

4-6

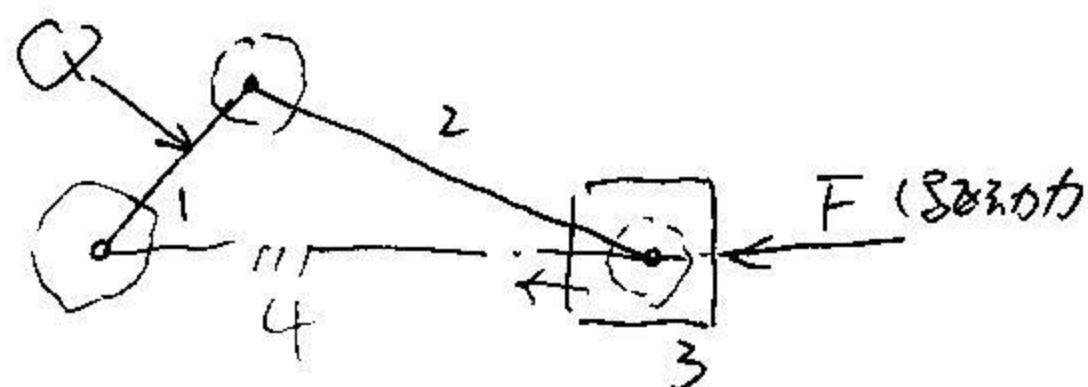
BC 为二力杆，曲柄 AB 逆时针转动

2. 受压

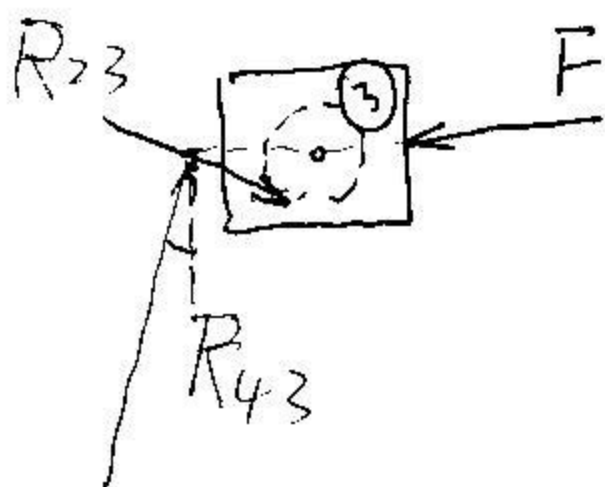
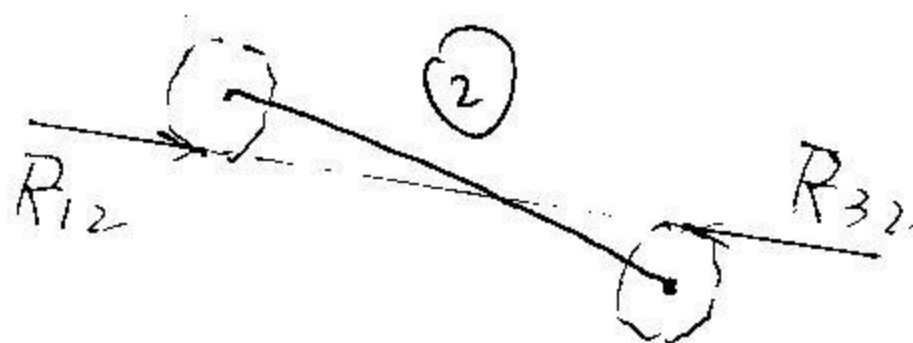
2 相对 1 顺时针转动

2 相对 3 顺时针转动

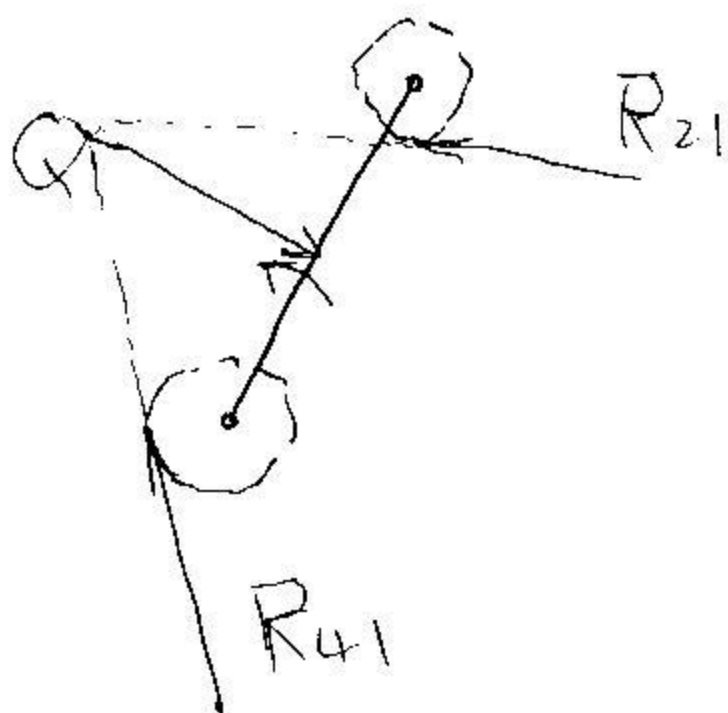
因此，杆件 2 的受力如图



2.



1.



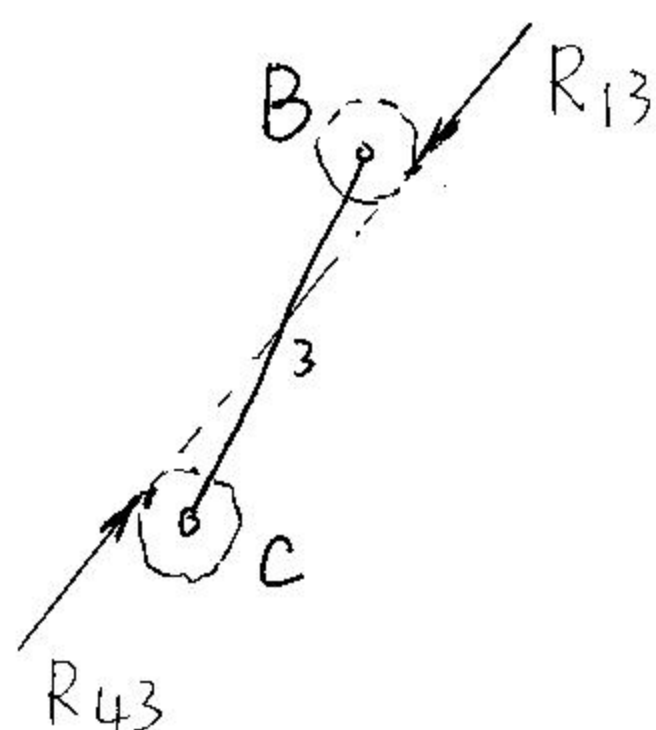
4-8

BC 为二力杆, 受压力, 逆时针方向转动 (相对机构)

③

BC 相对于 AB ① 逆时针

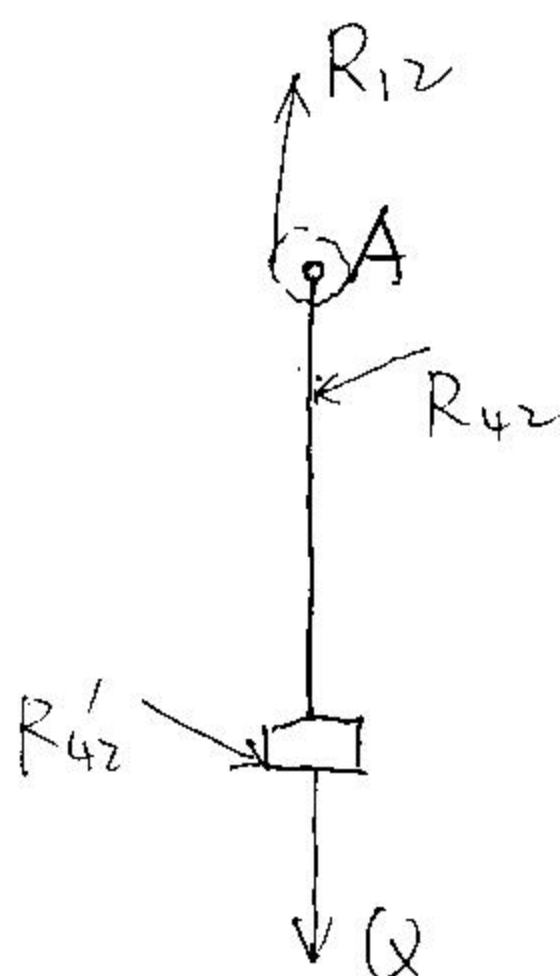
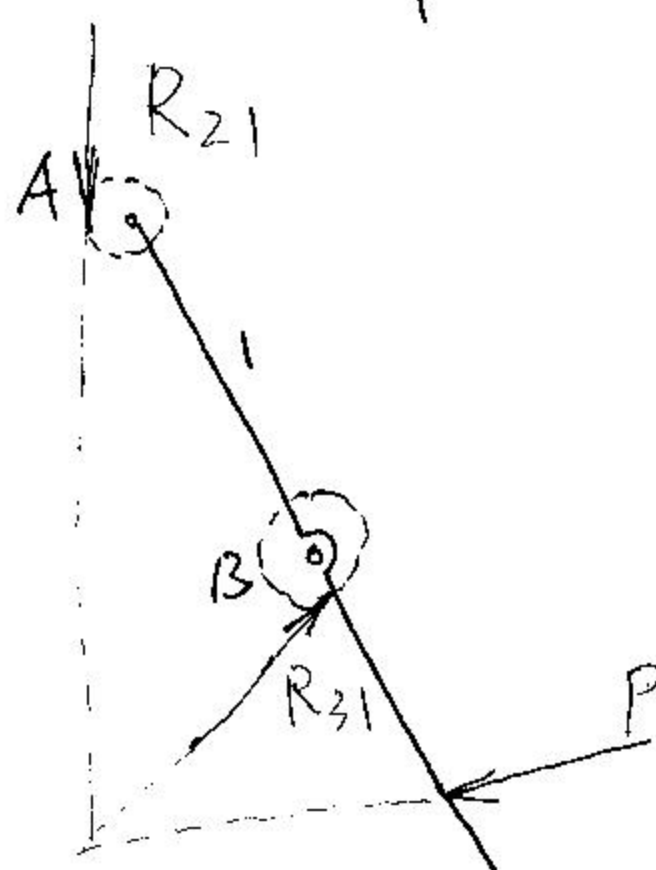
BC ③ 受力为



~~AB ① 相对~~

AB ① 相对于 ② 顺时针转动

A 点受力方向向下

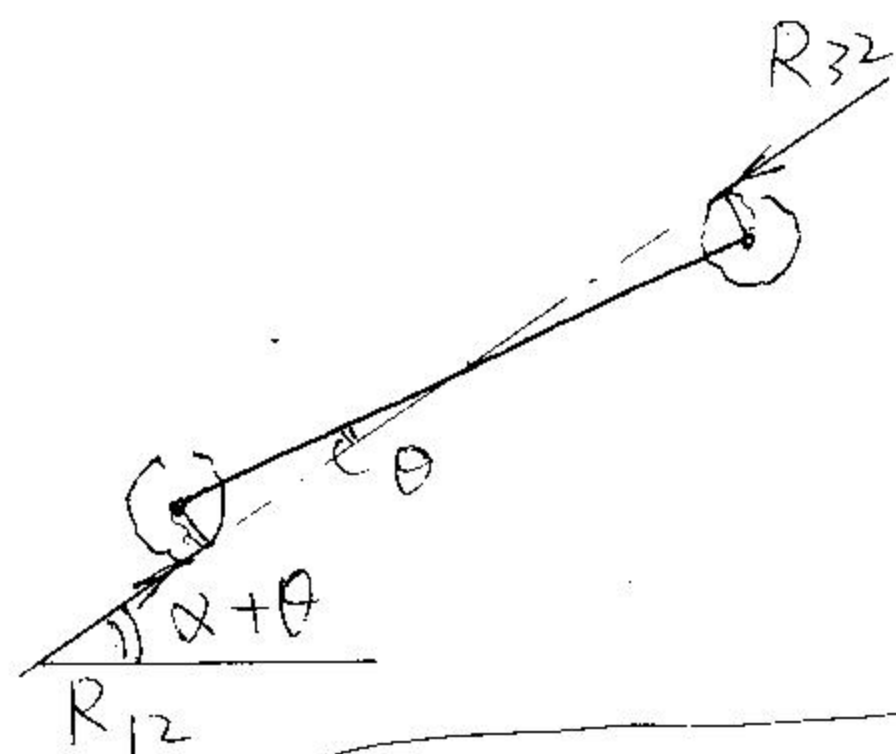


4-11 (1)

AB 为二力杆, 受压, ②

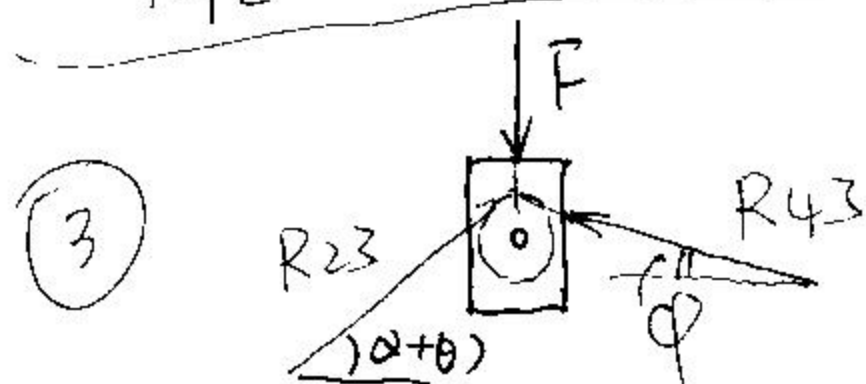
AB 相对于 ③ 顺时针转动

AB 相对于 ① 顺时针转动



$$\sin \theta = \frac{P}{l_2} = \frac{2P}{l} = \frac{2rf}{l}$$

$$\theta = \arcsin \frac{2rf}{l}$$



$$\varphi = \arctan f$$

$$\begin{cases} R_{43} \sin \varphi + R_{23} \sin(\alpha + \theta) = F \\ R_{43} \cos \varphi = R_{23} \cos(\alpha + \theta) \end{cases}$$

消去  $R_{43}$  :  $R_{23} \left[ \frac{\cos(\alpha + \theta)}{\cos \varphi} \times \sin \varphi + \sin(\alpha + \theta) \right] = F$

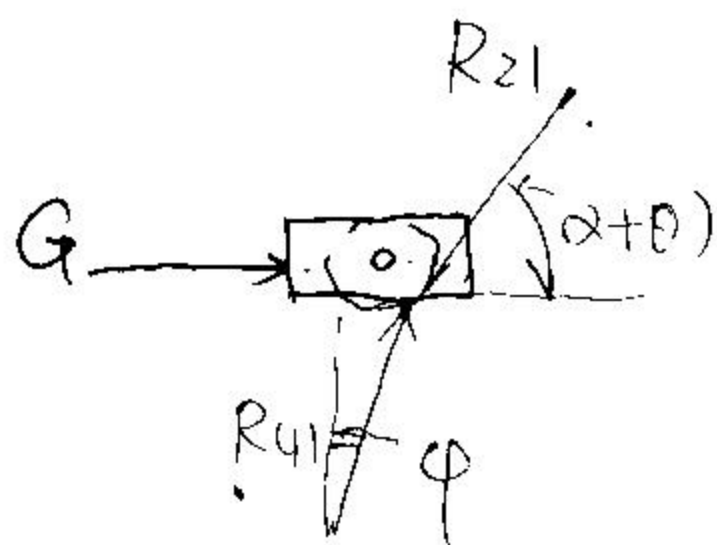
$$R_{23} [f \cos(\alpha + \theta) + \sin(\alpha + \theta)] = F$$

$$R_{23} = \frac{F}{f \cos(\alpha + \theta) + \sin(\alpha + \theta)}$$

$$\begin{cases} R_{21} \cos(\alpha + \theta) - R_{41} \sin \varphi = G \\ R_{21} \sin(\alpha + \theta) = R_{41} \cos \varphi \end{cases}$$

13 同样可得

$$R_{21} = \frac{G}{\cos(\alpha + \theta) - f \sin(\alpha + \theta)}$$



4-11 (2)

因  $R_{23} = R_{21}$

$$\therefore G = F \times \frac{\cos(\omega + \theta) - f \sin(\omega + \theta)}{f \cos(\omega + \theta) + \sin(\omega + \theta)}$$

无摩擦时  $G_0 = F \operatorname{ctg} \alpha$

效率

$$\boxed{\eta = \frac{G}{G_0}} = \frac{\cos(\omega + \theta) - f \sin(\omega + \theta)}{f \cos(\omega + \theta) + \sin(\omega + \theta)} \times \operatorname{tg} \alpha$$

自锁条件  $\eta = 0$

(1) 由  $\operatorname{tg} \alpha = 0$  得  $\alpha = 0 \rightarrow$  输出为 0

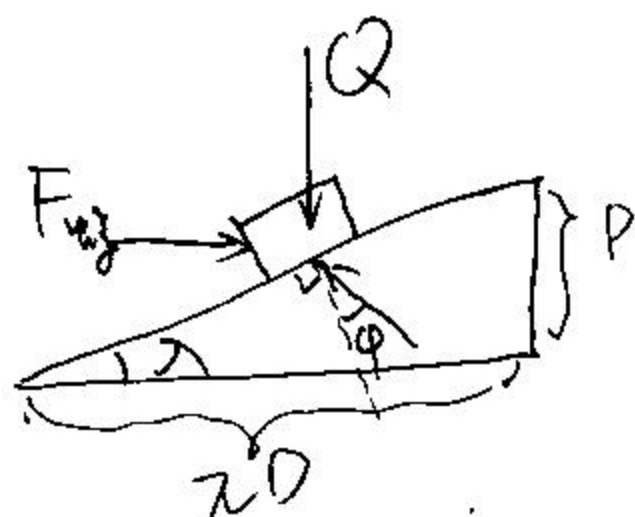
(2)  $\cos(\omega + \theta) - f \sin(\omega + \theta) = 0$

$$\operatorname{ctg}(\omega + \theta) = f$$

$$\alpha = \operatorname{arccctg} f - \theta$$

4-14: 蜗轮蜗杆

蜗轮中径  $D = 22 \text{ mm}$   $P = 4 \text{ mm}$



$$\lambda = \arctan \frac{P}{2D} = 3.3^\circ$$

$$\phi = \arctan f = \arctan 0.1 = 5.7^\circ$$

$$F_t = Q \tan(\lambda + \phi) = 0.158Q \text{ (N)}$$

$$\text{蜗轮面阻转矩 } M_t = F_t \times \frac{D}{2} = 1.738Q \text{ (N}\cdot\text{mm)}$$

蜗轮面 (环面)

环面蜗轮中径  $D_{\text{环}} = 46 \text{ mm}$

$$\text{蜗轮力 } F_{\text{环}} = Q \times f = 0.1Q$$

$$\text{蜗轮力矩 } M_{\text{环}} = F_{\text{环}} \times \frac{D_{\text{环}}}{2} = 0.1Q \times 23 = 2.3Q \text{ (N}\cdot\text{mm)}$$

$\therefore$  总阻力矩 = 蜗轮力矩

$$P \times l = M_t + M_{\text{环}}$$

$$P \times 300 = 1.738Q + 2.3Q$$

$$\boxed{Q = 75P}$$

当无蜗轮时  $F_t = 0.058Q$ ,  $M_t = 0.638Q$

$$M_{\text{环}} = 0$$

$$P \times 300 = 0.638Q_0 \quad Q_0 = 470P$$

$$\eta = Q/Q_0 = \frac{75}{470} = 15\%$$

有蜗轮  $P = 100 \text{ N}$ ,  $\boxed{Q = 75P = 7500 \text{ N}}$