

# Rotation

## 10.1 ROTATIONAL VARIABLES

### Learning Objectives

After reading this module, you should be able to . . .

- 10.1.1** Identify that if all parts of a body rotate around a fixed axis locked together, the body is a rigid body. (This chapter is about the motion of such bodies.)
- 10.1.2** Identify that the angular position of a rotating rigid body is the angle that an internal reference line makes with a fixed, external reference line.
- 10.1.3** Apply the relationship between angular displacement and the initial and final angular positions.
- 10.1.4** Apply the relationship between average angular velocity, angular displacement, and the time interval for that displacement.
- 10.1.5** Apply the relationship between average angular acceleration, change in angular velocity, and the time interval for that change.
- 10.1.6** Identify that counterclockwise motion is in the positive direction and clockwise motion is in the negative direction.
- 10.1.7** Given angular position as a *function of time*, calculate the instantaneous angular velocity at any particular time and the average angular velocity between any two particular times.

### Key Ideas

- To describe the rotation of a rigid body about a fixed axis, called the rotation axis, we assume a reference line is fixed in the body, perpendicular to that axis and rotating with the body. We measure the angular position  $\theta$  of this line relative to a fixed direction. When  $\theta$  is measured in radians,

$$\theta = \frac{s}{r} \quad (\text{radian measure}),$$

where  $s$  is the arc length of a circular path of radius  $r$  and angle  $\theta$ .

- Radian measure is related to angle measure in revolutions and degrees by

$$1 \text{ rev} = 360^\circ = 2\pi \text{ rad.}$$

- 10.1.8** Given a *graph* of angular position versus time, determine the instantaneous angular velocity at a particular time and the average angular velocity between any two particular times.
- 10.1.9** Identify instantaneous angular speed as the magnitude of the instantaneous angular velocity.
- 10.1.10** Given angular velocity as a *function of time*, calculate the instantaneous angular acceleration at any particular time and the average angular acceleration between any two particular times.
- 10.1.11** Given a *graph* of angular velocity versus time, determine the instantaneous angular acceleration at any particular time and the average angular acceleration between any two particular times.
- 10.1.12** Calculate a body's change in angular velocity by integrating its angular acceleration function with respect to time.
- 10.1.13** Calculate a body's change in angular position by integrating its angular velocity function with respect to time.

- A body that rotates about a rotation axis, changing its angular position from  $\theta_1$  to  $\theta_2$ , undergoes an angular displacement

$$\Delta\theta = \theta_2 - \theta_1,$$

where  $\Delta\theta$  is positive for counterclockwise rotation and negative for clockwise rotation.

- If a body rotates through an angular displacement  $\Delta\theta$  in a time interval  $\Delta t$ , its average angular velocity  $\omega_{\text{avg}}$  is

$$\omega_{\text{avg}} = \frac{\Delta\theta}{\Delta t}.$$

The (instantaneous) angular velocity  $\omega$  of the body is

$$\omega = \frac{d\theta}{dt}.$$

Both  $\omega_{\text{avg}}$  and  $\omega$  are vectors, with directions given by a right-hand rule. They are positive for counterclockwise rotation and negative for clockwise rotation. The magnitude of the body's angular velocity is the angular speed.

- If the angular velocity of a body changes from  $\omega_1$  to  $\omega_2$  in a time interval  $\Delta t = t_2 - t_1$ , the average angular acceleration  $\alpha_{\text{avg}}$  of the body is

$$\alpha_{\text{avg}} = \frac{\omega_2 - \omega_1}{t_2 - t_1} = \frac{\Delta\omega}{\Delta t}.$$

The (instantaneous) angular acceleration  $\alpha$  of the body is

$$\alpha = \frac{d\omega}{dt}.$$

Both  $\alpha_{\text{avg}}$  and  $\alpha$  are vectors.

## What Is Physics?

As we have discussed, one focus of physics is motion. However, so far we have examined only the motion of **translation**, in which an object moves along a straight or curved line, as in Fig. 10.1.1a. We now turn to the motion of **rotation**, in which an object turns about an axis, as in Fig. 10.1.1b.

You see rotation in nearly every machine, you use it every time you open a beverage can with a pull tab, and you pay to experience it everytime you go to an amusement park. Rotation is the key to many fun activities, such as hitting a long drive in golf (the ball needs to rotate in order for the air to keep it aloft longer) and throwing a curveball in baseball (the ball needs to rotate in order for the air to push it left or right). Rotation is also the key to more serious matters, such as metal failure in aging airplanes.



Mike Segar/Reuters/Newscom

**Figure 10.1.1** Figure skater Sasha Cohen in motion of (a) pure translation in a fixed direction and (b) pure rotation about a vertical axis.

We begin our discussion of rotation by defining the variables for the motion, just as we did for translation in Chapter 2. As we shall see, the variables for rotation are analogous to those for one-dimensional motion and, as in Chapter 2, an important special situation is where the acceleration (here the rotational acceleration) is constant. We shall also see that Newton's second law can be written for rotational motion, but we must use a new quantity called *torque* instead of just force. Work and the work–kinetic energy theorem can also be applied to rotational motion, but we must use a new quantity called *rotational inertia* instead of just mass. In short, much of what we have discussed so far can be applied to rotational motion with, perhaps, a few changes.

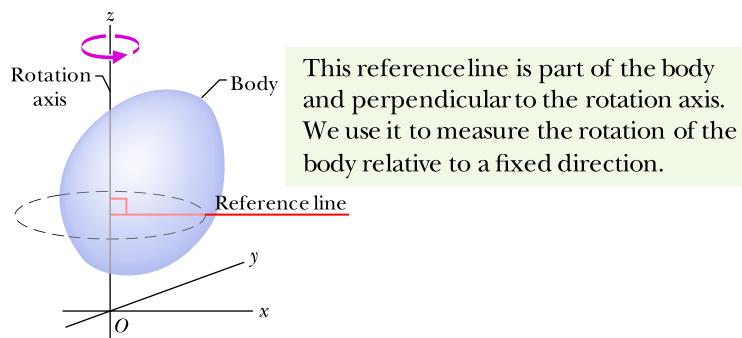
**Caution:** In spite of this repetition of physics ideas, many students find this and the next chapter very challenging. Instructors have a variety of reasons as to why, but two reasons stand out: (1) There are a lot of symbols (with Greek letters) to sort out. (2) Although you are very familiar with linear motion (you can get across the room and down the road just fine), you are probably very unfamiliar with rotation (and that is one reason why you are willing to pay so much for amusement park rides). If a homework problem looks like a foreign language to you, see if translating it into the one-dimensional linear motion of Chapter 2 helps. For example, if you are to find, say, an *angular distance*, temporarily delete the word *angular* and see if you can work the problem with the Chapter 2 notation and ideas.

## Rotational Variables

We wish to examine the rotation of a rigid body about a fixed axis. A **rigid body** is a body that can rotate with all its parts locked together and without any change in its shape. A **fixed axis** means that the rotation occurs about an axis that does not move. Thus, we shall not examine an object like the Sun, because the parts of the Sun (a ball of gas) are not locked together. We also shall not examine an object like a bowling ball rolling along a lane, because the ball rotates about a moving axis (the ball's motion is a mixture of rotation and translation).

Figure 10.1.2 shows a rigid body of arbitrary shape in rotation about a fixed axis, called the **axis of rotation** or the **rotation axis**. In pure rotation (*angular motion*), every point of the body moves in a circle whose center lies on the axis of rotation, and every point moves through the same angle during a particular time interval. In pure translation (*linear motion*), every point of the body moves in a straight line, and every point moves through the same *linear distance* during a particular time interval.

We deal now—one at a time—with the angular equivalents of the linear quantities position, displacement, velocity, and acceleration.



**Figure 10.1.2** A rigid body of arbitrary shape in pure rotation about the  $z$  axis of a coordinate system. The position of the *reference line* with respect to the rigid body is arbitrary, but it is perpendicular to the rotation axis. It is fixed in the body and rotates with the body.

## Angular Position

Figure 10.1.2 shows a *reference line*, fixed in the body, perpendicular to the rotation axis and rotating with the body. The **angular position** of this line is the angle of the line relative to a fixed direction, which we take as the **zero angular position**. In Fig. 10.1.3, the angular position  $\theta$  is measured relative to the positive direction of the  $x$  axis. From geometry, we know that  $\theta$  is given by

$$\theta = \frac{s}{r} \quad (\text{radian measure}). \quad (10.1.1)$$

Here  $s$  is the length of a circular arc that extends from the  $x$  axis (the zero angular position) to the reference line, and  $r$  is the radius of the circle.

An angle defined in this way is measured in **radians** (rad) rather than in revolutions (rev) or degrees. The radian, being the ratio of two lengths, is a pure number and thus has no dimension. Because the circumference of a circle of radius  $r$  is  $2\pi r$ , there are  $2\pi$  radians in a complete circle:

$$1 \text{ rev} = 360^\circ = \frac{2\pi r}{r} = 2\pi \text{ rad}, \quad (10.1.2)$$

and thus

$$1 \text{ rad} = 57.3^\circ = 0.159 \text{ rev}. \quad (10.1.3)$$

We do *not* reset  $\theta$  to zero with each complete rotation of the reference line about the rotation axis. If the reference line completes two revolutions from the zero angular position, then the angular position  $\theta$  of the line is  $\theta = 4\pi$  rad.

For pure translation along an  $x$  axis, we can know all there is to know about a moving body if we know  $x(t)$ , its position as a function of time. Similarly, for pure rotation, we can know all there is to know about a rotating body if we know  $\theta(t)$ , the angular position of the body's reference line as a function of time.

## Angular Displacement

If the body of Fig. 10.1.2 rotates about the rotation axis as in Fig. 10.1.4, changing the angular position of the reference line from  $\theta_1$  to  $\theta_2$ , the body undergoes an **angular displacement**  $\Delta\theta$  given by

$$\Delta\theta = \theta_2 - \theta_1. \quad (10.1.4)$$

This definition of angular displacement holds not only for the rigid body as a whole but also for *every particle within that body*.

**Clocks Are Negative.** If a body is in translational motion along an  $x$  axis, its displacement  $\Delta x$  is either positive or negative, depending on whether the body is moving in the positive or negative direction of the axis. Similarly, the angular displacement  $\Delta\theta$  of a rotating body is either positive or negative, according to the following rule:

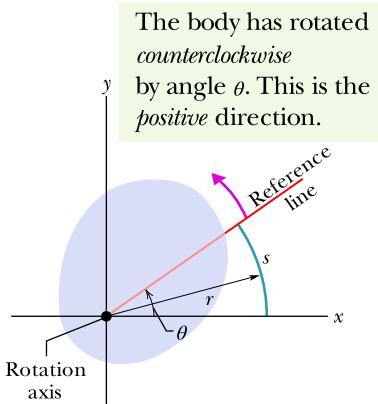


An angular displacement in the counterclockwise direction is positive, and one in the clockwise direction is negative.

The phrase “*clocks are negative*” can help you remember this rule (they certainly are negative when their alarms sound off early in the morning).

### Checkpoint 10.1.1

A disk can rotate about its central axis like a merry-go-round. Which of the following pairs of values for its initial and final angular positions, respectively, give a negative angular displacement: (a)  $-3 \text{ rad}$ ,  $+5 \text{ rad}$ , (b)  $-3 \text{ rad}$ ,  $-7 \text{ rad}$ , (c)  $7 \text{ rad}$ ,  $-3 \text{ rad}$ ?



**Figure 10.1.3** The rotating rigid body of Fig. 10.1.2 in cross section, viewed from above. The plane of the cross section is perpendicular to the rotation axis, which now extends out of the page, toward you. In this position of the body, the reference line makes an angle  $\theta$  with the  $x$  axis.

## Angular Velocity

Suppose that our rotating body is at angular position  $\theta_1$  at time  $t_1$  and at angular position  $\theta_2$  at time  $t_2$  as in Fig. 10.1.4. We define the **average angular velocity** of the body in the time interval  $\Delta t$  from  $t_1$  to  $t_2$  to be

$$\omega_{\text{avg}} = \frac{\theta_2 - \theta_1}{t_2 - t_1} = \frac{\Delta\theta}{\Delta t}, \quad (10.1.5)$$

where  $\Delta\theta$  is the angular displacement during  $\Delta t$  ( $\omega$  is the lowercase omega).

The **(instantaneous) angular velocity**  $\omega$ , with which we shall be most concerned, is the limit of the ratio in Eq. 10.1.5 as  $\Delta t$  approaches zero. Thus,

$$\omega = \lim_{\Delta t \rightarrow 0} \frac{\Delta\theta}{\Delta t} = \frac{d\theta}{dt}. \quad (10.1.6)$$

If we know  $\theta(t)$ , we can find the angular velocity  $\omega$  by differentiation.

Equations 10.1.5 and 10.1.6 hold not only for the rotating rigid body as a whole but also for *every particle of that body* because the particles are all locked together. The unit of angular velocity is commonly the radian per second (rad/s) or the revolution per second (rev/s). Another measure of angular velocity was used during at least the first three decades of rock: Music was produced by vinyl (phonograph) records that were played on turntables at “ $33\frac{1}{3}$  rpm” or “45 rpm,” meaning at  $33\frac{1}{3}$  rev/min or 45 rev/min.

If a particle moves in translation along an  $x$  axis, its linear velocity  $v$  is either positive or negative, depending on its direction along the axis. Similarly, the angular velocity  $\omega$  of a rotating rigid body is either positive or negative, depending on whether the body is rotating counterclockwise (positive) or clockwise (negative). (“Clocks are negative” still works.) The magnitude of an angular velocity is called the **angular speed**, which is also represented with  $\omega$ .

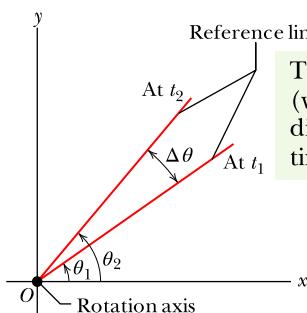
## Angular Acceleration

If the angular velocity of a rotating body is not constant, then the body has an angular acceleration. Let  $\omega_2$  and  $\omega_1$  be its angular velocities at times  $t_2$  and  $t_1$ , respectively. The **average angular acceleration** of the rotating body in the interval from  $t_1$  to  $t_2$  is defined as

$$\alpha_{\text{avg}} = \frac{\omega_2 - \omega_1}{t_2 - t_1} = \frac{\Delta\omega}{\Delta t}, \quad (10.1.7)$$

in which  $\Delta\omega$  is the change in the angular velocity that occurs during the time interval  $\Delta t$ . The **(instantaneous) angular acceleration**  $\alpha$ , with which we shall be most concerned, is the limit of this quantity as  $\Delta t$  approaches zero. Thus,

$$\alpha = \lim_{\Delta t \rightarrow 0} \frac{\Delta\omega}{\Delta t} = \frac{d\omega}{dt}. \quad (10.1.8)$$



This change in the angle of the reference line (which is part of the body) is equal to the angular displacement of the body itself during this time interval.

**Figure 10.1.4** The reference line of the rigid body of Figs. 10.1.2 and 10.1.3 is at angular position  $\theta_1$  at time  $t_1$  and at angular position  $\theta_2$  at a later time  $t_2$ . The quantity  $\Delta\theta (= \theta_2 - \theta_1)$  is the angular displacement that occurs during the interval  $\Delta t (= t_2 - t_1)$ . The body itself is not shown.

As the name suggests, this is the angular acceleration of the body at a given instant. Equations 10.1.7 and 10.1.8 also hold for *every particle of that body*. The unit of angular acceleration is commonly the radian per second-squared ( $\text{rad/s}^2$ ) or the revolution per second-squared ( $\text{rev/s}^2$ ).

### Sample Problem 10.1.1 Angular velocity derived from angular position

The disk in Fig. 10.1.5a is rotating about its central axis like a merry-go-round. The angular position  $\theta(t)$  of a reference line on the disk is given by

$$\theta = -1.00 - 0.600t + 0.250t^2 \quad (10.1.9)$$

with  $t$  in seconds,  $\theta$  in radians, and the zero angular position as indicated in the figure. (If you like, you can translate all this into Chapter 2 notation by momentarily dropping the word “angular” from “angular position” and replacing the symbol  $\theta$  with the symbol  $x$ . What you then have is an equation that gives the position as a function of time, for the one-dimensional motion of Chapter 2.)

(a) Graph the angular position of the disk versus time from  $t = -3.0$  s to  $t = 5.4$  s. Sketch the disk and its angular position reference line at  $t = -2.0$  s, 0 s, and 4.0 s, and when the curve crosses the  $t$  axis.

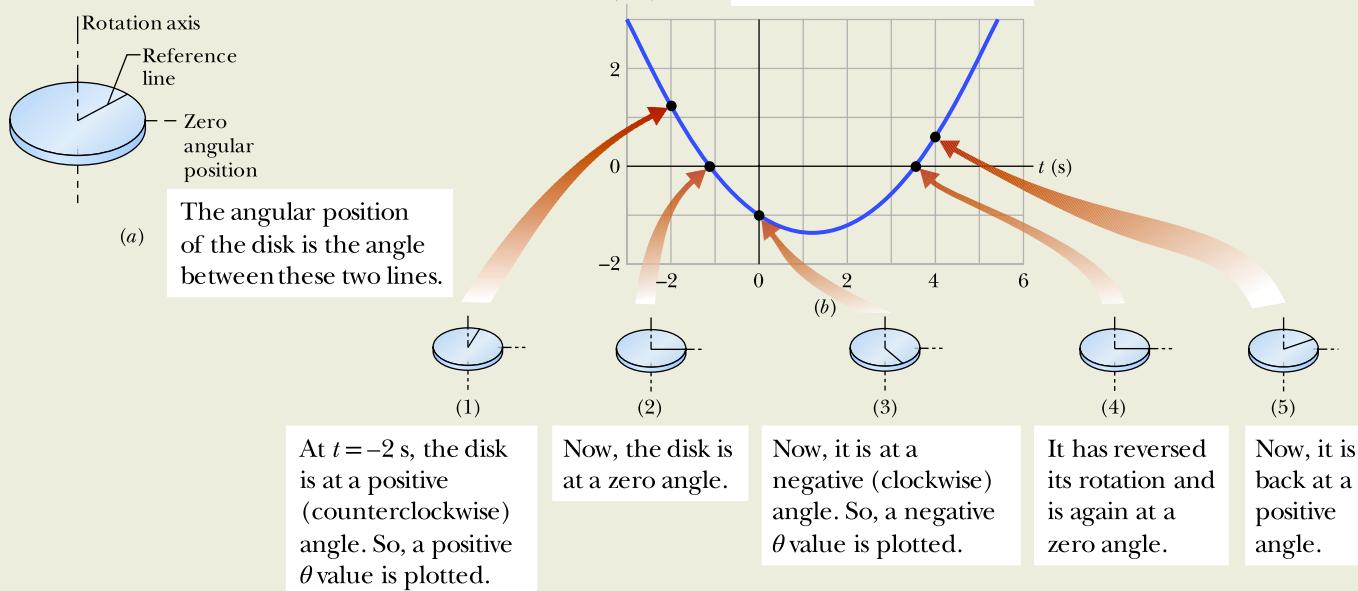
### KEY IDEA

The angular position of the disk is the angular position  $\theta(t)$  of its reference line, which is given by Eq. 10.1.9 as a function of time  $t$ . So we graph Eq. 10.1.9; the result is shown in Fig. 10.1.5b.

**Calculations:** To sketch the disk and its reference line at a particular time, we need to determine  $\theta$  for that time. To do so, we substitute the time into Eq. 10.1.9. For  $t = -2.0$  s, we get

$$\begin{aligned}\theta &= -1.00 - (0.600)(-2.0) + (0.250)(-2.0)^2 \\ &= 1.2 \text{ rad} = 1.2 \text{ rad} \frac{360^\circ}{2\pi \text{ rad}} = 69^\circ.\end{aligned}$$

This means that at  $t = -2.0$  s the reference line on the disk is rotated counterclockwise from the zero position by



**Figure 10.1.5** (a) A rotating disk. (b) A plot of the disk’s angular position  $\theta(t)$ . Five sketches indicate the angular position of the reference line on the disk for five points on the curve. (c) A plot of the disk’s angular velocity  $\omega(t)$ . Positive values of  $\omega$  correspond to counterclockwise rotation, and negative values to clockwise rotation.

angle  $1.2 \text{ rad} = 69^\circ$  (counterclockwise because  $\theta$  is positive). Sketch 1 in Fig. 10.1.5b shows this position of the reference line.

Similarly, for  $t = 0$ , we find  $\theta = -1.00 \text{ rad} = -57^\circ$ , which means that the reference line is rotated clockwise from the zero angular position by  $1.0 \text{ rad}$ , or  $57^\circ$ , as shown in sketch 3. For  $t = 4.0 \text{ s}$ , we find  $\theta = 0.60 \text{ rad} = 34^\circ$  (sketch 5). Drawing sketches for when the curve crosses the  $t$  axis is easy, because then  $\theta = 0$  and the reference line is momentarily aligned with the zero angular position (sketches 2 and 4).

(b) At what time  $t_{\min}$  does  $\theta(t)$  reach the minimum value shown in Fig. 10.1.5b? What is that minimum value?

### KEY IDEA

To find the extreme value (here the minimum) of a function, we take the first derivative of the function and set the result to zero.

**Calculations:** The first derivative of  $\theta(t)$  is

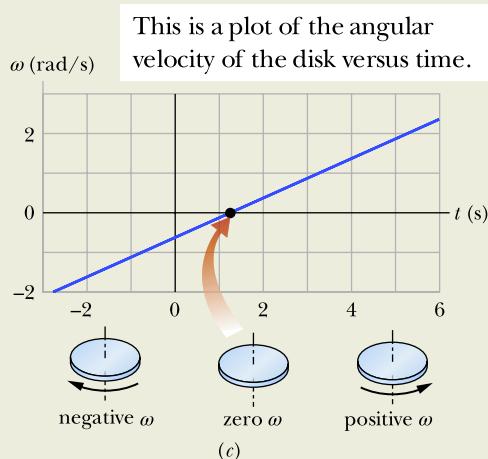
$$\frac{d\theta}{dt} = -0.600 + 0.500t. \quad (10.1.10)$$

Setting this to zero and solving for  $t$  give us the time at which  $\theta(t)$  is minimum:

$$t_{\min} = 1.20 \text{ s}. \quad (\text{Answer})$$

To get the minimum value of  $\theta$ , we next substitute  $t_{\min}$  into Eq. 10.1.9, finding

$$\theta = -1.36 \text{ rad} \approx -77.9^\circ. \quad (\text{Answer})$$



The angular velocity is initially negative and slowing, then momentarily zero during reversal, and then positive and increasing.

This *minimum* of  $\theta(t)$  (the bottom of the curve in Fig. 10.1.5b) corresponds to the *maximum clockwise* rotation of the disk from the zero angular position, somewhat more than is shown in sketch 3.

(c) Graph the angular velocity  $\omega$  of the disk versus time from  $t = -3.0 \text{ s}$  to  $t = 6.0 \text{ s}$ . Sketch the disk and indicate the direction of turning and the sign of  $\omega$  at  $t = -2.0 \text{ s}$ ,  $4.0 \text{ s}$ , and  $t_{\min}$ .

### KEY IDEA

From Eq. 10.1.6, the angular velocity  $\omega$  is equal to  $d\theta/dt$  as given in Eq. 10.1.10. So, we have

$$\omega = -0.600 + 0.500t. \quad (10.1.11)$$

The graph of this function  $\omega(t)$  is shown in Fig. 10.1.5c. Because the function is linear, the plot is a straight line. The slope is  $0.500 \text{ rad/s}^2$  and the intercept with the vertical axis (not shown) is  $-0.600 \text{ rad/s}$ .

**Calculations:** To sketch the disk at  $t = -2.0 \text{ s}$ , we substitute that value into Eq. 10.1.11, obtaining

$$\omega = -1.6 \text{ rad/s}. \quad (\text{Answer})$$

The minus sign here tells us that at  $t = -2.0 \text{ s}$ , the disk is turning clockwise (as indicated by the left-hand sketch in Fig. 10.1.5c).

Substituting  $t = 4.0 \text{ s}$  into Eq. 10.1.11 gives us

$$\omega = 1.4 \text{ rad/s}. \quad (\text{Answer})$$

The implied plus sign tells us that now the disk is turning counterclockwise (the right-hand sketch in Fig. 10.1.5c).

For  $t_{\min}$ , we already know that  $d\theta/dt = 0$ . So, we must also have  $\omega = 0$ . That is, the disk momentarily stops when the reference line reaches the minimum value of  $\theta$  in Fig. 10.1.5b, as suggested by the center sketch in Fig. 10.1.5c. On the graph of  $\omega$  versus  $t$  in Fig. 10.1.5c, this momentary stop is the zero point where the plot changes from the negative clockwise motion to the positive counterclockwise motion.

(d) Use the results in parts (a) through (c) to describe the motion of the disk from  $t = -3.0 \text{ s}$  to  $t = 6.0 \text{ s}$ .

**Description:** When we first observe the disk at  $t = -3.0 \text{ s}$ , it has a positive angular position and is turning clockwise but slowing. It stops at angular position  $\theta = -1.36 \text{ rad}$  and then begins to turn counterclockwise, with its angular position eventually becoming positive again.

### Sample Problem 10.1.2 Angular velocity derived from angular acceleration

A child's top is spun with angular acceleration

$$\alpha = 5t^3 - 4t,$$

with  $t$  in seconds and  $\alpha$  in radians per second-squared. At  $t = 0$ , the top has angular velocity 5 rad/s, and a reference line on it is at angular position  $\theta = 2$  rad.

(a) Obtain an expression for the angular velocity  $\omega(t)$  of the top. That is, find an expression that explicitly indicates how the angular velocity depends on time. (We can tell that there *is* such a dependence because the top is undergoing an angular acceleration, which means that its angular velocity *is* changing.)

#### KEY IDEA

By definition,  $\alpha(t)$  is the derivative of  $\omega(t)$  with respect to time. Thus, we can find  $\omega(t)$  by integrating  $\alpha(t)$  with respect to time.

**Calculations:** Equation 10.1.8 tells us

$$d\omega = \alpha dt,$$

so

$$\int d\omega = \int \alpha dt.$$

From this we find

$$\omega = \int (5t^3 - 4t) dt = \frac{5}{4}t^4 - \frac{4}{2}t^2 + C.$$

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## Are Angular Quantities Vectors?

We can describe the position, velocity, and acceleration of a single particle by means of vectors. If the particle is confined to a straight line, however, we do not really need vector notation. Such a particle has only two directions available to it, and we can indicate these directions with plus and minus signs.

In the same way, a rigid body rotating about a fixed axis can rotate only clockwise or counterclockwise as seen along the axis, and again we can select between the two directions by means of plus and minus signs. The question arises: "Can we treat the angular displacement, velocity, and acceleration of a rotating body as vectors?" The answer is a qualified "yes" (see the caution below, in connection with angular displacements).

**Angular Velocities.** Consider the angular velocity. Figure 10.1.6a shows a vinyl record rotating on a turntable. The record has a constant angular speed  $\omega$  ( $= 33\frac{1}{3}$  rev/min) in the clockwise direction. We can represent its angular velocity as a vector  $\vec{\omega}$  pointing along the axis of rotation, as in Fig. 10.1.6b. Here's how: We choose the length of this vector according to some convenient scale, for example, with 1 cm corresponding to 10 rev/min. Then we establish a direction for the vector  $\vec{\omega}$  by using a **right-hand rule**, as Fig. 10.1.6c shows: Curl

To evaluate the constant of integration  $C$ , we note that  $\omega = 5$  rad/s at  $t = 0$ . Substituting these values in our expression for  $\omega$  yields

$$5 \text{ rad/s} = 0 - 0 + C,$$

so  $C = 5$  rad/s. Then

$$\omega = \frac{5}{4}t^4 - 2t^2 + 5. \quad (\text{Answer})$$

(b) Obtain an expression for the angular position  $\theta(t)$  of the top.

#### KEY IDEA

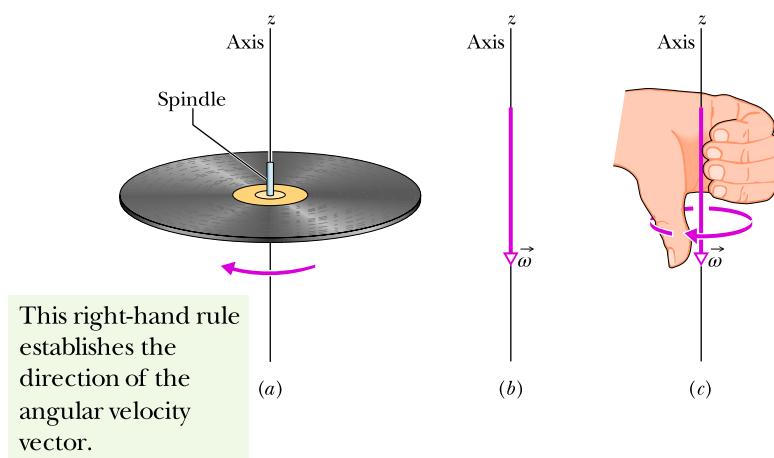
By definition,  $\omega(t)$  is the derivative of  $\theta(t)$  with respect to time. Therefore, we can find  $\theta(t)$  by integrating  $\omega(t)$  with respect to time.

**Calculations:** Since Eq. 10.1.6 tells us that

we can write

$$\begin{aligned} \theta &= \int \omega dt = \int \left( \frac{5}{4}t^4 - 2t^2 + 5 \right) dt \\ &= \frac{1}{4}t^5 - \frac{2}{3}t^3 + 5t + C' \\ &= \frac{1}{4}t^5 - \frac{2}{3}t^3 + 5t + 2, \end{aligned} \quad (\text{Answer})$$

where  $C'$  has been evaluated by noting that  $\theta = 2$  rad at  $t = 0$ .



**Figure 10.1.6** (a) A record rotating about a vertical axis that coincides with the axis of the spindle. (b) The angular velocity of the rotating record can be represented by the vector  $\vec{\omega}$ , lying along the axis and pointing down, as shown. (c) We establish the direction of the angular velocity vector as downward by using a right-hand rule. When the fingers of the right hand curl around the record and point the way it is moving, the extended thumb points in the direction of  $\vec{\omega}$ .

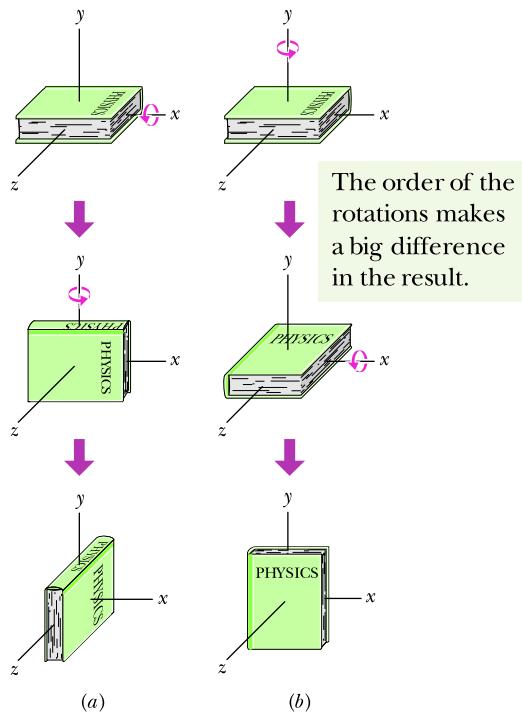
your right hand about the rotating record, your fingers pointing *in the direction of rotation*. Your extended thumb will then point in the direction of the angular velocity vector. If the record were to rotate in the opposite sense, the right-hand rule would tell you that the angular velocity vector then points in the opposite direction.

It is not easy to get used to representing angular quantities as vectors. We instinctively expect that something should be moving *along* the direction of a vector. That is not the case here. Instead, something (the rigid body) is rotating *around* the direction of the vector. In the world of pure rotation, a vector defines an axis of rotation, not a direction in which something moves. Nonetheless, the vector also defines the motion. Furthermore, it obeys all the rules for vector manipulation discussed in Chapter 3. The angular acceleration  $\vec{\alpha}$  is another vector, and it too obeys those rules.

In this chapter we consider only rotations that are about a fixed axis. For such situations, we need not consider vectors—we can represent angular velocity with  $\omega$  and angular acceleration with  $\alpha$ , and we can indicate direction with an implied plus sign for counterclockwise or an explicit minus sign for clockwise.

**Angular Displacements.** Now for the caution: Angular displacements (unless they are very small) *cannot* be treated as vectors. Why not? We can certainly give them both magnitude and direction, as we did for the angular velocity vector in Fig. 10.1.6. However, to be represented as a vector, a quantity must *also* obey the rules of vector addition, one of which says that if you add two vectors, the order in which you add them does not matter. Angular displacements fail this test.

Figure 10.1.7 gives an example. An initially horizontal book is given two  $90^\circ$  angular displacements, first in the order of Fig. 10.1.7a and then in the order of Fig. 10.1.7b. Although the two angular displacements are identical, their order is not, and the book ends up with different orientations. Here's another example. Hold your right arm downward, palm toward your thigh. Keeping your wrist



**Figure 10.1.7** (a) From its initial position, at the top, the book is given two successive  $90^\circ$  rotations, first about the (horizontal)  $x$  axis and then about the (vertical)  $y$  axis. (b) The book is given the same rotations, but in the reverse order.

rigid, (1) lift the arm forward until it is horizontal, (2) move it horizontally until it points toward the right, and (3) then bring it down to your side. Your palm faces forward. If you start over, but reverse the steps, which way does your palm end up facing? From either example, we must conclude that the addition of two angular displacements depends on their order and they cannot be vectors.

FCP

## 10.2 ROTATION WITH CONSTANT ANGULAR ACCELERATION

### Learning Objective

After reading this module, you should be able to . . .

**10.2.1** For constant angular acceleration, apply the relationships between angular position, angular

displacement, angular velocity, angular acceleration, and elapsed time (Table 10.2.1).

### Key Idea

- Constant angular acceleration ( $\alpha = \text{constant}$ ) is an important special case of rotational motion. The appropriate kinematic equations are

$$\begin{aligned}\omega &= \omega_0 + \alpha t, \\ \theta - \theta_0 &= \omega_0 t + \frac{1}{2}\alpha t^2, \\ \omega^2 &= \omega_0^2 + 2\alpha(\theta - \theta_0), \\ \theta - \theta_0 &= \frac{1}{2}(\omega_0 + \omega)t, \\ \theta - \theta_0 &= \omega t - \frac{1}{2}\alpha t^2.\end{aligned}$$

### Rotation with Constant Angular Acceleration

In pure translation, motion with a *constant linear acceleration* (for example, that of a falling body) is an important special case. In Table 2.4.1, we displayed a series of equations that hold for such motion.

In pure rotation, the case of *constant angular acceleration* is also important, and a parallel set of equations holds for this case also. We shall not derive them here, but simply write them from the corresponding linear equations, substituting equivalent angular quantities for the linear ones. This is done in Table 10.2.1, which lists both sets of equations (Eqs. 2.4.1 and 2.4.5 to 2.4.8; 10.2.1 to 10.2.5).

Recall that Eqs. 2.4.1 and 2.4.5 are basic equations for constant linear acceleration—the other equations in the Linear list can be derived from them. Similarly, Eqs. 10.2.1 and 10.2.2 are the basic equations for constant angular acceleration, and the other equations in the Angular list can be derived from them. To solve a simple problem involving constant angular acceleration, you can usually use an equation from the Angular list (*if you have the list*). Choose an equation for which the only unknown variable will be the variable requested in the problem. A better plan is to remember only Eqs. 10.2.1 and 10.2.2, and then solve them as simultaneous equations whenever needed.

#### Checkpoint 10.2.1

In four situations, a rotating body has angular position  $\theta(t)$  given by (a)  $\theta = 3t - 4$ , (b)  $\theta = -5t^3 + 4t^2 + 6$ , (c)  $\theta = 2/t^2 - 4/t$ , and (d)  $\theta = 5t^2 - 3$ . To which situations do the angular equations of Table 10.2.1 apply?

**Table 10.2.1** Equations of Motion for Constant Linear Acceleration and for Constant Angular Acceleration

Equation Number	Linear Equation	Missing Variable	Angular Equation	Equation Number	
(2.4.1)	$v = v_0 + at$	$x - x_0$	$\theta - \theta_0$	$\omega = \omega_0 + at$	(10.2.1)
(2.4.5)	$x - x_0 = v_0 t + \frac{1}{2}at^2$	$v$	$\omega$	$\theta - \theta_0 = \omega_0 t + \frac{1}{2}\alpha t^2$	(10.2.2)
(2.4.6)	$v^2 = v_0^2 + 2a(x - x_0)$	$t$	$t$	$\omega^2 = \omega_0^2 + 2\alpha(\theta - \theta_0)$	(10.2.3)
(2.4.7)	$x - x_0 = \frac{1}{2}(v_0 + v)t$	$a$	$\alpha$	$\theta - \theta_0 = \frac{1}{2}(\omega_0 + \omega)t$	(10.2.4)
(2.4.8)	$x - x_0 = vt - \frac{1}{2}at^2$	$v_0$	$\omega_0$	$\theta - \theta_0 = \omega t - \frac{1}{2}\alpha t^2$	(10.2.5)

### Sample Problem 10.2.1 Constant angular acceleration, grindstone

A grindstone (Fig. 10.2.1) rotates at constant angular acceleration  $\alpha = 0.35 \text{ rad/s}^2$ . At time  $t = 0$ , it has an angular velocity of  $\omega_0 = -4.6 \text{ rad/s}$  and a reference line on it is horizontal, at the angular position  $\theta_0 = 0$ .

- (a) At what time after  $t = 0$  is the reference line at the angular position  $\theta = 5.0 \text{ rev}$ ?

#### KEY IDEA

The angular acceleration is constant, so we can use the rotation equations of Table 10.2.1. We choose Eq. 10.2.2,

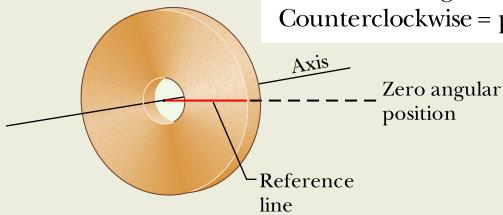
$$\theta - \theta_0 = \omega_0 t + \frac{1}{2}\alpha t^2,$$

because the only unknown variable it contains is the desired time  $t$ .

**Calculations:** Substituting known values and setting  $\theta_0 = 0$  and  $\theta = 5.0 \text{ rev} = 10\pi \text{ rad}$  give us

$$10\pi \text{ rad} = (-4.6 \text{ rad/s})t + \frac{1}{2}(0.35 \text{ rad/s}^2)t^2.$$

We measure rotation by using this reference line.  
Clockwise = negative  
Counterclockwise = positive



**Figure 10.2.1** A grindstone. At  $t = 0$  the reference line (which we imagine to be marked on the stone) is horizontal.

(We converted 5.0 rev to  $10\pi \text{ rad}$  to keep the units consistent.) Solving this quadratic equation for  $t$ , we find

$$t = 32 \text{ s.} \quad (\text{Answer})$$

Now notice something a bit strange. We first see the wheel when it is rotating in the negative direction and through the  $\theta = 0$  orientation. Yet, we just found out that 32 s later it is at the positive orientation of  $\theta = 5.0 \text{ rev}$ . What happened in that time interval so that it could be at a positive orientation?

- (b) Describe the grindstone's rotation between  $t = 0$  and  $t = 32 \text{ s}$ .

**Description:** The wheel is initially rotating in the negative (clockwise) direction with angular velocity  $\omega_0 = -4.6 \text{ rad/s}$ , but its angular acceleration  $\alpha$  is positive. This initial opposition of the signs of angular velocity and angular acceleration means that the wheel slows in its rotation in the negative direction, stops, and then reverses to rotate in the positive direction. After the reference line comes back through its initial orientation of  $\theta = 0$ , the wheel turns an additional 5.0 rev by time  $t = 32 \text{ s}$ .

- (c) At what time  $t$  does the grindstone momentarily stop?

**Calculation:** We again go to the table of equations for constant angular acceleration, and again we need an equation that contains only the desired unknown variable  $t$ . However, now the equation must also contain the variable  $\omega$ , so that we can set it to 0 and then solve for the corresponding time  $t$ . We choose Eq. 10.2.1, which yields

$$t = \frac{\omega - \omega_0}{\alpha} = \frac{0 - (-4.6 \text{ rad/s})}{0.35 \text{ rad/s}^2} = 13 \text{ s.} \quad (\text{Answer})$$

### Sample Problem 10.2.2 Constant angular acceleration, riding a Rotor

While you are operating a Rotor (a large, vertical, rotating cylinder found in amusement parks), you spot a passenger in acute distress and decrease the angular velocity of the cylinder from  $3.40 \text{ rad/s}$  to  $2.00 \text{ rad/s}$  in  $20.0 \text{ rev}$ , at constant

angular acceleration. (The passenger is obviously more of a “translation person” than a “rotation person.”) **FCP**

- (a) What is the constant angular acceleration during this decrease in angular speed?

**KEY IDEA**

Because the cylinder's angular acceleration is constant, we can relate it to the angular velocity and angular displacement via the basic equations for constant angular acceleration (Eqs. 10.2.1 and 10.2.2).

**Calculations:** Let's first do a quick check to see if we can solve the basic equations. The initial angular velocity is  $\omega_0 = 3.40 \text{ rad/s}$ , the angular displacement is  $\theta - \theta_0 = 20.0 \text{ rev}$ , and the angular velocity at the end of that displacement is  $\omega = 2.00 \text{ rad/s}$ . In addition to the angular acceleration  $\alpha$  that we want, both basic equations also contain time  $t$ , which we do not necessarily want.

To eliminate the unknown  $t$ , we use Eq. 10.2.1 to write

$$t = \frac{\omega - \omega_0}{\alpha},$$

which we then substitute into Eq. 10.2.2 to write

$$\theta - \theta_0 = \omega_0 \left( \frac{\omega - \omega_0}{\alpha} \right) + \frac{1}{2} \alpha \left( \frac{\omega - \omega_0}{\alpha} \right)^2.$$

Solving for  $\alpha$ , substituting known data, and converting 20 rev to 125.7 rad, we find

$$\begin{aligned} \alpha &= \frac{\omega^2 - \omega_0^2}{2(\theta - \theta_0)} = \frac{(2.00 \text{ rad/s})^2 - (3.40 \text{ rad/s})^2}{2(125.7 \text{ rad})} \\ &= -0.0301 \text{ rad/s}^2. \end{aligned} \quad (\text{Answer})$$

(b) How much time did the speed decrease take?

**Calculation:** Now that we know  $\alpha$ , we can use Eq. 10.2.1 to solve for  $t$ :

$$\begin{aligned} t &= \frac{\omega - \omega_0}{\alpha} = \frac{2.00 \text{ rad/s} - 3.40 \text{ rad/s}}{-0.0301 \text{ rad/s}^2} \\ &= 46.5 \text{ s}. \end{aligned} \quad (\text{Answer})$$

**WileyPLUS** Additional examples, video, and practice available at *WileyPLUS*

## 10.3 RELATING THE LINEAR AND ANGULAR VARIABLES

### Learning Objectives

After reading this module, you should be able to . . .

**10.3.1** For a rigid body rotating about a fixed axis, relate the angular variables of the body (angular position, angular velocity, and angular acceleration) and the linear variables of a particle on the body (position, velocity, and acceleration) at any given radius.

**10.3.2** Distinguish between tangential acceleration and radial acceleration, and draw a vector for each in a sketch of a particle on a body rotating about an axis, for both an increase in angular speed and a decrease.

### Key Ideas

● A point in a rigid rotating body, at a perpendicular distance  $r$  from the rotation axis, moves in a circle with radius  $r$ . If the body rotates through an angle  $\theta$ , the point moves along an arc with length  $s$  given by

$$s = \theta r \quad (\text{radian measure}),$$

where  $\theta$  is in radians.

● The linear velocity  $\vec{v}$  of the point is tangent to the circle; the point's linear speed  $v$  is given by

$$v = \omega r \quad (\text{radian measure}),$$

where  $\omega$  is the angular speed (in radians per second) of the body, and thus also the point.

● The linear acceleration  $\vec{a}$  of the point has both tangential and radial components. The tangential component is

$$a_t = \alpha r \quad (\text{radian measure}),$$

where  $\alpha$  is the magnitude of the angular acceleration (in radians per second-squared) of the body. The radial component of  $\vec{a}$  is

$$a_r = \frac{v^2}{r} = \omega^2 r \quad (\text{radian measure}).$$

● If the point moves in uniform circular motion, the period  $T$  of the motion for the point and the body is

$$T = \frac{2\pi r}{v} = \frac{2\pi}{\omega} \quad (\text{radian measure}).$$

## Relating the Linear and Angular Variables

In Module 4.5, we discussed uniform circular motion, in which a particle travels at constant linear speed  $v$  along a circle and around an axis of rotation. When a rigid body, such as a merry-go-round, rotates around an axis, each particle in the body

moves in its own circle around that axis. Since the body is rigid, all the particles make one revolution in the same amount of time; that is, they all have the same angular speed  $\omega$ .

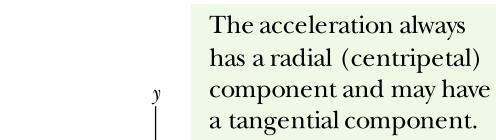
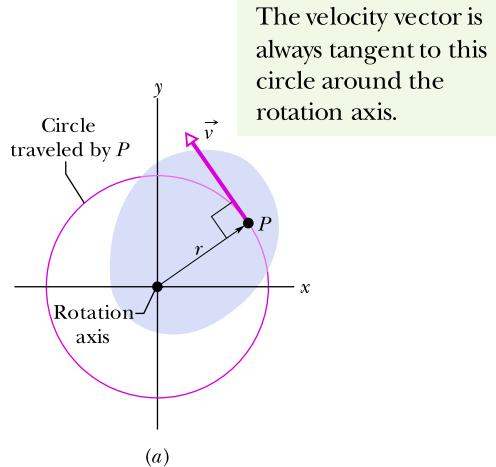
However, the farther a particle is from the axis, the greater the circumference of its circle is, and so the faster its linear speed  $v$  must be. You can notice this on a merry-go-round. You turn with the same angular speed  $\omega$  regardless of your distance from the center, but your linear speed  $v$  increases noticeably if you move to the outside edge of the merry-go-round.

We often need to relate the linear variables  $s$ ,  $v$ , and  $a$  for a particular point in a rotating body to the angular variables  $\theta$ ,  $\omega$ , and  $\alpha$  for that body. The two sets of variables are related by  $r$ , the *perpendicular distance* of the point from the rotation axis. This perpendicular distance is the distance between the point and the rotation axis, measured along a perpendicular to the axis. It is also the radius  $r$  of the circle traveled by the point around the axis of rotation.

### The Position

If a reference line on a rigid body rotates through an angle  $\theta$ , a point within the body at a position  $r$  from the rotation axis moves a distance  $s$  along a circular arc, where  $s$  is given by Eq. 10.1.1:

$$s = \theta r \quad (\text{radian measure}). \quad (10.3.1)$$



**Figure 10.3.1** The rotating rigid body of Fig. 10.1.2, shown in cross section viewed from above. Every point of the body (such as  $P$ ) moves in a circle around the rotation axis. (a) The linear velocity  $\vec{v}$  of every point is tangent to the circle in which the point moves. (b) The linear acceleration  $\vec{a}$  of the point has (in general) two components: tangential  $a_t$  and radial  $a_r$ .

This is the first of our linear-angular relations. *Caution:* The angle  $\theta$  here must be measured in radians because Eq. 10.3.1 is itself the definition of angular measure in radians.

### The Speed

Differentiating Eq. 10.3.1 with respect to time—with  $r$  held constant—leads to

$$\frac{ds}{dt} = \frac{d\theta}{dt} r.$$

However,  $ds/dt$  is the linear speed (the magnitude of the linear velocity) of the point in question, and  $d\theta/dt$  is the angular speed  $\omega$  of the rotating body. So

$$v = \omega r \quad (\text{radian measure}). \quad (10.3.2)$$

*Caution:* The angular speed  $\omega$  must be expressed in radian measure.

Equation 10.3.2 tells us that since all points within the rigid body have the same angular speed  $\omega$ , points with greater radius  $r$  have greater linear speed  $v$ . Figure 10.3.1a reminds us that the linear velocity is always tangent to the circular path of the point in question.

If the angular speed  $\omega$  of the rigid body is constant, then Eq. 10.3.2 tells us that the linear speed  $v$  of any point within it is also constant. Thus, each point within the body undergoes uniform circular motion. The period of revolution  $T$  for the motion of each point and for the rigid body itself is given by Eq. 4.5.2:

$$T = \frac{2\pi r}{v}. \quad (10.3.3)$$

This equation tells us that the time for one revolution is the distance  $2\pi r$  traveled in one revolution divided by the speed at which that distance is traveled. Substituting for  $v$  from Eq. 10.3.2 and canceling  $r$ , we find also that

$$T = \frac{2\pi}{\omega} \quad (\text{radian measure}). \quad (10.3.4)$$

This equivalent equation says that the time for one revolution is the angular distance  $2\pi$  rad traveled in one revolution divided by the angular speed (or rate) at which that angle is traveled.

### The Acceleration

Differentiating Eq. 10.3.2 with respect to time—again with  $r$  held constant—leads to

$$\frac{dv}{dt} = \frac{d\omega}{dt} r. \quad (10.3.5)$$

Here we run up against a complication. In Eq. 10.3.5,  $dv/dt$  represents only the part of the linear acceleration that is responsible for changes in the *magnitude*  $v$  of the linear velocity  $\vec{v}$ . Like  $\vec{v}$ , that part of the linear acceleration is tangent to the path of the point in question. We call it the *tangential component*  $a_t$  of the linear acceleration of the point, and we write

$$a_t = \alpha r \quad (\text{radian measure}), \quad (10.3.6)$$

where  $\alpha = d\omega/dt$ . *Caution:* The angular acceleration  $\alpha$  in Eq. 10.3.6 must be expressed in radian measure.

In addition, as Eq. 4.5.1 tells us, a particle (or point) moving in a circular path has a *radial component* of linear acceleration,  $a_r = v^2/r$  (directed radially inward), that is responsible for changes in the *direction* of the linear velocity  $\vec{v}$ . By substituting for  $v$  from Eq. 10.3.2, we can write this component as

$$a_r = \frac{v^2}{r} = \omega^2 r \quad (\text{radian measure}). \quad (10.3.7)$$

Thus, as Fig. 10.3.1b shows, the linear acceleration of a point on a rotating rigid body has, in general, two components. The radially inward component  $a_r$  (given by Eq. 10.3.7) is present whenever the angular velocity of the body is not zero. The tangential component  $a_t$  (given by Eq. 10.3.6) is present whenever the angular acceleration is not zero.

### Checkpoint 10.3.1

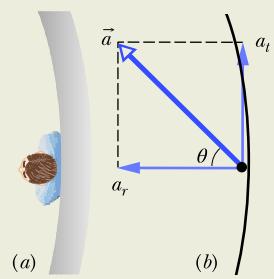
A cockroach rides the rim of a rotating merry-go-round. If the angular speed of this system (*merry-go-round + cockroach*) is constant, does the cockroach have (a) radial acceleration and (b) tangential acceleration? If  $\omega$  is decreasing, does the cockroach have (c) radial acceleration and (d) tangential acceleration?

### Sample Problem 10.3.1 Designing The Giant Ring, a large-scale amusement park ride

We are given the job of designing a large horizontal ring that will rotate around a vertical axis and that will have a radius of  $r = 33.1$  m (matching that of Beijing's The Great Observation Wheel, the largest Ferris wheel in the world). Passengers will enter through a door in the outer wall of the ring and then stand next to that wall (Fig. 10.3.2a). We decide that for the time interval  $t = 0$  to  $t = 2.30$  s, the angular position  $\theta(t)$  of a reference line on the ring will be given by

$$\theta = ct^3, \quad (10.3.8)$$

**Figure 10.3.2** (a) Overhead view of a passenger ready to ride The Giant Ring. (b) The radial and tangential acceleration components of the (full) acceleration.



with  $c = 6.39 \times 10^{-2} \text{ rad/s}^3$ . After  $t = 2.30 \text{ s}$ , the angular speed will be held constant until the end of the ride. Once the ring begins to rotate, the floor of the ring will drop away from the riders but the riders will not fall—indeed, they feel as though they are pinned to the wall. For the time  $t = 2.20 \text{ s}$ , let's determine a rider's angular speed  $\omega$ , linear speed  $v$ , angular acceleration  $\alpha$ , tangential acceleration  $a_t$ , radial acceleration  $a_r$ , and acceleration  $\vec{a}$ .

### KEY IDEAS

- (1) The angular speed  $\omega$  is given by Eq. 10.1.6 ( $\omega = d\theta/dt$ ).
- (2) The linear speed  $v$  (along the circular path) is related to the angular speed (around the rotation axis) by Eq. 10.3.2 ( $v = \omega r$ ).
- (3) The angular acceleration  $\alpha$  is given by Eq. 10.1.8 ( $\alpha = d\omega/dt$ ).
- (4) The tangential acceleration  $a_t$  (along the circular path) is related to the angular acceleration (around the rotation axis) by Eq. 10.3.6 ( $a_t = \alpha r$ ).
- (5) The radial acceleration  $a_r$  is given Eq. 10.3.7 ( $a_r = \omega^2 r$ ).
- (6) The tangential and radial accelerations are the (perpendicular) components of the (full) acceleration  $\vec{a}$ .

**Calculations:** Let's go through the steps. We first find the angular velocity by taking the time derivative of the given angular position function and then substituting the given time of  $t = 2.20 \text{ s}$ :

$$\begin{aligned}\omega &= \frac{d\theta}{dt} = \frac{d}{dt}(ct^3) = 3ct^2 & (10.3.9) \\ &= 3(6.39 \times 10^{-2} \text{ rad/s}^3)(2.20 \text{ s})^2 \\ &= 0.928 \text{ rad/s}. & (\text{Answer})\end{aligned}$$

From Eq. 10.3.2, the linear speed just then is

$$\begin{aligned}v &= \omega r = 3ct^2 r & (10.3.10) \\ &= 3(6.39 \times 10^{-2} \text{ rad/s}^3)(2.20 \text{ s})^2(33.1 \text{ m}) \\ &= 30.7 \text{ m/s}. & (\text{Answer})\end{aligned}$$

Although this is fast (111 km/h or 68.7 mi/h), such speeds are common in amusement parks and not alarming because (as mentioned in Chapter 2) your body reacts to accelerations but not to velocities. (It is an accelerometer, not a speedometer.) From Eq. 10.3.10 we see that the linear speed is increasing as the square of the time (but this increase will cut off at  $t = 2.30 \text{ s}$ ).

Next, let's tackle the angular acceleration by taking the time derivative of Eq. 10.3.9:

$$\begin{aligned}\alpha &= \frac{d\omega}{dt} = \frac{d}{dt}(3ct^2) = 6ct \\ &= 6(6.39 \times 10^{-2} \text{ rad/s}^3)(2.20 \text{ s}) = 0.843 \text{ rad/s}^2. & (\text{Answer})\end{aligned}$$

The tangential acceleration then follows from Eq. 10.3.6:

$$\begin{aligned}a_t &= \alpha r = 6ctr & (10.3.11) \\ &= 6(6.39 \times 10^{-2} \text{ rad/s}^3)(2.20 \text{ s})(33.1 \text{ m}) \\ &= 27.91 \text{ m/s}^2 \approx 27.9 \text{ m/s}^2, & (\text{Answer})\end{aligned}$$

or  $2.8g$  (which is reasonable and a bit exciting). Equation 10.3.11 tells us that the tangential acceleration is increasing with time (but it will cut off at  $t = 2.30 \text{ s}$ ). From Eq. 10.3.7, we write the radial acceleration as

$$a_r = \omega^2 r.$$

Substituting from Eq. 10.3.9 leads us to

$$\begin{aligned}a_r &= (3ct^2)^2 r = 9c^2 t^4 r & (10.3.12) \\ &= 9(6.39 \times 10^{-2} \text{ rad/s}^3)^2 (2.20 \text{ s})^4 (33.1 \text{ m}) \\ &= 28.49 \text{ m/s}^2 \approx 28.5 \text{ m/s}^2, & (\text{Answer})\end{aligned}$$

or  $2.9g$  (which is also reasonable and a bit exciting).

The radial and tangential accelerations are perpendicular to each other and form the components of the rider's acceleration  $\vec{a}$  (Fig. 10.3.2b). The magnitude of  $\vec{a}$  is given by

$$\begin{aligned}a &= \sqrt{a_r^2 + a_t^2} & (10.3.13) \\ &= \sqrt{(28.49 \text{ m/s}^2)^2 + (27.91 \text{ m/s}^2)^2} \\ &\approx 39.9 \text{ m/s}^2, & (\text{Answer})\end{aligned}$$

or  $4.1g$  (which is really exciting!). All these values are acceptable.

To find the orientation of  $\vec{a}$ , we can calculate the angle  $\theta$  shown in Fig. 10.3.2b:

$$\tan \theta = \frac{a_t}{a_r}.$$

However, instead of substituting our numerical results, let's use the algebraic results from Eqs. 10.3.11 and 10.3.12:

$$\theta = \tan^{-1} \left( \frac{6ctr}{9c^2 t^4 r} \right) = \tan^{-1} \left( \frac{2}{3ct^3} \right). \quad (10.3.14)$$

The big advantage of solving for the angle algebraically is that we can then see that the angle (1) does not depend on the ring's radius and (2) decreases as  $t$  goes from 0 to 2.20 s. That is, the acceleration vector  $\vec{a}$  swings toward being radially inward because the radial acceleration (which depends on  $t^4$ ) quickly dominates over the tangential acceleration (which depends on only  $t$ ). At our given time  $t = 2.20 \text{ s}$ , we have

$$\theta = \tan^{-1} \frac{2}{3(6.39 \times 10^{-2} \text{ rad/s}^3)(2.20 \text{ s})^3} = 44.4^\circ. \quad (\text{Answer})$$

# 10.4 KINETIC ENERGY OF ROTATION

## Learning Objectives

After reading this module, you should be able to . . .

**10.4.1** Find the rotational inertia of a particle about a point.

**10.4.2** Find the total rotational inertia of many particles moving around the same fixed axis.

**10.4.3** Calculate the rotational kinetic energy of a body in terms of its rotational inertia and its angular speed.

## Key Idea

The kinetic energy  $K$  of a rigid body rotating about a fixed axis is given by

$$K = \frac{1}{2}I\omega^2 \quad (\text{radian measure}),$$

in which  $I$  is the rotational inertia of the body, defined as

$$I = \sum m_i r_i^2$$

for a system of discrete particles.

## Kinetic Energy of Rotation

The rapidly rotating blade of a table saw certainly has kinetic energy due to that rotation. How can we express the energy? We cannot apply the familiar formula  $K = \frac{1}{2}mv^2$  to the saw as a whole because that would give us the kinetic energy only of the saw's center of mass, which is zero.

Instead, we shall treat the table saw (and any other rotating rigid body) as a collection of particles with different speeds. We can then add up the kinetic energies of all the particles to find the kinetic energy of the body as a whole. In this way we obtain, for the kinetic energy of a rotating body,

$$\begin{aligned} K &= \frac{1}{2}m_1 v_1^2 + \frac{1}{2}m_2 v_2^2 + \frac{1}{2}m_3 v_3^2 + \dots \\ &= \sum \frac{1}{2}m_i v_i^2, \end{aligned} \quad (10.4.1)$$

in which  $m_i$  is the mass of the  $i$ th particle and  $v_i$  is its speed. The sum is taken over all the particles in the body.

The problem with Eq. 10.4.1 is that  $v_i$  is not the same for all particles. We solve this problem by substituting for  $v$  from Eq. 10.3.2 ( $v = \omega r$ ), so that we have

$$K = \sum \frac{1}{2}m_i(\omega r_i)^2 = \frac{1}{2}(\sum m_i r_i^2)\omega^2, \quad (10.4.2)$$

in which  $\omega$  is the same for all particles.

The quantity in parentheses on the right side of Eq. 10.4.2 tells us how the mass of the rotating body is distributed about its axis of rotation. We call that quantity the **rotational inertia** (or **moment of inertia**)  $I$  of the body with respect to the axis of rotation. It is a constant for a particular rigid body and a particular rotation axis. (Caution: That axis must always be specified if the value of  $I$  is to be meaningful.)

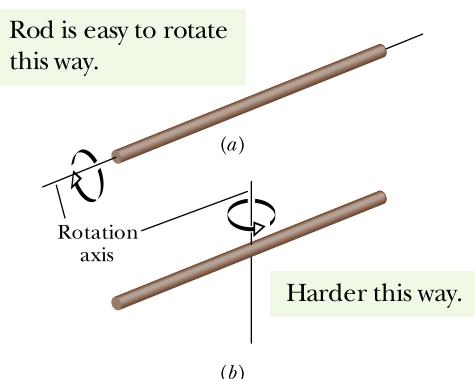
We may now write

$$I = \sum m_i r_i^2 \quad (\text{rotational inertia}) \quad (10.4.3)$$

and substitute into Eq. 10.4.2, obtaining

$$K = \frac{1}{2}I\omega^2 \quad (\text{radian measure}) \quad (10.4.4)$$

as the expression we seek. Because we have used the relation  $v = \omega r$  in deriving Eq. 10.4.4,  $\omega$  must be expressed in radian measure. The SI unit for  $I$  is the kilogram-square meter ( $\text{kg} \cdot \text{m}^2$ ).



**Figure 10.4.1** A long rod is much easier to rotate about (a) its central (longitudinal) axis than about (b) an axis through its center and perpendicular to its length. The reason for the difference is that the mass is distributed closer to the rotation axis in (a) than in (b).

**The Plan.** If we have a few particles and a specified rotation axis, we find  $mr^2$  for each particle and then add the results as in Eq. 10.4.3 to get the total rotational inertia  $I$ . If we want the total rotational kinetic energy, we can then substitute that  $I$  into Eq. 10.4.4. That is the plan for a few particles, but suppose we have a huge number of particles such as in a rod. In the next module we shall see how to handle such *continuous bodies* and do the calculation in only a few minutes.

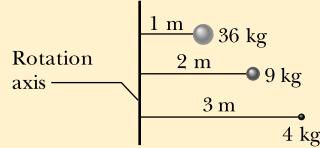
Equation 10.4.4, which gives the kinetic energy of a rigid body in pure rotation, is the angular equivalent of the formula  $K = \frac{1}{2}Mv_{\text{com}}^2$ , which gives the kinetic energy of a rigid body in pure translation. In both formulas there is a factor of  $\frac{1}{2}$ . Where mass  $M$  appears in one equation,  $I$  (which involves both mass and its distribution) appears in the other. Finally, each equation contains as a factor the square of a speed — translational or rotational as appropriate. The kinetic energies of translation and of rotation are not different kinds of energy. They are both kinetic energy, expressed in ways that are appropriate to the motion at hand.

We noted previously that the rotational inertia of a rotating body involves not only its mass but also how that mass is distributed. Here

is an example that you can literally feel. Rotate a long, fairly heavy rod (a pole, a length of lumber, or something similar), first around its central (longitudinal) axis (Fig. 10.4.1a) and then around an axis perpendicular to the rod and through the center (Fig. 10.4.1b). Both rotations involve the very same mass, but the first rotation is much easier than the second. The reason is that the mass is distributed much closer to the rotation axis in the first rotation. As a result, the rotational inertia of the rod is much smaller in Fig. 10.4.1a than in Fig. 10.4.1b. In general, smaller rotational inertia means easier rotation.

### Checkpoint 10.4.1

The figure shows three small spheres that rotate about a vertical axis. The perpendicular distance between the axis and the center of each sphere is given. Rank the three spheres according to their rotational inertia about that axis, greatest first.



## 10.5 CALCULATING THE ROTATIONAL INERTIA

### Learning Objectives

After reading this module, you should be able to . . .

**10.5.1** Determine the rotational inertia of a body if it is given in Table 10.5.1.

**10.5.2** Calculate the rotational inertia of a body by integration over the mass elements of the body.

### Key Ideas

- $I$  is the rotational inertia of the body, defined as

$$I = \sum m_i r_i^2$$

for a system of discrete particles and defined as

$$I = \int r^2 dm$$

for a body with continuously distributed mass. The  $r$  and  $r_i$  in these expressions represent the perpendicular distance from the axis of rotation to each mass element in the body, and the integration is carried out over the entire body so as to include every mass element.

**10.5.3** Apply the parallel-axis theorem for a rotation axis that is displaced from a parallel axis through the center of mass of a body.

- The parallel-axis theorem relates the rotational inertia  $I$  of a body about any axis to that of the same body about a parallel axis through the center of mass:

$$I = I_{\text{com}} + Mh^2.$$

Here  $h$  is the perpendicular distance between the two axes, and  $I_{\text{com}}$  is the rotational inertia of the body about the axis through the com. We can describe  $h$  as being the distance the actual rotation axis has been shifted from the rotation axis through the com.

## Calculating the Rotational Inertia

If a rigid body consists of a few particles, we can calculate its rotational inertia about a given rotation axis with Eq. 10.4.3 ( $I = \sum m_i r_i^2$ ); that is, we can find the product  $mr^2$  for each particle and then sum the products. (Recall that  $r$  is the perpendicular distance a particle is from the given rotation axis.)

If a rigid body consists of a great many adjacent particles (it is *continuous*, like a Frisbee), using Eq. 10.4.3 would require a computer. Thus, instead, we replace the sum in Eq. 10.4.3 with an integral and define the rotational inertia of the body as

$$I = \int r^2 dm \quad (\text{rotational inertia, continuous body}). \quad (10.5.1)$$

Table 10.5.1 gives the results of such integration for nine common body shapes and the indicated axes of rotation.

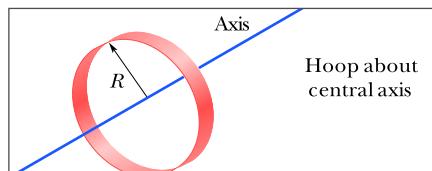
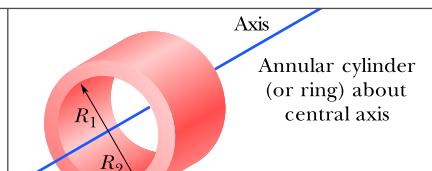
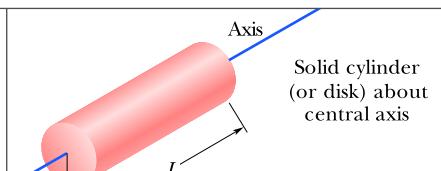
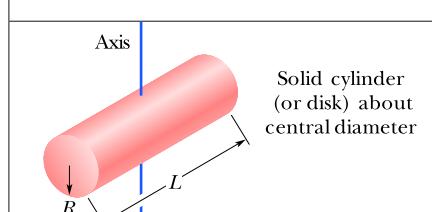
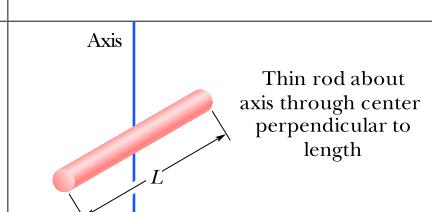
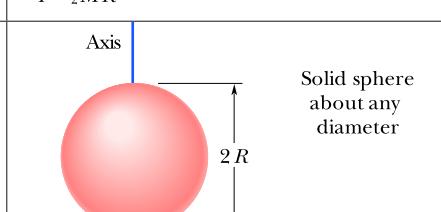
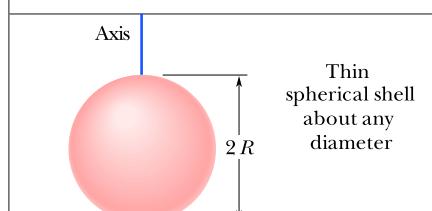
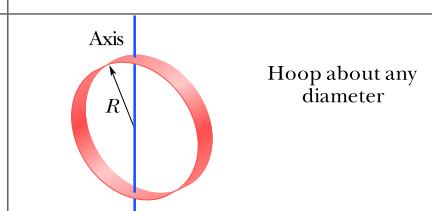
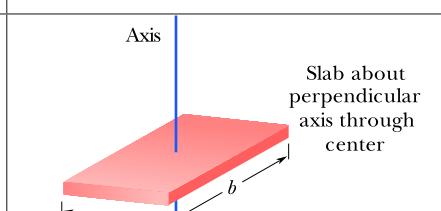
### Parallel-Axis Theorem

Suppose we want to find the rotational inertia  $I$  of a body of mass  $M$  about a given axis. In principle, we can always find  $I$  with the integration of Eq. 10.5.1. However, there is a neat shortcut if we happen to already know the rotational inertia  $I_{\text{com}}$  of the body about a *parallel* axis that extends through the body's center of mass. Let  $h$  be the perpendicular distance between the given axis and the axis through the center of mass (remember these two axes must be parallel). Then the rotational inertia  $I$  about the given axis is

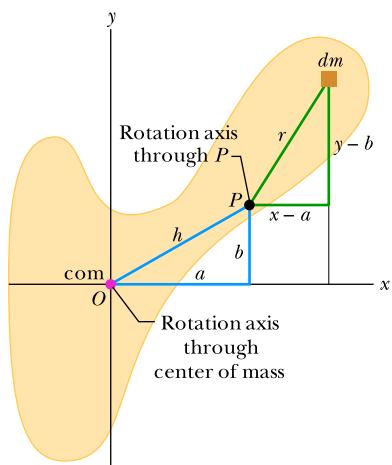
$$I = I_{\text{com}} + Mh^2 \quad (\text{parallel-axis theorem}). \quad (10.5.2)$$

Think of the distance  $h$  as being the distance we have shifted the rotation axis from being through the com. This equation is known as the **parallel-axis theorem**. We shall now prove it.

**Table 10.5.1** Some Rotational Inertias

 <p>Hoop about central axis</p> $I = MR^2$	 <p>Annular cylinder (or ring) about central axis</p> $I = \frac{1}{2}M(R_1^2 + R_2^2)$	 <p>Solid cylinder (or disk) about central axis</p> $I = \frac{1}{2}MR^2$
 <p>Solid cylinder (or disk) about central diameter</p> $I = \frac{1}{4}MR^2 + \frac{1}{12}ML^2$	 <p>Thin rod about axis through center perpendicular to length</p> $I = \frac{1}{12}ML^2$	 <p>Solid sphere about any diameter</p> $I = \frac{2}{5}MR^2$
 <p>Thin spherical shell about any diameter</p> $I = \frac{2}{3}MR^2$	 <p>Hoop about any diameter</p> $I = \frac{1}{2}MR^2$	 <p>Slab about perpendicular axis through center</p> $I = \frac{1}{12}M(a^2 + b^2)$

We need to relate the rotational inertia around the axis at  $P$  to that around the axis at the com.



**Figure 10.5.1** A rigid body in cross section, with its center of mass at  $O$ . The parallel-axis theorem (Eq. 10.5.2) relates the rotational inertia of the body about an axis through  $O$  to that about a parallel axis through a point such as  $P$ , a distance  $h$  from the body's center of mass.

### Proof of the Parallel-Axis Theorem

Let  $O$  be the center of mass of the arbitrarily shaped body shown in cross section in Fig. 10.5.1. Place the origin of the coordinates at  $O$ . Consider an axis through  $O$  perpendicular to the plane of the figure, and another axis through point  $P$  parallel to the first axis. Let the  $x$  and  $y$  coordinates of  $P$  be  $a$  and  $b$ .

Let  $dm$  be a mass element with the general coordinates  $x$  and  $y$ . The rotational inertia of the body about the axis through  $P$  is then, from Eq. 10.5.1,

$$I = \int r^2 dm = \int [(x-a)^2 + (y-b)^2] dm,$$

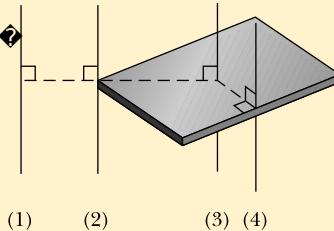
which we can rearrange as

$$I = \int (x^2 + y^2) dm - 2a \int x dm - 2b \int y dm + \int (a^2 + b^2) dm. \quad (10.5.3)$$

From the definition of the center of mass (Eq. 9.1.9), the middle two integrals of Eq. 10.5.3 give the coordinates of the center of mass (multiplied by a constant) and thus must each be zero. Because  $x^2 + y^2$  is equal to  $R^2$ , where  $R$  is the distance from  $O$  to  $dm$ , the first integral is simply  $I_{\text{com}}$ , the rotational inertia of the body about an axis through its center of mass. Inspection of Fig. 10.5.1 shows that the last term in Eq. 10.5.3 is  $Mh^2$ , where  $M$  is the body's total mass. Thus, Eq. 10.5.3 reduces to Eq. 10.5.2, which is the relation that we set out to prove.

### Checkpoint 10.5.1

The figure shows a book-like object (one side is longer than the other) and four choices of rotation axes, all perpendicular to the face of the object. Rank the choices according to the rotational inertia of the object about the axis, greatest first.



(1) (2) (3) (4)

### Sample Problem 10.5.1 Rotational inertia of a two-particle system

Figure 10.5.2a shows a rigid body consisting of two particles of mass  $m$  connected by a rod of length  $L$  and negligible mass.

- (a) What is the rotational inertia  $I_{\text{com}}$  about an axis through the center of mass, perpendicular to the rod as shown?

#### KEY IDEA

Because we have only two particles with mass, we can find the body's rotational inertia  $I_{\text{com}}$  by using Eq. 10.4.3

rather than by integration. That is, we find the rotational inertia of each particle and then just add the results.

**Calculations:** For the two particles, each at perpendicular distance  $\frac{1}{2}L$  from the rotation axis, we have

$$\begin{aligned} I &= \sum m r_i^2 = (m)(\frac{1}{2}L)^2 + (m)(\frac{1}{2}L)^2 \\ &= \frac{1}{2}mL^2. \end{aligned} \quad (\text{Answer})$$

- (b) What is the rotational inertia  $I$  of the body about an axis through the left end of the rod and parallel to the first axis (Fig. 10.5.2b)?

## KEY IDEAS

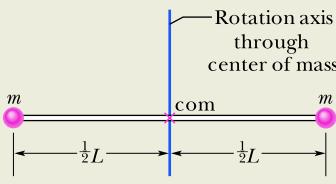
This situation is simple enough that we can find  $I$  using either of two techniques. The first is similar to the one used in part (a). The other, more powerful one is to apply the parallel-axis theorem.

**First technique:** We calculate  $I$  as in part (a), except here the perpendicular distance  $r_i$  is zero for the particle on the left and  $L$  for the particle on the right. Now Eq. 10.4.3 gives us

$$I = m(0)^2 + mL^2 = mL^2. \quad (\text{Answer})$$

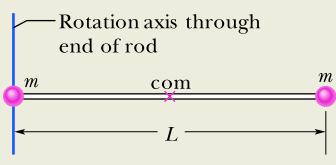
**Second technique:** Because we already know  $I_{\text{com}}$  about an axis through the center of mass and because the axis here is parallel to that “com axis,” we can apply the parallel-axis theorem (Eq. 10.5.2). We find

$$\begin{aligned} I &= I_{\text{com}} + Mh^2 = \frac{1}{2}mL^2 + (2m)\left(\frac{1}{2}L\right)^2 \\ &= mL^2. \end{aligned} \quad (\text{Answer})$$



(a)

Here the rotation axis is through the com.



(b)

Here it has been shifted from the com without changing the orientation. We can use the parallel-axis theorem.

**Figure 10.5.2** A rigid body consisting of two particles of mass  $m$  joined by a rod of negligible mass.

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### Sample Problem 10.5.2 Rotational inertia of a uniform rod, integration

Figure 10.5.3 shows a thin, uniform rod of mass  $M$  and length  $L$ , on an  $x$  axis with the origin at the rod’s center.

(a) What is the rotational inertia of the rod about the perpendicular rotation axis through the center?

## KEY IDEAS

(1) The rod consists of a huge number of particles at a great many different distances from the rotation axis. We certainly don’t want to sum their rotational inertias individually. So, we first write a general expression for the rotational inertia of a mass element  $dm$  at distance  $r$  from the rotation axis:  $r^2 dm$ . (2) Then we sum all such rotational inertias by integrating the expression (rather than adding them up one by one). From Eq. 10.5.1, we write

$$I = \int r^2 dm. \quad (10.5.4)$$

(3) Because the rod is uniform and the rotation axis is at the center, we are actually calculating the rotational inertia  $I_{\text{com}}$  about the center of mass.

**Calculations:** We want to integrate with respect to coordinate  $x$  (not mass  $m$  as indicated in the integral), so we must relate the mass  $dm$  of an element of the rod to its length  $dx$  along the rod. (Such an element is shown in Fig. 10.5.3.) Because the rod is uniform, the ratio of mass to length is the same for all the elements and for the rod as a whole. Thus, we can write

$$\frac{\text{element's mass } dm}{\text{element's length } dx} = \frac{\text{rod's mass } M}{\text{rod's length } L}$$

or

$$dm = \frac{M}{L} dx.$$

We can now substitute this result for  $dm$  and  $x$  for  $r$  in Eq. 10.5.4. Then we integrate from end to end of the rod (from  $x = -L/2$  to  $x = L/2$ ) to include all the elements. We find

$$\begin{aligned} I &= \int_{x=-L/2}^{x=+L/2} x^2 \left(\frac{M}{L}\right) dx \\ &= \frac{M}{3L} \left[x^3\right]_{-L/2}^{+L/2} = \frac{M}{3L} \left[\left(\frac{L}{2}\right)^3 - \left(-\frac{L}{2}\right)^3\right] \\ &= \frac{1}{12}ML^2. \end{aligned} \quad (\text{Answer})$$

(b) What is the rod’s rotational inertia  $I$  about a new rotation axis that is perpendicular to the rod and through the left end?

## KEY IDEAS

We can find  $I$  by shifting the origin of the  $x$  axis to the left end of the rod and then integrating from  $x = 0$  to  $x = L$ . However, here we shall use a more powerful (and easier) technique by applying the parallel-axis theorem (Eq. 10.5.2), in which we shift the rotation axis without changing its orientation.

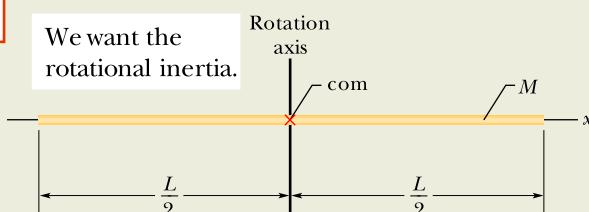
**Calculations:** If we place the axis at the rod’s end so that it is parallel to the axis through the center of mass, then we can use the parallel-axis theorem (Eq. 10.5.2). We know from part (a) that  $I_{\text{com}}$  is  $\frac{1}{12}ML^2$ . From Fig. 10.5.3, the perpendicular distance  $h$  between the new rotation axis and the center of mass is  $\frac{1}{2}L$ . Equation 10.5.2 then gives us

$$\begin{aligned} I &= I_{\text{com}} + Mh^2 = \frac{1}{12}ML^2 + (M)\left(\frac{1}{2}L\right)^2 \\ &= \frac{1}{3}ML^2. \end{aligned} \quad (\text{Answer})$$

Actually, this result holds for any axis through the left or right end that is perpendicular to the rod.



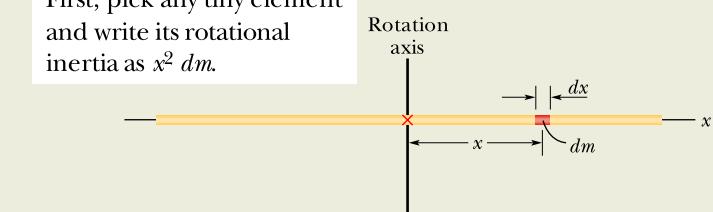
We want the rotational inertia.



**Figure 10.5.3** A uniform rod of length  $L$  and mass  $M$ . An element of mass  $dm$  and length  $dx$  is represented.

$$\begin{aligned}x &= -\frac{L}{2} \\ \text{Leftmost}\end{aligned}$$

First, pick any tiny element and write its rotational inertia as  $x^2 dm$ .



$$\begin{aligned}x &= \frac{L}{2} \\ \text{Rightmost}\end{aligned}$$

Then, using integration, add up the rotational inertias for *all* of the elements, from leftmost to rightmost.

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### Sample Problem 10.5.3 Rotational kinetic energy, spin test explosion

Large machine components that undergo prolonged, high-speed rotation are first examined for the possibility of failure in a *spin test system*. In this system, a component is *spun up* (brought up to high speed) while inside a cylindrical arrangement of lead bricks and containment liner, all within a steel shell that is closed by a lid clamped into place. If the rotation causes the component to shatter, the soft lead bricks are supposed to catch the pieces for later analysis.

In 1985, Test Devices, Inc. ([www.testdevices.com](http://www.testdevices.com)) was spin testing a sample of a solid steel rotor (a disk) of mass  $M = 272 \text{ kg}$  and radius  $R = 38.0 \text{ cm}$ . When the sample reached an angular speed  $\omega$  of 14 000 rev/min, the test engineers heard a dull thump from the test system, which was located one floor down and one room over from them. Investigating, they found that lead bricks had been thrown out in the hallway leading to the test room, a door to the room had been hurled into the adjacent parking lot, one lead brick had shot from the test site through the wall of a neighbor's kitchen, the structural beams of the test building had been damaged, the concrete floor beneath the spin chamber had been shoved downward by about 0.5 cm, and the 900 kg lid had been blown upward through the ceiling and had then crashed back onto the test equipment (Fig. 10.5.4). The exploding pieces had not penetrated the room of the test engineers only by luck.

How much energy was released in the explosion of the rotor?

**FCP**



Courtesy Test Devices, Inc.

**Figure 10.5.4** Some of the destruction caused by the explosion of a rapidly rotating steel disk.

### KEY IDEA

The released energy was equal to the rotational kinetic energy  $K$  of the rotor just as it reached the angular speed of 14 000 rev/min.

**Calculations:** We can find  $K$  with Eq. 10.4.4 ( $K = \frac{1}{2}I\omega^2$ ), but first we need an expression for the rotational inertia  $I$ . Because the rotor was a disk that rotated like a merry-go-round,  $I$  is given in Table 10.5.1c ( $I = \frac{1}{2}MR^2$ ). Thus,

$$I = \frac{1}{2}MR^2 = \frac{1}{2}(272 \text{ kg})(0.38 \text{ m})^2 = 19.64 \text{ kg} \cdot \text{m}^2.$$

The angular speed of the rotor was

$$\begin{aligned}\omega &= (14\,000 \text{ rev/min})(2\pi \text{ rad/rev})\left(\frac{1 \text{ min}}{60 \text{ s}}\right) \\ &= 1.466 \times 10^3 \text{ rad/s.}\end{aligned}$$

Then, with Eq. 10.4.4, we find the (huge) energy release:

$$\begin{aligned}K &= \frac{1}{2}I\omega^2 = \frac{1}{2}(19.64 \text{ kg} \cdot \text{m}^2)(1.466 \times 10^3 \text{ rad/s})^2 \\ &= 2.1 \times 10^7 \text{ J.}\end{aligned}\quad (\text{Answer})$$

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# 10.6 TORQUE

## Learning Objectives

After reading this module, you should be able to . . .

- 10.6.1 Identify that a torque on a body involves a force and a position vector, which extends from a rotation axis to the point where the force is applied.
- 10.6.2 Calculate the torque by using (a) the angle between the position vector and the force vector, (b) the line of action and the moment arm of the force, and (c) the force component perpendicular to the position vector.

## Key Ideas

- Torque is a turning or twisting action on a body about a rotation axis due to a force  $\vec{F}$ . If  $\vec{F}$  is exerted at a point given by the position vector  $\vec{r}$  relative to the axis, then the magnitude of the torque is

$$\tau = r F_t = r_{\perp} F = r F \sin \phi,$$

where  $F_t$  is the component of  $\vec{F}$  perpendicular to  $\vec{r}$  and  $\phi$  is the angle between  $\vec{r}$  and  $\vec{F}$ . The quantity  $r_{\perp}$  is the

- 10.6.3 Identify that a rotation axis must always be specified to calculate a torque.

- 10.6.4 Identify that a torque is assigned a positive or negative sign depending on the direction it tends to make the body rotate about a specified rotation axis: "Clocks are negative."

- 10.6.5 When more than one torque acts on a body about a rotation axis, calculate the net torque.

perpendicular distance between the rotation axis and an extended line running through the  $\vec{F}$  vector. This line is called the line of action of  $\vec{F}$ , and  $r_{\perp}$  is called the moment arm of  $\vec{F}$ . Similarly,  $r$  is the moment arm of  $F_t$ .

- The SI unit of torque is the newton-meter ( $N \cdot m$ ). A torque  $\tau$  is positive if it tends to rotate a body at rest counterclockwise and negative if it tends to rotate the body clockwise.

## Torque

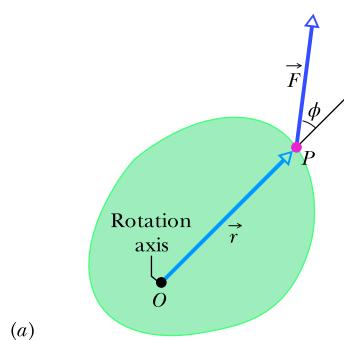
A doorknob is located as far as possible from the door's hinge line for a good reason. If you want to open a heavy door, you must certainly apply a force, but that is not enough. Where you apply that force and in what direction you push are also important. If you apply your force nearer to the hinge line than the knob, or at any angle other than  $90^\circ$  to the plane of the door, you must use a greater force than if you apply the force at the knob and perpendicular to the door's plane.

Figure 10.6.1a shows a cross section of a body that is free to rotate about an axis passing through  $O$  and perpendicular to the cross section. A force  $\vec{F}$  is applied at point  $P$ , whose position relative to  $O$  is defined by a position vector  $\vec{r}$ . The directions of vectors  $\vec{F}$  and  $\vec{r}$  make an angle  $\phi$  with each other. (For simplicity, we consider only forces that have no component parallel to the rotation axis; thus,  $\vec{F}$  is in the plane of the page.)

To determine how  $\vec{F}$  results in a rotation of the body around the rotation axis, we resolve  $\vec{F}$  into two components (Fig. 10.6.1b). One component, called the *radial component*  $F_r$ , points along  $\vec{r}$ . This component does not cause rotation, because it acts along a line that extends through  $O$ . (If you pull on a door parallel to the plane of the door, you do not rotate the door.) The other component of  $\vec{F}$  called the *tangential component*  $F_t$ , is perpendicular to  $\vec{r}$  and has magnitude  $F_t = F \sin \phi$ . This component *does* cause rotation.

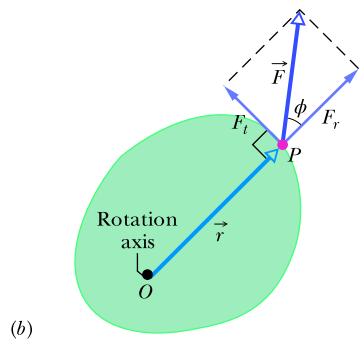
**Calculating Torques.** The ability of  $\vec{F}$  to rotate the body depends not only on the magnitude of its tangential component  $F_t$ , but also on just how far from  $O$  the force is applied. To include both these factors, we define a quantity called **torque**  $\tau$  as the product of the two factors and write it as

$$\tau = (r)(F \sin \phi). \quad (10.6.1)$$



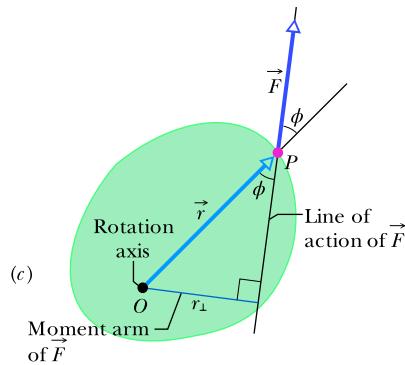
(a)

The torque due to this force causes rotation around this axis (which extends out toward you).



(b)

But actually only the *tangential* component of the force causes the rotation.



(c)

You calculate the same torque by using this moment arm distance and the full force magnitude.

**Figure 10.6.1** (a) A force  $\vec{F}$  acts on a rigid body, with a rotation axis perpendicular to the page. The torque can be found with (a) angle  $\phi$ , (b) tangential force component  $F_t$ , or (c) moment arm  $r_\perp$ .

Two equivalent ways of computing the torque are

$$\tau = (r)(F \sin \phi) = rF_t \quad (10.6.2)$$

and

$$\tau = (r \sin \phi)(F) = r_\perp F, \quad (10.6.3)$$

where  $r_\perp$  is the perpendicular distance between the rotation axis at  $O$  and an extended line running through the vector  $\vec{F}$  (Fig. 10.6.1c). This extended line is called the **line of action** of  $\vec{F}$ , and  $r_\perp$  is called the **moment arm** of  $\vec{F}$ . Figure 10.6.1b shows that we can describe  $r$ , the magnitude of  $\vec{r}$ , as being the moment arm of the force component  $F_t$ .

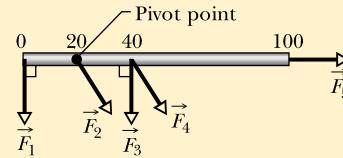
Torque, which comes from the Latin word meaning “to twist,” may be loosely identified as the turning or twisting action of the force  $\vec{F}$ . When you apply a force to an object—such as a screwdriver or torque wrench—with the purpose of turning that object, you are applying a torque. The SI unit of torque is the newton-meter ( $N \cdot m$ ). *Caution:* The newton-meter is also the unit of work. Torque and work, however, are quite different quantities and must not be confused. Work is often expressed in joules ( $1 J = 1 N \cdot m$ ), but torque never is.

**Clocks Are Negative.** In Chapter 11 we shall use vector notation for torques, but here, with rotation around a single axis, we use only an algebraic sign. If a torque would cause counterclockwise rotation, it is positive. If it would cause clockwise rotation, it is negative. (The phrase “clocks are negative” from Module 10.1 still works.)

Torques obey the superposition principle that we discussed in Chapter 5 for forces: When several torques act on a body, the **net torque** (or **resultant torque**) is the sum of the individual torques. The symbol for net torque is  $\tau_{\text{net}}$ .

### Checkpoint 10.6.1

The figure shows an overhead view of a meter stick that can pivot about the dot at the position marked 20 (for 20 cm). All five forces on the stick are horizontal and have the same magnitude. Rank the forces according to the magnitude of the torque they produce, greatest first.



## 10.7 NEWTON'S SECOND LAW FOR ROTATION

### Learning Objective

After reading this module, you should be able to . . .

**10.7.1** Apply Newton's second law for rotation to relate the net torque on a body to the body's rotational

inertia and rotational acceleration, all calculated relative to a specified rotation axis.

**Key Idea**

- The rotational analog of Newton's second law is

$$\tau_{\text{net}} = I\alpha,$$

where  $\tau_{\text{net}}$  is the net torque acting on a particle or rigid body,  $I$  is the rotational inertia of the particle or body about the rotation axis, and  $\alpha$  is the resulting angular acceleration about that axis.

**Newton's Second Law for Rotation**

A torque can cause rotation of a rigid body, as when you use a torque to rotate a door. Here we want to relate the net torque  $\tau_{\text{net}}$  on a rigid body to the angular acceleration  $\alpha$  that torque causes about a rotation axis. We do so by analogy with Newton's second law ( $F_{\text{net}} = ma$ ) for the acceleration  $a$  of a body of mass  $m$  due to a net force  $F_{\text{net}}$  along a coordinate axis. We replace  $F_{\text{net}}$  with  $\tau_{\text{net}}$ ,  $m$  with  $I$ , and  $a$  with  $\alpha$  in radian measure, writing

$$\tau_{\text{net}} = I\alpha \quad (\text{Newton's second law for rotation}). \quad (10.7.1)$$

**Proof of Equation 10.7.1**

We prove Eq. 10.7.1 by first considering the simple situation shown in Fig. 10.7.1. The rigid body there consists of a particle of mass  $m$  on one end of a massless rod of length  $r$ . The rod can move only by rotating about its other end, around a rotation axis (an axle) that is perpendicular to the plane of the page. Thus, the particle can move only in a circular path that has the rotation axis at its center.

A force  $\vec{F}$  acts on the particle. However, because the particle can move only along the circular path, only the tangential component  $F_t$  of the force (the component that is tangent to the circular path) can accelerate the particle along the path. We can relate  $F_t$  to the particle's tangential acceleration  $a_t$  along the path with Newton's second law, writing

$$F_t = ma_t.$$

The torque acting on the particle is, from Eq. 10.6.2,

$$\tau = F_t r = ma_t r.$$

From Eq. 10.3.6 ( $a_t = \alpha r$ ) we can write this as

$$\tau = m(ar)r = (mr^2)\alpha. \quad (10.7.2)$$

The quantity in parentheses on the right is the rotational inertia of the particle about the rotation axis (see Eq. 10.4.3, but here we have only a single particle). Thus, using  $I$  for the rotational inertia, Eq. 10.7.2 reduces to

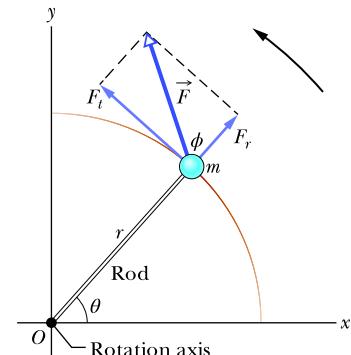
$$\tau = I\alpha \quad (\text{radian measure}). \quad (10.7.3)$$

If more than one force is applied to the particle, Eq. 10.7.3 becomes

$$\tau_{\text{net}} = I\alpha \quad (\text{radian measure}), \quad (10.7.4)$$

which we set out to prove. We can extend this equation to any rigid body rotating about a fixed axis, because any such body can always be analyzed as an assembly of single particles.

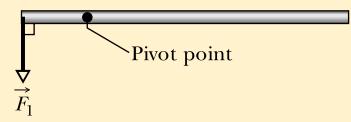
The torque due to the tangential component of the force causes an angular acceleration around the rotation axis.



**Figure 10.7.1** A simple rigid body, free to rotate about an axis through  $O$ , consists of a particle of mass  $m$  fastened to the end of a rod of length  $r$  and negligible mass. An applied force  $\vec{F}$  causes the body to rotate.

### Checkpoint 10.7.1

The figure shows an overhead view of a meter stick that can pivot about the point indicated, which is to the left of the stick's midpoint. Two horizontal forces,  $\vec{F}_1$  and  $\vec{F}_2$ , are applied to the stick. Only  $\vec{F}_1$  is shown. Force  $\vec{F}_2$  is perpendicular to the stick and is applied at the right end. If the stick is not to turn, (a) what should be the direction of  $\vec{F}_2$ , and (b) should  $F_2$  be greater than, less than, or equal to  $F_1$ ?



### Sample Problem 10.7.1 High heels

High heels (Fig. 10.7.2a) have long been popular in spite of the pain they commonly cause. Let's examine one of the causes. First, Fig. 10.7.2b is a simplified view of the forces on a foot when the person is standing still while wearing flat shoes with weight  $mg = 350 \text{ N}$  supported by each foot. The normal force  $F_{Nf}$  on the forefoot supports weight  $fmg$  with  $f = 0.40$  (that is, 40% of the weight on the foot) and acts at distance  $d_f = 0.18 \text{ m}$  from the ankle. The normal force  $F_{Nb}$  on the heel supports weight  $bmg$  with  $b = 0.60$ , at distance  $d_b = 0.070 \text{ m}$  from the ankle. The Achilles tendon (connecting the heel to the calf muscle) pulls on the heel with force  $\vec{T}$  at an angle of  $\phi = 5.0^\circ$  from a perpendicular to the plane of the foot. An unknown force from the leg bone acts downward on the ankle.

(a) What is the magnitude of  $\vec{T}$ ?

#### KEY IDEAS

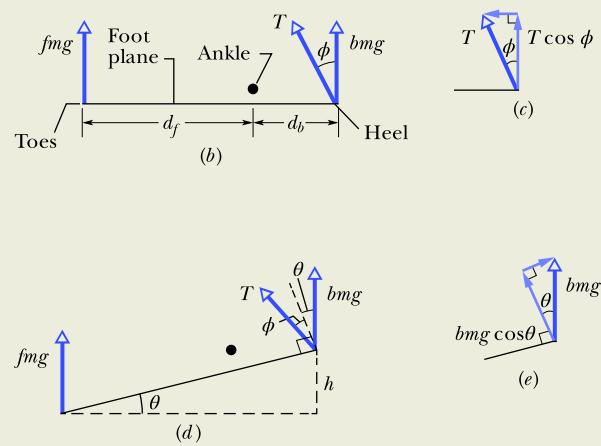
The foot is our system and is in equilibrium. Thus, the sum of the forces must balance both horizontally and vertically. Also, the sum of the torques around any point must balance.

**Calculations:** We cannot find the magnitude  $T$  of the pull from the Achilles tendon by balancing forces because we also do not know the force of the leg bone on the ankle. Instead, we can balance torques due to the forces by using a rotation axis through the ankle and perpendicular to the plane of the figure. The torque due to each force is then given by  $\tau = rF_t$  (Eq. 10.6.2), where  $r$  is the distance from the rotation axis to the point at which a force acts and  $F_t$  is the component of the force perpendicular to  $r$ , here, perpendicular to the plane of the foot.

On the forefoot, the normal force (a) is perpendicular to that plane, (b) has magnitude  $F_{Nf} = fmg$ , (c) acts at distance  $r = d_f = 0.18 \text{ m}$  from the rotation axis through the ankle, and (d) tends to rotate the foot in the (negative) clockwise direction. On the heel, the normal force (a) is also perpendicular to the foot plane, (b) has magnitude  $F_{Nb} = bmg$ , (c) acts at distance  $r = d_b = 0.070 \text{ m}$ , and (d) tends to rotate the foot in the (positive) counterclockwise direction. The Achilles tendon also acts at distance  $d_b$ . Its component perpendicular to the foot plane is  $T \cos \phi$  (Fig. 10.7.2c), which tends to produce a positive torque.



Evgeniy Skripichenko/123RF



**Figure 10.7.2** Sample Problem 10.7.1 (a) Moderate high heels. (b) Forces on forefoot and heel. (c) Components of force from Achilles tendon. (d) Elevated heel. (e) Components of the force from the shoe on the heel.

We can now write the balance of torques for this equilibrium situation as

$$\tau_{\text{net}} = 0$$

$$-d_f(fmg) + d_b(bmg) + d_b(T \cos \phi) = 0.$$

Solving for  $T$  and substituting known values, we find

$$T = \frac{d_f f - d_b b}{d_b \cos \phi} mg$$

$$= \frac{(0.18 \text{ m})(0.40) - (0.070 \text{ m})(0.60)}{(0.070 \text{ m}) \cos 5.0^\circ} (350 \text{ N})$$

$$= 151 \text{ N} \approx 0.15 \text{ kN.}$$

(b) The person next stands in shoes with moderate heel height  $h = 3.00 \text{ in.}$  ( $7.62 \text{ cm}$ ), again with the weight of  $350 \text{ N}$  on each foot. The values of  $d_f$  and  $d_b$  are unchanged but now  $f = 0.65$  ( $65\%$  of the weight is on the forefoot) and  $b = 0.35$ . Now what is the magnitude of  $\vec{T}$ ?

**Calculations:** From Fig. 10.7.2d, the plane of the foot is tilted at angle  $\theta$ :

$$\sin \theta = \frac{h}{d_f + d_b}$$

$$\theta = \sin^{-1} \frac{0.0762 \text{ m}}{(0.18 \text{ m} + 0.070 \text{ m})}$$

$$= 17.74^\circ.$$

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### Sample Problem 10.7.2 Using Newton's second law for rotation in a basic judo hip throw

To throw an  $80 \text{ kg}$  opponent with a basic judo hip throw, you intend to pull his uniform with a force  $\vec{F}$  and a moment arm  $d_1 = 0.30 \text{ m}$  from a pivot point (rotation axis) on your right hip (Fig. 10.7.3). You wish to rotate him about the pivot point with an angular acceleration  $\alpha$  of  $-6.0 \text{ rad/s}^2$ —that is, with an angular acceleration that is *clockwise* in the figure. Assume that his rotational inertia  $I$  relative to the pivot point is  $15 \text{ kg} \cdot \text{m}^2$ .

(a) What must the magnitude of  $\vec{F}$  be if, before you throw him, you bend your opponent forward to bring his center of mass to your hip (Fig. 10.7.3a)?

#### KEY IDEA

We can relate your pull  $\vec{F}$  on your opponent to the given angular acceleration  $\alpha$  via Newton's second law for rotation ( $\tau_{\text{net}} = I\alpha$ ).

**Calculations:** As his feet leave the floor, we can assume that only three forces act on him: your pull  $\vec{F}$ , a force  $\vec{N}$  on him from you at the pivot point (this force is not indicated in Fig. 10.7.3), and the gravitational force  $\vec{F}_g$ . To use  $\tau_{\text{net}} = I\alpha$ , we need the corresponding three torques, each about the pivot point.

On the heel, the vertical force is  $bmg$  and the component perpendicular to the plane of the foot is now  $bmg \cos \theta$  (Fig. 10.7.2e). On the forefoot, the vertical force is  $fmg$  and the component perpendicular to the plane of the foot is now  $fmg \cos \theta$ . The tendon's pull is still at  $5.0^\circ$  to a perpendicular to the plane of the foot. We now write the balance of torques as

$$\tau_{\text{net}} = 0$$

$$-d_f(fmg) \cos \theta + d_b(bmg) \cos \theta + d_b(T \cos \phi) = 0.$$

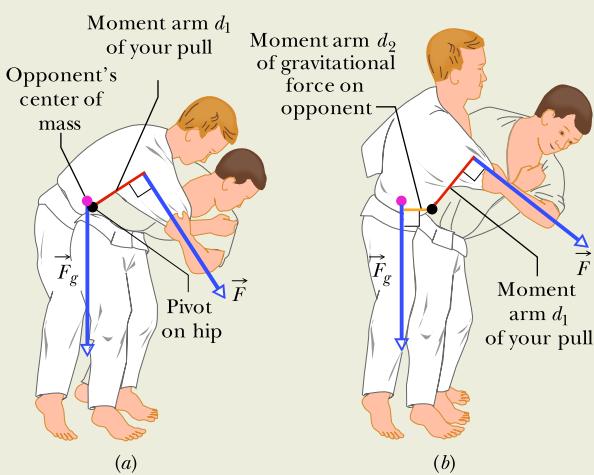
Solving for  $T$  and substituting known values, we find

$$T = \frac{d_f f - d_b b}{d_b \cos \phi} mg \cos \theta$$

$$= \frac{(0.18 \text{ m})(0.65) - (0.070 \text{ m})(0.35)}{(0.070 \text{ m}) \cos 5.0^\circ} (350 \text{ N}) (\cos 17.74^\circ)$$

$$= 442 \text{ N} \approx 0.44 \text{ kN.}$$

Thus, the force required of the Achilles tendon for simply standing still in even moderate high heels is severaltimes that required with flat shoes. Medical and physiological researchers believe sustained use of high heels permanently alters the tendon so much that walking barefooted or in flat shoes is then painful.



**Figure 10.7.3** A judo hip throw (a) correctly executed and (b) incorrectly executed.

From Eq. 10.6.3 ( $\tau = r_\perp F$ ), the torque due to your pull  $\vec{F}$  is equal to  $-d_1 F$ , where  $d_1$  is the moment arm  $r_\perp$  and the sign indicates the clockwise rotation this torque tends to cause. The torque due to  $\vec{N}$  is zero, because  $\vec{N}$  acts at the pivot point and thus has moment arm  $r_\perp = 0$ .

To evaluate the torque due to  $\vec{F}_g$ , we can assume that  $\vec{F}_g$  acts at your opponent's center of mass. With the center of mass at the pivot point,  $\vec{F}_g$  has moment arm  $r_\perp = 0$  and thus the torque due to  $\vec{F}_g$  is zero. So, the only torque on your opponent is due to your pull  $\vec{F}$ , and we can write  $\tau_{\text{net}} = I\alpha$  as

$$-d_1 F = I\alpha.$$

We then find

$$F = \frac{-I\alpha}{d_1} = \frac{-(15 \text{ kg} \cdot \text{m}^2)(-6.0 \text{ rad/s}^2)}{0.30 \text{ m}} \\ = 300 \text{ N.} \quad (\text{Answer})$$

(b) What must the magnitude of  $\vec{F}$  be if your opponent remains upright before you throw him, so that  $\vec{F}_g$  has a moment arm  $d_2 = 0.12 \text{ m}$  (Fig. 10.7.3b)?

### KEY IDEA

Because the moment arm for  $\vec{F}_g$  is no longer zero, the torque due to  $\vec{F}_g$  is now equal to  $d_2 mg$  and is positive because the torque attempts counterclockwise rotation.

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**Calculations:** Now we write  $\tau_{\text{net}} = I\alpha$  as

$$-d_1 F + d_2 mg = I\alpha,$$

which gives

$$F = -\frac{I\alpha}{d_1} + \frac{d_2 mg}{d_1}.$$

From (a), we know that the first term on the right is equal to 300 N. Substituting this and the given data, we have

$$F = 300 \text{ N} + \frac{(0.12 \text{ m})(80 \text{ kg})(9.8 \text{ m/s}^2)}{0.30 \text{ m}} \\ = 613.6 \text{ N} \approx 610 \text{ N.} \quad (\text{Answer})$$

The results indicate that you will have to pull much harder if you do not initially bend your opponent to bring his center of mass to your hip. A good judo fighter knows this lesson from physics. Indeed, physics is the basis of most of the martial arts, figured out after countless hours of trial and error over the centuries.

## 10.8 WORK AND ROTATIONAL KINETIC ENERGY

### Learning Objectives

After reading this module, you should be able to . . .

- 10.8.1 Calculate the work done by a torque acting on a rotating body by integrating the torque with respect to the angle of rotation.
- 10.8.2 Apply the work–kinetic energy theorem to relate the work done by a torque to the resulting change in the rotational kinetic energy of the body.

- 10.8.3 Calculate the work done by a constant torque by relating the work to the angle through which the body rotates.

- 10.8.4 Calculate the power of a torque by finding the rate at which work is done.

- 10.8.5 Calculate the power of a torque at any given instant by relating it to the torque and the angular velocity at that instant.

### Key Ideas

- The equations used for calculating work and power in rotational motion correspond to equations used for translational motion and are

$$W = \int_{\theta_i}^{\theta_f} \tau d\theta$$

and

$$P = \frac{dW}{dt} = \tau\omega.$$

- When  $\tau$  is constant, the integral reduces to

$$W = \tau(\theta_f - \theta_i).$$

- The form of the work–kinetic energy theorem used for rotating bodies is

$$\Delta K = K_f - K_i = \frac{1}{2}I\omega_f^2 - \frac{1}{2}I\omega_i^2 = W.$$

## Work and Rotational Kinetic Energy

As we discussed in Chapter 7, when a force  $F$  causes a rigid body of mass  $m$  to accelerate along a coordinate axis, the force does work  $W$  on the body. Thus, the body's kinetic energy ( $K = \frac{1}{2}mv^2$ ) can change. Suppose it is the only energy of the body that changes. Then we relate the change  $\Delta K$  in kinetic energy to the work  $W$  with the work–kinetic energy theorem (Eq. 7.2.8), writing

$$\Delta K = K_f - K_i = \frac{1}{2}mv_f^2 - \frac{1}{2}mv_i^2 = W \quad (\text{work–kinetic energy theorem}). \quad (10.8.1)$$

For motion confined to an  $x$  axis, we can calculate the work with Eq. 7.5.4,

$$W = \int_{x_i}^{x_f} F dx \quad (\text{work, one-dimensional motion}). \quad (10.8.2)$$

This reduces to  $W = Fd$  when  $F$  is constant and the body's displacement is  $d$ . The rate at which the work is done is the power, which we can find with Eqs. 7.6.2 and 7.6.7,

$$P = \frac{dW}{dt} = Fv \quad (\text{power, one-dimensional motion}). \quad (10.8.3)$$

Now let us consider a rotational situation that is similar. When a torque accelerates a rigid body in rotation about a fixed axis, the torque does work  $W$  on the body. Therefore, the body's rotational kinetic energy ( $K = \frac{1}{2}I\omega^2$ ) can change. Suppose that it is the only energy of the body that changes. Then we can still relate the change  $\Delta K$  in kinetic energy to the work  $W$  with the work–kinetic energy theorem, except now the kinetic energy is a rotational kinetic energy:

$$\Delta K = K_f - K_i = \frac{1}{2}I\omega_f^2 - \frac{1}{2}I\omega_i^2 = W \quad (\text{work–kinetic energy theorem}). \quad (10.8.4)$$

Here,  $I$  is the rotational inertia of the body about the fixed axis and  $\omega_i$  and  $\omega_f$  are the angular speeds of the body before and after the work is done.

Also, we can calculate the work with a rotational equivalent of Eq. 10.8.2,

$$W = \int_{\theta_i}^{\theta_f} \tau d\theta \quad (\text{work, rotation about fixed axis}), \quad (10.8.5)$$

where  $\tau$  is the torque doing the work  $W$ , and  $\theta_i$  and  $\theta_f$  are the body's angular positions before and after the work is done, respectively. When  $\tau$  is constant, Eq. 10.8.5 reduces to

$$W = \tau(\theta_f - \theta_i) \quad (\text{work, constant torque}). \quad (10.8.6)$$

The rate at which the work is done is the power, which we can find with the rotational equivalent of Eq. 10.8.3,

$$P = \frac{dW}{dt} = \tau\omega \quad (\text{power, rotation about fixed axis}). \quad (10.8.7)$$

Table 10.8.1 summarizes the equations that apply to the rotation of a rigid body about a fixed axis and the corresponding equations for translational motion.

### Proof of Eqs. 10.8.4 through 10.8.7

Let us again consider the situation of Fig. 10.7.1, in which force  $\vec{F}$  rotates a rigid body consisting of a single particle of mass  $m$  fastened to the end of a massless rod. During the rotation, force  $\vec{F}$  does work on the body. Let us assume that the only energy of the body that is changed by  $\vec{F}$  is the kinetic energy. Then we can apply the work–kinetic energy theorem of Eq. 10.8.1:

$$\Delta K = K_f - K_i = W. \quad (10.8.8)$$

Using  $K = \frac{1}{2}mv^2$  and Eq. 10.3.2 ( $v = \omega r$ ), we can rewrite Eq. 10.8.8 as

$$\Delta K = \frac{1}{2}mr^2\omega_f^2 - \frac{1}{2}mr^2\omega_i^2 = W. \quad (10.8.9)$$

From Eq. 10.4.3, the rotational inertia for this one-particle body is  $I = mr^2$ . Substituting this into Eq. 10.8.9 yields

$$\Delta K = \frac{1}{2}I\omega_f^2 - \frac{1}{2}I\omega_i^2 = W,$$

which is Eq. 10.8.4. We derived it for a rigid body with one particle, but it holds for any rigid body rotated about a fixed axis.

We next relate the work  $W$  done on the body in Fig. 10.7.1 to the torque  $\tau$  on the body due to force  $\vec{F}$ . When the particle moves a distance  $ds$  along its circular path, only the tangential component  $F_t$  of the force accelerates the particle along the path. Therefore, only  $F_t$  does work on the particle. We write that work  $dW$  as  $F_t ds$ . However, we can replace  $ds$  with  $r d\theta$ , where  $d\theta$  is the angle through which the particle moves. Thus we have

$$dW = F_t r d\theta. \quad (10.8.10)$$

From Eq. 10.6.2, we see that the product  $F_t r$  is equal to the torque  $\tau$ , so we can rewrite Eq. 10.8.10 as

$$dW = \tau d\theta. \quad (10.8.11)$$

The work done during a finite angular displacement from  $\theta_i$  to  $\theta_f$  is then

$$W = \int_{\theta_i}^{\theta_f} \tau d\theta,$$

which is Eq. 10.8.5. It holds for any rigid body rotating about a fixed axis. Equation 10.8.6 comes directly from Eq. 10.8.5.

We can find the power  $P$  for rotational motion from Eq. 10.8.11:

$$P = \frac{dW}{dt} = \tau \frac{d\theta}{dt} = \tau\omega,$$

which is Eq. 10.8.7.

**Table 10.8.1** Some Corresponding Relations for Translational and Rotational Motion

Pure Translation (Fixed Direction)		Pure Rotation (Fixed Axis)	
Position	$x$	Angular position	$\theta$
Velocity	$v = dx/dt$	Angular velocity	$\omega = d\theta/dt$
Acceleration	$a = dv/dt$	Angular acceleration	$\alpha = d\omega/dt$
Mass	$m$	Rotational inertia	$I$
Newton's second law	$F_{\text{net}} = ma$	Newton's second law	$\tau_{\text{net}} = I\alpha$
Work	$W = \int F dx$	Work	$W = \int \tau d\theta$
Kinetic energy	$K = \frac{1}{2}mv^2$	Kinetic energy	$K = \frac{1}{2}I\omega^2$
Power (constant force)	$P = Fv$	Power (constant torque)	$P = \tau\omega$
Work–kinetic energy theorem	$W = \Delta K$	Work–kinetic energy theorem	$W = \Delta K$

### Checkpoint 10.8.1

Here are four examples of a single torque being applied to a rigid body rotating around a fixed axis. At a certain instant, the table gives the torque and the body's angular velocity. (a) Rank the examples according to the power of the torque, most positive first, most negative last. (b) In which is the rotation slowing? (c) In which is positive work being done by the torque?

Example	Torque (N · m)	Angular Velocity (rad/s)
A	+5	+3
B	+5	-3
C	-5	-3
D	-5	+3

## Review & Summary

**Angular Position** To describe the rotation of a rigid body about a fixed axis, called the **rotation axis**, we assume a **reference line** is fixed in the body, perpendicular to that axis and rotating with the body. We measure the **angular position**  $\theta$  of this line relative to a fixed direction. When  $\theta$  is measured in **radians**,

$$\theta = \frac{s}{r} \quad (\text{radian measure}), \quad (10.1.1)$$

where  $s$  is the arc length of a circular path of radius  $r$  and angle  $\theta$ . Radian measure is related to angle measure in revolutions and degrees by

$$1 \text{ rev} = 360^\circ = 2\pi \text{ rad}. \quad (10.1.2)$$

**Angular Displacement** A body that rotates about a rotation axis, changing its angular position from  $\theta_1$  to  $\theta_2$ , undergoes an **angular displacement**

$$\Delta\theta = \theta_2 - \theta_1, \quad (10.1.4)$$

where  $\Delta\theta$  is positive for counterclockwise rotation and negative for clockwise rotation.

**Angular Velocity and Speed** If a body rotates through an angular displacement  $\Delta\theta$  in a time interval  $\Delta t$ , its **average angular velocity**  $\omega_{\text{avg}}$  is

$$\omega_{\text{avg}} = \frac{\Delta\theta}{\Delta t}. \quad (10.1.5)$$

The **(instantaneous) angular velocity**  $\omega$  of the body is

$$\omega = \frac{d\theta}{dt}. \quad (10.1.6)$$

Both  $\omega_{\text{avg}}$  and  $\omega$  are vectors, with directions given by the **right-hand rule** of Fig. 10.1.6. They are positive for counterclockwise rotation and negative for clockwise rotation. The magnitude of the body's angular velocity is the **angular speed**.

**Angular Acceleration** If the angular velocity of a body changes from  $\omega_1$  to  $\omega_2$  in a time interval  $\Delta t = t_2 - t_1$ , the **average angular acceleration**  $\alpha_{\text{avg}}$  of the body is

$$\alpha_{\text{avg}} = \frac{\omega_2 - \omega_1}{t_2 - t_1} = \frac{\Delta\omega}{\Delta t}. \quad (10.1.7)$$

The **(instantaneous) angular acceleration**  $\alpha$  of the body is

$$\alpha = \frac{d\omega}{dt}. \quad (10.1.8)$$

Both  $\alpha_{\text{avg}}$  and  $\alpha$  are vectors.

### The Kinematic Equations for Constant Angular Acceleration

Constant angular acceleration ( $\alpha = \text{constant}$ ) is an important special case of rotational motion. The appropriate kinematic equations, given in Table 10.2.1, are

$$\omega = \omega_0 + \alpha t, \quad (10.2.1)$$

$$\theta - \theta_0 = \omega_0 t + \frac{1}{2}\alpha t^2, \quad (10.2.2)$$

$$\omega^2 = \omega_0^2 + 2\alpha(\theta - \theta_0), \quad (10.2.3)$$

$$\theta - \theta_0 = \frac{1}{2}(\omega_0 + \omega)t, \quad (10.2.4)$$

$$\theta - \theta_0 = \omega t - \frac{1}{2}\alpha t^2. \quad (10.2.5)$$

**Linear and Angular Variables Related** A point in a rigid rotating body, at a *perpendicular distance*  $r$  from the rotation axis, moves in a circle with radius  $r$ . If the body rotates through an angle  $\theta$ , the point moves along an arc with length  $s$  given by

$$s = \theta r \quad (\text{radian measure}), \quad (10.3.1)$$

where  $\theta$  is in radians.

The linear velocity  $\vec{v}$  of the point is tangent to the circle; the point's linear speed  $v$  is given by

$$v = \omega r \quad (\text{radian measure}), \quad (10.3.2)$$

where  $\omega$  is the angular speed (in radians per second) of the body.

The linear acceleration  $\vec{a}$  of the point has both *tangential* and *radial* components. The tangential component is

$$a_t = \alpha r \quad (\text{radian measure}), \quad (10.3.6)$$

where  $\alpha$  is the magnitude of the angular acceleration (in radians per second-squared) of the body. The radial component of  $\vec{a}$  is

$$a_r = \frac{v^2}{r} = \omega^2 r \quad (\text{radian measure}). \quad (10.3.7)$$

If the point moves in uniform circular motion, the period  $T$  of the motion for the point and the body is

$$T = \frac{2\pi}{v} = \frac{2\pi}{\omega} \quad (\text{radian measure}). \quad (10.3.3, 10.3.4)$$

### Rotational Kinetic Energy and Rotational Inertia

The kinetic energy  $K$  of a rigid body rotating about a fixed axis is given by

$$K = \frac{1}{2}I\omega^2 \quad (\text{radian measure}), \quad (10.4.4)$$

in which  $I$  is the **rotational inertia** of the body, defined as

$$I = \sum m_i r_i^2 \quad (10.4.3)$$

for a system of discrete particles and defined as

$$I = \int r^2 dm \quad (10.5.1)$$

for a body with continuously distributed mass. The  $r$  and  $r_i$  in these expressions represent the perpendicular distance from the axis of rotation to each mass element in the body, and the integration is carried out over the entire body so as to include every mass element.

**The Parallel-Axis Theorem** The *parallel-axis theorem* relates the rotational inertia  $I$  of a body about any axis to that of the same body about a parallel axis through the center of mass:

$$I = I_{\text{com}} + Mh^2. \quad (10.5.2)$$

Here  $h$  is the perpendicular distance between the two axes, and  $I_{\text{com}}$  is the rotational inertia of the body about the axis through the com. We can describe  $h$  as being the distance the actual rotation axis has been shifted from the rotation axis through the com.

**Torque** *Torque* is a turning or twisting action on a body about a rotation axis due to a force  $\vec{F}$ . If  $\vec{F}$  is exerted at a point

## Questions

- 1 Figure 10.1 is a graph of the angular velocity versus time for a disk rotating like a merry-go-round. For a point on the disk rim, rank the instants  $a, b, c$ , and  $d$  according to the magnitude of the (a) tangential and (b) radial acceleration, greatest first.

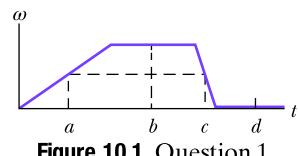


Figure 10.1 Question 1.

- 2 Figure 10.2 shows plots of angular position  $\theta$  versus time  $t$  for three cases in which a disk is rotated like a merry-go-round. In each case, the rotation direction changes at a certain angular position  $\theta_{\text{change}}$ . (a) For each case, determine whether  $\theta_{\text{change}}$  is clockwise or counterclockwise from  $\theta = 0$ , or whether it is at  $\theta = 0$ . For each case, determine (b) whether  $\omega$  is zero before, after, or at  $t = 0$  and (c) whether  $\alpha$  is positive, negative, or zero.

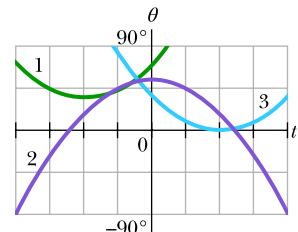


Figure 10.2 Question 2.

given by the position vector  $\vec{r}$  relative to the axis, then the magnitude of the torque is

$$\tau = rF_t = r_\perp F = rF \sin \phi. \quad (10.6.2, 10.6.3, 10.6.1)$$

where  $F_t$  is the component of  $\vec{F}$  perpendicular to  $\vec{r}$  and  $\phi$  is the angle between  $\vec{r}$  and  $\vec{F}$ . The quantity  $r_\perp$  is the perpendicular distance between the rotation axis and an extended line running through the  $\vec{F}$  vector. This line is called the **line of action** of  $\vec{F}$ , and  $r_\perp$  is called the **moment arm** of  $\vec{F}$ . Similarly,  $r$  is the moment arm of  $F_t$ .

The SI unit of torque is the newton-meter ( $N \cdot m$ ). A torque  $\tau$  is positive if it tends to rotate a body at rest counterclockwise and negative if it tends to rotate the body clockwise.

**Newton's Second Law in Angular Form** The rotational analog of Newton's second law is

$$\tau_{\text{net}} = I\alpha, \quad (10.7.4)$$

where  $\tau_{\text{net}}$  is the net torque acting on a particle or rigid body,  $I$  is the rotational inertia of the particle or body about the rotation axis, and  $\alpha$  is the resulting angular acceleration about that axis.

**Work and Rotational Kinetic Energy** The equations used for calculating work and power in rotational motion correspond to equations used for translational motion and are

$$W = \int_{\theta_i}^{\theta_f} \tau d\theta \quad (10.8.5)$$

$$\text{and} \quad P = \frac{dW}{dt} = \tau\omega. \quad (10.8.7)$$

When  $\tau$  is constant, Eq. 10.8.5 reduces to

$$W = \tau(\theta_f - \theta_i). \quad (10.8.6)$$

The form of the work–kinetic energy theorem used for rotating bodies is

$$\Delta K = K_f - K_i = \frac{1}{2}I\omega_f^2 - \frac{1}{2}I\omega_i^2 = W. \quad (10.8.4)$$

- 3 A force is applied to the rim of a disk that can rotate like a merry-go-round, so as to change its angular velocity. Its initial and final angular velocities, respectively, for four situations are: (a)  $-2 \text{ rad/s}$ ,  $5 \text{ rad/s}$ ; (b)  $2 \text{ rad/s}$ ,  $5 \text{ rad/s}$ ; (c)  $-2 \text{ rad/s}$ ,  $-5 \text{ rad/s}$ ; and (d)  $2 \text{ rad/s}$ ,  $-5 \text{ rad/s}$ . Rank the situations according to the work done by the torque due to the force, greatest first.

- 4 Figure 10.3b is a graph of the angular position of the rotating disk of Fig. 10.3a. Is the angular velocity of the disk positive,

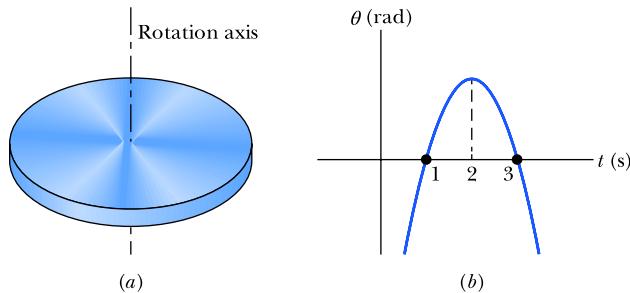


Figure 10.3 Question 4.

negative, or zero at (a)  $t = 1$  s, (b)  $t = 2$  s, and (c)  $t = 3$  s? (d) Is the angular acceleration positive or negative?

- 5** In Fig. 10.4, two forces  $\vec{F}_1$  and  $\vec{F}_2$  act on a disk that turns about its center like a merry-go-round. The forces maintain the indicated angles during the rotation, which is counterclockwise and at a constant rate. However, we are to decrease the angle  $\theta$  of  $\vec{F}_1$  without changing the magnitude of  $\vec{F}_1$ . (a) To keep the angular speed constant, should we increase, decrease, or maintain the magnitude of  $\vec{F}_2$ ? Do forces (b)  $\vec{F}_1$  and (c)  $\vec{F}_2$  tend to rotate the disk clockwise or counterclockwise?

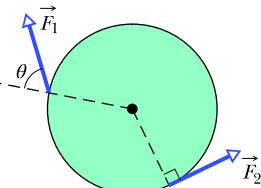


Figure 10.4 Question 5.

- 6** In the overhead view of Fig. 10.5, five forces of the same magnitude act on a strange merry-go-round; it is a square that can rotate about point  $P$ , at midlength along one of the edges. Rank the forces according to the magnitude of the torque they create about point  $P$ , greatest first.

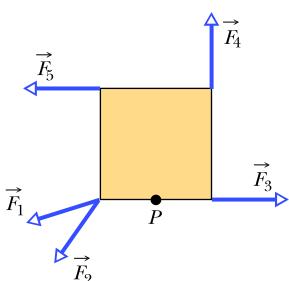


Figure 10.5 Question 6.

- 7** Figure 10.6a is an overhead view of a horizontal bar that can pivot; two horizontal forces act on the bar, but it is stationary. If the angle between the bar and  $\vec{F}_2$  is now decreased from  $90^\circ$  and the bar is still not to turn, should  $F_2$  be made larger, made smaller, or left the same?

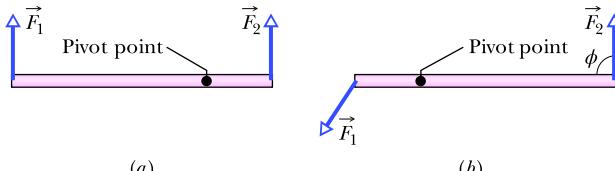


Figure 10.6 Questions 7 and 8.

- 8** Figure 10.6b shows an overhead view of a horizontal bar that is rotated about the pivot point by two horizontal forces,  $\vec{F}_1$  and  $\vec{F}_2$ , with  $\vec{F}_2$  at angle  $\phi$  to the bar. Rank the following values of  $\phi$  according to the magnitude of the angular acceleration of the bar, greatest first:  $90^\circ$ ,  $70^\circ$ , and  $110^\circ$ .

- 9** Figure 10.7 shows a uniform metal plate that had been square before 25% of it was snipped off. Three lettered points

are indicated. Rank them according to the rotational inertia of the plate around a perpendicular axis through them, greatest first.

- 10** Figure 10.8 shows three flat disks (of the same radius) that can rotate about their centers like merry-go-rounds. Each disk consists of the same two materials, one denser than the other (density is mass per unit volume). In disks 1 and 3, the denser material forms the outer half of the disk area. In disk 2, it forms the inner half of the disk area. Forces with identical magnitudes are applied tangentially to the disk, either at the outer edge or at the interface of the two materials, as shown. Rank the disks according to (a) the torque about the disk center, (b) the rotational inertia about the disk center, and (c) the angular acceleration of the disk, greatest first.

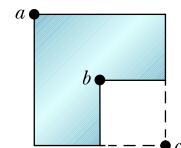


Figure 10.7  
Question 9.

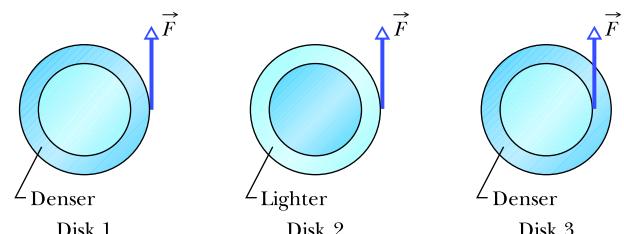


Figure 10.8 Question 10.

- 11** Figure 10.9a shows a meter stick, half wood and half steel, that is pivoted at the wood end at  $O$ . A force  $\vec{F}$  is applied to the steel end at  $a$ . In Fig. 10.9b, the stick is reversed and pivoted at the steel end at  $O'$ , and the same force is applied at the wood end at  $a'$ . Is the resulting angular acceleration of Fig. 10.9a greater than, less than, or the same as that of Fig. 10.9b?

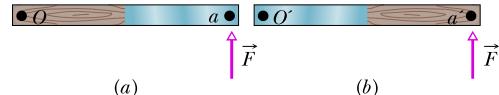


Figure 10.9

Question 11.

$R:$	1 m	2 m	3 m
$M:$	26 kg	7 kg	3 kg

(a) (b) (c)

Figure 10.10 Question 12.

- 12** Figure 10.10 shows three disks, each with a uniform distribution of mass. The radii  $R$  and masses  $M$  are indicated. Each disk can rotate around its central axis (perpendicular to the disk face and through the center). Rank the disks according to their rotational inertias calculated about their central axes, greatest first.

## Problems

Tutoring problem available (at instructor's discretion) in WileyPLUS

Worked-out solution available in Student Solutions Manual

Easy Medium Hard

Additional information available in *The Flying Circus of Physics* and at [flyingcircusofphysics.com](http://flyingcircusofphysics.com)

Requires calculus

Biomedical application

### Module 10.1 Rotational Variables

- 1 E** A good baseball pitcher can throw a baseball toward home plate at 85 mi/h with a spin of 1800 rev/min. How many revolutions does the baseball make on its way to home plate? For simplicity, assume that the 60 ft path is a straight line.

- 2 E** What is the angular speed of (a) the second hand, (b) the minute hand, and (c) the hour hand of a smoothly running analog watch? Answer in radians per second.

- 3 M FCP** When a slice of buttered toast is accidentally pushed over the edge of a counter, it rotates as it falls. If the distance to

the floor is 76 cm and for rotation less than 1 rev, what are the (a) smallest and (b) largest angular speeds that cause the toast to hit and then topple to be butter-side down?

**4 M CALC** The angular position of a point on a rotating wheel is given by  $\theta = 2.0 + 4.0t^2 + 2.0t^3$ , where  $\theta$  is in radians and  $t$  is in seconds. At  $t = 0$ , what are (a) the point's angular position and (b) its angular velocity? (c) What is its angular velocity at  $t = 4.0$  s? (d) Calculate its angular acceleration at  $t = 2.0$  s. (e) Is its angular acceleration constant?

**5 M** A diver makes 2.5 revolutions on the way from a 10-m-high platform to the water. Assuming zero initial vertical velocity, find the average angular velocity during the dive.

**6 M CALC** The angular position of a point on the rim of a rotating wheel is given by  $\theta = 4.0t - 3.0t^2 + t^3$ , where  $\theta$  is in radians and  $t$  is in seconds. What are the angular velocities at (a)  $t = 2.0$  s and (b)  $t = 4.0$  s? (c) What is the average angular acceleration for the time interval that begins at  $t = 2.0$  s and ends at  $t = 4.0$  s? What are the instantaneous angular accelerations at (d) the beginning and (e) the end of this time interval?

**7 H** The wheel in Fig. 10.11 has eight equally spaced spokes and a radius of 30 cm. It is mounted on a fixed axle and is spinning at 2.5 rev/s. You want to shoot a 20-cm-long arrow parallel to this axle and through the wheel without hitting any of the spokes. Assume that the arrow and the spokes are very thin. (a) What minimum speed must the arrow have? (b) Does it matter where between the axle and rim of the wheel you aim? If so, what is the best location?

**8 H CALC** The angular acceleration of a wheel is  $\alpha = 6.0t^4 - 4.0t^2$ , with  $\alpha$  in radians per second-squared and  $t$  in seconds. At time  $t = 0$ , the wheel has an angular velocity of +2.0 rad/s and an angular position of +1.0 rad. Write expressions for (a) the angular velocity (rad/s) and (b) the angular position (rad) as functions of time (s).

### Module 10.2 Rotation with Constant Angular Acceleration

**9 E** A drum rotates around its central axis at an angular velocity of 12.60 rad/s. If the drum then slows at a constant rate of 4.20 rad/s<sup>2</sup>, (a) how much time does it take and (b) through what angle does it rotate in coming to rest?

**10 E** Starting from rest, a disk rotates about its central axis with constant angular acceleration. In 5.0 s, it rotates 25 rad. During that time, what are the magnitudes of (a) the angular acceleration and (b) the average angular velocity? (c) What is the instantaneous angular velocity of the disk at the end of the 5.0 s? (d) With the angular acceleration unchanged, through what additional angle will the disk turn during the next 5.0 s?

**11 E** A disk, initially rotating at 120 rad/s, is slowed down with a constant angular acceleration of magnitude 4.0 rad/s<sup>2</sup>. (a) How much time does the disk take to stop? (b) Through what angle does the disk rotate during that time?

**12 E** The angular speed of an automobile engine is increased at a constant rate from 1200 rev/min to 3000 rev/min in 12 s.

(a) What is its angular acceleration in revolutions per minute-squared? (b) How many revolutions does the engine make during this 12 s interval?

**13 M** A flywheel turns through 40 rev as it slows from an angular speed of 1.5 rad/s to a stop. (a) Assuming a constant angular acceleration, find the time for it to come to rest. (b) What is its angular acceleration? (c) How much time is required for it to complete the first 20 of the 40 revolutions?

**14 M GO** A disk rotates about its central axis starting from rest and accelerates with constant angular acceleration. At one time it is rotating at 10 rev/s; 60 revolutions later, its angular speed is 15 rev/s. Calculate (a) the angular acceleration, (b) the time required to complete the 60 revolutions, (c) the time required to reach the 10 rev/s angular speed, and (d) the number of revolutions from rest until the time the disk reaches the 10 rev/s angular speed.

**15 M SSM** Starting from rest, a wheel has constant  $\alpha = 3.0$  rad/s<sup>2</sup>. During a certain 4.0 s interval, it turns through 120 rad. How much time did it take to reach that 4.0 s interval?

**16 M** A merry-go-round rotates from rest with an angular acceleration of 1.50 rad/s<sup>2</sup>. How long does it take to rotate through (a) the first 2.00 rev and (b) the next 2.00 rev?

**17 M** At  $t = 0$ , a flywheel has an angular velocity of 4.7 rad/s, a constant angular acceleration of  $-0.25$  rad/s<sup>2</sup>, and a reference line at  $\theta_0 = 0$ . (a) Through what maximum angle  $\theta_{\max}$  will the reference line turn in the positive direction? What are the (b) first and (c) second times the reference line will be at  $\theta = \frac{1}{2}\theta_{\max}$ ? At what (d) negative time and (e) positive time will the reference line be at  $\theta = 10.5$  rad? (f) Graph  $\theta$  versus  $t$ , and indicate your answers.

**18 H CALC** A pulsar is a rapidly rotating neutron star that emits a radio beam the way a lighthouse emits a light beam. We receive a radio pulse for each rotation of the star. The period  $T$  of rotation is found by measuring the time between pulses. The pulsar in the Crab nebula has a period of rotation of  $T = 0.033$  s that is increasing at the rate of  $1.26 \times 10^{-5}$  s/y. (a) What is the pulsar's angular acceleration  $\alpha$ ? (b) If  $\alpha$  is constant, how many years from now will the pulsar stop rotating? (c) The pulsar originated in a supernova explosion seen in the year 1054. Assuming constant  $\alpha$ , find the initial  $T$ .

### Module 10.3 Relating the Linear and Angular Variables

**19 E** What are the magnitudes of (a) the angular velocity, (b) the radial acceleration, and (c) the tangential acceleration of a spaceship taking a circular turn of radius 3220 km at a speed of 29 000 km/h?

**20 E CALC** An object rotates about a fixed axis, and the angular position of a reference line on the object is given by  $\theta = 0.40e^{2t}$ , where  $\theta$  is in radians and  $t$  is in seconds. Consider a point on the object that is 4.0 cm from the axis of rotation. At  $t = 0$ , what are the magnitudes of the point's (a) tangential component of acceleration and (b) radial component of acceleration?

**21 E FCP** Between 1911 and 1990, the top of the leaning bell tower at Pisa, Italy, moved toward the south at an average rate of 1.2 mm/y. The tower is 55 m tall. In radians per second, what is the average angular speed of the tower's top about its base?

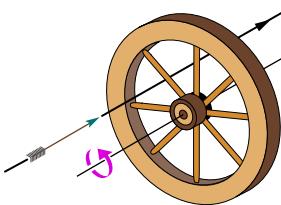


Figure 10.11 Problem 7.

**22 E BIO CALC** An astronaut is tested in a centrifuge with radius 10 m and rotating according to  $\theta = 0.30t^2$ . At  $t = 5.0$  s, what are the magnitudes of the (a) angular velocity, (b) linear velocity, (c) tangential acceleration, and (d) radial acceleration?

**23 E SSM** A flywheel with a diameter of 1.20 m is rotating at an angular speed of 200 rev/min. (a) What is the angular speed of the flywheel in radians per second? (b) What is the linear speed of a point on the rim of the flywheel? (c) What constant angular acceleration (in revolutions per minute-squared) will increase the wheel's angular speed to 1000 rev/min in 60.0 s? (d) How many revolutions does the wheel make during that 60.0 s?

**24 E** A vinyl record is played by rotating the record so that an approximately circular groove in the vinyl slides under a stylus. Bumps in the groove run into the stylus, causing it to oscillate. The equipment converts those oscillations to electrical signals and then to sound. Suppose that a record turns at the rate of  $33\frac{1}{3}$  rev/min, the groove being played is at a radius of 10.0 cm, and the bumps in the groove are uniformly separated by 1.75 mm. At what rate (hits per second) do the bumps hit the stylus?

**25 M SSM** (a) What is the angular speed  $\omega$  about the polar axis of a point on Earth's surface at latitude  $40^\circ$  N? (Earth rotates about that axis.) (b) What is the linear speed  $v$  of the point? What are (c)  $\omega$  and (d)  $v$  for a point at the equator?

**26 M** The flywheel of a steam engine runs with a constant angular velocity of 150 rev/min. When steam is shut off, the friction of the bearings and of the air stops the wheel in 2.2 h. (a) What is the constant angular acceleration, in revolutions per minute-squared, of the wheel during the slowdown? (b) How many revolutions does the wheel make before stopping? (c) At the instant the flywheel is turning at 75 rev/min, what is the tangential component of the linear acceleration of a flywheel particle that is 50 cm from the axis of rotation? (d) What is the magnitude of the net linear acceleration of the particle in (c)?

**27 M** A seed is on a turntable rotating at  $33\frac{1}{3}$  rev/min, 6.0 cm from the rotation axis. What are (a) the seed's acceleration and (b) the least coefficient of static friction to avoid slippage? (c) If the turntable had undergone constant angular acceleration from rest in 0.25 s, what is the least coefficient to avoid slippage?

**28 M** In Fig. 10.12, wheel *A* of radius  $r_A = 10$  cm is coupled by belt *B* to wheel *C* of radius  $r_C = 25$  cm. The angular speed of wheel *A* is increased from rest at a constant rate of  $1.6 \text{ rad/s}^2$ . Find the time needed for wheel *C* to reach an angular speed of 100 rev/min, assuming the belt does not slip. (*Hint:* If the belt does not slip, the linear speeds at the two rims must be equal.)

**29 M** Figure 10.13 shows an early method of measuring the speed of light that makes use of a rotating slotted wheel. A beam of light passes through one of the slots at the outside edge of the wheel, travels to a distant mirror, and returns to the wheel just in time to pass through the next slot in the wheel. One such slotted wheel has a radius of 5.0 cm and 500 slots around its edge. Measurements taken when the mirror is  $L = 500$  m from the wheel indicate a speed of light of

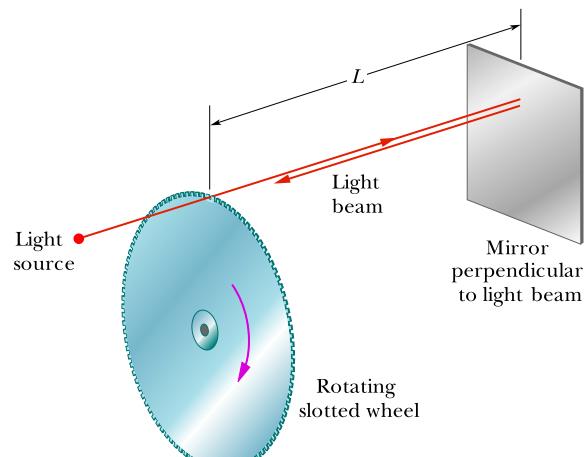


Figure 10.13 Problem 29.

$3.0 \times 10^5$  km/s. (a) What is the (constant) angular speed of the wheel? (b) What is the linear speed of a point on the edge of the wheel?

**30 M GO** A gyroscope flywheel of radius 2.83 cm is accelerated from rest at  $14.2 \text{ rad/s}^2$  until its angular speed is 2760 rev/min. (a) What is the tangential acceleration of a point on the rim of the flywheel during this spin-up process? (b) What is the radial acceleration of this point when the flywheel is spinning at full speed? (c) Through what distance does a point on the rim move during the spin-up?

**31 M GO** A disk, with a radius of 0.25 m, is to be rotated like a merry-go-round through  $800$  rad, starting from rest, gaining angular speed at the constant rate  $\alpha_1$  through the first  $400$  rad and then losing angular speed at the constant rate  $-\alpha_1$  until it is again at rest. The magnitude of the centripetal acceleration of any portion of the disk is not to exceed  $400 \text{ m/s}^2$ . (a) What is the least time required for the rotation? (b) What is the corresponding value of  $\alpha_1$ ?

**32 M** A car starts from rest and moves around a circular track of radius 30.0 m. Its speed increases at the constant rate of  $0.500 \text{ m/s}^2$ . (a) What is the magnitude of its net linear acceleration 15.0 s later? (b) What angle does this net acceleration vector make with the car's velocity at this time?

#### Module 10.4 Kinetic Energy of Rotation

**33 E SSM** Calculate the rotational inertia of a wheel that has a kinetic energy of 24 400 J when rotating at 602 rev/min.

**34 E** Figure 10.14 gives angular speed versus time  $\omega$  (rad/s) for a thin rod that rotates around one end. The scale on the  $\omega$  axis is set by  $\omega_s = 6.0 \text{ rad/s}$ . (a) What is the magnitude of the rod's angular acceleration? (b) At  $t = 4.0$  s, the rod has a rotational kinetic energy of 1.60 J. What is its kinetic energy at  $t = 0$ ?

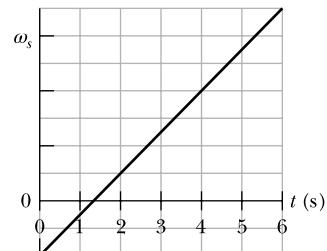


Figure 10.14 Problem 34.

**Module 10.5 Calculating the Rotational Inertia**

**35 E SSM** Two uniform solid cylinders, each rotating about its central (longitudinal) axis at 235 rad/s, have the same mass of 1.25 kg but differ in radius. What is the rotational kinetic energy of (a) the smaller cylinder, of radius 0.25 m, and (b) the larger cylinder, of radius 0.75 m?

**36 E** Figure 10.15a shows a disk that can rotate about an axis at a radial distance  $h$  from the center of the disk. Figure 10.15b gives the rotational inertia  $I$  of the disk about the axis as a function of that distance  $h$ , from the center out to the edge of the disk. The scale on the  $I$  axis is set by  $I_A = 0.050 \text{ kg} \cdot \text{m}^2$  and  $I_B = 0.150 \text{ kg} \cdot \text{m}^2$ . What is the mass of the disk?

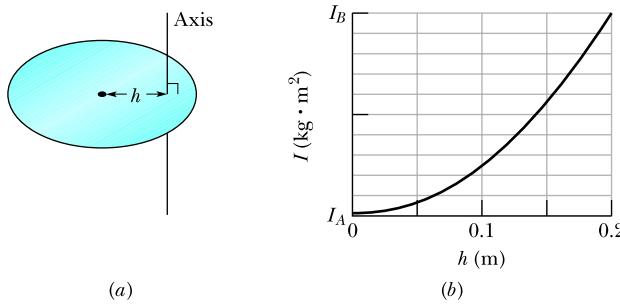


Figure 10.15 Problem 36.

**37 E SSM** Calculate the rotational inertia of a meter stick, with mass 0.56 kg, about an axis perpendicular to the stick and located at the 20 cm mark. (Treat the stick as a thin rod.)

**38 E** Figure 10.16 shows three 0.0100 kg particles that have been glued to a rod of length  $L = 6.00 \text{ cm}$  and negligible mass. The assembly can rotate around a perpendicular axis through point  $O$  at the left end. If we remove one particle (that is, 33% of the mass), by what percentage does the rotational inertia of the assembly around the rotation axis decrease when that removed particle is (a) the innermost one and (b) the outermost one?

**39 M** Trucks can be run on energy stored in a rotating flywheel, with an electric motor getting the flywheel up to its top speed of  $200\pi \text{ rad/s}$ . Suppose that one such flywheel is a solid, uniform cylinder with a mass of 500 kg and a radius of 1.0 m. (a) What is the kinetic energy of the flywheel after charging? (b) If the truck uses an average power of 8.0 kW, for how many minutes can it operate between chargings?

**40 M** Figure 10.17 shows an arrangement of 15 identical disks that have been glued together in a rod-like shape of length  $L = 1.0000 \text{ m}$  and (total) mass  $M = 100.0 \text{ mg}$ . The disks are uniform, and the disk arrangement can rotate about a perpendicular axis through its central disk at point  $O$ . (a) What is the rotational inertia of the arrangement about that axis? (b) If we approximated the arrangement as being a uniform rod of mass  $M$  and length  $L$ , what percentage error would we make in using the formula in Table 10.5.1e to calculate the rotational inertia?

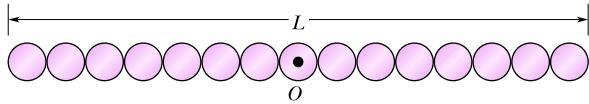


Figure 10.17 Problem 40.

**41 M SC** In Fig. 10.18, two particles, each with mass  $m = 0.85 \text{ kg}$ , are fastened to each other, and to a rotation axis at  $O$ , by two thin rods, each with length  $d = 5.6 \text{ cm}$  and mass  $M = 1.2 \text{ kg}$ . The combination rotates around the rotation axis with the angular speed  $\omega = 0.30 \text{ rad/s}$ . Measured about  $O$ , what are the combination's (a) rotational inertia and (b) kinetic energy?

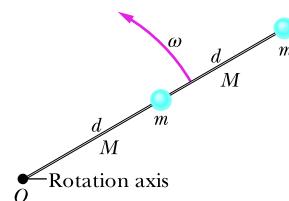


Figure 10.18 Problem 41.

**42 M** The masses and coordinates of four particles are as follows: 50 g,  $x = 2.0 \text{ cm}$ ,  $y = 2.0 \text{ cm}$ ; 25 g,  $x = 0$ ,  $y = 4.0 \text{ cm}$ ; 25 g,  $x = -3.0 \text{ cm}$ ,  $y = -3.0 \text{ cm}$ ; 30 g,  $x = -2.0 \text{ cm}$ ,  $y = 4.0 \text{ cm}$ . What are the rotational inertias of this collection about the (a)  $x$ , (b)  $y$ , and (c)  $z$  axes? (d) Suppose that we symbolize the answers to (a) and (b) as  $A$  and  $B$ , respectively. Then what is the answer to (c) in terms of  $A$  and  $B$ ?

**43 M SSM** The uniform solid block in Fig. 10.19 has mass 0.172 kg and edge lengths  $a = 3.5 \text{ cm}$ ,  $b = 8.4 \text{ cm}$ , and  $c = 1.4 \text{ cm}$ . Calculate its rotational inertia about an axis through one corner and perpendicular to the large faces.

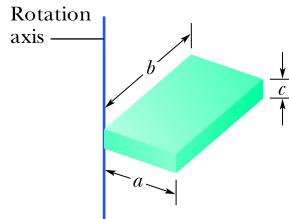


Figure 10.19 Problem 43.

**44 M** Four identical particles of mass 0.50 kg each are placed at the vertices of a  $2.0 \text{ m} \times 2.0 \text{ m}$  square and held there by four massless rods, which form the sides of the square. What is the rotational inertia of this rigid body about an axis that (a) passes through the midpoints of opposite sides and lies in the plane of the square, (b) passes through the midpoint of one of the sides and is perpendicular to the plane of the square, and (c) lies in the plane of the square and passes through two diagonally opposite particles?

**Module 10.6 Torque**

**45 E SSM** The body in Fig. 10.20 is pivoted at  $O$ , and two forces act on it as shown. If  $r_1 = 1.30 \text{ m}$ ,  $r_2 = 2.15 \text{ m}$ ,  $F_1 = 4.20 \text{ N}$ ,  $F_2 = 4.90 \text{ N}$ ,  $\theta_1 = 75.0^\circ$ , and  $\theta_2 = 60.0^\circ$ , what is the net torque about the pivot?

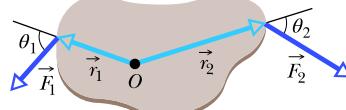


Figure 10.20 Problem 45.

**46 E** The body in Fig. 10.21 is pivoted at  $O$ . Three forces act on it:  $F_A = 10 \text{ N}$  at point  $A$ , 8.0 m from  $O$ ;  $F_B = 16 \text{ N}$  at  $B$ , 4.0 m from  $O$ ; and  $F_C = 19 \text{ N}$  at  $C$ , 3.0 m from  $O$ . What is the net torque about  $O$ ?

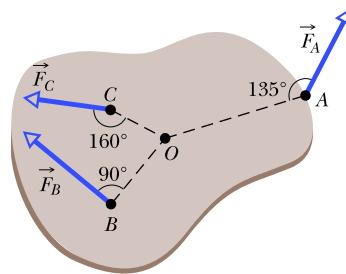


Figure 10.21 Problem 46.

**47 E SSM** A small ball of mass 0.75 kg is attached to one end of a 1.25-m-long massless rod, and the other end of the rod is hung from a pivot. When the resulting pendulum is  $30^\circ$  from the vertical, what is the magnitude of the gravitational torque calculated about the pivot?

**48 E** The length of a bicycle pedal arm is 0.152 m, and a downward force of 111 N is applied to the pedal by the rider. What is

the magnitude of the torque about the pedal arm's pivot when the arm is at angle (a)  $30^\circ$ , (b)  $90^\circ$ , and (c)  $180^\circ$  with the vertical?

### Module 10.7 Newton's Second Law for Rotation

**49 E SSM** During the launch from a board, a diver's angular speed about her center of mass changes from zero to  $6.20 \text{ rad/s}$  in  $220 \text{ ms}$ . Her rotational inertia about her center of mass is  $12.0 \text{ kg} \cdot \text{m}^2$ . During the launch, what are the magnitudes of (a) her average angular acceleration and (b) the average external torque on her from the board?

**50 E** If a  $32.0 \text{ N} \cdot \text{m}$  torque on a wheel causes angular acceleration  $25.0 \text{ rad/s}^2$ , what is the wheel's rotational inertia?

**51 M GO** In Fig. 10.22, block 1 has mass  $m_1 = 460 \text{ g}$ , block 2 has mass  $m_2 = 500 \text{ g}$ , and the pulley, which is mounted on a horizontal axle with negligible friction, has radius  $R = 5.00 \text{ cm}$ . When released from rest, block 2 falls  $75.0 \text{ cm}$  in  $5.00 \text{ s}$  without the cord slipping on the pulley. (a) What is the magnitude of the acceleration of the blocks? What are (b) tension  $T_2$  and (c) tension  $T_1$ ? (d) What is the magnitude of the pulley's angular acceleration? (e) What is its rotational inertia?

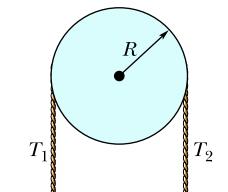


Figure 10.22  
Problems 51 and 83.

**52 M GO** In Fig. 10.23, a cylinder having a mass of  $2.0 \text{ kg}$  can rotate about its central axis through point  $O$ . Forces are applied as shown:  $F_1 = 6.0 \text{ N}$ ,  $F_2 = 4.0 \text{ N}$ ,  $F_3 = 2.0 \text{ N}$ , and  $F_4 = 5.0 \text{ N}$ . Also,  $r = 5.0 \text{ cm}$  and  $R = 12 \text{ cm}$ . Find the (a) magnitude and (b) direction of the angular acceleration of the cylinder. (During the rotation, the forces maintain their same angles relative to the cylinder.)

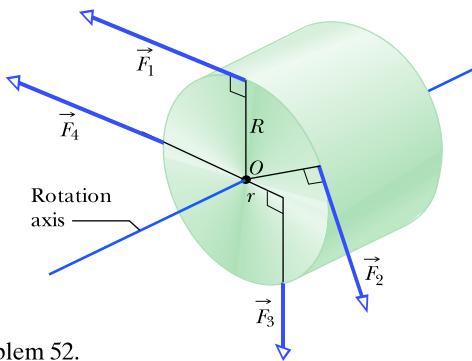


Figure 10.23 Problem 52.

**53 M GO** Figure 10.24 shows a uniform disk that can rotate around its center like a merry-go-round. The disk has a radius of  $2.00 \text{ cm}$  and a mass of  $20.0 \text{ g}$  and is initially at rest. Starting at time  $t = 0$ , two forces are to be applied tangentially to the rim as indicated, so that at time  $t = 1.25 \text{ s}$  the disk has an angular velocity of  $250 \text{ rad/s}$  counterclockwise. Force  $\vec{F}_1$  has a magnitude of  $0.100 \text{ N}$ . What is magnitude  $F_2$ ?



Figure 10.24  
Problem 53.

**54 M BIO FCP** In a judo foot-sweep move, you sweep your opponent's left foot out from under him while pulling on his gi (uniform) toward that side. As a result, your opponent rotates around his right foot and onto the mat. Figure 10.25 shows a simplified diagram of your opponent as you face him, with his left foot swept out. The rotational axis is through point  $O$ .

The gravitational force  $\vec{F}_g$  on him effectively acts at his center of mass, which is a horizontal distance  $d = 28 \text{ cm}$  from point  $O$ . His mass is  $70 \text{ kg}$ , and his rotational inertia about point  $O$  is  $65 \text{ kg} \cdot \text{m}^2$ . What is the magnitude of his initial angular acceleration about point  $O$  if your pull  $\vec{F}_a$  on his gi is (a) negligible and (b) horizontal with a magnitude of  $300 \text{ N}$  and applied at height  $h = 1.4 \text{ m}$ ?

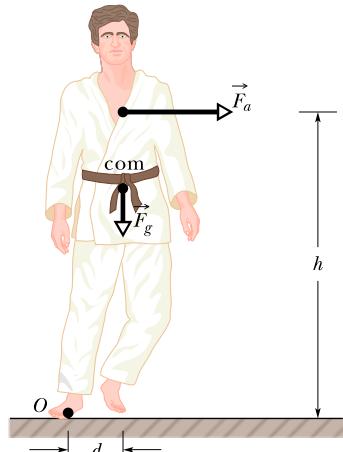


Figure 10.25 Problem 54.

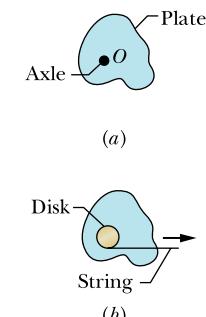


Figure 10.26  
Problem 55.

**55 M GO** In Fig. 10.26a, an irregularly shaped plastic plate with uniform thickness and density (mass per unit volume) is to be rotated around an axle that is perpendicular to the plate face and through point  $O$ . The rotational inertia of the plate about that axle is measured with the following method. A circular disk of mass  $0.500 \text{ kg}$  and radius  $2.00 \text{ cm}$  is glued to the plate, with its center aligned with point  $O$  (Fig. 10.26b). A string is wrapped around the edge of the disk the way a string is wrapped around a top. Then the string is pulled for  $5.00 \text{ s}$ . As a result, the disk and plate are rotated by a constant force of  $0.400 \text{ N}$  that is applied by the string tangentially to the edge of the disk. The resulting angular speed is  $114 \text{ rad/s}$ . What is the rotational inertia of the plate about the axle?

**56 M GO** Figure 10.27 shows particles 1 and 2, each of mass  $m$ , fixed to the ends of a rigid massless rod of length  $L_1 + L_2$ , with  $L_1 = 20 \text{ cm}$  and  $L_2 = 80 \text{ cm}$ . The rod is held horizontally on the fulcrum and then released. What are the magnitudes of the initial accelerations of (a) particle 1 and (b) particle 2?

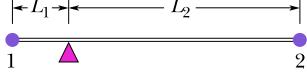


Figure 10.27 Problem 56.

**57 H CALC GO** A pulley, with a rotational inertia of  $1.0 \times 10^{-3} \text{ kg} \cdot \text{m}^2$  about its axle and a radius of  $10 \text{ cm}$ , is acted on by a force applied tangentially at its rim. The force magnitude varies in time as  $F = 0.50t + 0.30t^2$ , with  $F$  in newtons and  $t$  in seconds. The pulley is initially at rest. At  $t = 3.0 \text{ s}$  what are its (a) angular acceleration and (b) angular speed?

### Module 10.8 Work and Rotational Kinetic Energy

**58 E** A uniform disk with mass  $M$  and radius  $R$  is mounted on a fixed horizontal axis. A block with mass  $m$  hangs from a massless cord that is wrapped around the rim of the disk. (a) If  $R = 12 \text{ cm}$ ,  $M = 400 \text{ g}$ , and  $m = 50 \text{ g}$ , find the speed of the block after it has descended  $50 \text{ cm}$  starting from rest. Solve the problem using energy conservation principles. (b) Repeat (a) with  $R = 5.0 \text{ cm}$ .

**59 E** An automobile crankshaft transfers energy from the engine to the axle at the rate of  $100 \text{ hp}$  ( $= 74.6 \text{ kW}$ ) when rotating at a speed of  $1800 \text{ rev/min}$ . What torque (in newton-meters) does the crankshaft deliver?

**60 E** A thin rod of length 0.75 m and mass 0.42 kg is suspended freely from one end. It is pulled to one side and then allowed to swing like a pendulum, passing through its lowest position with angular speed 4.0 rad/s. Neglecting friction and air resistance, find (a) the rod's kinetic energy at its lowest position and (b) how far above that position the center of mass rises.

**61 E** A 32.0 kg wheel, essentially a thin hoop with radius 1.20 m, is rotating at 280 rev/min. It must be brought to a stop in 15.0 s. (a) How much work must be done to stop it? (b) What is the required average power?

**62 M** In Fig. 10.16, three 0.0100 kg particles have been glued to a rod of length  $L = 6.00$  cm and negligible mass and can rotate around a perpendicular axis through point  $O$  at one end. How much work is required to change the rotational rate (a) from 0 to 20.0 rad/s, (b) from 20.0 rad/s to 40.0 rad/s, and (c) from 40.0 rad/s to 60.0 rad/s? (d) What is the slope of a plot of the assembly's kinetic energy (in joules) versus the square of its rotation rate (in radians-squared per second-squared)?

**63 M SSM** A meter stick is held vertically with one end on the floor and is then allowed to fall. Find the speed of the other end just before it hits the floor, assuming that the end on the floor does not slip. (*Hint:* Consider the stick to be a thin rod and use the conservation of energy principle.)

**64 M** A uniform cylinder of radius 10 cm and mass 20 kg is mounted so as to rotate freely about a horizontal axis that is parallel to and 5.0 cm from the central longitudinal axis of the cylinder. (a) What is the rotational inertia of the cylinder about the axis of rotation? (b) If the cylinder is released from rest with its central longitudinal axis at the same height as the axis about which the cylinder rotates, what is the angular speed of the cylinder as it passes through its lowest position?

**65 H GO FCP** A tall, cylindrical chimney falls over when its base is ruptured. Treat the chimney as a thin rod of length 55.0 m. At the instant it makes an angle of 35.0° with the vertical as it falls, what are (a) the radial acceleration of the top, and (b) the tangential acceleration of the top. (*Hint:* Use energy considerations, not a torque.) (c) At what angle  $\theta$  is the tangential acceleration equal to  $g$ ?

**66 H CALC GO** A uniform spherical shell of mass  $M = 4.5$  kg and radius  $R = 8.5$  cm can rotate about a vertical axis on frictionless bearings (Fig. 10.28). A massless cord passes around the equator of the shell, over a pulley of rotational inertia  $I = 3.0 \times 10^{-3}$  kg · m<sup>2</sup> and radius  $r = 5.0$  cm, and is attached to a small object of mass  $m = 0.60$  kg. There is no friction on the pulley's axle; the cord does not slip on the pulley. What is the speed of the object when it has fallen 82 cm after being released from rest? Use energy considerations.

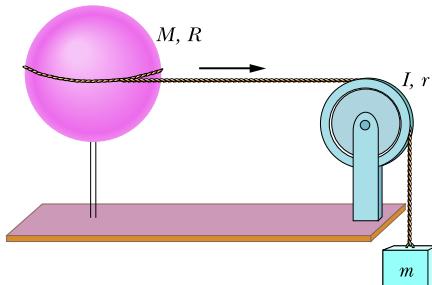


Figure 10.28 Problem 66.

**67 H GO** Figure 10.29 shows a rigid assembly of a thin hoop (of mass  $m$  and radius  $R = 0.150$  m) and a thin radial rod (of mass  $m$  and length  $L = 2.00R$ ). The assembly is upright, but if we give it a slight nudge, it will rotate around a horizontal axis in the plane of the rod and hoop, through the lower end of the rod. Assuming that the energy given to the assembly in such a nudge is negligible, what would be the assembly's angular speed about the rotation axis when it passes through the upside-down (inverted) orientation?

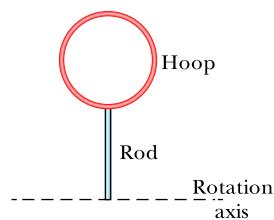


Figure 10.29 Problem 67.

### Additional Problems

**68** Two uniform solid spheres have the same mass of 1.65 kg, but one has a radius of 0.226 m and the other has a radius of 0.854 m. Each can rotate about an axis through its center. (a) What is the magnitude  $\tau$  of the torque required to bring the smaller sphere from rest to an angular speed of 317 rad/s in 15.5 s? (b) What is the magnitude  $F$  of the force that must be applied tangentially at the sphere's equator to give that torque? What are the corresponding values of (c)  $\tau$  and (d)  $F$  for the larger sphere?

**69** In Fig. 10.30, a small disk of radius  $r = 2.00$  cm has been glued to the edge of a larger disk of radius  $R = 4.00$  cm so that the disks lie in the same plane. The disks can be rotated around a perpendicular axis through point  $O$  at the center of the larger disk. The disks both have a uniform density (mass per unit volume) of  $1.40 \times 10^3$  kg/m<sup>3</sup> and a uniform thickness of 5.00 mm. What is the rotational inertia of the two-disk assembly about the rotation axis through  $O$ ?

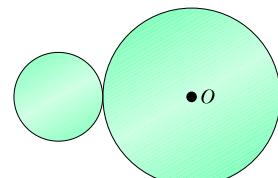


Figure 10.30 Problem 69.

**70** A wheel, starting from rest, rotates with a constant angular acceleration of 2.00 rad/s<sup>2</sup>. During a certain 3.00 s interval, it turns through 90.0 rad. (a) What is the angular velocity of the wheel at the start of the 3.00 s interval? (b) How long has the wheel been turning before the start of the 3.00 s interval?

**71 SSM** In Fig. 10.31, two 6.20 kg blocks are connected by a massless string over a pulley of radius 2.40 cm and rotational inertia  $7.40 \times 10^{-4}$  kg · m<sup>2</sup>. The string does not slip on the pulley; it is not known whether there is friction between the table and the sliding block; the pulley's axis is frictionless. When this system is released from rest, the pulley turns through 0.130 rad in 91.0 ms and the acceleration of the blocks is constant. What are (a) the magnitude of the pulley's angular acceleration, (b) the magnitude of either block's acceleration, (c) string tension  $T_1$ , and (d) string tension  $T_2$ ?

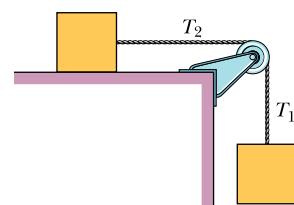


Figure 10.31 Problem 71.

**72** Attached to each end of a thin steel rod of length 1.20 m and mass 6.40 kg is a small ball of mass 1.06 kg. The rod is constrained to rotate in a horizontal plane about a vertical axis through its midpoint. At a certain instant, it is rotating at

39.0 rev/s. Because of friction, it slows to a stop in 32.0 s. Assuming a constant retarding torque due to friction, compute (a) the angular acceleration, (b) the retarding torque, (c) the total energy transferred from mechanical energy to thermal energy by friction, and (d) the number of revolutions rotated during the 32.0 s. (e) Now suppose that the retarding torque is known not to be constant. If any of the quantities (a), (b), (c), and (d) can still be computed without additional information, give its value.

**73 CALC** A uniform helicopter rotor blade is 7.80 m long, has a mass of 110 kg, and is attached to the rotor axle by a single bolt. (a) What is the magnitude of the force on the bolt from the axle when the rotor is turning at 320 rev/min? (*Hint:* For this calculation the blade can be considered to be a point mass at its center of mass. Why?) (b) Calculate the torque that must be applied to the rotor to bring it to full speed from rest in 6.70 s. Ignore air resistance. (The blade cannot be considered to be a point mass for this calculation. Why not? Assume the mass distribution of a uniform thin rod.) (c) How much work does the torque do on the blade in order for the blade to reach a speed of 320 rev/min?

**74 Racing disks.** Figure 10.32 shows two disks that can rotate about their centers like a merry-go-round. At time  $t = 0$ , the reference lines of the two disks have the same orientation. Disk A is already rotating, with a constant angular velocity of 9.5 rad/s. Disk B has been stationary but now begins to rotate at a constant angular acceleration of 2.2 rad/s<sup>2</sup>. (a) At what time  $t$  will the reference lines of the two disks momentarily have the same angular displacement  $\theta$ ? (b) Will that time  $t$  be the first time since  $t = 0$  that the reference lines are momentarily aligned?

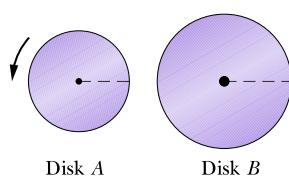


Figure 10.32 Problem 74.

**75 BIO FCP** A high-wire walker always attempts to keep his center of mass over the wire (or rope). He normally carries a long, heavy pole to help: If he leans, say, to his right (his com moves to the right) and is in danger of rotating around the wire, he moves the pole to his left (its com moves to the left) to slow the rotation and allow himself time to adjust his balance. Assume that the walker has a mass of 70.0 kg and a rotational inertia of 15.0 kg · m<sup>2</sup> about the wire. What is the magnitude of his angular acceleration about the wire if his com is 5.0 cm to the right of the wire and (a) he carries no pole and (b) the 14.0 kg pole he carries has its com 10 cm to the left of the wire?

**76** Starting from rest at  $t = 0$ , a wheel undergoes a constant angular acceleration. When  $t = 2.0$  s, the angular velocity of the wheel is 5.0 rad/s. The acceleration continues until  $t = 20$  s, when it abruptly ceases. Through what angle does the wheel rotate in the interval  $t = 0$  to  $t = 40$  s?

**77 SSM** A record turntable rotating at  $33\frac{1}{3}$  rev/min slows down and stops in 30 s after the motor is turned off. (a) Find its (constant) angular acceleration in revolutions per minute-squared. (b) How many revolutions does it make in this time?

**78 GO** A rigid body is made of three identical thin rods, each with length  $L = 0.600$  m, fastened together in the form of a letter **H** (Fig. 10.33). The body is free to rotate about a horizontal axis

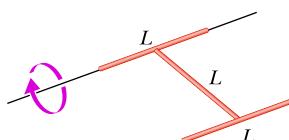


Figure 10.33 Problem 78.

that runs along the length of one of the legs of the **H**. The body is allowed to fall from rest from a position in which the plane of the **H** is horizontal. What is the angular speed of the body when the plane of the **H** is vertical?

**79 SSM** (a) Show that the rotational inertia of a solid cylinder of mass  $M$  and radius  $R$  about its central axis is equal to the rotational inertia of a thin hoop of mass  $M$  and radius  $R/\sqrt{2}$  about its central axis. (b) Show that the rotational inertia  $I$  of any given body of mass  $M$  about any given axis is equal to the rotational inertia of an *equivalent hoop* about that axis, if the hoop has the same mass  $M$  and a radius  $k$  given by

$$k = \sqrt{\frac{I}{M}}.$$

The radius  $k$  of the equivalent hoop is called the *radius of gyration* of the given body.

**80** A disk rotates at constant angular acceleration, from angular position  $\theta_1 = 10.0$  rad to angular position  $\theta_2 = 70.0$  rad in 6.00 s. Its angular velocity at  $\theta_2$  is 15.0 rad/s. (a) What was its angular velocity at  $\theta_1$ ? (b) What is the angular acceleration? (c) At what angular position was the disk initially at rest? (d) Graph  $\theta$  versus time  $t$  and angular speed  $\omega$  versus  $t$  for the disk, from the beginning of the motion (let  $t = 0$  then).

**81 GO** The thin uniform rod in Fig. 10.34 has length 2.0 m and can pivot about a horizontal, frictionless pin through one end. It is released from rest at angle  $\theta = 40^\circ$  above the horizontal. Use the principle of conservation of energy to determine the angular speed of the rod as it passes through the horizontal position.

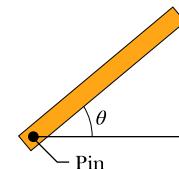


Figure 10.34  
Problem 81.

**82 FCP** George Washington Gale Ferris, Jr., a civil engineering graduate from Rensselaer Polytechnic Institute, built the original Ferris wheel for the 1893 World's Columbian Exposition in Chicago. The wheel, an astounding engineering construction at the time, carried 36 wooden cars, each holding up to 60 passengers, around a circle 76 m in diameter. The cars were loaded 6 at a time, and once all 36 cars were full, the wheel made a complete rotation at constant angular speed in about 2 min. Estimate the amount of work that was required of the machinery to rotate the passengers alone.

**83** In Fig. 10.22, two blocks, of mass  $m_1 = 400$  g and  $m_2 = 600$  g, are connected by a massless cord that is wrapped around a uniform disk of mass  $M = 500$  g and radius  $R = 12.0$  cm. The disk can rotate without friction about a fixed horizontal axis through its center; the cord cannot slip on the disk. The system is released from rest. Find (a) the magnitude of the acceleration of the blocks, (b) the tension  $T_1$  in the cord at the left, and (c) the tension  $T_2$  in the cord at the right.

**84 Newton's second law for rotation.** Figure 10.35 shows a uniform disk, with mass  $M = 2.5$  kg and radius  $R = 20$  cm, mounted on a fixed horizontal axle. A block with mass  $m = 1.2$  kg hangs from a massless cord that is wrapped around the rim of the disk. Find (a) the acceleration of the falling block, (b) the tension in the cord, and (c) the angular acceleration of the disk. The cord does not slip, and there is no friction at the axle.

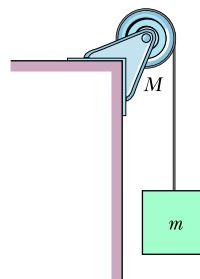


Figure 10.35  
Problem 84.

**85** *Earth's rotation rate, then and now.* Research with an extinct type of clams that lived 70 million years ago involves the daily growth rings that formed on the shells. Measurements reveal that the day back then was 23.5 hours long. (a) In radians per hour, what is Earth's current rate of rotation  $\omega$ ? (b) What was it back then? (c) Back then, how many days were in a year, the time Earth takes to make a complete revolution about the Sun?

**86 BIO CALC** *Bone screw insertion.* An increasingly common method of surgically stabilizing a broken bone is by inserting a screw into the bone with an automated surgical screwdriver. As the screw enters the bone, the medical team monitors the torque applied to the screw. The purpose is to drive the screw inward until the screw head meets the bone and then to rotate the screw a bit more to tighten the screw threads against the bone threads that the screw has cut along its path. The danger is to tighten the screw too much because then the screw threads destroy (*strip*) the bone threads. Figure 10.36 shows an idealized plot of torque magnitude  $\tau$  versus angle of rotation  $\theta$ , all the way to the failure stage. Initially, as more of the screw enters the bone, the required torque increases until it reaches a short plateau at  $\tau_{\text{plateau}} = 0.10 \text{ N} \cdot \text{m}$ , which occurs as the head makes contact. Then the torque sharply increases as the screw tightens. The surgical team would like to stop at or near the peak at  $\tau_{\text{peak}} = 1.7 \text{ N} \cdot \text{m}$  and avoid passing into the failure region. They might be able to predict the peak from the plateau and the work done on the screw by the screwdriver. (a) What multiple of the plateau torque gives the peak torque? (b) How much work is done from the left end of the plot to the peak?

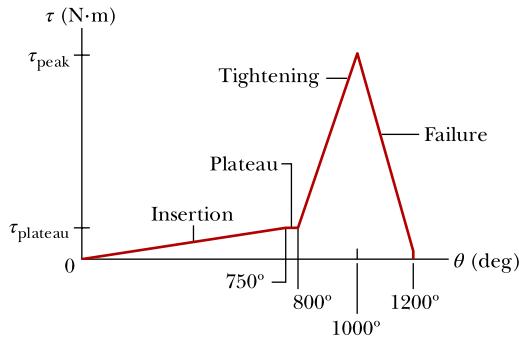


Figure 10.36 Problem 86.

**87** *Pulsars.* When a star with a mass at least ten times that of the Sun explodes outward in a supernova, its core can be collapsed into a pulsar, which is a spinning star that emits electromagnetic radiation (radio waves or light) in two tight beams in opposite directions. If a beam sweeps across Earth during the rotation, we can detect repeated pulses of the radiation, one per revolution. (a) The first pulsar was discovered by Jocelyn Bell Burnell and Antony Hewish in 1967; its pulses are separated by 1.3373 s. What is its angular speed in revolutions per second? (b) To date, the fastest spinning pulsar has an angular speed of 716 rev/s. What is the separation of its detected pulses in milliseconds?

**88** *Fastest spinning star.* The star VFTS102 in the Large Magellanic Cloud (a satellite galaxy to our Milky Way Galaxy) is spinning so fast that it exceeds traditional expectations. The star has 25 times the mass of the Sun and if we consider it to be a solid rotating sphere, the surface at the equator is moving at a speed of  $2.0 \times 10^6 \text{ km/h}$ . To find its radius, assume that it has

the same density as the Sun. What are (a) the star's radius, (b) its rotational period, and (c) the magnitude of the centripetal acceleration of a section on the equatorial surface?

**89** *Rod rotation.* Figure 10.37 shows a 2.0 kg uniform rod that is 3.0 m long. The rod is mounted to rotate freely about a horizontal axis perpendicular to the rod that passes through a point 1.0 m from one end of the rod. The rod is released from rest when it is horizontal. (a) What is its angular acceleration just then? (b) If the rod's mass were increased, would the answer increase, decrease, or stay the same?

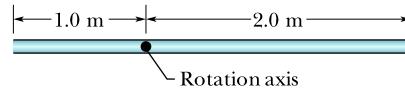


Figure 10.37 Problem 89.

**90 BIO** *Ballet en pointe.* When a ballerina stands en pointe, her weight is supported by only the tips of her toes in a rigid toe box in her shoes (Fig. 10.38a). Her center of mass must be directly above the toes, but that positioning is difficult to maintain. To see how her height affects the balancing, treat her as a uniform rod of length  $L$  that is balanced vertically on one end (Fig. 10.38b). (a) What is the angular acceleration  $\alpha$  around that end if the rod leans by a small angle  $\theta$  from the vertical? (b) For a given angle, is  $\alpha$  larger or smaller for a taller ballerina? (Does a taller ballerina have less time or more time to correct an imbalance?)

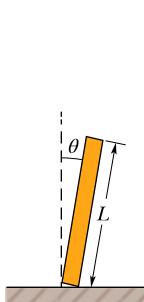


Figure 10.38 Problem 90.

**91** *Different rotation axes.* Five particles, positioned in the  $xy$  plane according to the following table, form a rigidly connected body. What is the rotational inertia of the body about (a) the  $x$  axis, (b) the  $y$  axis, and (c) the  $z$  axis? (d) Where is the center of mass of the body?

Object	1	2	3	4	5
Mass (g)	500	400	300	600	450
$x$ (cm)	15	-13	17	-4.0	-5.0
$y$ (cm)	20	13	-6.0	-7.0	9.0

**92 BIO** *The Michael Jackson lean.* In his music video "Smooth Criminal," Michael Jackson planted his feet on the stage and then leaned forward rigidly by  $45^\circ$ , seemingly defying the gravitational force because his center of mass was then well forward of his supporting feet (Fig. 10.39a). The secret was in the shoes patented by Jackson: Each heel had a vee-shaped notch that he caught on a nail head slightly protruding from the stage. Once the heels were snagged on the nail heads, he could lean forward

without toppling. The rotation axis was through each nail head, which was just below an ankle. The feat required tremendous leg strength, particularly of the Achilles tendon that connects the calf muscle (at distance  $d = 40$  cm from the ankle) to the heel (Fig. 10.39b). That tendon is at  $\phi = 5.0^\circ$  from the leg bone and from the rigid orientation of Jackson's body. Jackson's mass  $m$  was 60 kg, his height  $h$  was 1.75 m, and his center of mass was  $0.56h$  from his ankle. What was the tension  $T$  in the tendon when Jackson's body was at  $\theta = 45^\circ$  from the stage?

Entertainment Pictures/Zuma Press



(a)

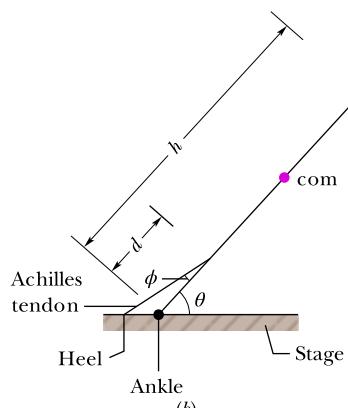


Figure 10.39 Problem 92.

**93 Strobing.** A disk that rotates clockwise at angular speed  $10\pi$  rad/s is illuminated by only a flashing stroboscopic light. The flashes reveal a small black dot on the rim of the disk. In the first flash, the dot appears at the 12:00 position (as on an analog clock face). Where does it appear in the next five flashes if the time between flashes is (a) 0.20 s, (b) 0.050 s, and (c) 40 ms?

**94 Roundabout management.** Figure 10.40 shows an overhead view of a single-lane roundabout where access is computer controlled. Car 1 is allowed to enter at access point  $A$  at time  $t = 0$ . It accelerates at  $a = 3.0 \text{ m/s}^2$  to the speed limit of  $v = 13.4 \text{ m/s}$  as it moves around the roundabout and past access point  $B$  where car 2 waits to enter. The radius  $R$  of the circular road is 45 m, the angle  $\theta$  subtended between  $A$  and  $B$  is  $120^\circ$ , and both cars have length  $L = 4.5 \text{ m}$ . Car 2 is allowed to enter when the rear of car 1 is 2.0 car lengths past point  $B$ . At what time  $t$  is car 2 released?

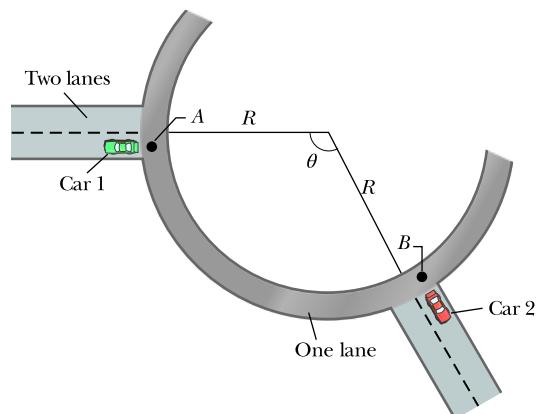


Figure 10.40 Problem 94.

**95 BIO Grip.** In the engineering design of handles (such as on manual and powered hand tools) and rails (such as for stairways), the grip and possible slip of a hand must be considered. If a hand grips a cylindrical handle with a diameter of 30 mm and with a normal force on the hand at 150 N, what is the maximum tangential frictional torque when the coefficient of static friction is 0.25?

**96 BIO Wheelchair work.** A manual (nonmotorized) wheelchair (Fig. 10.41) is propelled over level ground when the person forces the hand rim to rotate forward. Suppose that the rim has a diameter  $D$  of 0.55 m, the forward rotation  $\Delta\theta$  of each push is  $88^\circ$ , the average tangential force  $F_{\text{avg}}$  on the rim during a push is 39 N, the time  $\Delta t$  for a push is 0.38 s, and the frequency  $f$  of pushing is 53 pushes per minute. How much work is done in (a) each push and (b) 3.0 min? What is the average power output in (c) each push and (d) 3.0 min?



Figure 10.41 Problem 96.

MachineHeadz/Stock/Gett Images

**97 Coin on turntable.** A coin is placed a distance  $R$  from the center of a phonograph turntable. The coefficient of static friction is  $\mu_s$ . The angular spin of the turntable is slowly increased. When it reaches  $\omega_0$ , the coin is on the verge of sliding off. Find  $\omega_0$  in terms of  $\mu_s$ ,  $R$ , and  $g$ .