

Summary

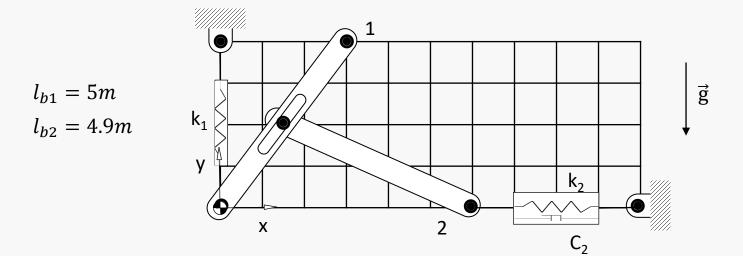
Exercises on the kinematic and dynamic analysis of mechanical systems using Cartesian Coordinates.

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Exam 2014 Part I

The two bodies shown in the figure are connected by a pin in a slot and are acted upon by springs and dampers. All data are supplied, though you are supposed to answer all questions without substituting the variables by their numerical value or performing any numerical calculations (except for f)).

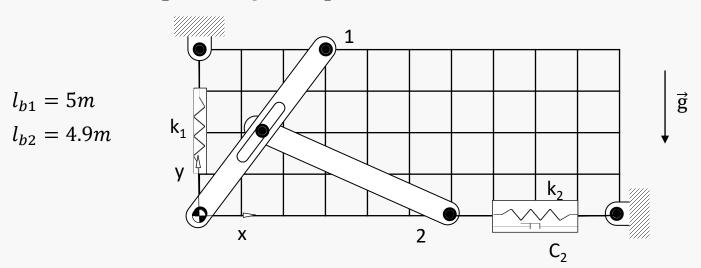
$$m_1 = 5kg$$
 $m_2 = 6kg$ $k_1 = 20N/m$ $k_2 = 30N/m$ $C_2 = 6Ns/m$ $J_1 = 10kgm^2$ $J_2 = 12.5kgm^2$ $l_1^0 = 5m$ $l_2^0 = 4.5m$





Exam 2014 Part I

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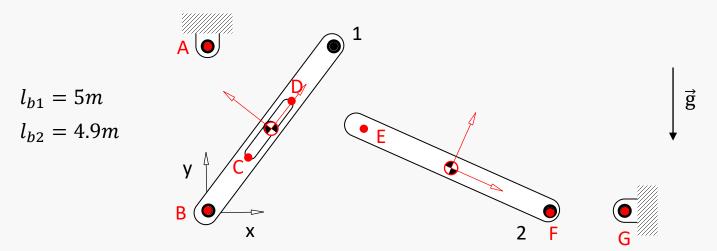
- a) Identify all bodies, locate the body-fixed coordinate frames and identify the coordinates of all points required to formulate the kinematic joints (1 pt).
- b) Write the equations of the constraints for the kinematic joints. For the kinematic constraints identified, formulate the Jacobian matrix and the right-hand side vectors of the velocity and acceleration equations (2 pts).
- c) Write the equations of motion of the system, identifying, in the process, the force vectors (1 pt).



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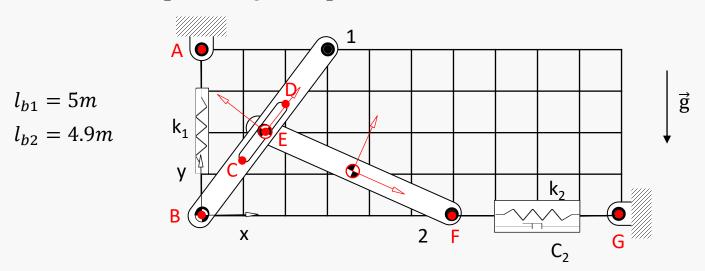


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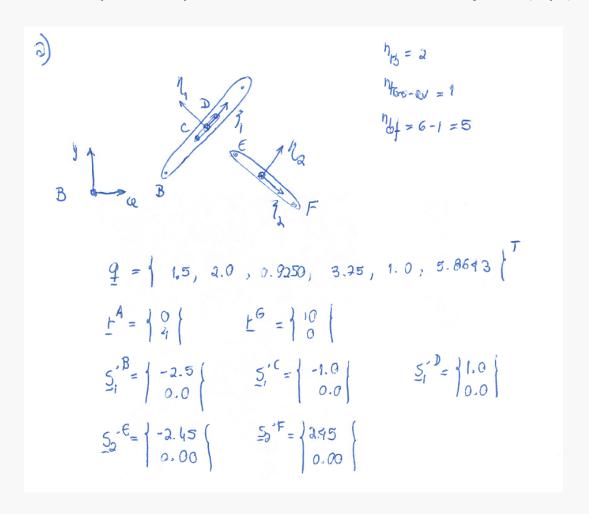
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a) Identify all bodies, locate the body-fixed coordinate frames and identify the coordinates of all points required to formulate the kinematic joints (1 pt).





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b) Write the equations of the constraints for the kinematic joints. For the kinematic constraints identified, formulate the Jacobian matrix and the right-hand side vectors of the velocity and acceleration equations (2 pts).

Vectors for the formulation of the translation -revolute joint

Tendation axis in body
$$1 \Rightarrow 5_1 = \underline{r_1}^D - \underline{r_1}^C = 4_1 \left(\underline{s_1}^T - \underline{s_1}^C \right)$$

Vector perpendicular to the translation vector $= b_1 = A_{90} \cdot \underline{s_1}$, where $A_{90} = \begin{bmatrix} 0 & -1 \\ 1 & 0 \end{bmatrix}$

Vector from a point in translation axis $= \underline{r_1}^D - \underline{r_2}^C = 1$

to point in translation axis $= \underline{r_1}^D + \underline{r_2}^C = 1$

The constraint for the translation -revolvte joint is given by

 $A_{10} = A_{10} \cdot \underline{s_1}^C = 1$



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b) Write the equations of the constraints for the kinematic joints. For the kinematic constraints identified, formulate the Jacobian matrix and the right-hand side vectors of the velocity and acceleration equations (2 pts).

The velocity equations are obtained from the first derivative of the Constraints with respect to time:

$$\dot{D} = 0 \implies \dot{c}^{T} \dot{h}_{1} + \dot{h}_{1}^{T} \dot{c}^{\dagger} = 0 \implies \\
\iff \dot{c}^{T} \left(Ag_{0} B_{1} \left(S_{1}^{*} \dot{b} - S_{1}^{*} \dot{c}^{\dagger} \right) \dot{\theta}_{1} \right) + \dot{h}_{1}^{T} \left(\dot{f}_{1} + B_{1} S_{1}^{*} \dot{\theta}_{1}^{\dagger} - \dot{f}_{2}^{\dagger} - B_{3} S_{3}^{*} \dot{\theta}_{3}^{\dagger} \right) = 0 \implies \\
\iff \dot{b}^{T} \quad \dot{c}^{T} Ag_{0} B_{1} \left(S_{1}^{*} \dot{b} - S_{1}^{*} \dot{c}^{\dagger} \right) + \dot{h}_{1}^{T} B_{1} S_{1}^{*} \dot{b} - \dot{h}_{1}^{T} - \dot{h}_{1}^{T} B_{2} S_{3}^{*} \dot{\theta}_{3}^{\dagger} \right) = 0 \implies \\
\Leftrightarrow \dot{b}^{T} \quad \dot{c}^{T} Ag_{0} B_{1} \left(S_{1}^{*} \dot{b} - S_{1}^{*} \dot{c}^{\dagger} \right) + \dot{h}_{1}^{T} B_{2} S_{3}^{*} \dot{\theta}_{3}^{\dagger} - \dot{h}_{1}^{T} B_{2} S_{3}^{*} \dot{\theta}_{3}^{\dagger} \right) = 0 \implies \\
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b) Write the equations of the constraints for the kinematic joints. For the kinematic constraints identified, formulate the Jacobian matrix and the right-hand side vectors of the velocity and acceleration equations (2 pts).

The acceleration equations are:

$$\vec{D} = 0$$

$$\Rightarrow \underline{c}^{T} \vec{b}_{1} + \underline{b}_{1}^{T} \vec{c}_{2}^{T} + \underline{b}_{1}^{T} \vec{c}_{2}^{T} + \underline{c}_{1}^{T} \vec{c}_{2}^{T} + \underline{c}_{2}^{T} \vec{b}_{1}^{T} = 0$$

$$\Rightarrow \underline{c}^{T} \vec{b}_{1} + \underline{b}_{1}^{T} \vec{c}_{2}^{T} = -2\underline{c}^{T} \vec{b}_{1}^{T} = 0$$

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$$\Rightarrow \underline{c}^{T} \vec{b}_{1} + \underline{b}_{1}^{T} \vec{c}_{1}^{T} + \underline{b}_{1}^{T} \vec{c}_{1}^{T} = -2\underline{c}^{T} \vec{b}_{1}^{T} = 0$$

$$\Rightarrow \underline{c}^{T} \vec{b}_{1} + \underline{b}_{1}^{T} \vec{c}_{1}^{T} \vec{c}_{1}^{T} + \underline{b}_{1}^{T} \vec{c}_{1}^{T} + \underline{$$



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c) Write the equations of motion of the system, identifying, in the process, the force vectors (1 pt).

e) The equations of notion for the system are:

$$\begin{bmatrix}
M & \text{Implies of notion for the system are:} \\
\text{Implies o$$



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c) Write the equations of motion of the system, identifying, in the process, the force vectors (1 pt).

the fore vector for each body inclodes the gratitational force and the spring-compartors:

$$g_i = g_{gai} + g_{gbi}, \quad \text{where} \quad g_{gai} = \begin{cases} m_i g_g \\ 0 \end{cases}$$
For a generic spring compart system
$$\frac{d}{d} = r_i + A_i s_i r_i - r_j - A_j s_i r_j \qquad \text{where} \qquad g_{gai} = \begin{cases} m_i g_g \\ 0 \end{cases}$$

$$U = \frac{d}{d} \qquad \qquad U = (d^T d)^{1/2} \qquad \qquad U =$$



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c) Write the equations of motion of the system, identifying, in the process, the force vectors (1 pt).

For the spring system 1

$$\int_{S} ds_{1} = K_{1} \left(I_{1} - I_{1}^{0} \right) U_{1} \quad , \quad \text{where} \qquad \qquad C_{1} = L^{A} - I_{3} - I_{4} S_{1}^{B} \\
I_{1} = \left(C_{1}^{T} C_{1}^{B} \right)^{1/2} \\
U_{1} = \frac{C_{1}}{I_{1}} \\
\text{The force vector of body 1 is } g_{1} = \int_{1}^{1} m_{1} \, dg + \int_{S} ds_{1} \int_{1}^{1} \cdot \sin c \, dL \text{ force of the spring is not exting on the center of mass, the equivalent system of the center of mass is the force itself plus a transport nument. This transport moment is given by

$$\Omega_{1} = S_{1}^{B} \times \int_{S} ds_{1} = \int_{1}^{1} m_{1} \, dg + \int_{1}^{1} S \, ds_{1} \, ds_{2} \, ds_{3} \, ds_{3$$$$



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for the spring-compar system a

$$\int_{Sd_2} = k_2 \left(l_2 - l_2^{\circ} \right) \underline{u}_2 + l_2 \, l_2^{\circ} \, \underline{u}_2, \quad \text{where} \quad \underline{l}_2 = \underline{l}_3 + \underline{l}_2 \, \underline{s}_4^{\circ} \, \overline{l}_{-\underline{l}}^{\circ} \, \underline{l}_3 = \left(\underline{l}_4^{\top} \cdot \underline{l}_a \right)^{\prime \underline{k}} \\
\underline{l}_2 = \underline{u}_2^{\top} \cdot \underline{l}_2 \\
\underline{l}_2 = \underline{r}_2^{\cdot} + \underline{B}_2 \, \underline{s}_3^{\circ} \, \overline{l}_3^{\circ} \\
\underline{u}_2 = \underline{l}_2 \\
\underline{l}_2$$

The fora vector of body a is $\underline{g}_2 = \left(\underline{m}_2 \, \underline{g}_2 - \underline{f}_3 \, \underline{s}_2 \right) \\
\underline{f}_3^{\dagger} \cdot \underline{f}_3 \cdot \underline{s}_4^{\circ} = \underline{f}_3^{\circ} \cdot \underline{f}_3^{\circ} \cdot \underline{f}_3^{\circ} = \underline{f}_3^{\circ} \cdot \underline{f}_3^{\circ} = \underline{f}_3^{\circ} \cdot \underline{f}_3^{\circ} = \underline{f}_3^{\circ} \cdot \underline{f}_3^{\circ} = \underline{f}_3^{\circ} \cdot \underline{f}_3^$



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c) Write the equations of motion of the system, identifying, in the process, the force vectors (1 pt).

The vector of externally applied forces is given by:

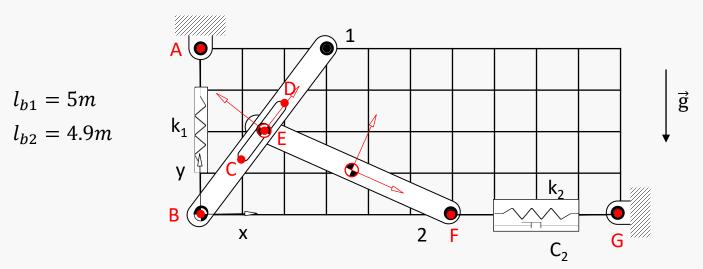
$$\frac{g}{2} = \left\{ \begin{array}{c} m, \underline{a}g + \underline{f}_{5}\underline{b}_{1} \\ \underline{h}_{5}\underline{b}_{1} \\ \underline{f}_{5}\underline{b}_{2} \\ -\underline{f}_{5}\underline{b}_{2} \\ \underline{f}_{5}\underline{b}_{5} \\ \underline{f}_{5}\underline{f}_{5}\underline{b}_{5} \\ \underline{f}_{5}\underline{b}_{5} \\ \underline{f}_{5}\underline{b}_{5}\underline{b}_{5} \\ \underline{f}_$$



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- d) Show how to calculate the accelerations and the joint reaction forces (1 pt).
- e) Substitute the spring and damper between body 2 and ground by a massless link (I = 4m). Write the equations of motion of the system (2 pts).
- f) Provided that you can measure the initial positions in the figure, and if the initial velocities of the system, formulated in b), are $\dot{x}_1 = 0.5 \, m/s$, $\dot{y}_1 = -0.2 \, m/s$, $\dot{x}_2 = 0.5 \, m/s$, $\dot{y}_2 = -0.2 \, m/s$, $\dot{\theta}_1 = -0.2 \, rad/s$, and $\dot{\theta}_2 = -0.1 \, rad/s$, show if the velocity constraint equations of the system are satisfied (2 pts).



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d) Show how to calculate the accelerations and the joint reaction forces (1 pt).

c)
$$| \underline{M} \hat{\mathbf{q}} + \underline{\mathbf{p}}_{\mathbf{q}}^{\mathsf{T}} \underline{\mathcal{L}} = \underline{\mathbf{q}} | \underline{M} \hat{\mathbf{q}} = \underline{\mathbf{q}} - \underline{\mathbf{p}}_{\mathbf{q}}^{\mathsf{T}} \underline{\mathcal{L}} | \underline{\mathbf{q}}_{\mathbf{q}}^{\mathsf{T}} \underline{\mathcal{L}} = \underline{\mathbf{q}} | \underline{\mathbf{q}}_{\mathbf{q}}^{\mathsf{T}} \underline{\mathcal{L}} = \underline{\mathbf{q$$



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e) Substitute the spring and damper between body 2 and ground by a massless link (I = 4m). Write the equations of motion of the system (2 pts).



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The acceleration equations are given by

$$\ddot{D} = 0 \implies a \dot{c}_{1} \dot{c}_{2} + a \dot{c}_{2} \dot{c}_{3} = 0 \implies$$

$$\Rightarrow a \dot{c}_{1} \left(-\dot{r}_{3} - B_{3} \dot{s}^{2} \dot{\theta}_{2} + A_{3} \dot{s}_{2}^{2} \dot{\theta}_{2}^{2} \right) = -a \dot{c}_{3} \dot{c}_{2} \implies$$

$$\Rightarrow \left[0 \quad 0 \quad -a \dot{c}_{1}^{T} \quad -a \dot{c}_{2}^{T} B_{3} \dot{s}^{2} \right] \left[\ddot{r}_{1} \right] = -a \dot{c}_{3}^{T} \dot{c}_{2} - a \dot{c}_{3}^{T} A_{2} \dot{s}^{2} \dot{\theta}_{3}^{2}$$

$$\Rightarrow \left[0 \quad 0 \quad -a \dot{c}_{1}^{T} \quad -a \dot{c}_{2}^{T} B_{3} \dot{s}^{2} \right] \left[\ddot{r}_{1} \right] = -a \dot{c}_{3}^{T} \dot{c}_{2} - a \dot{c}_{3}^{T} A_{2} \dot{s}^{2} \dot{\theta}_{3}^{2}$$

$$\Rightarrow \left[\dot{a}_{3} \right] \left[\dot{a}$$



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e) Substitute the spring and damper between body 2 and ground by a massless link (I = 4m). Write the equations of motion of the system (2 pts).

The equations of motion are
$$\begin{bmatrix} M & Qq \\ \overline{Q}_{q} & O & A \\ \overline{Q}_{q} & \overline{Q}_{q} & \overline{Q}_{q} \end{bmatrix} = \underbrace{QQ}_{Y}, \quad \text{where}$$

$$\overline{Q}_{q} = \begin{bmatrix} h_{1}^{T} & d^{T} A_{20} & G_{1}^{\prime D} - S_{1}^{\prime C} \\ O & O & -2 d^{T} \\ O & O & -2 d^{T} B_{2} S_{1}^{\prime C} \end{bmatrix}$$

$$Y = \begin{cases} -a & d^{T} h_{1}^{\prime} + d^{T} A_{20} & A_{1} & (S_{1}^{\prime D} - S_{1}^{\prime C}) & \partial_{1}^{\prime A} \\ -2 & d^{T} d_{2} & -2 d^{T} A_{2} S_{2}^{\prime C} & \partial_{2}^{\prime A} \end{bmatrix}$$

$$Q = \begin{cases} m_{1} \otimes q + f_{2} & f_{2} \\ m_{2} \otimes q + f_{2} & f_{2} \\ m_{3} \otimes q + f_{3} & f_{3} & f_{3} & f_{3} & f_{3} \\ 0 & 0 & 0 \end{cases}$$



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f) Provided that you can measure the initial positions in the figure, and if the initial velocities of the system, formulated in b), are $\dot{x}_1 = 0.5 \, m/s$, $\dot{y}_1 = -0.2 \, m/s$, $\dot{x}_2 = 0.5 \, m/s$, $\dot{y}_2 = -0.2 \, m/s$, $\dot{\theta}_1 = -0.2 \, rad/s$, and $\dot{\theta}_2 = -0.1 \, rad/s$, show if the velocity constraint equations of the system are satisfied (2 pts).

$$\frac{f}{g} = \frac{g}{g} = \frac{g}{g} = \frac{g}{g}$$

$$\Rightarrow \begin{bmatrix} h, T & \frac{1}{2} Ag_0 B_1 (s, D - s, C) + h, T_{B_1} S, D & -h, T - h, B_2 S, C \\ 0 & -2 \frac{1}{2} T - 2 \frac{1}{2} T_{B_2} S, F \end{bmatrix} \begin{bmatrix} \frac{1}{2} & \frac{1}{2} & \frac{1}{2} \\ \frac{1}{2} & \frac{1}{2} & \frac{1}{2} \\ \frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} \\ \frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} \\ \frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} & \frac{1}{2} \\ \frac{1}{2} & \frac{1}{2} \\ \frac{1}{2} & \frac$$



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f) Provided that you can measure the initial positions in the figure, and if the initial velocities of the system, formulated in b), are $\dot{x}_1=0.5\,m/s$, $\dot{y}_1=-0.2\,m/s$, $\dot{x}_2=0.5\,m/s$, $\dot{y}_2=-0.2\,m/s$, $\dot{\theta}_1=-0.2\,rad/s$, and $\dot{\theta}_2=-0.1\,rad/s$, show if the velocity constraint equations of the system are satisfied (2 pts).

$$\Rightarrow \begin{bmatrix} -1.5972 & 1.2037 & 0.0087 & 1.5972 & -1.8037 & 1.1023 \\ 0 & 0 & -8.0236 & 0.0070 & -7.9801 \end{bmatrix} \begin{bmatrix} 0.5 \\ -0.2 \\ -0.2 \\ 0.5 \\ -0.2 \\ -0.1 \end{bmatrix}$$

$$\Rightarrow \begin{bmatrix} -0.1120 & 7 & 0 \\ -3-2152 & 0 \end{bmatrix} \begin{bmatrix} 7 & 0 \\ 7 & -2.2152 & 0 \end{bmatrix}$$
The veberly constraint equations are not satisfied.