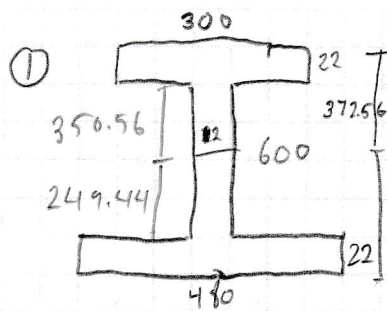


CivE 310 A5



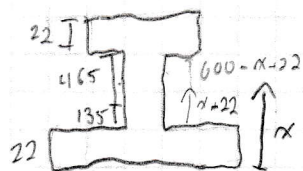
300 mm, $F_y = 300 \text{ MPa}$, $F_u = 450 \text{ MPa}$

$L < L_u$ assumed since
internally braced

$$a) E. NA = \bar{y} = \frac{11(22 \times 480) + 322(12 \times 600) + 633(22 \times 300)}{(300 + 480)(22) + (600)(12)}$$

$$= 271.44 \text{ mm}$$

P. NA = Point of equal areas, assumed in flange



$$480 \times 22 + (x - 22)(12) = 300 \times 22 + (600 + 22 - x)(12)$$

$$10296 + 12x = 14064 - 12x$$

$$24x = 3768$$

$$x = 157 \text{ mm}$$

$$I_x = \int y^2 dA = \int b y^2 dy$$

$$= \frac{12}{3} (249.44)^3 + \frac{12}{3} (350.56)^3 + \frac{300}{3} (372.56^3 - 350.56^3) + \frac{480}{3} (271.44^3 - 249.44^3)$$

$$= 1814160932 \text{ mm}^4$$

$$S_x = I_x / y$$

$$= \frac{1814160932}{372.56}$$

$$= 4869446.35 \text{ mm}^3$$

$$Z_x = A_{cy} + A_{ty}$$

$$= 480 \cdot 22 \cdot (135 + 11) + 135 \cdot 12 \cdot 67.5 + 465 \cdot 12 \cdot 232.5 + 300 \cdot 22 \cdot 476$$

$$= 6090060 \text{ mm}^3$$

$$\text{Shape Factor} = \frac{M_p}{M_y} = \frac{Z}{S} = \frac{6090060}{4869446.35} = 1.25$$

b) Section Classification

Flange

$$\frac{b_{eff}}{t} = \frac{480}{22.2} = 10.91 \leq \frac{145}{\sqrt{300}} = 8.37, \frac{170}{\sqrt{300}} = 9.81, \frac{200}{\sqrt{300}} = 11.547$$

\therefore Flange class 3

Web

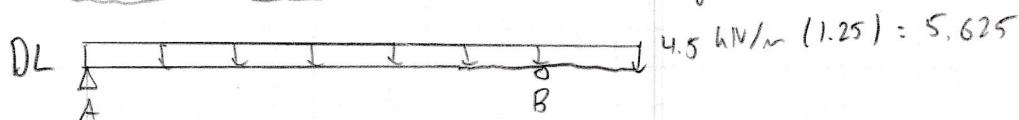
$$\frac{h}{w} = \frac{600}{12} = 50, \frac{1100}{\sqrt{300}} = 63.5, \therefore \text{Class 1}$$

\therefore Section is class 3, fails at yield stress

$$M_{max} = M_y = S F_y = 4869446.35 \cdot 300 \times 0.9^{\phi} \cdot \left(\frac{1}{1000}\right)^2$$

$$= 1314.75 \text{ kN.m}$$

To find P_{max} , consider moment diagrams for DL and LL, respectively



$$M_B = 5.625(2.5)(11.25)$$

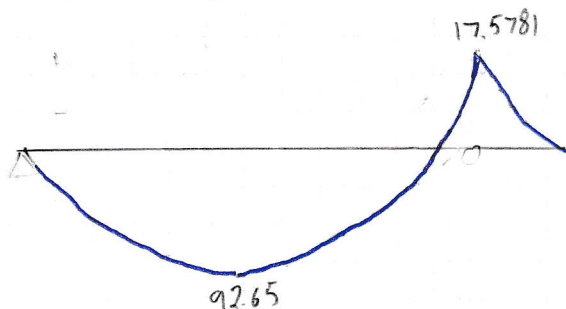
$$= 17.5781 \text{ kN.m}$$

Max moment between supports =

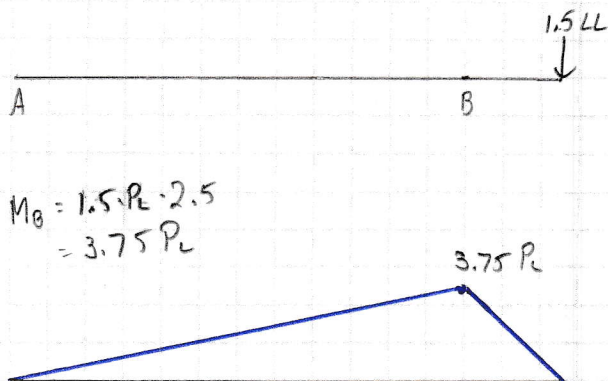
$$\frac{w}{8l^2} (l+a)^2 (l-a)^2$$

$$= \frac{5.625}{8 \cdot (12)^2} (12+2.5)^2 (12-2.5)^2$$

$$= 92.65 \text{ kN.m}$$



LL



$$M_B = 1.5 \cdot P_L \cdot 2.5 \\ = 3.75 P_L$$

From these two bending moment diagrams, it is clear that for large values of P_L , the maximum moment is found at reaction B . Since the maximum moment in the span is $92.65 < 1314.75$, failure can only happen at B . (assuming $M_{span} > M_B$)

$$\therefore 5.625(2.5)(2.5/2) + 3.75 P_L = 1314.75$$

$$P_L = 345.913 \text{ kN at failure}$$

$$c) E = 2E5 \text{ N/mm}^2, w = 4.5 \text{ kN/m} = 4.5 \text{ N/mm}$$

$$\Delta_{\text{deflection}} = \frac{w a}{24 E I} (4 a^2 l - l^3 + 3 a^3) + \frac{P a^2}{3 E I} (l + a)$$

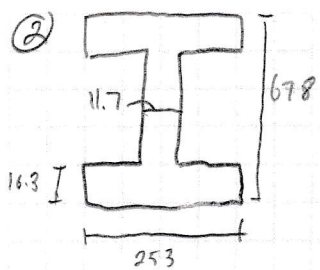
$$= \frac{4.5(2500)}{24 \cdot 2E5 \cdot 1814160932} (4 \cdot 2500^2 \cdot 12000 - 12000^3 + 3 \cdot 2500^3) + \frac{345913 \cdot 2500^2}{3 \cdot 2E5 \cdot 1814160932} (14500)$$

$$= -1.71 + 28.80$$

$$= 27.09 \text{ mm} \downarrow$$

$$\frac{L}{360} = \frac{2500}{360} = 6.94 \text{ mm} \downarrow$$

\therefore Excessive deflection



125 kg/m

$$S_{xx} = 3500 \times 10^3 \text{ mm}^3$$

$$Z_{xx} = 4010 \times 10^3 \text{ mm}^3$$

Joists @ 2500 mm o/c

$$\text{Factored load} = 1.25(15) + 1.5(38) = 75.75$$

a) Unbraced length = 2500 mm due to joist spacing

From beam table, $L_u = 3190$.

Since $2500 < L_u$, lateral-torsional buckling will not control

b) Section classification

Flange

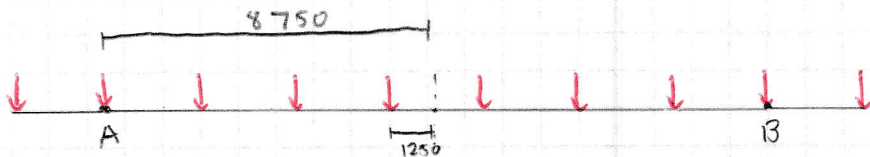
$$\text{Flange } \frac{b_{fl}}{t} = \frac{253/2}{16.3} = 7.76 \leq \frac{145}{\sqrt{345}} = 7.8, \therefore \text{Class 1}$$

$$\text{Web } \frac{h}{w} = \frac{678 - 16.3(2)}{11.7} = 55.16 \leq \frac{1100}{\sqrt{345}} = 59.22, \therefore \text{Class 1}$$

\therefore Can reach plastic moment

$$M_p = \phi Z_x F_y = 0.9 \cdot 4010 \text{ E}3 \times 345 \times \left(\frac{1}{1000}\right)^2 = 1245.105 \text{ kN}\cdot\text{m}$$

$$P = 1.25(15) + 1.5(38) = 75.75 \text{ kN}$$

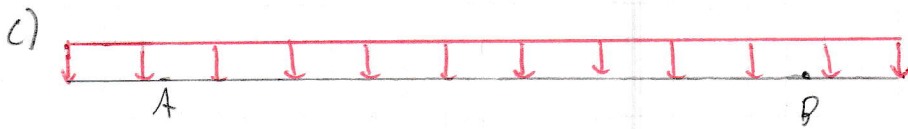


$$M_A = M_B = 75.75 \cdot 2.5 = 189.375 \text{ kN}\cdot\text{m}$$

$$A_y = B_y = \frac{75.75 \times 10}{2} = 378.75 \text{ kN}$$

$$\begin{aligned} \text{Max moment in midspan, } \therefore M &= 75.75 [1.25 + (1.25 + 2.5) + (1.25 + 2 \times 2.5) + (1.25 + 3 \times 2.5) + (1.25 + 4 \times 2.5)] \\ &\quad - 378.75 (8.75) \\ &= 2367.19 - 3314.0625 \\ &= 946.875 \text{ kN}\cdot\text{m} \end{aligned}$$

$$946.875 < 1245.105, \text{ OK}$$



$$w_d = \frac{15 \cdot 1.25 \times 10}{22.5} = 25/3 \text{ kN/m}$$

$$M_A = M_B = \frac{101}{3} \cdot 22.5 \times 1.25$$

$$= 105.208 \text{ kNm}$$

$$w_c = \frac{1.5 \cdot 38 \cdot 10}{22.5} = 76/3 \text{ kN/m}$$

$$\therefore w_T = \frac{101}{3} \text{ kN/m}$$

$$A_g = B_g = \frac{\frac{101}{3} \times 22.5}{2} = 378.75 \text{ kN}$$

$$M_{midspan} = \frac{101}{3} \times \frac{(8.75 + 22.5)^2}{2} - 378.75 \cdot 8.75$$

$$= 2130.4688 - 3314.0625$$

$$M_{midspan} = 1183.59 \text{ kNm}$$

This method is very conservative, yielding a moment 236.72 kNm greater than reality. However, the moment calculated is below the plastic moment, and therefore it passes.

Thus, both methods demonstrate that the beam passes, but the latter is less accurate.