

Lecture 11: Hydraulic motors, transmission lines

- Hydraulic motors
- Hydraulic transmission lines
- (Electrical transmission lines)

Book: 4.1-4.6, (1.6)

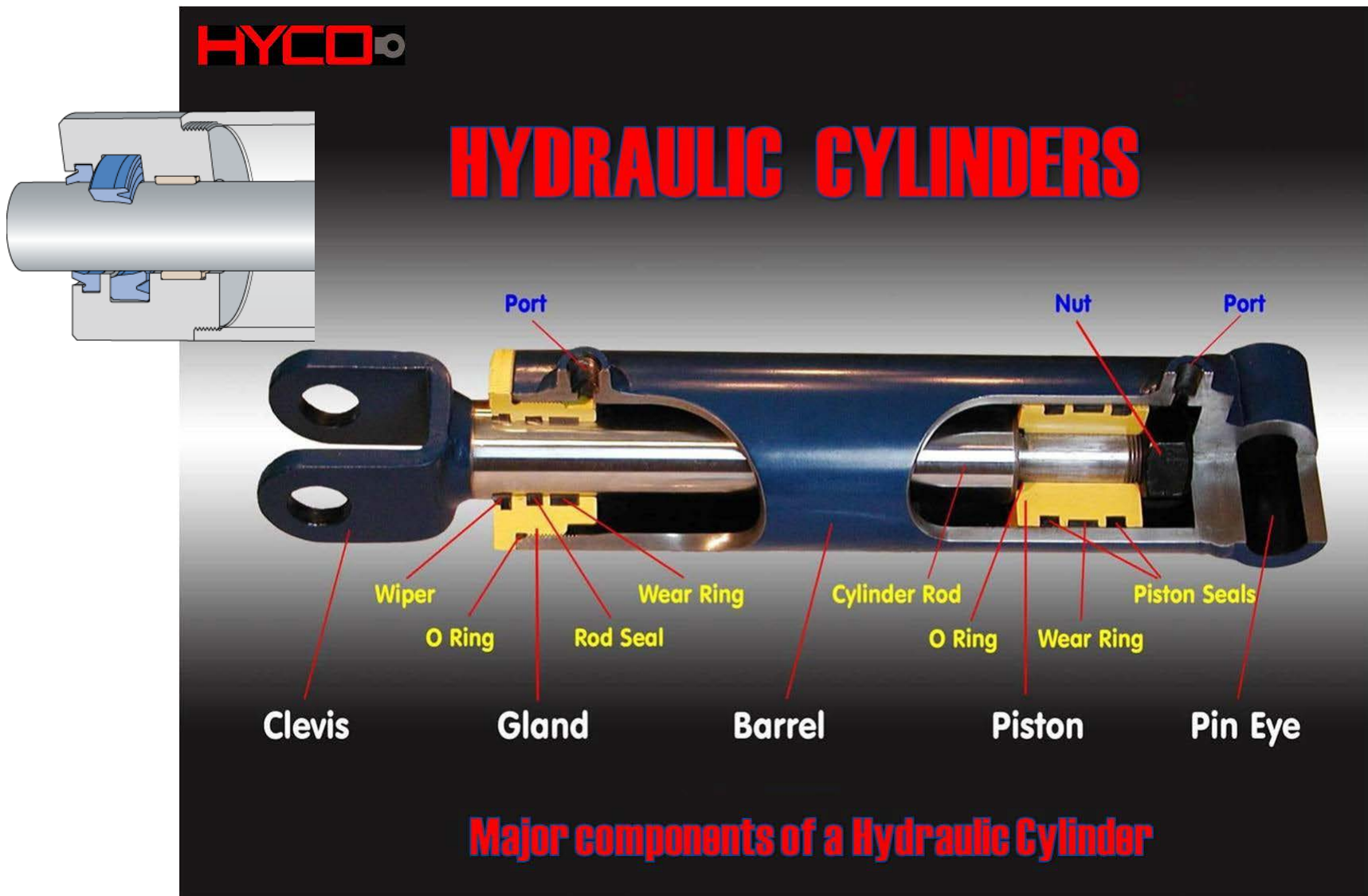
Systems using hydraulics to produce motion

- Excavators

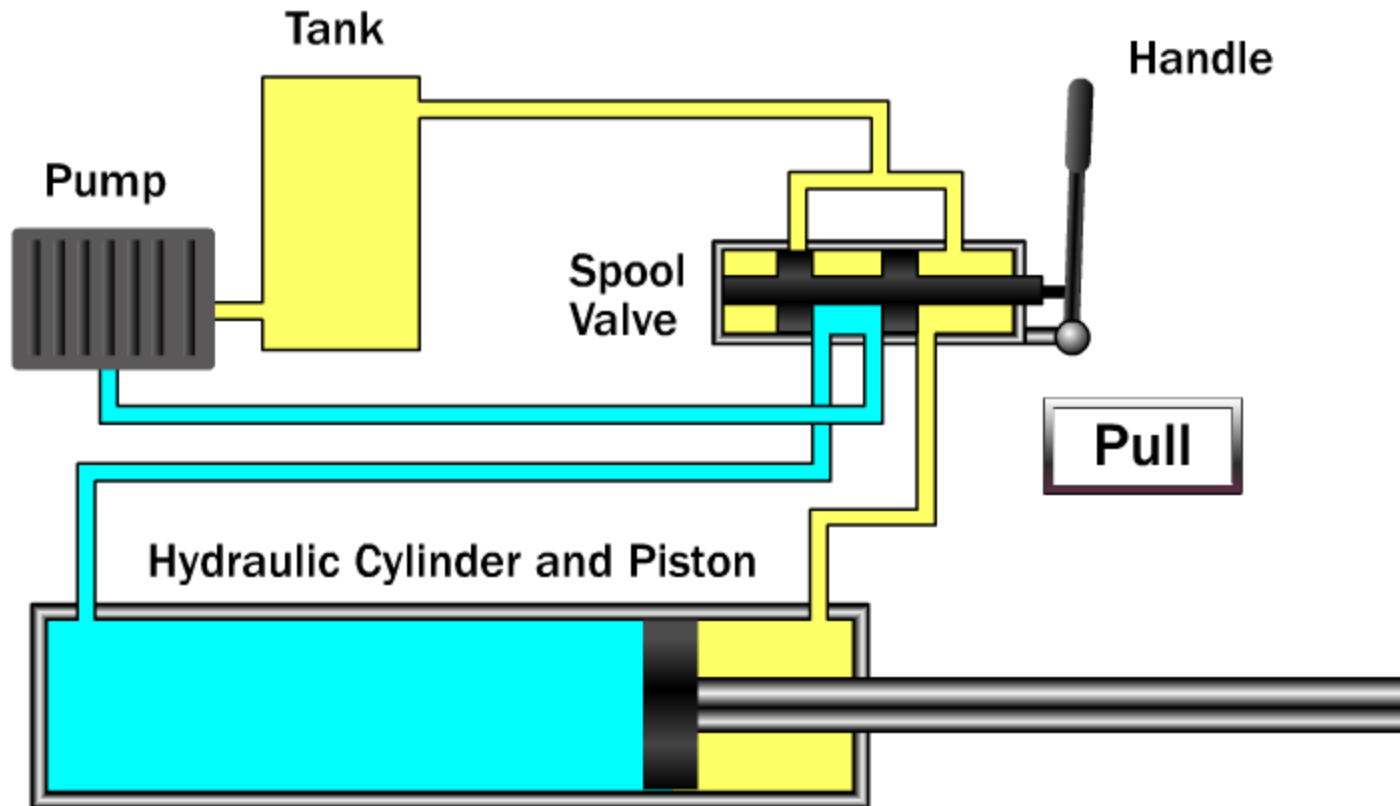


- Robots, cranes, etc.
- To control motion of these systems, we need models of the hydraulic actuators

Hydraulic cylinder



Hydraulic system



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Anna Konda – The fire fighting snake robot



Moody chart

- Circular pipe
- Darcy-Weisbach factor with Reynolds number and relative roughness

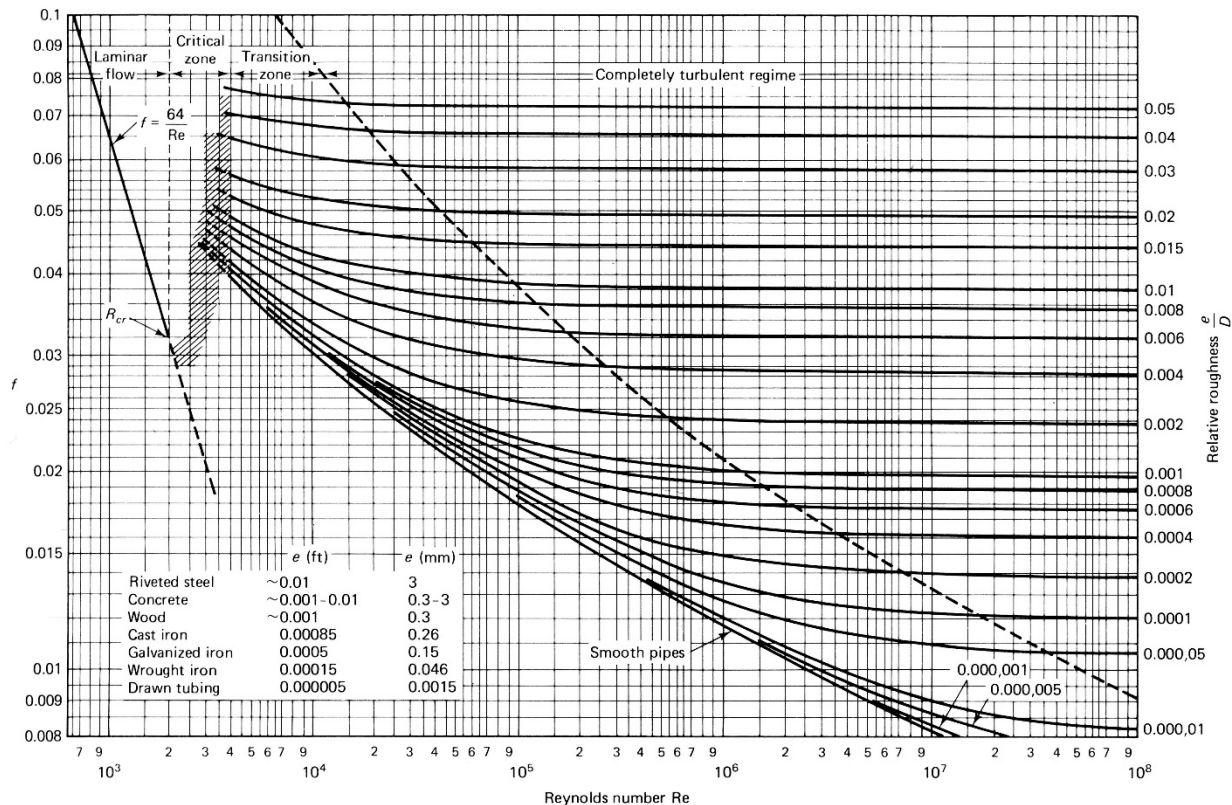


Figure 7.13 Moody diagram. (From L. F. Moody, *Trans. ASME*, Vol. 66, 1944.)

Hydraulic cylinder

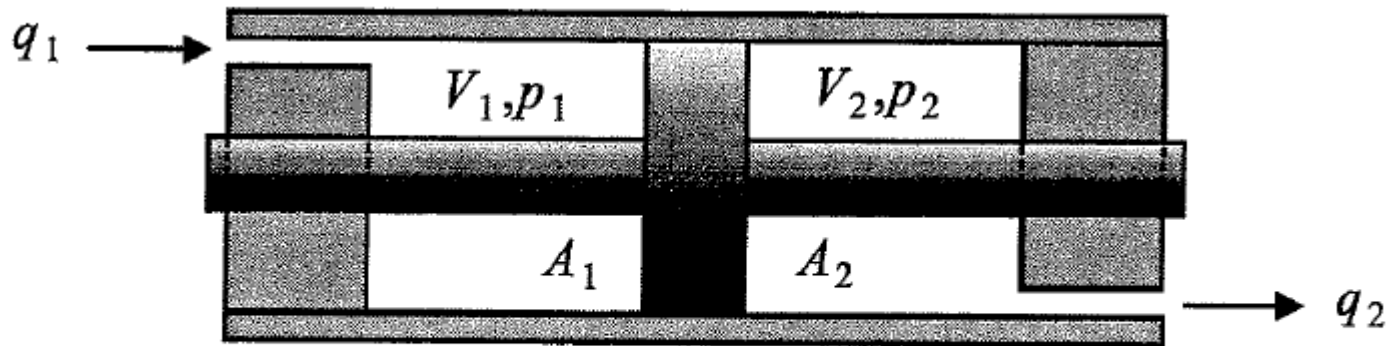


Figure 4.9: Symmetric hydraulic cylinder

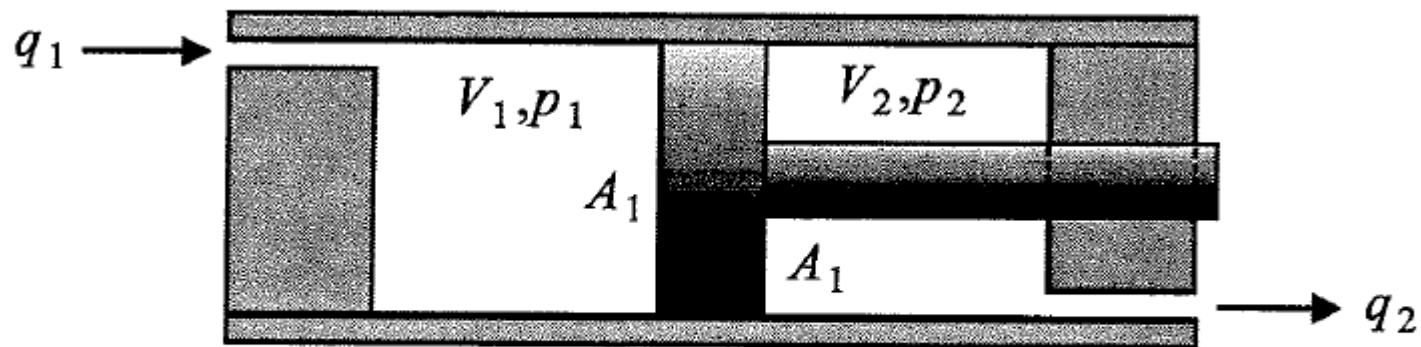


Figure 4.10: Single-rod hydraulic piston

Rotational hydraulic motor

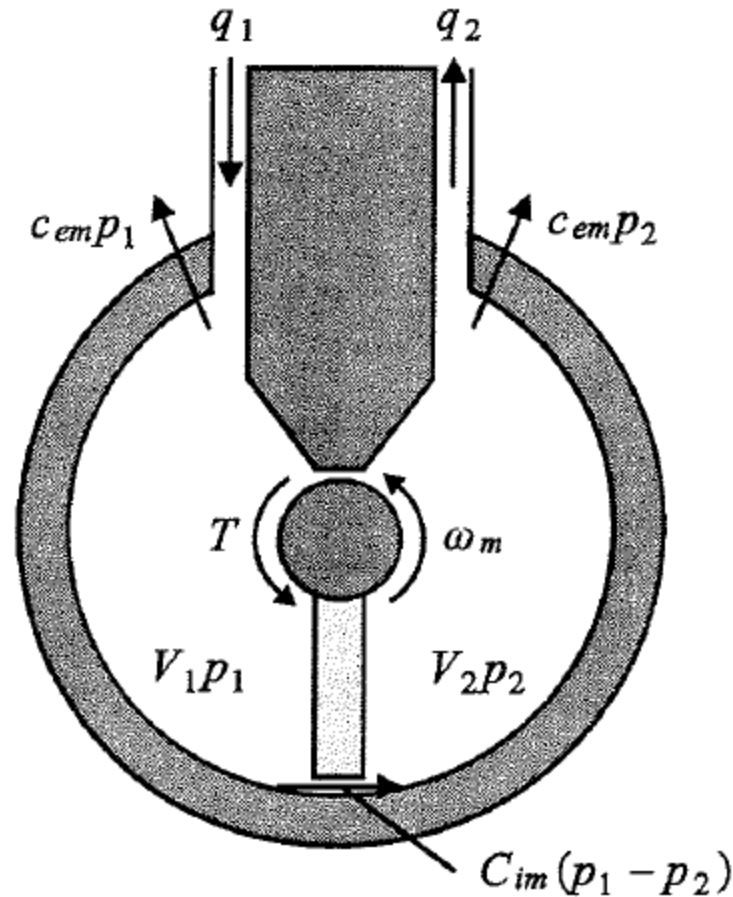
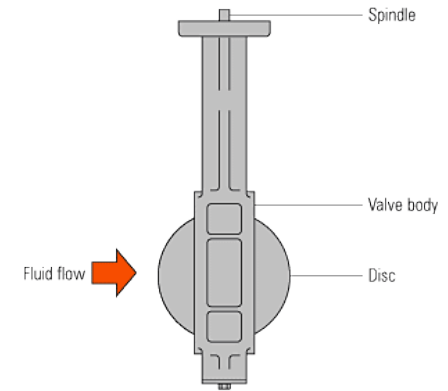
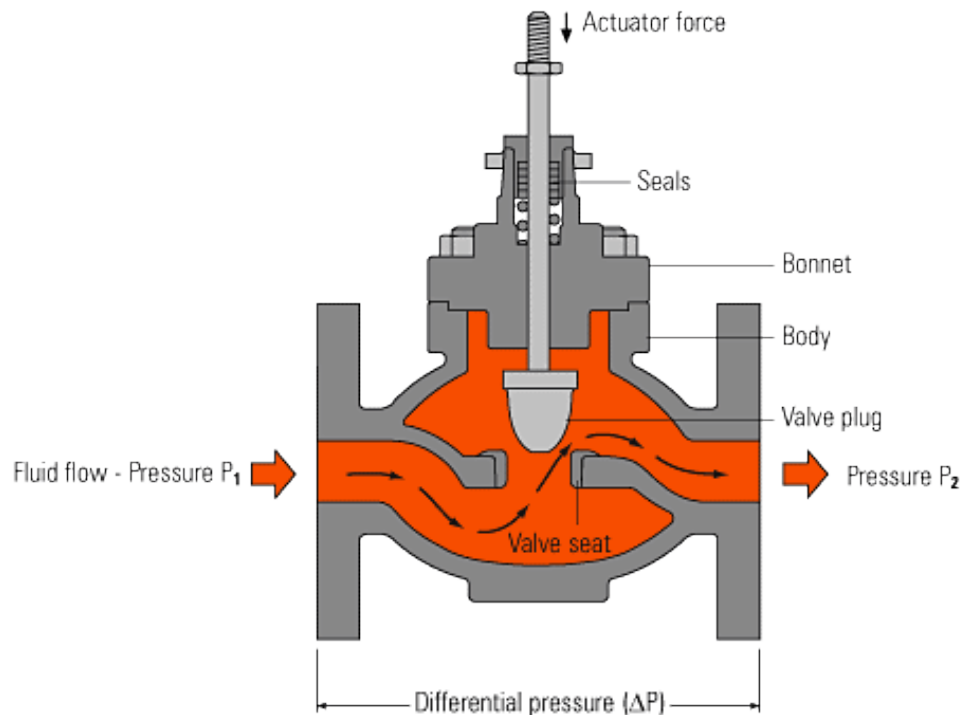


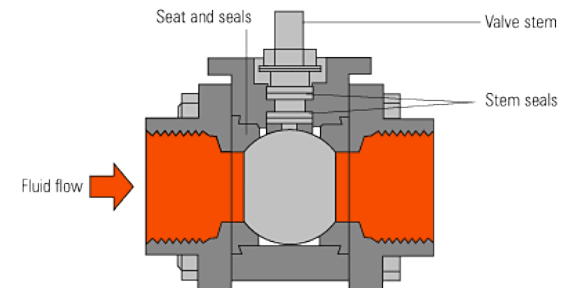
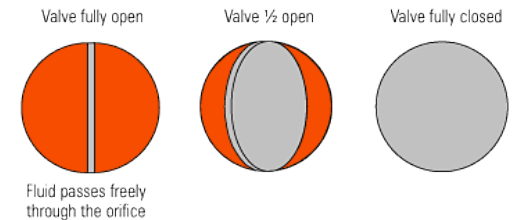
Figure 4.7: Rotational hydraulic motor of the single vane type with limited travel.

Valves

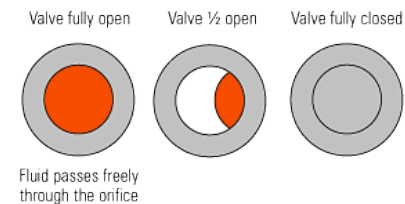
- Device that regulates flow
- Many different types of valves exist
 - Globe valve, ball valve, butterfly valve, ...



End view of the disc within the butterfly valve at different stages of rotation



End view of the ball within the ball valve at different stages of rotation

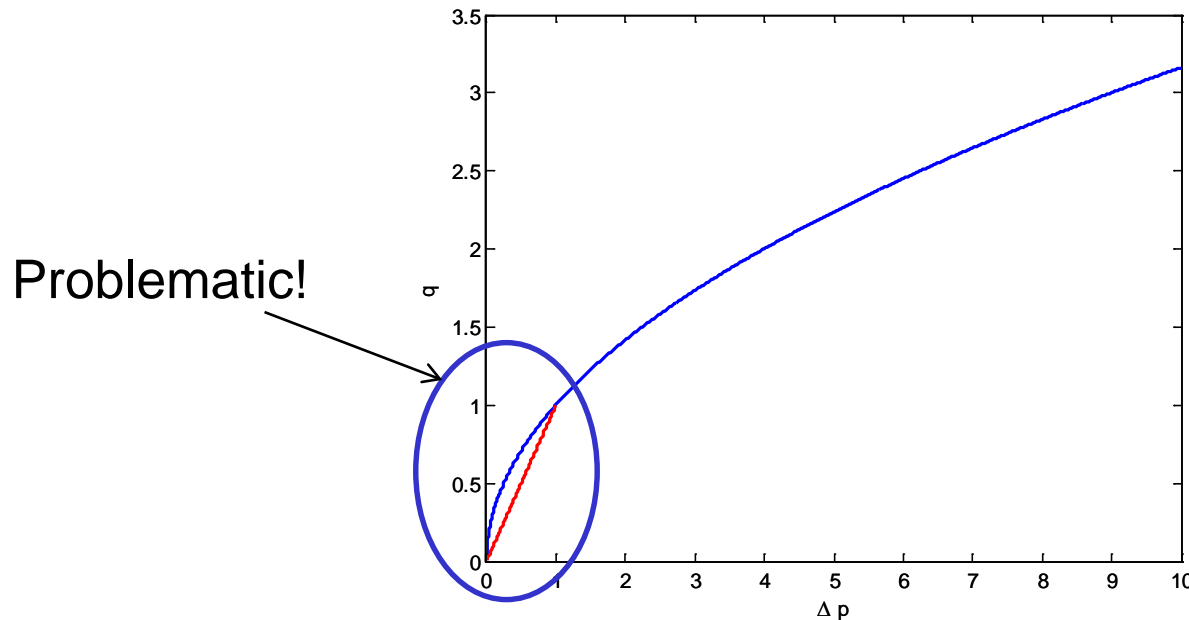


Valve models

(book 4.2)

- Flow through a restriction is generally turbulent

$$q = C_d A \sqrt{\frac{2}{\rho} \Delta p}$$

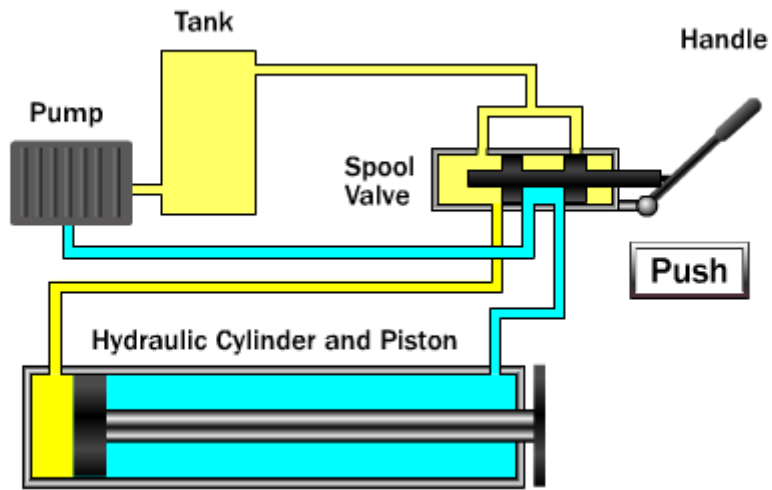


- Solution: Regularize by assuming laminar flow for small Δp

$$q = C_l \Delta p$$

- Book: Make transition smooth

Four-way valve



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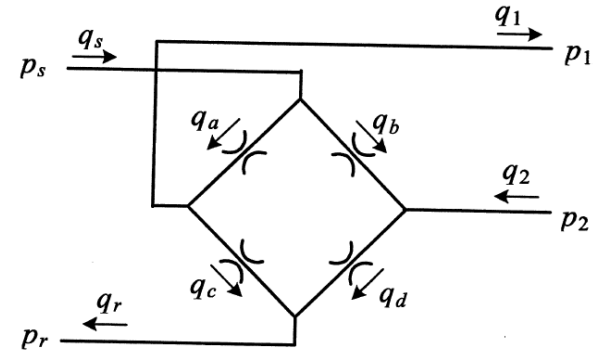
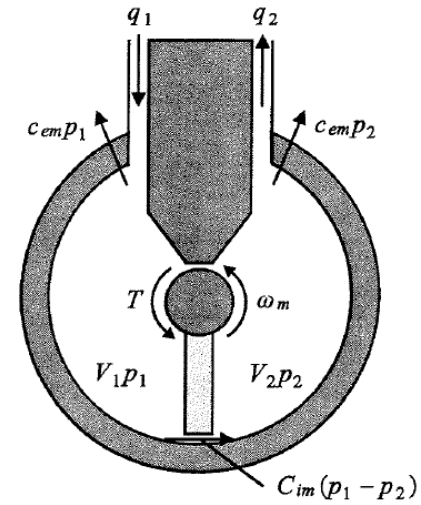
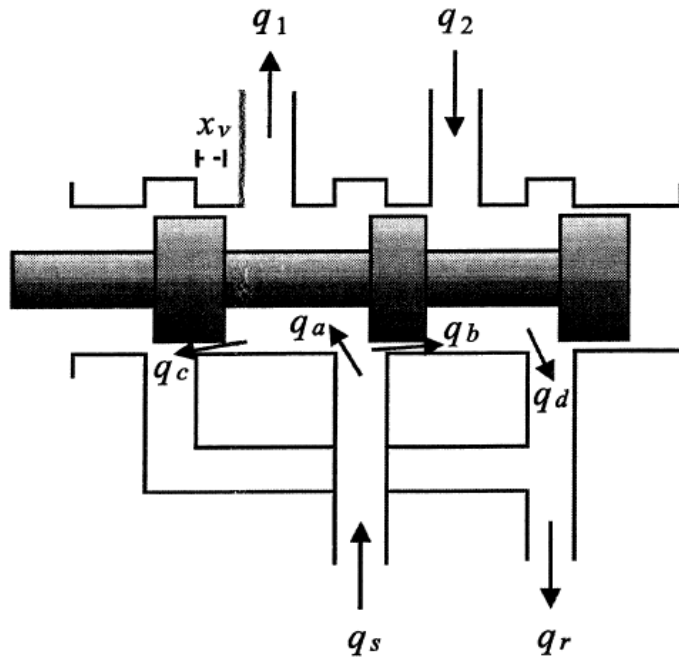


Figure 4.1: Four-way valve

Figure 4.2: A matched and symmetric four-way valve.

Modeling of four-way valve

- Define load pressure

$$p_L = p_1 - p_2$$

- Define load flow

$$q_L = \frac{q_1 + q_2}{2}$$

- Symmetric load assumption (motor)

$$q_1 = q_2$$

- Symmetric valve and symmetric load

$$q_L = C_d b x_v \sqrt{\frac{1}{\rho} (p_s - \text{sign}(x_v) p_L)}$$

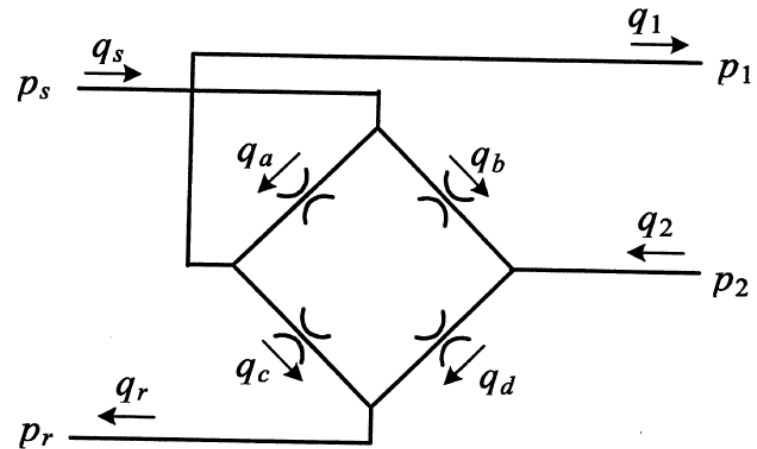


Figure 4.1: Four-way valve

Characteristic of four-way valve

$$q_L = C_d b x_v \sqrt{\frac{1}{\rho} (p_s - \text{sign}(x_v) p_L)}$$

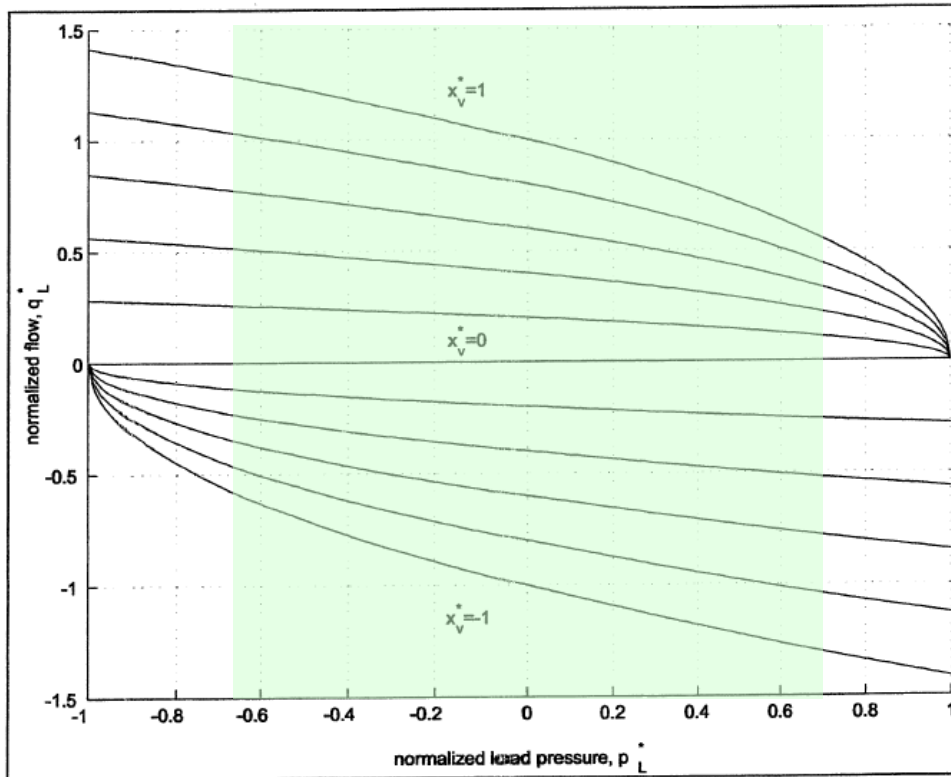


Figure 4.3: Valve characteristic

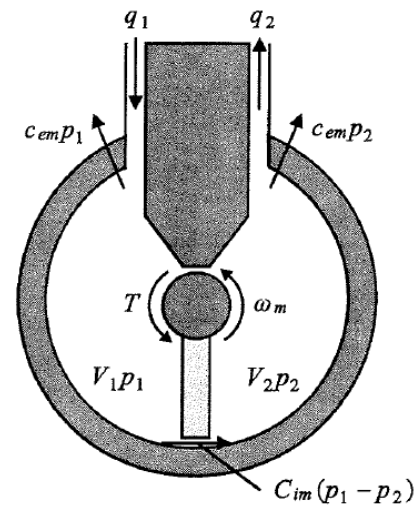
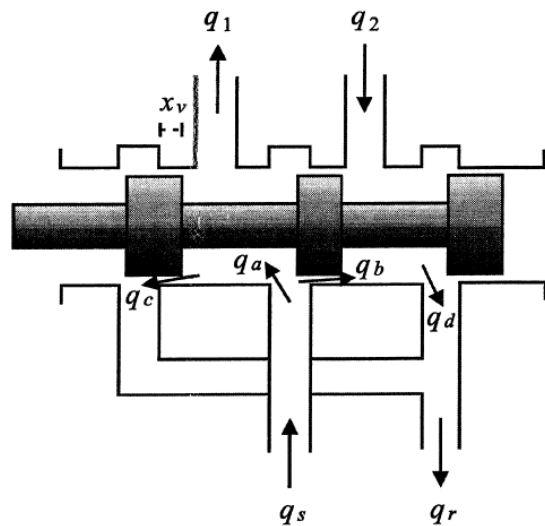
Linearized model:

$$|p_L| \leq \frac{2}{3} p_s : \quad q_L = K_q x_v - K_c p_L$$

Gain uncertainty:

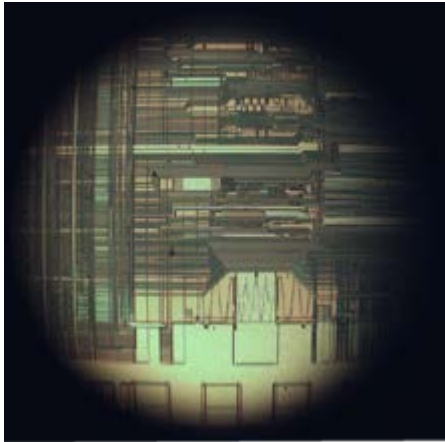
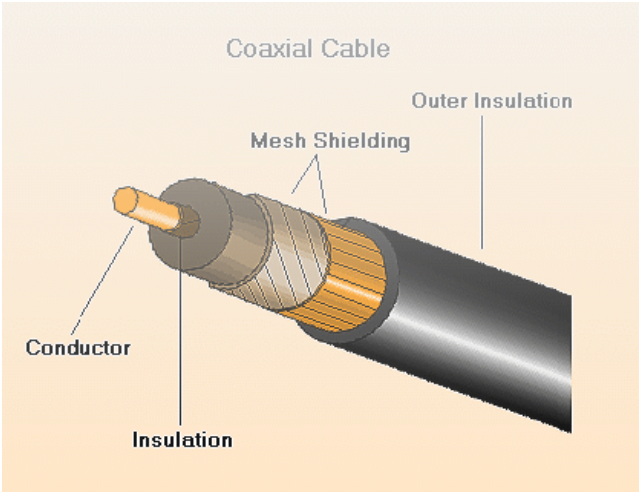
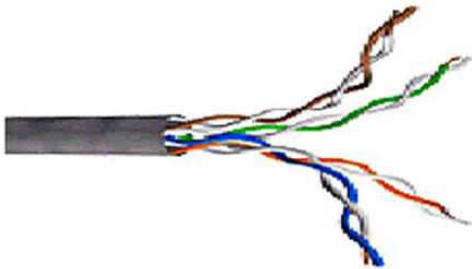
$$0.58 K_{q0} \leq K_q \leq 1.29 K_{q0}$$

Transfer function valve+motor



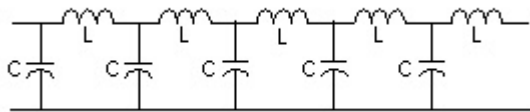
$$\theta_m(s) = \frac{\frac{K_q}{D_m} x_v(s) - \frac{K_{ce}}{D_m^2} \left(1 + \frac{s}{\omega_t}\right) T_L(s)}{s \left(1 + 2\zeta_h \frac{s}{\omega_h} + \frac{s^2}{\omega_h^2}\right)}$$

Electrical transmission lines

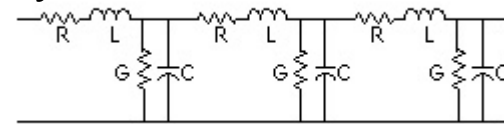


Telegrapher's equation (Wave equation)

- Lossless:



- Lossy:



- Model (Ch. 1.6):

$$\frac{\partial u(x, t)}{\partial x} = -Ri(x, t) - L \frac{\partial i(x, t)}{\partial t}$$

$$\frac{\partial i(x, t)}{\partial x} = -Gu(x, t) - C \frac{\partial u(x, t)}{\partial t}$$
- Laplace:

$$\frac{\partial u(x, s)}{\partial x} = -X(s)i(x, s)$$

$$\frac{\partial i(x, s)}{\partial x} = -Y(s)u(x, s)$$

Series impedance:

$$X(s) = R + Ls$$

Parallel admittance:

$$Y(s) = G + Cs$$

Characteristic impedance:

$$Z_c(s) = \sqrt{\frac{X(s)}{Y(s)}}$$

Same equations for electrical and fluid/hydraulical transmission lines

Electrical transmission lines:

$$\frac{\partial u(x, t)}{\partial x} = -Ri(x, t) - L \frac{\partial i(x, t)}{\partial t}$$

$$\frac{\partial i(x, t)}{\partial x} = -Gu(x, t) - C \frac{\partial u(x, t)}{\partial t}$$

Fluid transmission lines:

$$\frac{\partial p(x, t)}{\partial t} = -\frac{\beta}{A} \frac{\partial q(x, t)}{\partial x}$$

$$\frac{\partial q(x, t)}{\partial t} = -\frac{A}{\rho} \frac{\partial p(x, t)}{\partial x} - \frac{F[q(x, t)]}{\rho}$$

- Current and flow “same” variables, as is voltage and pressure
- In both cases, we can define line impedance, characteristic impedance, propagation operator, etc.
- Solution to equations have same structure/form: waves propagating back and forth

Solution: Waves

- Solution:

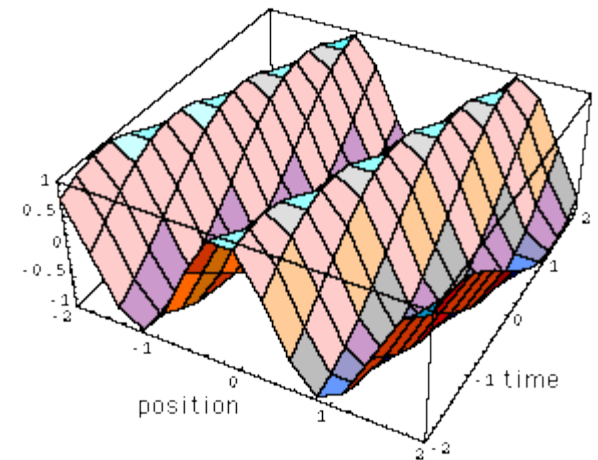
$$u_{out}(s) = e^{-\Gamma(s)} u_{in}(s)$$

- Propagation operator $\Gamma(s) = L\sqrt{X(s)Y(s)}$

- Attenuation factor $\Re[\Gamma(j\omega)]$: How much is wave reduced
- Phase factor: $\Im[\Gamma(j\omega)]$: How long does it take

- Lossless ($R = G = 0$): $\Gamma(s) = Ts$

- Attenuation factor: 0
- Phase factor: Pure time-delay



When should we care?

- Solution lossless case: Time delay

$$e^{-Ts}$$

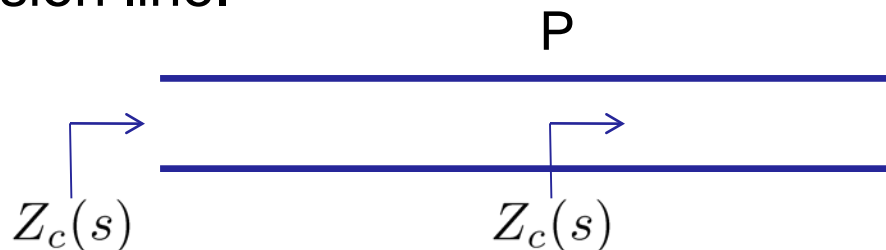
- Rule-of-thumb from control theory: We can ignore time-delay for frequencies much less than $1/T$

$$\omega \leq \frac{1}{T} \Rightarrow 2\pi \frac{c}{\lambda} \leq \frac{c}{L} \Rightarrow L \leq \frac{\lambda}{2\pi}$$

- Rule-of-thumb for transmission lines: When L is larger than one tenth of wavelength, treat as transmission line
- Power lines, $f = 50\text{Hz}$: $\lambda = 6000\text{km}$
- Personal computers, $f = 10\text{GHz}$: $\lambda = 1.5\text{cm}$

Impedance matching

- Suppose we have an imaginary joint at P in a very long transmission line.



The wave goes through the joint without reflection because there is actually no joint (just imagined).

- Now, let us terminate a resistance of value Z_c *at the same position of this* imaginary joint. The wave will go through without reflection too.



This is called a **matched load**.