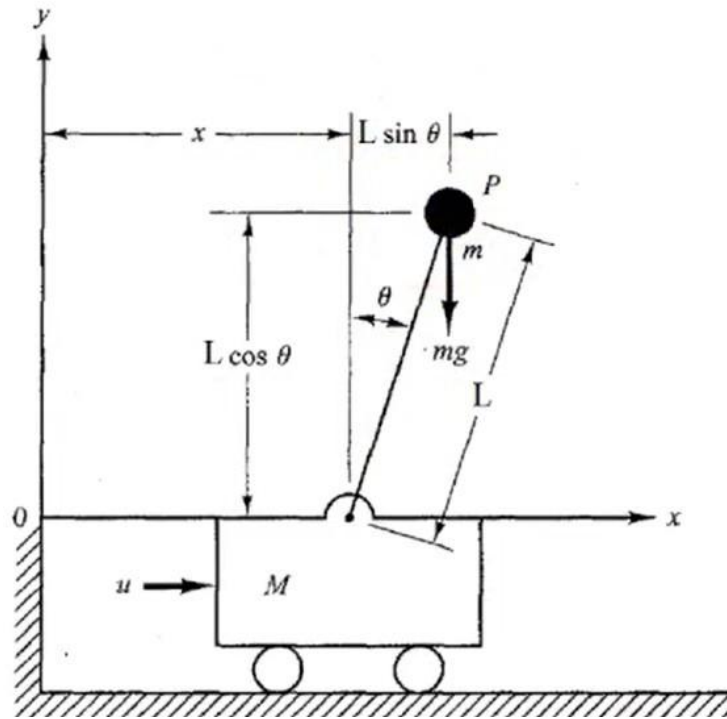


## Lagrangian Calculation:

$$L = T - V$$



### Part 1: Kinetic Energy(T)

#### 1. Pendulum:

$$T_p = \frac{1}{2}mv^2 + \frac{1}{2}I\dot{\theta}^2$$

And we have:

$$v = \frac{d}{dt}((x + L \sin(\theta))i + L \cos(\theta)j) = (i + L\dot{\theta} \cos(\theta))i + (-L\dot{\theta} \sin(\theta))j$$

$$\rightarrow v^2 = \dot{x}^2 + (L\dot{\theta})^2 + 2\dot{x}L\dot{\theta} \cos(\theta)$$

**2. Cart:** The cart has kinetic energy due to its horizontal motion

$$T_{cart} = \frac{1}{2} m_c \dot{x}^2$$

Thus, the total kinetic energy is equal to:

$$\begin{aligned} T &= T_{pendulum} + T_{cart} \\ &= \frac{1}{2} m_p \left[ (\dot{x} + L\dot{\theta} \cos \theta)^2 + (L\dot{\theta} \sin \theta)^2 \right] + \frac{1}{2} m_c \dot{x}^2 \end{aligned}$$

## Part 2: Potential Energy(V)

**1. Pendulum:** The pendulum has gravitational potential energy.

$$V_{pendulum} = m_p g L \cos \theta$$

**2. Cart:** the cart has no potential energy

$$V_{cart} = 0$$

Thus, the total potential energy is equal to:

$$V = V_{pendulum} + V_{cart} = m_p g L \cos \theta$$

Now the Lagrangian can be computed as follows:

$$\begin{aligned} \mathcal{L} &= T - V \\ &= \frac{1}{2} m_p \left[ \dot{x}^2 + (L\dot{\theta})^2 + 2\dot{x}L\dot{\theta} \cos(\theta) \right] + \frac{1}{2} I \dot{\theta}^2 + \frac{1}{2} m_c \dot{x}^2 - m_p g L \cos \theta \end{aligned}$$

As we know, the Euler Lagrange Equation is as follows:

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{q}_i} \right) - \frac{\partial \mathcal{L}}{\partial q_i} = Q_i$$

Where,  $Q_i$  generally represents external forces.

Here, since we have two degrees of freedom, we have to solve the Euler-Lagrange equations twice.

**1. For  $x$  (cart's horizontal position)**

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{x}} \right) - \frac{\partial \mathcal{L}}{\partial x} = F$$

**1. Computing  $\frac{\partial \mathcal{L}}{\partial \dot{x}}$ :**

$$\frac{\partial \mathcal{L}}{\partial \dot{x}} = m_c \dot{x} + m_p (\dot{x} + L \dot{\theta} \cos \theta)$$

**2. Computing  $\frac{\partial \mathcal{L}}{\partial x}$ :**

$$\frac{\partial \mathcal{L}}{\partial x} = 0$$

**3. Final Equation:**

$$(m_c + m_p) \ddot{x} + m_p L \ddot{\theta} \cos \theta - m_p L \dot{\theta}^2 \sin \theta = F$$

2. For  $\theta$ (pendulum angle)

$$\frac{d}{dt} \left( \frac{\partial \mathcal{L}}{\partial \dot{\theta}} \right) - \frac{\partial \mathcal{L}}{\partial \theta} = 0$$

1. Computing  $\frac{\partial \mathcal{L}}{\partial \dot{\theta}}$ :

$$\frac{\partial \mathcal{L}}{\partial \dot{\theta}} = m_p L^2 \ddot{\theta} + m_p L \dot{x} \cos \theta + I \dot{\theta}$$

2. Computing  $\frac{\partial \mathcal{L}}{\partial \theta}$ :

$$\frac{\partial \mathcal{L}}{\partial \theta} = -m_p L \dot{x} \dot{\theta} \sin \theta + m_p g L \sin \theta$$

3. Final Equation:

$$m_p L^2 \ddot{\theta} + m_p L \ddot{x} \cos \theta - m_p g L \sin \theta + I \ddot{\theta} + m_p L \dot{x} \dot{\theta} \sin \theta = 0$$

Finally, as the Euler-Lagrange model we have:

$$\begin{cases} m_p L^2 \ddot{\theta} + m_p L \ddot{x} \cos \theta - m_p g L \sin \theta + I \ddot{\theta} + m_p L \dot{x} \dot{\theta} \sin \theta = 0 \\ (m_c + m_p) \ddot{x} + m_p L \ddot{\theta} \cos \theta - m_p L \dot{\theta}^2 \sin \theta = F \end{cases}$$

Solving for  $\ddot{x}$  and  $\ddot{\theta}$  we get:

$$\begin{cases} \ddot{\theta} = f_1(\theta) + g_1(\theta)F \\ \ddot{x} = f_2(x) + g_2(x)F \end{cases} \begin{cases} f(\theta) = \frac{(m_c + m_p)g \sin \theta - m_p L \sin \theta \cos \theta \dot{\theta}^2}{(m_c + m_p \sin^2 \theta)L} \\ g(\theta) = \frac{-\cos \theta}{(m_c + m_p \sin^2 \theta)L} \\ f(x) = \frac{-m_p g \sin \theta \cos \theta + m_p L \sin \theta \dot{\theta}^2}{m_c + m_p \sin^2 \theta} \\ g(x) = \frac{1}{m_c + m_p \sin^2 \theta} \end{cases}$$

To turn it into the state space form, we define the state variables as:

$$x = [x \ \dot{x} \ \theta \ \dot{\theta}]^T$$

Having:

$$\begin{bmatrix} \dot{x} \\ \ddot{x} \\ \dot{\theta} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} x_2 \\ f_2(x_1) + g_2(x_1)x_4 \\ x_4 \\ f_1(x_3) + g_1(x_3) \end{bmatrix}$$

And as said in the exercise instructions, the model parameters are:

$$\begin{cases} m_c = 0.5 \text{ Kg} \\ m_p = 2 \text{ Kg} \\ L = 0.2 \text{ m} \\ g = 9.81 \frac{\text{m}}{\text{s}^2} \end{cases}$$

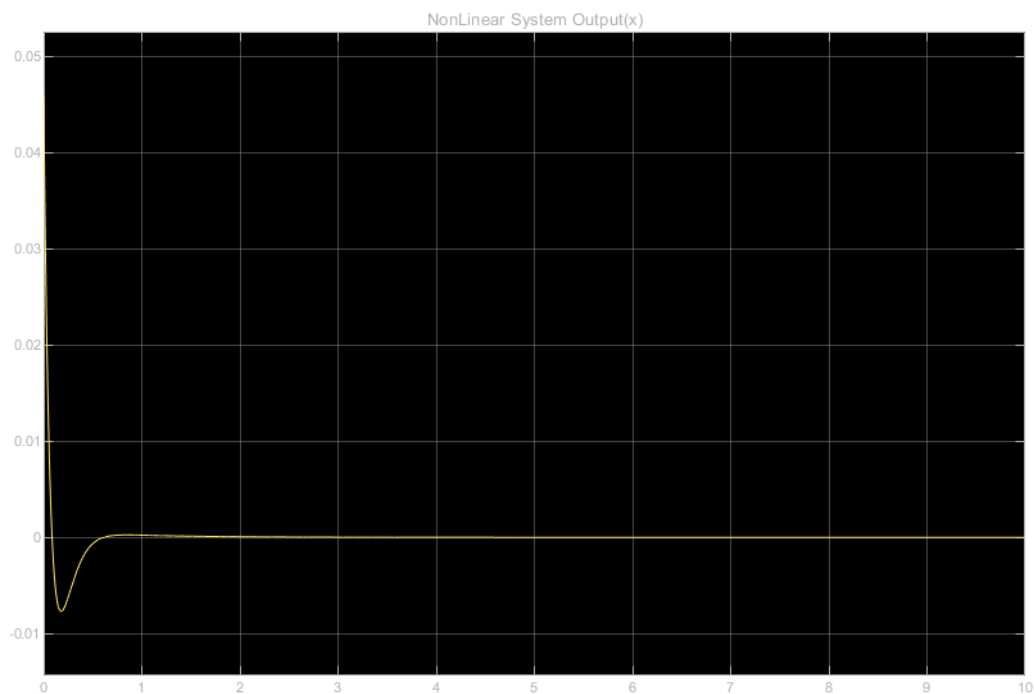
**After this, the system is linearized by calculating its Jacobian and substituting the equilibrium point (which is 90 degrees because the pendulum axis is assumed to be upwards).**

**Then a PID Controller is designed using MATLAB PIDTuner and applied as a regulator to keep the system at zero.**

**Everything is done in MATLAB using Symbolic Math Toolbox and the system is simulated in Simulink.**

**And the results are as follows:**

#### **Non-Linear System Model:**



## Linearized System:

