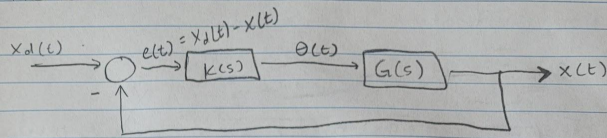


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Pre-lab

2)



$$3) \begin{cases} E(s) = X_d(s) - X(s) \\ \Theta(s) = K(s) \cdot E(s) \end{cases}$$

$$\Rightarrow \frac{\Theta(s)}{X_d(s)} = \frac{K(s)}{1 + K(s)G(s)}$$

4) In specification 3:

$$\Theta(t) \Big|_{t=0} < 30^\circ \text{ or } \frac{\pi}{6} \text{ rad}$$

$$\Theta(s) = \frac{K(s)}{1 + K(s)G(s)} \cdot X_d(s) = \frac{1}{s} \cdot \frac{K(s)}{1 + K(s)G(s)}$$

$$G(s) = \frac{7}{s^2}$$

$$K(s) = \frac{K \prod_{i=1}^M (s - z_i)}{\prod_{j=1}^N (s - p_j)} \quad \begin{array}{l} z_i, p_j - \text{zeros, poles} \\ K - \text{control gain} \end{array}$$

$$\Theta(t) \Big|_{t=0} = \lim_{s \rightarrow \infty} s \cdot \Theta(s) = \lim_{s \rightarrow \infty} \frac{K(s)}{1 + K(s)G(s)}$$

$$= K$$

$$\therefore K < \frac{\pi}{6}$$

$$5) \frac{X(s)}{X_d(s)} = \frac{\Theta(s)G(s)}{X_d(s)} = \frac{K(s)G(s)}{1 + K(s)G(s)}$$

pole > zero
reduces
damping
increases factor

For lag compensator
When done, poles
go closer to zero

more
poles
toward
zero
reduces
SS error

6) from specification 1

$$2 \leq T_s \leq 4s$$

$$T_s = \frac{4.6}{\sigma} \quad \sigma - \text{real part of pole / dominant pole}$$

$$\Rightarrow 1.15 \leq \sigma \leq 2.3$$

7) dominant poles:

compare the real parts of poles since
the real part determines how fast the response decreases

ax) poles:
$$\frac{-\zeta\omega_n \pm j\omega_n\sqrt{1-\zeta^2}}{-a_r}$$

step response:

$$C(s) = \frac{A}{s} + \frac{B(s + \zeta\omega_n) + C\omega_d}{(s + \zeta\omega_n)^2 + \omega_d^2} + \frac{D}{s + a_r}$$

$$c(t) = Au(t) + e^{-\zeta\omega_n t} (B\cos\omega_d t + C\sin\omega_d t) + D e^{-a_r t}$$

if $a_r \gg \zeta\omega_n$

8) If Type 1 or Type 2 specification 1 is met
 $K(s)G(s)$ should be at least type 1

control design:

$$K(s) = \frac{K_s}{s+p}$$

$$K(s)G(s) = \frac{K}{s(s+p)} \quad \begin{matrix} K > 0 \\ p > 0 \end{matrix}$$

In practice, zero and pole cancellation is not allowed
unless the pole is strictly in LHP. Also, since the poles at zero
indicates the type of system, we usually do not cancel the pole at zero