

3. System Response

Poles, Zeros, and System Response

First-Order Systems

Second-Order Systems

Poles, Zeros, and System Response

1. *Poles of a transfer function are:*
 - 1) The values of the Laplace transform variable, s , that cause the transfer function to become infinite.
 - 2) _____ of the transfer function.
 - 3) Poles are shown as \times 's in the complex s -plane.
 - 4) A pole of the input function generates the form of the _____ .
 - 5) A pole of the transfer function generates the form of the _____ or _____ .
2. *Zeros of a transfer function are:*
 - 1) The values of the Laplace transform variable, s , that cause the transfer function to become zero.
 - 2) Roots of the numerator of the transfer function.
 - 3) Zeros are shown as o 's in the complex s -plane.
 - 4) The zeros and poles generate the *amplitudes* for both the forced and natural responses.

Example 1: Poles and Zeros

- 1) Unit step response of the system $G(s)$ in (a) is:

$$C(s) = \frac{(s+2)}{s(s+5)} = \frac{A}{s} + \frac{B}{s+5}$$

$$= \frac{1}{s} + \frac{3}{s+5}$$

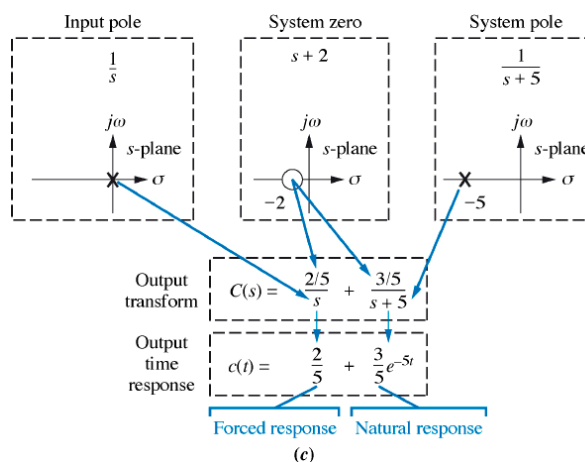
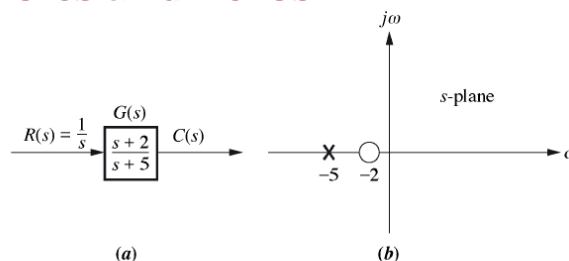
where

$$A = \frac{(s+2)}{(s+5)} \Big|_{s \rightarrow 0} =$$

$$B = \frac{(s+2)}{s} \Big|_{s \rightarrow -5} =$$

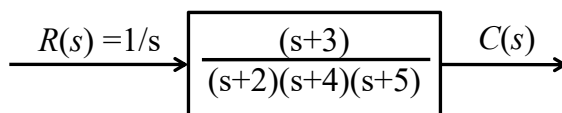
Thus,

$$c(t) =$$



Response vs. Poles

- Given the system, write the output and specify which terms are related to the input and which terms are from the system's own characteristics.



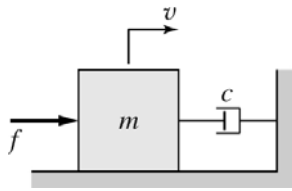
1. By inspection, each system pole generates an exponential as part of the natural response.

$$C(s) \equiv \underbrace{\frac{K_1}{s}} + \underbrace{\frac{K_2}{(s+2)} + \frac{K_3}{(s+4)} + \frac{K_4}{(s+5)}}_{\text{Natural response}}$$

2. Taking the inverse Laplace transform, we get

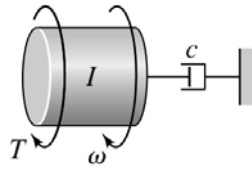
$$c(t) \equiv \underbrace{K_1}_{\text{Forced response}} + \underbrace{K_2 e^{-2t} + K_3 e^{-4t} + K_4 e^{-5t}}_{\text{Natural response}}$$

First-Order Systems



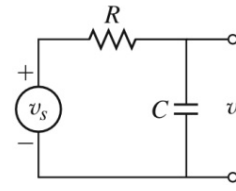
$$m \frac{dv}{dt} + cv = f$$

$$\tau = \frac{m}{c}$$



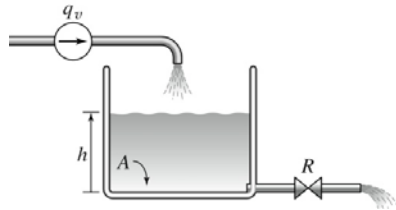
$$I \frac{d\omega}{dt} + c\omega = T$$

$$\tau = \frac{I}{c}$$



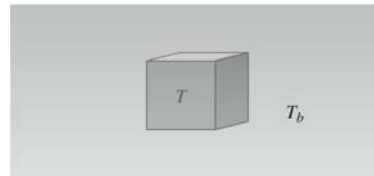
$$RC \frac{dv}{dt} + v = v_s$$

$$\tau = RC$$



$$AR \frac{dh}{dt} + gh = Rq_v$$

$$\tau = \frac{AR}{g}$$



$$mc_p R \frac{dT}{dt} + T = T_b$$

$$\tau = mc_p R$$

First-Order System (1/2)

1. Step response of first-order system

- 1) Consider the following first-order system.

$$\dot{x} + ax = af(t)$$

- 2) The characteristic equation is _____ and its root, _____ is called the system _____.
- 3) For a **step response**, a unit step function, $u(t)$ is applied to the first-order system. The resulting system equation becomes

$$\dot{x} + ax = au_s(t)$$

- 4) The solution of the equation with zero initial conditions is

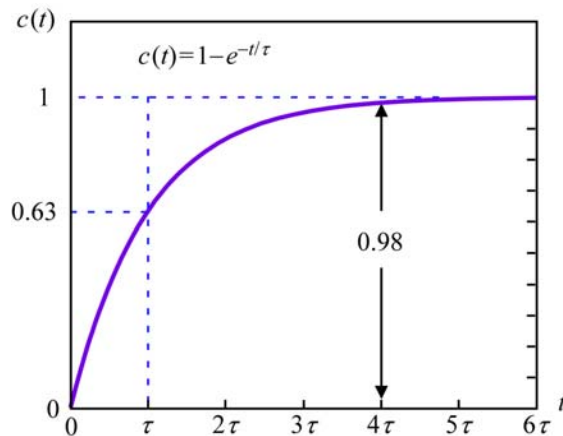
2. Time constant

- 1) The time constant, τ is defined as _____ (positive reciprocal of the system pole) and the step response can be rewritten as $x(t) = 1 - e^{-t/\tau}$.
- 2) Systems with **smaller** time constants respond _____ than those with larger time constants.

First-Order System (2/2)

2. Time constant (cont'd)

- 3) At $t = \tau$, $e^{-t/\tau}$ decays to 37% of its initial value and $x(t)$ rises to _____ of its final value.
- 4) At $t = 4\tau$, the response is _____ of the steady-state value, and at $t = 5\tau$, it is 99% of steady state.
- 5) For most engineering purposes, we regard that $x(t)$ reaches steady state at $t = 4\tau$.



Time (t)	$1 - e^{-t/\tau}$
$t = 0$	0
$t = \tau$	0.63
$t = 2\tau$	0.86
$t = 3\tau$	0.95
$t = 4\tau$	0.98
$t = 5\tau$	0.99

The University of Alabama, ME 475

7

Example: First-Order System

- A system model is found to be $\dot{x} + 50x = 50f$. Find the time constant and the time that the system takes to reach 50% and 98% of the steady-state value when a unit step input is applied.

1. Time constant, $\tau =$ _____.
2. When $t = 4\tau$, the system reaches 98% of the final value. Thus, after _____, the response rises to 98% of the final value.
3. In order to find the time that it reaches 50% of the final value, the following equation needs to be solved.

$$1 - e^{-50t} = 0.5 \Rightarrow e^{-50t} = 0.5 \Rightarrow \ln(e^{-50t}) = \ln(0.5)$$

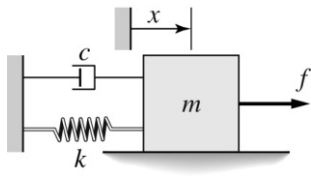
$$\Rightarrow t = \underline{\hspace{2cm}}$$

First order systems do not exhibit any oscillatory behavior!

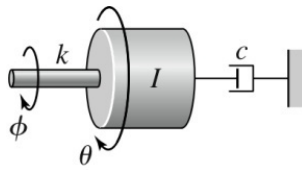
The University of Alabama, ME 475

8

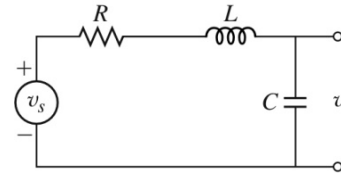
Second-Order Systems



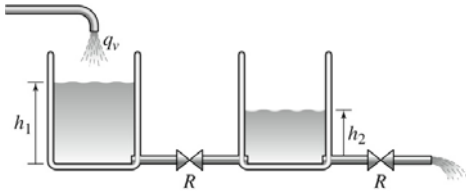
$$m \frac{d^2 x}{dt^2} + c \frac{dx}{dt} + kx = f$$



$$I \frac{d^2 \theta}{dt^2} + c \frac{d\theta}{dt} + k\theta = k\phi$$

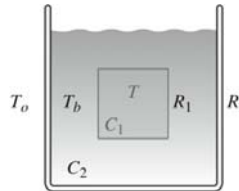


$$LC \frac{d^2 v}{dt^2} + RC \frac{dv}{dt} + v = v_s$$



$$RA \frac{dh_1}{dt} + g(h_1 - h_2) = Rq_v$$

$$RA \frac{dh_2}{dt} + g(h_2 - h_1) + gh_2 = 0$$



$$R_1 C_1 \frac{dT}{dt} + T = T_b$$

$$R_1 R_2 C_2 \frac{dT_b}{dt} + (R_1 + R_2) T_b = R_2 T + R_1 T_o$$

Second-Order System (1/3)

1. Step response of second-order system

- 1) Consider the following second-order system.

$$\ddot{x} + a\dot{x} + bx = bf(t)$$

- 2) The characteristic equation is $s^2 + as + b = 0$ and the system **poles** are _____.

- 2) For a step response, a unit step function, $u(t)$ is applied to the second-order system. The resulting system equation becomes

$$\ddot{x} + a\dot{x} + bx = bu_s(t)$$

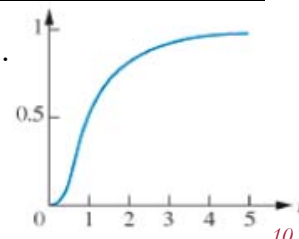
2. Overdamped case

- 1) If $a^2 - 4b > 0$, the char. eq. has two distinct real roots, $-\sigma_1$ and $-\sigma_2$.
- 2) The general step response is _____.

- 3) Two time constants exist: $\tau_1 = 1/\sigma_1$ and $\tau_2 = 1/\sigma_2$.

- 4) Example: If $a = b = 9$,

$$x(t) = 1 + 0.171e^{-7.854t} - 1.171e^{-1.146t}$$



Second-Order System (2/3)

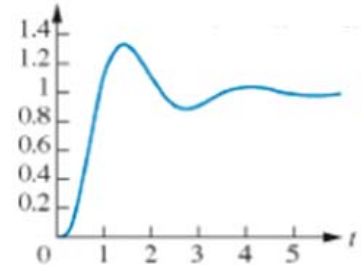
3. Underdamped case

- 1) If $a^2 - 4b < 0$, the char. eq. has two complex roots, $-\sigma_d \pm j\omega_d$.
- 2) The general step response is

- 3) Time constant: _____.

- 4) Example: If $a = 2$ and $b = 9$,

$$x(t) = 1 - e^{-t} \cos(\sqrt{8}t) - \frac{\sqrt{8}}{8} e^{-t} \sin(\sqrt{8}t)$$



- 5) Response characteristics: Damped sinusoid with an exponential envelope whose time constant is equal to the reciprocal of the pole's real part. The radian frequency of the sinusoid, the _____ of oscillation, is equal to the imaginary part of the poles.

Second-Order System (3/3)

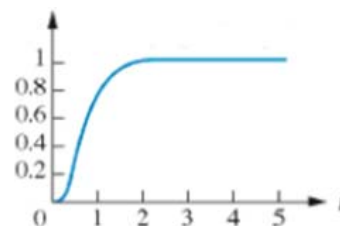
4. Critically-Damped case

- 1) If $a^2 - 4b = 0$, the char. eq. has _____ roots, $-\sigma_1$.
- 2) The general step response is _____.

- 3) Time constant: $\tau = 1/\sigma_1$.

- 4) Example: If $a = 6$ and $b = 9$,

$$x(t) = 1 - 3te^{-3t} - e^{-3t}$$

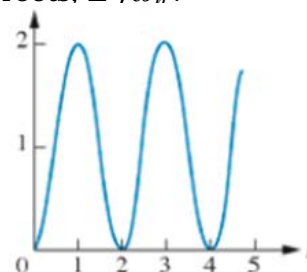


5. Undamped case

- 1) If $a = 0$, the char. eq. has two imaginary roots, $\pm j\omega_n$.
- 2) The general step response is

$$\begin{aligned} x(t) &= C_0 + C_1 \sin(\omega_n t) + C_2 \cos(\omega_n t) \\ &= C_0 + A \sin(\omega_n t + \phi) \end{aligned}$$

- 3) Example: If $a = 0$ and $b = 9$,
 $x(t) = 1 - \cos 3t$



Example (Overdamped case)

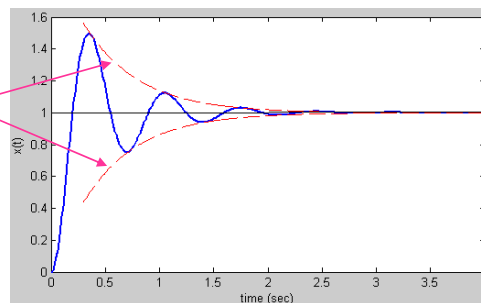
Identify the transient, steady-state, free, and forced responses of the following equation with $\dot{x}(0)$ and $x(0)$ as I.C.'s.

$$\ddot{x} + 7\dot{x} + 10x = c$$

Example (Underdamped Case)

Determine the response of the SDOF mass-spring-damper system when $m=1$ kg, $c=4$ Ns/m, and $k=85$ N/m with the initial conditions $x(0) = 0$ and $\dot{x}(0) = 0$, and the external input $f= 85$.

Vibration decays exponentially !



Example (Zero-Damping Case)

Determine the response of the SDOF mass-spring-damper system when $m=1$ kg, $c=0$, and $k=85$ N/m with the initial conditions $x(0)=0$ and $\dot{x}(0)=0$, and the external input $f=85$.

Solution) $\ddot{x} + 85x = 85$ $x(0)=0$ $\dot{x}(0)=0$

$$x_c = A \cos \sqrt{85}t + B \sin \sqrt{85}t \text{ and } x_p = 1.$$

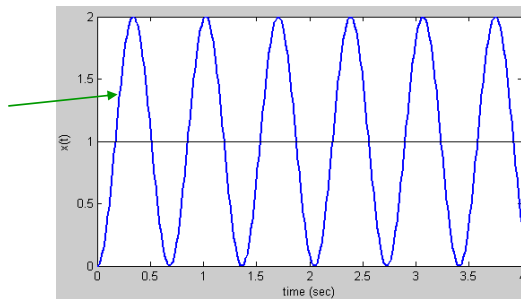
The general solution becomes $x(t) = A \cos \sqrt{85}t + B \sin \sqrt{85}t + 1$.

By applying the I.C.'s, the final solution is found to be

$$x(t) =$$

steady-state response

Without damping, no energy is dissipated and vibration continues forever !



The University of Alabama, ME 475

15

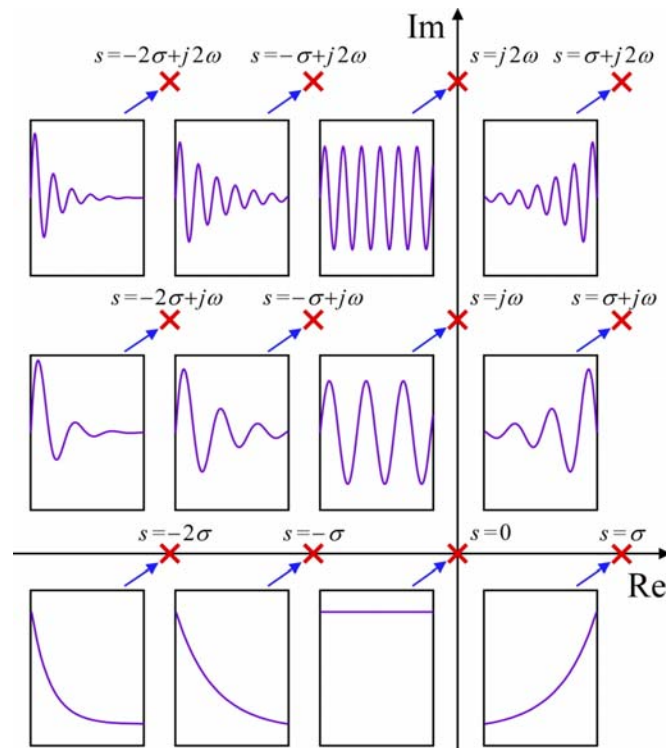
Effect of Pole Location

1. Location of system poles (characteristic roots)
 - The system behavior is dependent on the location of the roots of the characteristic equation. They are also called the **system poles**.
2. General tendency
 - 1) Unstable behavior occurs when any pole lies in the _____.
 - 2) Neutrally stable behavior occurs when a pole lies on the imaginary axis.
 - 3) Since the system parameters ζ , ω_n , ω_d , σ , and τ are normally for stable systems only, it is assumed here that all poles are located in the _____ (stable region) when those system parameters are defined.
 - 4) The response oscillates only when the pole has a nonzero _____.
 - 5) The greater the imaginary part, the _____ the frequency of the oscillation. (Imaginary part of system pole = Frequency of oscillation)
 - 6) The farther to the left the root lies in the S-plane, the _____ the response decays. (Real part of system pole = Decay rate of response)

The University of Alabama, ME 475

16

Effect of Pole Location (Free Response)



The University of Alabama, ME 475

17

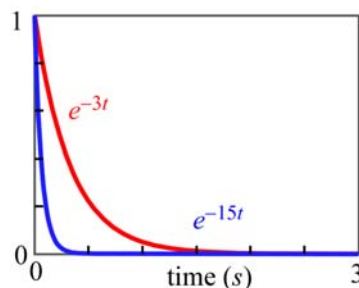
Dominant-Root Analysis

1. Dominant root

- 1) If the real parts of two different characteristic roots are $-\sigma_1$ and $-\sigma_2$, the response will have the exponentials $e^{-\sigma_1 t}$ and $e^{-\sigma_2 t}$ associated with each of these roots.
- 2) If both roots are negative, the system is stable and time constants can be defined such that $\tau_1 = 1/\sigma_1$ and $\tau_2 = 1/\sigma_2$.
- 3) Since a time constant τ is a measure of the decay rate of an exponential $e^{-t/\tau}$, the two exponentials will decay at different rates.
- 4) The exponential having the **largest time constant decays the** _____ and thus _____. Its corresponding root is thus called the dominant root, and its time constant is the dominant time constant.

2. Example

- 1) If the system poles of a second-order system are $s = -3, -15$, the associated time constants are _____ and _____. Thus, the second exponential will disappear five times faster than the first exponential.



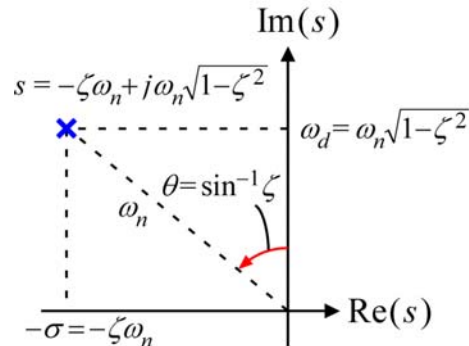
The University of Alabama, ME 475

18

Pole Location vs. System Parameters

□ System response vs. pole location

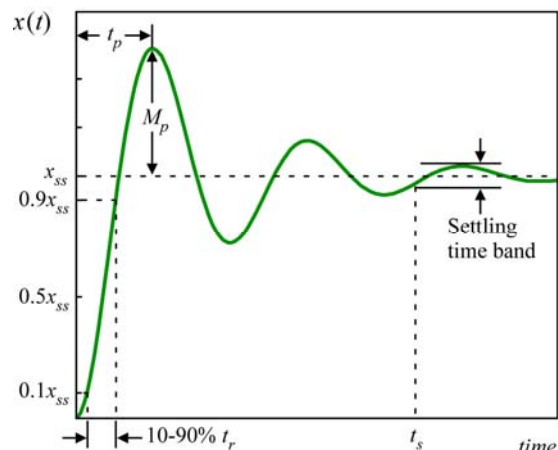
- 1) To visualize the connection between the pole location and the system behavior, the poles are plotted in the complex plane or S-plane.
- 2) The real part of the system pole is _____ and determines how fast the response decays.
- 3) The imaginary part of the system pole is _____ and is the damped natural frequency.
- 3) The distance between the origin and the pole is _____, the natural frequency.
- 4) The damping ratio, ζ is related to θ , the angle measured from the imaginary axis such that _____.



Time Domain Specifications

1. Definitions

- 1) Maximum or peak _____, M_p : the maximum deviation of the output above its steady-state value, x_{ss} . It is sometimes expressed as a percentage of the final value.
- 2) **Peak time, t_p** : the time at which the maximum overshoot occurs.
- 3) _____, t_s : the time required for the oscillations to stay within some specified small percentage (2% or 5%) of the final value.
- 4) _____, t_r : the time required for the output to rise from a certain low value (0% or 10%) to a certain high value (100% or 90%).

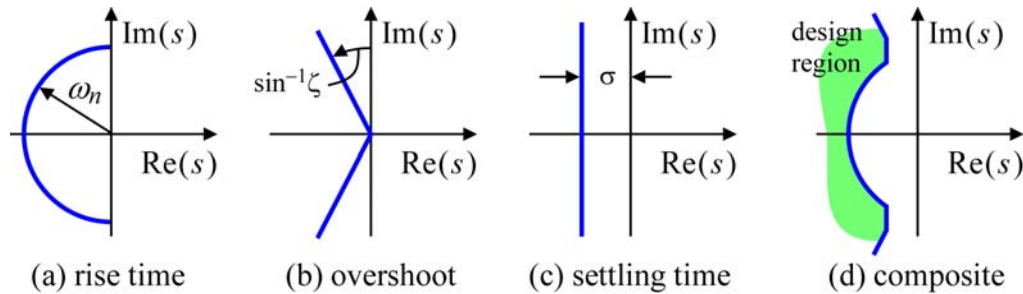


Pole Location vs. Time Domain Specifications

2. General design guideline

- 1) **Peak time, t_p :** $t_p = \frac{\pi}{\omega_d} = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}}$
- 2) **Overshoot, M_p (%) :** $M_p = 100 e^{-\pi\zeta/\sqrt{1-\zeta^2}}$
- 3) **Settling time (2%), t_s :** $t_s = \frac{4}{\sigma} = \frac{4}{\zeta\omega_n}$
- 4) **Rise time (10% – 90%), t_r :** $t_r \cong \frac{1.8}{\omega_n}$

3. Graphs of regions in s -plane for certain transient requirements



Example: Specifications to the S-Plane

- Find the allowable regions in the s -plane for the poles of a system if the system response requirements are $t_r \leq 0.6$ sec, $M_p \leq 10\%$, and $t_s \leq 3$ sec.

Solution)

$$M_p = 100 e^{-\pi\zeta/\sqrt{1-\zeta^2}} \Rightarrow 100 e^{-\pi\zeta/\sqrt{1-\zeta^2}} = 10 \Rightarrow \underline{\hspace{2cm}}$$

$$\Rightarrow \theta \geq \sin^{-1} \zeta = \underline{\hspace{2cm}}$$

$$t_r \cong \frac{1.8}{\omega_n} \Rightarrow \omega_n \geq \underline{\hspace{2cm}}$$

$$t_s = \frac{4}{\sigma} \Rightarrow \sigma \geq \underline{\hspace{2cm}}$$

