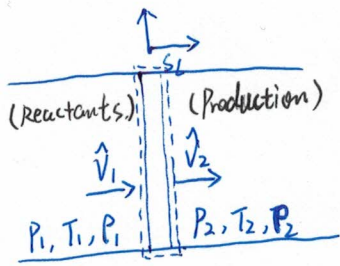


§ Governing Equations for 1D premixed combustion.



- ① Moving coordinates.
- ② Consider the tiny control volume covering the flame zone.
- ③ 1D Euler equations.
- ④ Rankine-Hugoniot conditions. $\Delta \bar{F} = S \cdot \Delta U$

$$\frac{\partial U}{\partial t} + \frac{\partial F(U)}{\partial x} = 0$$

where:

$$U = \begin{bmatrix} P \\ \rho u \\ \rho(e + \frac{1}{2}u^2) \end{bmatrix}, \quad F = \begin{bmatrix} \rho u \\ \rho u^2 + P \\ u(\rho(e + \frac{1}{2}u^2) + P) \end{bmatrix}$$

R-H condition

$S=0$

$$\Delta \bar{F} = 0 : \begin{cases} P_1 \hat{u}_1 = P_2 \hat{u}_2 \\ P_1 \hat{u}_1^2 + P_1 = P_2 \hat{u}_2^2 + P_2 \\ P_1 \hat{u}_1 (e_1 + \frac{1}{2} \hat{u}_1^2 + \frac{P_1}{\rho_1}) = P_2 \hat{u}_2 (e_2 + \frac{1}{2} \hat{u}_2^2 + \frac{P_2}{\rho_2}) \end{cases}$$

The energy equation can be simplified with the continuity equation, $\Rightarrow h_1 + \frac{1}{2} \hat{u}_1^2 = h_2 + \frac{1}{2} \hat{u}_2^2$

Full equation set:

$$\begin{cases} P_1 \hat{u}_1 = P_2 \hat{u}_2 & ① \\ P_1 \hat{u}_1^2 + P_1 = P_2 \hat{u}_2^2 + P_2 & ② \\ h_1 + \frac{1}{2} \hat{u}_1^2 = h_2 + \frac{1}{2} \hat{u}_2^2 & ③ \end{cases} \quad \& \quad \begin{cases} h = h_0 + C_p \Delta T & ④ \\ P_1 = P_1 R T & ⑤ \\ P_2 = P_2 R T & ⑥ \end{cases}$$

§ Rayleigh Line & Rankine-Hugoniot Curve.

① When the mass flux Q is given, relations between pressure P and density P (or $v = \frac{1}{P}$) can be described using ① & ②. (The so-called Rayleigh Line)

$$P_1 \hat{u}_1 = P_2 \hat{u}_2 \equiv Q \Rightarrow \frac{Q^2}{P_1} + P_1 = \frac{Q^2}{P_2} + P_2 \Rightarrow \frac{P_2 - P_1}{\frac{1}{P_2} - \frac{1}{P_1}} = -Q^2 \quad \text{or} \quad \left(\frac{P_2 - P_1}{v_2 - v_1} = -Q^2 \right)$$

$$\text{or} (Q^2 v_1 + P_1 = Q^2 v_2 + P_2)$$

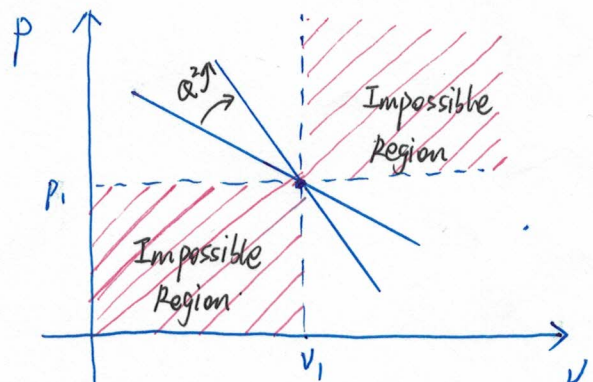
namely:

$$P = -Q^2 v + (P_1 + Q^2 v_1)$$

Comments:

(1) There're 2 impossible regions as the slope is never positive

(2) When the initial condition is given, the line gets steeper when $Q \uparrow$



- ② When the energy equation is also considered, relations between pressure P and density ρ (or $v = \frac{1}{\rho}$) can be further described. (The so-called Rankine-Hugoniot Curve)

Typically, we re-write the energy equation ③ with ④ as:

$$C_p T_1 + \frac{\hat{u}_1^2}{2} + q = C_p T_2 + \frac{\hat{u}_2^2}{2}$$

where $q = \sum_i Y_i h_{f,i}^0 - \sum_i Y_i h_{f,i}^0$, indicating the total heat released during the combustion

Combining ①, ② & ⑤, ⑥ yields:

$$C_p(T_2 - T_1) + \frac{1}{2}(\hat{u}_2^2 - \hat{u}_1^2) - q = 0$$

$$\bar{T} = \frac{P}{\rho R}$$

$$\hat{u}_2 - \hat{u}_1 = -\frac{P_2 - P_1}{Q}$$

$$\hat{u}_2 + \hat{u}_1 = Q\left(\frac{1}{\rho_1} + \frac{1}{\rho_2}\right)$$

$$\Rightarrow \frac{C_p}{R} \left(\frac{P_2}{\rho_2} - \frac{P_1}{\rho_1} \right) - \frac{1}{2} (P_2 - P_1) \left(\frac{1}{\rho_1} + \frac{1}{\rho_2} \right) - q = 0$$

$$\Downarrow \gamma \triangleq \frac{C_p}{C_v}, C_p - C_v = R$$

$$\frac{\gamma}{\gamma - 1} (P_2 v_2 - P_1 v_1) - \frac{1}{2} (P_2 - P_1) (v_2 + v_1) - q = 0$$

When q, P_1, v_1 are given, relations between P and v can be plotted:

D and E are the so-called Chapman-Jouguet points.

The curve is divided into 5 parts:

- (1) I: Beyond point D:

Strong detonation:

$$Ma_2 < 1, Ma_1 \gg 1$$

- (2) II: Between D and B:

Weak detonation:

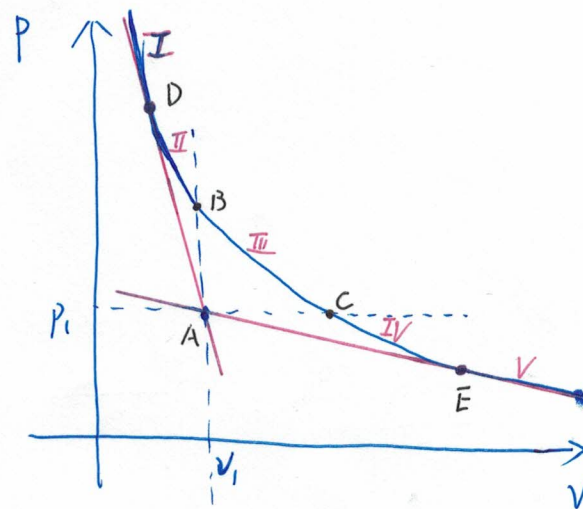
$$Ma_2 > 1, Ma_1 \gg 1$$

- (3) III: Between B and C: Impossible.

- (4) IV: Between C and E:

Weak deflagration:

$$Ma_2 < 1, Ma_1 \leq 1$$



Detonation: a type of combustion involving a supersonic exothermic front accelerating through a medium that eventually drives a shock front propagating in front of it directly.

Deflagration: Subsonic combustion propagating through heat transfer.

(5) V: Below point E:

Strong deflagration.

$$Ma_2 > 1 \quad Ma_1 \leq 1$$

Q: How to determine / understand Ma_2 in different regions?

A: Take derivative of the Rankine-Hugoniot curve:

$$\frac{d}{dv} \left[\frac{\gamma}{\gamma-1} (Pv - P_1 v_1) - \frac{1}{2} (P - P_1)(v + v_1) - Q \right] = 0$$

$$\Leftrightarrow: \frac{\gamma}{\gamma-1} \left(\frac{dP}{dv} \cdot v + P \right) = \frac{1}{2} \frac{dP}{dv} (v + v_1) + (P - P_1)$$

$$\text{at CJ points: } \frac{dP}{dv} = \frac{P_2 - P_1}{v_2 - v_1}$$

$$\Rightarrow \frac{\gamma}{\gamma-1} \left(\frac{P_2 - P_1}{v_2 - v_1} v_2 + P_2 \right) = \frac{1}{2} \frac{P_2 - P_1}{v_2 - v_1} (v_2 + v_1) + (P_2 - P_1)$$

$$\Leftrightarrow \frac{P_2 - P_1}{v_2 - v_1} = - \frac{\gamma P_2}{v_2} = - \gamma P_2 P_2$$

$$\Rightarrow Ma_2^2 \triangleq \frac{\hat{Q}_2^2}{a_2^2} = \frac{(P_2 \hat{u}_2)^2}{(P_2 a_2)^2} = \frac{Q^2}{\gamma P_2 P_2} = - \frac{P_2 - P_1}{v_2 - v_1} \frac{1}{\gamma P_2 P_2} = 1 \quad (\text{at D and E})$$

Some comments on Ma_2^2 :

$$Ma_2^2 = \frac{Q^2}{\gamma P_2 P_2} = \frac{Q^2 v_2}{\gamma P_2}$$

In Region I: Q^2 and v_2 varies slower than P_2 , thus $Ma_2^2 < 1$

In Region II: Q^2 increase rapidly towards $+\infty$, while P_2 and v_2 varies little, $\Rightarrow Ma_2^2 > 1$

In Region IV: Q^2 tends to 0, while P_2 and v_2 holds nearly constant, $\Rightarrow Ma_2^2 < 1$

In Region V: Q^2 changes little, P_2 is confined, but $v_2 \rightarrow +\infty$, $\Rightarrow Ma_2^2 > 1$

It should be noted that: (in reality) weak detonation (II) and strong deflagration (V) hardly occur!

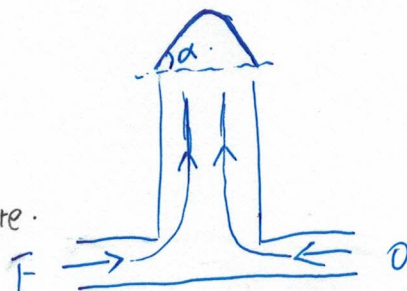
§. Laminar Premixed Flame Structure:

Take the Bunsen burner for example.

$\phi < 1$: deep-violet, CH dominate

$\phi > 1$: green, C_2H_2 dominate

$\phi \gg 1$: intense-yellow, Carbon particle dominate.



The high temperature zone lies in the front of the flame, but the reaction rate is small as the reaction process is ending.

For burned gas, $T \uparrow$ results in $\rho \downarrow$, due to continuity, the normal velocity $V_n \uparrow$.

§ Flame Speed & Remarks on Q2-HW3

* General Structure (1D, laminar, premixed)

Governing Equation for energy:

$$\rho_u S_L C_p \frac{dT}{dx} - \frac{d}{dx} \left(\lambda \frac{dT}{dx} \right) = - \sum h_i W_i$$

① Consider the pre-heat zone:

It's assumed that there's no reaction!

Hence the chemical source term vanished!

In other words; gas entering the pre-heat zone is heated by the conduction through interface 2.

$$\Rightarrow \rho_u S_L C_p \frac{dT}{dx} = \frac{d}{dx} \left(\lambda \frac{dT}{dx} \right) \xrightarrow{\int_{-\infty}^{x_i}} \rho_u S_L C_p (T_i - T_0) = \lambda \frac{dT}{dx} \Big|_{x_i} = \lambda \frac{dT}{dx} \Big|_{x=x_i}$$

$$\left(\frac{dT}{dx} \Big|_{x=-\infty} = 0 \right)$$

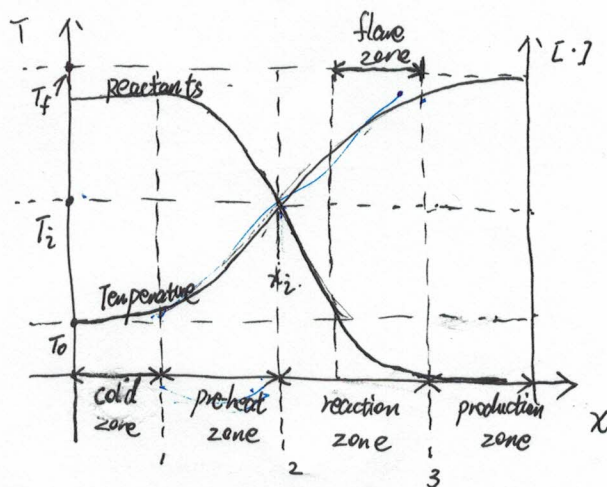
Suppose the width of flame δ is known, then flame speed can be approximated.

by the Thermal Theory as the gradient of T can be approximated by $\frac{T_f - T_i}{\delta}$.

$$\Rightarrow \rho_u S_L C_p (T_i - T_0) = \lambda \frac{T_f - T_i}{\delta} \Rightarrow S_L = \frac{\lambda (T_f - T_i)}{\rho C_p (T_i - T_0)} \frac{1}{\delta} = \alpha \frac{T_f - T_i}{T_i - T_0} \frac{1}{\delta}$$

thermal diffusivity.

Only related to the properties of mixture, independent of flame configuration.



② Consider the reaction zone.

Energy from convection (due to the temperature difference) is much smaller compared to the diffusion term. Hence can be neglected!

Indicating that the conduction through interface 2 is sustained by reaction!

$$\Rightarrow -\frac{d}{dx}\left(\lambda \frac{dT}{dx}\right) = -\bar{z}h_i w_i \xrightarrow{x_i} \lambda \frac{dT}{dx}\bigg|_{x=x_i} = \int_{x_i}^{+\infty} -\bar{z}h_i w_i dx \quad (\text{not convenient to solve!})$$

With some mathematical manipulation by multiply $\frac{dT}{dx}$ to both sides!

$$-\frac{d}{dx}\left(\lambda \frac{dT}{dx}\right) \cdot \frac{dT}{dx} = -\bar{z}h_i w_i \cdot \frac{dT}{dx} \xrightarrow[\text{Integrate by parts}]{x_i} \left(\lambda \frac{dT}{dx}\right)^2\bigg|_{x=x_i} = -\int_{T_i}^{T_f} 2\lambda \bar{z}h_i w_i dT$$

(LHS = $-\frac{1}{2\lambda} \frac{d}{dx}\left(\lambda \frac{dT}{dx}\right)^2$, $\frac{dT}{dx}\bigg|_{x=+\infty} = 0$)

$$\Rightarrow \lambda \frac{dT}{dx}\bigg|_{x=x_i} = \sqrt{-2\lambda \int_{T_i}^{T_f} \bar{z}h_i w_i dT}$$

Combined with previous expression of $\lambda \frac{dT}{dx}\bigg|_{x=x_i}$ yields:

$$\rho S_L C_p (T_i - T_0) = \sqrt{-2\lambda \int_{T_i}^{T_f} \bar{z}h_i w_i dT} \Rightarrow S_L = \sqrt{\underbrace{\frac{\lambda}{\rho C_p}}_{\alpha} \cdot \frac{-2}{\rho C_p (T_i - T_0)} \cdot \frac{1}{T_i - T_0} \int_{T_i}^{T_f} \bar{z}h_i w_i dT}$$

This is the so-called ZFK Theory!

Q: What's the dependence of S_L to ϕ ?

A: As ϕ is related to z by: $\phi = \frac{z}{1-z} \left(\frac{1-z_{st}}{z_{st}} \right)$: monotonic!

Recall that the Adiabatic Flame temperature T_{ad} is related to z . (HW2)

Hence, S_L follows the same shape as T_{ad} !

* Special Case: $Le \triangleq \frac{\lambda}{\rho C_p D} = \frac{\alpha}{D} = 1$.

Governing Equation: $m \cdot C_p \frac{dT}{dx} - \lambda \frac{d^2 T}{dx^2} = \dot{q}_c \cdot \dot{w}$ (energy)

$m \frac{dY}{dx} - \rho D \frac{d^2 Y}{dx^2} = -\dot{w}$ (specie).

Q: How to Eliminate \dot{w} ?

A: multiply the species equation by ρc_p and adding them up!

$$\Rightarrow \dot{m} c_p \frac{dT}{dx} + \dot{m} \rho c_p \frac{d\tilde{Y}_i}{dx} = \lambda \frac{dT}{dx^2} + \rho D \rho c_p \frac{d\tilde{Y}_i}{dx^2}$$

With proper non-dimensionalized:

$$\tilde{T} = \frac{c_p}{\rho c_p T_u} T, \quad \tilde{Y}_i = \frac{Y_i}{Y_{u,i}}, \quad \tilde{x} = \frac{c_p \dot{m}}{\lambda} x$$

$$\Rightarrow \frac{d\tilde{T}}{d\tilde{x}} + \frac{d\tilde{Y}}{d\tilde{x}} = \frac{d^2\tilde{T}}{d\tilde{x}^2} + \frac{1}{Le} \frac{d^2\tilde{Y}}{d\tilde{x}^2} \xrightarrow{\int_{-\infty}} (\tilde{T} + \tilde{Y})|_{-\infty} = \frac{d\tilde{T}}{d\tilde{x}} + \frac{1}{Le} \frac{d\tilde{Y}}{d\tilde{x}}$$

$$(\frac{d\tilde{T}}{d\tilde{x}}|_{\tilde{x}=-\infty} = 0, \quad \frac{d\tilde{Y}}{d\tilde{x}}|_{\tilde{x}=-\infty} = 0, \quad \tilde{T}|_{\tilde{x}=-\infty} = \tilde{T}_u, \quad \tilde{Y}|_{\tilde{x}=-\infty} = Y_u/Y_u = 1)$$

$$\Rightarrow (\tilde{T} + \tilde{Y}) - (\tilde{T}_u + 1) = \frac{d\tilde{T}}{d\tilde{x}} + \frac{1}{Le} \frac{d\tilde{Y}}{d\tilde{x}} \quad (\text{no simplification!})$$

Suppose $Le=1$, and denote $\tilde{W} = \tilde{T} + \tilde{Y}$, then it becomes a simple ODE:

$$\frac{d\tilde{W}}{d\tilde{x}} = \tilde{W} + C_0 \quad (C_0 = \text{const}).$$

$$\Rightarrow \tilde{W} = C' e^{\tilde{x}} - C_0 \Rightarrow C' = 0, \text{ otherwise } \tilde{W} \text{ diverges!} \Rightarrow \tilde{W} = \text{const!}$$

(Theoretical Solution!)

§ Flame Thickness & Remarks on Q1-HW3

Directly follows from the thermal theory, with $\frac{dT}{dx}|_{x=x_i}$ being approximated as:

$$\frac{dT}{dx}|_{x=x_i} \doteq \frac{T_f - T_0}{\delta_f}$$

$$\rho S_L c_p (T_i - T_0) = \lambda \frac{dT}{dx}|_{x=x_i} \Rightarrow \delta_f = \frac{\lambda}{\rho c_p} \frac{T_f - T_0}{T_i - T_0} \frac{1}{S_L}$$

if S_L is given / measured from experiments, then δ_f can be estimated provided that the thermal and transport properties are given! → ~ O(1) as $T_f \sim T_i$

Typically, for $\text{CH}_4 - \text{O}_2$, $p = 1 \text{ atm}$, $T \approx 1500 \text{ K}$. $\delta_f \sim 1 \text{ mm}$.

for $\text{H}_2 - \text{O}_2$, $p = 1 \text{ atm}$, $T \approx 1500 \text{ K}$ $\delta_f \sim 0.3 \text{ mm}$.

§ Thermal Explosion.

chemical reactions promotes the temperature! , Constant Volume!

* Adiabatic.

$$\rho C_v \frac{dT}{dt} = B \cdot Q_c C_F e^{-T_0/T} = -Q_c \frac{dC_F}{dt} \quad (\text{balance})$$

Simplify with non-dimension variables:

$$\tilde{T} = \frac{C_v \rho_0}{Q_c \cdot C_{F,0}} T, \quad \tilde{C}_F = \frac{C_F}{C_{F,0}} \Rightarrow \frac{d\tilde{T}}{d\tilde{t}} = -\frac{d\tilde{C}_F}{d\tilde{t}} = B \tilde{C}_F e^{(-\frac{\tilde{T}_0}{\tilde{T}})}$$

$$\Rightarrow \frac{d}{d\tilde{t}}(\tilde{T} + \tilde{C}_F) = 0 \Rightarrow \tilde{T} + \tilde{C}_F = \text{const} \Rightarrow \tilde{T} - \tilde{T}_0 = 1 - \tilde{C}_F$$

At initial ($t \rightarrow 0$). $\tilde{T} \sim \tilde{T}_0$, $\tilde{C}_F \sim 1$

perturbation: \tilde{T} as: $\tilde{T} = \tilde{T}_0 + \epsilon \theta(t) + O(\epsilon^2)$, $\epsilon = \frac{\tilde{T}_0^2}{\tilde{T}_a} \ll 1$.

$$\text{with } \tilde{t} = \frac{B e^{-\tilde{T}_0/\tilde{T}_0}}{\epsilon} t, \Rightarrow \frac{d\theta}{d\tilde{t}} = e^{(-\frac{\tilde{T}_0}{\tilde{T}} + \frac{\tilde{T}_0}{\tilde{T}_0})} = e^{\frac{\tilde{T}_0}{\tilde{T}_0}(\tilde{T} - \tilde{T}_0)}$$

$$= e^{\frac{\tilde{T}_0}{\tilde{T}_0} \epsilon \theta} \xrightarrow{t \rightarrow 0} e^{\theta}$$

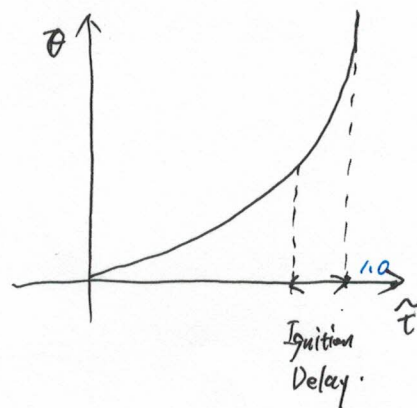
$$\Rightarrow \theta = -\ln(1 - \tilde{t}) \Rightarrow \tilde{T} \text{ diverges when } \tilde{t} \rightarrow 1 \Rightarrow t_I = \tau_v \frac{T_0^2}{T_a} \frac{e^{T_0/T_0}}{Q_c Y_{F,0} B}$$

* Non-Adiabatic

Difference: Heat loss through boundary!

Employ the notion learnt in heat transfer course:

$$-Sh(T - T_0)$$



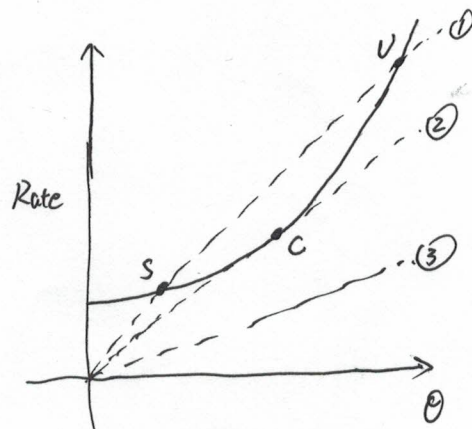
Governing Equation:

$$V \rho_0 \cdot C_v = V \cdot Q_c \cdot B C_F e^{-T_0/T} - Sh(T - T_0)$$

$$\text{Again: } \tilde{T} = \frac{C_v \rho_0}{Q_c \cdot C_{F,0}} T, \quad \tilde{C}_F = \frac{C_F}{C_{F,0}}, \quad \tilde{t} = t/\tau_c$$

$$\text{New: } \tilde{\tau}_c = \frac{\epsilon}{B e^{-\tilde{T}_0/\tilde{T}_0}}, \quad \tilde{\tau}_h = \frac{C_v \rho_0}{(Sh/V)h}, \quad \tilde{h} = \frac{\tau_c}{\tilde{\tau}_h}$$

$$\Rightarrow \frac{d\theta}{d\tilde{t}} = e^{\theta} - \tilde{h}\theta$$



§ Ignition and Distinction (Well-Stirred Reactor Analogy) & Remarks on Q4-HW3

Governing Equation: (Energy Balance)

$$\dot{V} \rho C_p (T_f - T_0) = V \cdot Q_c \cdot B \cdot C_f \cdot e^{-T_a/T_f}$$

Use C_p instead of C_v !
Open system, flow exists!

V : reactor volume, T_0 : initial temperature of mixture.

T_f : Reaction temperature. C_f : Reactant concentration in the burner.

Similar non-dimensional form: (with $\tilde{T} - \tilde{T}_0 = 1 - \tilde{C}_F$)

$$\tilde{T}_f - \tilde{T}_0 = Da \cdot (\tilde{T}_{ad} - \tilde{T}_f) e^{-\tilde{T}_a/\tilde{T}_f}$$

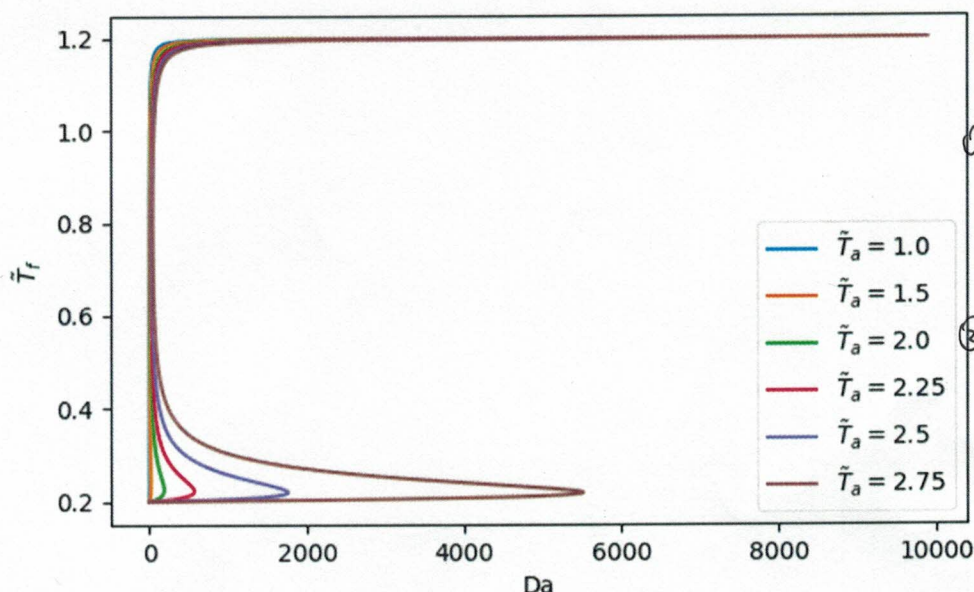
where:

$$\tilde{T} = \frac{C_p}{Q_c \cdot Y_{F,0}} T, \quad \tilde{T}_{ad} - \tilde{T}_f = \tilde{C}_F, \quad \tilde{T}_{ad} = 1 + \tilde{T}_0$$

and $Da = \frac{B}{\dot{V}/V} = \frac{\dot{V}/V}{1/B} = \frac{\text{Flow time}}{\text{Reaction time}}.$

In numerical calculation, \tilde{T}_0 is pre-defined, and Da is calculated with given \tilde{T}_f .

For example, take $\tilde{T}_0 = 0.2$, then $\tilde{T}_{ad} = 1.2$, $\tilde{T}_f \in (\tilde{T}_0, \tilde{T}_{ad})$, the plot of $\tilde{T}_f - Da$ is shown as:



Understanding:

① $Da \uparrow \Rightarrow$ Flow time \uparrow

Enough time of reaction!

\hookrightarrow Ignition!

② $Da \downarrow \Rightarrow$ Flow time \downarrow

Insufficient reaction

\hookrightarrow Extinction!