# Thermoelastic fracture problems using Extended Finite Element Method



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#### Outline

- Introduction
- Literature Survey
- · Objectives and work done
- Computer implementation
- Validation of FEM program
- Example problems
- Conclusion





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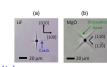
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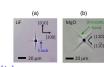
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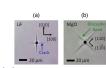
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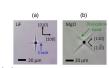
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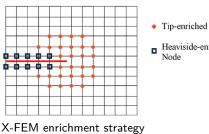
**source**: Tian, X., Shen. (2006). A direct finite element method study of generalized thermoelastic problems. *International Journal of Solids and Structures*, 43(7), 2050-2063.



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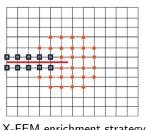
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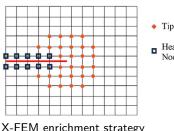
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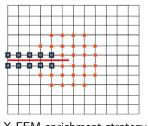
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  - No need of remeshing



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## Enrichment functions in X-FEM



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 The nodes which belongs to the elements totally cut by the crack, are enriched by and Heaviside function.

$$h(x,y) = \begin{cases} 1, & \text{for } y \ge 0 \\ -1, & \text{for } y \le 0 \end{cases}$$



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• The nodes of elements which contains cracktip are enriched by  $\gamma$ :

$$u^{h} = \sum_{i} N_{i}(x)u_{i} + \sum_{j \in J} N_{j}(x)h(x)a_{j} + \sum_{k \in K} N_{k}(x) \left(\sum_{l=1}^{4} \gamma_{l}(x)b_{kl}\right)$$

$$v^{h} = \sum_{i} N_{i}(x)v_{i} + \sum_{j \in J} N_{j}(x)h(x)c_{j} + \sum_{k \in K} N_{k}(x) \left(\sum_{l=1}^{4} \gamma_{l}(x)d_{kl}\right)$$

$$where, \quad \gamma = \left[\sqrt{r}\cos\left(\frac{\theta}{2}\right), \sqrt{r}\sin\left(\frac{\theta}{2}\right), \sqrt{r}\sin\left(\frac{\theta}{2}\right)\sin(\theta), \sqrt{r}\cos\left(\frac{\theta}{2}\right)\sin(\theta)\right]$$

Source:Belytschko, T., & Black, T. (1999). Elastic crack growth in finite elements with minimal remeshing. International journal for numerical methods in engineering, 45(5), 601-620. 5 of 22



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$$\varepsilon_{ij} = \varepsilon_{ij}^{(M)} + \varepsilon_{ij}^{(T)} = \frac{1+\nu}{E} \sigma_{ij} - \frac{\nu}{E} \sigma_{kk} \delta_{ij} + \alpha (T-T_0) \delta_{ij}$$



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• It can be inverted to get following stress-strain relationship:

$$\begin{cases} \sigma_{x} \\ \sigma_{y} \\ \tau_{xy} \end{cases} = \frac{E}{(1 - \nu^{2})} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & 1 - \nu \end{bmatrix} \begin{cases} \varepsilon_{x} - \alpha \Delta T \\ \varepsilon_{y} - \alpha \Delta T \\ \varepsilon_{xy} \end{cases}$$

# Governing Equations of Thermoelasticity



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• The governing equations of the thermo-elasticity is derived as:

$$\begin{split} \frac{\partial}{\partial x} \left[ c_{11} \frac{\partial u}{\partial x} + c_{12} \frac{\partial v}{\partial y} \right] + \frac{\partial}{\partial y} \left[ c_{66} \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \right] - (c_{11} + c_{12}) \alpha \frac{\partial T}{\partial x} - f_x &= 0 \\ \frac{\partial}{\partial x} \left[ c_{66} \left( \frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \right] + \frac{\partial}{\partial y} \left[ c_{12} \frac{\partial u}{\partial x} + c_{22} \frac{\partial v}{\partial y} \right] - (c_{11} + c_{12}) \alpha \frac{\partial T}{\partial y} - f_y &= 0 \\ k \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) &= q \end{split}$$



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• We can develop a week form of above equations by approximating u,v and T over a typical finite element  $\Omega^e$  as:

$$u(x,y) = \sum_{i=1}^{n} N_i(x,y)u_i \ , v(x,y) = \sum_{i=1}^{n} N_i(x,y)v_i \ , T(x,y) = \sum_{i=1}^{n} N_i(x,y)T_i$$



$$\begin{split} -\int_{\Omega} \left[ c_{11} \frac{\partial N_{i}}{\partial x} \frac{\partial N_{j}}{\partial x} u_{j} + c_{12} \frac{\partial N_{i}}{\partial x} \frac{\partial N_{j}}{\partial y} v_{j} \right] dx dy + \int_{\Omega} \left[ c_{66} \frac{\partial N_{i}}{\partial y} \frac{\partial N_{j}}{\partial y} u_{j} dx dy + \frac{\partial N_{i}}{\partial y} \frac{\partial N_{j}}{\partial x} v_{j} \right] dx dy \\ -\int_{\Omega} \frac{\partial N_{i}}{\partial x} \beta N_{j} T_{j} dx dy + \int_{\Omega} N_{i} f_{x} dx dy + \int_{\Gamma} N_{i} \vec{t} dx = 0 \end{split}$$



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$$k \int_{\Omega} \left( \frac{\partial N_i}{\partial x} \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial N_j}{\partial y} \right) T_j dx dy - \int_{\Omega} N_i N_j q dx dy = \int_{\Gamma} N_i \bar{Q} ds$$



• Neglecting the body forces, above equations can be written in matrix form as:

$$\begin{bmatrix} K_{11} & K_{12} \\ 0 & K_{22} \end{bmatrix} \begin{Bmatrix} U^e \\ T^e \end{Bmatrix} = \begin{Bmatrix} F \\ Q \end{Bmatrix}$$



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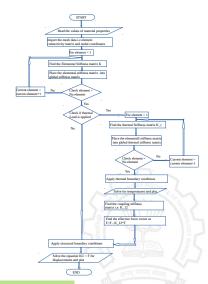
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$$[B] = \begin{bmatrix} \frac{\partial N_1}{\partial x} & 0 & \frac{\partial N_2}{\partial x} & 0 & \dots & \frac{\partial N_n}{\partial x} & 0 \\ 0 & \frac{\partial N_1}{\partial y} & 0 & \frac{\partial N_2}{\partial y} & \dots & 0 & \frac{\partial N_n}{\partial y} \\ \frac{\partial N_1}{\partial y} & \frac{\partial N_1}{\partial x} & \frac{\partial N_2}{\partial y} & \frac{\partial N_2}{\partial x} & \dots & \frac{\partial N_n}{\partial y} & \frac{\partial N_n}{\partial x} \end{bmatrix}, \ [B^{\theta}] = \begin{bmatrix} \frac{\partial N_1}{\partial y} & \frac{\partial N_2}{\partial y} & \dots & \frac{\partial N_n}{\partial y} \\ \frac{\partial N_1}{\partial y} & \frac{\partial N_2}{\partial y} & \dots & \frac{\partial N_n}{\partial y} \end{bmatrix}$$

$$N = \begin{bmatrix} N_1 & 0 & N_2 & 0 & \dots & N_n & 0 \\ 0 & N_1 & 0 & N_2 & \dots & 0 & N_n \end{bmatrix}, \ N^{\theta} = [N_1 & \dots & N_n]$$

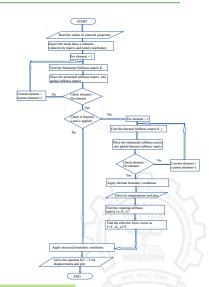


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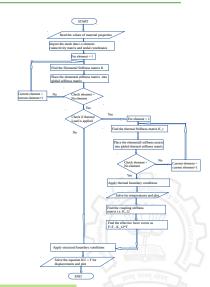
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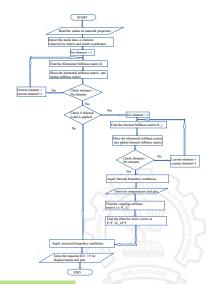


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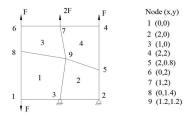
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A flow-chart showing the steps of the programming is shown in the figure.





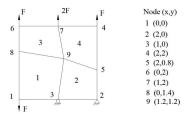
• A square plate is taken and meshed with 4 elements as shown in figure below.

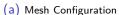






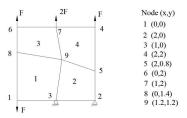
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- Minimum number of essential boundary conditions is fixed to eliminate the rigid body motions







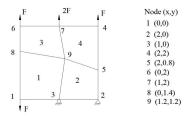
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(a) Mesh Configuration



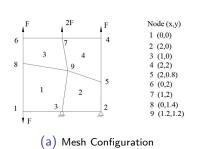
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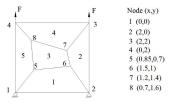


#### (b) Results of Patch Test 1

	Gauss Points	$\sigma_x/2F$	$\sigma_y/2F$	$\tau_{xy}/2F$
Element 1	1	$-0.222 \times 10^{-15}$	1	$0.1665 \times 10^{-15}$
	2	$0.2498 \times 10^{-15}$	1	$0.1665 \times 10^{-15}$
	3	$0.3058 \times 10^{-15}$	1	$0.222 \times 10^{-15}$
	4	0	1	0
Element 2	1	$-0.222 \times 10^{-15}$	1	0
	2	$41633 \times 10^{-15}$	1	$0.111 \times 10^{-15}$
	3	$0.02775 \times 10^{-15}$	1	$0.138 \times 10^{-15}$
	4	$-0.1110 \times 10^{-15}$	-1	0
	1	$0.222 \times 10^{-15}$	F 1 3	$0.1665 \times 10^{-15}$
Element 3	2	$0.0555 \times 10^{-15}$	1	$0.222 \times 10^{-15}$
	3	$-0.0555 \times 10^{-15}$	1	0
	4	$0.22204 \times 10^{-15}$	1	0
Element 4	1	$-0.222 \times 10^{-15}$	1	$0.1665 \times 10^{-15}$
	2	$0.222 \times 10^{-15}$	1	$0.1665 \times 10^{-15}$
	3	$0.3058 \times 10^{-15}$	1 6	$0.222 \times 10^{-15}$
	4	$0.222 \times 10^{-15}$	1	0



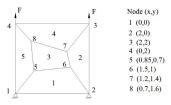
 Another patch test is performed on a plate with 5 element mesh and solved and results are tabulated as shown below







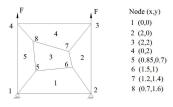
- Another patch test is performed on a plate with 5 element mesh and solved and results are tabulated as shown below
- The  $2 \times 2$  quadrature rule is used for numerical integration



(a) Mesh Configuration



- Another patch test is performed on a plate with 5 element mesh and solved and results are tabulated as shown below
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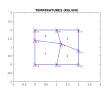
(a) Mesh Configuration

#### (b) Results of Patch Test 2

	Gauss Point	$\sigma_x/F$	$\sigma_y/F$	$\tau_{xy}/F$
Element 1	1	$0.166 \times 10^{-15}$	1	$0.1665 \times 10^{-15}$
	2	$-0.033 \times 10^{-15}$	1	$0.1665 \times 10^{-15}$
	3	$-0.063 \times 10^{-15}$	1	$0.222 \times 10^{-15}$
	4	0	1	0
Element 2	1	$0.062 \times 10^{-15}$	1	0
	2	$41633 \times 10^{-15}$	1	$-0.222 \times 10^{-15}$
	3	$0.02775 \times 10^{-15}$	1	$0.138 \times 10^{-15}$
	4	$-0.2220 \times 10^{-15}$	1	0
Element 3	1	$0.222 \times 10^{-15}$	1	$0.1665 \times 10^{-15}$
	2	$0.0555 \times 10^{-15}$	1	$0.222 \times 10^{-15}$
	3	$-0.0555 \times 10^{-15}$	1	White Ought Of
	4	$0.22204 \times 10^{-15}$	1	0
Element 4	1	$-0.222 \times 10^{-15}$	/1/	$0.1665 \times 10^{-15}$
	2	$0.222 \times 10^{-15}$	- 1	$0.1665 \times 10^{-15}$
	3	$-0.063 \times 10^{-15}$	1	$0.222 \times 10^{-15}$
	4	$0.222 \times 10^{-15}$	1	0
Element 5	1	$-0.422 \times 10^{-15}$	1	$0.1665 \times 10^{-15}$
	2	$0.222 \times 10^{-15}$	1	$0.1665 \times 10^{-15}$
	3	$-0.063 \times 10^{-15}$	1	$0.222 \times 10^{-15}$
	4	$0.222 \times 10^{-15}$	1	0



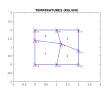
• Another patch was performed to validate the thermal code.



(a) Temperature distribution



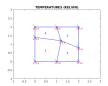
- Another patch was performed to validate the thermal code.
- Same problem as patch test was taken



(a) Temperature distribution



- Another patch was performed to validate the thermal code.
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- Temperature loads are applied to get the constant heat flux throughout the body



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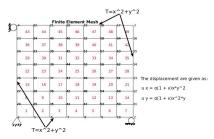
(b) Results of Patch test 3

TEMPERATURES (KELVIN)					
2.5					
2	Are Are				
1.5	4 )				
1	a)re				
0.5	2 206				
0	271x 271x 271x				
-0.5	-				
-1	-0.5 0 0.5 1 1.5 2 2.5 3				

(a)	Temperature distribution

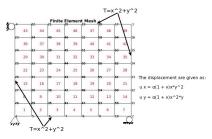
Nodes	Temperatures	$Q_y(W)$	$Q_{x}(W)$
1	273	200	$0.125 \times 10^{-10}$
2	273	200	$0.155 \times 10^{-10}$
3	273	200	$0.222 \times 10^{-10}$
4	281	200	0
5	276	200	0
6	281	200	INDIAN OSTITUTA
7	281	200	$0.111 \times 10^{-10}$
8	279	200	$0.138 \times 10^{-10}$
9	278	200	0



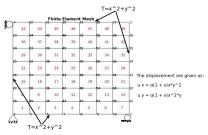


• A square plate of dimension  $2 \times 2$  is taken and meshed as shown





- $\bullet$  A square plate of dimension  $2\times 2$  is taken and meshed as shown
- Temperature distribution of  $T = x^2 + y^2$  is applied on 4 boundaries of plate

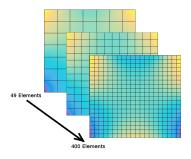


- A square plate of dimension  $2 \times 2$  is taken and meshed as shown
- Temperature distribution of  $T = x^2 + y^2$  is applied on 4 boundaries of plate
- a heat source  $q=rac{\partial^2 T}{\partial x^2}+rac{\partial^2 T}{\partial y^2}=$  4 is applied throughout the body
- Mesh is refined and to get the more accurate results

# Mesh Refinements

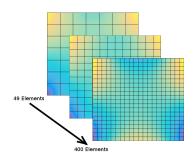


# Mesh Refinements





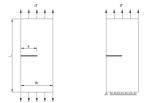
### Mesh Refinements



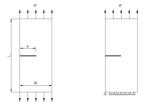
#### Errors obtained after mesh refinements

Number of Elements	Error in u <sub>x</sub>	Error in u <sub>y</sub>	Error in $\sigma_x$	Error in $\sigma_y$	Error in $ au_{xy}$
7 × 7	$5.783469 \times 10^{-10}$	$5.783469 \times 10^{-10}$	$6.668255 \times 10^{2}$	$6.668255 \times 10^{2}$	$3.615391 \times 10^{2}$
12 × 12	$1.764697 \times 10^{-11}$	$1.764697 \times 10^{-11}$	1.000597121	1.000597121	0.031722
20 × 20	$1.265804 \times 10^{-12}$	$1.265804 \times 10^{-12}$	0.994299467	0.994299467	0.005781

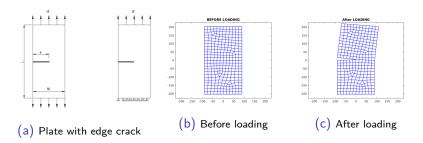




- (a) Plate with edge crack
- A plate with edge crack is meshed in ANSYS and imported to MATLAB



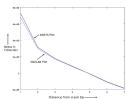
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- Boundary conditions are applied and stresses ahead of cracktip is plotted



- A plate with edge crack is meshed in ANSYS and imported to MATLAB
- Boundary conditions are applied and stresses ahead of cracktip is plotted
- The results obtained by MATLAB code is compared with the ANSYS solution



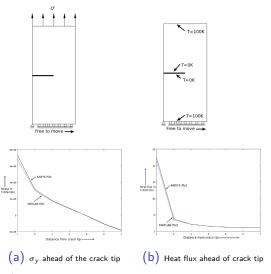




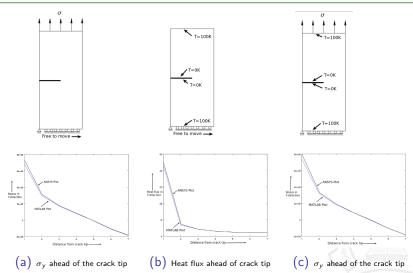
(a)  $\sigma_v$  ahead of the crack tip





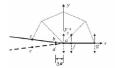




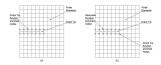


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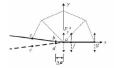


(a) Crack closure technique

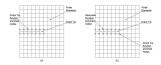


(b) X-FEM enrichment of nodes



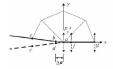


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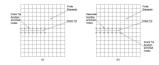


(b) X-FEM enrichment of nodes

Crack closure integral: 
$$G_I = \frac{W}{B\Delta a} = \frac{F_y^a u_y^b}{2B\Delta a}$$
 Thus,  $K_I = \sqrt{\frac{G_I}{E}} = 5.24586910 \times 10^9$  Pa $\sqrt{m}$  Analytical:  $K_I = C * \sigma \sqrt{\pi a} = 4.726065 \times 10^9$  Pa $\sqrt{m}$ 



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Method used	% Error = $\{K_{theoretical} - K_{numerical}/K_{theoretical}\} \times 100$			
Finite Element Method	11.1%		THE	
Extended Finite Element Method	4%			



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- Q8 elements will be applied in our program to solve curved boundary problems.
- As the hydrogen diffusion problems governed by Laplace's equation same as thermoelastic problems, we will modify the X-FEM program to solve these problems.

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