

Lecture 1: Dynamical Systems, Equilibrium, and Stability

Textbook Sections 1.1, 1.2, & 1.3

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Model-Based Design

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- Model-based design uses coverall approach to standard embedded systems development → time to port between modeling software and embedded systems can outweigh temporal value for alternative lab-based design

Model-Based Design Software

- Uses graphical modeling tools: generic and unified graphical modeling environment
 - Reduces complexity of model designs by breaking into hierarchies of individual design blocks
 - Helps design engineers to conceptualize entire system, typically with **graphical user interface (GUI)**
 - Connection between signals and systems in these GUIs typically depicted using **block diagrams**, for which each “block” represents system and each “arrow” represents signal

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- Example of basic block diagram:
 - System: G
 - Input signal: u
 - Output signal: y



Planning, Control, and Perception

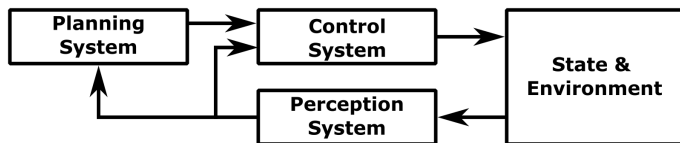
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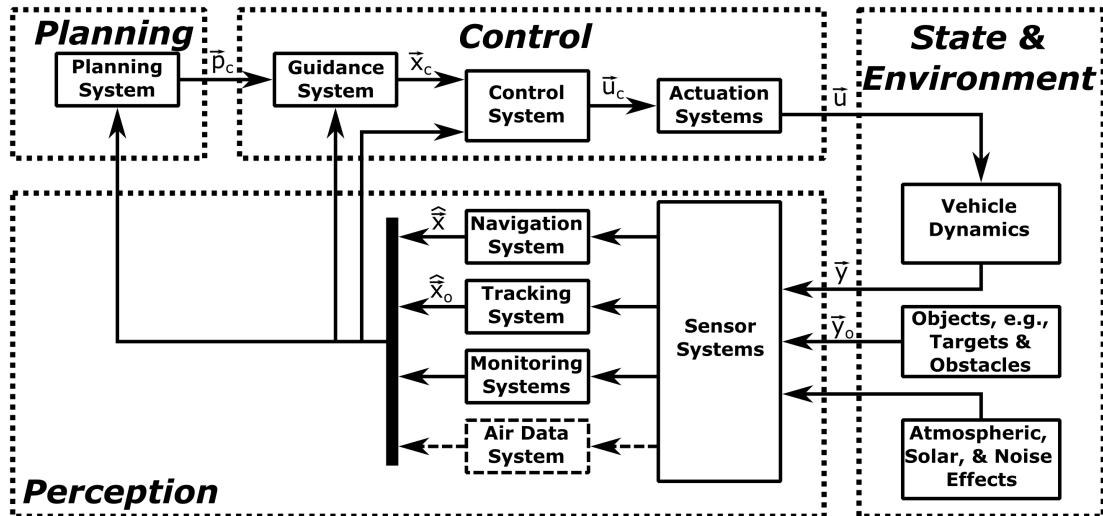
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- In modern aerospace applications, design of **planning, control, and perception** systems, fundamental to operations of aerospace vehicles, a.k.a. flight vehicles
- Three systems understood as connected together through:



- Planning system: determines what mission/path/trajectory, \vec{p}_c , aerospace vehicle should fly
- Control system: executes path/trajectory for aerospace vehicle by actuating control inputs, \vec{u}
- Perception system(s): senses and estimates state of aerospace vehicle, $\hat{\vec{x}}$, state of object(s) in environment, $\hat{\vec{x}}_o$, and state of environment
 - For aerospace vehicles, perception systems provide feedback to planning system & control system

Aerospace Planning, Control, and Perception System



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- 1950s and 1960s: space race generated interest in embedded control and perception systems
 - Engineers constructed control and perception systems as part of end product, e.g. engine control units and flight simulators
- 1970s, computer-based control and perception systems introduced and became standard
 - Drastic shift in control and perception system design
 - Led to modern use of model-based design for control and perception systems

4 Steps of Model-Based Design

- 1 **Plant modeling:** consists of identifying system model to be controlled, i.e. **plant** which comes from “powerplant” in early days of control and perception systems
 - Plant modeling can be based on first principles or machine learning
 - First principles modeling implements physics-derived mathematical model for plant dynamics, e.g. Newton-Euler EOMs
 - Machine learning modeling processes raw data from real-world system and uses learning algorithms to identify data-driven model for plant dynamics

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- 2 Control and perception design:** consists of **control and perception analysis** and **control and perception synthesis**
 - Mathematical plant model developed in first step used to identify suitable **design requirements** for controlled plant
 - Based on these identified requirements, control and perception strategies chosen
 - Control system and perception system synthesized: explicit **control law** and **observer law**, i.e. two mathematical algorithms for computing plant inputs automatically based on external commands and sensor data

4 Steps of Model-Based Design (continued)

- 3 System simulation:** performed as multiple simulations with lower/higher fidelity plant models as well as offline/real-time simulations
- Allow specification, requirements, and modeling errors to be found immediately, rather than later in overall system design effort
 - Real-time simulation: test control system, perception system, and plant on real-time modeling computer, a.k.a. **software-in-the-loop (SIL)** simulation
 - Control and perception systems implemented on embedded system hardware and simulated with real-time plant and sensor models, a.k.a. **hardware-in-the-loop (HIL)** simulation

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- 4 **Validation and verification (V&V)** of control and perception system
 - Control and perception systems implemented on embedded system hardware and tested with actual plant, e.g. aerospace vehicle, and sensors
 - Control and perception systems highly unlikely to work exactly same on actual plant and sensors as in simulation → iterative debugging process by analyzing test results and updating control and perception system designs until design requirements met in actuality

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- **System:** process that produces output signals in response to input signals
 - Output signal from system, a.k.a. **system response**
 - System characterized by its different properties

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- Dynamical systems theory encompasses three broad topics:
 - Simulation
 - Control
 - **System identification (SID)**
 - Course: advanced simulation and control for aerospace vehicles

Dynamical System Characterization

- Characterize dynamical systems: signal types
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 - **Time-varying system**: mathematical rule that depends *explicitly* on time
 - **Deterministic system**: always produces same output signal for given input signal
 - **Stochastic system**: never produces same output signal for given input signal

Dynamical Systems Theory Scope

- Characterize system: number of input and output signals
 - **Single Input, Single Output (SISO) system**
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- Assumptions simplify mathematical theory of dynamical systems & provide valuable insight into FDC

State-Space Representation

- Continuous-time MIMO dynamical systems: **continuous-time state-space model**

$$\begin{aligned}\dot{\vec{x}}(t) &= f(t, \vec{x}, \vec{u}) \\ \vec{y}(t) &= h(t, \vec{x}, \vec{u})\end{aligned}\tag{1}$$

- $\vec{u}(t) \in \mathbb{R}^{n_u}$: **input vector** of n_u input signals
- $\vec{y}(t) \in \mathbb{R}^{n_y}$: **output vector** of n_y output signals
- $\vec{x}(t) \in \mathbb{R}^{n_x}$: **state vector** of n_x^{th} -order dynamical system

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- First vector-valued differential equation: **dynamics equation** for state-space systems, a.k.a. **state equation**
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- System state \vec{x} dynamically *controlled* by input \vec{u} and *observed* through output \vec{y}

State-Space by Component

- Vector-valued state-space system rewritten as n_x first order ODEs for state equation and n_y algebraic equations for output equation:

$$\begin{aligned}\dot{x}_1 &= f_1(t, x_1, \dots, x_{n_x}, u_1, \dots, u_{n_u}) \\ \vdots &= \vdots \\ \dot{x}_{n_x} &= f_{n_x}(t, x_1, \dots, x_{n_x}, u_1, \dots, u_{n_u}) \\ y_1 &= h_1(t, x_1, \dots, x_{n_x}, u_1, \dots, u_{n_u}) \\ \vdots &= \vdots \\ y_{n_y} &= h_{n_y}(t, x_1, \dots, x_{n_x}, u_1, \dots, u_{n_u})\end{aligned}\tag{2}$$

- x_i denotes i^{th} element of \vec{x}
- u_j denotes j^{th} element of \vec{u}
- y_k denotes k^{th} element of \vec{y}

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 - Let output $y_1(t)$ be solution to linear ODE with input $u_1(t)$ and zero initial conditions
 - Scaling property: for any real number c , solution of linear ODE with input $u_s(t) = cu_1(t)$ and zero initial conditions is $y_s(t) = cy_1(t)$

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 - Let $y_2(t)$ be solution with $u_2(t)$ and zero initial conditions
 - Additivity property: for solution of linear ODE with input $u_a(t) = u_1(t) + u_2(t)$ and zero initial conditions is $y_a(t) = y_1(t) + y_2(t)$

Continuous-Time Linear System Models

- **Continuous-time linear state-space model** general form:

$$\begin{aligned}\dot{\vec{x}}(t) &= A(t)\vec{x}(t) + B(t)\vec{u}(t) \\ \vec{y}(t) &= C(t)\vec{x}(t) + D(t)\vec{u}(t)\end{aligned}\tag{3}$$

- $A(t) \in \mathbb{R}^{n_x \times n_x}$: **state matrix**
- $B(t) \in \mathbb{R}^{n_x \times n_u}$: **input matrix**
- $C(t) \in \mathbb{R}^{n_y \times n_x}$: **output matrix**
- $D(t) \in \mathbb{R}^{n_y \times n_u}$: **feedthrough matrix**

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- **Continuous-time linear time-invariant state-space models** general form

$$\begin{aligned}\dot{\vec{x}}(t) &= A\vec{x}(t) + B\vec{u}(t) \\ \vec{y}(t) &= C\vec{x}(t) + D\vec{u}(t)\end{aligned}\tag{5}$$

- LTI systems theory borrows heavily from mathematical methods in differential equations and linear algebra, utilized throughout course

Notes on LTI Systems

- Particular LTI state-space model can be denoted by the quadruple (A, B, C, D)
 - Different choices for (A, B, C, D) that represent same dynamical system in terms of input-to-output relationship
 - Internal state different for each representation

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- To set $\vec{y} = \vec{x}$: set output matrix to **identity matrix**

$$C = I = \begin{bmatrix} 1 & \cdots & 0 \\ \vdots & \ddots & \vdots \\ 0 & \vdots & 1 \end{bmatrix} \quad (6)$$

& set feedthrough matrix to **zero matrix**

$$D = \begin{bmatrix} 0 & \cdots & 0 \\ \vdots & \ddots & \vdots \\ 0 & \cdots & 0 \end{bmatrix} \quad (7)$$

Transfer Function

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- Numerator and denominator: polynomials \rightarrow have real and/or complex roots
 - **Zero** of transfer function z : root of numerator
 - $|G(s)| \rightarrow 0$ for s equal to zero
 - **Pole** of transfer function p : root of denominator
 - $|G(s)| \rightarrow \infty$ for s equal to pole
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 - Plotting $|G(s)|$ over complex plane, ∞ looks like tent pole in 3D
- Can always factor transfer function as

$$G(s) = \frac{K(s - z_1) \cdots (s - z_m)}{(s - p_1) \cdots (s - p_n)} \quad (10)$$

Notes on Transfer Functions

- Can always factor transfer function as **partial-fraction decomposition**, i.e.

$$G(s) = \frac{r_1}{(s - p_1)} + \cdots + \frac{r_n}{(s - p_n)} \quad (11)$$

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- **Final Value Theorem (FVT)**: if every pole of transfer function $F(s)$ not purely imaginary *except*, at most, single pole at origin, then:

$$\lim_{t \rightarrow \infty} f(t) = \lim_{s \rightarrow 0} sF(s) \quad (12)$$

- $s \rightarrow 0$ denotes s approaching through positive numbers

Transfer Functions in Multivariate Domain

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- Transform to Laplace domain with zero initial conditions as

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$$(sI - A)\vec{x}(s) = B\vec{u}(s) \quad (17)$$

Transfer Function Matrix

- To find **transfer function matrix**, $[G(s)]$, maps each input to each output through matrix of transfer functions, i.e.

$$\vec{y}(s) = [G(s)] \vec{u}(s) \quad (15)$$

- Requires finding $\vec{x}(s)$ in terms of $\vec{u}(s)$ and substituting into output equation

$$s\vec{x}(s) - A\vec{x}(s) = B\vec{u}(s) \quad (16)$$

$$(sI - A) \vec{x}(s) = B\vec{u}(s) \quad (17)$$

$$\vec{x}(s) = (sI - A)^{-1} B\vec{u}(s) \quad (18)$$

- $(sI - A)^{-1}$: **resolvent** of A

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- By definition of transfer function matrix, **standard transfer function matrix**:

$$[G(s)] = C(sI - A)^{-1} B + D \quad (21)$$

- $n \times m$ matrix of transfer function elements, $G_{ij}(s)$, i.e.

$$[G(s)] = \begin{bmatrix} G_{11}(s) & \cdots & G_{1m}(s) \\ \vdots & \ddots & \vdots \\ G_{n1} & \cdots & G_{nm}(s) \end{bmatrix} \quad (22)$$

Equilibrium Points for State-Space Models

- State-input pair (\vec{x}, \vec{u}) **equilibrium point**: if $\dot{\vec{x}}$ zero for all $t > 0$, i.e. if

$$f(t, \vec{x}, \vec{u}) = 0 \quad (23)$$

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 - I.e. \vec{x} “steady”
- Also may be multiple, even infinite, solutions for particular **trim state**, \vec{x}
 - Fewer equations than free variables for vector-valued state equation
 - I.e. n_x elements and $n_x + n_u$ free variables

Lyapunov Stability Definitions

- Important characterization of system equilibrium points: stability
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 - 2 For highly nonlinear ODEs one typically analyzes system's stability using Lyapunov's second method as linearized stability neighborhood too small to be practically useful
- LTI FDC considers Lyapunov's first method for time-invariant systems
 - Provide methods for assessing flight vehicle's stability and designing control systems
 - Nonlinear and time-varying control: discusses Lyapunov's second method

Multivariate Linearization

- **Taylor Series** for multivariate function $f(\vec{x}, \vec{u})$ about vector pair $(\vec{\bar{x}}, \vec{\bar{u}})$:

$$\dot{\vec{x}}(t) = f(\vec{x}, \vec{u}) = f(\vec{\bar{x}}, \vec{\bar{u}}) + \left[\frac{\partial f}{\partial \vec{x}}(\vec{\bar{x}}, \vec{\bar{u}}) \right] (\vec{x} - \vec{\bar{x}}) + \left[\frac{\partial f}{\partial \vec{u}}(\vec{\bar{x}}, \vec{\bar{u}}) \right] (\vec{u} - \vec{\bar{u}}) + \text{HOT} \quad (24)$$

&

$$\vec{y}(t) = h(\vec{x}, \vec{u}) = h(\vec{\bar{x}}, \vec{\bar{u}}) + \left[\frac{\partial h}{\partial \vec{x}}(\vec{\bar{x}}, \vec{\bar{u}}) \right] (\vec{x} - \vec{\bar{x}}) + \left[\frac{\partial h}{\partial \vec{u}}(\vec{\bar{x}}, \vec{\bar{u}}) \right] (\vec{u} - \vec{\bar{u}}) + \text{HOT} \quad (25)$$

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- $\left[\frac{\partial f}{\partial \vec{x}}(\vec{\bar{x}}, \vec{\bar{u}}) \right]$: **Jacobian** of $f()$
- Multivariate linearization a.k.a. **Jacobian linearization**

Perturbation Vectors

- State **perturbation vector** about constant \vec{x} :

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- Recognize for trim, $f(\vec{x}, \vec{u}) = 0$ and $h(\vec{x}, \vec{u}) = \vec{y}$:

$$\Delta \dot{\vec{x}}(t) = \dot{\vec{x}}(t) = f(\vec{x}, \vec{u}) = f(\vec{x}, \vec{u}) + \left[\frac{\partial f}{\partial \vec{x}}(\vec{x}, \vec{u}) \right] \Delta \vec{x}(t) + \left[\frac{\partial f}{\partial \vec{u}}(\vec{x}, \vec{u}) \right] \Delta \vec{u}(t) + \text{HOT} \quad (29)$$

Time-Invariant State-Space Model

- Recall general form for time-invariant state-space model:

$$\begin{aligned}\dot{\vec{x}} &= f(\vec{x}, \vec{u}) \\ \vec{y} &= h(\vec{x}, \vec{u})\end{aligned}\tag{31}$$

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- Set

$$A = \left[\frac{\partial f}{\partial \vec{x}}(\vec{x}, \vec{u}) \right] = \begin{bmatrix} \frac{\partial f_1}{\partial x_1}(\vec{x}, \vec{u}) & \cdots & \frac{\partial f_1}{\partial x_{n_x}}(\vec{x}, \vec{u}) \\ \vdots & \ddots & \vdots \\ \frac{\partial f_{n_x}}{\partial x_1}(\vec{x}, \vec{u}) & \cdots & \frac{\partial f_{n_x}}{\partial x_{n_x}}(\vec{x}, \vec{u}) \end{bmatrix}\tag{32}$$

$$B = \left[\frac{\partial f}{\partial \vec{u}}(\vec{x}, \vec{u}) \right] = \begin{bmatrix} \frac{\partial f_1}{\partial u_1}(\vec{x}, \vec{u}) & \cdots & \frac{\partial f_1}{\partial u_{n_u}}(\vec{x}, \vec{u}) \\ \vdots & \ddots & \vdots \\ \frac{\partial f_{n_x}}{\partial u_1}(\vec{x}, \vec{u}) & \cdots & \frac{\partial f_{n_x}}{\partial u_{n_u}}(\vec{x}, \vec{u}) \end{bmatrix}\tag{33}$$

Time-Invariant State-Space Model (continued)

$$C = \left[\frac{\partial h}{\partial \vec{x}}(\vec{x}, \vec{u}) \right] = \begin{bmatrix} \frac{\partial h_1}{\partial x_1}(\vec{x}, \vec{u}) & \cdots & \frac{\partial h_1}{\partial x_{n_x}}(\vec{x}, \vec{u}) \\ \vdots & \ddots & \vdots \\ \frac{\partial h_{n_y}}{\partial x_1}(\vec{x}, \vec{u}) & \cdots & \frac{\partial h_{n_y}}{\partial x_{n_x}}(\vec{x}, \vec{u}) \end{bmatrix} \quad (34)$$

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- Yields approximate LTI state-space model about $\vec{\bar{x}}$ and $\vec{\bar{u}}$:

$$\begin{aligned} \Delta \dot{\vec{x}}(t) &\approx A \Delta \vec{x}(t) + B \Delta \vec{u}(t) \\ \Delta \vec{y}(t) &\approx C \Delta \vec{x}(t) + D \Delta \vec{u}(t) \end{aligned} \quad (36)$$

General Solution for LTI Systems

- General LTI state-space form

$$\begin{aligned}\dot{\vec{x}}(t) &= A\vec{x}(t) + B\vec{u}(t) \\ \vec{y}(t) &= C\vec{x}(t) + D\vec{u}(t)\end{aligned}\tag{37}$$

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- Boundary conditions: $\vec{x}(t_0)$
- General solution for vector-valued first-order state equation:

$$\begin{aligned}\vec{x}(t) &= e^{A(t-t_0)}\vec{x}(t_0) + \int_{t_0}^t e^{A(t-\tau)}B\vec{u}(\tau)d\tau \\ \vec{y}(t) &= C\left[e^{A(t-t_0)}\vec{x}(t_0) + \int_{t_0}^t e^{A(t-\tau)}B\vec{u}(\tau)d\tau\right] + D\vec{u}(t)\end{aligned}\tag{38}$$

State-Space Characteristic Equation

- **State-space characteristic equation** for general LTI systems:

$$\det [\lambda I - A] = 0 \quad (39)$$

- Solutions to equation: n_x roots of state-space characteristic polynomial, may be repeated and will either be real numbers or complex-conjugate pairs

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- Roots of characteristic polynomials a.k.a. **system poles**
 - If no pole-zero cancellations, then poles of any $G_{ij}(s)$ also system poles
- Each distinct real pole and each distinct complex-conjugate pair: **system mode**
 - Poles in LHP: stable

Stability

- Real part of each individual mode: characterize stability of each mode
 - **Stable** mode: real part of $\lambda < 0$
 - **Marginally stable** mode: real part of $\lambda = 0$
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- Stable LTI system: *all* modes stable
 - If *any* mode unstable \rightarrow unstable LTI system
- Eigenvalues may be complex-valued \rightarrow system poles located in left half of complex plane, i.e. **left half plane (LHP)**, correspond to stable modes

Modal Characteristics

- Each mode corresponds to unique **modal time constant**
 - Describes exponential decay/growth rate

$$\tau = \frac{1}{\text{Real}(\lambda)} \quad (40)$$

- $\text{Real}(\lambda)$ represents the real part of λ , i.e. if $\tau < 0$, then stable mode
- Modes with faster decay on output = modes with higher eigenvalues

Transfer Function Modal Analysis

- For complex-conjugate pairs, $\lambda = \frac{1}{\tau} \pm j\omega_d$, define **modal damped frequency**: ω_d
 - Related to **modal undamped natural frequency**: ω_n
 - **Modal damping ratio**, $0 < \zeta_i < 1$

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- Note: one can compare:

$$\left(\lambda - \frac{1}{\tau} + j\omega_d\right)\left(\lambda - \frac{1}{\tau} - j\omega_d\right) \quad (43)$$

$$\lambda^2 + 2\zeta\omega_n\lambda + \omega_n^2 \quad (44)$$

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- Course focus: MIMO, continuous-time dynamical systems
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