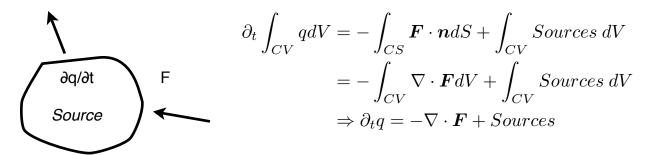
Finite volume method

Natural for unstructured meshed. Also, working with fluxes leads into hyperbolic equations.

A. Recall origin of p.d.e.'s



Don't take second step: apply discretization to the integral balance. Control volume is polyline (polyhedron in 3-D).

Divergence theorem (Gauss') is used in f.v. method. Rationale:

 $f = \int df/dx dx$ but now $f \rightarrow \int \mathbf{F} \cdot d\mathbf{S}$ and $df/dx dx \rightarrow$

$$[\nabla \cdot \mathbf{F} d\mathbf{V} = [\nabla \cdot \mathbf{F} d\mathbf{S} dx_n = [d\mathbf{F}/dx_n dx_n d\mathbf{S} =] \mathbf{n} \cdot \mathbf{F} d\mathbf{S}]$$

Divergence theorem <-> fundamental theorem of calculus: Sometimes useful to start with p.d.e. (see below)

1. Control volumes are defined by mesh:

$$\partial_t \int_{CV} q dV = -\int_{CS} {m F} \cdot {m n} dS$$
 (without sources). The integrals are

$$\partial_t \int q \, dV = -\Sigma \int F \cdot \mathbf{n} \, dS$$
 S_1 V S_6 S_5

This is exact. In 2-D, V = area and S = length: $\partial_t \int q dA = -\Sigma \int F \cdot \mathbf{n} d\ell$

The integrals must be discretized.

What is F? Convective $\mathbf{u}T$; diffusive $-\kappa \nabla T$ --- How does one compute the gradient?

2. Sometimes have to think of starting with diff eq. E.g., down gradient heat flux:

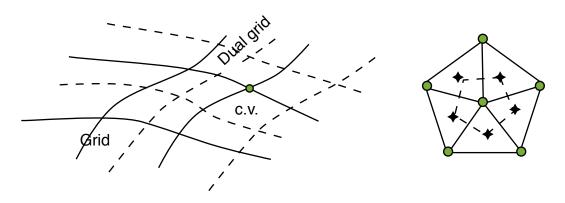
$$\int -\nabla T \, dV = -\int \nabla \cdot (\mathbf{I} \, T) \, dV = -\int T \, (\mathbf{n} \cdot \mathbf{I}) \, dS = -\int T \, \mathbf{n} dS$$
 $\mathbf{n} = \text{outward normal}$

Temperature gradient is computed from face temperatures: $\int -\nabla T \, dV = -\Sigma \int \mathbf{n} \, T \, dS$

B. Control volumes

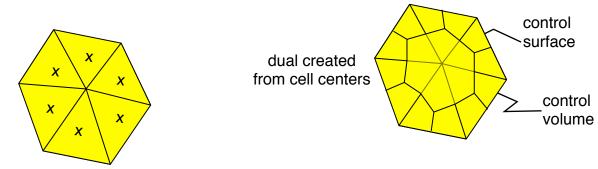
- 1. Order of accuracy determined by `reconstruction'.
- 2. Natural approach for unstructured grids (vs. finite difference)
- 3. Can derive by integrating diff. eqs. over control volume = mesh cell
- 4. Centers and vertices define dual grids

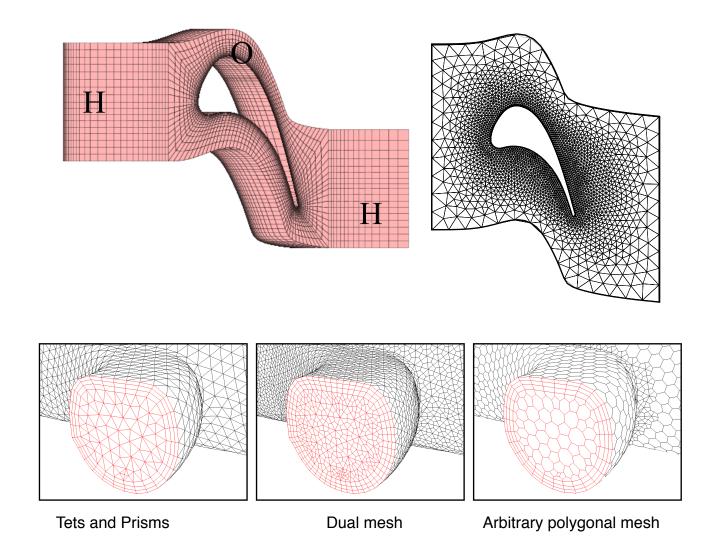
Data can be stored on grid (vertex) or on dual (cell center)



Mesh = set of control volumes. $\partial_t \int q \, dV = -\Sigma \int F \cdot \mathbf{n} \, dS$ Note flux out of one cell = flux into neighbor.

X's can be computational nodes of triangular c.v.s. Or vertices can be nodes and dual defines c.v.'s





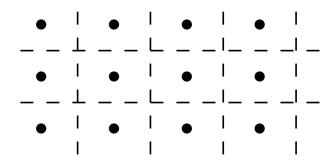
C. Finite Volume approximation

Reminder: $\partial_t T = \partial_x (\kappa \partial_x T) = -\partial_x F$ where $F = -\kappa \partial_x T$

$$\stackrel{\circ}{\underset{\hat{\mathbf{n}}}{\longleftarrow}} \stackrel{\circ}{\underset{\text{grid}}{\longleftarrow}} \stackrel{\circ}{\underset{\hat{\mathbf{n}}}{\longleftarrow}} \stackrel{\circ}$$

 $\partial_t \int T dx = F_{i-1/2} - F_{i+1/2}$ is exact conservation. Approximation: $\partial_t \int T dx \sim \partial_t T_i \Delta x$.

1. Flux balance applied to cell centers; volumes defined by mesh, think of Cartesian case



2. q = quantity per unit volume

 $\int_{CV} \partial_t q \, dV = -\int_{CS} F \cdot \tilde{\mathbf{n}} \, dS + \int_{CV} \text{ sources } dV$: Flux = convection $\mathbf{u}q$ or diffusion $-\kappa \nabla q$ Poisson: $\partial_t q = 0$ and $F = \nabla \psi$, source = ω : $\int_{CS} \tilde{\mathbf{n}} \cdot \nabla \psi \, dS = \int_{CV} \omega \, dV$

but **▼** complicates finite volume, analogous to Laplacian in finite diff.

3. Approximate the integrals to second order (midpoint rule)

$$\int_{cell} \partial_t q \, dV \approx \partial_t q_{cell \ center} \Delta V$$

$$\int_{cs} \mathbf{F} \cdot \hat{\mathbf{n}} dS = \sum_{faces} \int \mathbf{F} \cdot \hat{\mathbf{n}}_i dS_i \approx \sum_i (\mathbf{F} \cdot \hat{\mathbf{n}}_i)_{face \ center} \Delta S_i$$

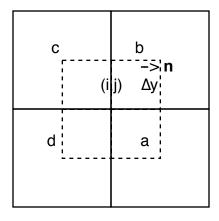
Higher order: use polynomial 'reconstruction' from nodal values. Discrete conservation equation

$$\partial_t q_{cc} = \Sigma_i [(F \cdot \hat{\mathbf{n}})_{|i} \Delta S_i]_{fc} / \Delta V$$

E.g. continuity q=p= constant:

$$0 = \Sigma_i \, \boldsymbol{F} \cdot \boldsymbol{\hat{n}} \, \Delta S_i \, / \Delta \boldsymbol{V}$$

With $\mathbf{F} = \rho \mathbf{u}$ and rectangular control volume:



$$n=(1,0), \Delta I_{a-}b=\Delta y$$

$$n=(0,-1), \Delta I_{d-a}=x_a -x_d=\Delta x$$

$$0 = [-\Delta x \ v_{i,j-1/2} + \Delta y \ u_{i+1/2,j} + \Delta x \ v_{i,j+1/2} - \Delta y \ u_{i-1/2,j}] \ / \ \Delta x \Delta y$$

$$0 = (u_{i+1/2,j} - u_{i-1/2,j})/\Delta x + (v_{i,j+1/2} - v_{i,j-1/2})/\Delta y \text{ as expected -- } \nabla \cdot \mathbf{u}.$$
 With $u_{i+1/2} = (u_{i+1} + u_i)/2$
$$0 = (u_{i+1,j} - u_{i-1,j})/2\Delta x + (v_{i,j+1} - v_{i,j-1})/2\Delta y$$

Equilateral triangle control volume . $\hat{\bf n}_2 = (\pm \sin \pi/6, \cos \pi/6) = (\sqrt{3}, 1)/2$

$$\hat{\mathbf{n}}_1 \Delta S = (0,-1) \ell$$

$$\hat{\bf n}_2 \Delta S = (\sqrt{3}, 1)/2 \ \ell$$

$$\hat{\bf n}_3 \Delta S = (-\sqrt{3}, 1)/2 \ \ell$$

$$V = \sqrt{3} \ell^2/4$$

$$\frac{(-v_1 + \sqrt{3}/2 \ u_2 + v_2/2 - \sqrt{3}/2 \ u_3 + v_3/2) \ell}{\sqrt{3} \ \ell^2/4} = \frac{[v_2 + v_3 - 2v_1]}{\sqrt{3} \ \ell/2} + \frac{2(u_2 - u_3)}{\ell}$$