Thermoelastic fracture problems using Extended Finite Element Method



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Outline

- Introduction
- Literature Survey
- · Objectives and work done
- Computer implementation
- Validation of FEM program
- Example problems
- Conclusion



Introduction: Thermo-elastic Fracture Mechanics

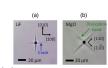
- Thermo-elastic Fracture mechanics: Study of propagation of crack in presence of temperature field.
- Atkinson's solution of Laplace's equation: $u(x,y) = r^{\frac{\alpha}{\phi}} \sin \alpha \theta$
- For $\phi = 2\pi$: $u \propto r^{\frac{1}{2}}$ and $u' \propto r^{-\frac{1}{2}}$
- Application areas:



(a) Ultra fast laser



(C) The cylinder-nozzle intersection.



(b) Cracks generated in material

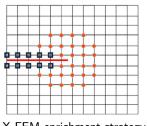


(d) Cracked baffle bolt of Belgian Nuclear Reactor

source: Tian, X., Shen. (2006). A direct finite element method study of generalized thermoelastic problems. *International Journal of Solids and Structures*, 43(7), 2050-2063.

Extended Finite Element Method in Thermoelasticity

- In FEM we have to use very fine mesh to capture the behaviour of crack.
- In X-FEM, we enrich the polynomial approximation to include the effects of singular discontinuous field.
- Advantages over FEM:
 - Accurate solutions without the very fine mesh at the discontinuity
 - No need of remeshing



· Tip-enriched Node

 Heaviside-enriched Node



Enrichment functions in X-FEM

 The nodes which belongs to the elements totally cut by the crack, are enriched by and Heaviside function.

$$h(x,y) = \begin{cases} 1, & \text{for } y \ge 0 \\ -1, & \text{for } y \le 0 \end{cases}$$

• The nodes of elements which contains cracktip are enriched by γ :

$$\begin{split} u^h &= \sum_i N_i(x) u_i + \sum_{j \in J} N_j(x) h(x) a_j + \sum_{k \in K} N_k(x) \left(\sum_{l=1}^4 \gamma_l(x) b_{kl} \right) \\ v^h &= \sum_i N_i(x) v_i + \sum_{j \in J} N_j(x) h(x) c_j + \sum_{k \in K} N_k(x) \left(\sum_{l=1}^4 \gamma_l(x) d_{kl} \right) \\ where, \quad \gamma &= \left[\sqrt{r} \cos \left(\frac{\theta}{2} \right), \sqrt{r} \sin \left(\frac{\theta}{2} \right), \sqrt{r} \sin \left(\frac{\theta}{2} \right) \sin(\theta), \sqrt{r} \cos \left(\frac{\theta}{2} \right) \sin(\theta) \right] \end{split}$$

Source:Belytschko, T., & Black, T. (1999). Elastic crack growth in finite elements with minimal remeshing. International journal for numerical methods in engineering, 45(5), 601-620.

Objectives and work Done

Objectives:

- To develop an Extended Finite Element Program for coupled thermoelastic fracture problems
- To utilize the developed X-FEM Program in other fields which is governed by Laplace's equation i.e hydrogen diffusion

Work Done:

- Finite Element Formulation of coupled thermoelasticity
- Development of a MATLAB program for semi-coupled thermoelastic problems.
- Patch tests to validate the developed FEM program.
- Solution of various thermoelastic fracture problems and comparison with the analytical solutions.
- Comparison between the results of FEM and X-FEM programs.

Finite Element Formulation of Thermo-elasticity

- We have formulated the semi-coupled in which we neglected the effect of displacements on temperature field.
- In thermoelastic case the total strain is given as:

$$\varepsilon_{ij} = \varepsilon_{ij}^{(M)} + \varepsilon_{ij}^{(T)} = \frac{1+\nu}{E} \sigma_{ij} - \frac{\nu}{E} \sigma_{kk} \delta_{ij} + \alpha (T-T_0) \delta_{ij}$$

It can be inverted to get following stress-strain relationship:

$$\begin{cases} \sigma_{x} \\ \sigma_{y} \\ \tau_{xy} \end{cases} = \frac{E}{(1 - \nu^{2})} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & 1 - \nu \end{bmatrix} \begin{cases} \varepsilon_{x} - \alpha \Delta T \\ \varepsilon_{y} - \alpha \Delta T \\ \varepsilon_{xy} \end{cases}$$

Governing Equations of Thermoelasticity

• The governing equations of the thermo-elasticity is derived as:

$$\begin{split} \frac{\partial}{\partial x} \left[c_{11} \frac{\partial u}{\partial x} + c_{12} \frac{\partial v}{\partial y} \right] + \frac{\partial}{\partial y} \left[c_{66} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \right] - (c_{11} + c_{12}) \alpha \frac{\partial T}{\partial x} - f_x &= 0 \\ \frac{\partial}{\partial x} \left[c_{66} \left(\frac{\partial u}{\partial y} + \frac{\partial v}{\partial x} \right) \right] + \frac{\partial}{\partial y} \left[c_{12} \frac{\partial u}{\partial x} + c_{22} \frac{\partial v}{\partial y} \right] - (c_{11} + c_{12}) \alpha \frac{\partial T}{\partial y} - f_y &= 0 \\ k \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} \right) &= q \end{split}$$

• We can develop a week form of above equations by approximating u,v and T over a typical finite element Ω^e as:

$$u(x,y) = \sum_{i=1}^{n} N_i(x,y)u_i \ , v(x,y) = \sum_{i=1}^{n} N_i(x,y)v_i \ , T(x,y) = \sum_{i=1}^{n} N_i(x,y)T_i$$

Weak Form Equations of Coupled Thermoelasticity

$$\begin{split} -\int_{\Omega} \left[c_{11} \frac{\partial N_{i}}{\partial x} \frac{\partial N_{j}}{\partial x} u_{j} + c_{12} \frac{\partial N_{i}}{\partial x} \frac{\partial N_{j}}{\partial y} v_{j} \right] dx dy + \int_{\Omega} \left[c_{66} \frac{\partial N_{i}}{\partial y} \frac{\partial N_{j}}{\partial y} u_{j} dx dy + \frac{\partial N_{i}}{\partial y} \frac{\partial N_{j}}{\partial x} v_{j} \right] dx dy \\ -\int_{\Omega} \frac{\partial N_{i}}{\partial x} \beta N_{j} T_{j} dx dy + \int_{\Omega} N_{i} f_{x} dx dy + \int_{\Gamma} N_{i} \vec{t} dx = 0 \end{split}$$

$$\begin{split} -\int_{\Omega} \left[c_{11} \frac{\partial N_{i}}{\partial y} \frac{\partial N_{j}}{\partial y} v_{j} + c_{12} \frac{\partial N_{i}}{\partial y} \frac{\partial N_{j}}{\partial x} u_{j} \right] dx dy + \int_{\Omega} \left[c_{66} \frac{\partial N_{i}}{\partial x} \frac{\partial N_{j}}{\partial x} v_{j} dx dy + \frac{\partial N_{i}}{\partial x} \frac{\partial N_{j}}{\partial y} u_{j} \right] dx dy \\ - \int_{\Omega} \frac{\partial N_{i}}{\partial y} \beta N_{j} T_{j} dx dy + \int_{\Omega} N_{i} f_{y} dx dy + \int_{\Gamma_{\text{MARM MSTrype}}} N_{i} \vec{t} dy = 0 \end{split}$$

$$k \int_{\Omega} \left(\frac{\partial N_i}{\partial x} \frac{\partial N_j}{\partial x} + \frac{\partial N_i}{\partial y} \frac{\partial N_j}{\partial y} \right) T_j dx dy - \int_{\Omega} N_i N_j q dx dy = \int_{\Gamma} N_i \bar{Q} ds$$

Finite Element Model

Neglecting the body forces, above equations can be written in matrix form as:

$$\begin{bmatrix} K_{11} & K_{12} \\ 0 & K_{22} \end{bmatrix} \begin{Bmatrix} U^e \\ T^e \end{Bmatrix} = \begin{Bmatrix} F \\ Q \end{Bmatrix}$$

Where,

$$[K_{11}^e] = \int_{\Omega} [B]^T [C] [B] dx dy \quad [K_{12}^e] = \int_{\Omega} [B]^T [\beta] [N^{\theta}] dx dy \quad [K_{22}^e] = \int_{\Omega} [B^{\theta}]^T [K] [B^{\theta}] dx dy$$

$$\{F\} = \int_{\Gamma} [N]^T \overline{t} ds \quad \{Q\} = \int_{\Gamma} [N^{\theta}]^T \overline{Q} ds$$

$$[B] = \begin{bmatrix} \frac{\partial N_1}{\partial x} & 0 & \frac{\partial N_2}{\partial x} & 0 & \dots & \frac{\partial N_n}{\partial x} & 0 \\ 0 & \frac{\partial N_1}{\partial y} & 0 & \frac{\partial N_2}{\partial y} & \dots & 0 & \frac{\partial N_n}{\partial y} \\ \frac{\partial N_1}{\partial y} & \frac{\partial N_1}{\partial x} & \frac{\partial N_2}{\partial y} & \frac{\partial N_2}{\partial x} & \dots & \frac{\partial N_n}{\partial y} & \frac{\partial N_n}{\partial x} \end{bmatrix}, \ [B^{\theta}] = \begin{bmatrix} \frac{\partial N_1}{\partial y} & \frac{\partial N_2}{\partial y} & \dots & \frac{\partial N_n}{\partial y} \\ \frac{\partial N_1}{\partial y} & \frac{\partial N_2}{\partial y} & \dots & \frac{\partial N_n}{\partial y} \end{bmatrix}$$

$$N = \begin{bmatrix} N_1 & 0 & N_2 & 0 & \dots & N_n & 0 \\ 0 & N_1 & 0 & N_2 & \dots & 0 & N_n \end{bmatrix}, \ N^{\theta} = [N_1 & \dots & N_n]$$

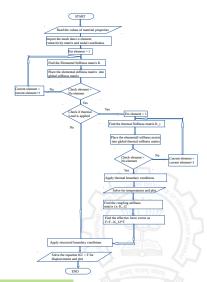
Computer implementation

A MATLAB program is developed to solve the 2-dimensional thermoelasticity problems.

Quadrilateral (Q4) elements were used for meshing the body.

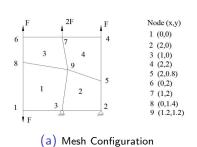
 2×2 Gauss quadrature rule is used for numerical integration.

A flow-chart showing the steps of the programming is shown in the figure.



Patch Test 1

- A square plate is taken and meshed with 4 elements as shown in figure below.
- Minimum number of essential boundary conditions is fixed to eliminate the rigid body motions
- Loads are applied such that there is constant state of stress in the body
- Numerical integration is performed using 2×2 Gauss quadrature rule.

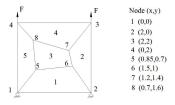


(b) Results of Patch Test 1

	Gauss Points	$\sigma_x/2F$	$\sigma_y/2F$	$\tau_{xy}/2F$
Element 1	1	-0.222×10^{-15}	1	0.1665×10^{-15}
	2	0.2498×10^{-15}	1	0.1665×10^{-15}
Element 1	3	0.3058×10^{-15}	1	0.222×10^{-15}
	4	0	1	0
	1	-0.222×10^{-15}	1	0
Element 2	2	41633×10^{-15}	1	0.111×10^{-15}
Element 2	3	0.02775×10^{-15}	1	0.138×10^{-15}
	4	-0.1110×10^{-15}	-1	0
	1	0.222×10^{-15}	2 1 s	0.1665×10^{-15}
Element 3	2	0.0555×10^{-15}	1	0.222×10^{-15}
Element 3	3	-0.0555×10^{-15}	1	0
	4	0.22204×10^{-15}	1	0
Element 4	1	-0.222×10^{-15}	1	0.1665×10^{-15}
	2	0.222×10^{-15}	1	0.1665×10^{-15}
	3	0.3058×10^{-15}	1 6	0.222×10^{-15}
	4	0.222×10^{-15}	1	0

Patch Test 2

- Another patch test is performed on a plate with 5 element mesh and solved and results are tabulated as shown below
- The 2×2 quadrature rule is used for numerical integration



(a) Mesh Configuration

(b) Results of Patch Test 2

	Gauss Point		σ_y/F	τ_{xy}/F
	1	0.166×10^{-15}	1	0.1665×10^{-15}
Element 1	2	-0.033×10^{-15}	1	0.1665×10^{-15}
Element 1	3	-0.063×10^{-15}	1	0.222×10^{-15}
	4	0	16×10^{-15} 1 0.1 66×10^{-15} 1 0.1 63×10^{-15} 1 0.1 63×10^{-15} 1 0.1 63×10^{-15} 1 0.1 9.2×10^{-15} 1 0.2 9.2×10^{-15} 1 0.2 9.2×10^{-15} 1 0.2 9.2×10^{-15} 1 0.5 9.2×10^{-15} 1 0.5 9.2×10^{-15} 1 0.1	0
	1	0.062×10^{-15}	1	0
Element 2	2	41633×10^{-15}	1	-0.222×10^{-15}
Liement 2	3	0.02775×10^{-15}	1	0.138×10^{-15}
	4	-0.2220×10^{-15}	1	0
	1	0.222×10^{-15}	1	0.1665×10^{-15}
Element 3	2	0.0555×10^{-15}	1	0.222×10^{-15}
Element 3	3	-0.0555×10^{-15}	1	WDIAN ONS LITE
	4	0.22204×10^{-15}	1	0
	1	-0.222×10^{-15}	/1/	0.1665×10^{-15}
Element 4	2	0.222×10^{-15}	1	0.1665×10^{-15}
	3	-0.063×10^{-15}	1	0.222×10^{-15}
	4	0.222×10^{-15}	1	0
Element 5	1	-0.422×10^{-15}	1	0.1665×10^{-15}
	2	0.222×10^{-15}	1	0.1665×10^{-15}
	3	-0.063×10^{-15}	1	0.222×10^{-15}
	4	0.222×10^{-15}	1	0

Patch Test 3: Thermal code

- Another patch was performed to validate the thermal code.
- Same problem as patch test was taken
- Temperature loads are applied to get the constant heat flux throughout the body

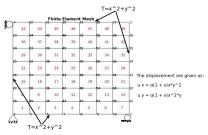
(b) Results of Patch test 3

TEMPERATURES (KELVIN)				
3				
2.5				
2	after after			
1.5	4 ,			
1	700			
0.5	1 / 2 41			
0	ne ne			
-0.5	-			
-1-1 -0	5 0 05 1 15 2 25 3			

(a)	Temperature	distribution

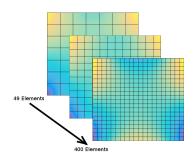
Nodes	Temperatures	$Q_{y}(W)$	$Q_{\times}(W)$
1	273	200	0.125×10^{-10}
2	273	200	0.155×10^{-10}
3	273	200	0.222×10^{-10}
4	281	200	0
5	276	200	0
6	281	200	INDIAN OSTITUTA
7	281	200	0.111×10^{-10}
8	279	200	0.138×10^{-10}
9	278	200	0

Patch Test 4: Thermo-elastic code



- A square plate of dimension 2×2 is taken and meshed as shown
- Temperature distribution of $T = x^2 + y^2$ is applied on 4 boundaries of plate
- a heat source $q=rac{\partial^2 T}{\partial x^2}+rac{\partial^2 T}{\partial y^2}=$ 4 is applied throughout the body
- Mesh is refined and to get the more accurate results

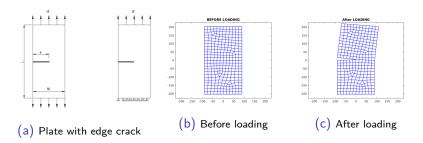
Mesh Refinements



Errors obtained after mesh refinements

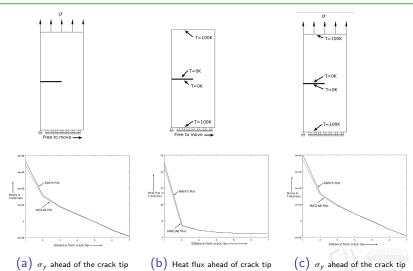
Number of Elements	Error in ux	Error in u _y	Error in σ_x	Error in σ_y	Error in $ au_{xy}$
7 × 7	5.783469×10^{-10}	5.783469×10^{-10}	6.668255×10^{2}	6.668255×10^{2}	3.615391×10^{2}
12 × 12	1.764697×10^{-11}	1.764697×10^{-11}	1.000597121	1.000597121	0.031722
20 × 20	1.265804×10^{-12}	1.265804×10^{-12}	0.994299467	0.994299467	0.005781

Plate with edge crack



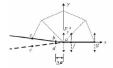
- A plate with edge crack is meshed in ANSYS and imported to MATLAB
- Boundary conditions are applied and stresses ahead of cracktip is plotted
- The results obtained by MATLAB code is compared with the ANSYS solution

Stresses Ahead of Crack-tip: Comparison with ANSYS

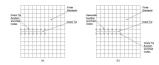


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Comparison of FEM and X-FEM methods



(a) Crack closure technique



(b) X-FEM enrichment of nodes

Crack closure integral:
$$G_I = \frac{W}{B\Delta a} = \frac{F_y^a u_y^b}{2B\Delta a}$$
 Thus, $K_I = \sqrt{\frac{G_I}{E}} = 5.24586910 \times 10^9$ Pa \sqrt{m} Analytical: $K_I = C * \sigma \sqrt{\pi a} = 4.726065 \times 10^9$ Pa \sqrt{m}

Method used	$\%$ Error = $\{K_{theoretical} - K_{numerical}/\}$	$K_{theoretical}\}$ ×	100
Finite Element Method	11.1%		THE
Extended Finite Element Method	4%	निय और्यो	

Conclusions and Future Work

- A FEM program is developed for Thermo-elastci Fracture problems based on semi-coupled formulation and patch tests were performed to validate the program.
- It is shown that when solving fracture problems with traditional FEM program, solutions are not accurate around the crack tip
- Same problem is solved in an X-FEM program developed by parnaik to show that the solution improves drastically when using X-FEM
- The Extended Finite Element enrichments will be applied to both displacement and temperature fields of thermoelasticity program in stage 2 of the project
- Q8 elements will be applied in our program to solve curved boundary problems.
- As the hydrogen diffusion problems governed by Laplace's equation same as thermoelastic problems, we will modify the X-FEM program to solve these problems.

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