Response of a Single DOF System

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Here follows a straightforward implementation of the (approximate) impulse-momentum relationship, valid for durations of the loading much shorter than the natural period of the system. The integral of a triangular *curve* is of course simply the area of the triangle...

```
from math import sqrt

m = 20.0 # kg
k = 800.0 # N/m
p_max = 1200.0 # N
Delta_t = 0.02 # s
h = 0.01 # s

w = sqrt(k/m) # rad/s

Delta_moment = p_max*Delta_t/2 # N \times s = kg m / s
initial_velocity = Delta_moment/m # m / s
amplitude_of_motion = initial_velocity/w # m
```

and the result is

0.1 Linear Acceleration

The damping, the time step, the formula for the modified stiffness.

```
| c = 0 # N / (m/s) - undamped system
| h = 0.01 # s
| x = k + 3*c/h + 6*m/h**2
```

The initial conditions

```
| x0 = 0.0 
| v0 = 0.0
```

The loading function and the initial acceleration

```
p0 = 0.0

p1 = 1200.0

p2 = 0.0

a0 = (p0 - k*x0 - c*v0) / m
```

We are ready for the first step, 1st the modified load, then the displacement increment, the velocity increment and the quantities at the end of the step

```
dp1 = (p1-p0) + c*(a0*h/2+3*v0) + m*(3*a0+6*v0/h)
dx1 = dp1/x
dv1 = 3*dx1/h - 3*v0 - a0*h/2
x1 = x0+dx1
v1 = v0+dv1
a1 = (p1 - k*x1 - c*v1) / m
```

and the 2nd step is just the first one with all the indices incremented by one...

```
dp2 = (p2-p1) + c*(a1*h/2+3*v1) + m*(3*a1+6*v1/h)
dx2 = dp2/x
dv2 = 3*dx2/h - 3*v1 - a1*h/2
x2 = x1+dx2
v2 = v1+dv2
a2 = (p2 - k*x2 - c*v2) / m
```

Our results (x_{max} happens during free vibrations and its amplitude is $\sqrt{x_2^2 + v_2^2/\omega_n^2}$).

 $v(t_2) = 598.402 \text{ mm/s}$ $a(t_2) = -0.240 \text{ m/s}^2$

 $x_{max} = 94.805 \text{ mm}$