Thermoelastic fracture problems using Extended Finite Element Method

By
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Outline

- Introduction
- Motivation
- Literature Survey
- Objectives
- Computer implementation
- Example problems
- Conclusion and future work

Introduction: Thermo-elastic Fracture Mechanics

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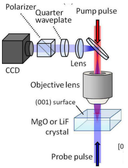

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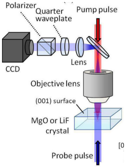

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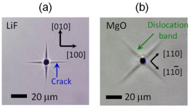

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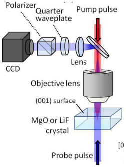

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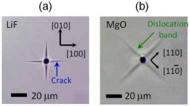

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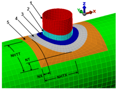

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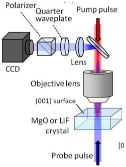

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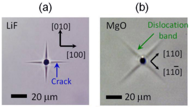

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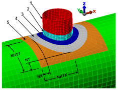

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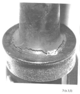

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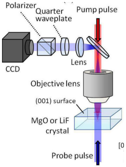

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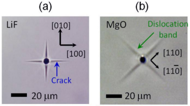

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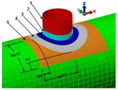

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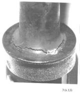

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source: Tian, X., Shen. (2006). A direct finite element method study of generalized thermoelastic problems. *International Journal of Solids and Structures*, 43(7), 2050-2063.

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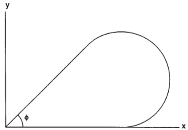

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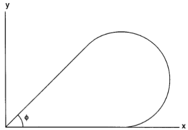

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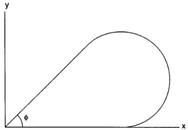

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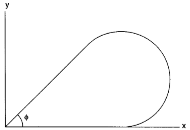

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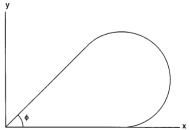

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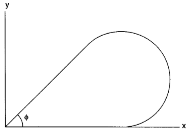

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- Thus, around the crack tip, thermal stresses also has square root singularity : our motivation for analyzing a thermo-elastic crack problem.

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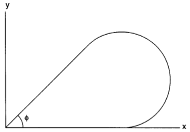

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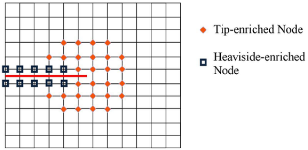

X-FEM enrichment strategy

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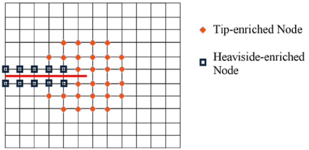

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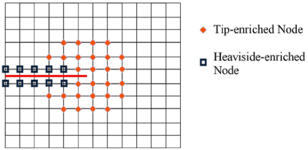

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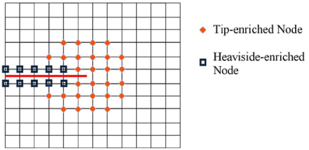

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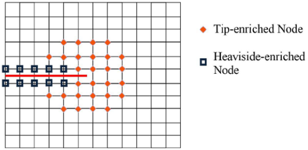

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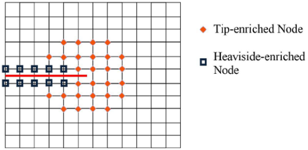

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Source: Belytschko, T., & Black, T. (1999). Elastic crack growth in finite elements with minimal remeshing. *International journal for numerical methods in engineering*, 45(5), 601-620.

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- In presence of temperature field, the Hooke's law can be given by:

$$\begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} = \frac{E}{(1-\nu^2)} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & 1-\nu \end{bmatrix} \begin{Bmatrix} \varepsilon_x - \alpha \Delta T \\ \varepsilon_y - \alpha \Delta T \\ \varepsilon_{xy} \end{Bmatrix}$$

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- The governing equations of the thermoelasticity are derived as follows:

$$\frac{\partial}{\partial x}\left[c_{11}\frac{\partial u}{\partial x} + c_{12}\frac{\partial v}{\partial y}\right] + \frac{\partial}{\partial y}\left[c_{66}\left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}\right)\right] - (c_{11}+c_{12})\alpha\frac{\partial T}{\partial x} - f_x = 0$$

$$\frac{\partial}{\partial x}\left[c_{66}\left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x}\right)\right] + \frac{\partial}{\partial y}\left[c_{12}\frac{\partial u}{\partial x} + c_{22}\frac{\partial v}{\partial y}\right] - (c_{11}+c_{12})\alpha\frac{\partial T}{\partial y} - f_y = 0$$

$$k\left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2}\right) = q$$

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$$-\int_{\Omega}\left[c_{11}\frac{\partial N_i}{\partial x}\frac{\partial N_j}{\partial x}u_j + c_{12}\frac{\partial N_i}{\partial x}\frac{\partial N_j}{\partial y}v_j\right]dxdy + \int_{\Omega}\left[c_{66}\frac{\partial N_i}{\partial y}\frac{\partial N_j}{\partial y}u_j dxdy + \frac{\partial N_i}{\partial y}\frac{\partial N_j}{\partial x}v_j\right]dxdy$$

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$$k\int_\Omega \left(\frac{\partial N_i}{\partial x}\frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y}\frac{\partial N_j}{\partial y}\right)T_j dxdy - \int_\Omega N_i N_j q dxdy = \int_\Gamma N_i \bar{Q} ds$$

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- Neglecting the body forces, above equations can be written in matrix form as:

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Where,

$$[K_{11}^e] = \int_\Omega [B]^T [C][B] dxdy \quad [K_{12}^e] = \int_\Omega [B]^T [\beta][N^\theta] dxdy \quad [K_{22}^e] = \int_\Omega [B^\theta]^T [K][B^\theta] dxdy$$

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Finite Element Model

- Neglecting the body forces, above equations can be written in matrix form as:

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Patch Test 1: Validation of Elasticity code

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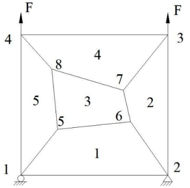
- A square plate is taken and meshed with 5 elements as shown in the figure below.



Node	(x,y)
1	(0,0)
2	(2,0)
3	(2,2)
4	(0,2)
5	(0.85,0.7)
6	(1.5,1)
7	(1.2,1.4)
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(a) Mesh Configuration

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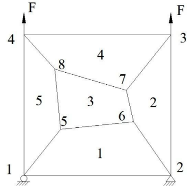
- Minimum number of essential boundary conditions is fixed to eliminate the rigid body motions



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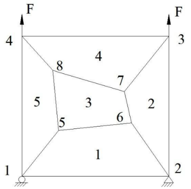
- Loads are applied such that there is constant state of stress in the body



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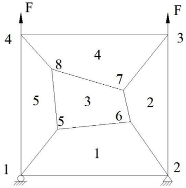

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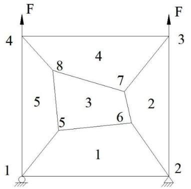

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(a) Mesh Configuration

(b) Results of Patch Test

	Gauss Point	σ_x/F	σ_y/F	τ_{xy}/F
Element 1	1	0.166×10^{-15}	1	0.1665×10^{-15}
	2	-0.033×10^{-15}	1	0.1665×10^{-15}
	3	-0.063×10^{-15}	1	0.222×10^{-15}
	4	0	1	0
Element 2	1	0.062×10^{-15}	1	0
	2	$-.41633 \times 10^{-15}$	1	-0.222×10^{-15}
	3	0.02775×10^{-15}	1	0.138×10^{-15}
	4	-0.2220×10^{-15}	1	0
Element 3	1	0.222×10^{-15}	1	0.1665×10^{-15}
	2	0.0555×10^{-15}	1	0.222×10^{-15}
	3	-0.0555×10^{-15}	1	0
	4	0.22204×10^{-15}	1	0
Element 4	1	-0.222×10^{-15}	1	0.1665×10^{-15}
	2	0.222×10^{-15}	1	0.1665×10^{-15}
	3	-0.063×10^{-15}	1	0.222×10^{-15}
	4	0.222×10^{-15}	1	0
Element 5	1	-0.422×10^{-15}	1	0.1665×10^{-15}
	2	0.222×10^{-15}	1	0.1665×10^{-15}
	3	-0.063×10^{-15}	1	0.222×10^{-15}
	4	0.222×10^{-15}	1	0

Patch Test 2: Validation of Thermoelasticity code

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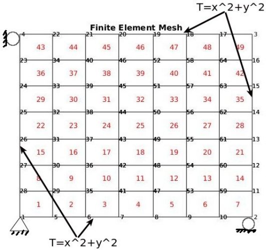
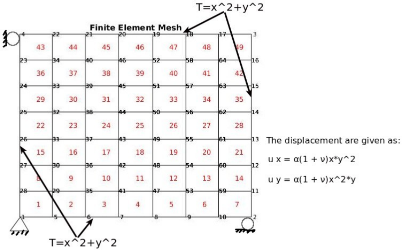


The displacement are given as:

u x = α(1 + ν)x*y^2

u y = α(1 + ν)x^2*y

Patch Test 2: Validation of Thermoelasticity code



- A temperature distribution of $T = x^2 + y^2$ is applied on 4 boundaries of plate which satisfies the Poisson's equation $\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} = 4$.

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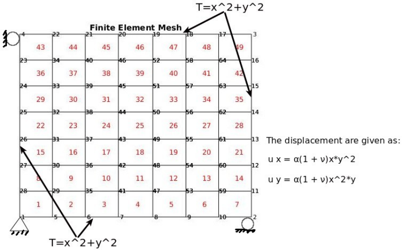

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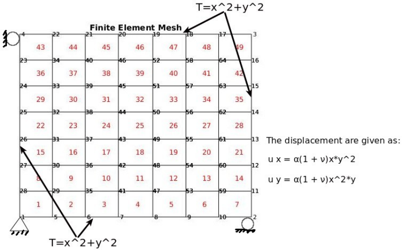

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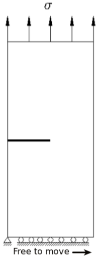
- Heat source $q = 4$ is applied throughout the body.
- Results are compared with the analytical solutions in the following table:

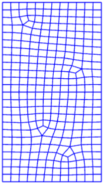
Nodes	% Error in Temperature T	% Error in displacement u_y
30	$-0.0523342237 \times 10^{-12}$	0.00034894013
33	$0.1046684475 \times 10^{-12}$	0.00082764871
63	$0.0916486147 \times 10^{-12}$	0.00015321553

Plate with edge crack

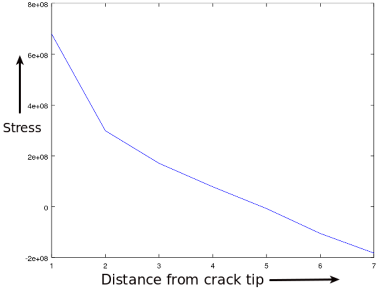
Plate with edge crack



(a) Edge crack

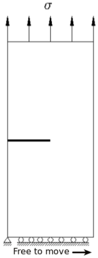


(b) Mesh configuration



(c) σ_y ahead of the crack tip

Plate with edge crack



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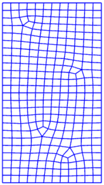

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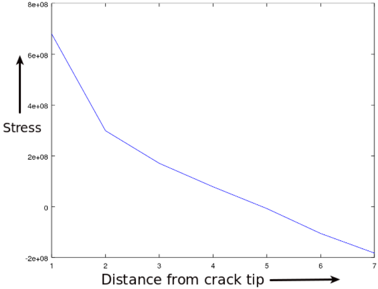
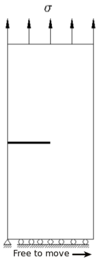


(c) σ_y ahead of the crack tip

- An edge crack problem is solved applying stress $\sigma_y = 100$ Mpa and taking E=200 Gpa and ν=0.3.

Plate with edge crack



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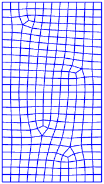

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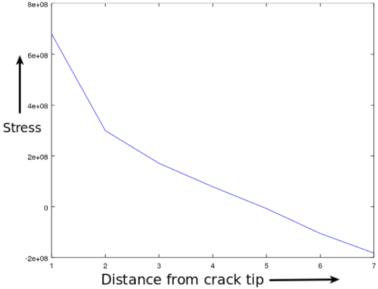

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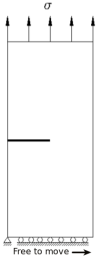

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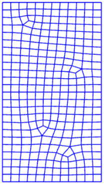

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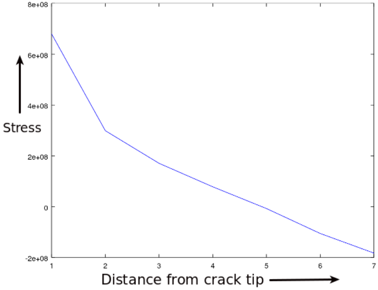

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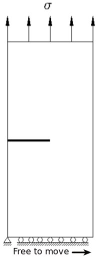

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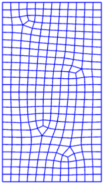

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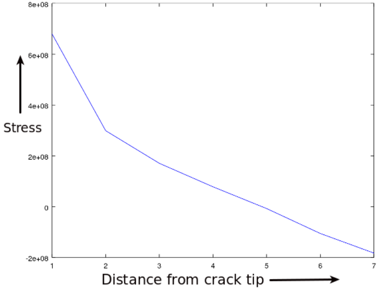

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source: Koustubh Parnaik. (2013). Implementation of Extended Finite Element Method (X-FEM) for static fracture problems. *Department of Mechanical Engineering, IIT Bombay.*

Comparison of FEM and X-FEM methods

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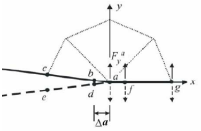

(a) Crack closure technique

Crack closure integral: $$G_I = \frac{W}{B\Delta a} = \frac{F_y^a u_y^b}{2B\Delta a}$$

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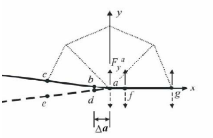

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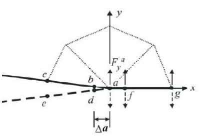

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Analytical: $K_I = C * \sigma\sqrt{\pi a} = 4.726065 \times 10^9 \;\; Pa\sqrt{m}$

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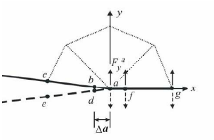

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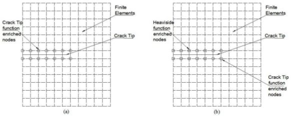

(b) X-FEM enrichment of nodes

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Method used	$\%\ Error = \{K_{theoretical} - K_{numerical}/K_{theoretical}\} \times 100$
Finite Element Method	11.1%
Extended Finite Element Method	4%

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  - Utilization of the X-FEM program in other fields which is governed by the Laplace's equation.

Thank You!