

# 5.1 Linear mass-spring models

a lecture for MATH F302 Differential Equations

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for textbook: D. Zill, *A First Course in Differential Equations with Modeling Applications*, 11th ed.

## a good reason

- in Chapter 4 we solved 2nd-order linear DEs

$$ay'' + by' + cy \stackrel{*}{=} g(t)$$

- a good reason is that

anything that smoothly oscillates has \* for a model

- ① a mass suspended on a spring oscillates up and down
  - ② the current in an electrical circuit flows back-and-forth
  - ③ a pendulum swings back and forth
  - ④ the earth moves up and down in an earthquake
  - ⑤ magnetic field in a radio wave oscillates
  - ⑥ a drum-head vibrates
  - ⑦ a photon is
- 5.1 and 5.3 slides cover ① – ③

## 1st-order linear: no oscillation

- why is 2nd-order needed for oscillation?
- *background assumption*: laws of nature are autonomous
- 1st-order linear autonomous DEs cannot generate oscillation

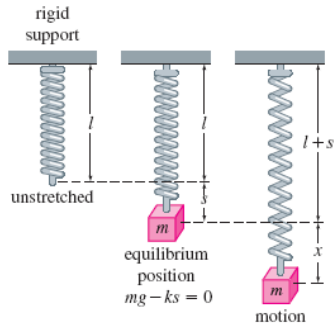
$$\begin{aligned}y' &= ay + b \\ \int \frac{dy}{ay + b} &= \int dt \\ \frac{1}{a} \ln |ay + b| &= t + c \\ y(t) &= \frac{1}{a} (Ce^{at} - b)\end{aligned}$$

- solutions are *always* growing/decaying exponentials
- 1st-order *nonlinear* DEs would be nearly-linear for small solutions
- summary: we expect oscillation models are 2nd-order
  - we know examples:  $y'' + y = 0 \iff y = c_1 \cos t + c_2 \sin t$

## mass-spring model: the setup

a specific set-up so that the equations are clear:

- hang spring from rigid support
  - length  $\ell$  and spring constant  $k$
- choose mass  $m$  and hook to the spring
- it stretches distance  $s$  down to equilibrium position
- mark length scale:
  - $x = 0$  is equilibrium position
  - positive  $x$  is downward
- $x$  is the displacement from additional stretch of the spring, i.e. *downward displacement of the mass from its equilibrium position*



## Newton's law

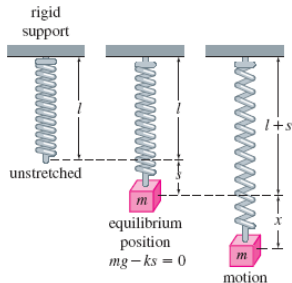
- Newton's second law is  $ma = F$
- for our first mass-spring model:

$$m \frac{d^2x}{dt^2} = mg - k(x + s)$$

- but  $mg = ks$  so

$$m \frac{d^2x}{dt^2} = -kx$$

- “Hooke's law”  $F_{spring} = -kx$  is a *model* for how springs work
  - not a bad model for small motions
  - improved model in 5.3
- in practice:  
 $k$  is determined from  $mg = ks$



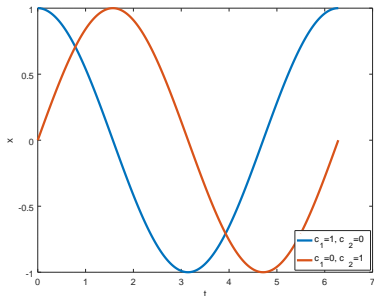
## (undamped) mass-spring solution

- from last slide:  $m \frac{d^2x}{dt^2} + kx = 0$
- constant coefficient: substitute  $x(t) = e^{rt}$  and get

$$mr^2 + k = 0 \quad \Longleftrightarrow \quad r = \pm \sqrt{\frac{k}{m}}i = \pm \omega i$$

- $\omega = \sqrt{\frac{k}{m}}$
- general solution:

$$x(t) = c_1 \cos(\omega t) + c_2 \sin(\omega t)$$



## the meaning of $\omega$

- general solution:  $x(t) = c_1 \cos(\omega t) + c_2 \sin(\omega t)$
- suppose  $t$  is measured in seconds
- then  $\omega = \sqrt{\frac{k}{m}}$  is *frequency of oscillation* in radians per second
  - units are correct because  $\omega t$  must be in radians
- time  $T = \frac{2\pi}{\omega}$  is *period of oscillation*
  - equation  $\omega T = 2\pi$  gives the smallest  $T > 0$  so that

$$\cos(\omega T) = \cos(0) \quad \text{and} \quad \sin(\omega T) = \sin(0)$$

- ... general solution has period  $T$

## exercise #3 in §5.1

- we are ready for a “free undamped motion” example
3. *A mass weighing 24 pounds, attached to the end of a spring, stretches it 4 inches. Initially the mass is released from rest from a point 3 inches above the equilibrium position. Find the solution for the motion.*



## mass/weight English unit stupidity

- “kilograms” is the SI unit for *mass*  $m$ 
  - $g = 9.8 \text{ m/s}^2$  is acceleration of gravity
  - $mg$  is a force in newtons  $N = \text{kg m/s}^2$
- “pounds” is a unit for *force*  $mg$ 
  - it is a *weight* not a mass
- “slugs” are a unit for *mass*  $m$ 
  - old English system . . .
  - and you need:  $g = 32 \text{ ft/s}^2$

## amplitude and phase of $x(t)$

- for any  $c_1, c_2$ , this formula is a wave or oscillation:

$$x(t) = c_1 \cos \omega t + c_2 \sin \omega t$$

- what is its amplitude?
  - only an easy question if either  $c_1 = 0$  or  $c_2 = 0$

**Problem:** find *amplitude*  $A$  and *phase angle*  $\phi$  so that

$$x(t) = c_1 \cos \omega t + c_2 \sin \omega t = A \sin(\omega t + \phi)$$

**Solution:** use  $\sin(a + b) = \sin a \cos b + \cos a \sin b$  so

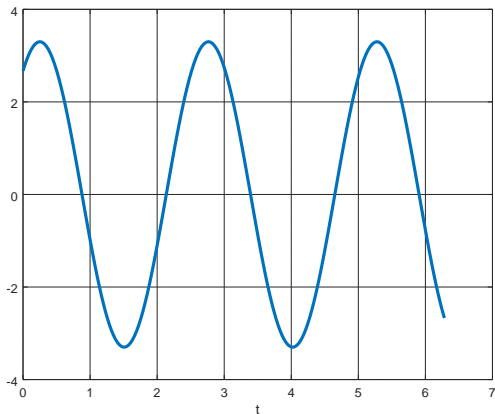
$$A \sin(\omega t + \phi) = A \sin(\omega t) \cos \phi + A \cos(\omega t) \sin \phi$$

$$\implies c_1 = A \sin \phi, \quad c_2 = A \cos \phi$$

$$\implies A^2 = c_1^2 + c_2^2, \quad \tan \phi = \frac{c_1}{c_2}$$

## illustration

- *example:* graph  $x(t) = A \sin(\omega t + \phi)$  for frequency  $\omega = 2.7$ , amplitude  $A = 3.3$ , and phase angle  $\phi = 0.3\pi$ 
  - period  $T = 2\pi/\omega = 2.51$
  - $x(t) = 2.67 \cos(\omega t) + 1.94 \sin(\omega t)$



## exercise #6 in §5.1

- another “free undamped motion” exercise

*6. A force of 400 newtons stretches a spring 2 meters. A mass of 50 kilograms is attached to the end of the spring and is initially released from the equilibrium position with an upward velocity of 10 m/s. Find the motion  $x(t)$ .*

# damped mass-spring model

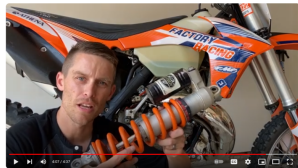
- actual mass-springs don't oscillate forever
- friction or drag is called “damping”
  - simple case: mass is surrounded by water or other fluid
- *model*: damping is proportional to velocity

$$F_{\text{damping}} = -\beta v = -\beta \frac{dx}{dt}$$

- $\beta > 0$  so damping force opposes motion
- same model as drag force for projectiles in sections 1.3, 3.1
- Newton's 2nd law again:

$$\boxed{m \frac{d^2x}{dt^2} = -kx - \beta \frac{dx}{dt}}$$

$$\text{or } mx'' = -kx - \beta x'$$



[youtu.be/IPg695IXbPo](https://youtu.be/IPg695IXbPo)

## damped solution method

- recall *undamped* mass-spring model with  $\omega = \sqrt{\frac{k}{m}}$ :

$$mx'' = -kx \quad \Longleftrightarrow \quad x'' + \omega^2 x = 0$$

- new damped mass-spring model:

$$mx'' = -kx - \beta x' \quad \Longleftrightarrow \quad x'' + 2\lambda x' + \omega^2 x = 0$$

- $\lambda = \frac{\beta}{2m}$
- auxiliary equation from  $x(t) = e^{rt}$ :

$$r^2 + 2\lambda r + \omega^2 = 0$$

- has roots:

$$r = \frac{-2\lambda \pm \sqrt{4\lambda^2 - 4\omega^2}}{2} = -\lambda \pm \sqrt{\lambda^2 - \omega^2} = r_1, r_2$$

- are  $r_1, r_2$  distinct? real? complex?

## exercise #27 in §5.1

- “free damped motion” exercise

*27. A 1 kilogram mass is attached to a spring whose constant is 16 N/m. The entire system is submerged in a liquid that imparts a damping force numerically equal to 10 times the instantaneous velocity. Determine the equations of motion if the mass is initially released from rest from a point 1 meter below the equilibrium position.*

## slight variation comes out different

*A 1 kilogram mass is attached to a spring whose constant is 16 N/m. The entire system is submerged in a liquid that imparts a damping force numerically equal to 6 times the instantaneous velocity. Determine the equations of motion if the mass is initially released from rest from a point 1 meter below the equilibrium position.*



## damping cases

$$\frac{d^2x}{dt^2} + 2\lambda \frac{dx}{dt} + \omega^2 x = 0$$

- *undamped* if  $\lambda = 0$ :

$$x(t) = c_1 \cos(\omega t) + c_2 \sin(\omega t)$$

- *overdamped* if  $\lambda^2 - \omega^2 > 0$ :

$$r_1, r_2 = -\lambda \pm \sqrt{\lambda^2 - \omega^2}$$

$$x(t) = e^{-\lambda t} \left( c_1 e^{\sqrt{\lambda^2 - \omega^2} t} + c_2 e^{-\sqrt{\lambda^2 - \omega^2} t} \right)$$

- *critically damped* if  $\lambda^2 - \omega^2 = 0$ :

$$r_1 = r_2 = -\lambda$$

$$x(t) = e^{-\lambda t} (c_1 + c_2 t)$$

- *underdamped* if  $\lambda^2 - \omega^2 < 0$ :

$$r_1, r_2 = -\lambda \pm \sqrt{\omega^2 - \lambda^2} i$$

$$x(t) = e^{-\lambda t} \left( c_1 \cos(\sqrt{\omega^2 - \lambda^2} t) + c_2 \sin(\sqrt{\omega^2 - \lambda^2} t) \right)$$

## damping cases pictured

- consider  $m = 1$ ,  $k = 4$

- $\omega = \sqrt{\frac{k}{m}} = 2$ :

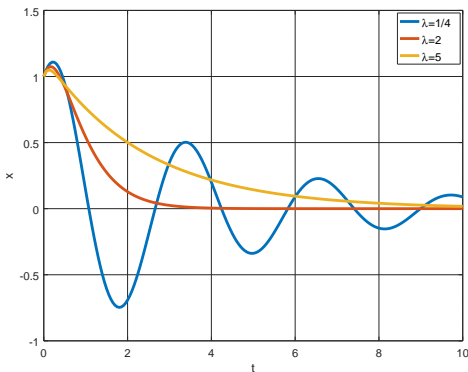
$$\frac{d^2x}{dt^2} + 2\lambda \frac{dx}{dt} + 4x = 0$$

- with initial values  
 $x(0) = 1, x'(0) = 1$

- picture cases

$$\lambda = 1/4, 2, 5$$

- recall  $\lambda = \frac{\beta}{2m}$
- so  $\beta = 1/2, 4, 10$



## a plotting code: massspringplot.m

```
function massspringplot(m,beta,k,x0,v0,T)
% MASSSPRINGPLOT Make a plot on  $0 < t < T$  of solution to
%  $m x'' + \beta x' + k x = 0$ 
% with initial conditions  $x(0) = x_0$ ,  $x'(0) = v_0$ .

omega = sqrt(k/m); lambda = beta/(2*m);
D = lambda^2 - omega^2;
t = 0:T/200:T; % 200 points enough for smooth graph
if D > 0
    fprintf('overdamped\n')
    Z = sqrt(D); c = [1, 1; -lambda+Z, -lambda-Z] \ [x0; v0];
    x = exp(-lambda*t) .* (c(1) * exp(Z*t) + c(2) * exp(-Z*t));
elseif D == 0
    fprintf('critically damped\n')
    c = [x0; v0 + lambda * x0];
    x = exp(-lambda*t) .* (c(1) + c(2) * t);
else % D < 0
    fprintf('underdamped\n')
    W = sqrt(-D); c = [x0; (v0 + lambda * x0) / W];
    x = exp(-lambda*t) .* (c(1) * cos(W*t) + c(2) * sin(W*t));
end
plot(t,x), grid on, xlabel('t'), ylabel('x')
```

## example

*example:* solve the IVP

$$mx'' = -kx - \beta x', \quad x(0) = x_0, \quad x'(0) = v_0$$

in the critically-damped case

## forced

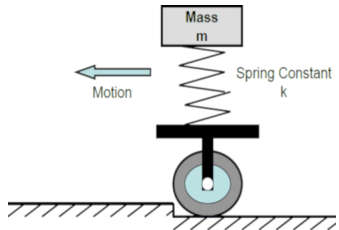
- the nonhomogeneous version is called a *driven*, damped mass-spring where force  $f(t)$  is applied to the mass:

$$m \frac{d^2 x}{dt^2} = -kx - \beta \frac{dx}{dt} + f(t)$$

- equivalently, after dividing by  $m$ :

$$\frac{d^2 x}{dt^2} + 2\lambda \frac{dx}{dt} + \omega^2 x = F(t)$$

- a version of this model is a damped mass-spring formed by your car
  - force is applied to the support and your car is the mass



## mass-spring DEs

	Newton's law: $ma = F$	$\omega$ form
undamped	$m \frac{d^2x}{dt^2} = -kx$	$\frac{d^2x}{dt^2} + \omega^2 x = 0$
damped	$m \frac{d^2x}{dt^2} = -kx - \beta \frac{dx}{dt}$	$\frac{d^2x}{dt^2} + 2\lambda \frac{dx}{dt} + \omega^2 x = 0$
damped and driven	$m \frac{d^2x}{dt^2} = -kx - \beta \frac{dx}{dt} + f(t)$	$\frac{d^2x}{dt^2} + 2\lambda \frac{dx}{dt} + \omega^2 x = F(t)$

notes:

- $\omega = \sqrt{k/m}$ ,  $\lambda = \beta/(2m)$ ,  $F(t) = f(t)/m$
- with driving force  $f(t)$  the problem is *nonhomogeneous*
- you would solve the damped and driven problems by undetermined coefficients to find a particular solution (section 4.4)

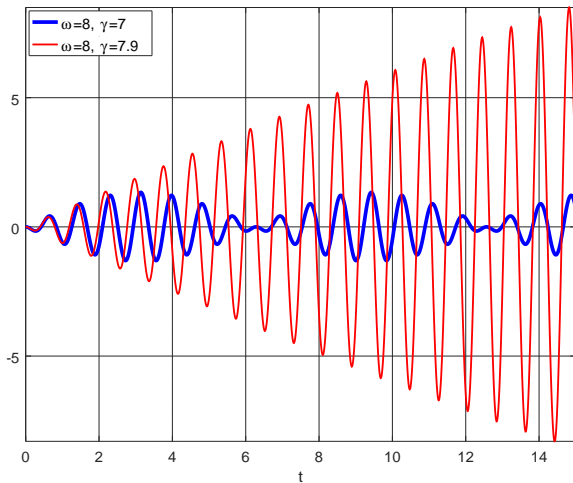
exercise #43 in §5.1

*Solve the IVP*

$$\frac{d^2x}{dt^2} + \omega^2 x = F_0 \cos \gamma t, \quad x(0) = 0, \quad x'(0) = 0$$

*and compute  $\lim_{\gamma \rightarrow \omega} x(t)$*

## exercise #43 pictured

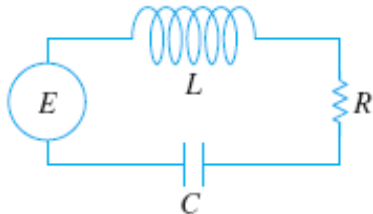


- idea: **resonance** can occur in driven mass-spring systems



## RLC circuit

- consider the electrical circuit:



- has electrical source ( $E = E(t)$ ), an inductor ( $L$ ), a resistor ( $R$ ), and a capacitor ( $C$ )
- a differential equation for the *charge*  $q$  is

$$L \frac{d^2 q}{dt^2} + R \frac{dq}{dt} + \frac{1}{C} q = E(t)$$

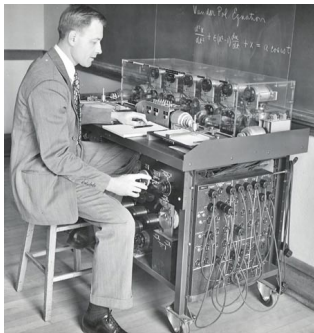
- because  $dq/dt = I$ , a differential equation for the *current*  $I$  is

$$L \frac{d^2 I}{dt^2} + R \frac{dI}{dt} + \frac{1}{C} I = E'(t)$$

## circuit analogy

mass-spring	electrical circuit
mass $m$	inductance $L$
drag $\beta$	resistance $R$
spring constant $k$	inverse of capacitance $1/C$
applied driving force $f(t)$	applied voltage source $E(t)$
$mx'' + \beta x' + kx = f(t)$	$Lq'' + Rq' + \frac{1}{C}q = E(t)$

- this is how radios are understood
  - *tuning a radio* means choosing the capacitance  $C$  to cause resonance at the frequency you want to hear from the input  $E(t)$  from the antenna
- based on this idea there were *analog computers* which used a configurable electrical circuit to model mechanical motions



## expectations

to learn this material, just listening to a lecture is *not* enough

- *read* section 5.1 in the textbook
  - material on “double spring systems” (p. 201) can be skipped
  - while I discussed electrical circuits in these slides, I will not ask about it on quizzes or exams
- do Homework 5.1