

EE132 Automatic Control

Lab 5: Ball Beam & Root Locus

Buddy Ugwumba, Chris Hwang, Nick Andrade

Objective:

Use Root Locus method to design a controller or compensator for a ball and beam system. Our plant is the ball and beam system represented by the following transfer function:

$$G(s) = \frac{x(s)}{\theta(s)} = \frac{7}{s^2} \text{ <---- Double integrator system}$$

where: $x(s)$, $x(t)$, $\theta(s)$, $\theta(t)$ are the respective position and angle of the ball and beam system in the s- and t-domain. With this in mind, we would like to design a controller that generates $\theta(t)$ and we will use that controller to control the double integrator.

Controller Objective

- in unity feedback loop
- design a controller: $K(s) = \frac{\theta(s)}{E(s)}$ which will take the error signal as input
- Satisfy the following conditions:

1) $2 \leq T_s \leq 4$

2) Tracking error must be zero

3) The desired position is a step function with the initial value of $\theta(t) \leq 30$. Then we will use the initial value theorem to determine the initial value.

Use root locus methods to design a cascade compensator for the ball and beam system.

2) root locus

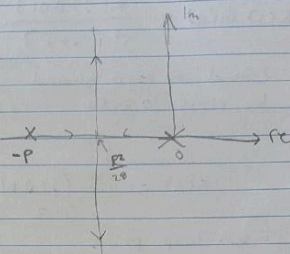
breakout point: $P_d = -\frac{p}{2}$

$$1 + K(s)G(s) = 0$$

or

$$\begin{cases} |K(s)G(s)| = 1 \\ \angle K(s)G(s) = 180^\circ \end{cases}$$

$$K_d = \frac{p^2}{2b}$$



3) Specification 1

$$-2.3 \leq \sigma \leq -1.15$$

choose

choose complex poles for the system

$$-2.3 \leq -\zeta/\omega_n \leq -1.15 \Rightarrow 4.6 \geq \zeta \geq 2.3$$

4) Specification 3

$$K < 0.5236$$

for complex pole, $K > K_d$

$$\frac{p^2}{2b} < K < 0.5236 \quad \Leftarrow \quad \frac{p^2}{2b} < 0.5236$$

$$5) \quad 2.3 \leq p \leq 3.8289$$

pick p

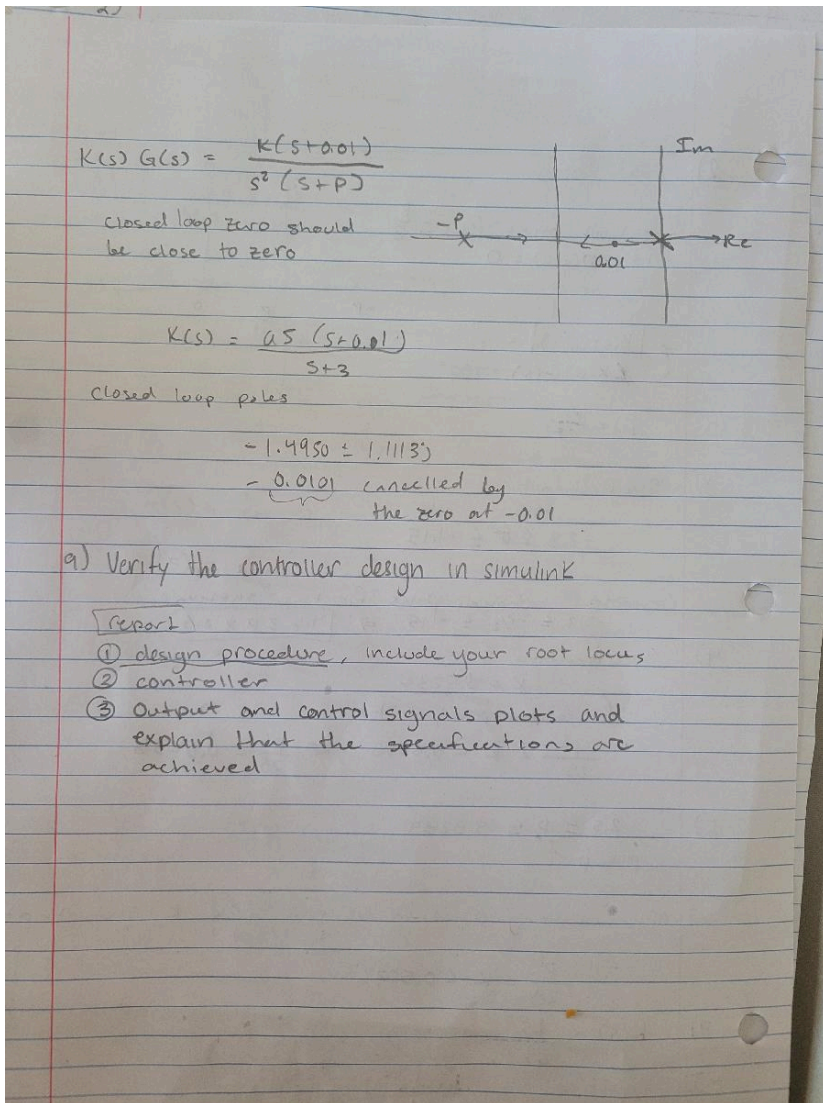
6) After choosing p, then we can pick K such that

$$\frac{p^2}{2b} < K < 0.5236$$

$$7) \quad K(s) = \frac{Ks}{s+p}$$

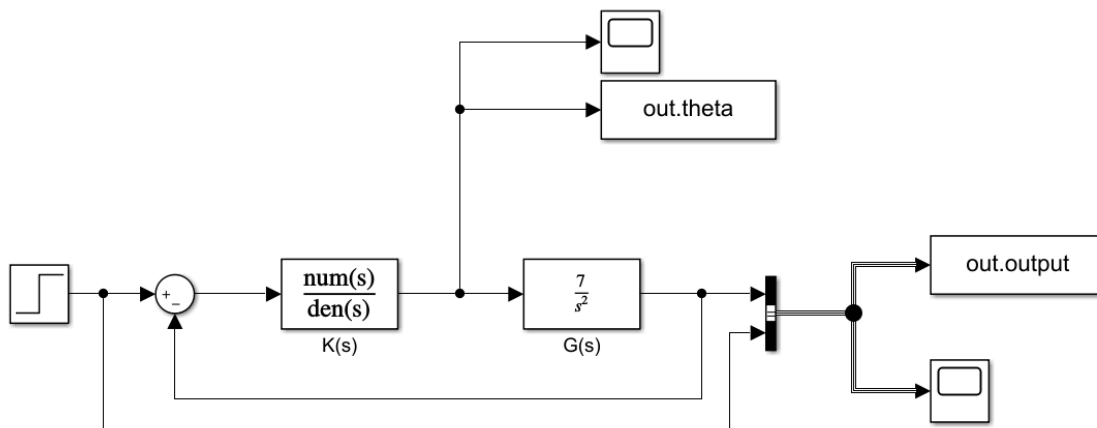
$$8) \quad \text{replace } s \text{ with } s+0.01$$

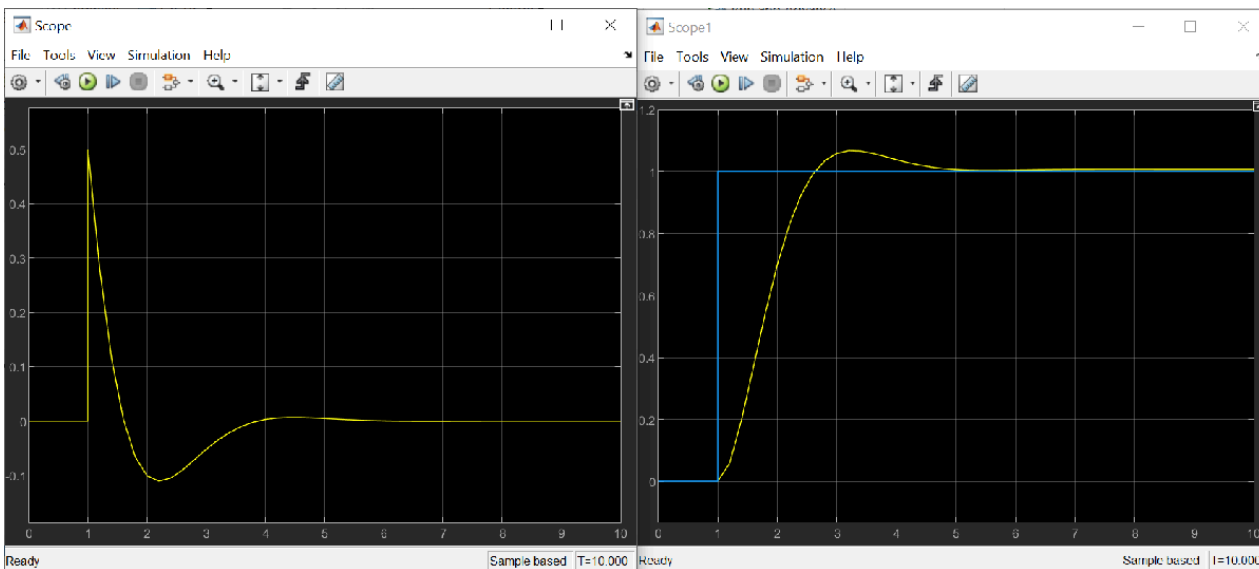
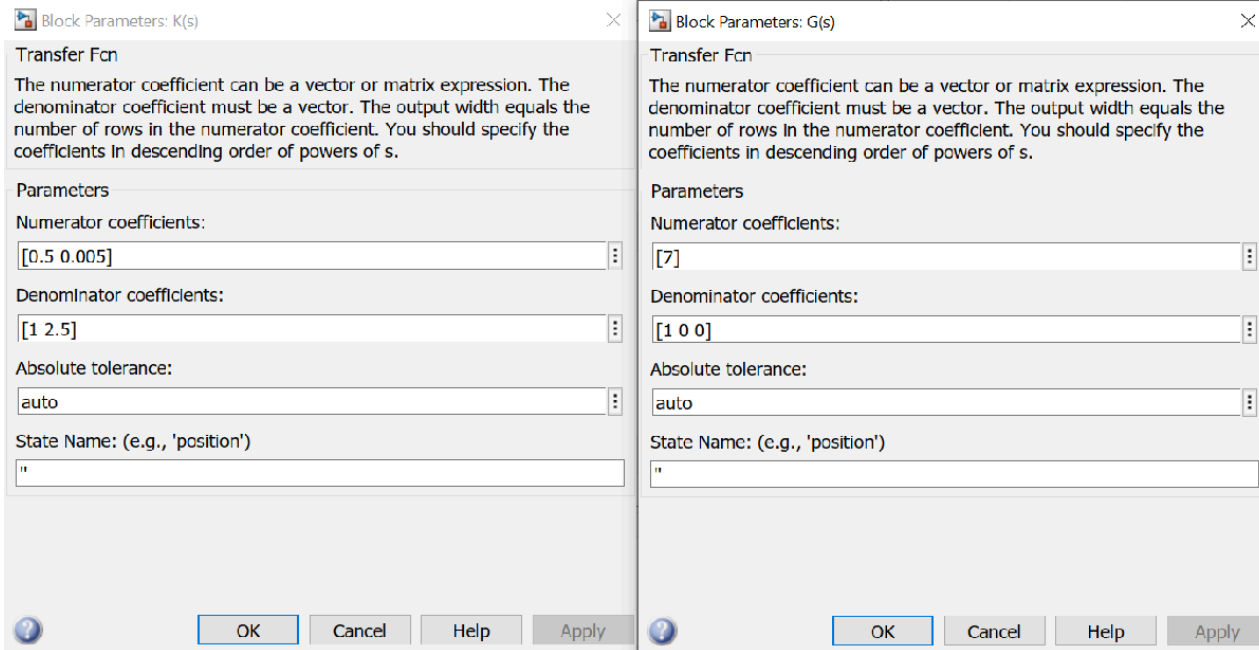
$$K(s) = \frac{K(s+0.01)}{s+p}$$



k = 0.5

p = 2.5





Regarding the figure on the right, we can see that our desired value for both K and p were correct because our controller meets the design specifications outlined in the lab; e.g. the settling time must be greater than or equal to 2 and less than or equal to 4. Moreover, we can see that the tracking error is none as t approaches infinity. Regarding the image on the left, we can see that our design specifications were met as well because the angle of the ball beam system is not greater than 30 degrees. Our calculations showed that $\pi/6$ is about 0.5 rads.

```
tf([3.5 0.035],[1 2.5 3.5 0.035])
```

```
ans =
```

```

      3.5 s + 0.035
-----
s^3 + 2.5 s^2 + 3.5 s + 0.035
```

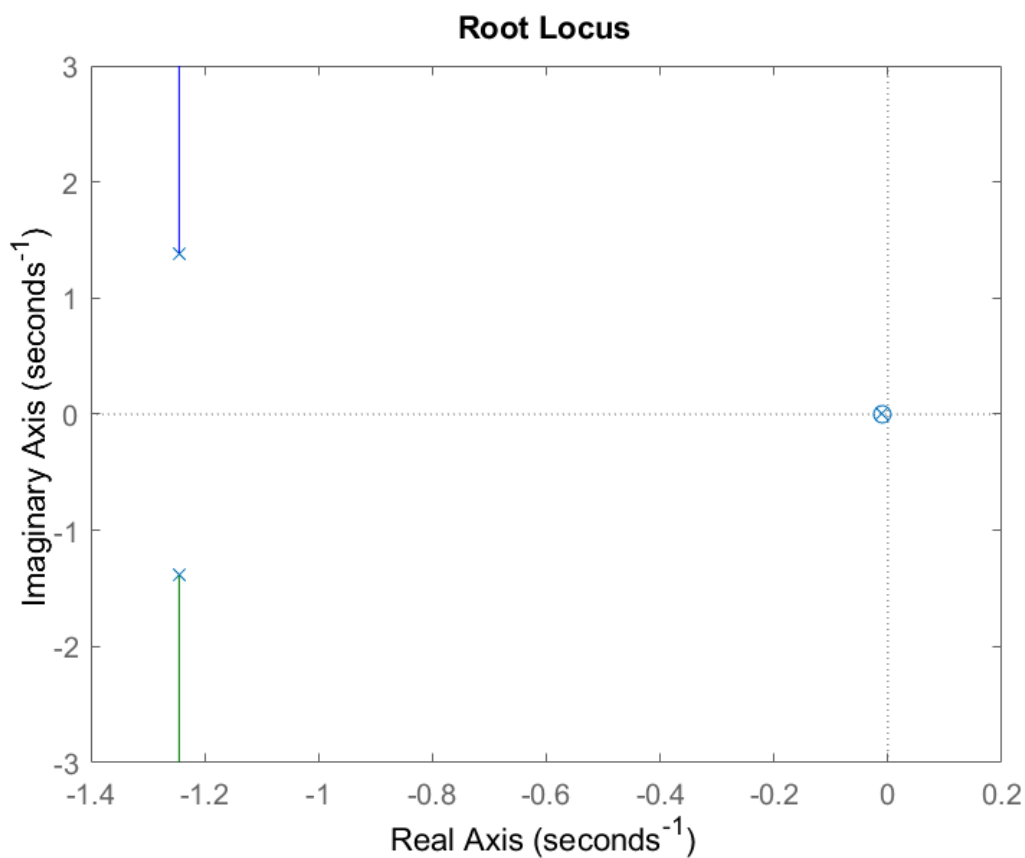
Continuous-time transfer function.

```
stepinfo(sys)
```

```
ans = struct with fields:
```

```
    RiseTime: 1.0771
    SettlingTime: 3.4019
    SettlingMin: 0.9106
    SettlingMax: 1.0672
    Overshoot: 6.7200
    Undershoot: 0
         Peak: 1.0672
    PeakTime: 2.2564
```

```
rlocus(sys)
```

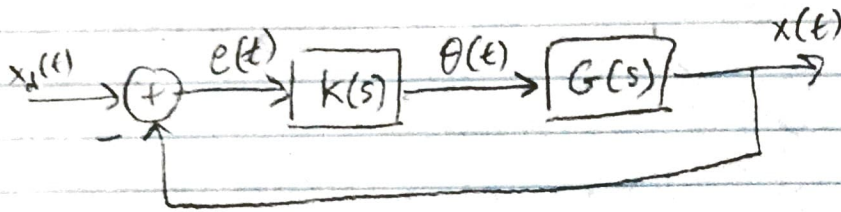


Concerning the root locus image we can see that our controller is stable because both the poles and zeros are strictly on the LHP. The complex conjugate pairs of poles tell us that this second order system is type 1.

Pre-lab 5

2)

$$e(t) = x_d(t) - x(t)$$



3)

$$\begin{cases} E(s) = x_d(s) - X(s) \\ \Theta(s) = K(s) \cdot E(s) \end{cases}$$

so

$$\frac{\Theta(s)}{X_d(s)} = \frac{K(s)}{1 + K(s)G(s)}$$

4)

$$\left. \Theta(s) \right|_{t=0} < 30^\circ \text{ or } \frac{\pi}{6} \text{ rad} \quad \leftarrow \text{Specification 3}$$

$$\Theta(s) = \frac{K(s)}{1 + K(s)G(s)} \cdot X_d(s) = \frac{1}{s} \cdot \frac{K(s)}{1 + K(s)G(s)}$$

$$G(s) = \frac{7}{s}$$

$$K(s) = \frac{K \prod_{i=1}^m (s - z_i)}{\prod_{j=1}^n (s - p_j)}$$

$$\left. \Theta(s) \right|_{t=0} = \lim_{s \rightarrow \infty} s \cdot \Theta(s) = \lim_{s \rightarrow \infty} \frac{K(s)}{1 + K(s)G(s)} = K$$

so

$$K < \frac{\pi}{6}$$

$$5) \frac{X(s)}{X_d(s)} = \frac{\theta(s) \cdot G(s)}{X_d(s)}$$

$$\frac{X(s)}{X_d(s)} = \frac{K(s) \cdot G(s)}{1 + K(s)G(s)}$$

$$6) \text{ specification } 2 \leq T_s \leq 4$$

$$T_s = \frac{4.6}{\sigma}$$

$$\sigma = \frac{4.6}{T_s}$$

$$\Rightarrow 1.15 \leq \sigma \leq 2.3$$

7) dominant poles

compare the real parts of poles since the real parts determine the speed of the response, decrease.

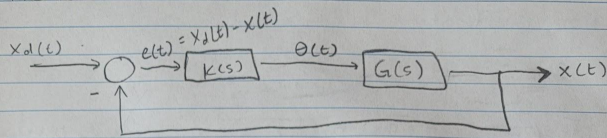
$$\text{poles} = -\zeta\omega_n \pm j\omega_n\sqrt{1-\zeta^2}$$

$$\text{step response } C(s) = \frac{A}{s} + \frac{B(s + \zeta\omega_n) + C\omega_d}{(s + \zeta\omega_n)^2 + \omega_d^2} + \frac{D}{s + \omega_r}$$

8) $K(s)G(s)$ should be type 1. Specification 2 is met. There would a 0 tracking error,

Pre-lab

2)



$$3) \begin{cases} E(s) = X_d(s) - X(s) \\ \Theta(s) = K(s) \cdot E(s) \end{cases}$$

$$\Rightarrow \frac{\Theta(s)}{X_d(s)} = \frac{K(s)}{1 + K(s)G(s)}$$

4) In specification 3:

$$\Theta(t) \big|_{t=0} < 30^\circ \text{ or } \frac{\pi}{6} \text{ rad}$$

$$\Theta(s) = \frac{K(s)}{1 + K(s)G(s)} * X_d(s) = \frac{1}{s} * \frac{K(s)}{1 + K(s)G(s)}$$

$$G(s) = \frac{\pi}{s^2}$$

$$K(s) = K \frac{\prod_{i=1}^M (s - z_i)}{\prod_{j=1}^N (s - p_j)} \quad \begin{array}{l} z_i, p_j - \text{zeros, poles} \\ K - \text{control gain} \end{array}$$

$$\Theta(t) \big|_{t=0} = \lim_{s \rightarrow \infty} s \cdot \Theta(s) = \lim_{s \rightarrow \infty} \frac{K(s)}{1 + K(s)G(s)}$$

$$= K$$

$$\therefore K < \frac{\pi}{6}$$

$$5) \frac{X(s)}{X_d(s)} = \frac{\Theta(s)G(s)}{X_d(s)} = \frac{K(s)G(s)}{1 + K(s)G(s)}$$

pole \rightarrow zero
reduces
damping
increases

For lag compensator
When done, poles
go closer to zero

more
poles
toward
zero
reduces
 S_s error

6) from specification 1

$$2 \leq T_s \leq 4s$$

$$T_s = \frac{4.6}{\sigma} \quad \sigma = \text{real part of pole / dominant pole}$$

$$\Rightarrow 1.15 \leq \sigma \leq 2.3$$

7) dominant poles:

compare the real parts of poles since
the real part determines how fast the response decreases

ax) poles:
$$\frac{-\zeta\omega_n \pm j\omega_n\sqrt{1-\zeta^2}}{-\sigma_r}$$

step response:

$$C(s) = \frac{A}{s} + \frac{B(s + \zeta\omega_n) + C\omega_d}{(s + \zeta\omega_n)^2 + \omega_d^2} + \frac{D}{s + \sigma_r}$$

$$c(t) = Au(t) + e^{-\sigma_r t} (B \cos \omega_d t + C \sin \omega_d t) + D e^{-\sigma_r t}$$

if $\sigma_r \gg \zeta\omega_n$

8) If Type 1 or Type 2 specification 1 is met
 $K(s)G(s)$ should be at least type 1

control design:

$$K(s) = \frac{K_s}{s+p}$$

$$K(s)G(s) = \frac{K}{s(s+p)} \quad \begin{matrix} K > 0 \\ p > 0 \end{matrix}$$

In practice, zero and pole cancellation is not allowed
unless the pole is strictly in LHP. Also, since the poles at zero
indicates the type of system, we usually do not cancel the pole at zero

$x(t) \rightarrow x_d(t)$ as $t \rightarrow \infty$

(3) $x_d(t)$ is a step function

the initial value of $\theta(t) \big|_{t=0} < 30^\circ$

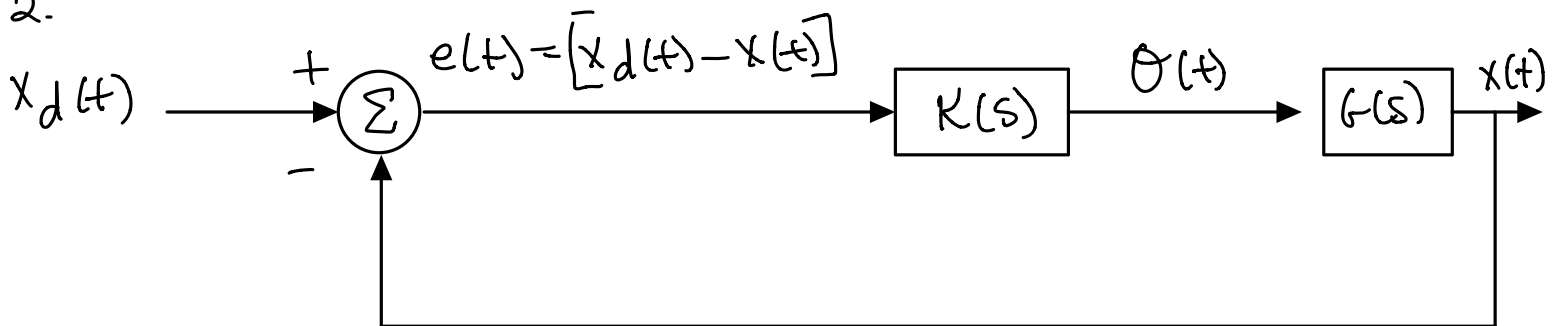
then we use initial value theorem to determine the initial value

$$\theta(t) \big|_{t=0} = \lim_{s \rightarrow 0} s(\theta(s))$$

To find constraints on the control input

Prelab

2.



3) $\frac{K(s)G(s)}{1 + K(s)G(s)} = T(s)$

? what happened to the $G(s)$ we're not finding the output

$$\begin{cases} E(s) = X_d(s) - X(s) \\ \theta(s) = K(s)G(s) \end{cases} \Rightarrow \frac{\theta(s)}{X_d(s)} = \frac{K(s)}{1 + K(s)G(s)}$$

4.)

$$\theta(t) \big|_{t=0} < 30^\circ \text{ or } \frac{\pi}{6}$$

$$O(s) = \frac{K(s)}{1 + K(s)G(s)} \cdot X_d(s) = \frac{1}{s} \cdot \frac{K(s)}{1 + K(s)G(s)}$$

$$G(s) = \frac{7}{s^2} \quad = \frac{1}{s} \cdot \frac{K(s)}{1 + K(s)} \cdot \frac{7}{s^2}$$

$$= K$$

$$\text{Thus } K < \frac{\pi}{6}$$

5.

$$\frac{X(s)}{X_d(s)} = \frac{O(s)G(s)}{X_d(s)}$$

$$= \frac{K(s)G(s)}{1 + K(s)G(s)}$$

6) From specifications

$$2 \leq T_s \leq 4s$$

Use constant of dominant pole to estimate settling time

$$T_s = \frac{4.6}{\sigma} \quad \sigma - \text{real part of pole}$$

$$1.1s \leq \sigma \leq 2.3$$

7) compare the real parts of poles since the real parts determine how fast the response decays

8) should be at least type 1