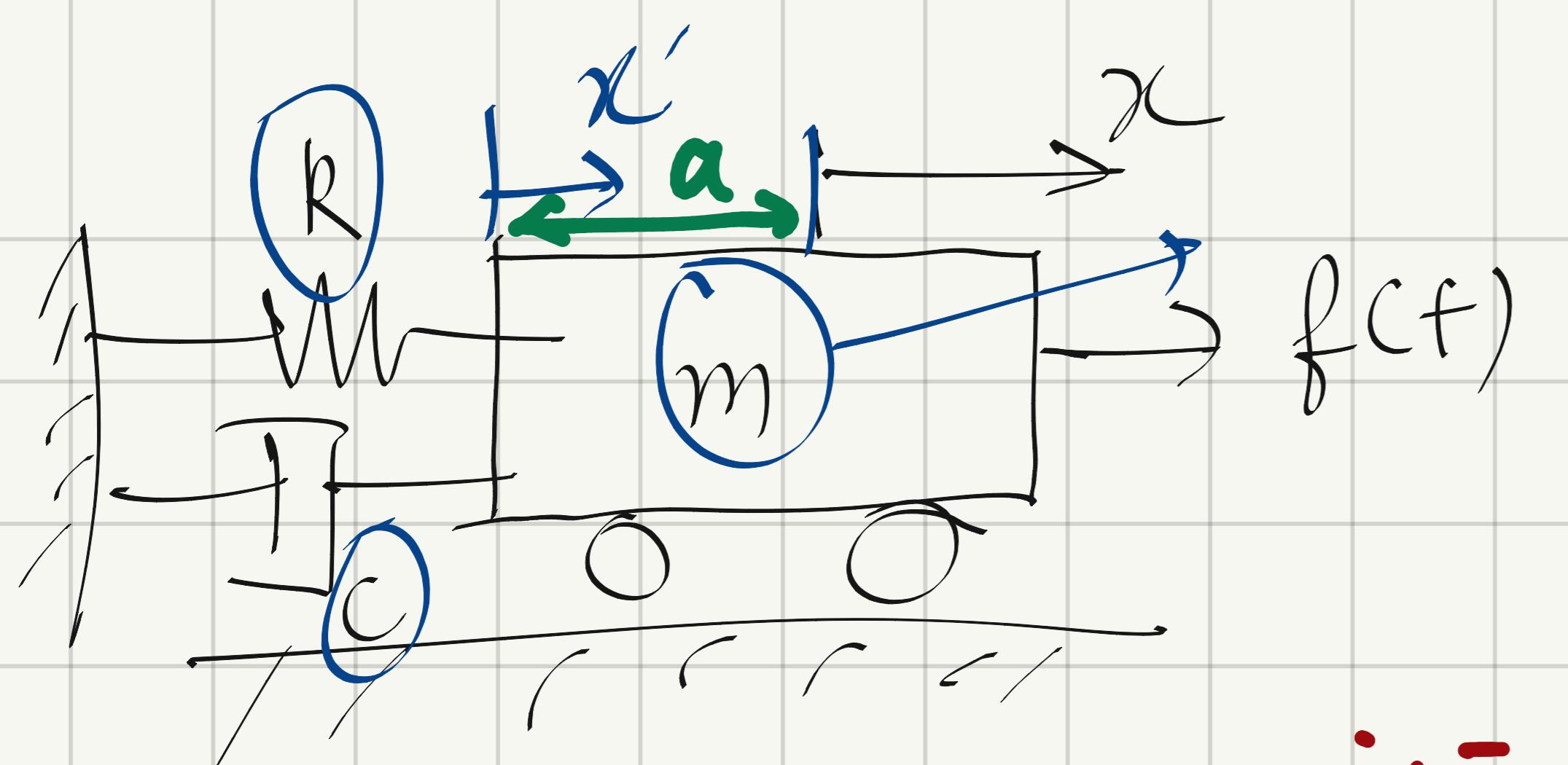
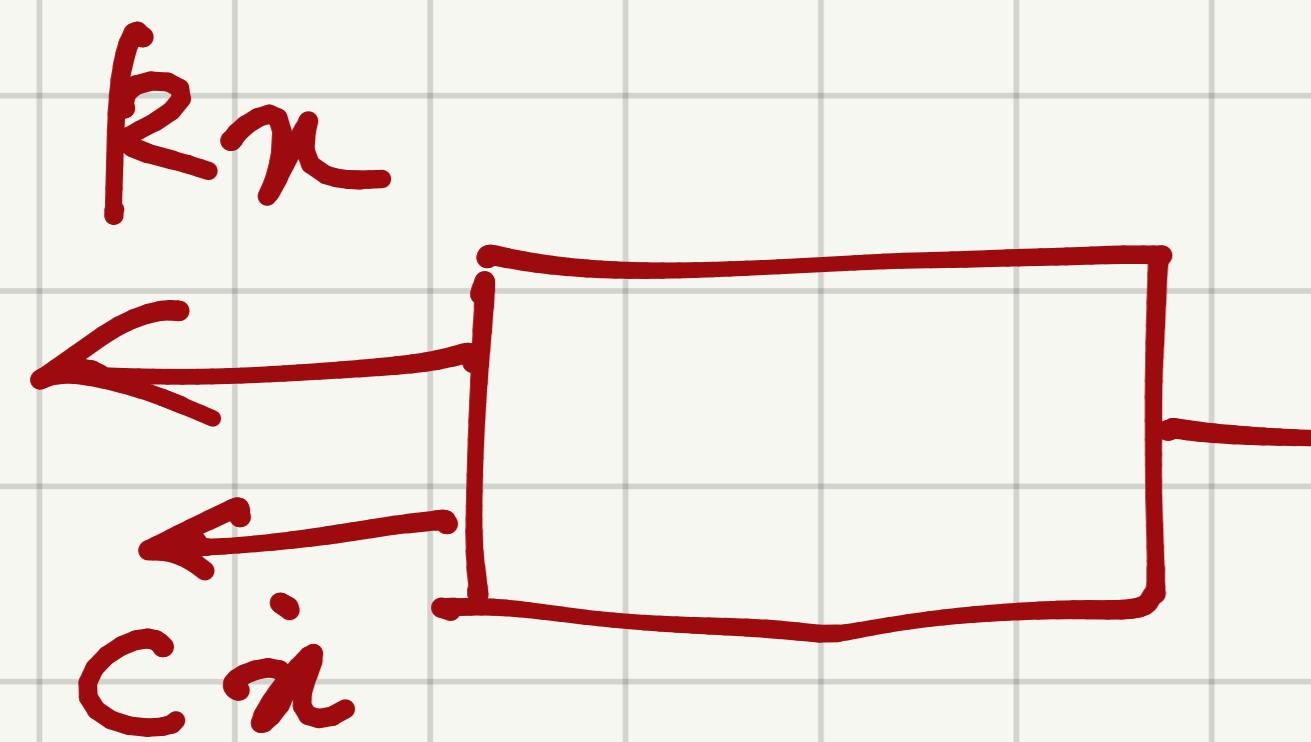


Laplace transforms

Modeling in frequency domain



$$F = ma$$



$$f(t) - kx - cx = m \ddot{x}$$

$$m \ddot{x} + c \dot{x} + kx = f(t)$$

$$\begin{aligned}\dot{x} &= \frac{dx}{dt} \\ \ddot{x} &= \frac{d^2x}{dt^2}\end{aligned}$$

Given Initial Conditions, system is solvable.

Solving for $x(t)$

2nd order

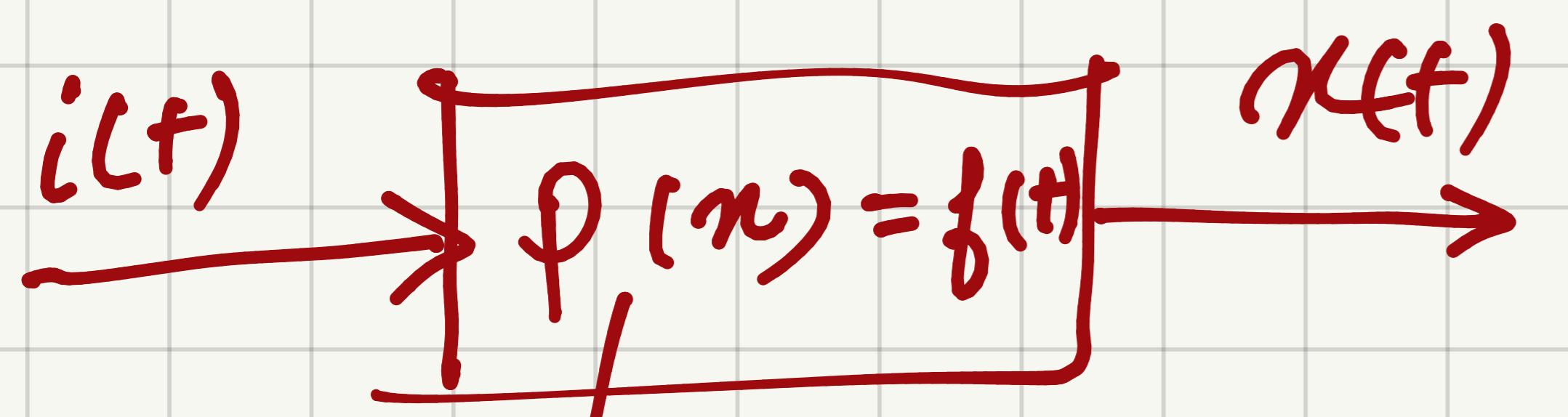
Non-Homogeneous
ODE

Time invariant
Linear

$$x = (x - a) \quad a \quad x = x - a$$

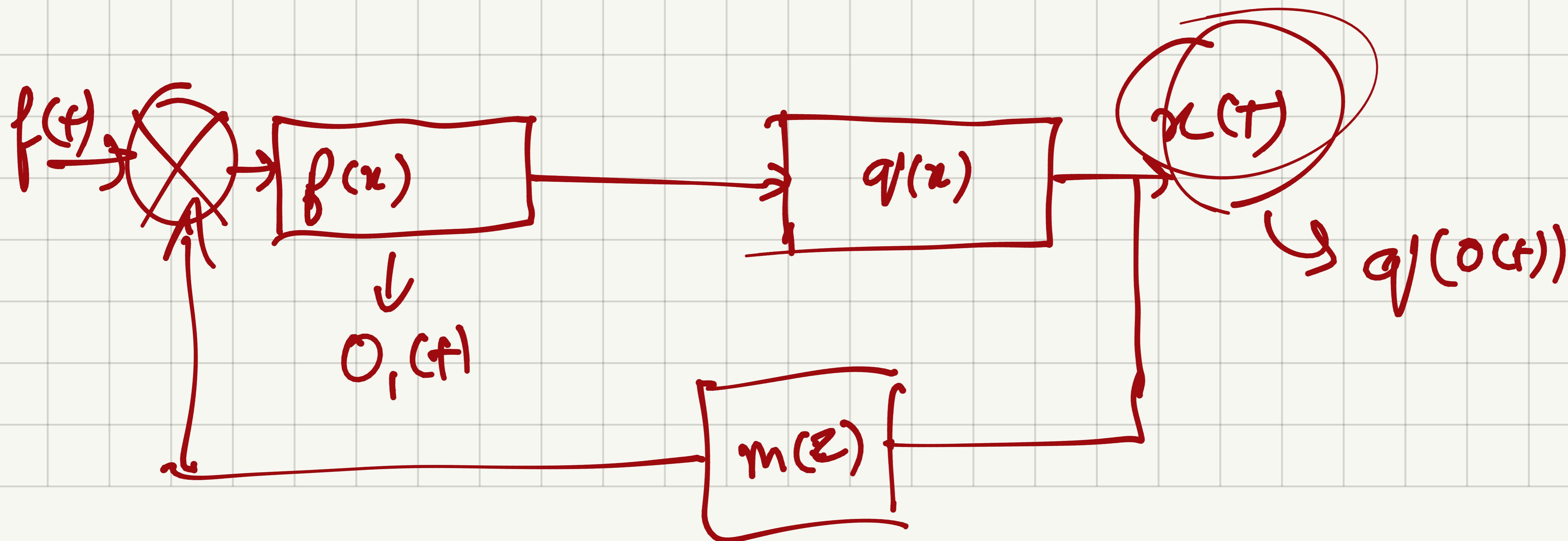
$$\begin{aligned}x &= x' \\ \dot{x} &= \dot{x}' \\ \ddot{x} &= \ddot{x}'\end{aligned}$$

$$f(x+a) = f(x) + f(a)$$



Differential
eqn.
for a system

$$x(t) = \text{sol}^m \text{ of } p(x) = f(t)$$



Laplace Transform

transform a differential eqn into algebraic equation

$$\mathcal{L} \left[f(t) \right] = F(s)$$

time domain \uparrow s -domain

$s = \sigma + j\omega$

$$f_0 = \int_0^\infty f(t) e^{-st} dt$$

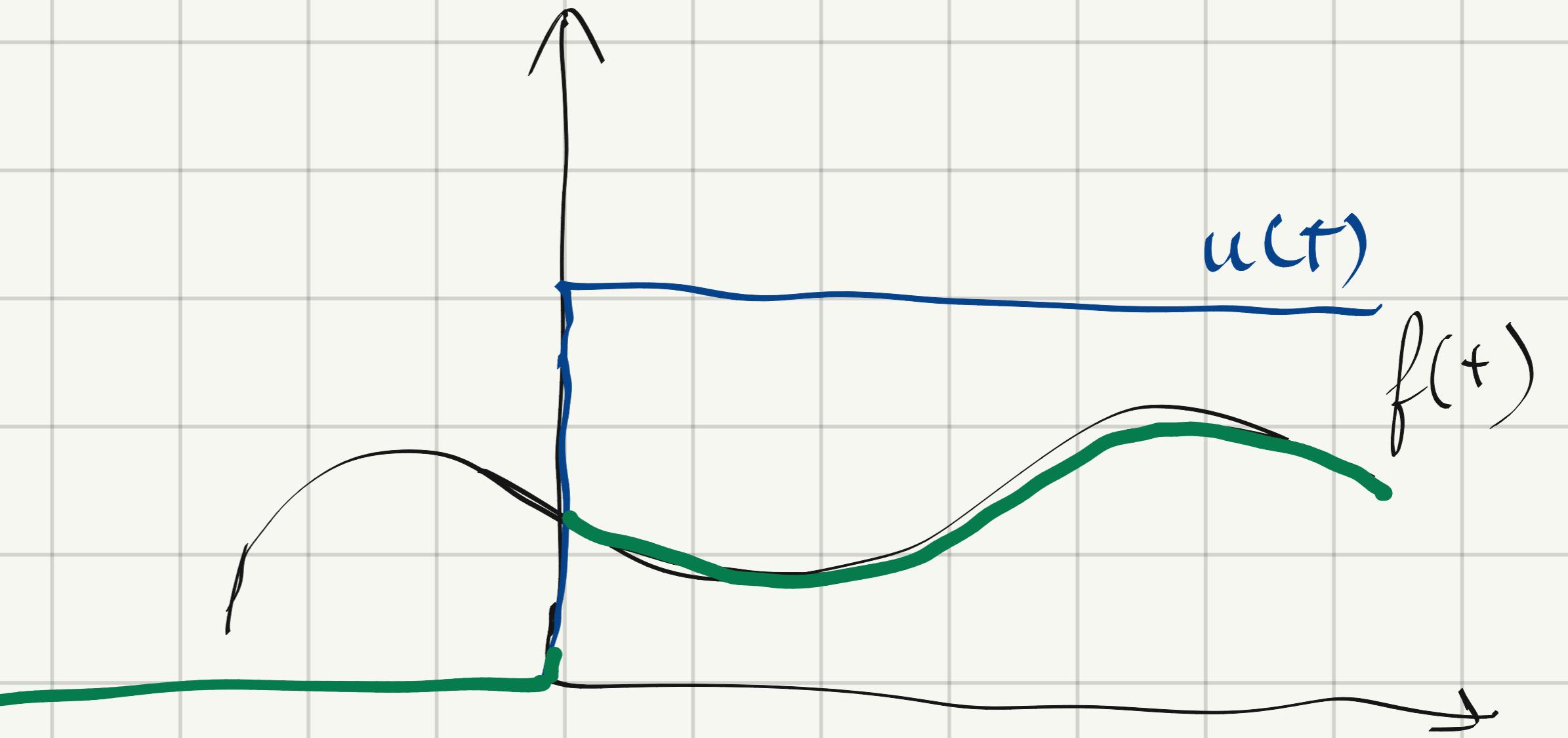
If there is a discontinuity at $t=0$

\mathcal{L} exists as long as $f(s)$ converges.

Inverse Laplace

$$\mathcal{L}^{-1}[F(s)] = \frac{1}{2\pi j} \int_{\sigma-j\omega}^{\sigma+j\omega} F(s) e^{st} ds = f(t) u(t)$$

unit step function



| Item no. | $f(t)$ | $F(s)$ |
|----------|----------------------------------|---------------------------------|
| 1. | <u>$\delta(t)$</u> | 1 |
| 2. | <u>$u(t)$</u> | $\frac{1}{s}$ |
| 3. | $t u(t)$ | $\frac{1}{s^2}$ |
| 4. | <u>$t^n u(t)$</u> | $\frac{n!}{s^{n+1}}$ |
| 5. | <u>$e^{-at} u(t)$</u> | $\frac{1}{s+a}$ |
| 6. | $\sin \omega t u(t)$ | $\frac{\omega}{s^2 + \omega^2}$ |
| 7. | $\cos \omega t u(t)$ | $\frac{s}{s^2 + \omega^2}$ |

$$\int_0^\infty t e^{-st} dt = \frac{1}{s^2}$$

$$\mathcal{L}[f(t)] = F(s)$$

$$\mathcal{L}[e^{-at} f(t)] = F(s+a)$$

$$\mathcal{L}\left[e^{at} t^n u(t)\right] = \frac{n!}{(s-a)^{n+1}}$$

$$\mathcal{L}\left[f(t) e^{-ct}\right] = \int f(t) e^{-(s+c)t} dt$$

Properties of Laplace Transform

$$\mathcal{L}[f(t)] = F(s)$$

$$\mathcal{L}[kf(s)]$$

$$= \int k f(t) e^{-st} dt = \boxed{k F(s)}$$

} linearity
theorem

$$\mathcal{L}[f_1(t) + f_2(t)]$$

$$= \int (f_1 + f_2) e^{-st} dt = \int f_1 e^{-st} dt + \int f_2 e^{-st} dt \\ = \boxed{F_1(s) + F_2(s)}$$

$$\mathcal{L}[e^{-at} f(t)]$$

$$= F(s+a) \rightarrow \text{Frequency shift theorem.}$$

$$\mathcal{L}[f(t-T)]$$

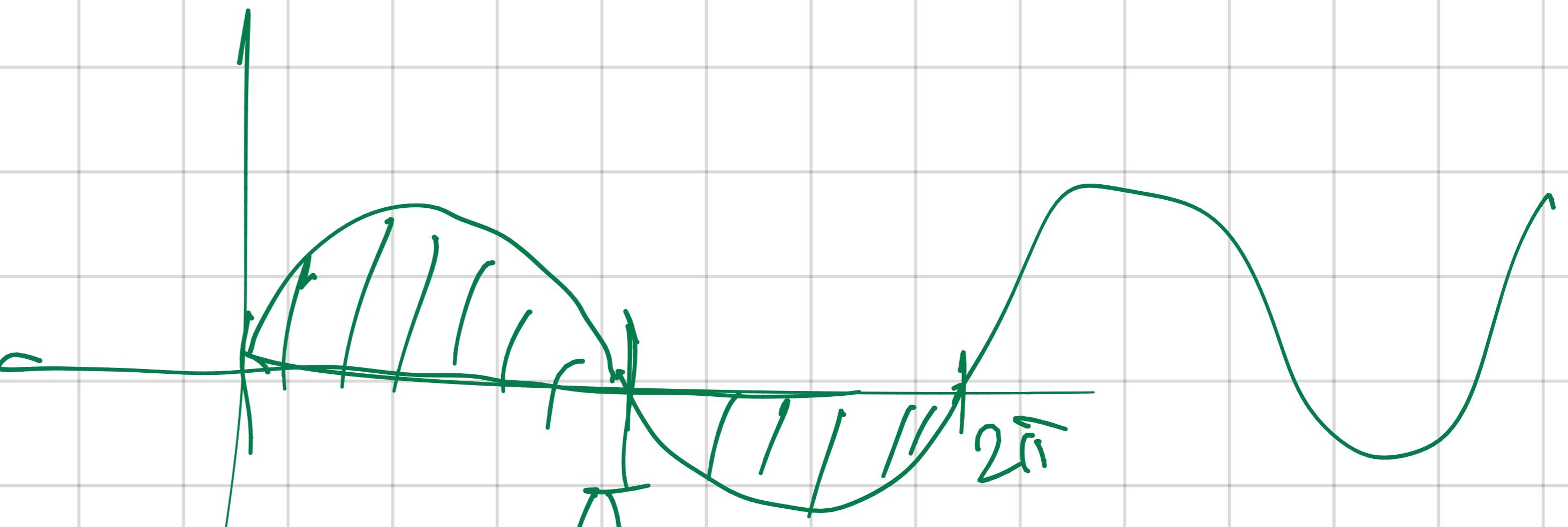
$$= \int_0^\infty f(t-T) e^{-st} dt = \int_{0+T}^\infty f(z) e^{-s(z-T)} dz = \boxed{e^{-sT} F(s)}$$

Replace $t-T=\tau$

Time shift theorem

$$\int_0^\pi \sin x \, dx = 1$$

$$\int_{-\infty}^{\infty} f(t) e^{-st} dt = \int_{-\infty}^{\infty} f(z) e^{-sz} dz$$



$$\underline{\underline{L[f(at)]}}$$

$$= \frac{1}{a} F\left(\frac{s}{a}\right)$$

Scaling theorem.

$$\underline{\underline{L\left[\frac{df}{dt}\right]}}$$

$$= sf(s) - \underbrace{f(0_-)}_{\text{Initial value of function.}}$$

$$\underline{\underline{L\left[\frac{d^2f}{dt^2}\right]}}$$

$$= s^2 f(s) - \underbrace{sf(0_-)}_{\cdot} - \underbrace{f'(0_-)}_{\text{initial value of first derivative.}}$$

$$\underline{\underline{L\left[\frac{d^n f}{dt^n}\right]}}$$

$$= s^n f(s) - \sum_{k=1}^n s^{n-k} f^{(k-1)}(0_-) \rightarrow \text{Sum of initial values upto } (n-1) \text{ derivative.}$$

C

Differentiation theorem.

$$\mathcal{L} \left[\int_{0^-}^t f(\tau) d\tau \right] = \frac{F(s)}{s}$$

Integration theorem.

$f(\infty)$
Final value

$f(0)$

Initial Value

$$= \lim_{s \rightarrow \infty} s F(s)$$

$$= \lim_{s \rightarrow 0} s F(s)$$

Final value theorem

* For this to yield correct result roots of denominator of $F(s)$ must be (-)ve with no more than 1 root at origin.

$$F(s) = \frac{1}{s(s+1)(s+2)}$$

$$f(\infty) = \frac{1}{2}$$

$$\frac{1}{s^2(s+1)(s+2)}$$

$$f(\infty) = \infty \quad N \neq N$$

Solution of a differential equation

$$\boxed{\frac{d^2y}{dt^2}} + 12 \frac{dy}{dt} + 32y = 32u(t) \quad \checkmark$$

given all I.C.s are 0.

$$\mathcal{L}[\text{LHS}] = \mathcal{L}[\text{RHS}]$$

$$s^2 Y(s) - \cancel{sY(0)}^{\rightarrow 0} - \cancel{y(0)}^{\rightarrow 0} + 12s Y(s) - \cancel{y'(0)}^{\rightarrow 0} + 32Y(s) = \frac{32}{s}$$

$$Y(s) [s^2 + 12s + 32] = \frac{32}{s}$$

$$Y(s) = \frac{32}{s(s^2 + 12s + 32)}$$

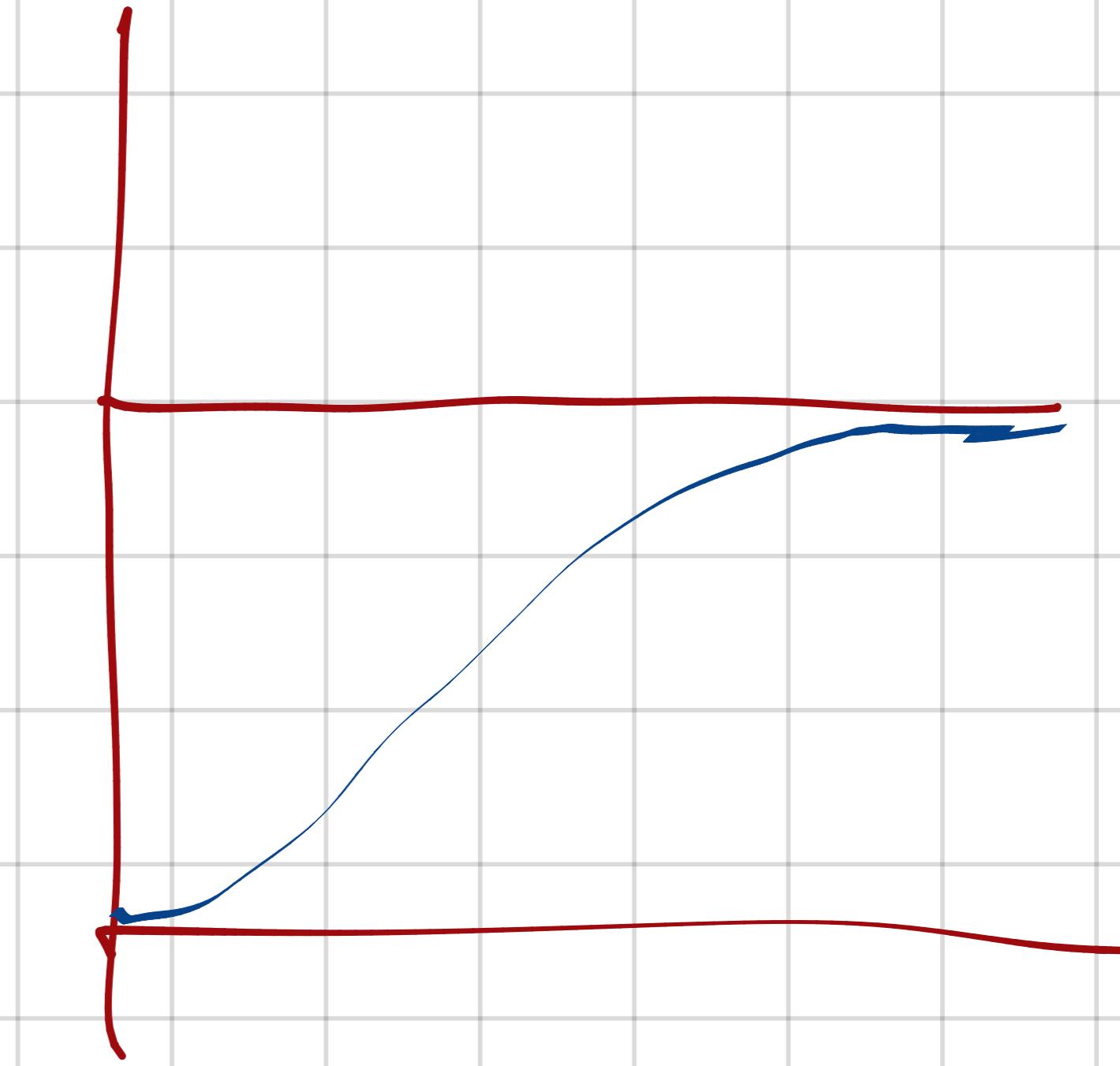
$$\frac{32}{s(s+4)(s+8)}$$

$$= \frac{K_1}{s} + \frac{K_2}{(s+4)} + \frac{K_3}{(s+8)}$$

$$K_1 = 1$$

$$K_2 = -2$$

$$K_3 = 1$$



$$\mathcal{L}^{-1}[Y(s)] = u(t) - 2e^{-4t}u(t) + e^{-8t}u(t)$$

↑ decay ↓ decay

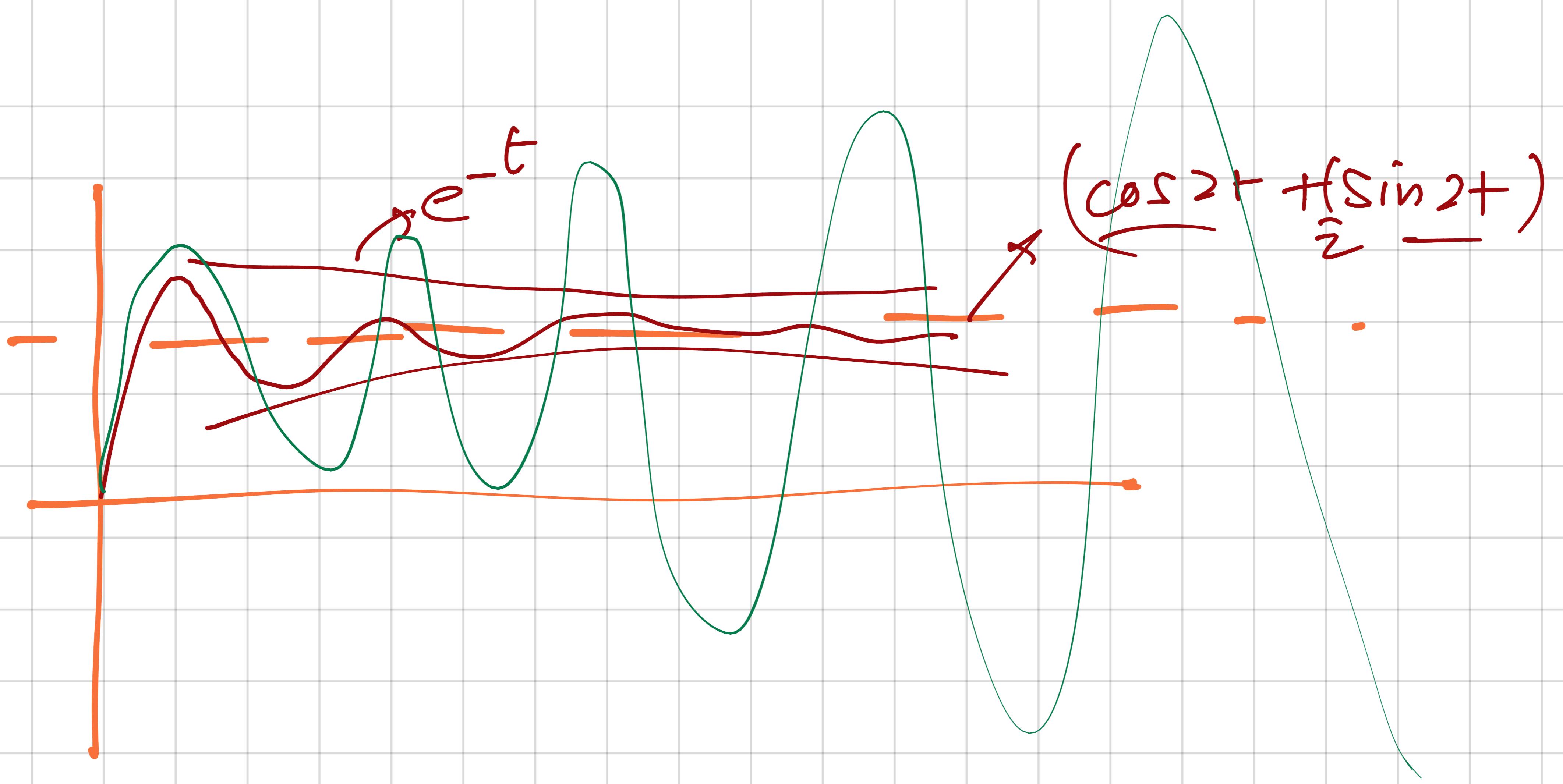
What if roots are complex?

$$s=0, -1 \pm 2i$$

eg- $f(s) = \frac{3}{s(s^2+2s+5)}$ = $\frac{3}{5} \frac{1}{s} - \frac{3}{5} \left(\frac{s+2}{s^2+2s+5} \right)$

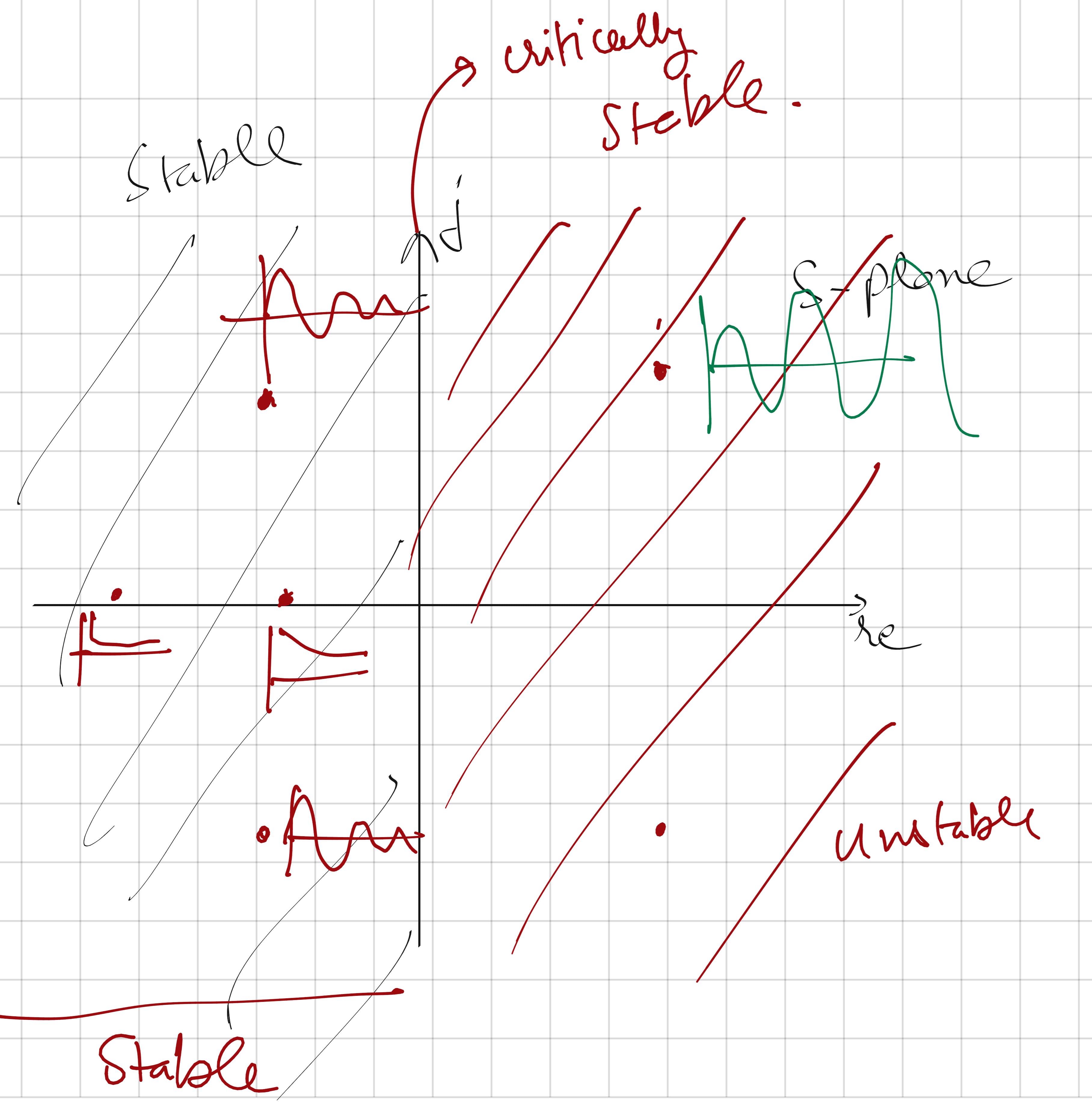
$$= \frac{3}{ss} - 3 \left[\frac{s+1}{\underline{(s+1)^2 + 2^2}} + \frac{1 \times 2}{2 \cdot (s+1)^2 + 2^2} \right]$$
$$\frac{s}{s^2+\omega^2}$$

$$\mathcal{L}^{-1} \left\{ f(s) \right\} = \frac{3}{5} u(t) - \frac{3}{5} e^{-t} \left[\cos 2t + \frac{1}{2} \sin 2t \right]$$



For a system to have a stable response, roots of denominator must lie in the left half of s-plane.

Left Half of s-Plane



critically
stable .

Roots of denominator
determine stability
of system

TRANSFER FUNCTIONS

$$a_n \frac{d^n c(t)}{dt^n} + a_{n-1} \frac{d^{n-1} c(t)}{dt^{n-1}} + \cdots + a_0 c(t) = b_m \frac{d^m r(t)}{dt^m} + b_{m-1} \frac{d^{m-1} r(t)}{dt^{m-1}} + \cdots + b_0 r(t)$$

$c(f) \rightarrow$ output.

$r(t) \rightarrow \underline{\text{input}}$

$$\underline{C(t)} = \underline{F} \cdot \underline{r(t)} \rightarrow \text{Desired relationship - to find the response}$$

Take Laplace on both sides:

$$\frac{a_n s^n c(s) + a_{n-1} s^{n-1} \underline{c(s)} + \dots + a_0 \underline{c(s)}}{\underline{}} + \begin{array}{l} \text{initial condition} \\ \text{terms involving } c(t) \\ \& \text{its derivatives.} \end{array}$$

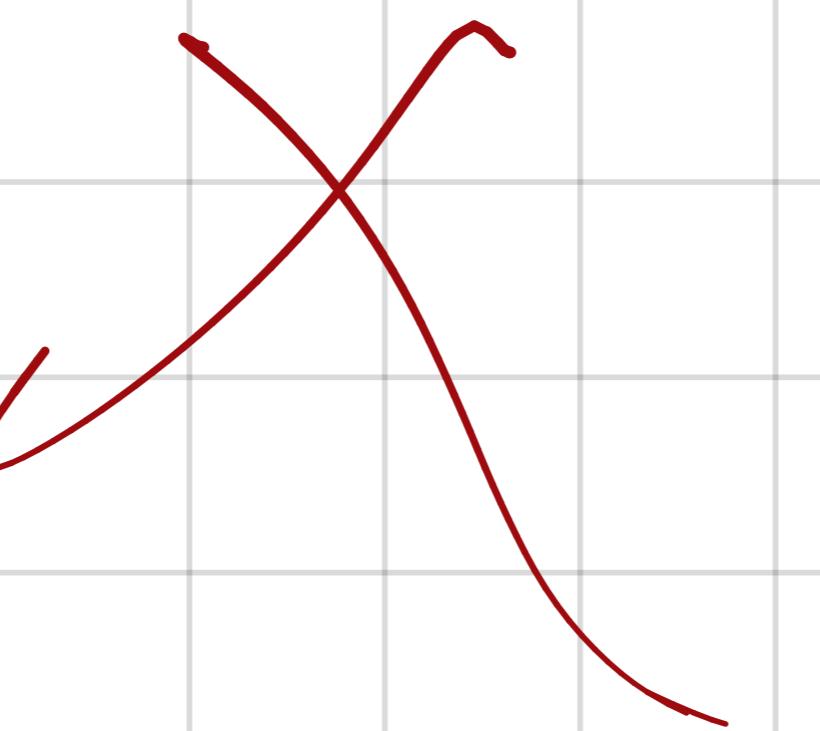
$$= b_m s^m R(s) + b_{m-1} s^{m-1} \underbrace{R(s)}_{\text{I.C. terms of } r(t) \text{ & derivatives.}} + \dots + b_0 R(s)$$

if all I.C. are 0

$$\frac{C(s)}{R(s)} = G(s) = \frac{(b_m s^m + b_{m-1} s^{m-1} + \dots + b_0)}{(a_n s^n + a_{n-1} s^{n-1} + \dots + a_0)}$$

Transfer function

$$C(t) = r(t) g(t)$$

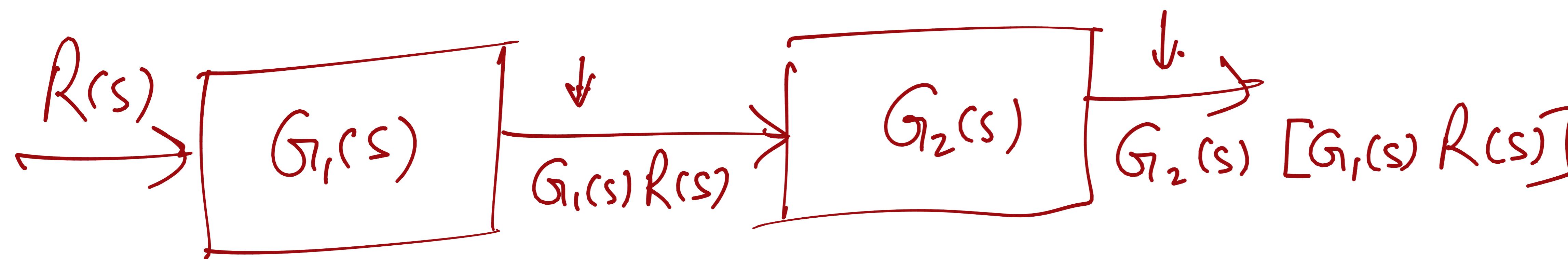
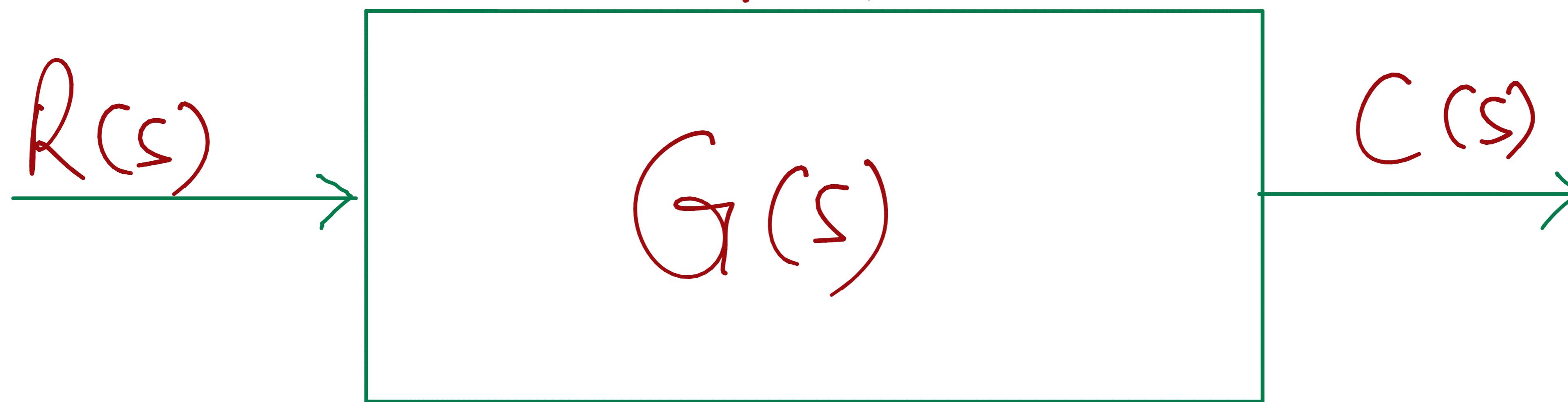


$$\mathcal{L}^{-1}[G(s) \cdot f(s)] \neq g(t) f(t)$$

$$C(t) = f(r(t))$$

$$\mathcal{L}^{-1}[C(s)] = \mathcal{L}^{-1}[G(s) \cdot R(s)]$$

• Transfer function •



System response from transfer function

$$m\ddot{x} + c\dot{x} + Rx = \underline{r(t)}$$

Transfer function

$$G(s) \rightarrow \frac{X(s)}{R(s)} = ?$$

$$G(s) = \frac{1}{ms^2 + cs + R}$$

$$X(s) = R(s) \cdot \frac{1}{ms^2 + cs + R}$$

$$m=1, c=3, R=2 \quad (\text{say})$$

$$(s^2 + 3s + 2) = (s+2)(s+1)$$

Input

Impulse

R_{CC}

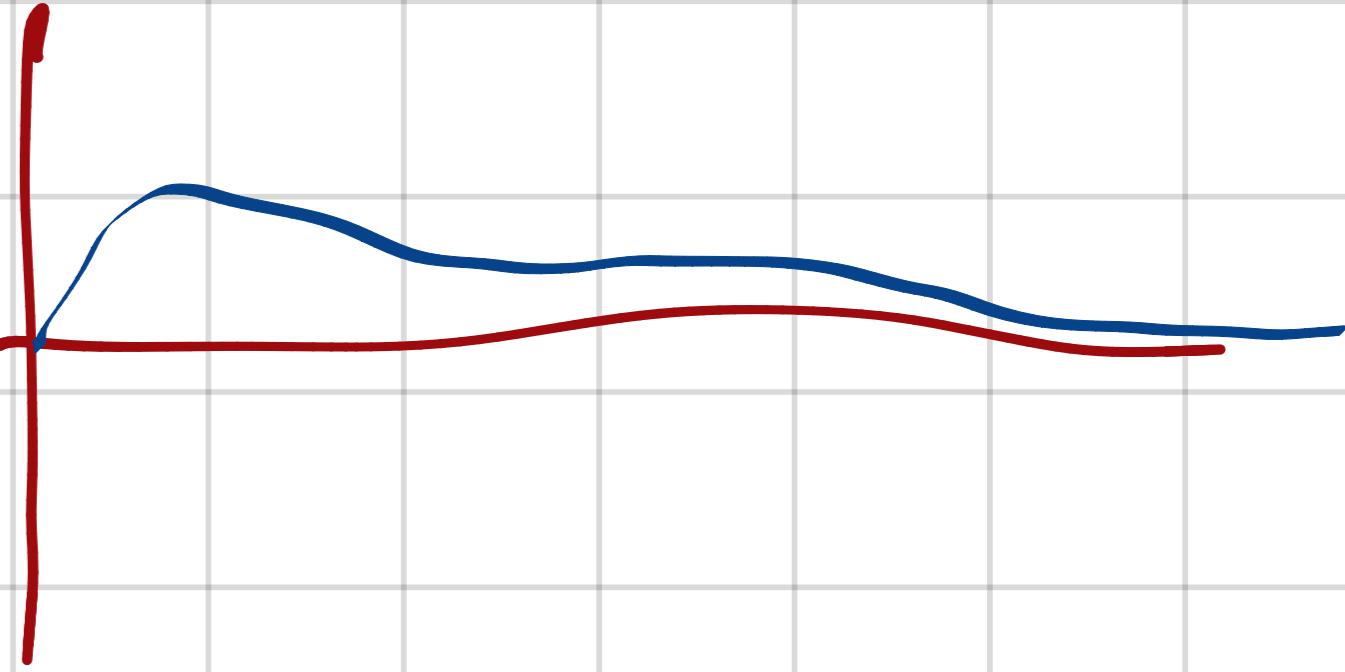
1

$X(s)$

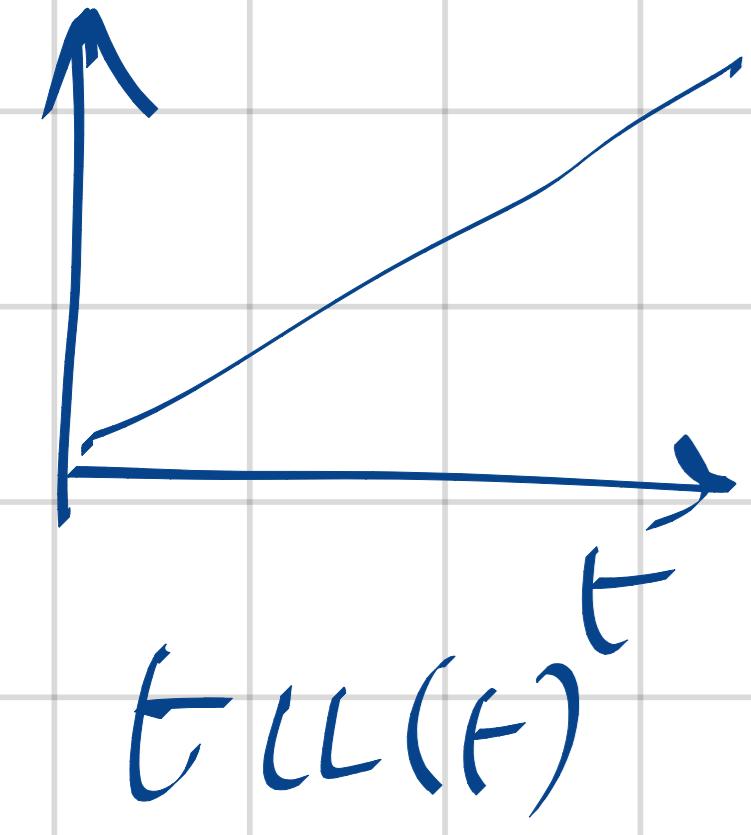
$$\frac{1}{(s+1)(s+2)}$$

$x(t)$

$$[e^{-t} - e^{-2t}] u(t)$$



Remainder
I/P.



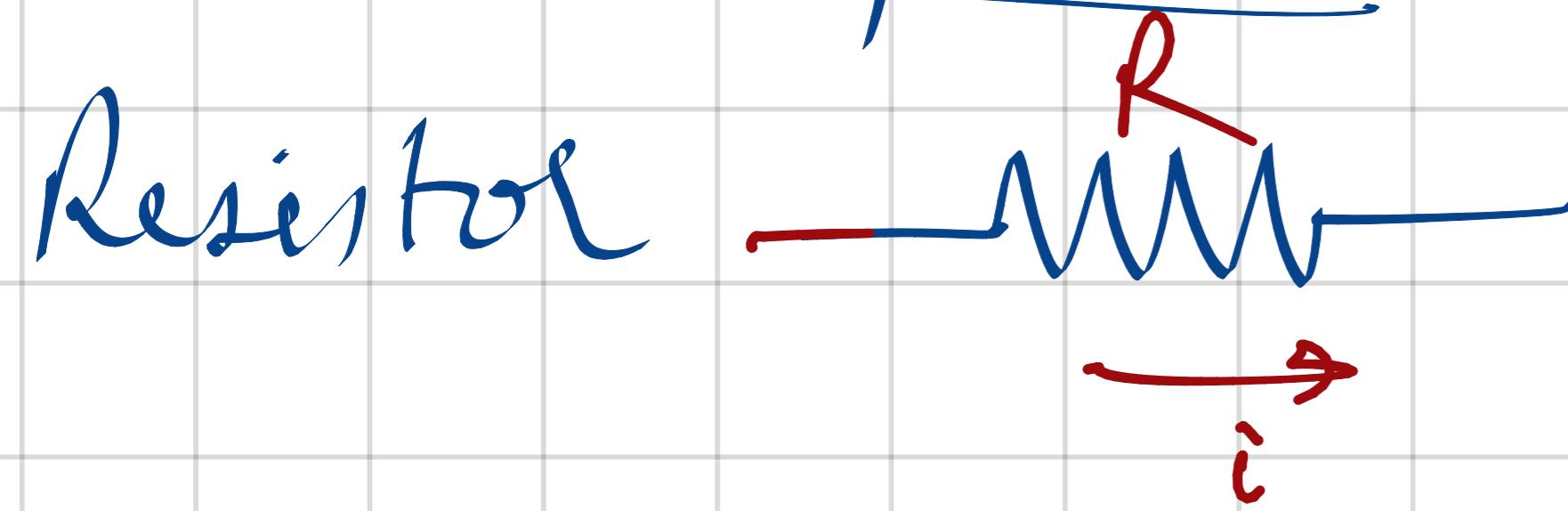
$$\frac{1}{s^2}$$

$$\frac{1}{s^2} \frac{1}{(s+1)(s+2)}$$

Real Systems

Electrical Systems

Passive Components



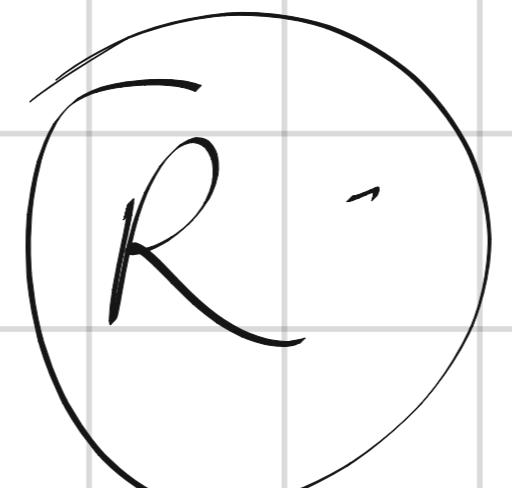
$V \rightarrow I$

$$V(t) = R i(t)$$

$$V(t) = \frac{1}{C} \int i dt$$

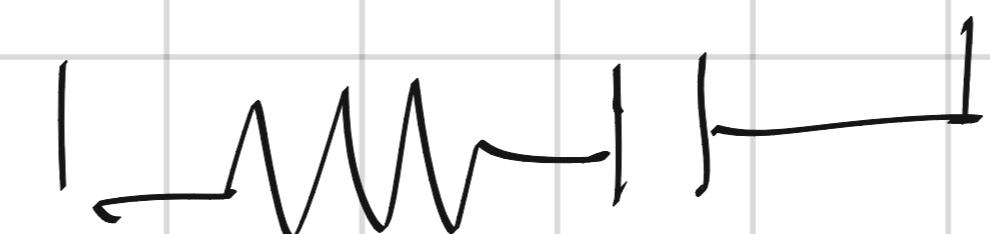
$$V(t) = L \frac{di}{dt}$$

Impedance $Z(s)$
Transfer function
 $V(s) / I(s)$



$$\frac{1}{Cs}$$

$$Ls$$



$V - q$

$$V = R \frac{dq}{dt}$$

$$V = \frac{q}{C}$$

$$V = L \frac{d^2q}{dt^2}$$

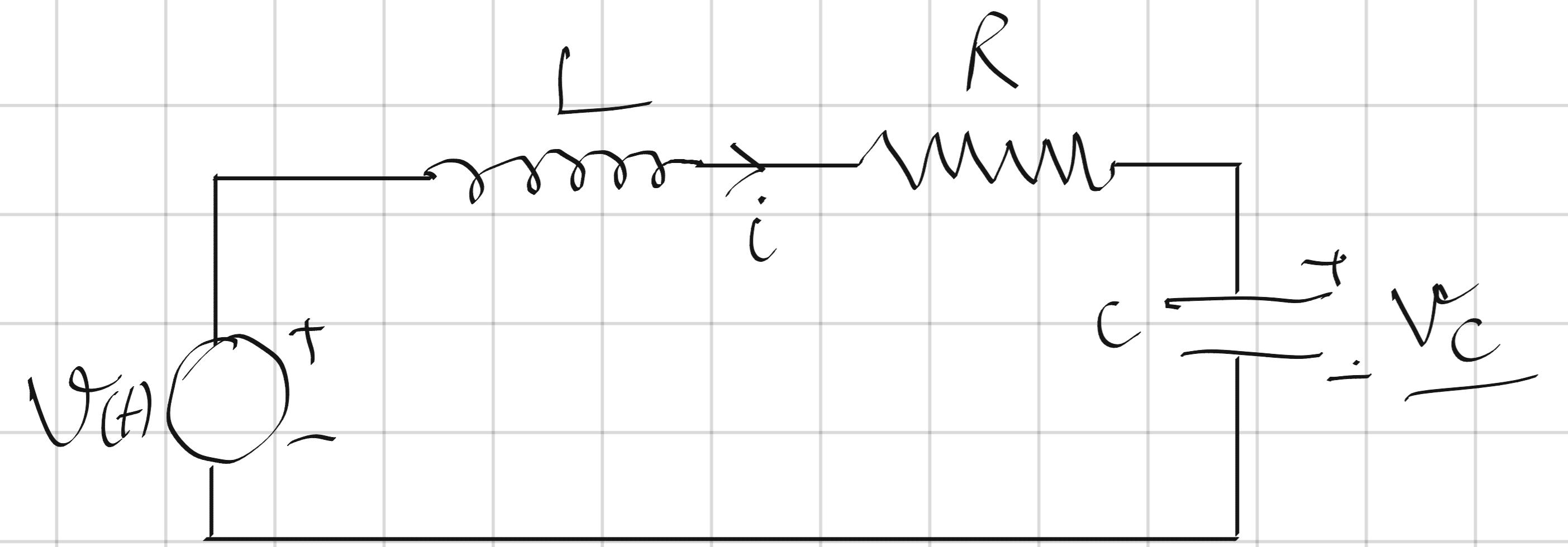
Admittance
 $I(s) / V(s)$

$$\frac{1}{R} = G$$

$$Cs$$

$$\frac{1}{Ls}$$

eg - Single loop



$$LC \frac{d^2V_c}{dt^2} + RC \frac{dV_c}{dt} + V_c = V$$

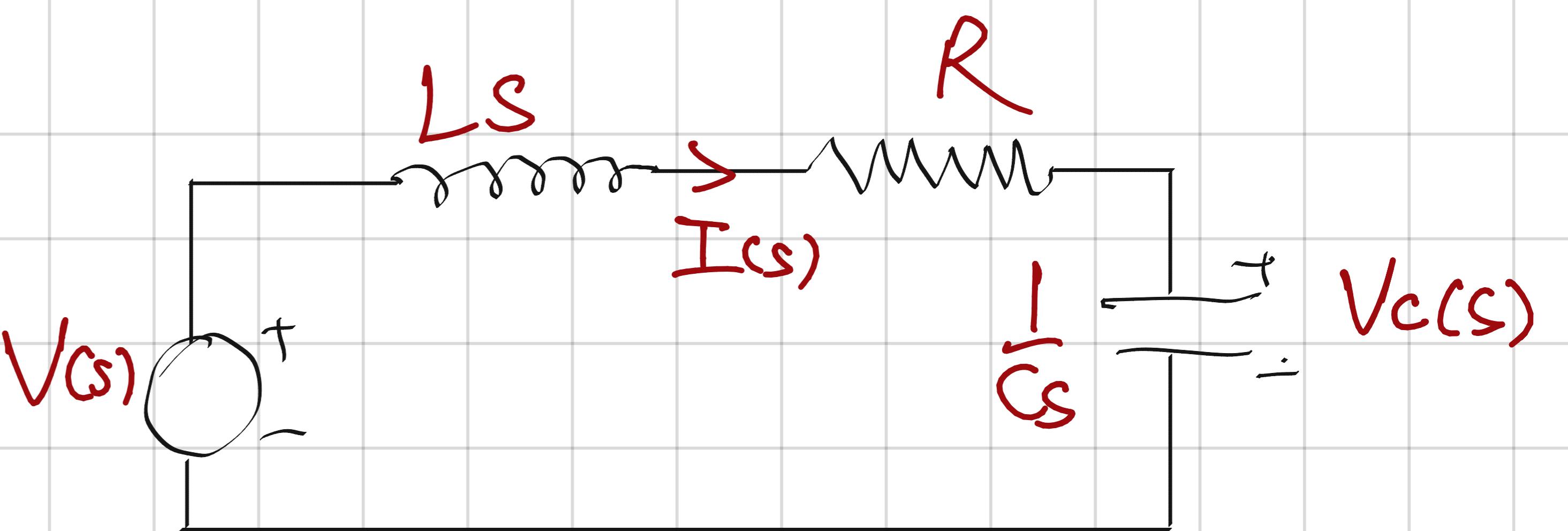
$$L \frac{di}{dt} + Ri + \frac{1}{C} \int i \cdot dz = V$$

Substitute $i = \frac{dq}{dt}$

$$L \frac{d^2q}{dt^2} + R \frac{dq}{dt} + \frac{q}{C} = V$$

$$q = CV_c$$

$$\left(Ls + R + \frac{1}{Cs}\right) I(s) = V(s)$$



$$\frac{V(s)}{I(s)} = \frac{1}{Cs}$$

$$I(s) = Cs V_c$$

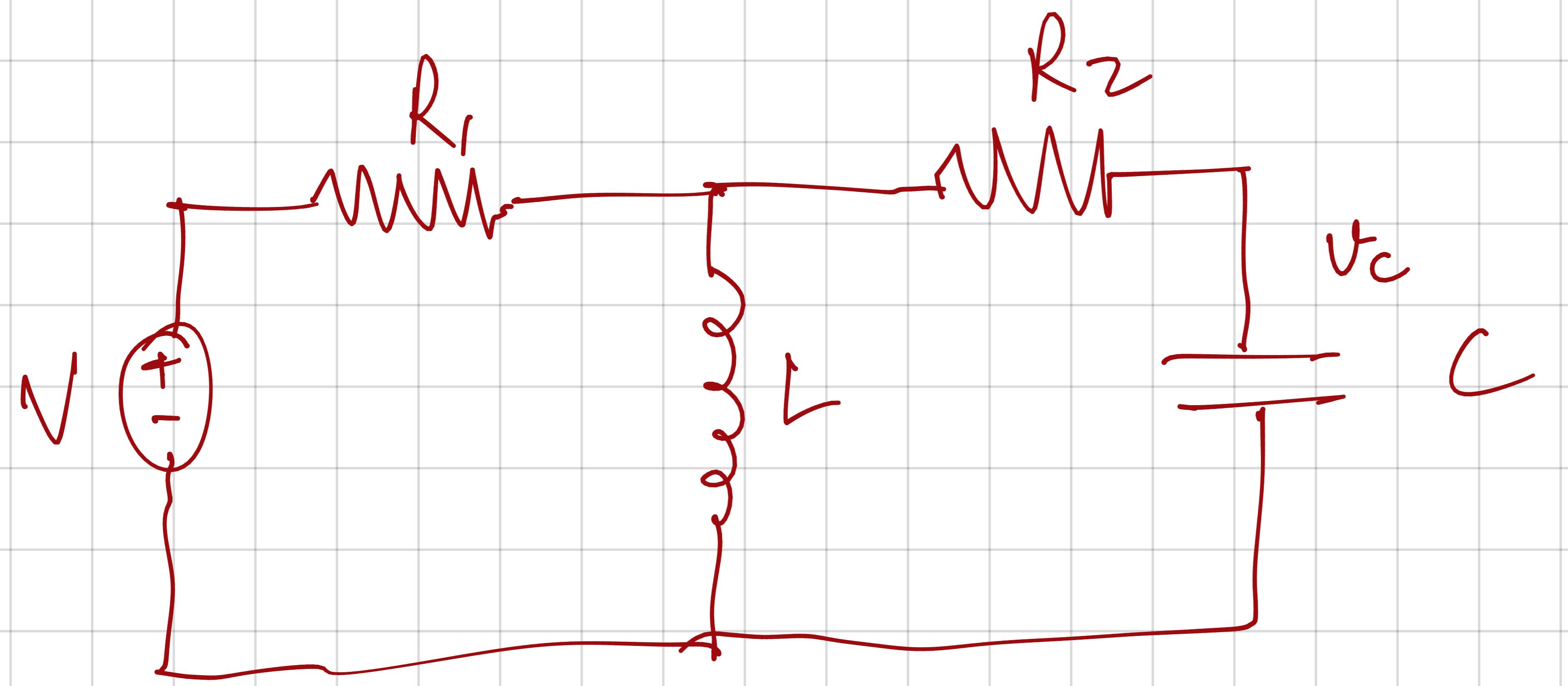
$$\left(Ls + R + \frac{1}{Cs}\right) Cs V_c = V$$

$$\boxed{\left(LCs^2 + RCs + 1\right) V_c = V} \equiv s'$$

$$LC \frac{d^2 V_c}{dt^2} + RC \frac{d V_c}{dt} + V_c = V$$

t

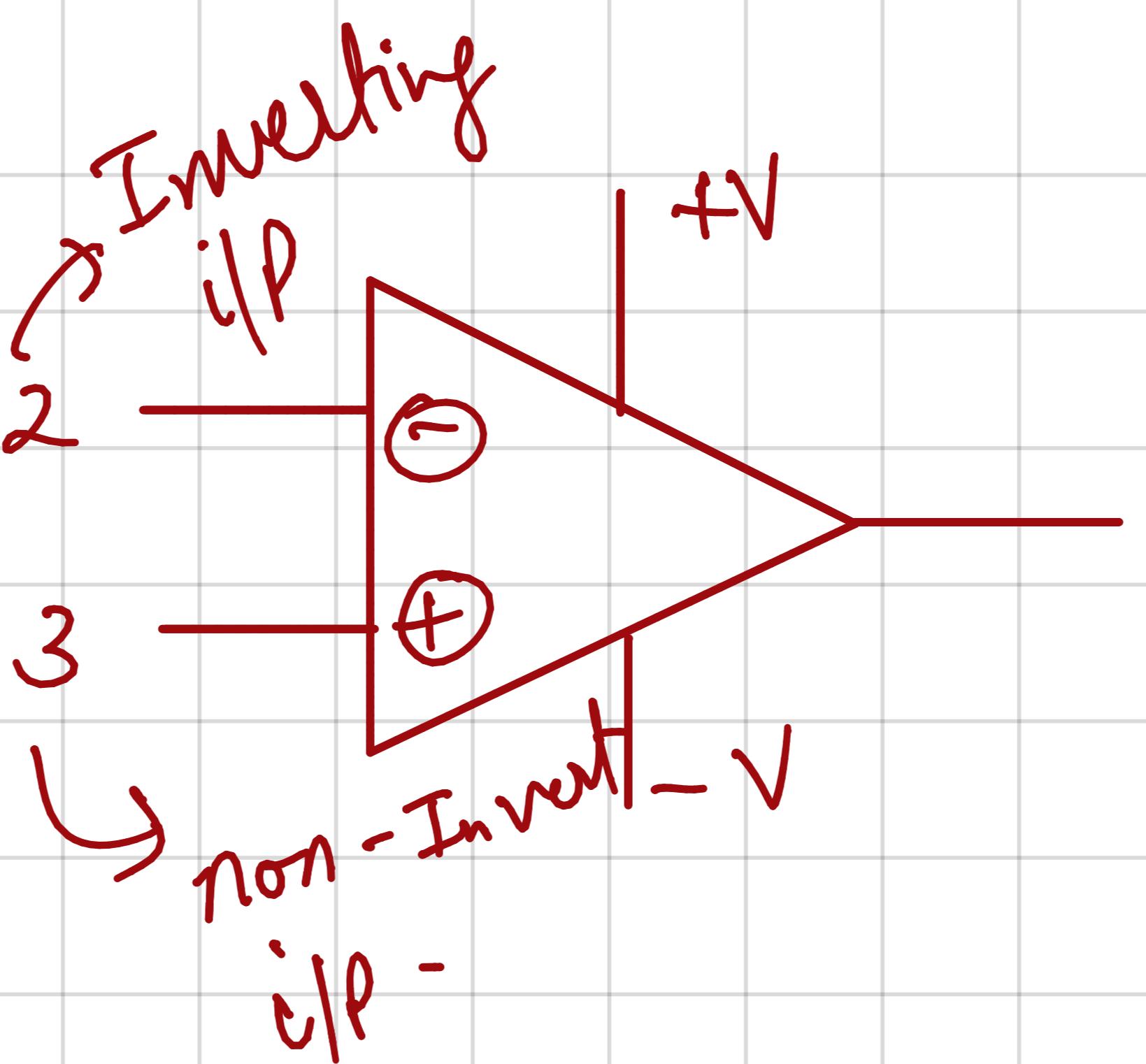
given $I \cdot Cs = 0$



Opamps

Ideal opamp assumptions:-

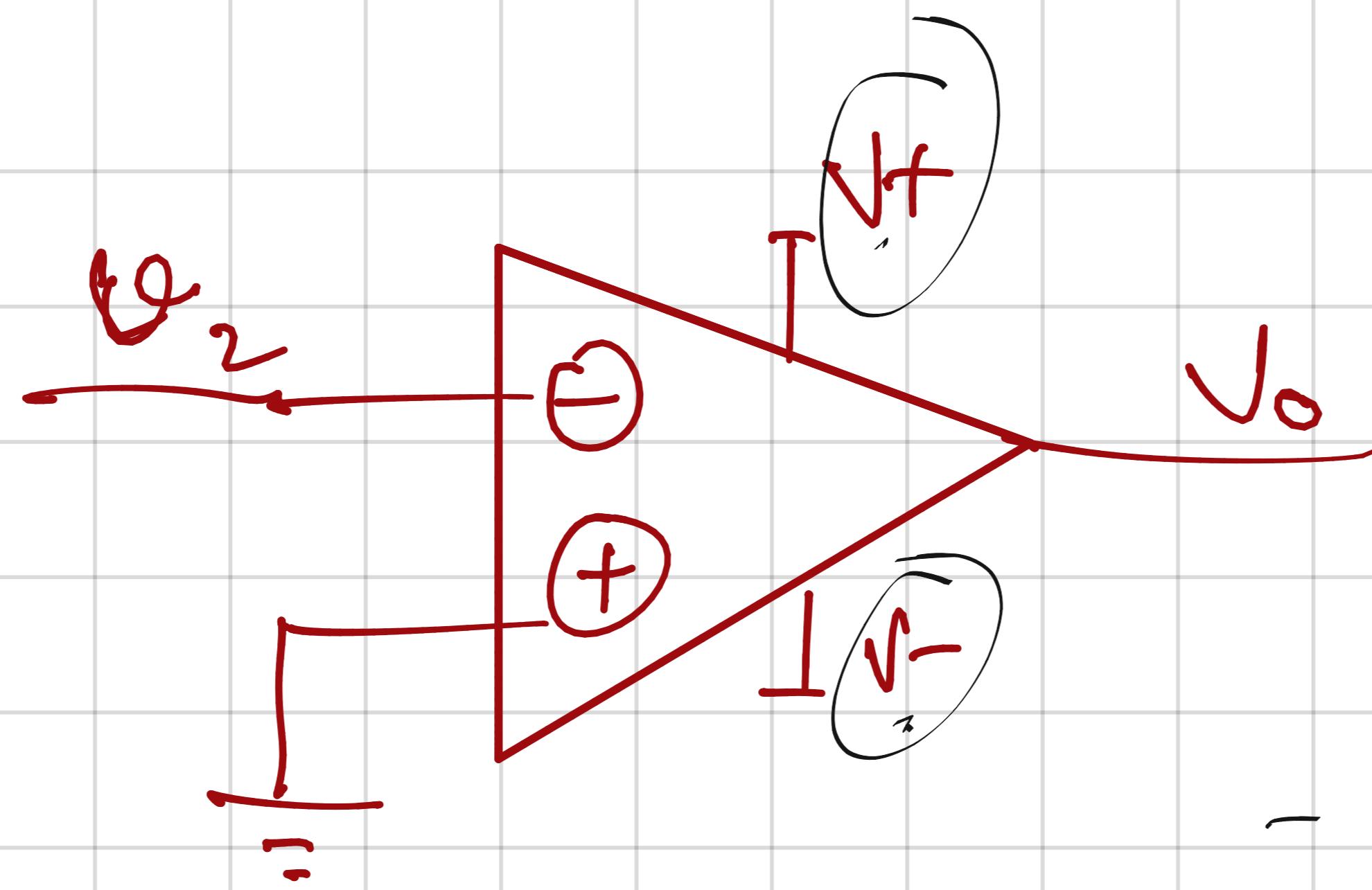
- Infinite resistance b/w
2 & 3



- Low o/p impedance = 0

- Infinite I/p impedance $Z_2 = Z_3 = \infty$ i.e. no current flows from source to opamp.

Inverting opamp

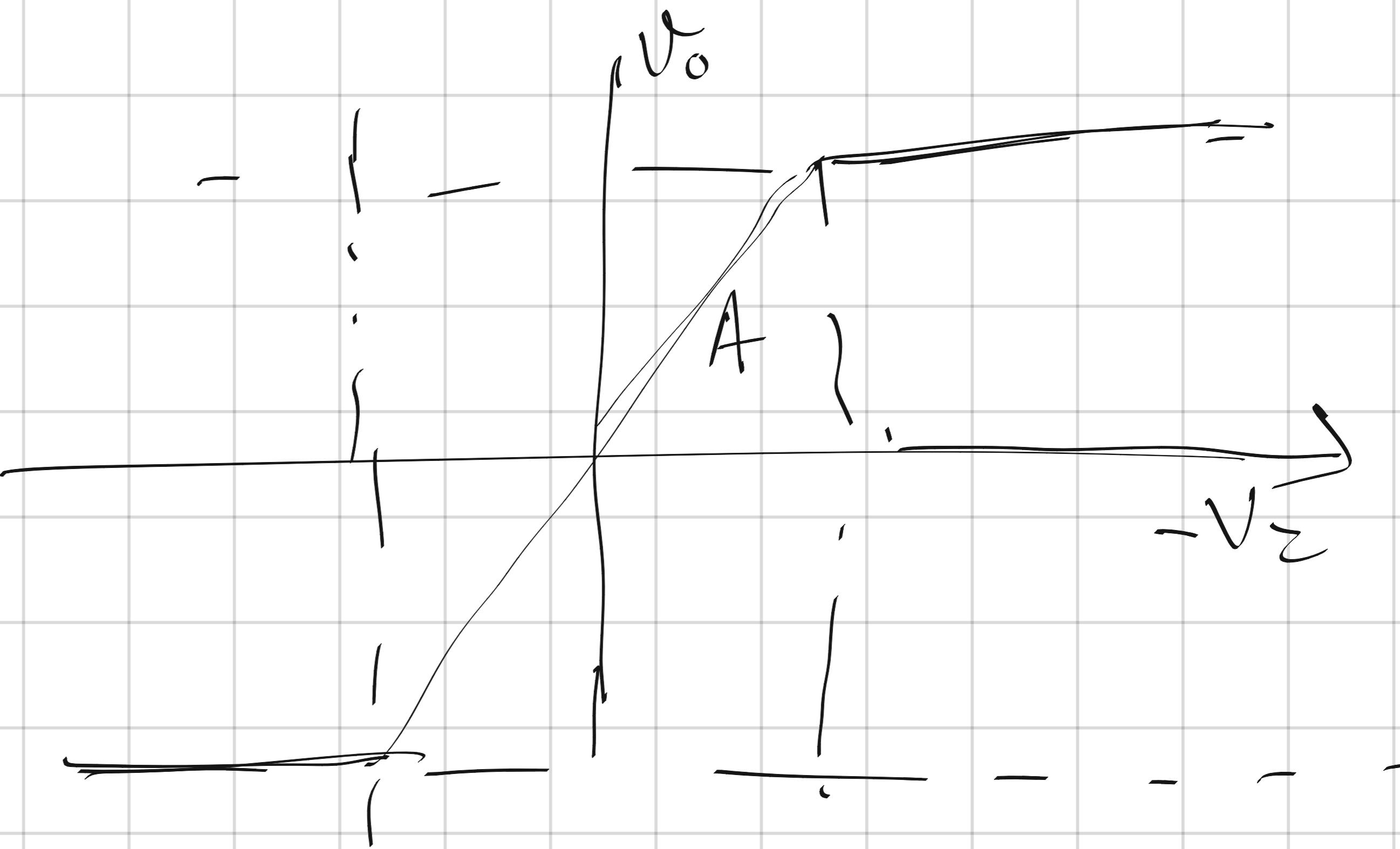


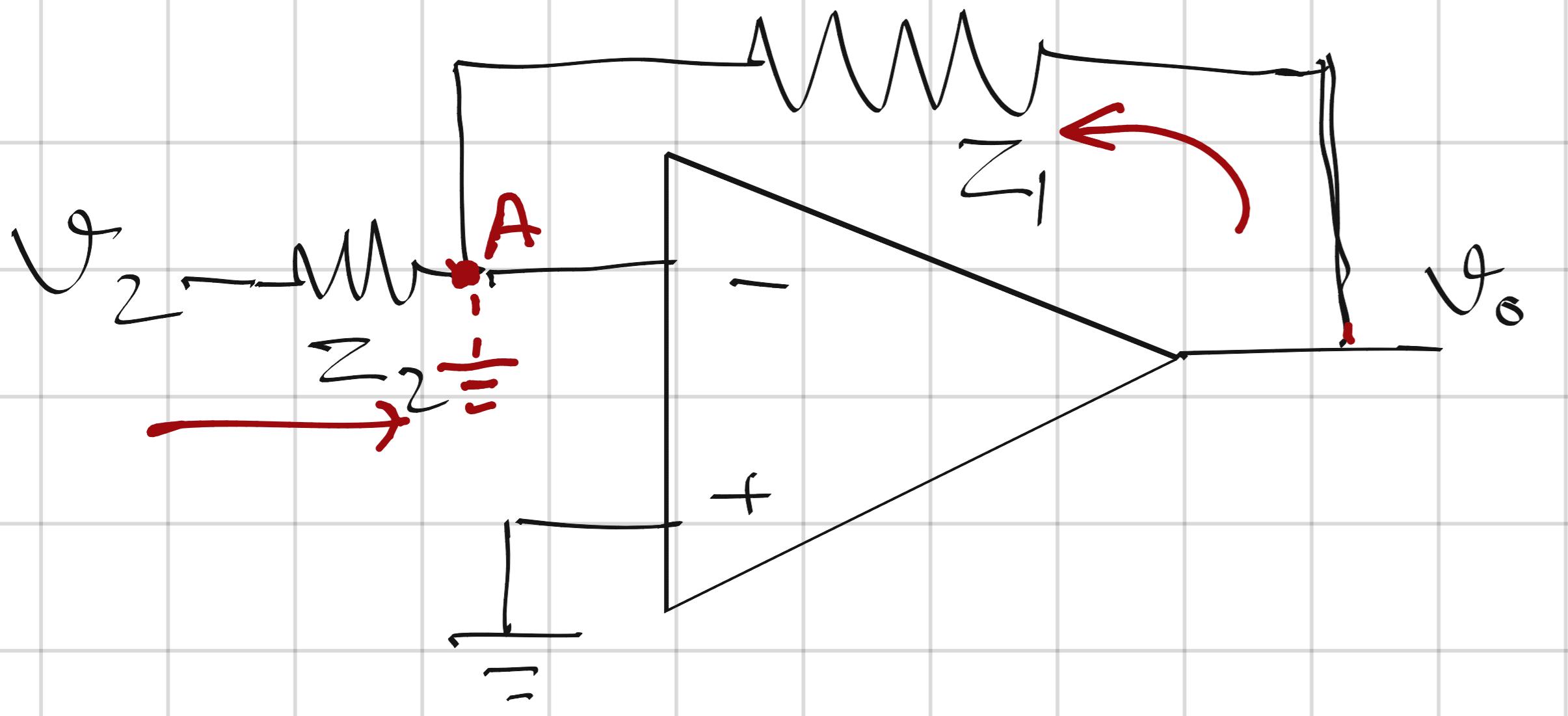
$$V_o = -A v_2$$

Ideally $A \rightarrow \infty$

$$= -v_-$$

open loop gain = A





$A \rightarrow$ virtual ground.

$$-\frac{V_0}{Z_1}$$

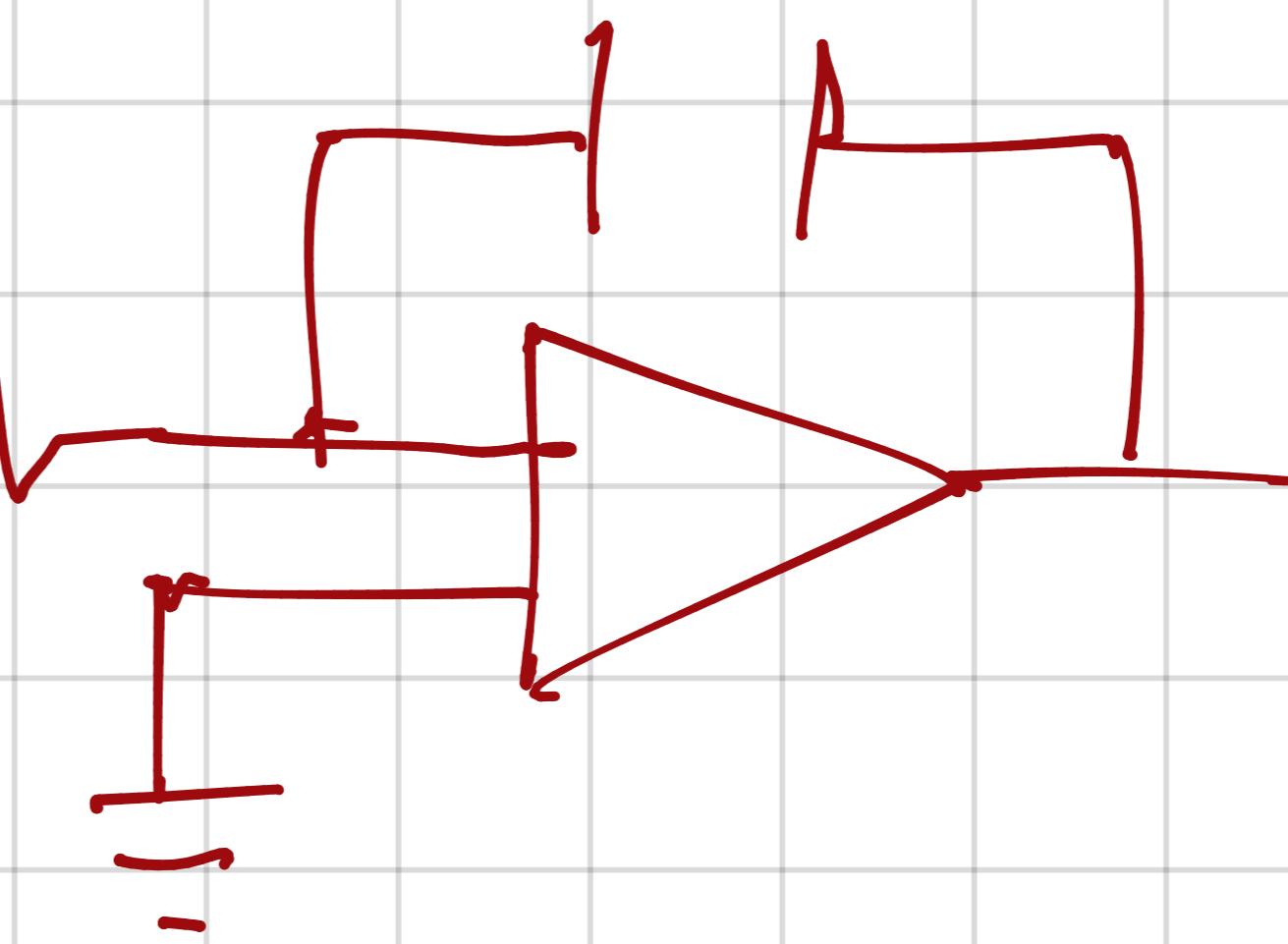
$$\frac{V_2}{Z_2}$$

$$V_0 = -V_2 \cdot \frac{(Z_1)}{(Z_2)}$$

eg - write Z_1 & Z_2 in laplace

$$Z_1 = R_1, Z_2 = R_2 \Rightarrow V_0 = -V_2 \frac{R_1}{R_2}$$

$$Z_1 = \frac{1}{CS}, Z_2 = R_2 \Rightarrow V_0 = -V_2 \frac{1}{R_2 CS}$$



Time Response of Systems

$$G(s) = \frac{(s+a)}{(s+b)(s+c)} + 1 = \frac{F_1}{(s+b)} + \frac{F_2}{(s+c)}$$

$$= K_1 e^{-bt} + K_2 e^{-ct}$$

$$ax + bx^2 + c \int_0^x dz$$

$$as + b + \frac{c}{s} = 1$$

$$G = \frac{(s+a)}{(s+b)(s+c)}$$

Zeros :

$$\begin{aligned} G &\rightarrow 0 \\ s &\rightarrow -a \end{aligned}$$

$a \rightarrow$ is a zero of
the system

Poles :

$$s \rightarrow -b, -c \quad G \rightarrow \infty$$

↳ Poles of the system

Any poles that are common with zeros
& vice versa are also poles & zeros

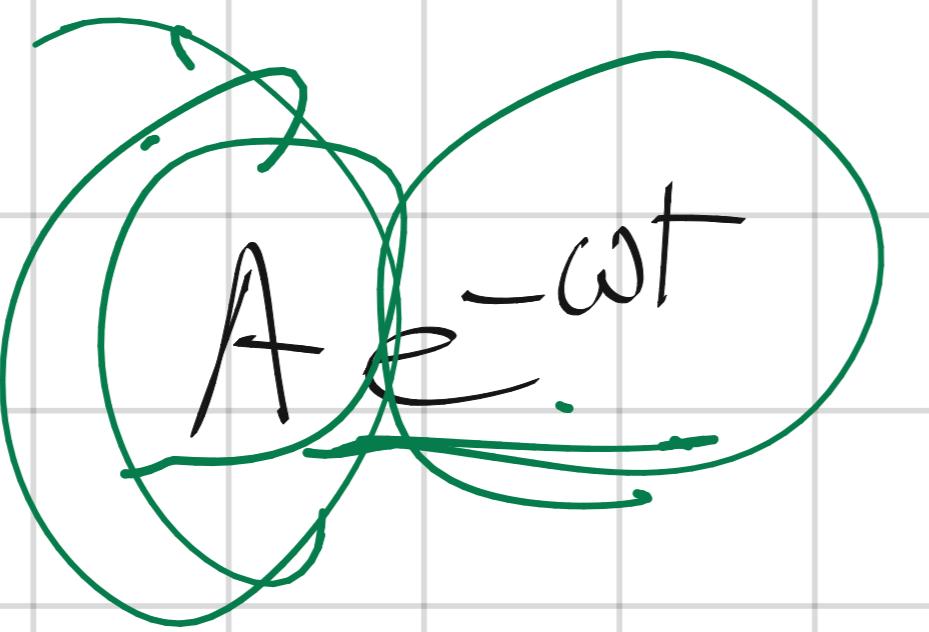
Step response of system

$$G(s) \cdot \frac{1}{s} \rightarrow \frac{1}{s} \cdot \frac{(s+a)}{(s+b)(s+c)} \rightarrow \frac{K_1}{s} + \frac{K_2}{s+b} + \frac{K_3}{s+c} \quad - (1)$$

$$\mathcal{L}^{-1}[1] = (K_1 q(t) + [K_2 e^{-bt} + K_3 e^{-ct}] u(t)) \quad t \rightarrow \infty$$

$q(t) \rightarrow K_1$

dynamics



Dynamics of the system i.e. transient response primarily governs the system response.

$$\left[\frac{s+3}{s+7} \right]$$

$$\frac{\frac{s+11}{(s+10)(s+15)}}{e^{-t}} = \frac{1}{s+1} e^{-t j\omega}$$

Poles $\rightarrow X$

Zeros $\rightarrow 0$



$$G(s) = \frac{s+3}{(s+7)}$$

1st order system

$$O(s) = I(s) \cdot G(s)$$

$1, \frac{1}{s}, \frac{1}{s^2}$

→ Step response

$$\frac{1}{s} \cdot \frac{s+3}{(s+7)}$$

$$= -\frac{3}{7} \frac{1}{s} + \frac{4}{7} \frac{1}{s+7} \quad \text{--- (2)}$$

$$\mathcal{L}^{-1}[(2)] \rightarrow \frac{3}{7} u(t) + \frac{4}{7} e^{-7t} u(t)$$

Forced response Natural response

Static gain.

$$\frac{3}{7}$$

→ Impulse response?

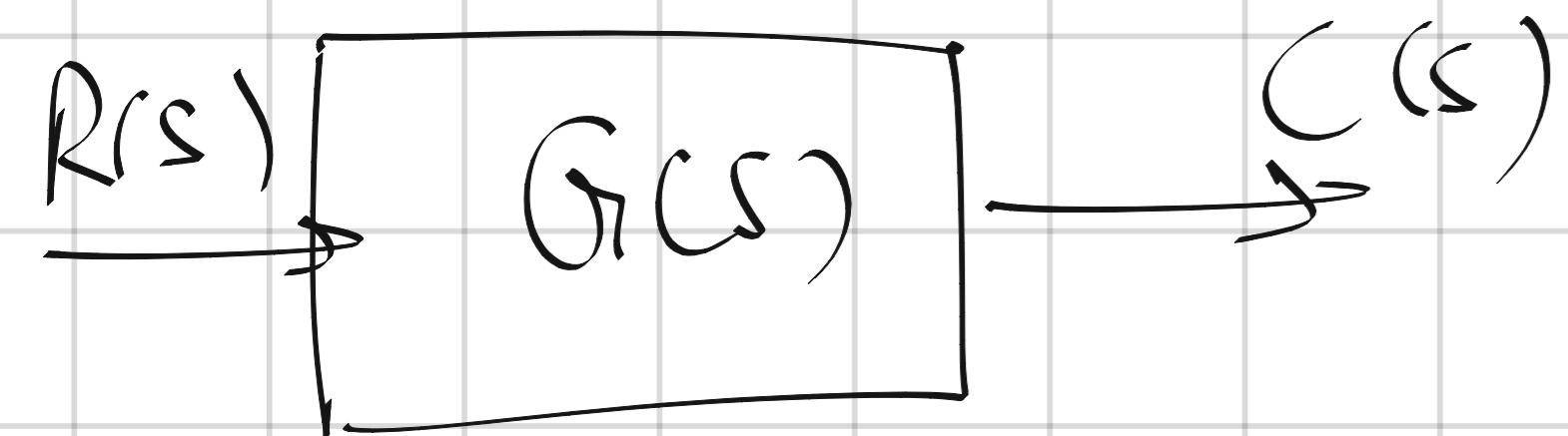
$$\mathcal{L}^{-1}\left[\frac{s+3}{(s+7)}\right] = \mathcal{L}^{-1}\left[1 - \frac{4}{s+7}\right]$$

$$= \delta(t) u(f) - 4e^{-7t}$$

First Order Systems: Response and Performance Characteristics

$$G(s) = \frac{a}{(s+a)}$$

sys response to step i/p -

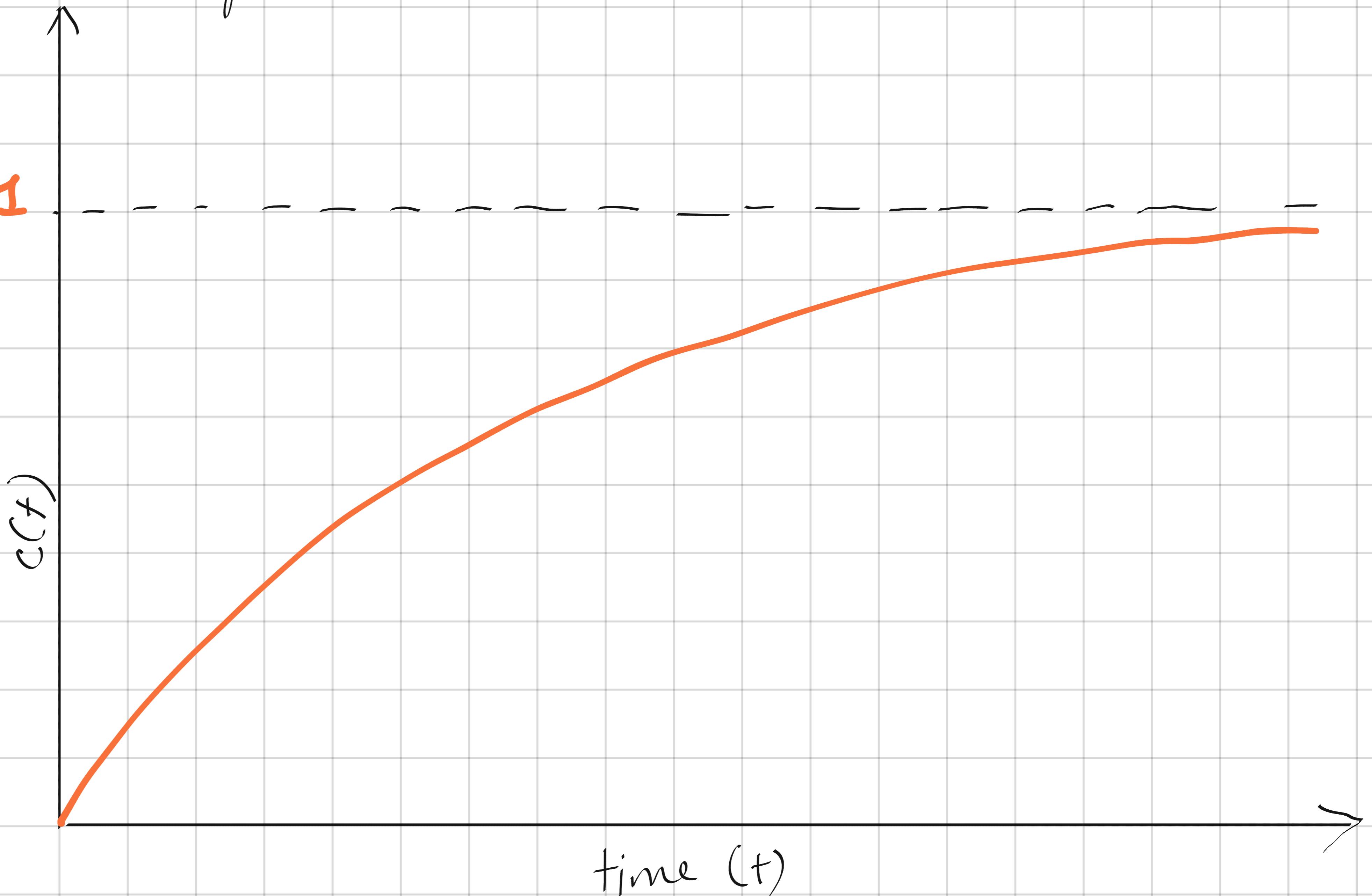


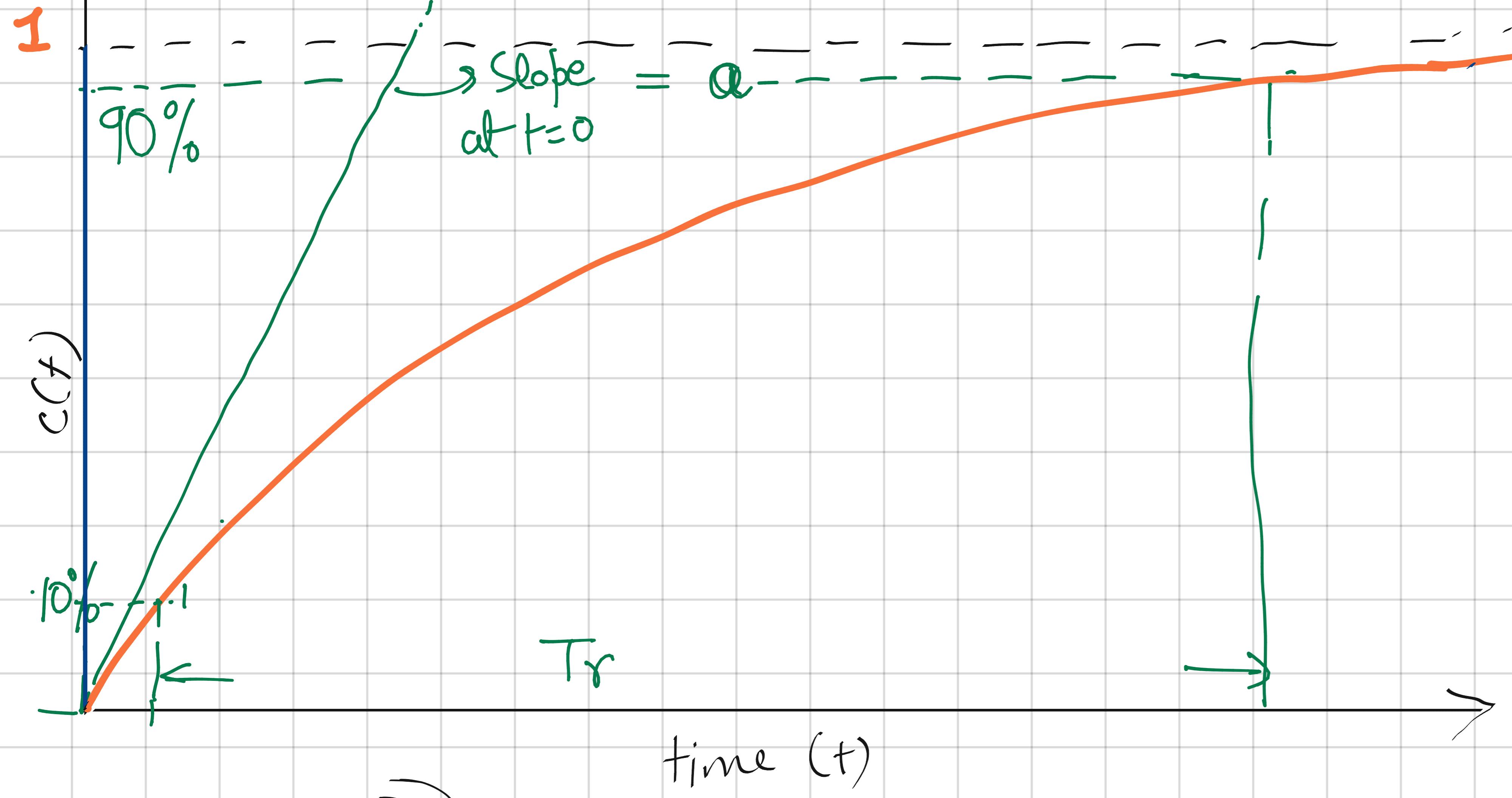
$$C(s) = G(s) R(s) = \frac{a}{s(s+a)}$$

$$= \frac{1}{s} - \frac{1}{s+a}$$

$$\mathcal{L}^{-1}[C(s)] = [1 - e^{-at}] u(t)$$

$a \rightarrow$ only parameter required to describe the sys.

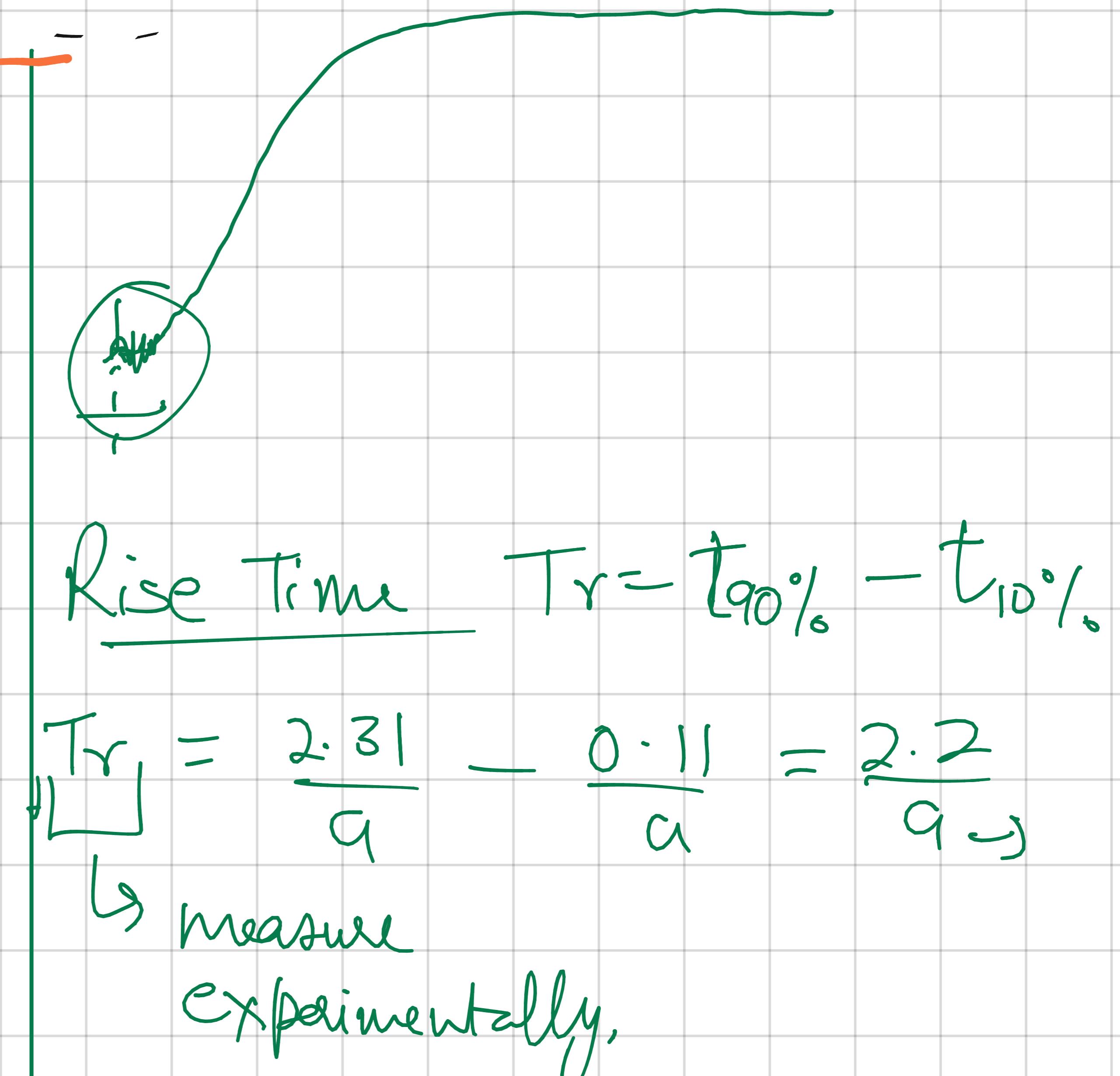




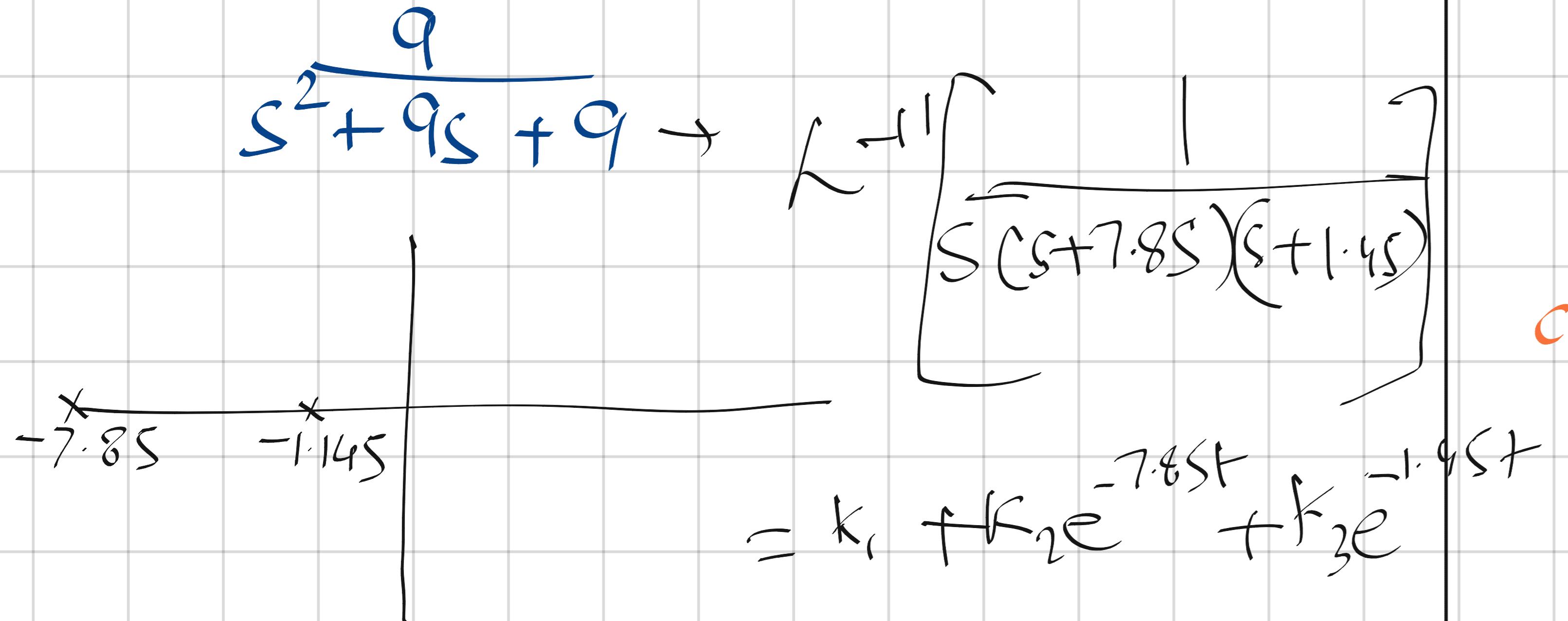
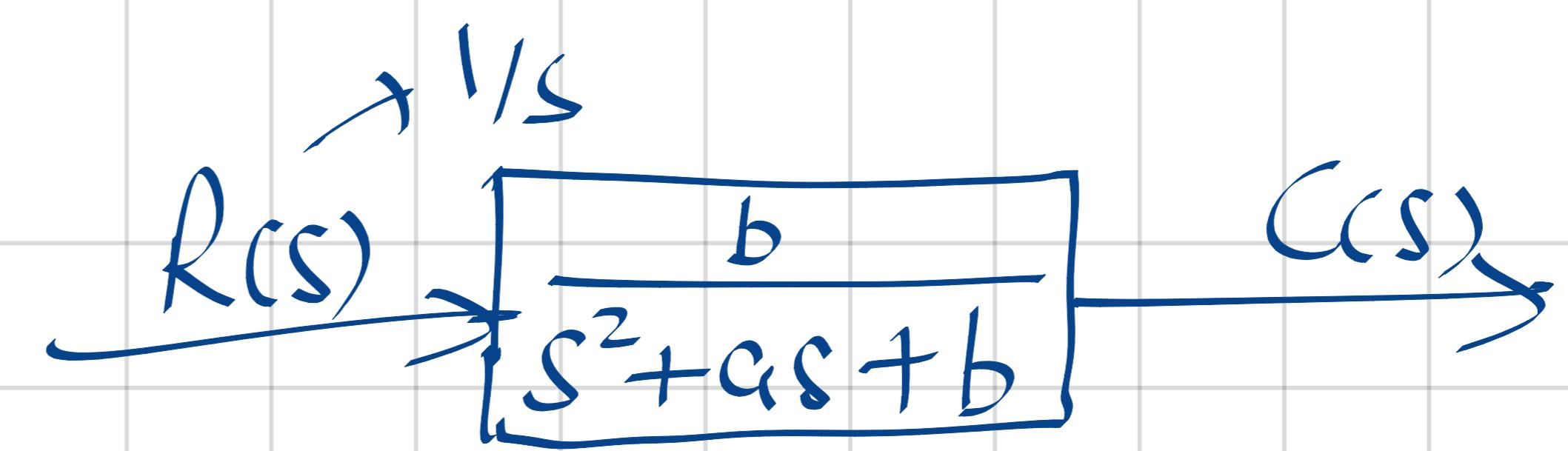
$$c(t) = 1 - e^{-t/\tau}$$

$$= 0.63$$

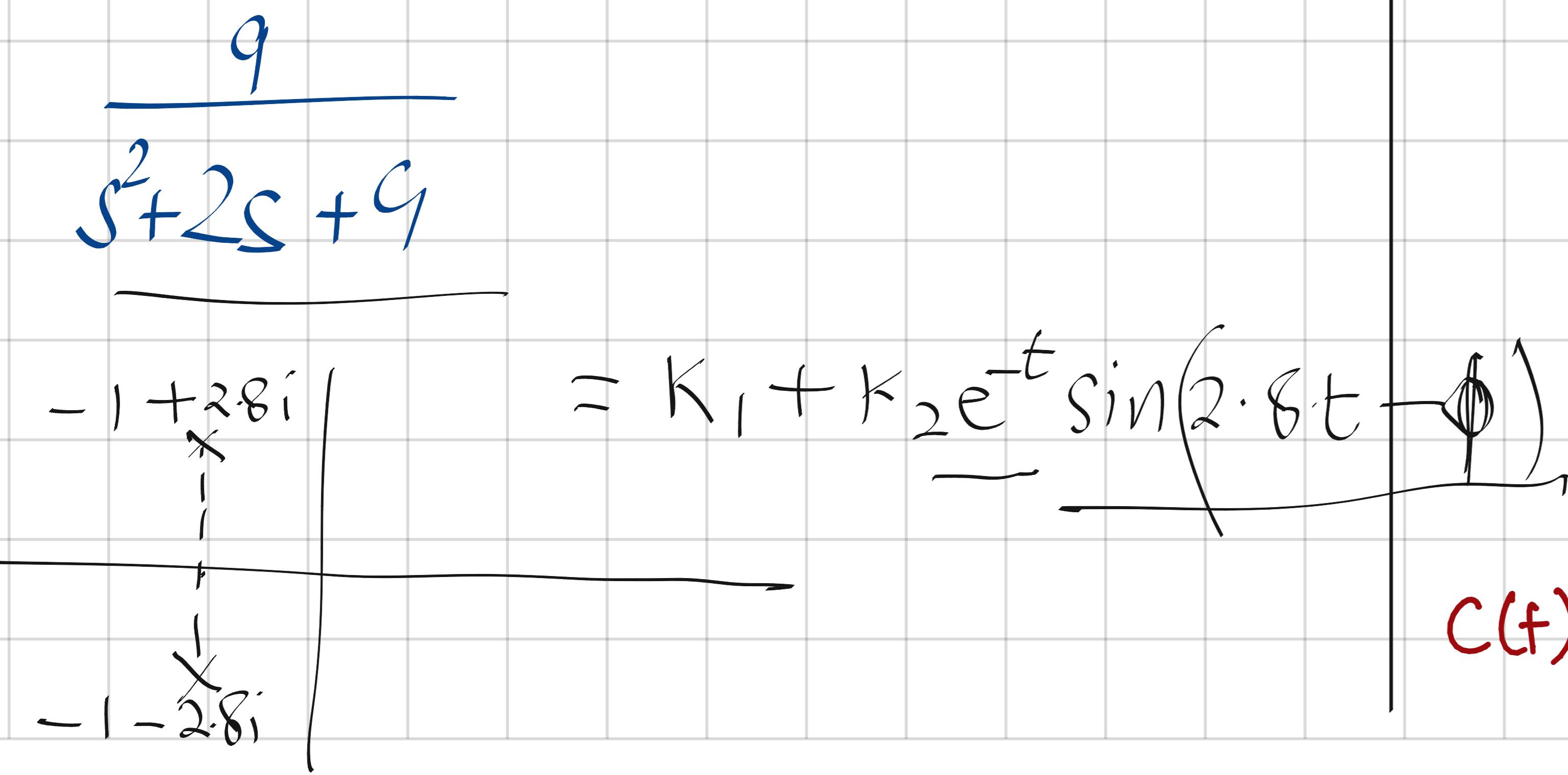
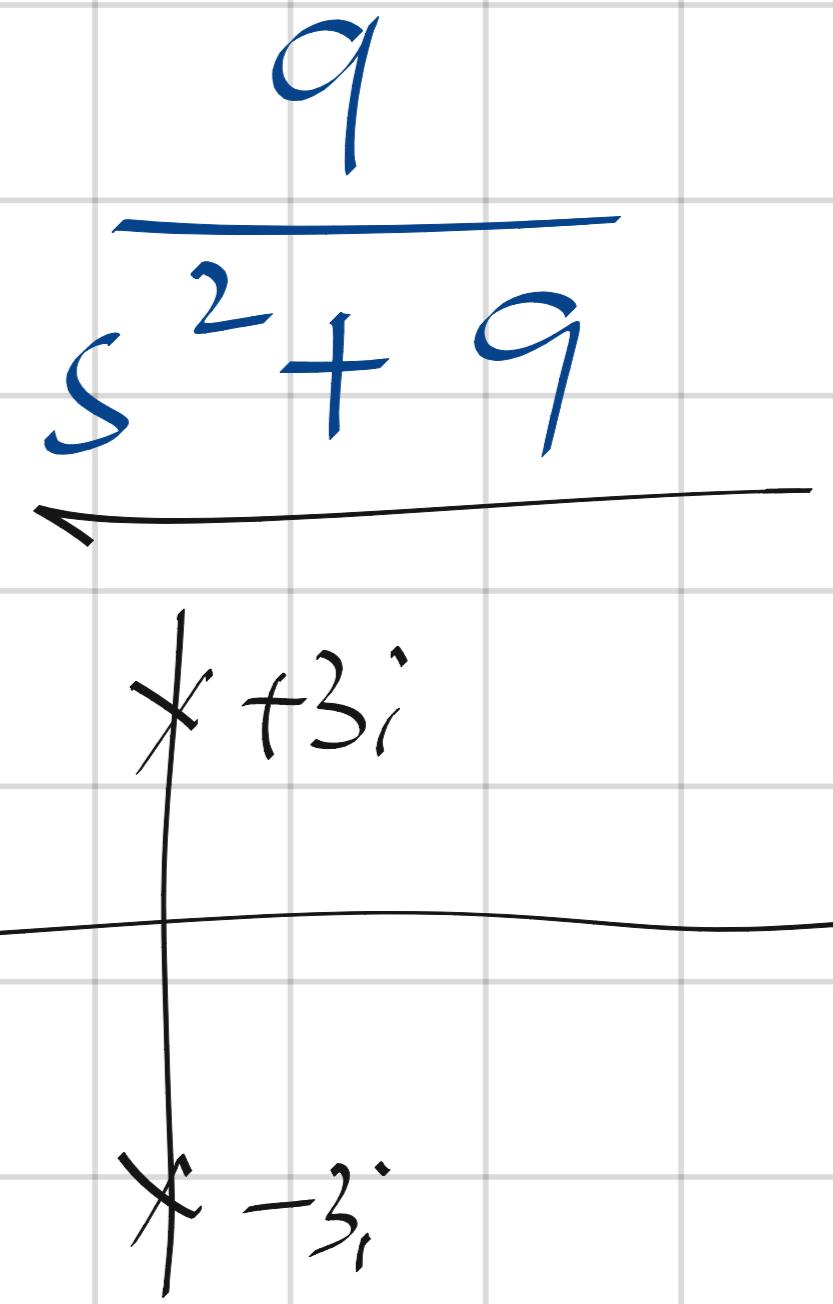
→ Useful for system I.D.



Second Order Systems

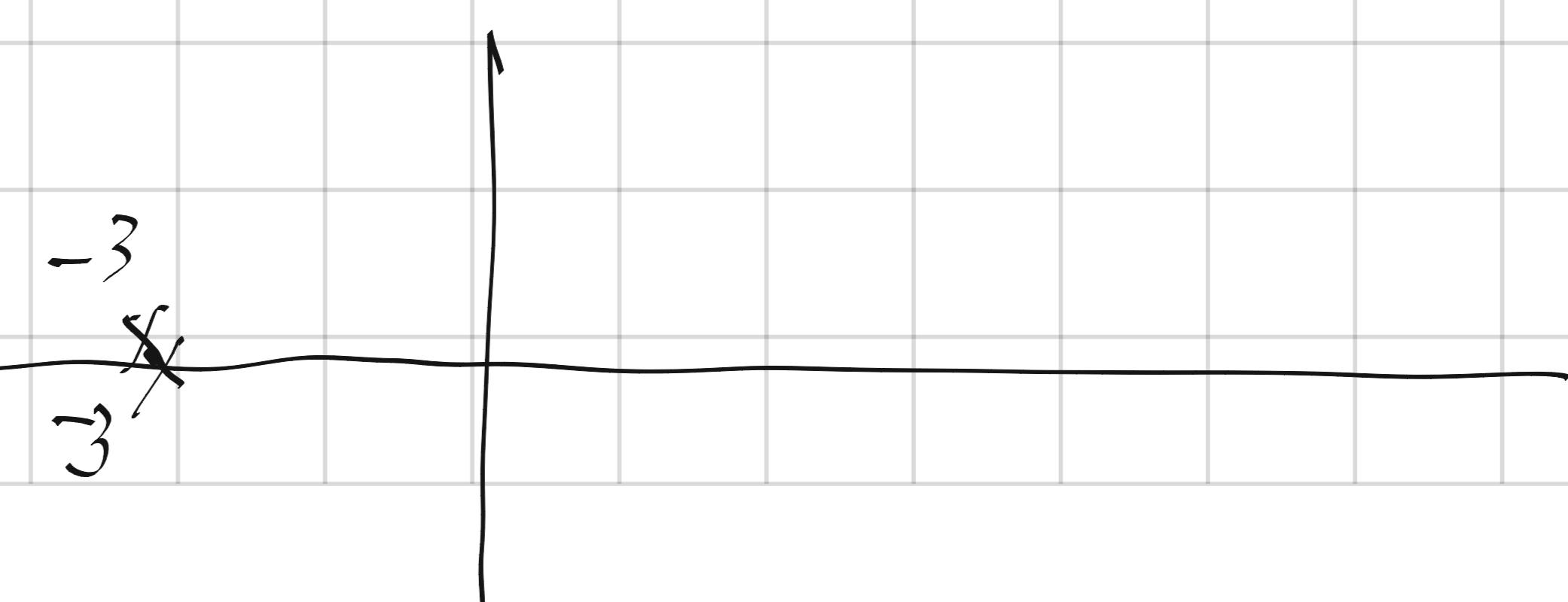


$$C(t) = 1 - \cos 3t$$

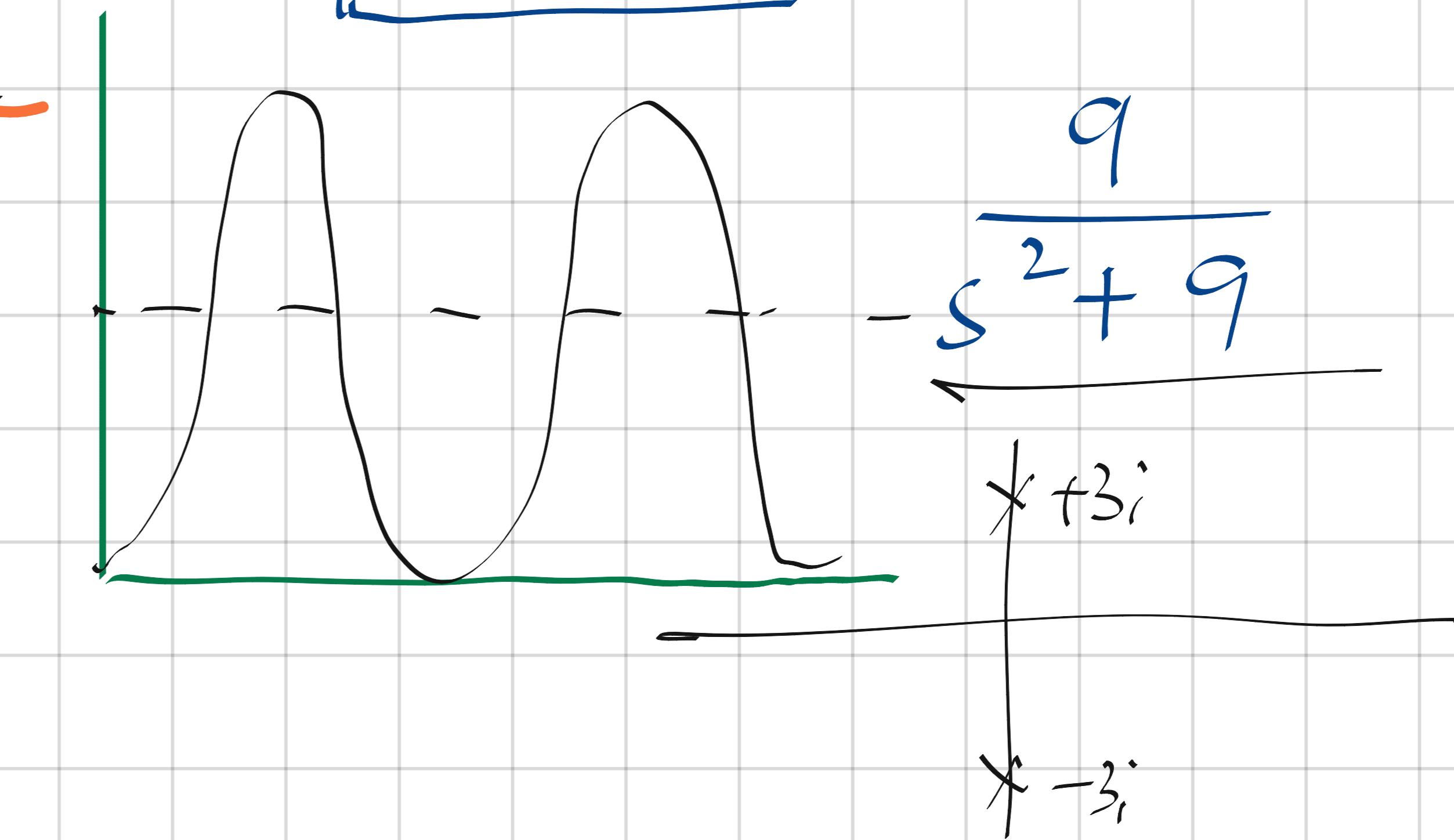
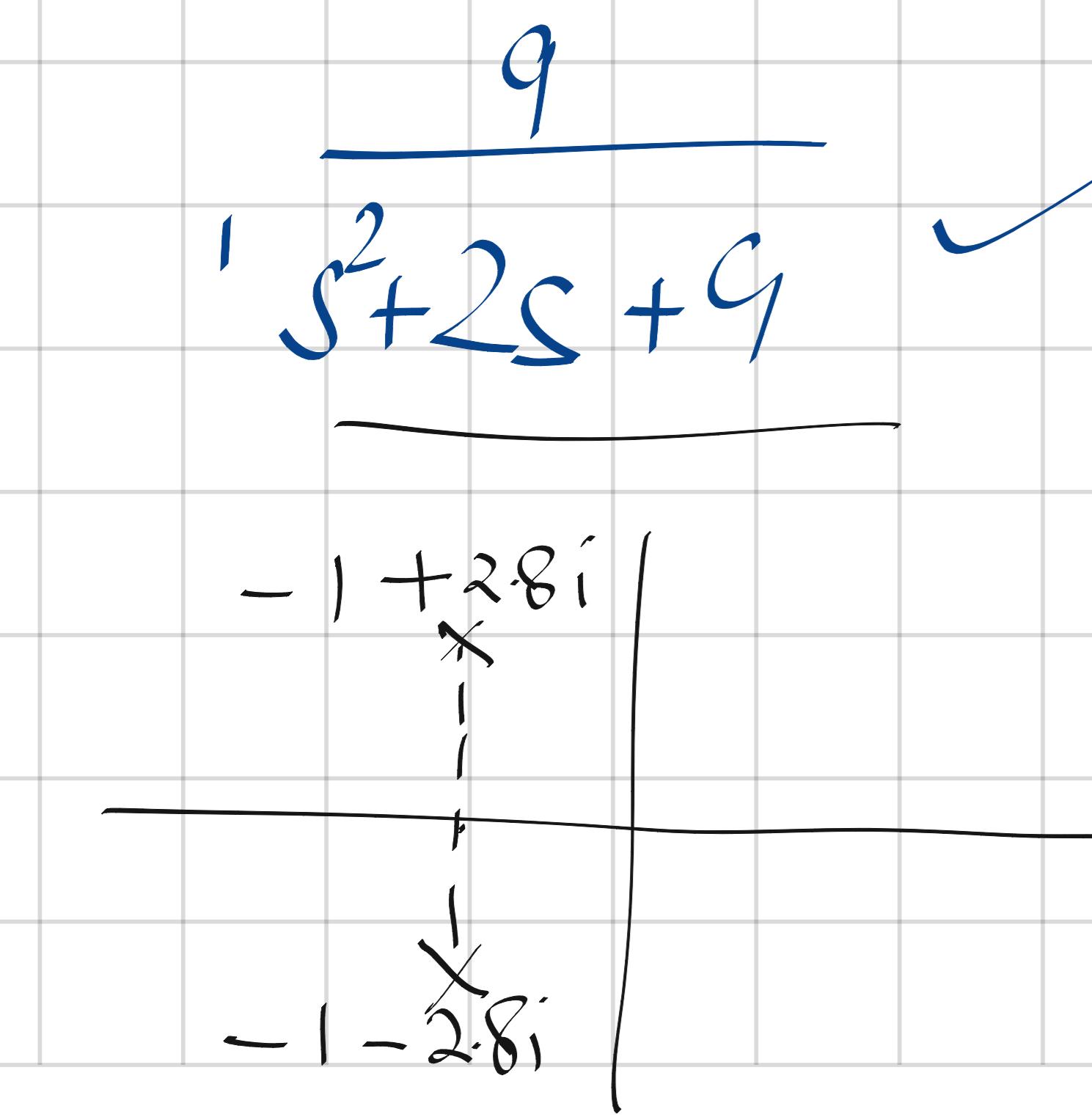
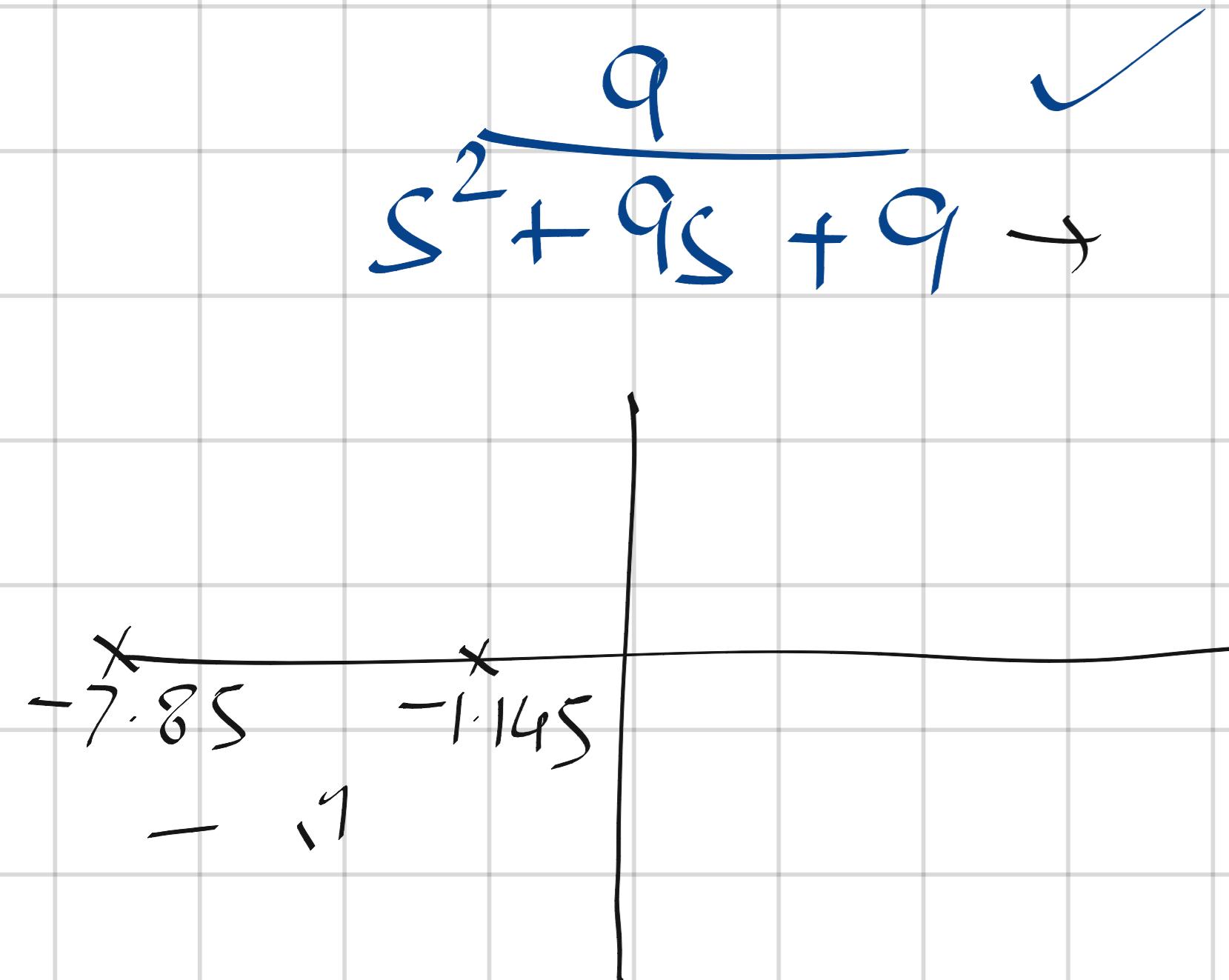
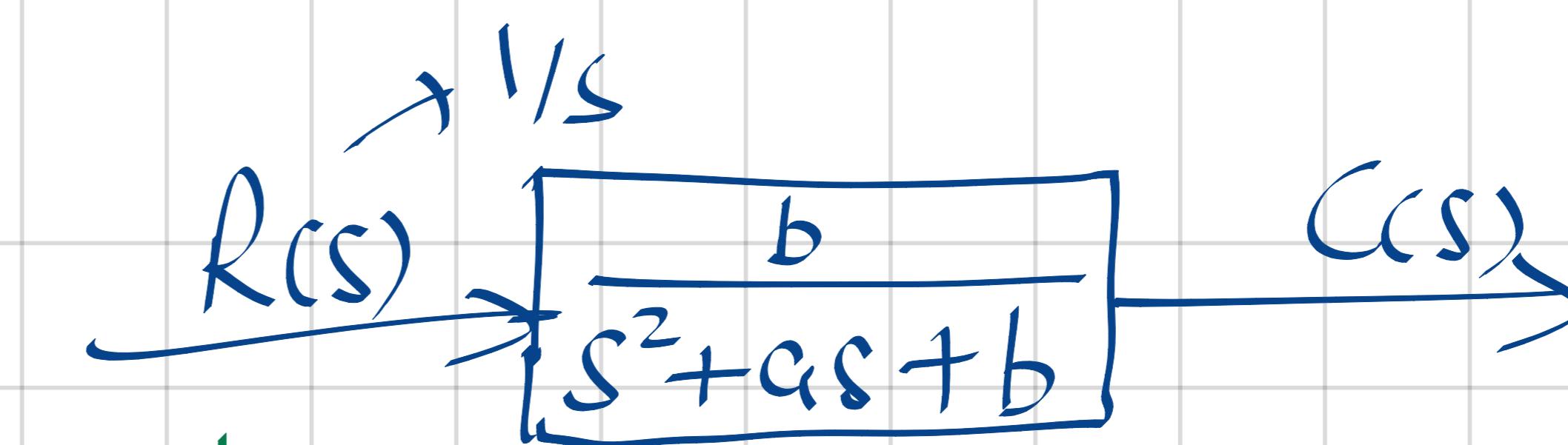


$$\frac{1}{s(s+3)^2} = \frac{K_1}{s} + \frac{K_2}{s+3} + \frac{K_3}{(s+3)^2}$$

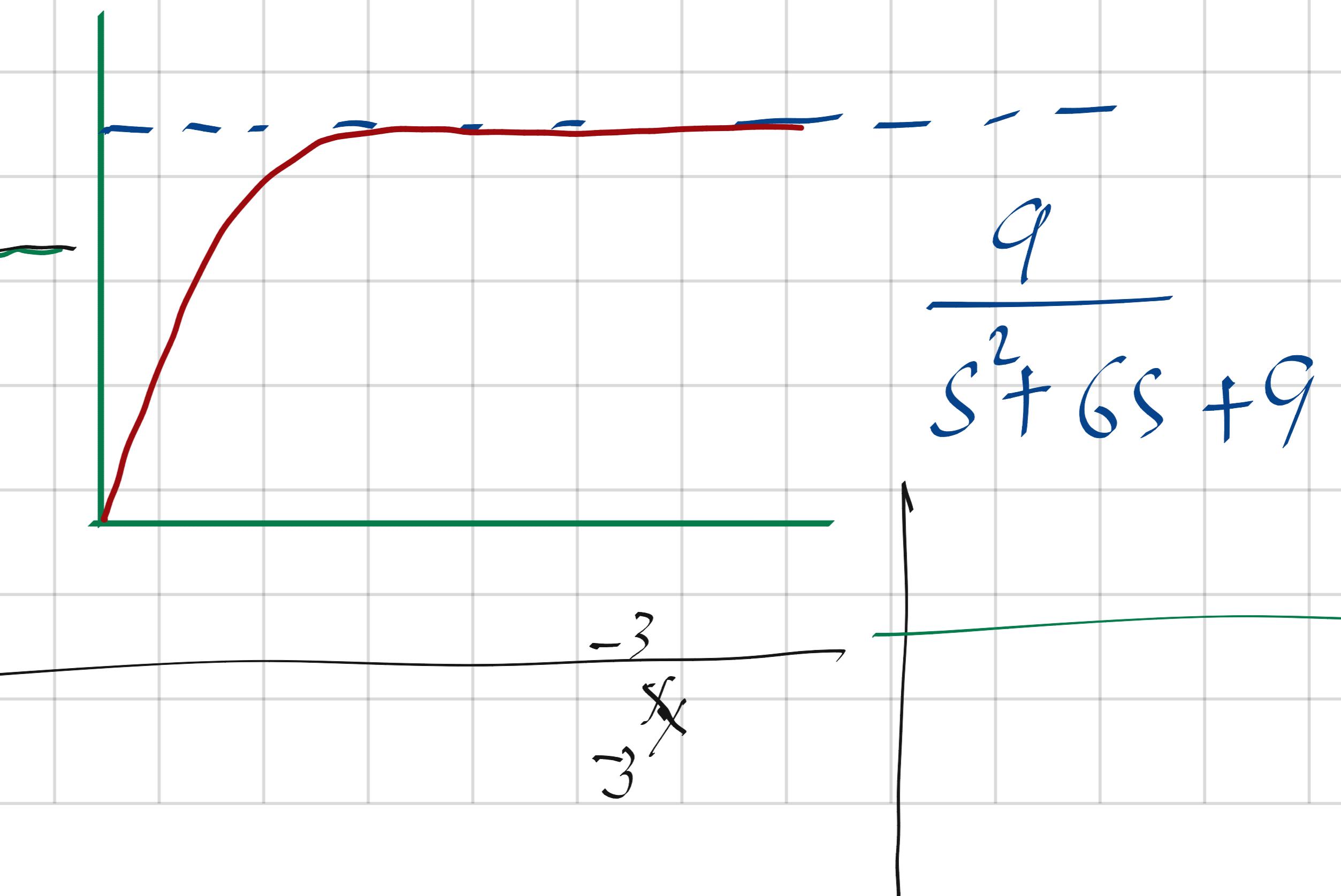
$$C(t) = -3t e^{-3t} - e^{-3t}$$



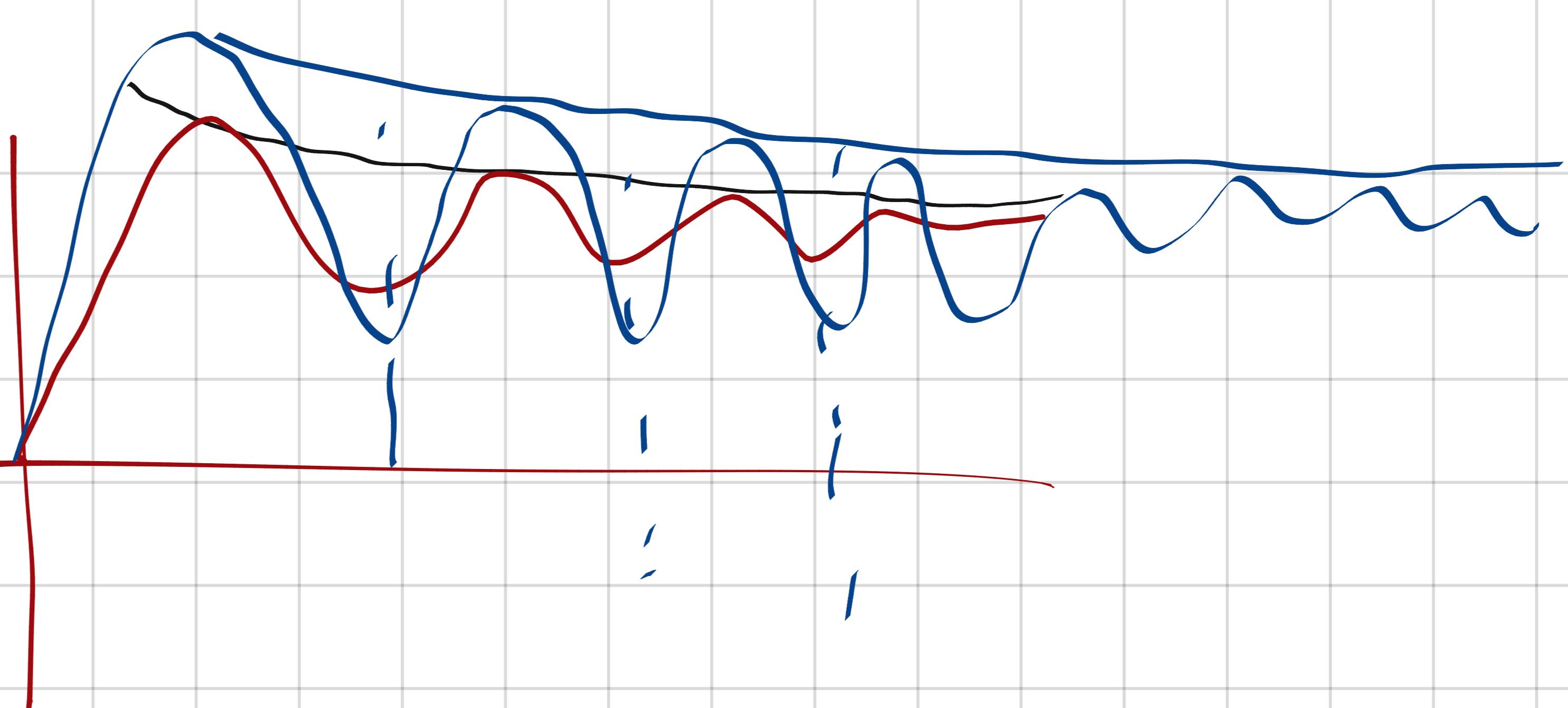
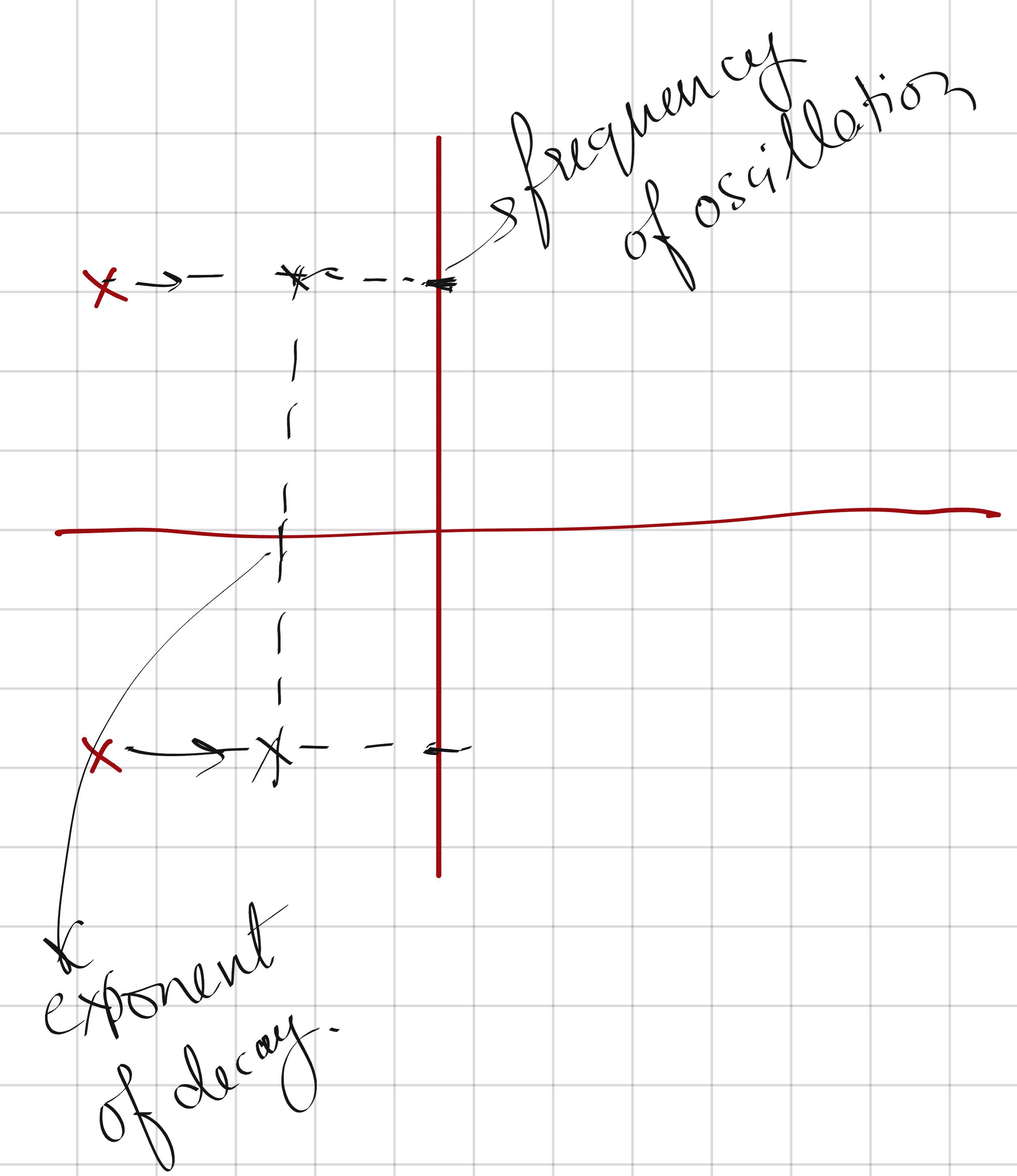
Second order Systems



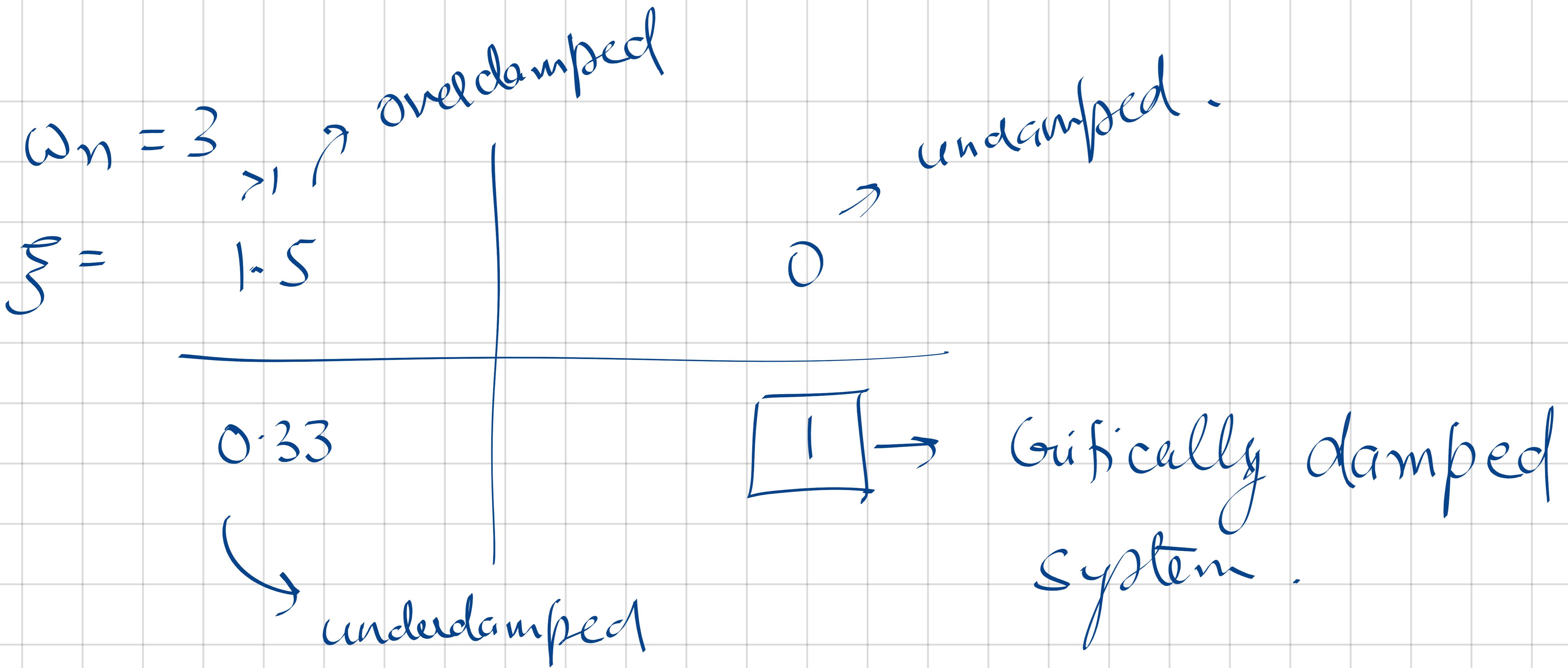
$$\frac{q}{s^2 + 9}$$



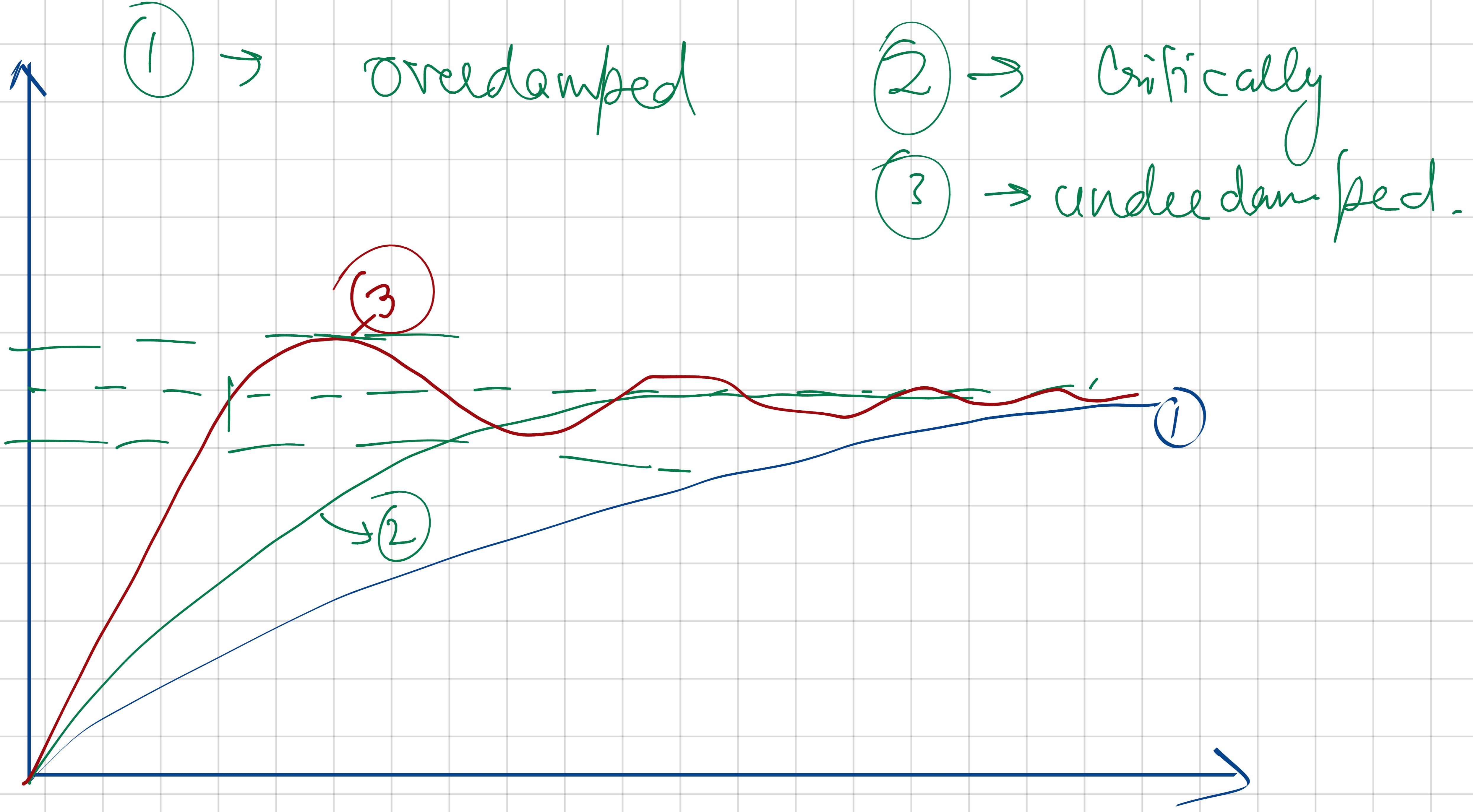
$$\frac{q}{s^2 + 6s + 9}$$



$\omega_n \rightarrow$ natural frequency of system
 $\zeta \rightarrow$ damping ratio
 $\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \rightarrow$ Generalized 2nd order representation



| | | |
|-----------|-----------------|--------------------------|
| $\xi > 1$ | No oscillation, | Asymptotic convergence. |
| $\xi = 1$ | No oscillation. | full convergence. |
| $\xi < 1$ | Oscillations | oscillating convergence. |



Underdamped response using poles:

By inspection, write the form of response of the following system to step input.

$$\frac{9}{(s^2 + 2s + 9)}$$

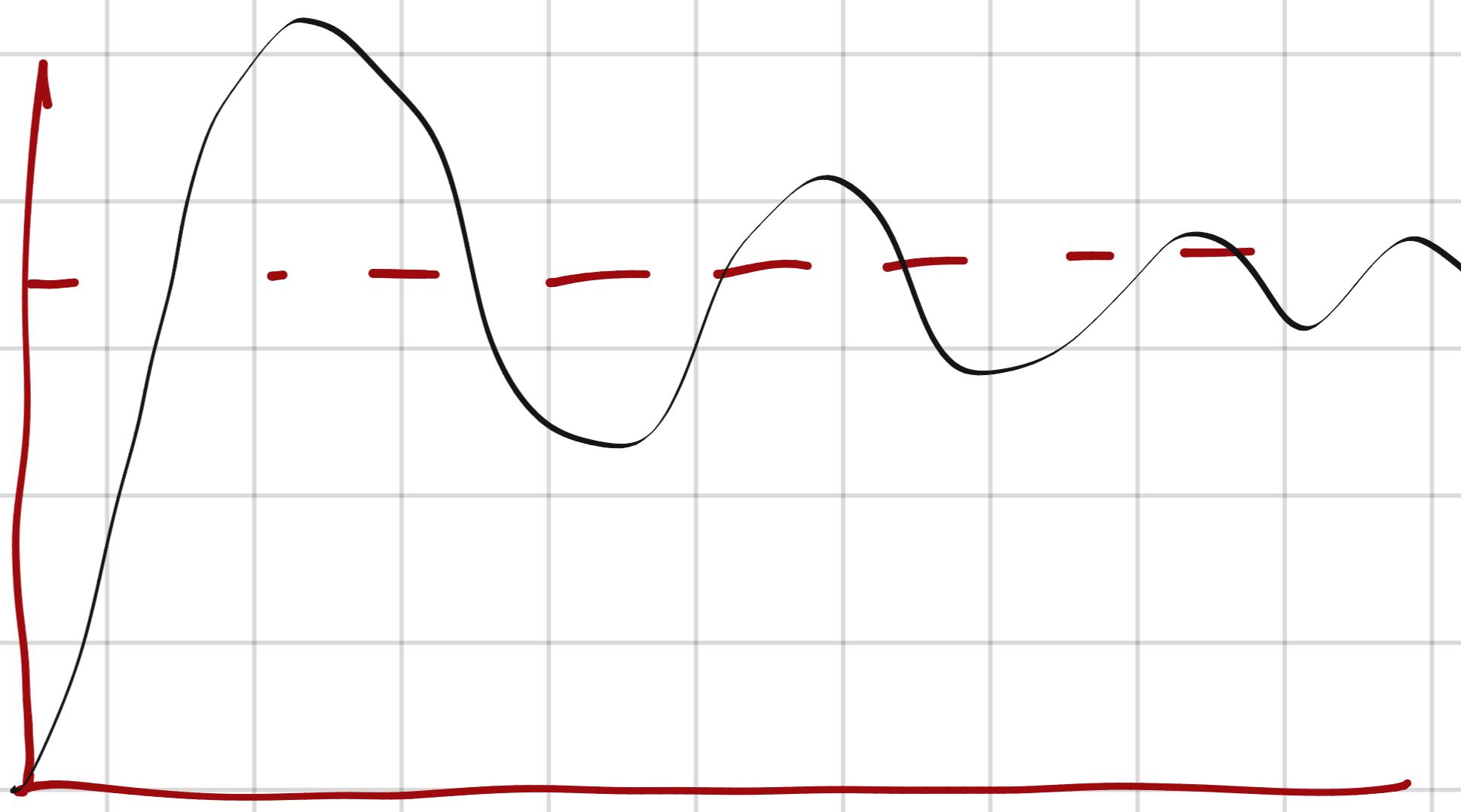
$$s = -1 \pm 2.828i \text{ [poles]}$$

$$c(t) = 1 + k_1 e^{-t} [k_2 \sin 2.828t + k_3 \cos 2.828t]$$

$$= 1 + k_4 e^{-t} [\cos(2.828t - \phi)]$$

$$\phi = \tan^{-1} k_2 / k_3$$

$$k_4 = k_1 \times \sqrt{k_2^2 + k_3^2}$$



ω_n = undamped frequency of oscillations

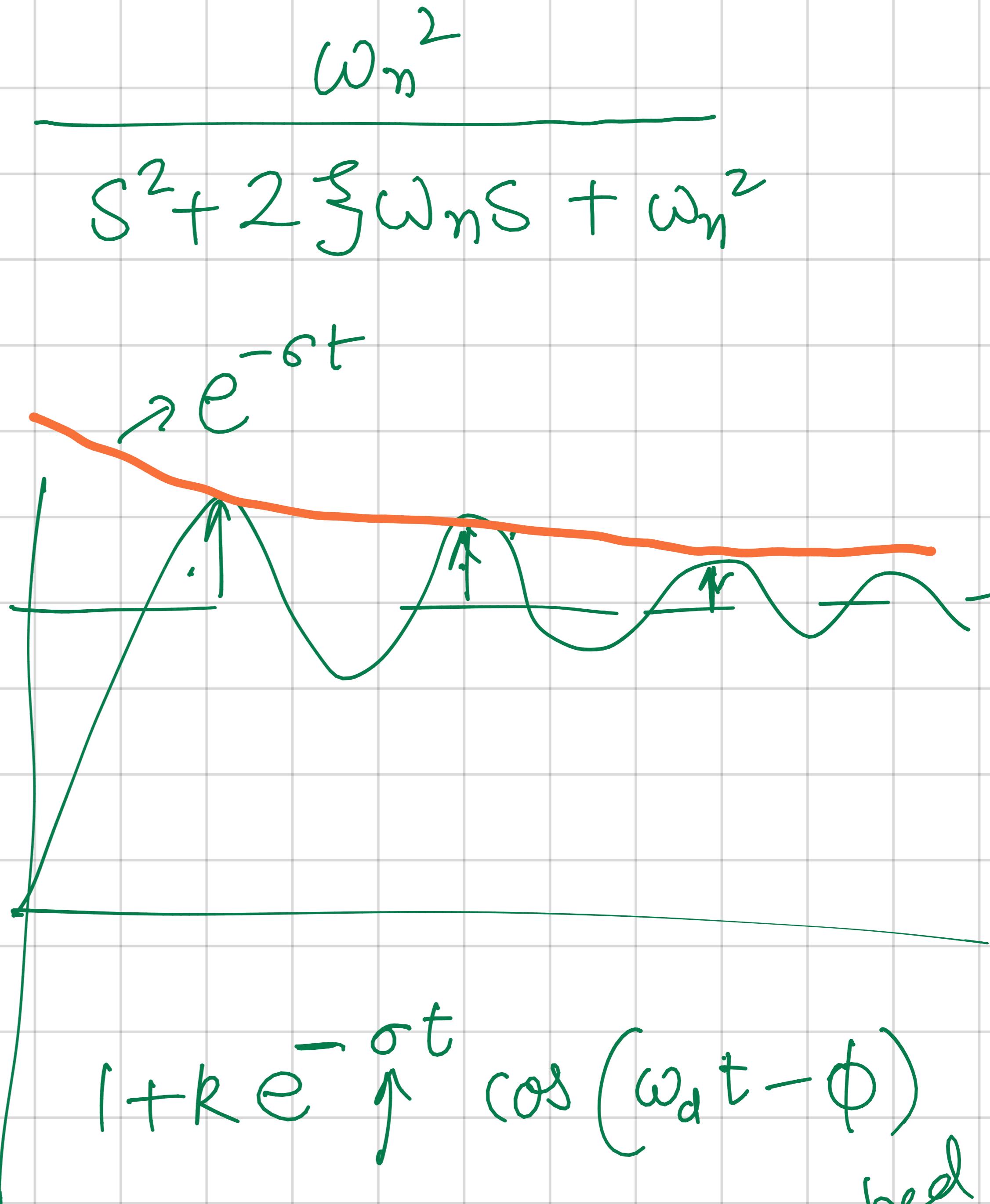
$\xi = \frac{\text{exponential decay frequency}}{\text{natural frequency}}$

$$= \frac{1}{2\pi} \frac{\text{time period of undamp oscillation}}{\text{Exponential time constant.}}$$

1st order sys. \rightarrow

$$\frac{q}{s+a} = \frac{1}{\frac{s}{\tau} + 1}$$

$$= 1 - e^{-at}$$



$$1 + Re^{j\omega_d t} \cos(\omega_d t - \phi)$$

$s = -\sigma \pm i\omega_d \rightarrow$ damped frequency of oscillation

$\frac{1}{\tau} = a = \text{exponential decay freq.}$

Underdamped Second Order System

Step response:

$$C(s) = \frac{\omega_n^2}{s(s^2 + 2\xi\omega_n s + \omega_n^2)} = \frac{K_1}{s} + \frac{K_2 s + K_3}{s^2 + 2\xi\omega_n s + \omega_n^2}$$

$\xi < 1$

$$C(t) = 1 - \frac{1}{\sqrt{1-\xi^2}} e^{-\xi\omega_n t} \cos(\omega_n \sqrt{1-\xi^2} t - \phi)$$

$$1 - e^{-\xi\omega_n t} \left[\cos(\omega_n \sqrt{1-\xi^2} t) + \frac{\xi}{\sqrt{1-\xi^2}} \sin(\omega_n \sqrt{1-\xi^2} t) \right] \phi = \tan^{-1} \frac{\xi}{\sqrt{1-\xi^2}}$$

Metrics:

Peak Time (T_p)

$$c(t) = 0$$

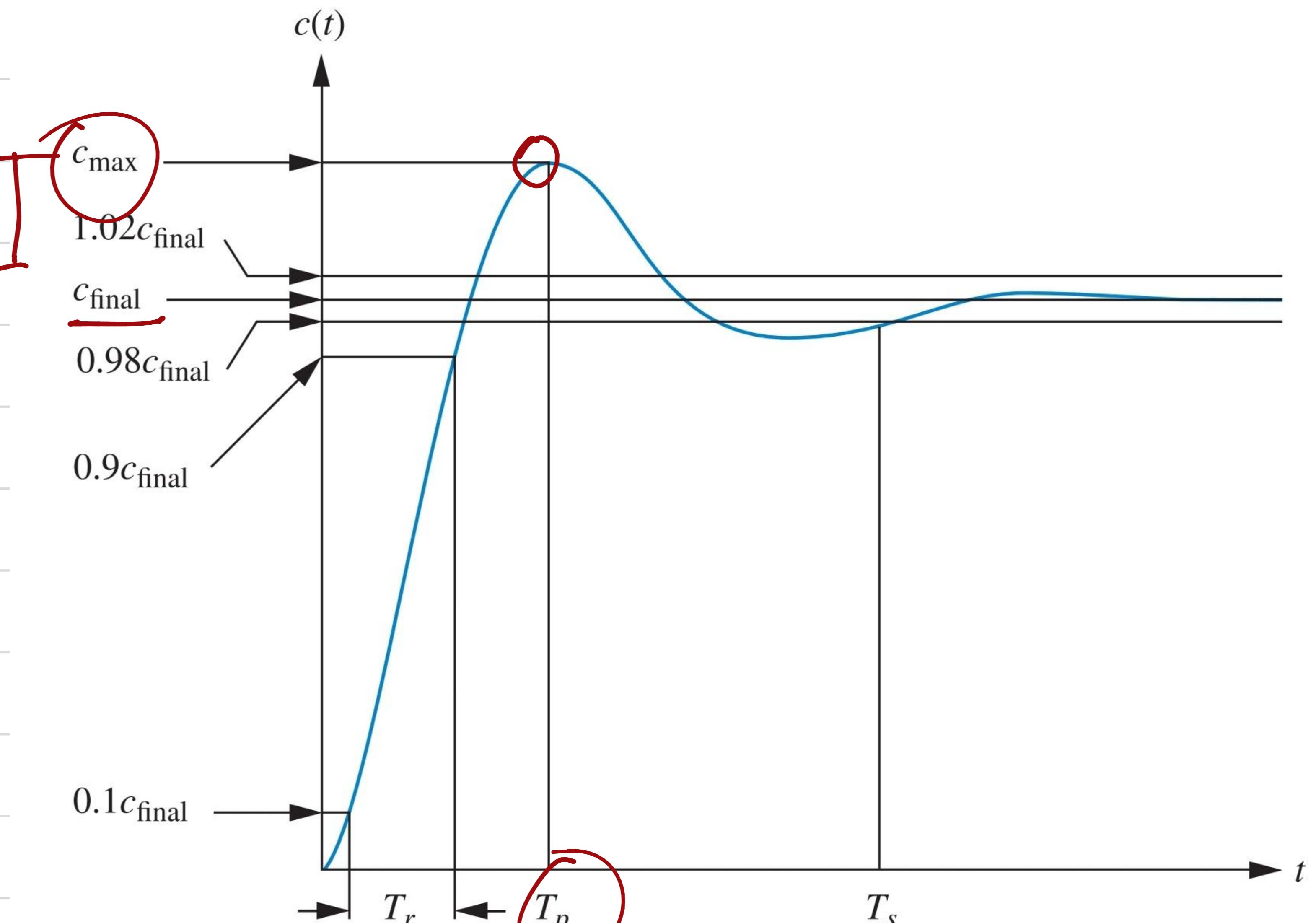
$$\frac{\omega_n}{\sqrt{1-\xi^2}} e^{-\xi \omega_n t} \sin \omega_n \sqrt{1-\xi^2} t = 0$$

$$\Rightarrow \sin(\omega_n \sqrt{1-\xi^2} t) = 0$$

$$\omega_n \sqrt{1-\xi^2} t = n\pi$$

$$t = \frac{n\pi}{\omega_n \sqrt{1-\xi^2}}$$

$t_p \text{ (at) } n=1$
 i.e. $t_p = \frac{\pi}{\omega_n \sqrt{1-\xi^2}}$



Peak time

$$= \frac{\pi}{\omega_d} = f(\omega_n, \xi)$$

Metrics:

To find overshoot \rightarrow

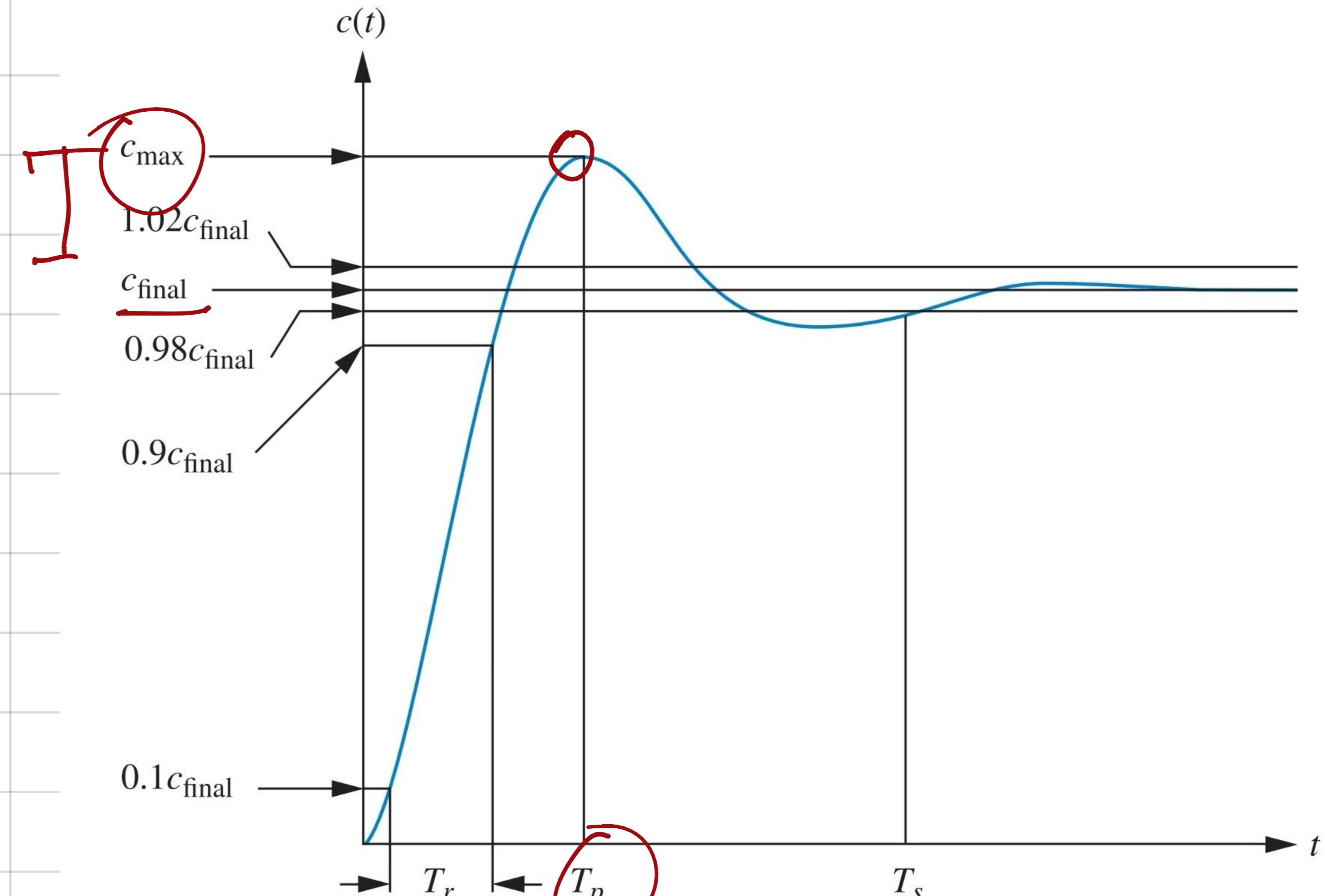
$$C(t_p) = e^{-\xi\pi/\sqrt{1-\xi^2}} + 1$$

$$\% OS = \frac{C(t_p) - C_{final}}{C_{final}} \times 100$$

$$\boxed{\% OS = 100 \cdot e^{-\xi\pi/\sqrt{1-\xi^2}}}$$

$$\hookrightarrow f(\xi)$$

$$\xi = \frac{-\ln (\% OS / 100)}{\sqrt{\pi^2 + \ln^2 (\% OS / 100)}}$$



To find the desired ξ for a specified OS

Peak time

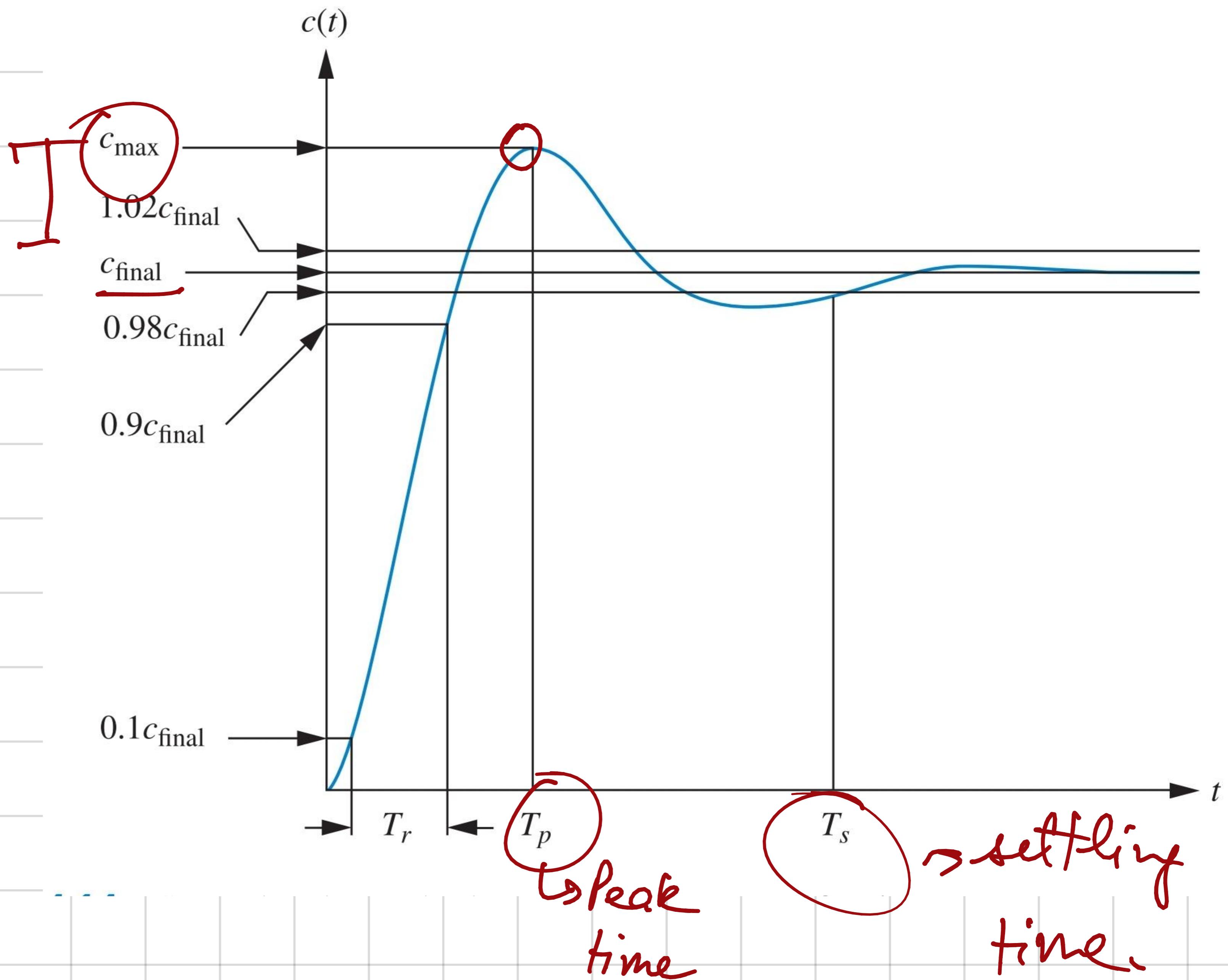
Metrics:

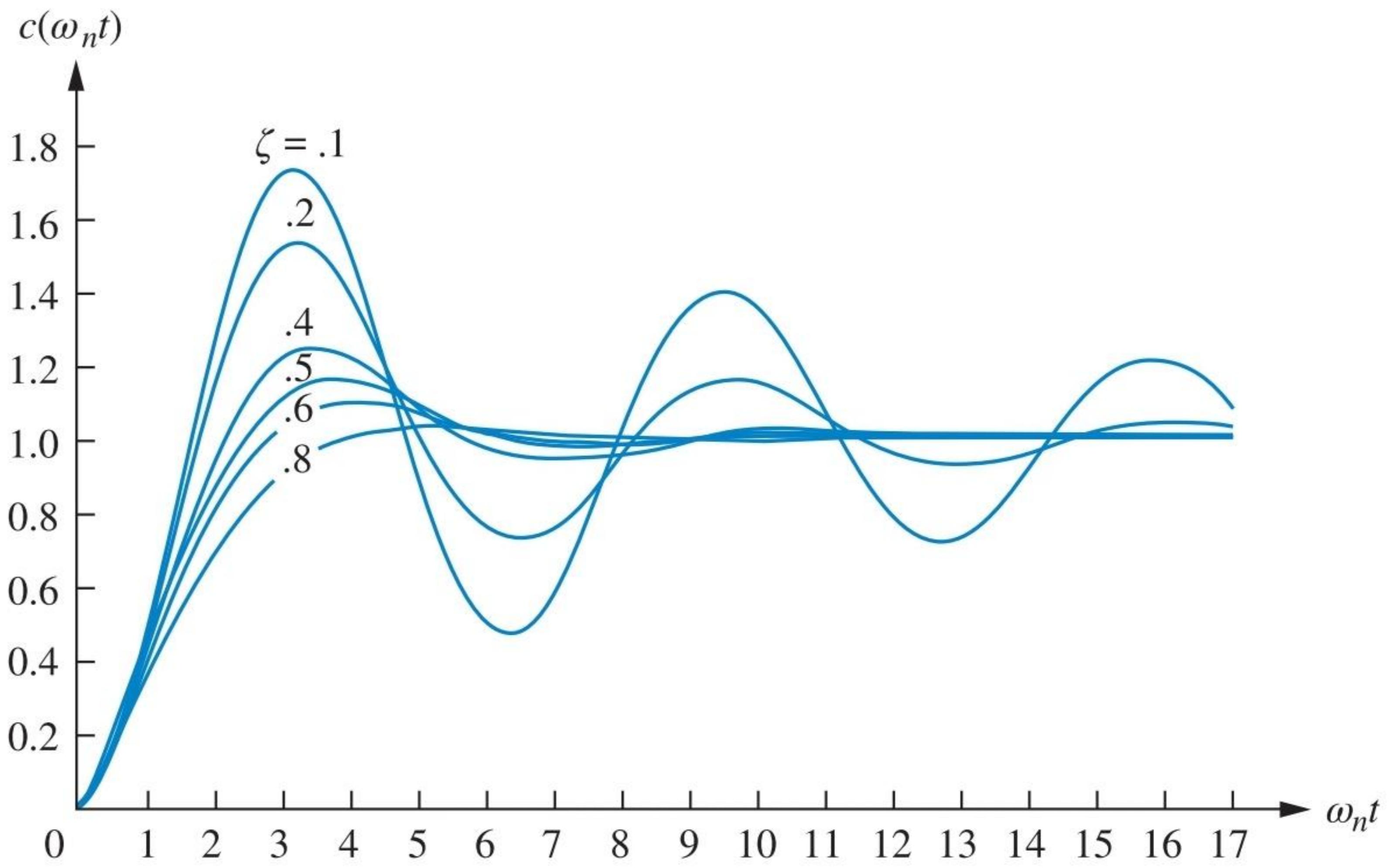
$$\left| \frac{e^{-\xi \omega_n t}}{\sqrt{1 - \xi^2}} \right| = 0.02$$

approximation assuming

$$\cos(\omega_n \sqrt{1 - \xi^2} t - \phi) \sim 1$$

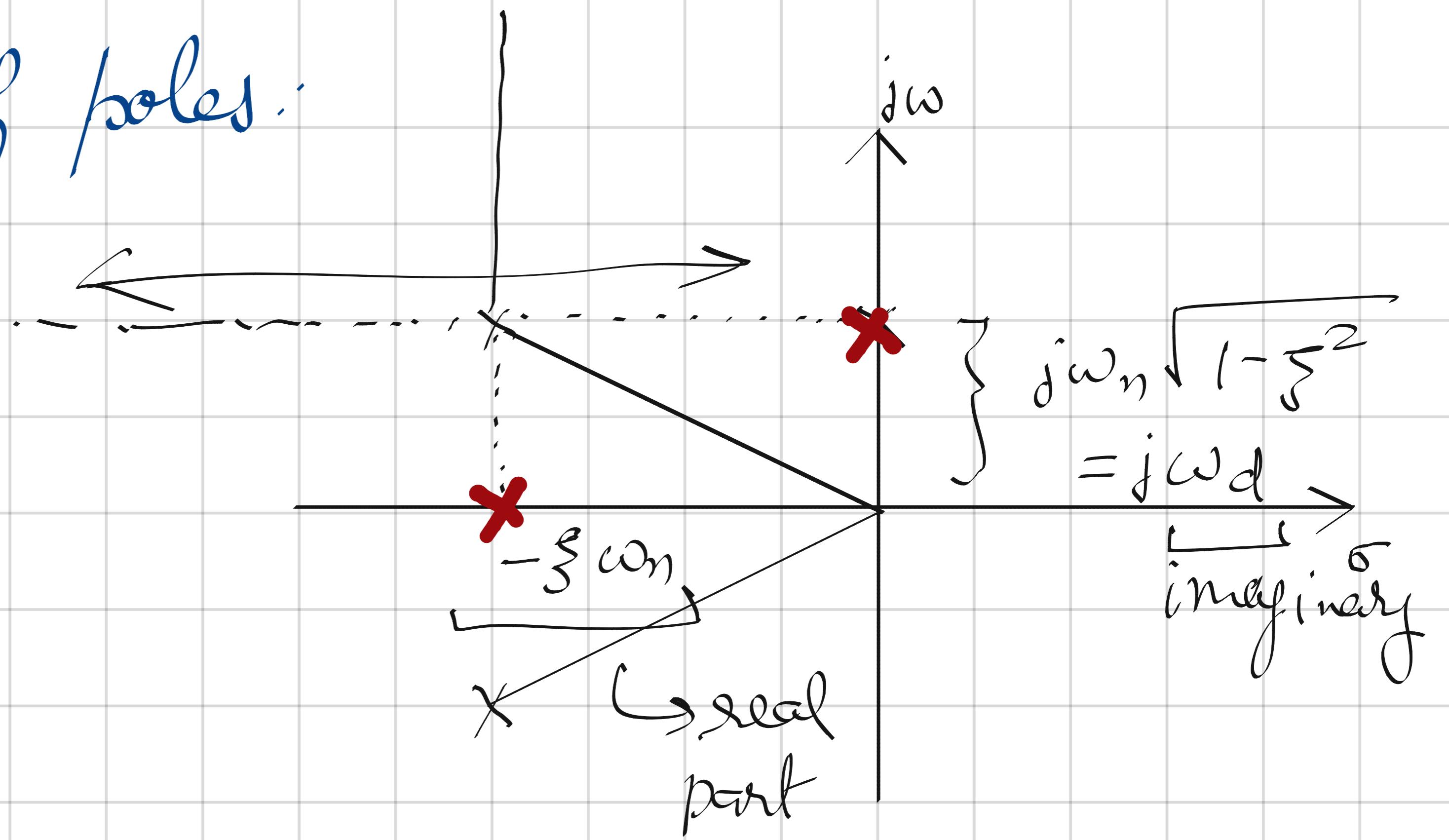
$$T_s = \frac{4}{\xi \omega_n}$$

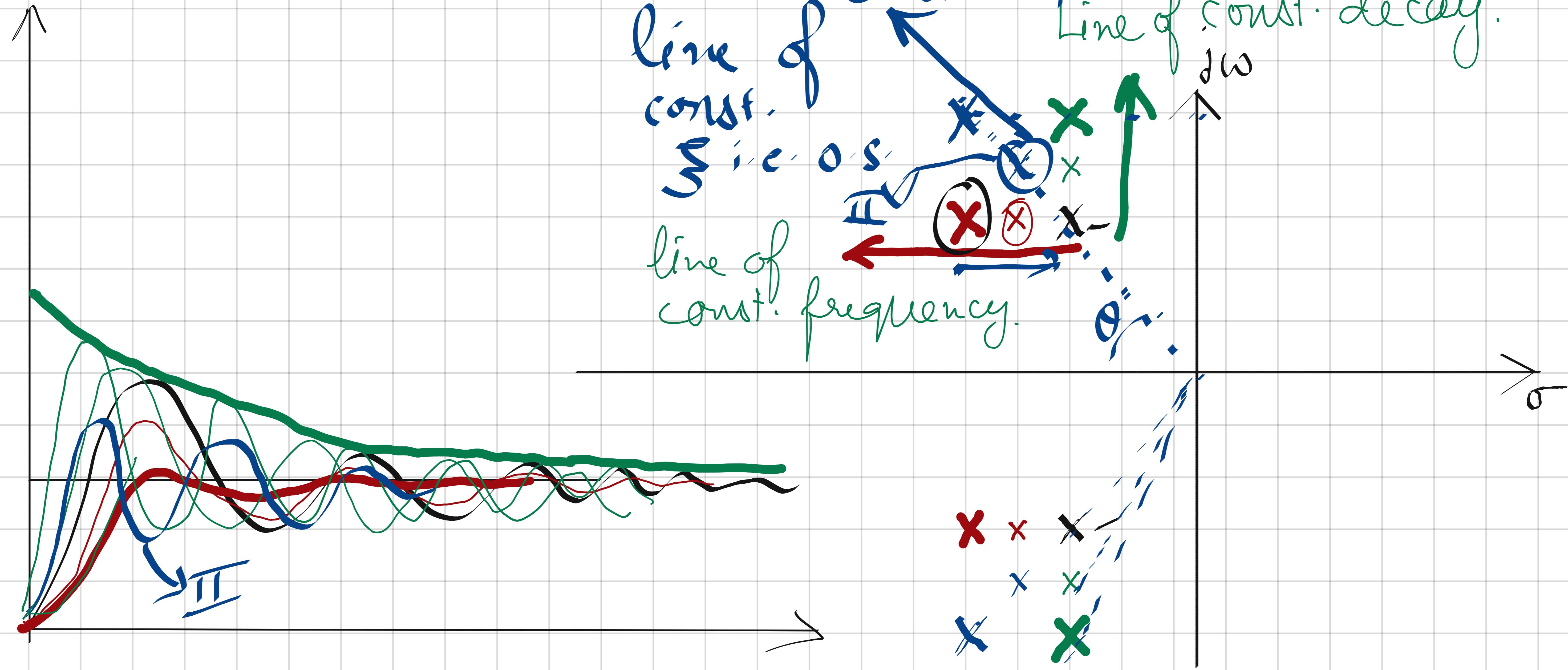




Graphical Understanding of effect of poles:

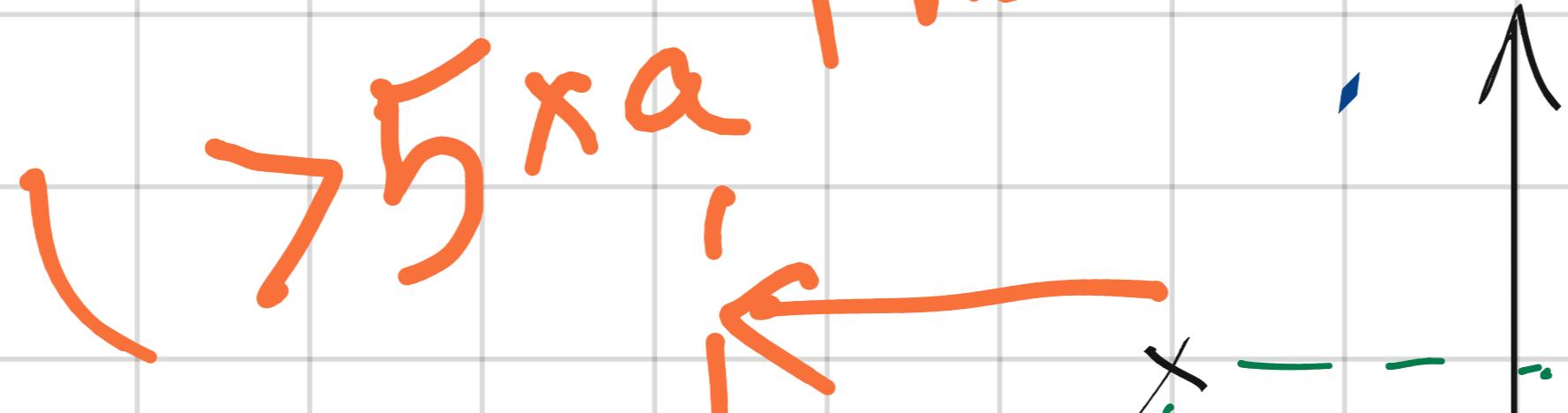
$$s = -\xi \omega_n \pm j\omega_n \sqrt{1 - \xi^2}$$





System response with additional Poles 🇵🇱

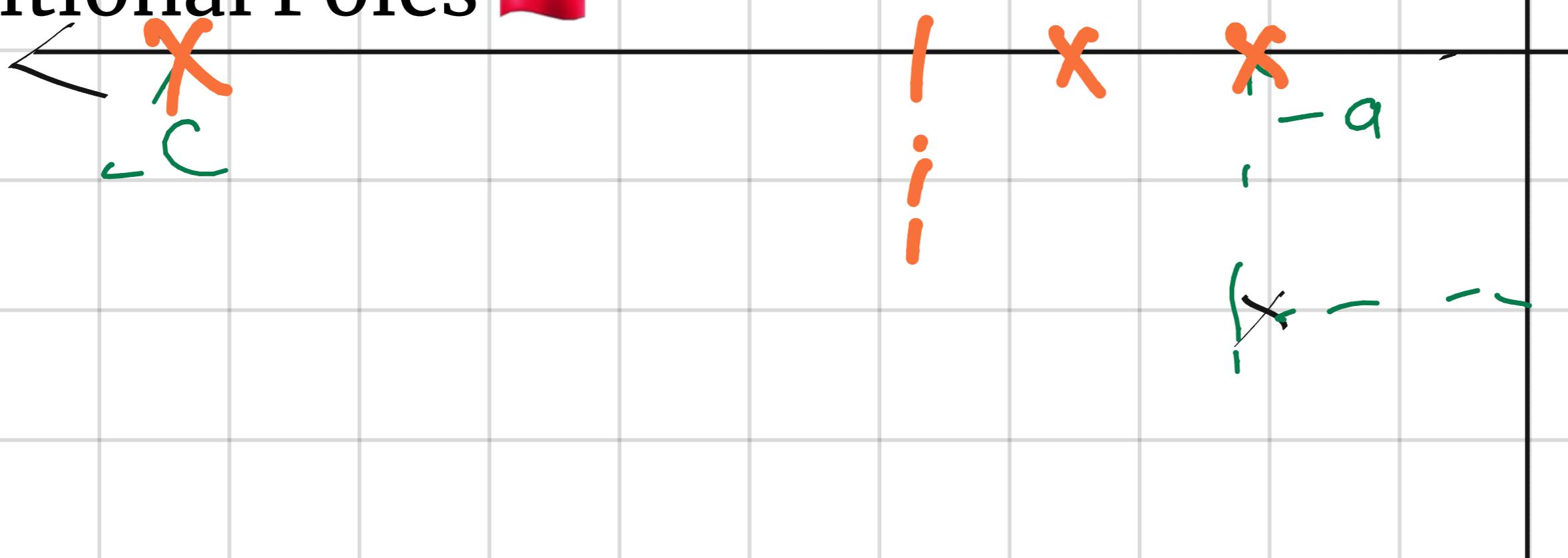
1 non dominant pole -



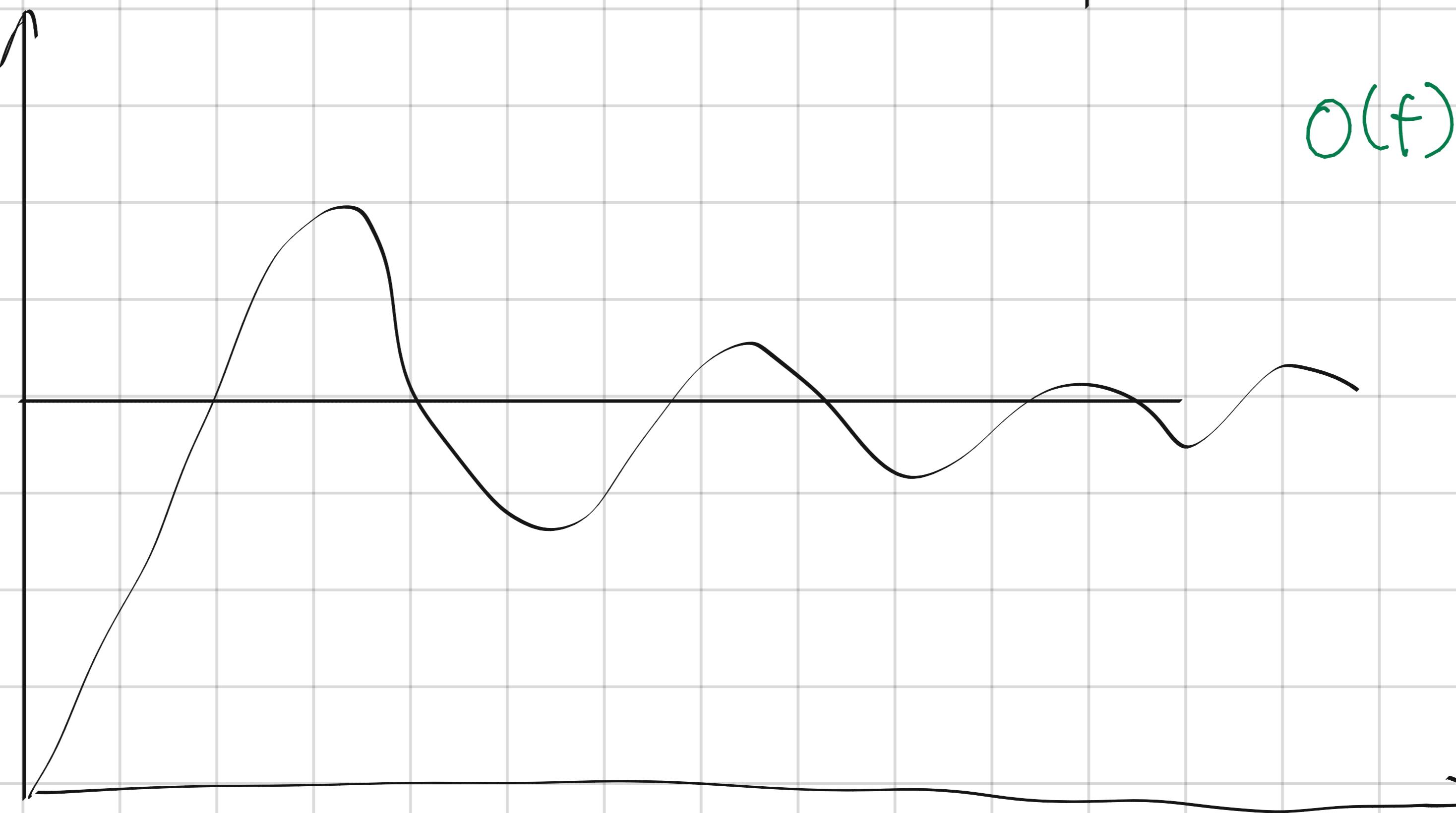
$$\frac{K_1}{(s - (-a+jb))(s - (-a-jb))}$$

System response with additional Poles 🇵🇱

~~X~~



$$(s+c)\frac{K_2}{(s - (-a+jb))(s - (-a-jb))}$$



$$O(f) = 1 - e^{-ct} - e^{-R_z t} \cos(\omega_d t - \phi)$$

System response with zeros

$$G(s) = \frac{1}{(s+1)(s+2)}$$

unit step response \rightarrow

$$c(t) = 1 - k_1 e^{-t} - k_2 e^{-2t}$$

$$= 0.5 - e^{-t} + \frac{1}{2} e^{-2t}$$

New system

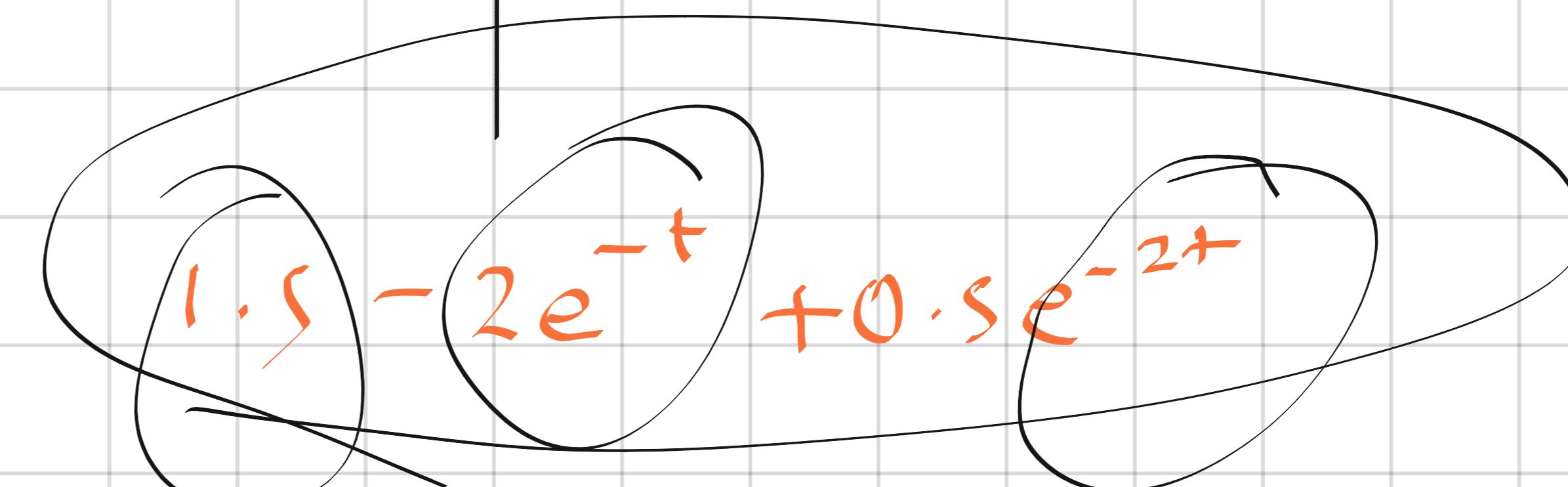
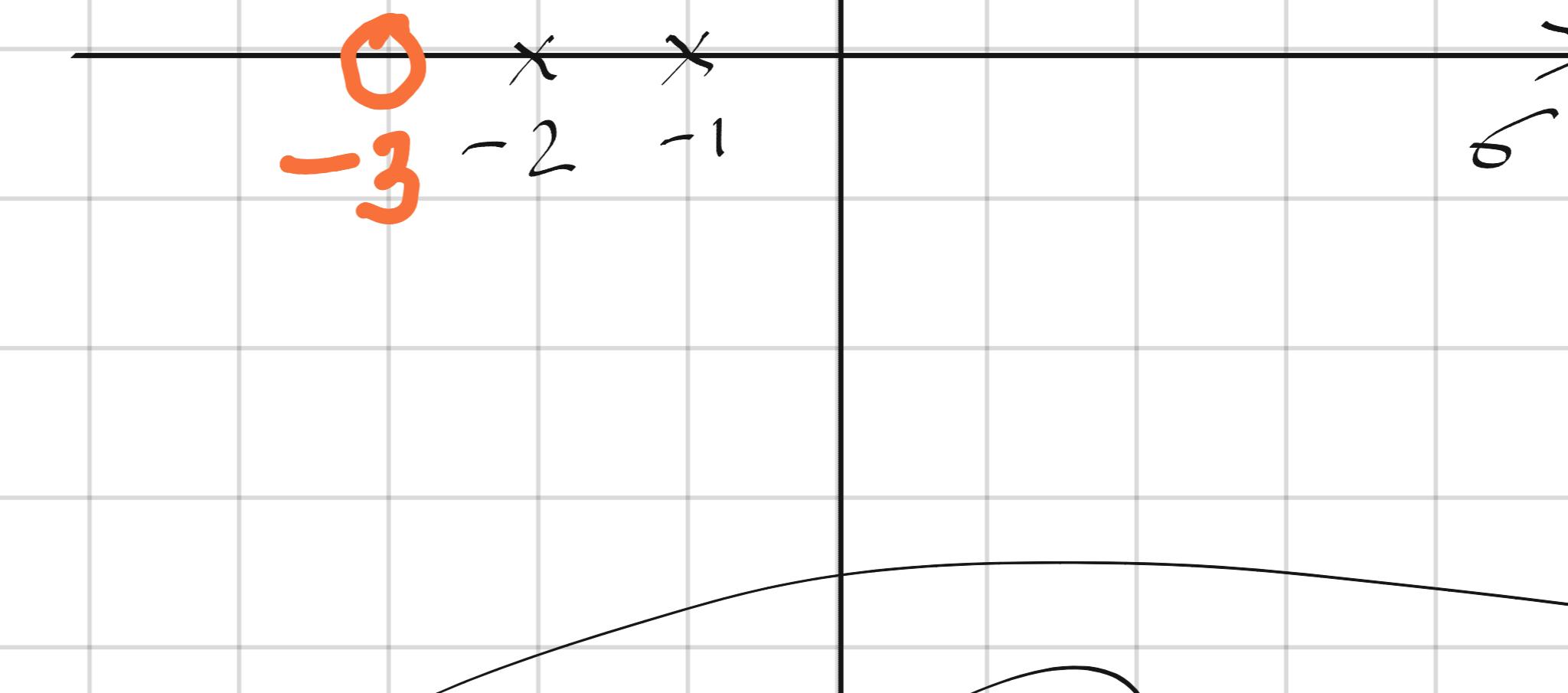
$$G_2(s) = \frac{s+3}{(s+1)(s+2)}$$

response to unit step I/P \rightarrow

$$G_2(s) = 3 \left[\frac{1}{s+1} - \frac{1}{2} \frac{1}{(s+2)} \right]$$

$$\frac{1}{s(s+1)(s+2)} = \frac{k_1}{s} + \frac{k_2}{s+1} + \frac{k_3}{s+2}$$

$s' \omega$

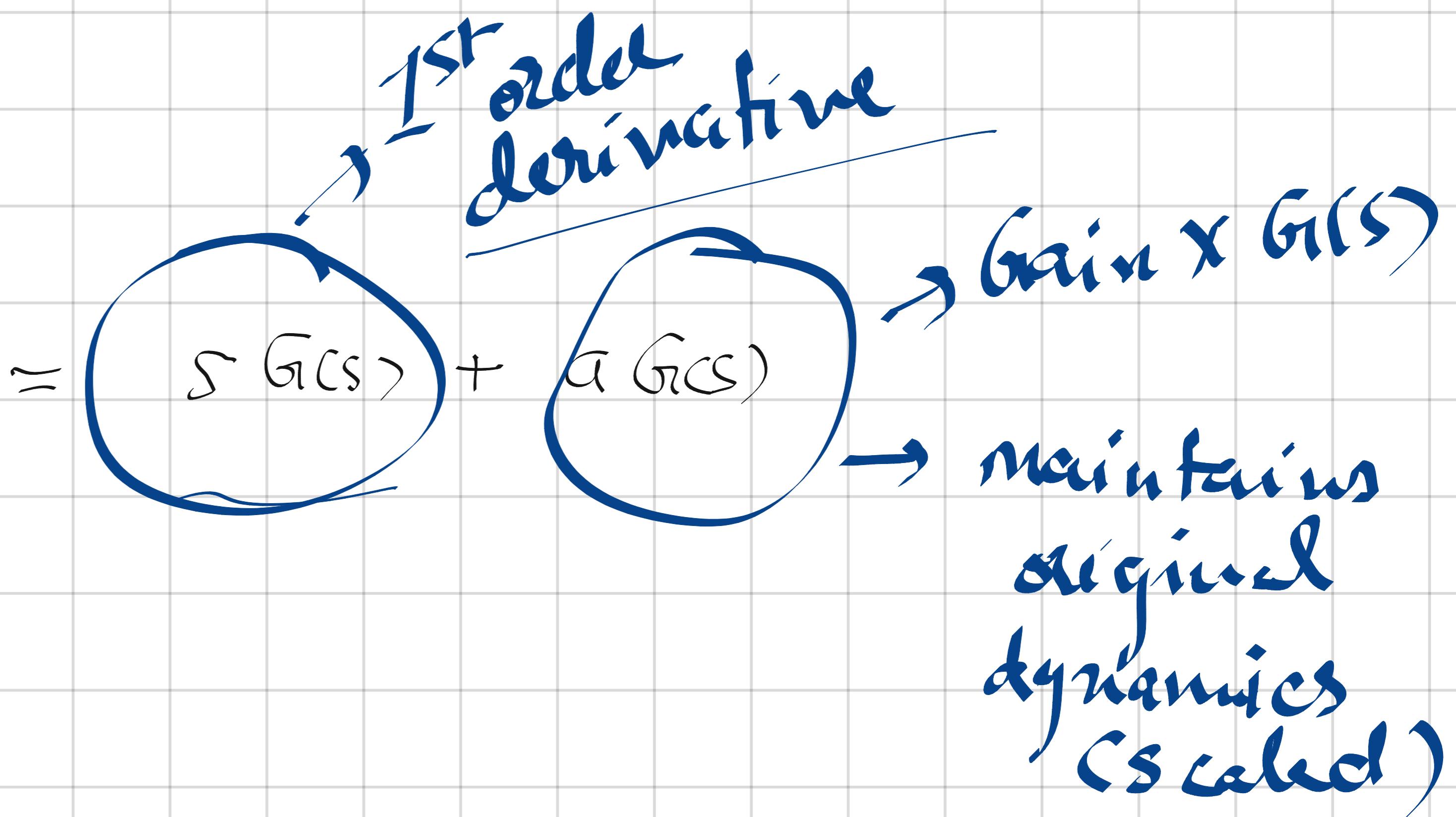


$$G(s) = \frac{9}{s^2 + 2s + 9}$$

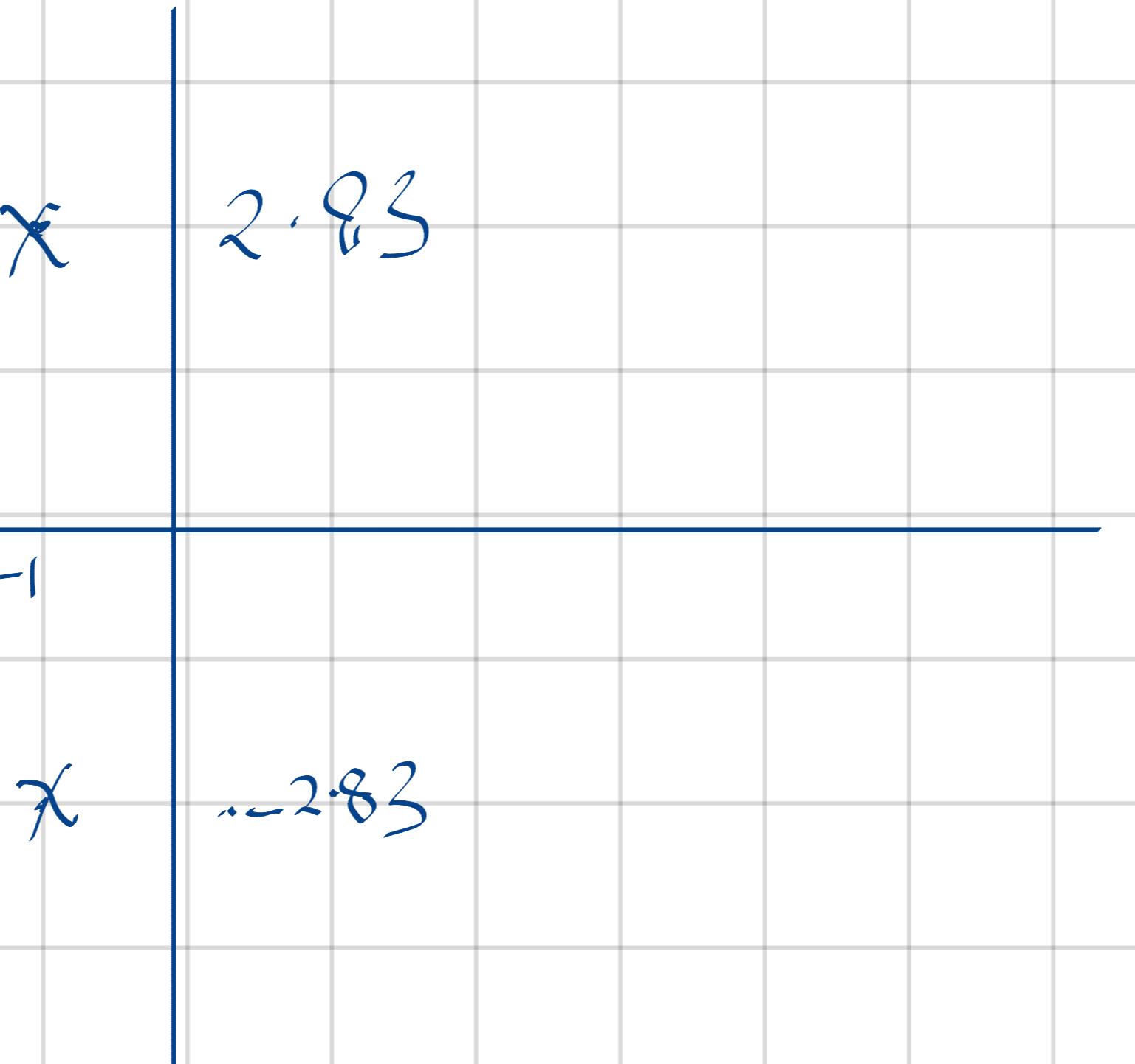
$$(s+a)$$

$$(s+a) G(s) = G_2(s) =$$

$$\frac{9(s+a)}{(s^2 + 2s + 9)}$$



$$\frac{9(s+3)}{s^2 + 2s + 9}$$

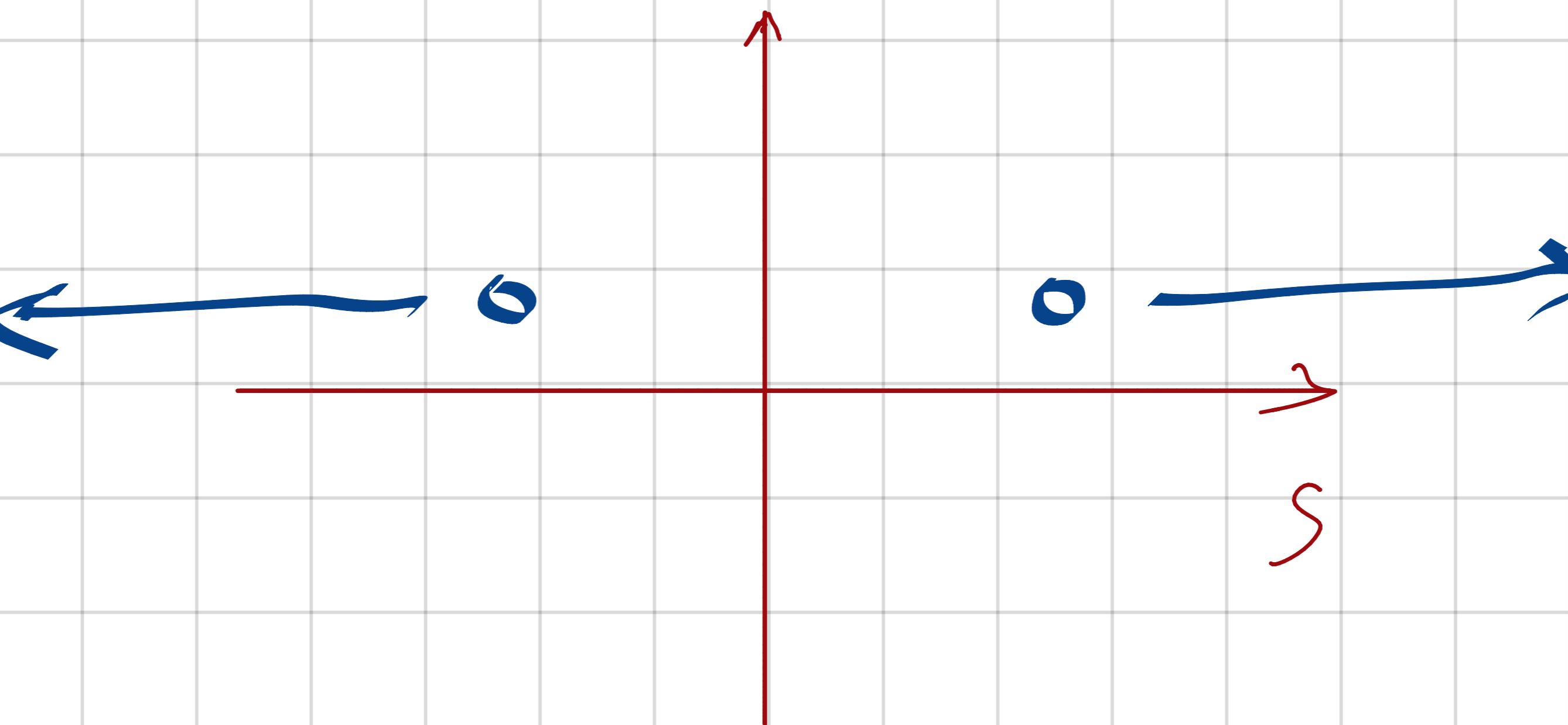
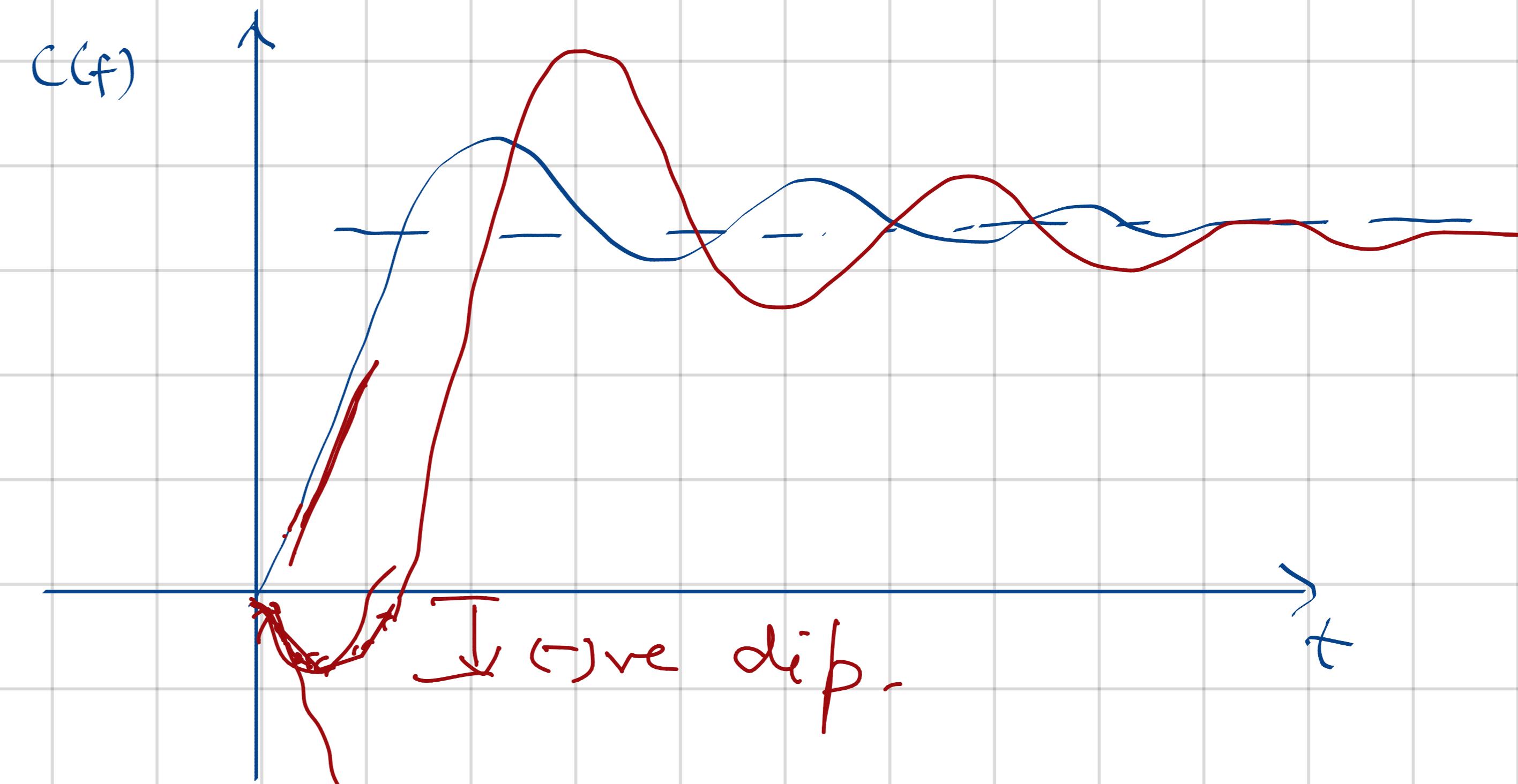


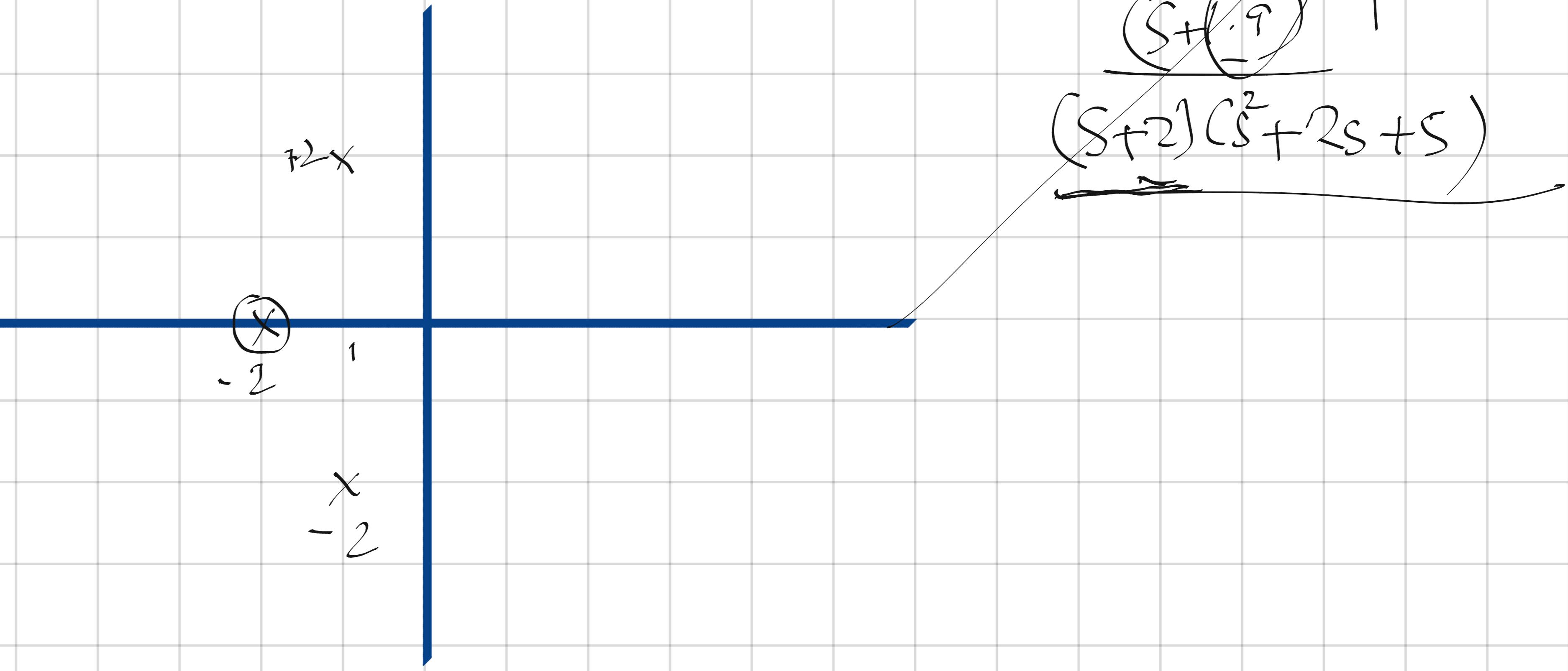
$$sG(s) = a G(s)$$

$$sG(s) + b G(s)$$

$\downarrow -3$

$a \rightarrow$ the number
Zero is in
RHP.





Stability

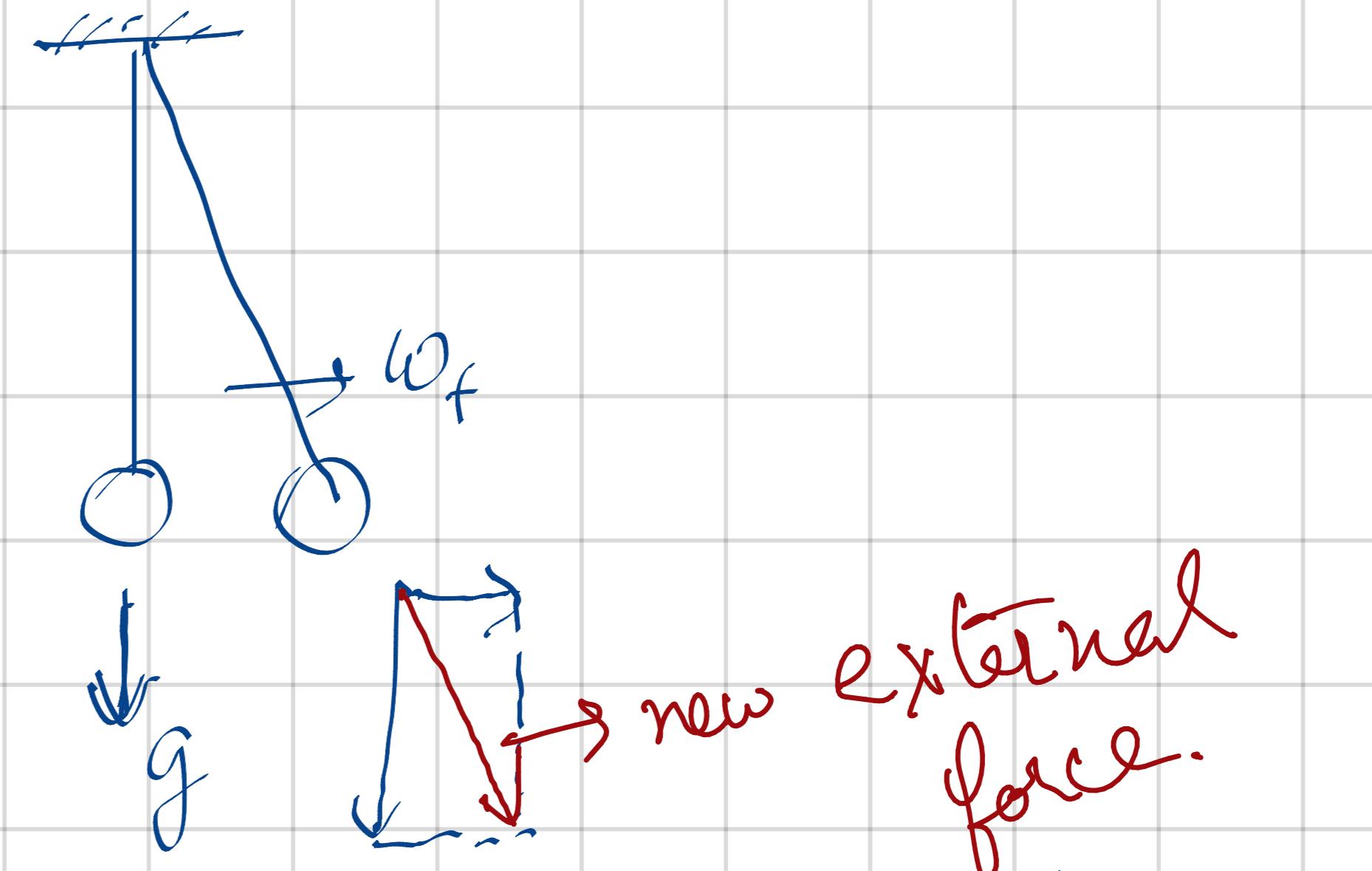
Criterions for stability.

System tends to a finite value.

System returns to its stable position when subjected to small disturbance.

Error wrt target $\rightarrow 0$ as $t \rightarrow \infty$

\rightarrow finite value



$$\text{denominator } a_n s^n + a_{n-1} s^{n-1} + a_{n-2} s^{n-2} \dots + a_0$$

Routh - Hurwitz Criterion

$$\begin{array}{|c|} \hline N(s) \\ \hline a_4 s^4 + a_3 s^3 + a_2 s^2 + a_1 s + a_0 \\ \hline \end{array}$$

Routh table to identify how many poles of the sys' are in the RHP without calculating the roots.

$$a_4 s^4 + a_3 s^3 + a_2 s^2 + a_1 s + a_0$$

$$\begin{array}{cc} a_3 & a_1 \\ b_1 & b_2 \\ \hline -\frac{\left| \begin{array}{cc} a_3 & a_1 \\ b_1 & b_2 \end{array} \right|}{b_1} \end{array}$$

Column 1
 a_4
 a_3
 b_1
 c_1
 d_1

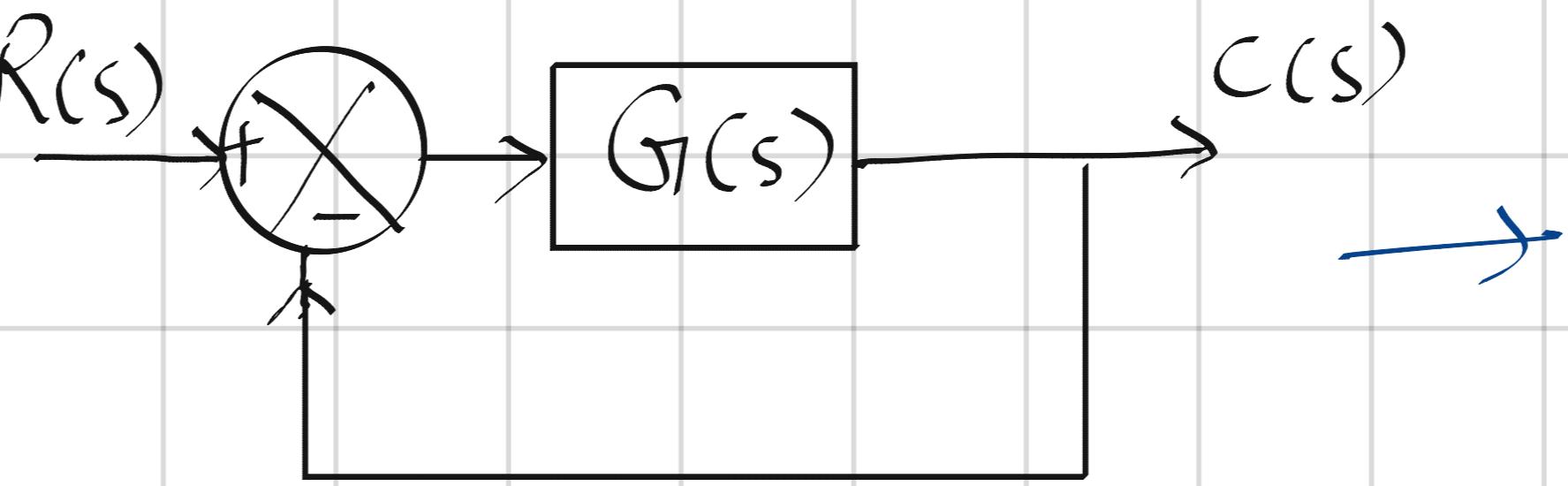
poles in RHP
 $= \# \text{ sign changes in column 1}$

s^4
 s^3
 s^2
 s^1
 s^0

$$\begin{aligned} & \begin{array}{c} a_4 \\ a_3 \\ a_2 \\ a_1 \\ a_0 \\ 0 \\ 0 \end{array} \\ & -\frac{\left| \begin{array}{cc} a_4 & a_2 \\ a_3 & a_1 \end{array} \right|}{a_3} = b_1 \\ & -\frac{\left| \begin{array}{cc} a_4 & a_0 \\ a_3 & 0 \end{array} \right|}{a_3} = b_2 \\ & -\frac{\left| \begin{array}{cc} a_3 & a_1 \\ b_1 & b_2 \end{array} \right|}{b_1} = c_1 \\ & -\frac{\left| \begin{array}{cc} b_1 & b_2 \\ c_1 & 0 \end{array} \right|}{c_1} = d_1 \end{aligned}$$

eg.
 $\begin{array}{r} 1 \\ 3 \\ 2 \\ 1 \\ 2 \end{array}$
 $\downarrow \quad \downarrow \quad \downarrow \quad \downarrow$
2 poles
in RHP

$$G(s) = \frac{1000}{(s+2)(s+3)(s+5)}$$



$$\frac{G(s)}{1+G(s)}$$

$$\frac{1000}{(s+2)(s+3)(s+5)+1000}$$

$$= \frac{1000}{s^3 + 10s^2 + 31s + 1030}$$

Roots $\rightarrow -13 \cdot 413$

$$\underline{1.7068 \pm 3.595i}$$

$$\begin{aligned}q_3 &= 1 \\q_2 &= 10 \\q_1 &= 31 \\q_0 &= 1030\end{aligned}$$

| | | | |
|-------|------|-------|---|
| s^3 | 1 | 31 | 0 |
| s^2 | 10 | +1030 | 0 |
| s^1 | -72 | -Q | 0 |
| s^0 | 1030 | +0 | 0 |

$$ss + 2s^4 + 3s^3 + 6s^2 + 5s + 3$$

$$ss \quad | \quad 3 \quad 5 \quad 0$$

$$s^4 \quad 2 \quad 6 \quad 3 \quad 0$$

$$s^3 \quad \cancel{0\varepsilon} \quad 3.s \quad 0 \quad 0$$

$$s^2 \quad \frac{6\varepsilon - 7}{\varepsilon} \quad (3 \quad 0) \quad 0$$

$$\frac{42\varepsilon - 49 - 6\varepsilon^2}{12\varepsilon - 19}$$

$$s^0 \quad 3$$

$\varepsilon \rightarrow$ infinitesimally
small number

$\varepsilon \rightarrow (+)$ ve

$\varepsilon \rightarrow (-)$ ve

$$ss \quad + \quad +$$

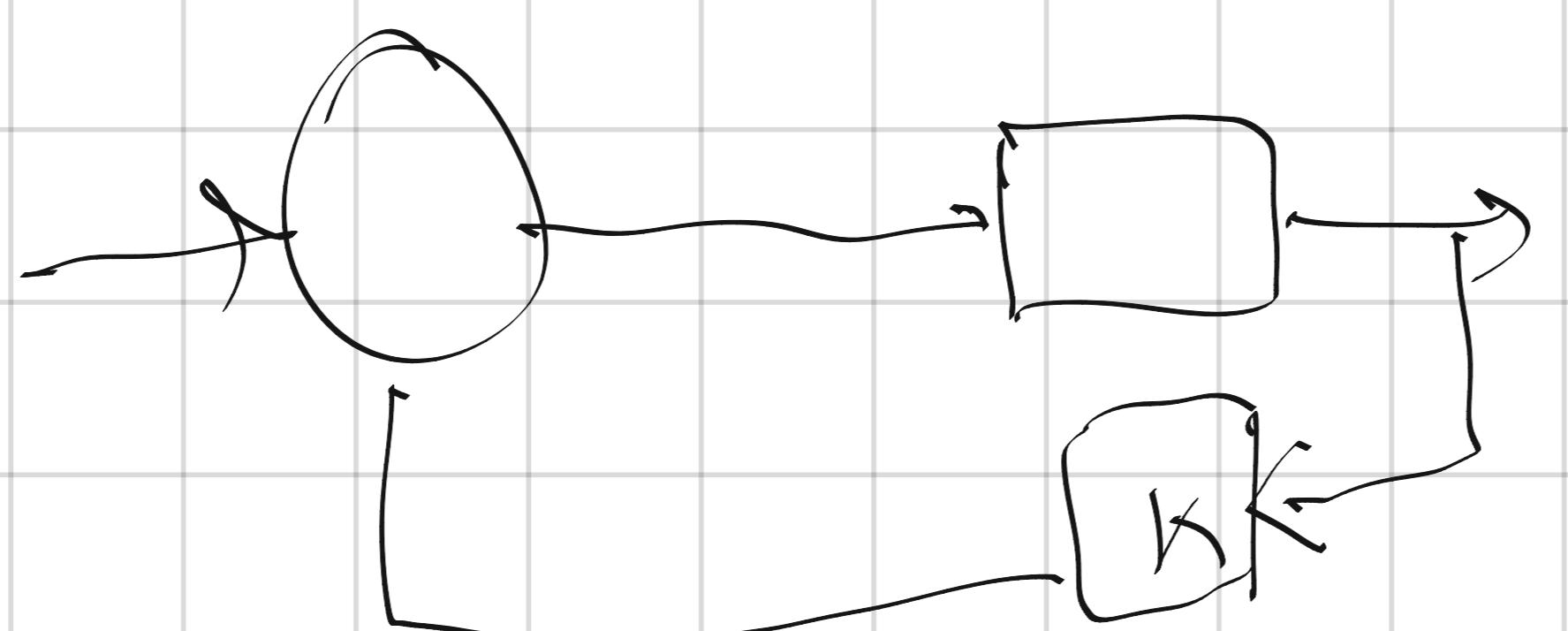
$$s^4 \quad + \quad +$$

$$s^3 \quad + \quad -$$

$$s^2 \quad - \quad +$$

$$s^1 \quad + \quad +$$

$$s^0 \quad + \quad +$$



↑ 2 sign
changed

