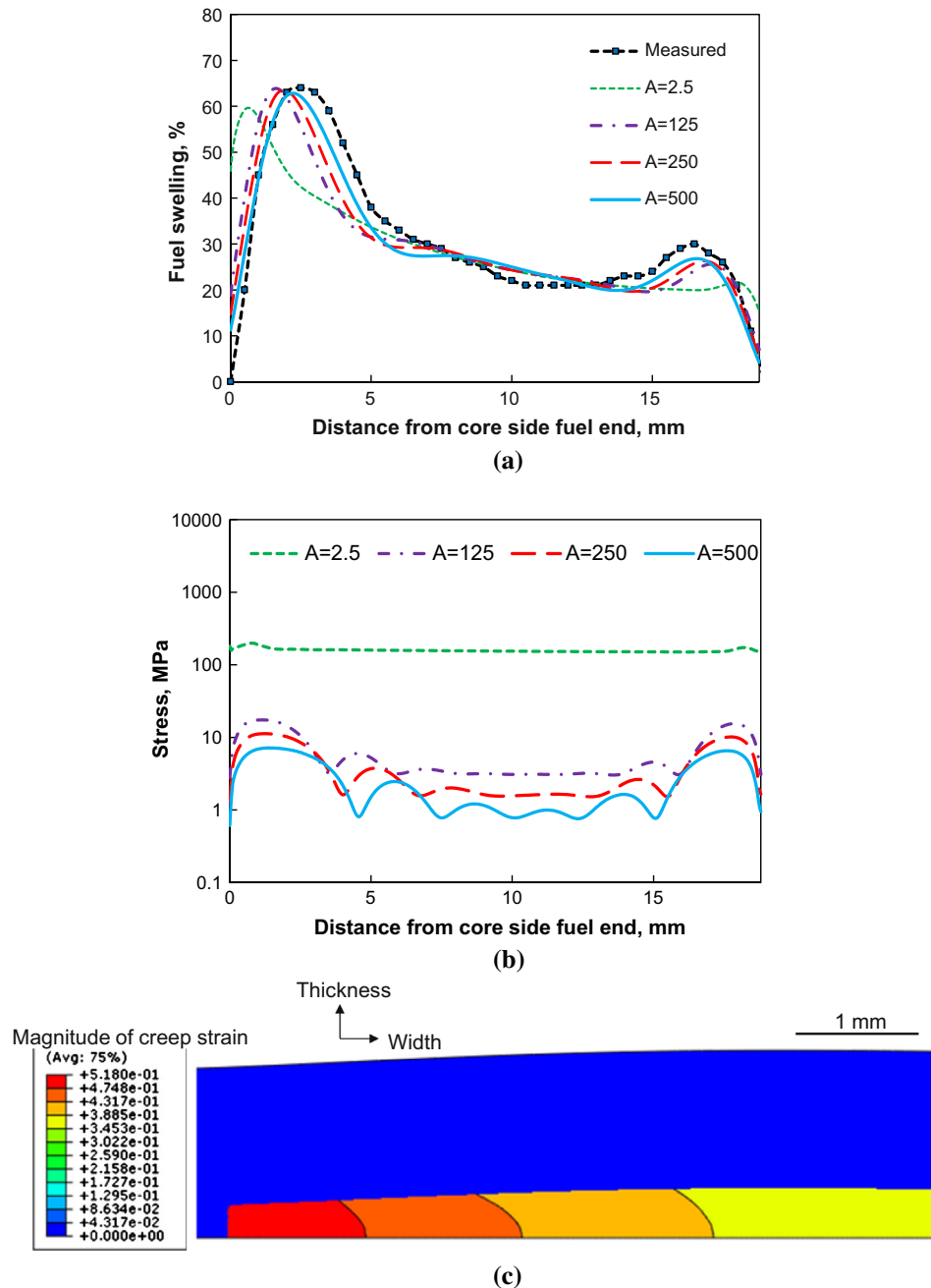


Fig. 7. Finite element modeling for L1P04A with edge-on loading and symmetric cladding thickness. (a) Schematic of fuel plate cross section and (b) finite element modeling.





**Fig. 8.** Finite element simulation results to fit creep rate coefficient using the measured data for L1P04A. (a) Fitting of the fuel swelling by using different creep rate coefficients  $A$  (in  $10^{-25} \text{ cm}^3/\text{MPa}$ ), (b) Von Mises stress corresponding to the creep rate coefficient used in (a), and (c) Contour of fuel volume expansion by the combination of fuel swelling and creep-induced mass transfer from the foil end region to the foil width central region.

the obtained creep rate constant should also include considerable uncertainty although it is difficult to quantify.

The corresponding Von Mises stresses for the fuel swelling given in Fig. 8a are plotted in Fig. 8b. It is considered that the stresses obtained with  $A = 500 \times 10^{-25} \text{ cm}^3/\text{MPa}$  is reasonable. The stress becomes zero at the foil end to satisfy the traction free boundary condition. It is also worth noting that the stress at the fuel central region in the width direction becomes smallest. The wave nature in stress suggests that this uneven stress state may be the major driving force to cause separation of fuel foil from cladding (i.e., debonding) during irradiation observed in some plates.

The simulation result of the combination of fission product induced fuel swelling and fuel mass transfer from the foil end to

the foil central region in L1P04A is shown in Fig. 8c. The FEA simulation also implies that fuel volume increase by fission product-induced fuel swelling at the foil end region is effectively removed and accumulated instead at the region showing peak foil thickness. Fuel flow is the highest at the foil thickness centerline and the lowest at the foil-cladding interface where the fuel is assumed to be perfectly bonded with the cladding.

The driving force for the mass transfer is provided by shear stress in the foil width direction. The evolution of the shear stress at the foil-cladding interface in the thickness direction was calculated at BOL, MOL and EOL and shown in Fig. 9. The shear stress increases as fission product induced fuel swelling increases. As a result, mass transfer increases.

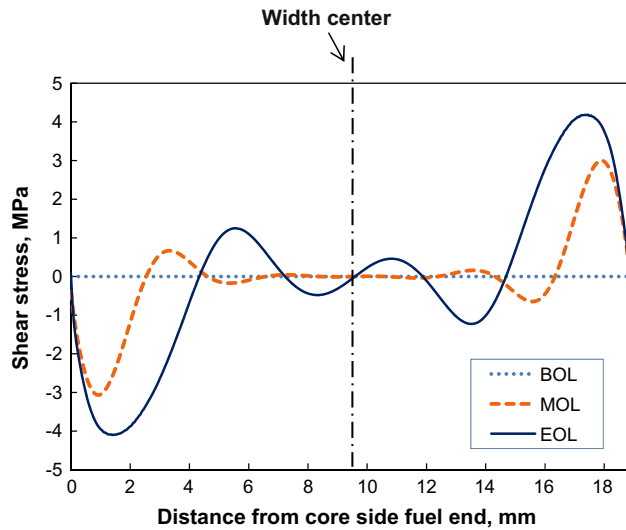


Fig. 9. Calculated shear stress at the foil-cladding interface at three points in irradiation time of L1P04A. The signs are opposite across the center in the width direction due to direction change.

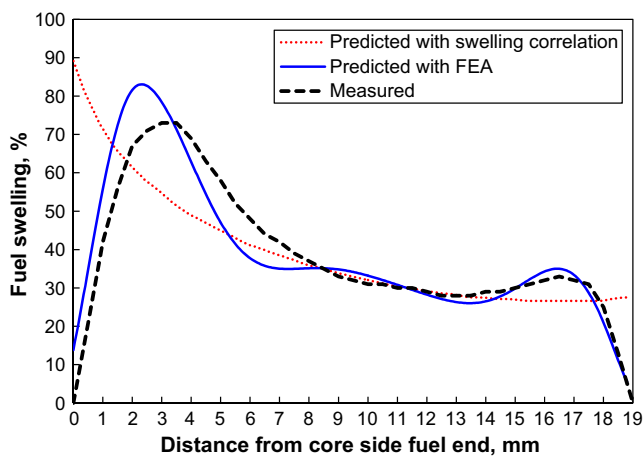


Fig. 10. Comparison of the predicted fuel swelling by FEA to the measured swelling of plate L1P05A. The predicted fuel swelling by the empirical correlation is also provided as a reference.

#### 4.3. Validation of FEA simulation

The creep rate coefficient obtained in the previous subsection  $A = 500 \times 10^{-25} \text{ cm}^3/\text{MPa}$  and the FEA simulation results were validated by applying this value in the simulation for other plates. Among the measured plates given in Table 2, three plates with different fabrication geometry and plate loading direction were selected.

L1P04A and L1P05A had symmetric cladding thickness on both plate sides and edge-on loaded. FEA simulation was performed for L1P05A using the creep rate coefficient  $A = 500 \times 10^{-25} \text{ cm}^3/\text{MPa}$ . The FEA result is in reasonable agreement with the measurement (Fig. 10), although the simulated apex at the core side of L1P05A exhibits a slight discrepancy with the measurement.

L1P12Z was inadvertently fabricated with different cladding thickness on each side. FEA simulation was made for L1P12Z, using  $A = 500 \times 10^{-25} \text{ cm}^3/\text{MPa}$ , to include the effect of the asymmetric cladding thickness. The schematic of fuel cross section and the fuel FEA modeling scheme are shown in Fig. 11. As shown in Fig. 12a, the FEA result is in excellent agreement with the measured fuel swelling. Fuel deformation obtained by FEA shown in Fig. 12b occurs dominantly on the thin cladding side of the plate compared to the thick side, which is in accord with the metallography shown in Fig. 12c.

L1P755 from the RERTR-12 test is a face-on loaded. This loading scheme was employed to provide a more uniform power distribution across the plate width. As shown in Fig. 13a, however, gamma scanning during PIE showed that power peaking still exists although less than the edge-on loaded plates. The FEA simulation result for L1P755, using  $A = 500 \times 10^{-25} \text{ cm}^3/\text{MPa}$  shown Fig. 13b compares the FEA simulation result with the measured fuel swelling and the calculated fuel swelling without considering creep, in which the FEA simulation is consistent with the measurement.

#### 5. Discussion

The consistent FEA simulations with the measured data for L1P05A, L1P12Z and L1P755 using the creep rate coefficient  $A = 500 \times 10^{-25} \text{ cm}^3/\text{MPa}$  in general implies that the fuel mass transfer is indeed enabled by a creep mechanism and that ABAQUS FEA is applicable in simulation of the fission induced creep observed for  $A = 500 \times 10^{-25}$  U–Mo alloy fuel regardless of fuel plate fabrication and loading types. ABAQUS FEA also demonstrates that

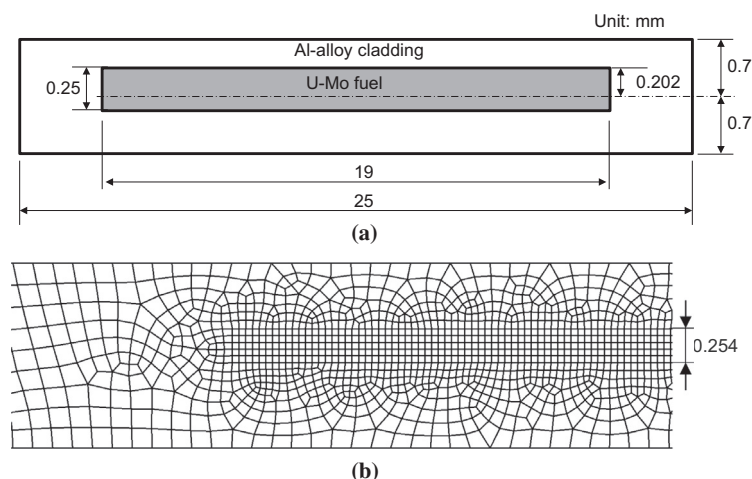
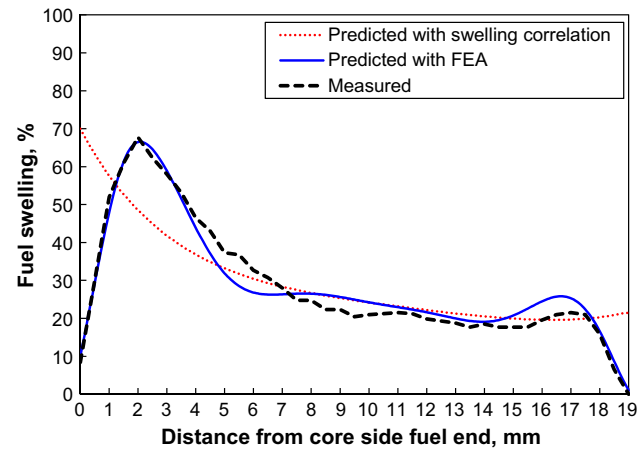
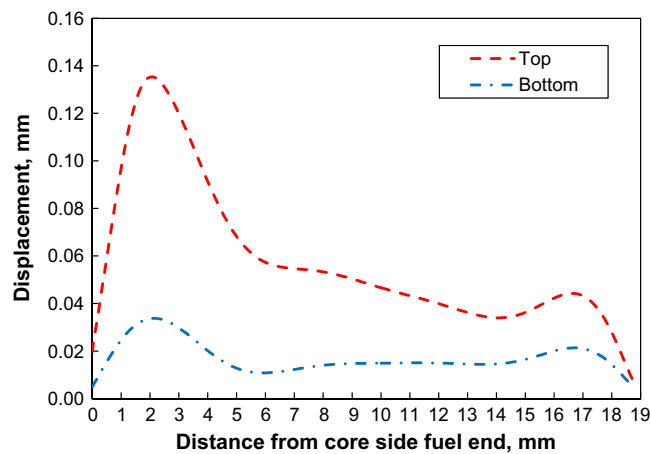


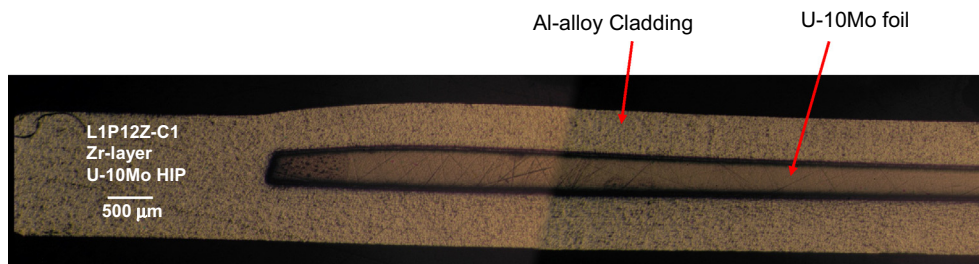
Fig. 11. Finite element modeling for L1P12Z with edge-on loading and asymmetric cladding thickness on each side. (a) Schematic of fuel plate cross section and (b) finite element modeling.



(a) Comparison of the predicted fuel swelling by FEA to the measured swelling of plate L1P12Z. The predicted fuel swelling by the empirical correlation is also provided as a reference.



(b) Fuel displacements for both halves of the foil.



(c) Optical micrograph of cross section of L1P12Z in the width direction

**Fig. 12.** Finite element analysis results and PIE data of L1P12Z.

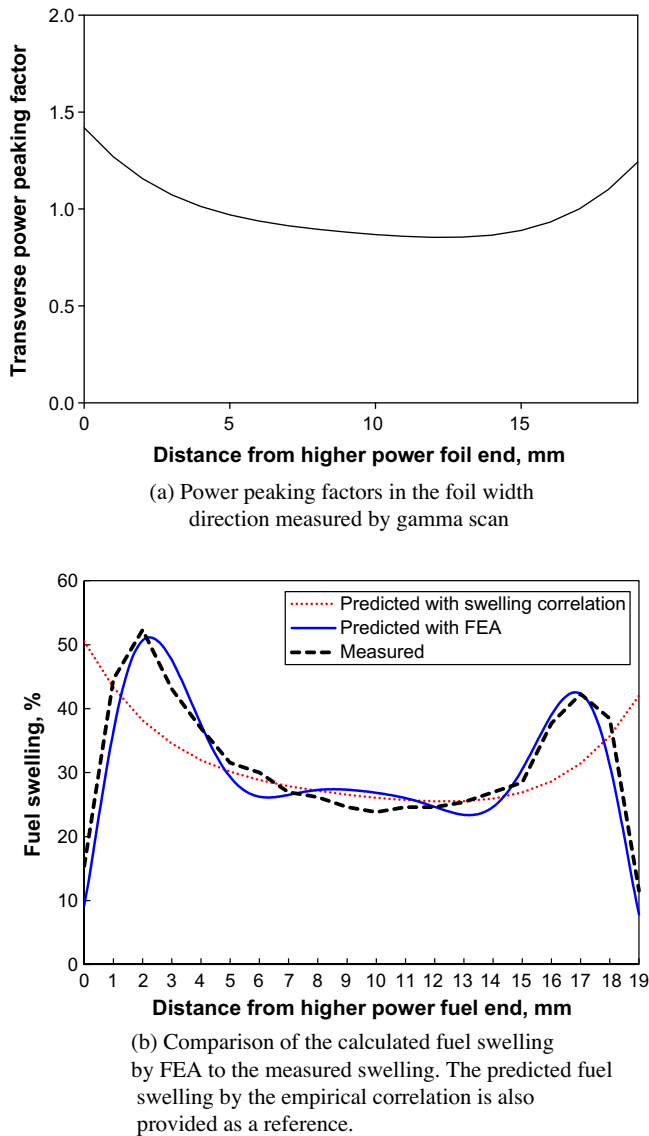
not only the obtained creep rate constant is acceptable, but also the FEA modeling itself is valid in simulating the measured data.

Fission enhanced creep at low homologous temperatures was reported for all uranium fuels and was first identified in  $\alpha$ -U in the 1950s by Russian [14] and British [15] workers, and subsequently in ceramic fuels by various researchers [11,16–22]. Common for all, the creep rate was found to be athermal having a linear dependence on the applied stress and fission rate, as in Eq. (2). The obtained creep rate coefficient  $\text{cm}^3/\text{MPa}$  is compared with other U-fuels in the literature at homologous temperatures relative to their melting points in Table 3. The obtained value for U–10Mo

lies between values of pure uranium and MOX, and is about a half of the value of pure U. The slightly lower creep rate of the U–10Mo compared to pure U is most likely due to its higher strength compared to the pure U.

The high creep rate enables the extent of observed fuel lateral mass transfer, which is the effective mechanism that lessens the stresses caused by fission product induced fuel swelling at the foil end region where peak fission density is usually achieved. This mechanism allows U–Mo fuel to achieve high burnup without failure, by reducing the potential to plate buckling due to lateral stresses induced by fuel swelling. The high creep rate can also explain





**Fig. 13.** Power distribution across foil width and FEA analysis results for plate L1P755.

**Table 3**  
Creep rate coefficient (*A*) in Eq. (2).

Fuel	<i>A</i> ( $10^{-25}$ cm <sup>3</sup> /MPa)	$\sim T/T_m$	Reference
U	800	0.3	[14,17]
U–10Mo wt.%	500	0.3	Present study
MOX	56	0.25	[18]
UO <sub>2</sub>	7	0.25	[11,19,20]
UN	3	0.3	[21,22]
UC	1	0.3	[20]

*T* = test temperature, *T<sub>m</sub>* = melting point.

the extent of fuel particle deformation observed in high burnup dispersion fuel plates shown in Fig. 14, in which the spherical U–Mo particles, when they were as-fabricated, underwent significant deformation during irradiation although deformation of fuel particles can also occur to some extent during production of the particles and plate fabrication. This plate also shows sintering between particles. The analysis of creep behavior of the dispersion fuel plates is not pursued in this work.

The rather large increase in local fuel loading at the peak thickness location resulting from the lateral fuel creep may have to be considered for hot-spot calculations for reactors that operate fuel at high power after substantial burnup. The additional plate thickness increase by creep in addition to fission product induced fuel swelling at the peak thickness location must be incorporated in safety analyses. The additional foil thickness increase by creep is  $\sim 25\%$  from the as-fabricated foil thickness at a fission density of  $7 \times 10^{21}$  f/cm<sup>3</sup>, which may be considerable in a safety analysis.

In the FEA analyses performed in this study, we used life-averaged fission rates instead of the time-dependent ones given in Fig. 3 for the ease of calculation. The validity of this simplistic approach was examined. Fission induced creep is a product of stress and fission rate; however, the stress is more important because it provides the driving force and the fission rate determines amplitude of the creep rate. The stress is produced by the fission product induced fuel swelling, which is a function of fission density. The total creep strain is a time integration of the creep rate given in Eq. (2). Therefore, the case with the time-dependent fission rate has higher fission density in early life than the average-fission rate case because the fission rate decreases with burnup in all of the tests, except for RERTR-9A. FEA simulation for L1F140 at middle of life (MOL) compares the two cases in Fig. 15. The fuel swelling at peak thickness location was slightly ( $\sim 2\%$ ) larger for the time-dependent fission rate case, whereas no difference was found at EOL between the two cases. This result is undoubtedly due to the linear stress on fission rate dependence of the creep rate.

Studies of the effect of porosity on thermal creep are available in the literature (see for example [23]). Porosity typically enhances the thermal creep rate. Since fission gas pores are formed in U–Mo, the porosity effect may be important for a more accurate analysis. Porosity in some high burnup U–Mo plates becomes considerable, greater than 10%. Hayes et al. [23] estimated that  $\sim 12\%$  augmentation in the creep rate was needed at this porosity in UO<sub>2</sub>. Unlike thermal creep, however, for which the exponent of the stress term ranges 3–4, the porosity effect for fission induced creep is smaller because the exponent for the stress term is one. For this reason, a detailed study to accurately factor in this effect is not explored in this study. Nonetheless, it should be noted that the constant creep rate coefficient *A* employed in this study is simplistic. A burnup dependent creep rate coefficient is more appropriate particularly at high burnup when porosity increases considerably by fission gas bubbles.

The temperature effect has not been considered in the FEA performed in this study, which follows the findings reported in literature [11,14–22]. The temperatures of the plates analyzed in this work are narrowly bounded between 110 and 190 °C (Table 1). This low and narrow range of temperatures is the prerequisite for athermal creep.

An implication from this work that affects the measurement of fission product induced fuel swelling is that one must avoid measuring it at or near the foil end regions. Measurements at these regions will lead to erroneous results affected by fuel lateral mass transfer.

## 6. Conclusions

The taper of U–Mo fuel foil observed at the foil width end region, where fission density is highest, has been reviewed. The underlying mechanism is lateral mass transfer by fission induced creep, which is athermal and dependent on fission rate and stress that builds up by fission product induced fuel swelling in the area. The creep of U–Mo fuel is the effective mechanism that lowers the stresses at the foil end region where fission product induced fuel swelling is the highest because of peak fission density there, which

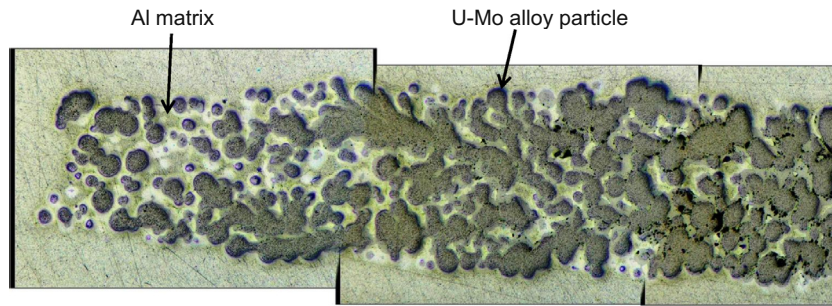


Fig. 14. Optical micrograph of the fuel meat end region of R6R018 where the meat-averaged fission density is  $\sim 10 \times 10^{21}$  f/cm<sup>3</sup>-fuel-particle.

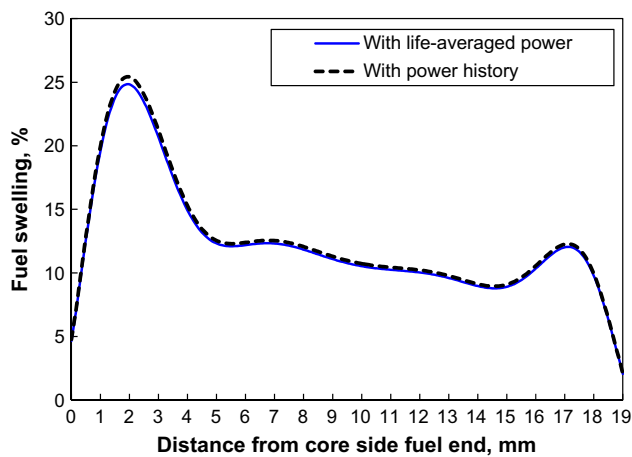


Fig. 15. Comparison of FEA results for fuel swelling between the cases with the time-dependent fission rate and life-average fission rate at MOL for fuel plate L1F140.

enables U–Mo alloy fuel to achieve high burnup without failure. It also provides a realistic explanation for the observed anisotropic swelling in the fuel plates.

ABAQUS finite element analysis (FEA) simulation coupled with a model for fission product induced fuel swelling and creep, produced consistent results using the measured fuel swelling data for all fuel plate types and loading schemes with physical-mechanical data available in the literature. This proves that the fuel mass transfer is indeed induced by a creep mechanism and validates that ABAQUS FEA is applicable in simulation of the fission induced creep observed for U–Mo alloy fuel.

ABAQUS simulation also produced the best-fit creep rate constant for U–10Mo alloy fuel  $500 \times 10^{-25}$  cm<sup>3</sup>/MPa, a value that is approximately a factor of two lower than that for pure uranium. However, the obtained creep rate constant includes considerable uncertainty, for which quantification is not tried in this study.

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