

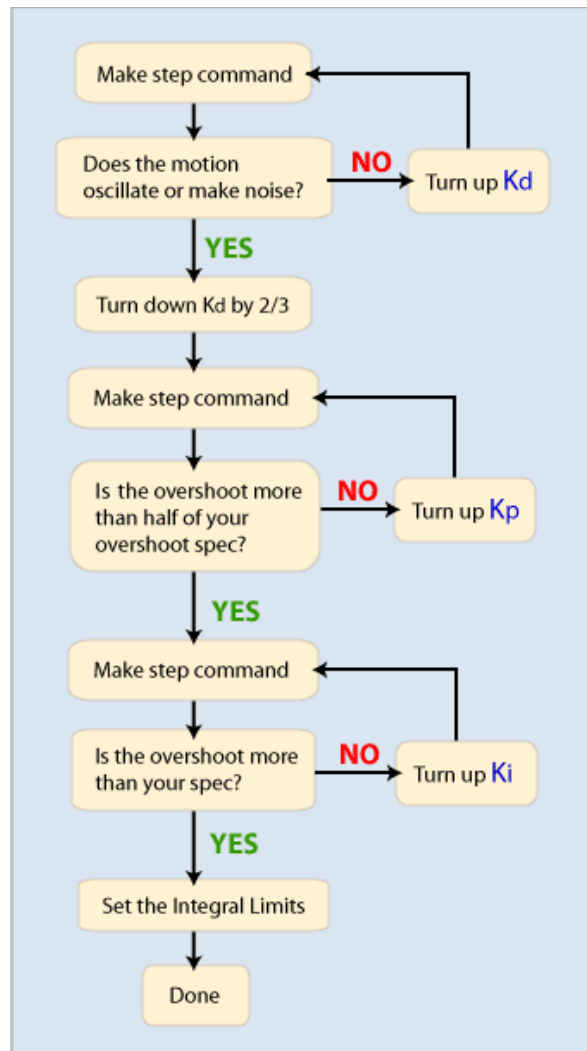


PID Tuning: *Step Behavior - Overshoot Criteria*

PID tuning with step behavior is accomplished by turning up a gain parameter until unwanted behavior (oscillation or noise) is achieved and then turn down the gain parameter until the desired behavior is achieved and the stage is stable. This method primarily focuses on tuning the position overshoot parameter.

The diagram below shows the natural progression of steps for this tuning method. To create a disturbance on the stage, a small commanded step is made.

1. [Optimize Kd](#)
2. [Optimize Kp](#)
3. [Optimize Ki](#)



When using this tuning method, be sure that the command output (DAC) is not clipping the step command. If the command output does clip the step command, you have a non-linear system. If the system is non-linear, the generalizations made on this page may not apply. If you are determined to tune despite the non-linearities, make sure that there is a repeatable way to generate the non-linearities, as well as the ability to try many different scenarios.

Before You Start

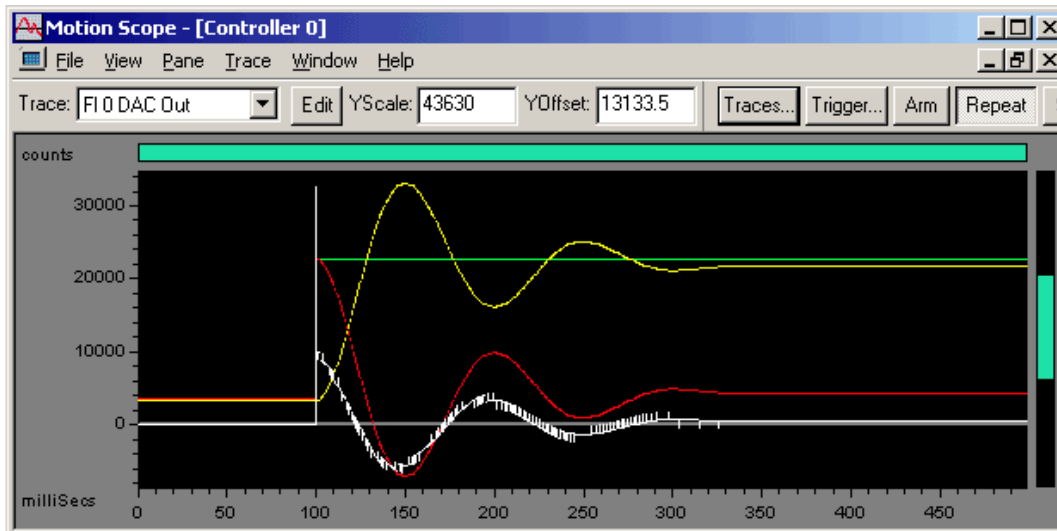
Kd: With Kp, Ki, Kpff, Kaff, Kvff, and Kfff all set to zero, command a small motion. You are going to make your step command with this motion, so choose an appropriate length. (Small moves are usually best.) In any case, do not clip the command output (DAC) severely, as that will make the result less representative of a normal move.

Use an extremely large acceleration, deceleration, and velocity ($1e10$ counts / sec^2 should be enough for most systems -- make sure the commanded position makes a sharp step as seen in Motion Scope) to make the commanded motion a step. The key to making a step response on the XMP is to turn the acceleration, deceleration, and velocity up high enough to complete the entire move in one servo sample.

The Motion Scope screenshot below shows the following traces:

- Commanded Position - Green
- Actual Position - Yellow
- Position Error - Red
- DAC Output - White

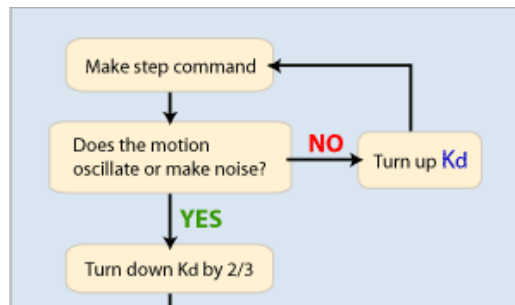
These traces help highlight the position step input (Commanded Position), and the response (Actual Position). The Position Error is helpful to see the difference in the commanded and actual position when the difference is small. The DAC output shows the torque command to the motor. High frequency oscillations are sometimes easier to see on the DAC signal than on the position plots.



100 Count Step Commanded (Trapezoidal move, $1e10$ accel, decel, vel)

- Green - Commanded Position
- Yellow - Actual Position
- Red - Position Error
- White - DAC Output

1. Optimize Kd



GOAL: To find the largest Kd value that can be used to command a step without getting any oscillation and then turn down Kd to a fixed amount (approx. 1/3 of the limit value).

It is best to start with a small Kd value like 1.0. After commanding a step, look and listen for oscillation.

If you can see oscillation in Motion Scope or hear/see an objectionable oscillation from the motor, turn Kd down and repeat.

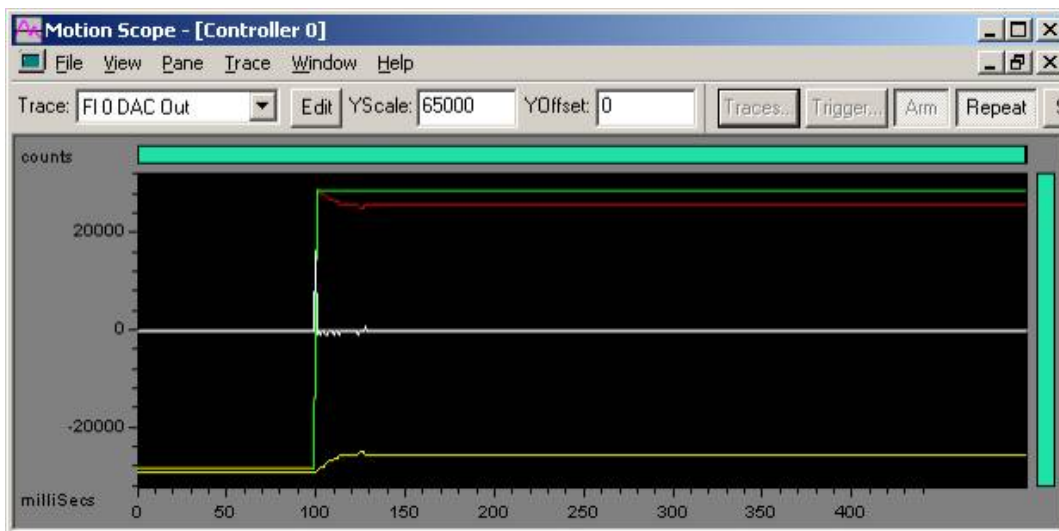
If you do NOT notice any oscillation, turn Kd up and repeat.

Stop adjusting Kd once you've found the largest Kd value that you can use to command a step without getting any oscillation.

After you find this limit Kd value, turn it down a fixed amount. Setting Kd to 1/3 of the limit value is a safe place to start. You can try turning Kd down if you need more performance. Keep in mind that the less you turn down Kd, the less stable your system will be. At some point, if Kd is too large, you will get oscillation (buzzing, humming, etc). Remember, if you are tuning a machine for a production release, you can actually decrease the stability of the machine if the Kd is too large. It is recommended that you do not push Kd too far and allow yourself a buffer for safety and stability reasons.

It is often difficult to visibly see small and high frequency oscillations. It is usually easier to see oscillations on DAC output plots. Remember, the goal is to find the high-end limitation of Kd, not to find the most accurate motion.

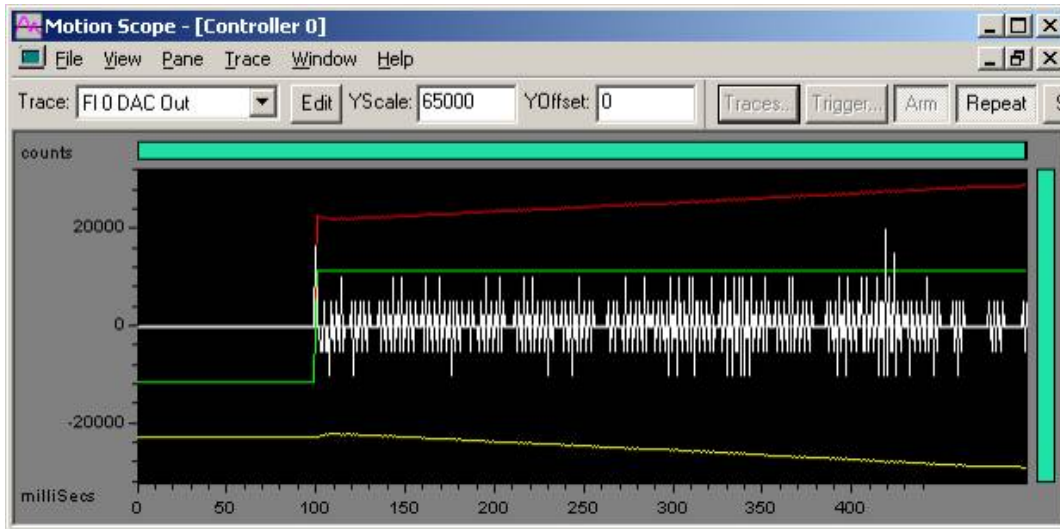
A Trust TA 9000 was used to generate the following examples, unless otherwise noted.
(<http://www.trustautomation.com>)



Acceptable motion on 100 count step move

(No objectionable audible noise on move and no audible noise when move is settled)

Green - Commanded Position
 Yellow - Actual Position
 Red - Position Error
 White - DAC Output
 Kd = 2000



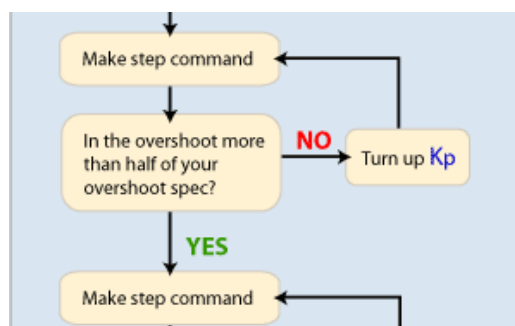
Excessive oscillation on 100 count step move

Green - Commanded Position
 Yellow - Actual Position
 Red - Position Error
 White - DAC Output
 Kd = 7000

In the plot above, when Kd = 7000, instability is found. When Kd > 6000, it results in audible noise and an erratic plot in Motion Scope.

Turn down Kd by 2/3 ($6000 / 3 = 2000$) and continue to Step 2 and tune Kp.

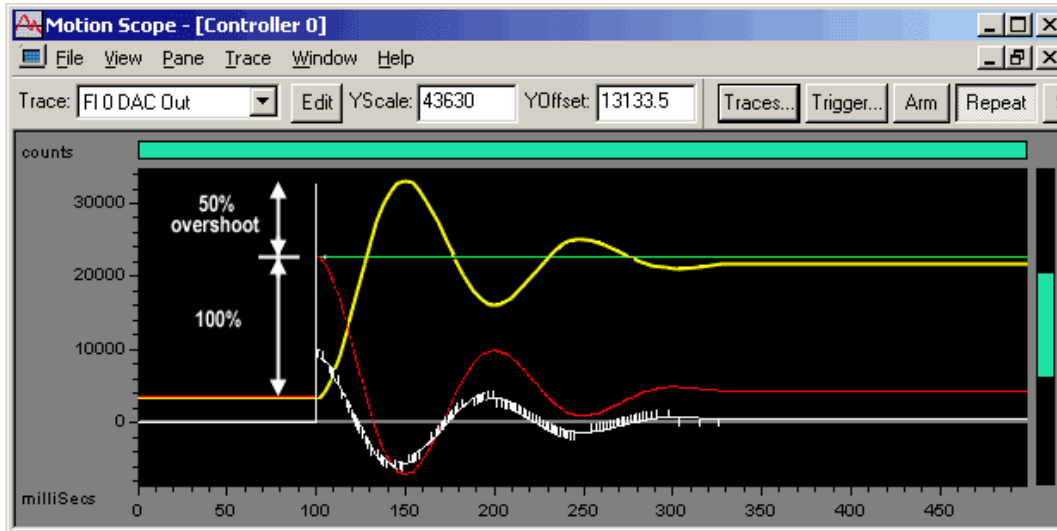
2. Optimize Kp



GOAL: To turn up Kp until there is half the overshoot of a step command.

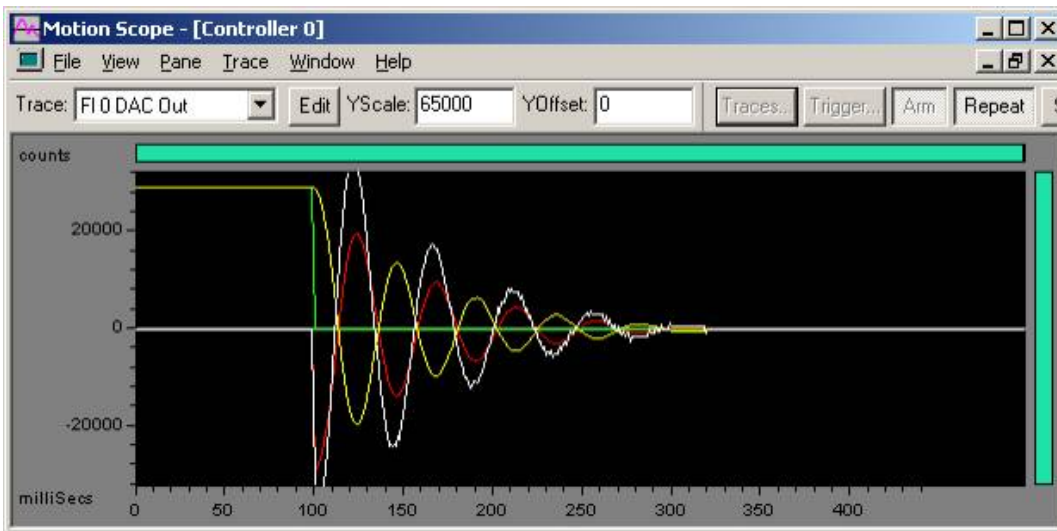
Once you have tuned Kd, start to tune Kp. Turn up Kp until you get an overshoot of 50% of the specified step command. For example, if you command a 100 count step, a 50% overshoot means the peak actual position value is 150 counts. The 50% overshoot is a

simple, repeatable measure of PID loop stability. Since your overshoot will automatically get larger when you tune K_i , this will provide some tuning flexibility.



In the plot above, the yellow trace (actual position) shows a 50% overshoot.

We now start with a $K_p = 500$.



Excessive oscillation on 100 count step move

Green - Commanded Position

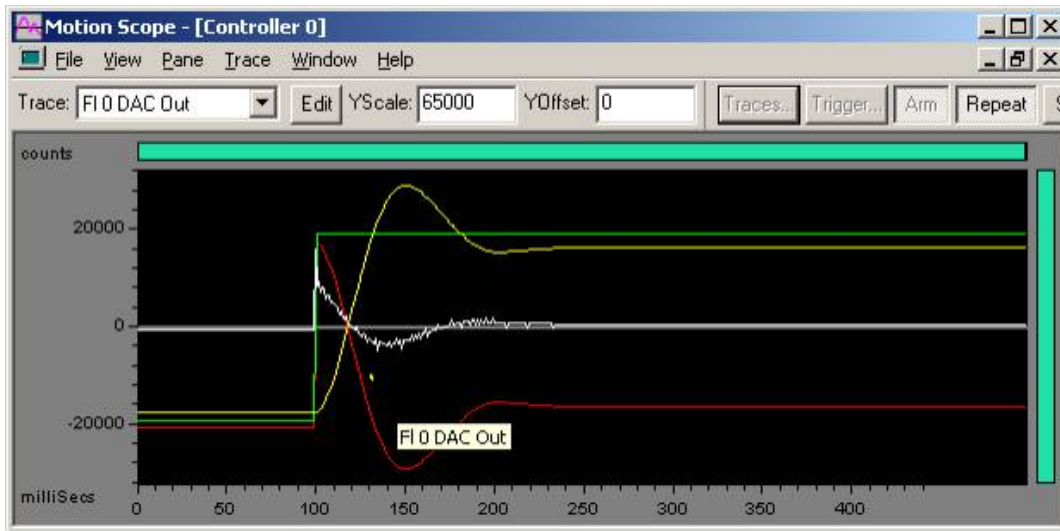
Yellow - Actual Position

Red - Position Error

White - DAC Output

$K_p = 500$

$K_d = 2000$



Acceptable overshoot (26%) on 100 count step move

Green - Commanded Position

Yellow - Actual Position

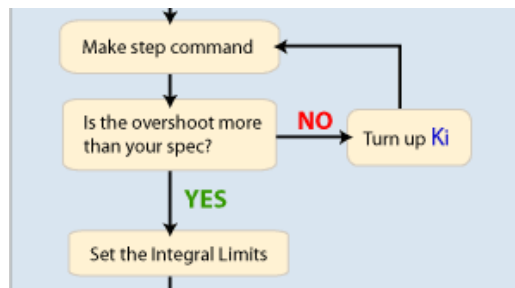
Red - Position Error

White - DAC Output

$K_p = 100$

$K_d = 2000$

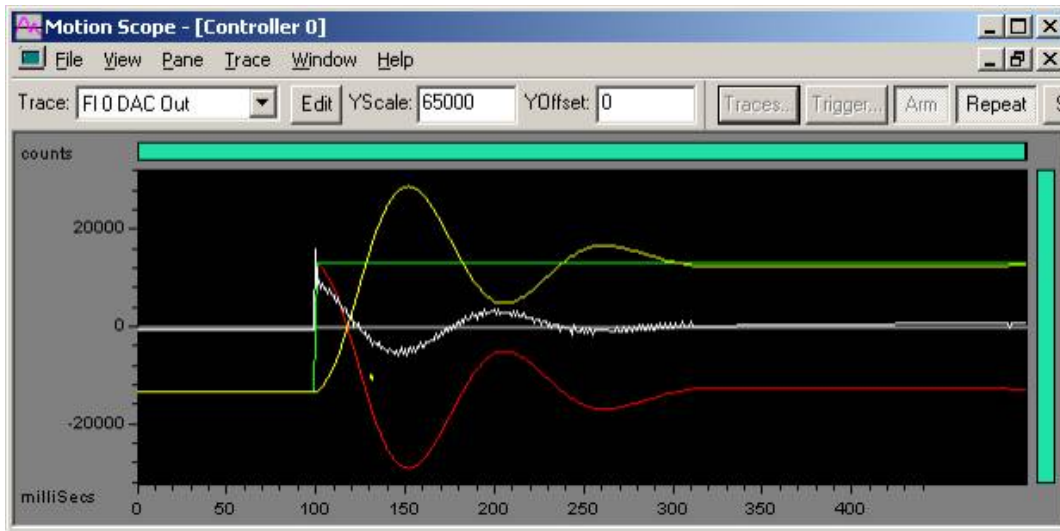
3. Optimize K_i



Once you have tuned K_p , start to tune K_i . Set $I_{maxMoving}$ and $I_{maxRest}$ to the same value as Output Limit. In this case Output Limit is 32767.

$I_{maxMoving} = I_{maxRest} = \text{Output Limit} = 32767$

K_i is the last closed loop parameter, so we will want to turn up K_i until we get the specified overshoot. Turning up K_i will close the steady state error.



Acceptable overshoot (50%) on 100 count step move

Green - Commanded Position

Yellow - Actual Position

Red - Position Error

White - DAC Output

 $K_i = 0.43$ $K_p = 100$ $K_d = 2000$

Integral Limits

In a PID system, the proportional (K_p), derivative (K_d), and integral (K_i) terms all contribute to the total controller output (the three terms are added together). When a system oscillates at low frequency (the frequency varies, but is often a few Hertz lower), the integral term is likely the term that is oscillating. K_p and K_d are along for the ride and are not affected. In an effort to control an oscillation due to the integral term, we can limit how much it contributes to the controller output. We limit the maximum contribution of the integral term while the stage is at rest with $IMaxRest$. We limit the maximum contribution of the integral term during a motion with $IMaxLimit$. There are several ways you can set these limits:

Full integral effect always - By setting $IMaxRest$ and $IMaxLimit$ equal to Output Limit, you will have no effective limit on the integral term. ($IMaxRest = IMaxLimit = Output\ Limit$)

Full integral at rest - Some systems may already cause enough disturbances when set in motion. The integral will simply make things worse by trying to compensate for the disturbances. If an integral term is still needed to close steady state error, setting $IMaxLimit$ to zero and $IMaxRest$ equal to Output Limit will turn off the integral during motion, thereby enhancing stability.

To close steady state error when **in motion**, (Integral is ON)

$IMaxLimit = 0$

$IMaxRest = Output\ Limit$

To close steady state error when **at rest**, (Integral is OFF)

Half integral limit - It is possible to tune a stage so that it visually appears stable when in reality it is very unstable and may result in violent oscillation when the stage is grossly disturbed (ex: some large external force jars the stage). This is likely the result of an oscillating integral. By setting the integral limit to far less than the Output limit, integral

oscillation can be damped out by the proportional and derivative terms. This is why it is important to set the integral limit to be larger than what is typically used during normal operation while also making sure that it is still significantly less than the output limit. It is often good to start with half of the output limit.

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