

Fields, Forms and Flows 3/34

Solution Sheet 4

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1. Lie derivative of a vector field.

- (a) (5 marks) For $n = 1$, the required formula is just Proposition 1.8.10. Now use induction – suppose it's true for n , and verify for $n + 1$ as follows:

$$\begin{aligned} \frac{\partial^{n+1}}{\partial s^{n+1}} \Psi_{s*} \mathbb{X} &= \frac{\partial}{\partial s} \frac{\partial^n}{\partial s^n} \Psi_{s*} \mathbb{X} = \frac{\partial}{\partial s} (-1)^n L_{\mathbb{Y}}^n \Psi_{s*} \mathbb{X} \text{ (by induction hypothesis)} \\ &= (-1)^n L_{\mathbb{Y}}^n \frac{\partial}{\partial s} \Psi_{s*} \mathbb{X} \text{ (as } \mathbb{Y} \text{ does not depend on } s) \\ &= (-1)^n L_{\mathbb{Y}}^n (-L_{\mathbb{Y}} \Psi_{s*} \mathbb{X}) = (-1)^{n+1} L_{\mathbb{Y}}^{n+1} \Psi_{s*} \mathbb{X}, \end{aligned}$$

as required.

- (b) (5 marks) We have the formal power series

$$\Psi_{s*} \mathbb{X} = \sum_{n=0}^{\infty} \frac{\partial^n}{\partial s^n} \Psi_{s*} \mathbb{X} \Big|_{s=0} \frac{s^n}{n!}.$$

From part (a), since $\Psi_{0*} \mathbb{X} = \mathbb{X}$,

$$\frac{\partial^n}{\partial s^n} \Psi_{s*} \mathbb{X} \Big|_{s=0} = (-1)^n L_{\mathbb{Y}}^n \Psi_{s*} \mathbb{X} \Big|_{s=0} = (-1)^n L_{\mathbb{Y}}^n \mathbb{X}.$$

Substitute to get

$$\Psi_{s*} \mathbb{X} = \left(\sum_{n=0}^{\infty} \frac{s^n}{n!} (-1)^n L_{\mathbb{Y}}^n \right) \mathbb{X} = e^{-sL_{\mathbb{Y}}} \mathbb{X},$$

as required. Since $L_{\mathbb{Y}} \mathbb{Y} = [\mathbb{Y}, \mathbb{Y}] = 0$, it follows that $L_{\mathbb{Y}}^n \mathbb{Y} = 0$ for $n \geq 1$, so that

$$\Psi_{s*} \mathbb{Y} = \mathbb{Y}.$$

(We had a different, flow-based proof of this fact in Proposition 1.8.11).

- (c) (5 marks) Just go from $L_{\mathbb{X}}$ -notation to bracket-notation and back again, as follows:

$$\begin{aligned} (L_{\mathbb{X}} L_{\mathbb{Y}} - L_{\mathbb{Y}} L_{\mathbb{X}}) \mathbb{Z} &= L_{\mathbb{X}} [\mathbb{Y}, \mathbb{Z}] - L_{\mathbb{Y}} [\mathbb{X}, \mathbb{Z}] = [\mathbb{X}, [\mathbb{Y}, \mathbb{Z}]] - [\mathbb{Y}, [\mathbb{X}, \mathbb{Z}]] \\ &= [\mathbb{X}, [\mathbb{Y}, \mathbb{Z}]] + [[\mathbb{X}, \mathbb{Z}], \mathbb{Y}] \\ &= [[\mathbb{X}, \mathbb{Y}], \mathbb{Z}] \text{ (from the Jacobi identity)} = L_{[\mathbb{X}, \mathbb{Y}]} \mathbb{Z}, \end{aligned}$$

as required.

2. Parallel parking.

- (a) (5 marks) Let \mathbb{S} be given by

$$\mathbb{S} = (0, 0, 0, \omega), \tag{1}$$

where ω is a constant. The flow of \mathbb{S} satisfies the system of differential equations

$$\dot{x} = 0, \quad \dot{y} = 0, \quad \dot{\theta} = 0, \quad \dot{\alpha} = \omega,$$

with solutions $x(t) = x_0$, $y(t) = y_0$, $\theta(t) = \theta_0$, $\alpha(t) = \alpha_0 + \omega t$. Thus, Φ_t is given by

$$\Phi_t(x, y, \theta, \alpha) = (x, y, \theta, \alpha + \omega t). \tag{2}$$

- (b) (10 marks) $P = (x, y)$ specifies the centre of the back axle, and θ specifies the inclination of the back axle to the vertical. If the back wheels roll forward a distance $v\delta t$, P undergoes a displacement δP given by

$$\delta P = v\delta t(\sin \theta, \cos \theta). \quad (3)$$

$Q = (x + L \sin \theta, y + L \cos \theta)$ specifies the centre of the front axle, and $\theta + \alpha$ specifies the inclination of the front axle to the vertical. If the front wheels roll forward a distance $\delta\sigma$, Q undergoes a displacement δQ given by

$$\delta Q = \delta\sigma(\sin(\theta + \alpha), \cos(\theta + \alpha)). \quad (4)$$

The magnitude of this displacement, $\delta\sigma$, is determined by requiring that the distance between $P + \delta P$ and $Q + \delta Q$ is L (the car can't stretch). We have that

$$|(P + \delta P) - (Q + \delta Q)|^2 = |P - Q|^2 + 2(P - Q) \cdot (\delta P - \delta Q) + O(2).$$

Since $|P - Q| = L$, we must have $2(P - Q) \cdot (\delta P - \delta Q) = 0$, or

$$L(\sin \theta, \cos \theta) \cdot (v\delta t \sin \theta - \delta\sigma \sin(\theta + \alpha), v\delta t \cos \theta - \delta\sigma \cos(\theta + \alpha)) = 0.$$

This simplifies to

$$v\delta t - \delta\sigma(\cos \theta \cos(\theta + \alpha) + \sin \theta \sin(\theta + \alpha)) = v\delta t - \delta\sigma \cos \alpha = 0,$$

or

$$\delta\sigma = \frac{v}{\cos \alpha} \delta t.$$

Note the singularity at $\alpha = \pm\pi/2$. If the front wheels are perpendicular to the car, P can't move (at least not without the front wheels skidding).

As θ specifies the orientation of the car with respect to the vertical, it is given by, eg, $(Q - P) \cdot (1, 0) = L \sin \theta$. Thus $((Q + \delta Q) - (P + \delta P)) \cdot (1, 0) = L \sin(\theta + \delta\theta)$. At first order in the displacements, we get

$$(\delta Q - \delta P) \cdot (1, 0) = L \cos \theta \delta\theta.$$

Using the expressions (3) and (4) for δP and δQ , we get

$$\delta\theta = \frac{v\delta t}{L \cos \theta} \left(\frac{\sin(\theta + \alpha)}{\cos \alpha} - \sin \theta \right) = \frac{v}{L} \tan \alpha \delta t.$$

The vector field \mathbb{D} which describes the displacement $P \rightarrow P + \delta P$, $\theta \rightarrow \theta + \delta\theta$ is given by

$$\mathbb{D} = (v \sin \theta, v \cos \theta, \frac{v}{L} \tan \alpha, 0). \quad (5)$$

The flow of \mathbb{D} satisfies the system of differential equations

$$\dot{x} = v \sin \theta, \quad \dot{y} = v \cos \theta, \quad \dot{\theta} = (v/L) \tan \alpha, \quad \dot{\alpha} = 0.$$

The solutions are $\alpha(t) = \alpha_0$, $\theta(t) = \theta_0 + (vt/L) \tan \alpha$, $x(t) = x_0 - L(\cos \theta(t) - \cos \theta_0)/\tan \alpha$, and $y(t) = y_0 + L(\sin \theta(t) - \sin \theta_0)/\tan \alpha$. Thus, the flow is given by

$$\begin{aligned} \Psi_t(x, y, \theta, \alpha) &= \left(x - \frac{L}{\tan \alpha} \left(\cos \left(\theta + \frac{vt}{L} \tan \alpha \right) - \cos \theta \right), \right. \\ &\quad y + \frac{L}{\tan \alpha} \left(\sin \left(\theta + \frac{vt}{L} \tan \alpha \right) - \sin \theta \right), \\ &\quad \theta + \frac{vt}{L} \tan \alpha, \\ &\quad \left. \alpha \right). \end{aligned} \quad (6)$$

Under Ψ_t , the point P describes a circle centred at $(x + L \cos \theta / \tan \alpha, y - L \sin \theta / \tan \alpha)$ with radius $L / \tan \alpha$. Note that when $\alpha = 0$ (the front wheels are not turned), the turning radius is infinite, and the car moves in a straight line.

- (c) (5 marks) Compute $\Psi_\epsilon(x, y, \theta, \epsilon\Omega)$ through order ϵ^2 by expanding in (6). The x -component is given by

$$x - \frac{L}{\tan \epsilon\Omega} \left(\cos \left(\theta + \frac{v\epsilon}{L} \tan \epsilon\Omega \right) - \cos \theta \right).$$

Note that $\tan \epsilon\Omega = \epsilon\Omega + O(\epsilon^3)$, so that

$$\cos \left(\theta + \frac{v\epsilon}{L} \tan \epsilon\Omega \right) - \cos \theta = \cos \left(\theta + \frac{\epsilon^2 \Omega v}{L} + O(\epsilon^4) \right) - \cos \theta = -\epsilon^2 \frac{\Omega v}{L} \sin \theta + O(\epsilon^4).$$

Substitute into the preceding expression to get the following expression for the x -component:

$$x + \frac{L}{\epsilon\Omega} \left(\epsilon^2 \frac{\Omega v}{L} \sin \theta + O(\epsilon^4) \right) = x + \epsilon v \sin \theta + O(\epsilon^3).$$

A similar calculation gives the y -component as

$$y + \epsilon v \cos \theta + O(\epsilon^3).$$

Next, the θ -component is given by

$$\theta + \frac{v\epsilon}{L} \tan(\epsilon\Omega) = \theta + \epsilon^2 \frac{\Omega v}{L} + O(\epsilon^4).$$

Finally, the α -component is just $\epsilon\Omega$, since α is unchanged by Ψ_t . Collecting results we have that

$$\Psi_\epsilon(x, y, \theta, \epsilon\Omega) = (x + \epsilon v \sin \theta, y + \epsilon v \cos \theta, \theta + \epsilon^2 \Omega v / L, \epsilon\Omega) + O(\epsilon^3). \quad (7)$$

- (d) (10 marks)

$$\mathbb{A} = [\mathbb{S}, \mathbb{D}] = (\mathbb{S} \cdot \nabla) \mathbb{D} - (\mathbb{D} \cdot \nabla) \mathbb{S}.$$

As \mathbb{S} is constant, $(\mathbb{D} \cdot \nabla) \mathbb{S} = 0$. Also, $(\mathbb{S} \cdot \nabla) \mathbb{D} = \omega \partial \mathbb{D} / \partial \alpha$, so that

$$\mathbb{A} = \left(0, 0, \frac{\omega v}{L} \sec^2 \alpha, 0 \right).$$

Thus, \mathbb{A} has a component in the θ -direction only. Γ_a , the flow of \mathbb{A} , is given by

$$\Gamma_a(x, y, \theta, \alpha) = (x, y, \theta + (\omega v \sec^2 \alpha / L) a, \alpha). \quad (8)$$

Next,

$$\mathbb{B} = [[\mathbb{S}, \mathbb{D}], \mathbb{D}] = [\mathbb{A}, \mathbb{D}] = (\mathbb{A} \cdot \nabla) \mathbb{D} - (\mathbb{D} \cdot \nabla) \mathbb{A}.$$

Since \mathbb{A} has a non-zero component in the θ -direction only, we have that

$$(\mathbb{A} \cdot \nabla) \mathbb{D} = \frac{\omega v}{L} \sec^2 \alpha \frac{\partial \mathbb{D}}{\partial \theta} = \frac{\omega v^2}{L} \sec^2 \alpha (\cos \theta, -\sin \theta, 0, 0),$$

and, since \mathbb{A} depends only α and the α -component of \mathbb{D} vanishes, $(\mathbb{D} \cdot \nabla) \mathbb{A} = 0$. Therefore,

$$\mathbb{B} = \frac{\omega v^2}{L} \sec^2 \alpha (\cos \theta, -\sin \theta, 0, 0).$$

Thus, \mathbb{B} has components in the x - and y -directions only; it is a pure translation. Its flow, Δ_b , is given by

$$\Delta_b(x, y, \theta, \alpha) = \left(x + \frac{\omega v^2}{L} \sec^2 \alpha \cos \theta b, y - \frac{\omega v^2}{L} \sec^2 \alpha \sin \theta b, \theta, \alpha \right). \quad (9)$$

Note that the direction of the translation is perpendicular to the axis of the car.

Linear independence: construct a 4×4 matrix whose rows are the components of the vector fields \mathbb{D} , \mathbb{B} , \mathbb{A} and \mathbb{S} (in that order),

$$\begin{pmatrix} v \sin \theta & v \cos \theta & (v/L) \tan \alpha & 0 \\ \frac{\omega v^2}{L} \sec^2 \alpha \cos \theta & -\frac{\omega v^2}{L} \sec^2 \alpha \sin \theta & 0 & 0 \\ 0 & 0 & \frac{\omega v}{L} \sec^2 \alpha & 0 \\ 0 & 0 & 0 & \omega \end{pmatrix}$$

Linear independence of \mathbb{A} , \mathbb{B} , \mathbb{S} and \mathbb{D} is equivalent to this matrix being nonsingular, ie having nonzero determinant. The determinant is given by $-\omega^3 v^4 \sec^4 \alpha / L^2$, which is well defined and nonzero provided $\cos \alpha \neq 0$.

(e) (10 marks)

Let $M_\epsilon = \Psi_{-\epsilon} \circ \Phi_{-\epsilon} \circ \Psi_\epsilon \circ \Phi_\epsilon$ (ie, the sequence steer right, drive forward, steer left, drive back, for a time ϵ). Let

$$p = \Phi_\epsilon(x, y, \theta, 0), \quad q = \Psi_\epsilon(p), \quad r = \Phi_{-\epsilon}(q), \quad s = \Psi_{-\epsilon}(r) = M_\epsilon(x, y, \theta, 0).$$

The flows $\Phi_{\pm\epsilon}$ and $\Psi_{\pm\epsilon}$ are given (through $O(\epsilon^3)$) by (2) and (7). Use these expressions to compute p , q , r and s , as follows:

$$\begin{aligned} p &= (x, & y, \theta, & \epsilon\omega) + O(\epsilon^3), \\ q &= (x + \epsilon v \sin \theta, & y + \epsilon v \cos \theta, \theta + \epsilon^2 \omega v / L, & \epsilon\omega) + O(\epsilon^3), \\ r &= (x + \epsilon v \sin \theta, & y + \epsilon v \cos \theta, \theta + \epsilon^2 \omega v / L, & 0) + O(\epsilon^3), \\ s &= (x, & y, \theta + \epsilon^2 \omega v / L, & 0) + O(\epsilon^3) \\ &= \Gamma_{\epsilon^2}(x, y, \theta, 0) + O(\epsilon^3). \end{aligned}$$

Thus,

$$M_\epsilon(x, y, \theta, 0) = \Gamma_{\epsilon^2}(x, y, \theta, 0) + O(\epsilon^3), \quad (10)$$

as required.

Let $(M_\epsilon)^j$ denote M_ϵ repeated j times, ie

$$(M_\epsilon)^j = \underbrace{M_\epsilon \circ \dots \circ M_\epsilon}_{j \text{ times}}.$$

From (10), it follows that

$$(M_\epsilon)^j(x, y, \theta, 0) = \Gamma_{j\epsilon^2}(x, y, \theta, 0) + O(j\epsilon^3),$$

so that

$$(M_\epsilon)^{[\epsilon^{-2}a]}(x, y, \theta, 0) = \Gamma_a(x, y, \theta, 0) + O(\epsilon), \quad (11)$$

where $[x]$ denotes the integer part of x .

Suppose your car initially has configuration $(x, y, \theta, 0)$, and you want to turn it so that it points in the direction $\theta + \Theta$, all the while keeping x and y to within $O(\epsilon)$ of their initial values. In view of (10) and (11), it suffices to execute the maneuver M_ϵ j times in succession, where

$$j = \left\lceil \frac{L}{\epsilon^2 \omega v} \Theta \right\rceil.$$

ϵ can be made arbitrarily small, but the number of maneuvers j grows as ϵ^{-2} .

(f) (10 marks) Let

$$P_\delta = \Psi_{-\delta} \circ \Gamma_{-\delta T} \circ \Psi_\delta \circ \Gamma_{\delta T}.$$

Let

$$p = \Gamma_{\delta T}(x, y, \theta, 0), \quad q = \Psi_\delta(p), \quad r = \Gamma_{-\delta T}(q), \quad s = \Psi_{-\delta}(r) = P_\delta(x, y, \theta, 0).$$

The flows Ψ_δ and $\Gamma_{\delta T}$ are given (through $O(\delta^3)$) by (7) and (8). Use these expressions to compute p , q , r and s as follows, letting $\nu = \omega v T / L$:

$$\begin{aligned} p &= (x, & y, \theta + \delta\nu, & 0), \\ q &= (x + \delta v \sin \theta + \delta^2 v \nu \cos \theta, & y + \delta v \cos \theta - \delta^2 v \nu \sin \theta, \theta + \delta\nu, & 0) + O(\delta^3), \\ r &= (x + \delta v \sin \theta + \delta^2 v \nu \cos \theta, & y + \delta v \cos \theta - \delta^2 v \nu \sin \theta, \theta, & 0) + O(\delta^3), \\ s &= (x + \delta^2 v \nu \cos \theta, & y - \delta^2 v \nu \sin \theta, \theta, & 0) + O(\delta^3) \\ &= \Delta_{\delta^2 T}(x, y, \theta, 0) + O(\delta^3). \end{aligned}$$

Thus,

$$P_\delta(x, y, \theta, 0) = \Delta_{\delta^2 T}(x, y, \theta, 0) + O(\delta^3), \quad (12)$$

as required.

Let $(P_\delta)^k$ denote P_δ repeated k times, ie

$$(P_\delta)^k = \underbrace{P_\delta \circ \dots \circ P_\delta}_{k \text{ times}}.$$

From (12) it follows that

$$(P_\delta)^k(x, y, \theta, 0) = \Delta_{k\delta^2 T}(x, y, \theta, 0) + O(k\delta^3),$$

so that

$$(P_\delta)^{[\delta^{-2}b/T]}(x, y, \theta, 0) = \Delta_b(x, y, \theta, 0) + O(\delta). \quad (13)$$

From part (e) above, we have that

$$\Gamma_{\delta T}(x, y, \theta, 0) = (M_{(\delta T)^2})^{[(\delta T)^{-3}]}(x, y, \theta, 0) + O(\delta^3).$$

Therefore, $\Gamma_{\delta T}(x, y, \theta, 0)$ can be realised, to within $O(\delta^3)$, by a sequence of $M_{\delta^2 T^2}$ maneuvers, ie by a sequence of steers and drives. Therefore, $P_\delta(x, y, \theta, 0)$ can also be realised, to within $O(\delta^3)$, by a sequence of steers and drives.

Suppose your car initially has configuration $(x, y, \theta, 0)$, and you want to shift it a distance d perpendicular to its length while keeping its direction θ to within $O(\delta)$ of its original value. In view of (12) and (13), it suffices to execute the maneuver P_δ k times in succession, where

$$k = \left\lceil \frac{Ld}{\delta^2 T \omega v^2} \Theta \right\rceil.$$

3. Noncommutativity of rotations in \mathbb{R}^3 .

- (a) (5 marks) One way to proceed is to use the result of Question 4 of Problem Sheet 3 and then verify explicitly that

$$\begin{aligned} [A, B] &= AB - BA = C, \\ [B, C] &= BC - CB = A, \\ [C, A] &= CA - AC = B. \end{aligned}$$

Alternatively, we can compute the Jacobi brackets directly. We have that

$$X_A(r) = (0, -z, y), \quad X_B(r) = (z, 0, -x), \quad X_C(r) = (-y, x, 0).$$

Then

$$\begin{aligned} [X_A, X_B] &= \left(\left(-z \frac{\partial}{\partial y} + y \frac{\partial}{\partial z} \right) (z, 0, -x) - \left(z \frac{\partial}{\partial x} - x \frac{\partial}{\partial z} \right) (0, -z, y) \right) \\ &= (y, 0, 0) - (0, x, 0) = (y, -x, 0) = -X_C. \end{aligned}$$

The other relations are similarly proved.

- (b) (5 marks) The system $\dot{r} = X_C(r)$ yields

$$\dot{x} = -y, \quad \dot{y} = x, \quad \dot{z} = 0.$$

$\dot{z} = 0$ implies that $z(t) = z_0$, and the equations for x and y can be solved as follows. We have that $\ddot{x} = -\dot{y} = -x$, which has solution $x(t) = \cos t x_0 + \sin t \dot{x}_0$. Similarly, $\ddot{y} = \dot{x} = -y$, which has solution $y(t) = \cos t y_0 + \sin t \dot{y}_0$. But the original differential equations imply that $\dot{x}_0 = -y_0$ and $\dot{y}_0 = x_0$. Therefore,

$$\begin{aligned} x(t) &= \cos t x_0 - \sin t y_0, \\ y(t) &= \sin t x_0 + \cos t y_0. \end{aligned}$$

Collectively the solutions are described by the matrix equation

$$\begin{pmatrix} x \\ y \\ z \end{pmatrix} (t) = \begin{pmatrix} \cos t & -\sin t & 0 \\ \sin t & \cos t & 0 \\ 0 & 0 & 1 \end{pmatrix} \begin{pmatrix} x_0 \\ y_0 \\ z_0 \end{pmatrix}.$$

Therefore, we can write

$$\Phi_{Ct}(r) = \mathcal{R}_C(t) \cdot r,$$

where

$$\mathcal{R}_C(t) = \begin{pmatrix} \cos t & -\sin t & 0 \\ \sin t & \cos t & 0 \\ 0 & 0 & 1 \end{pmatrix}.$$

(c) (5 marks) Similarly,

$$\Phi_{At}(r) = \mathcal{R}_A(t) \cdot r, \quad \Phi_{Bt}(r) = \mathcal{R}_B(t) \cdot r,$$

where

$$\mathcal{R}_A(t) = \begin{pmatrix} 1 & 0 & 0 \\ 0 & \cos t & -\sin t \\ 0 & \sin t & \cos t \end{pmatrix}, \quad \mathcal{R}_B(t) = \begin{pmatrix} \cos t & 0 & \sin t \\ 0 & 1 & 0 \\ -\sin t & 0 & \cos t \end{pmatrix}.$$

(d) (5 marks) We have that

$$(\Phi_{B\theta} \circ \Phi_{A\theta} \circ \Phi_{-B\theta} \circ \Phi_{-A\theta})(r) = \mathcal{R}_B(\theta)\mathcal{R}_A(\theta)\mathcal{R}_B(-\theta)\mathcal{R}_A(-\theta) \cdot r.$$

Up to and including terms of second order in θ , $\sin \theta = \theta$ and $\cos \theta = 1 - \theta^2/2$, so that

$$\mathcal{R}_A(\theta) = \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 - \theta^2/2 & -\theta \\ 0 & \theta & 1 - \theta^2/2 \end{pmatrix}, \quad \mathcal{R}_B(\theta) = \begin{pmatrix} 1 - \theta^2/2 & 0 & \theta \\ 0 & 1 & 0 \\ -\theta & 0 & 1 - \theta^2/2 \end{pmatrix}.$$

To second order in θ ,

$$\begin{aligned} \mathcal{R}_B(-\theta)\mathcal{R}_A(-\theta) &= \begin{pmatrix} 1 - \theta^2/2 & 0 & -\theta \\ 0 & 1 & 0 \\ \theta & 0 & 1 - \theta^2/2 \end{pmatrix} \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 - \theta^2/2 & \theta \\ 0 & -\theta & 1 - \theta^2/2 \end{pmatrix} \\ &= \begin{pmatrix} 1 - \theta^2/2 & \theta^2 & -\theta \\ 0 & 1 - \theta^2/2 & \theta \\ \theta & -\theta & 1 - \theta^2 \end{pmatrix}. \end{aligned}$$

Then multiply the preceding by $\mathcal{R}_A(\theta)$ on the left and keep terms through θ^2 to get

$$\begin{aligned} \mathcal{R}_A(\theta)\mathcal{R}_B(-\theta)\mathcal{R}_A(-\theta) &= \begin{pmatrix} 1 & 0 & 0 \\ 0 & 1 - \theta^2/2 & -\theta \\ 0 & \theta & 1 - \theta^2/2 \end{pmatrix} \begin{pmatrix} 1 - \theta^2/2 & \theta^2 & -\theta \\ 0 & 1 - \theta^2/2 & \theta \\ \theta & -\theta & 1 - \theta^2 \end{pmatrix} \\ &= \begin{pmatrix} 1 - \theta^2/2 & \theta^2 & -\theta \\ -\theta^2 & 1 & 0 \\ \theta & 0 & 1 - \theta^2/2 \end{pmatrix}. \end{aligned}$$

Multiply the preceding by $\mathcal{R}_B(\theta)$ on the left and keep terms through θ^2 to get

$$\begin{aligned} \mathcal{R}_B(\theta)\mathcal{R}_A(\theta)\mathcal{R}_B(-\theta)\mathcal{R}_A(-\theta) &= \begin{pmatrix} 1 - \theta^2/2 & 0 & \theta \\ 0 & 1 & 0 \\ -\theta & 0 & 1 - \theta^2/2 \end{pmatrix} \begin{pmatrix} 1 - \theta^2/2 & \theta^2 & -\theta \\ -\theta^2 & 1 & 0 \\ \theta & 0 & 1 - \theta^2/2 \end{pmatrix} \\ &= \begin{pmatrix} 1 & \theta^2 & 0 \\ -\theta^2 & 1 & 0 \\ 0 & 0 & 1 \end{pmatrix}. \end{aligned}$$

But this last expression is equal to $\mathcal{R}_C(-\theta^2)$ through terms of order θ^2 .