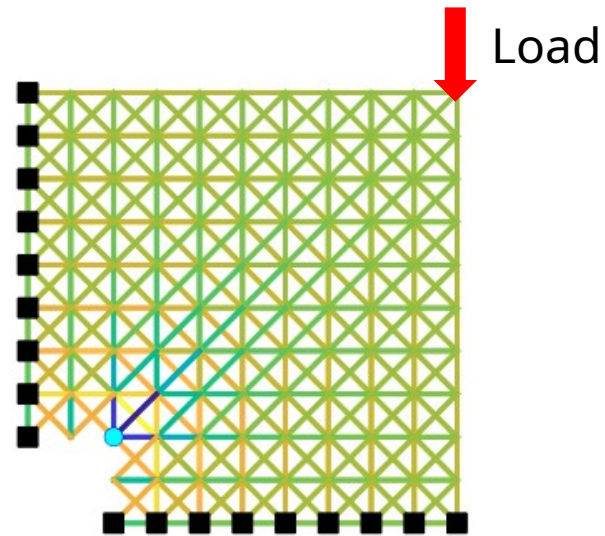


Columns: Euler Buckling





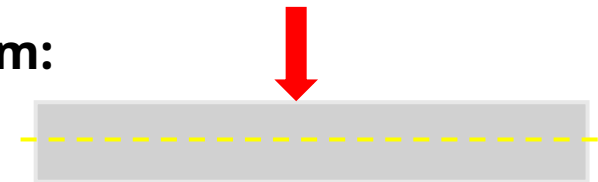
Parthenon: Greece

**2 =
Column:**

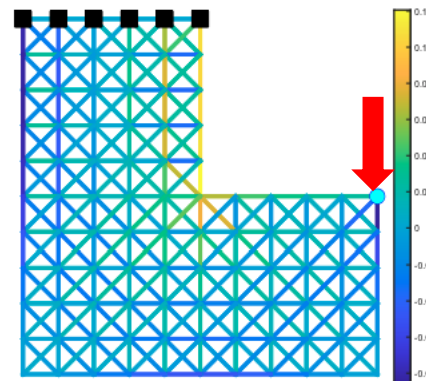


- Loads act along the axis of the structure

1 = Beam:

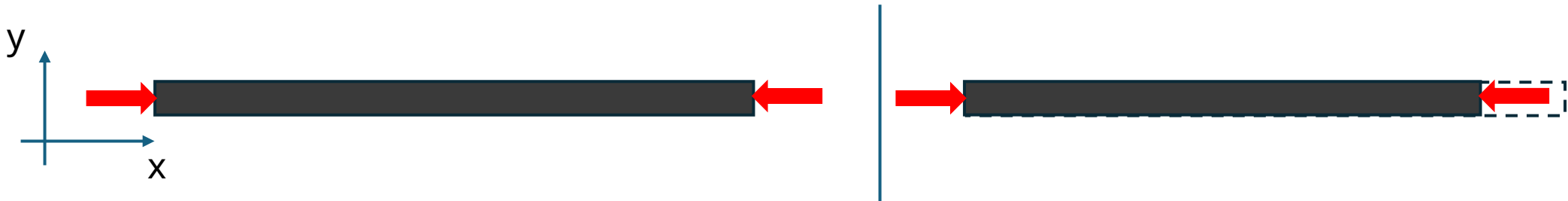


- Loads act perpendicular to the axis of the structure



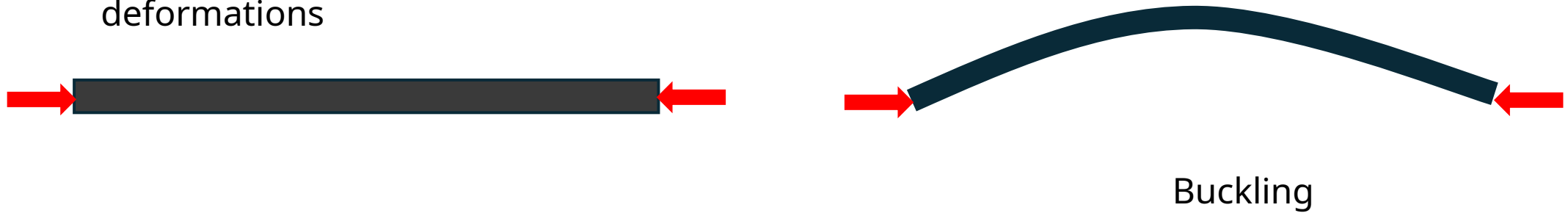
Case 1: Low levels of load

- The deformations are axial only






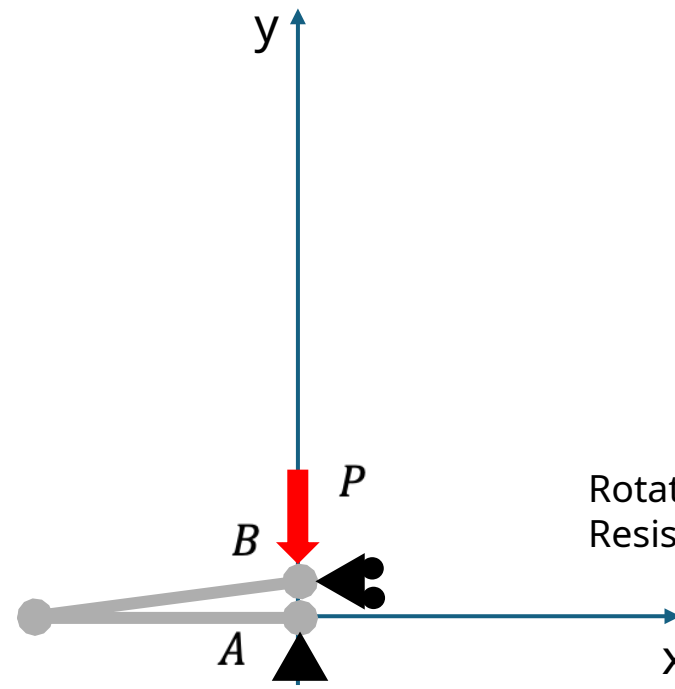
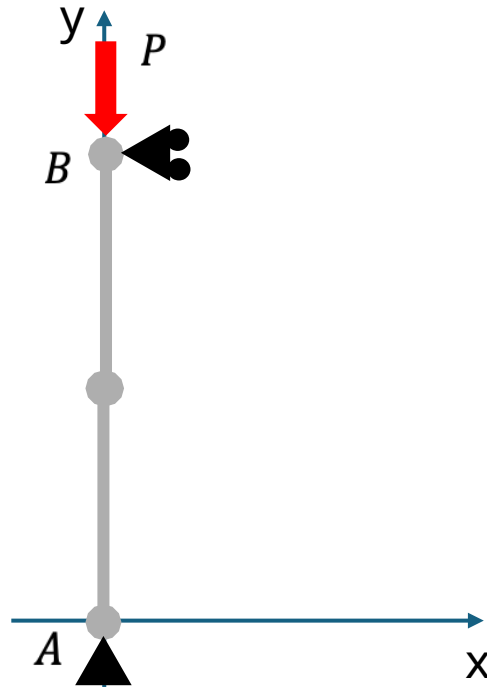
Case 2: High levels of load

- The deformations/deflections are governed by the bending with large deformations

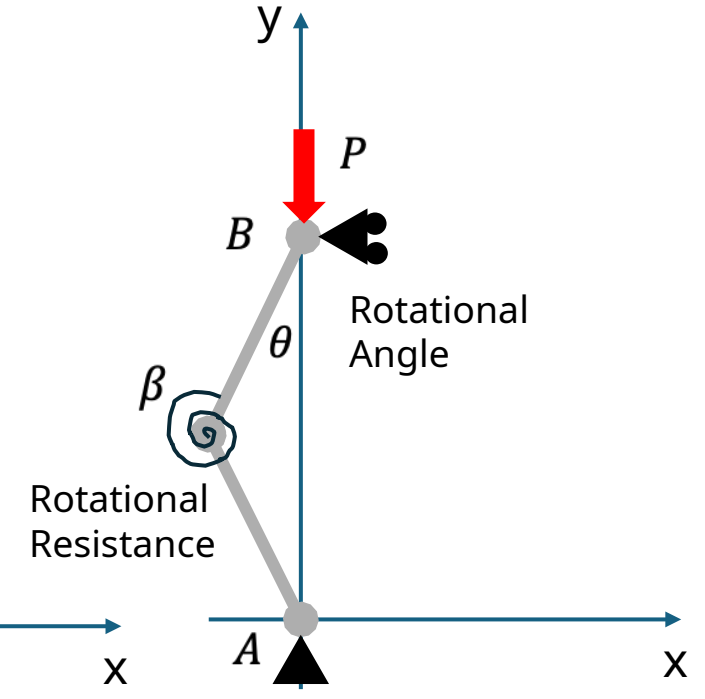


Buckling: Simple Case Study

-  Free to move in x but not y
-  Free to move in y but not x
-  Fixed






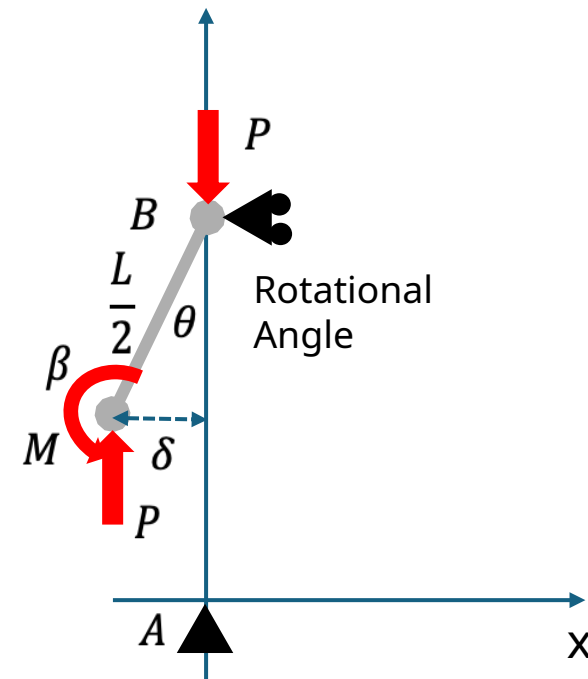
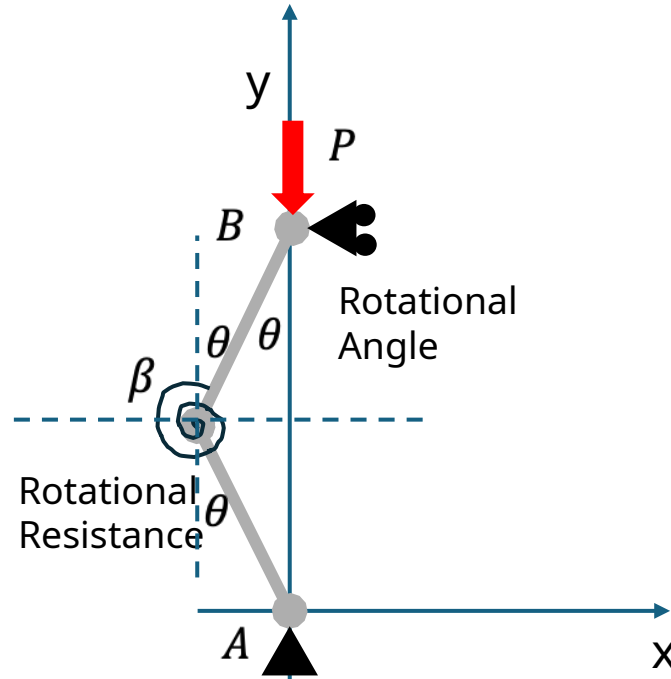
Simple Pinned Structure



Actual Engineering Structure

Buckling: Simple Case Study

-  Free to move in x but not y
-  Free to move in y but not x
-  Pinned



Actual Engineering
Structure

Moment Balance: $P\delta = M = 2\beta\theta$

Small Angle Assumption: $\delta = \frac{L}{2}\theta$

Load at which the structure is in
equilibrium:

$$P = \frac{4\beta}{L}$$

Force Balance, for structure to be in equilibrium:

$$P\delta = M = 2\beta\theta$$

Case #1: Restoring moment ($2\beta\theta$) is **GREATER** than the moment of due to the applied load ($P\delta$)

$$2\beta\theta > P\delta \longrightarrow 2\beta\theta > P\frac{L}{2}\theta$$
$$P < \frac{4\beta}{L}$$

STABLE EQUILIBRIUM

Case #2: Restoring moment ($2\beta\theta$) is **LESS** than the moment of due to the applied load ($P\delta$)

$$2\beta\theta < P\delta$$

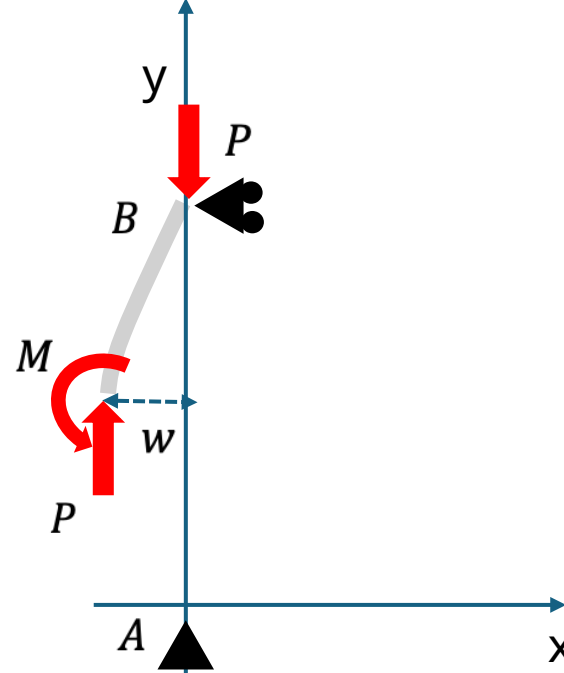
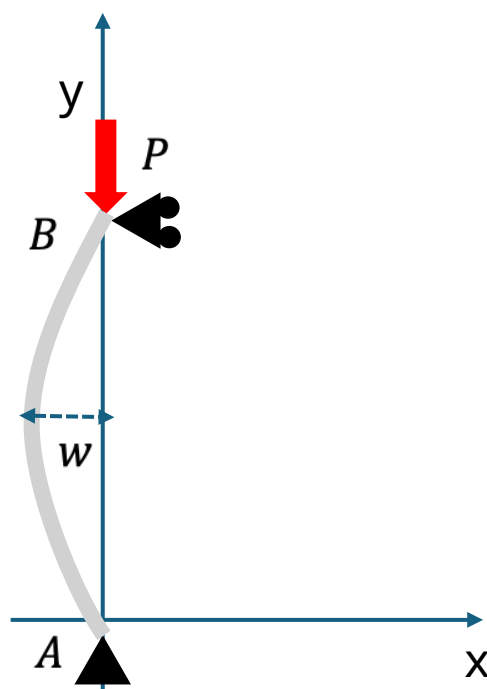
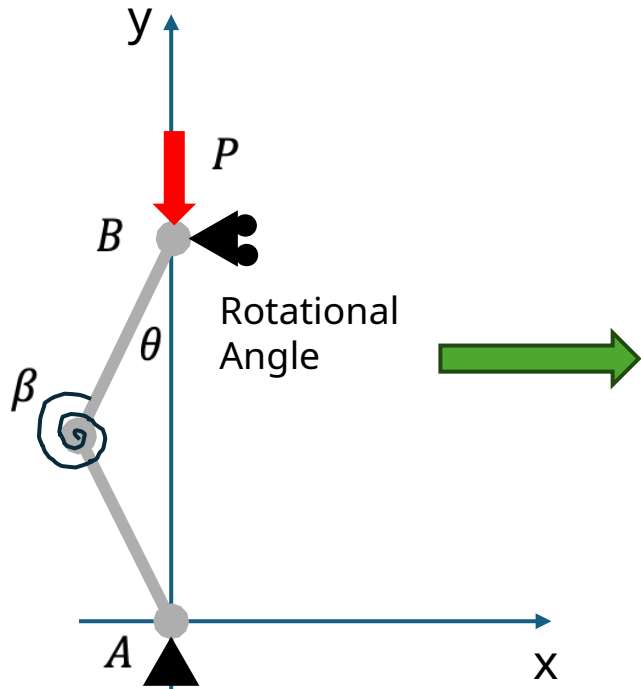
$$P > \frac{4\beta}{L}$$

**UNSTABLE
EQUILIBRIUM**

Rough Work

$$\delta = \frac{L}{2}\theta$$

Pinned-Pinned Columns



Moment Balance:

$$M - P(-w) = 0$$

$$M + Pw = 0$$

Bending Moment Equation:

$$\frac{d^2w}{dy^2} = \frac{M}{EI}$$

Differential Equation of Deflection Curve:

$$EI \frac{d^2w}{dy^2} + Pw = 0$$

$$\frac{d^2w}{dy^2} + k^2w = 0 \quad \text{where, } k^2 = \frac{P}{EI}$$

$$\frac{d^2w}{dy^2} + k^2w = 0 \quad \text{where, } k^2 = \frac{P}{EI}$$

General solution of the differential equation:

$$w = C_1 \sin(ky) + C_2 \cos(ky)$$

Boundary Conditions:

$$w = 0 \text{ at } y = 0,$$

$$C_2 = 0$$

$$y = L$$

$$C_1 \sin(kL) = 0$$

$$\text{Either: } C_1 = 0 \text{ or } \sin(kL) = 0$$

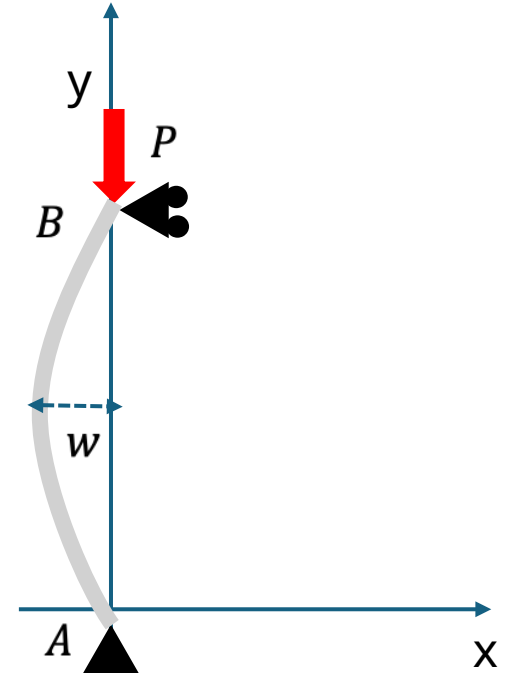
Trivial Solution:

Both C_1 and C_2 are zero

Buckling Equation:

$$\sin(kL) = 0$$

$$\sin(kL) = 0 \rightarrow kL = 0, \pi, 2\pi, \dots$$



$$kL = 0, \pi, 2\pi, \dots \quad \text{where, } k^2 = \frac{P}{EI}$$

Case #1: $kL = 0 \rightarrow P = 0$

Case #2: $kL = n\pi \rightarrow P = \frac{n^2\pi^2 EI}{L^2}$

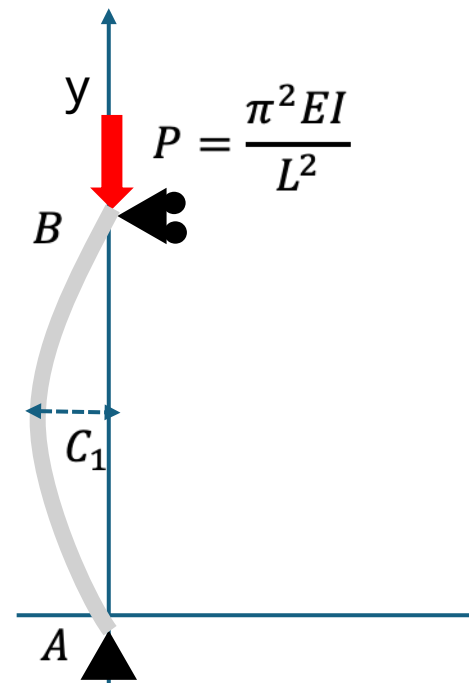
Critical loads on the structure

Deflection Equation: $\rightarrow w = C_1 \sin\left(\frac{n\pi}{L}y\right)$

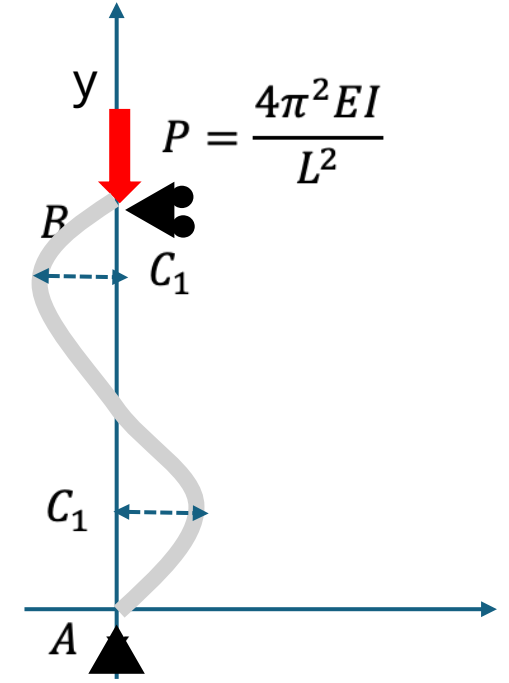
For $n = 1$, **Fundamental Case** of column buckling:

Load $P = \frac{\pi^2 EI}{L^2}$

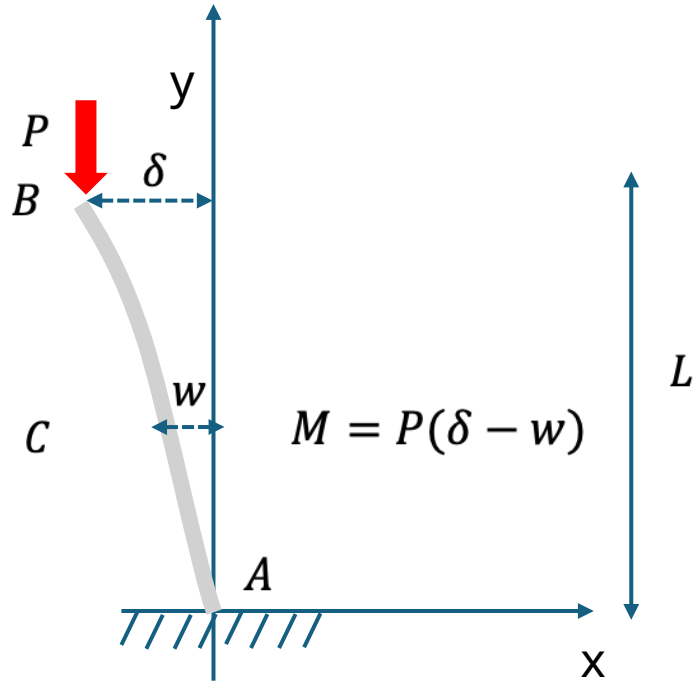
Deflection $w = C_1 \sin\left(\frac{\pi}{L}y\right)$



Deflection shape for $n=1$



Deflection shape for $n=2$



Fixed-Free Columns

$$\frac{d^2 w}{dy^2} = \frac{M}{EI}$$

$$EI \frac{d^2 w}{dy^2} = P(\delta - w)$$

$$\frac{d^2 w}{dy^2} + k^2 w = k^2 \delta$$

$$w = C_1 \sin(ky) + C_2 \cos(ky) + \delta$$

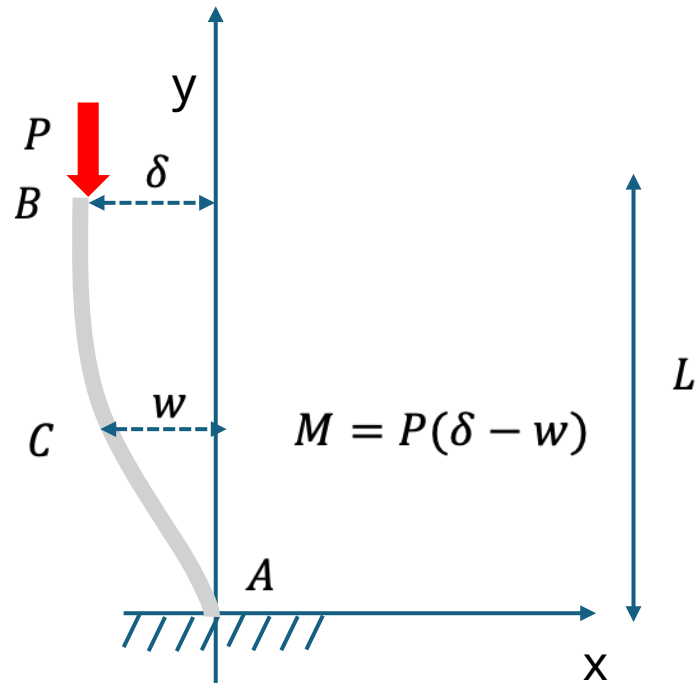
Boundary Conditions: $w = 0$ and $w' = 0$ at $y = 0$

$$C_2 = -\delta$$

$$C_1 = 0$$

$$w = \delta(1 - \cos(ky))$$

Boundary Conditions: $w = \delta$ at $y = L$ $\longrightarrow \cos(kL) = 0$



$$\cos(kL) = 0$$

$$kL = \frac{n\pi}{2} \quad n = 1, 3, 5 \dots$$

$$P = \frac{n^2 \pi^2 EI}{4L^2}$$

Deflection Equation: $\rightarrow w = \delta \left(1 - \cos\left(\frac{n\pi}{2L}y\right) \right)$

For $n = 1$, **Fundamental Case** of column buckling:

Load

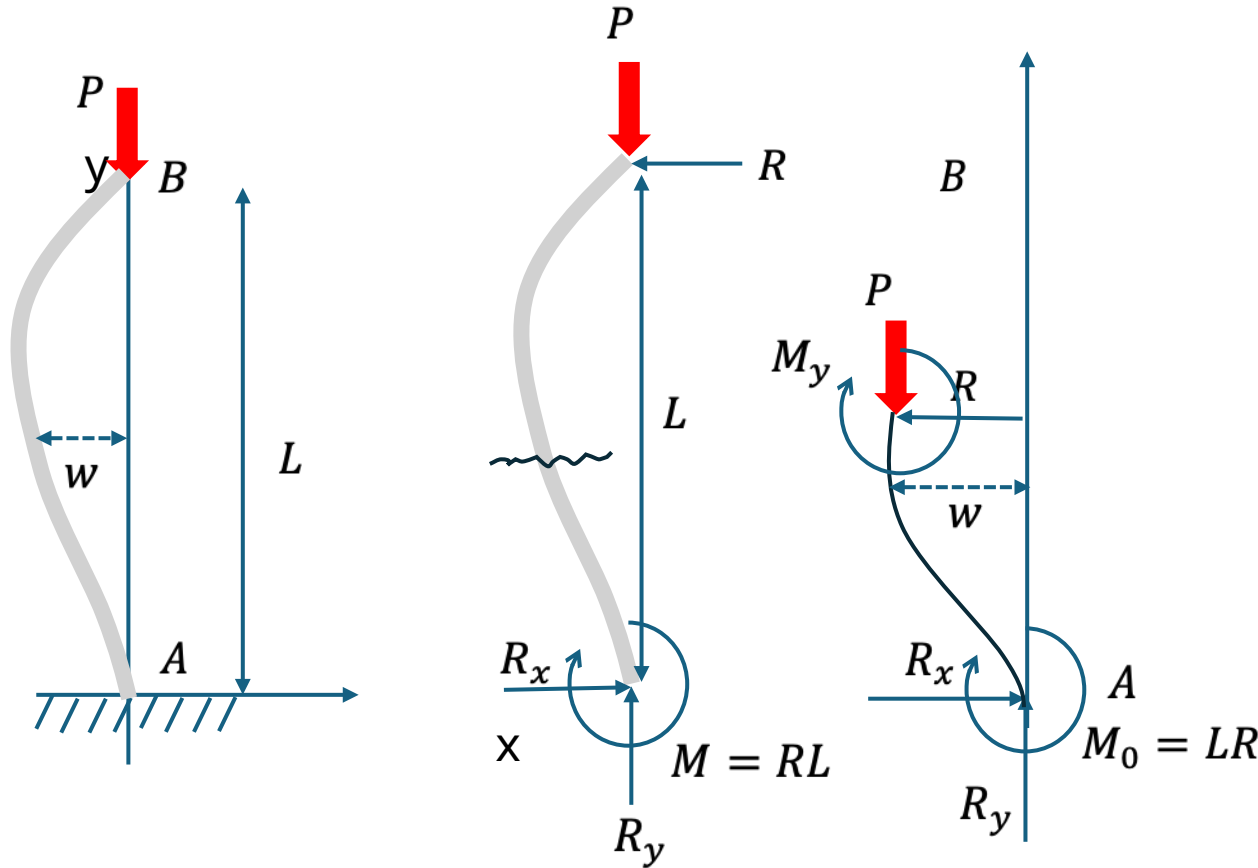
$$P_{cr} = \frac{\pi^2 EI}{4L^2}$$

Deflection

$$w = \delta \left(1 - \cos\left(\frac{\pi}{2L}y\right) \right)$$

Fixed-Pinned Columns

$$\mathbf{M} = [\mathbf{r}] \times [\mathbf{F}]$$



Force Balance

$$R_x = R$$

$$R_y = P$$

Moment Balance about A:

$$M_P = [-w\hat{i}] \times [-P\hat{j}] = wP\hat{k}$$

$$M_R = [y\hat{j}] \times [-R\hat{i}] = yR\hat{k}$$

$$M_y + wP + yR = RL$$

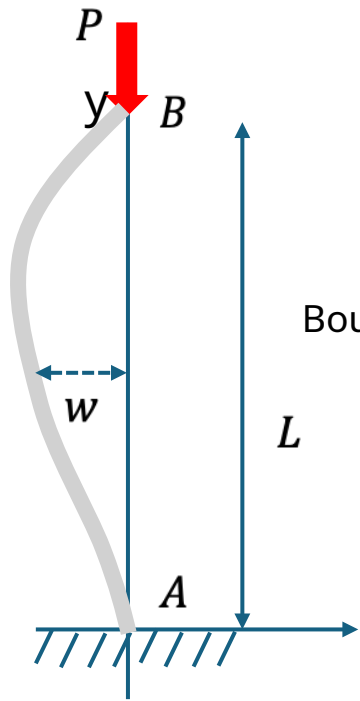
$$M_y = RL - Ry - Pw$$

$$\frac{d^2w}{dy^2} = \frac{M}{EI}$$

$$\frac{d^2w}{dy^2} + k^2w = \frac{R}{EI}(L - y)$$

Differential Equation of Deflection Curve:

Deflection curve: $w = C_1 \sin(ky) + C_2 \cos(ky) + \frac{R}{P}(L - y)$



$$w = C_1 \sin(ky) + C_2 \cos(ky) + \frac{R}{P}(L - y)$$

Boundary Conditions: $w = 0$ and $w' = 0$ at $y = 0$ $w = 0$ at $y = L$

$$C_2 + \frac{RL}{P} = 0$$

$$C_1 k - \frac{R}{P} = 0$$

$$\left. \begin{array}{l} C_2 + \frac{RL}{P} = 0 \\ C_1 k - \frac{R}{P} = 0 \end{array} \right\} C_2 = -C_1 kL$$

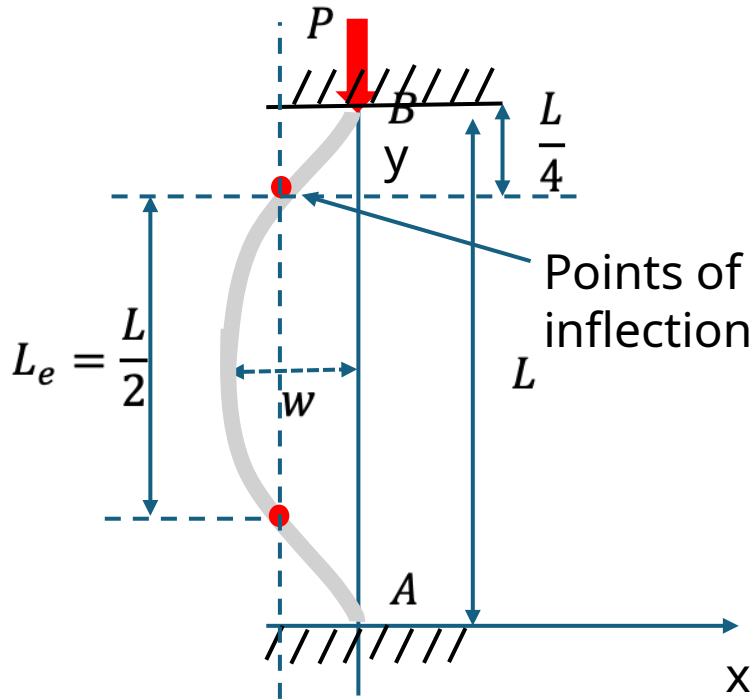
$$C_1 \tan(kL) + C_2 = 0 \quad \rightarrow \quad kL = \tan(kL)$$

$$kL = 4.49$$

$$P_{cr} = \frac{20.19EI}{L^2} \quad \rightarrow \quad P_{cr} = \frac{2.046\pi^2 EI}{L^2}$$

$$w = C_1 [\sin(ky) - kL \cos(ky) + k(L - y)]$$

Fixed-Fixed Columns

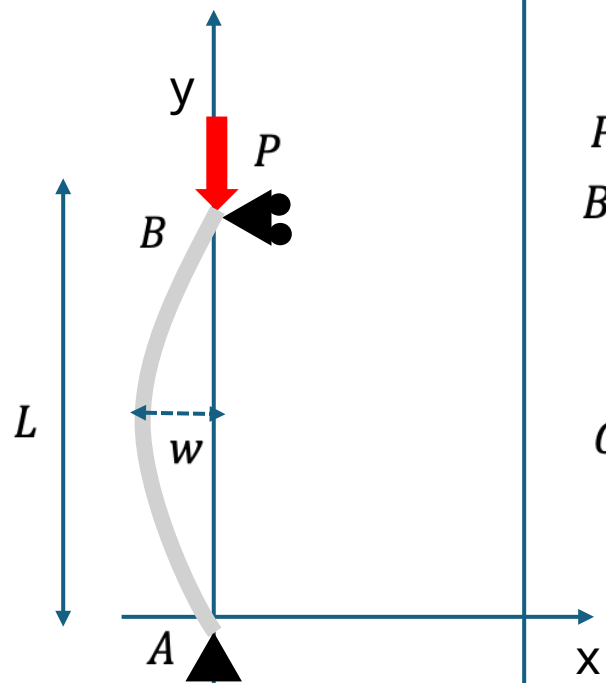


1. Points of inflection: Points at which the sign of the curvature changes
2. Moment at the points of inflection is zero
3. Deflection is symmetrical with slope of deflection curve zero at top, base and middle

$$P_{cr} = \frac{\pi^2 EI}{L_e^2} \quad \rightarrow \quad P_{cr} = \frac{4\pi^2 EI}{L^2}$$

The critical load for a column with fixed ends is 4 times the critical load for column with pinned ends

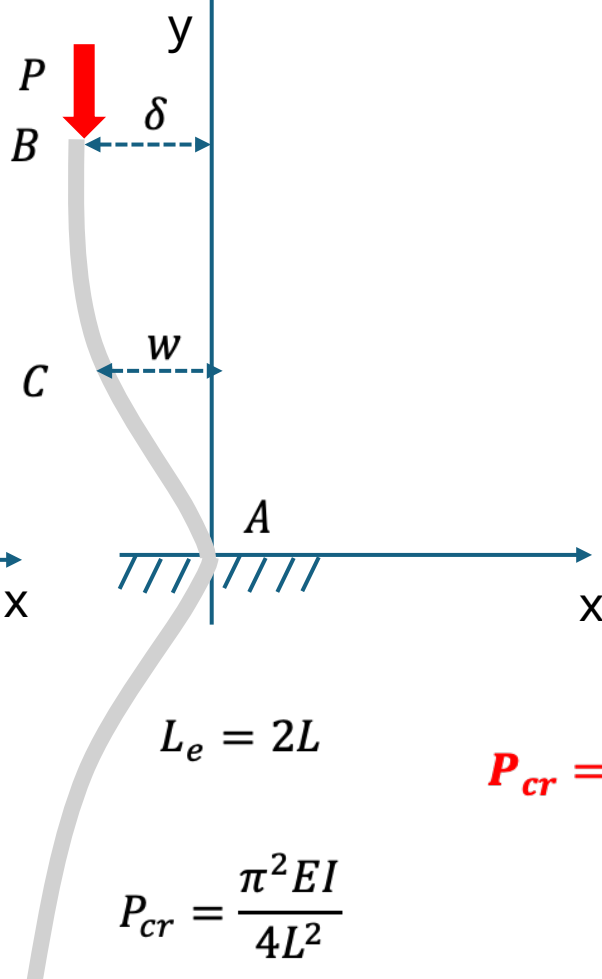
Pinned-Pinned Columns



$$L_e = L$$

$$P_{cr} = \frac{\pi^2 EI}{L^2}$$

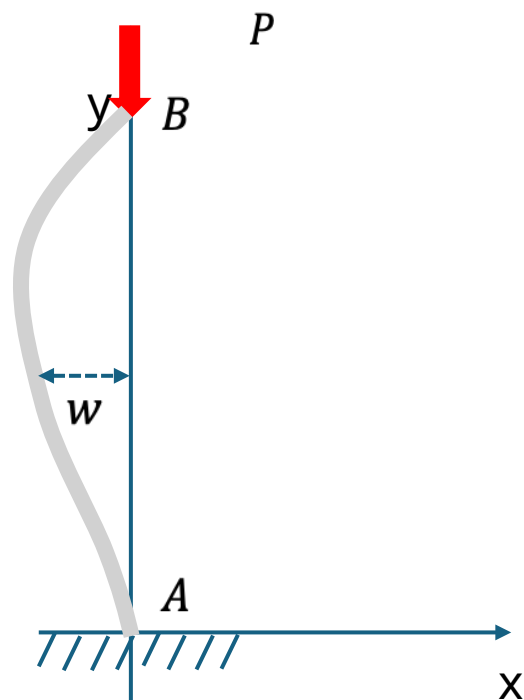
Fixed-Free Columns



$$L_e = 2L$$

$$P_{cr} = \frac{\pi^2 EI}{4L^2}$$

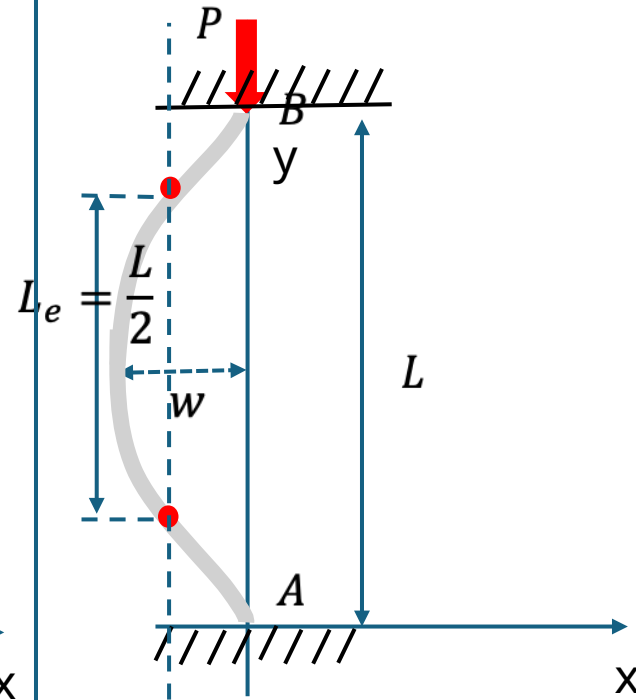
Fixed-Pinned Columns



$$L_e = 0.699L$$

$$P_{cr} = \frac{2.046\pi^2 EI}{L^2}$$

Fixed-Fixed Columns



$$L_e = 0.5L$$

$$P_{cr} = \frac{4\pi^2 EI}{L^2}$$

$$P_{cr} = \frac{\pi^2 EI}{L_e^2}$$

THANK YOU!

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