

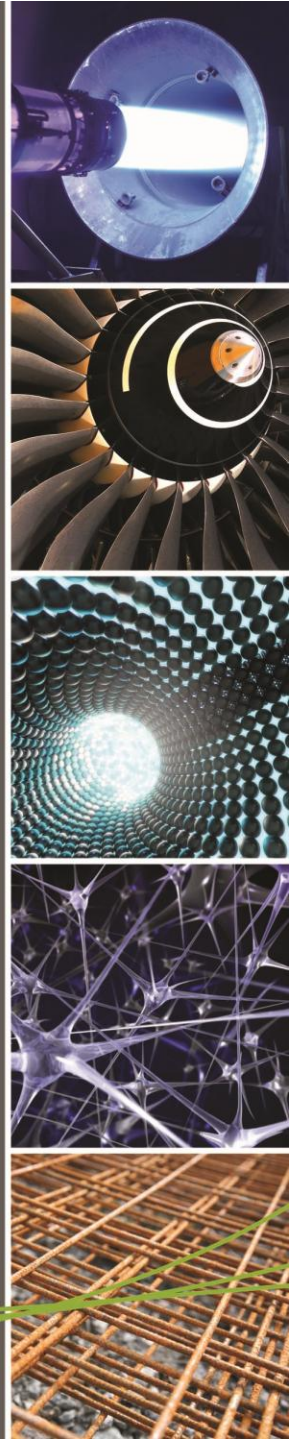


Swansea University  
Prifysgol Abertawe

# Advanced Structural Analysis

## EGF316

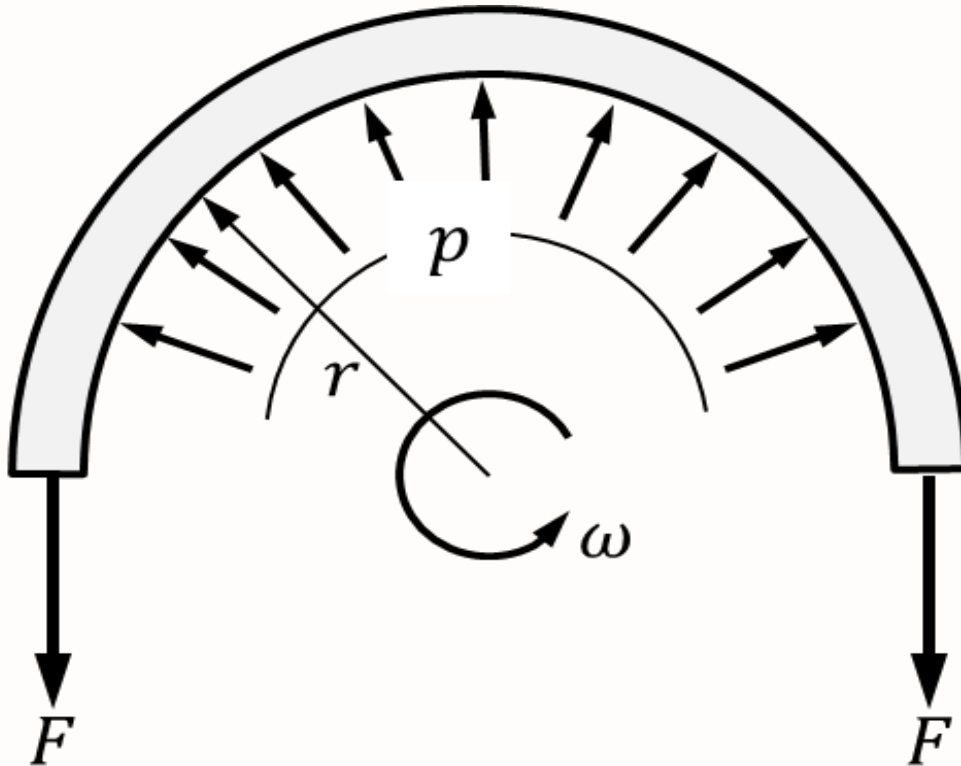
### Rotating Discs



# Lecture Content

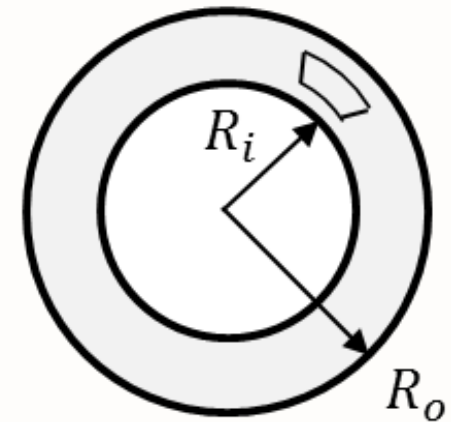
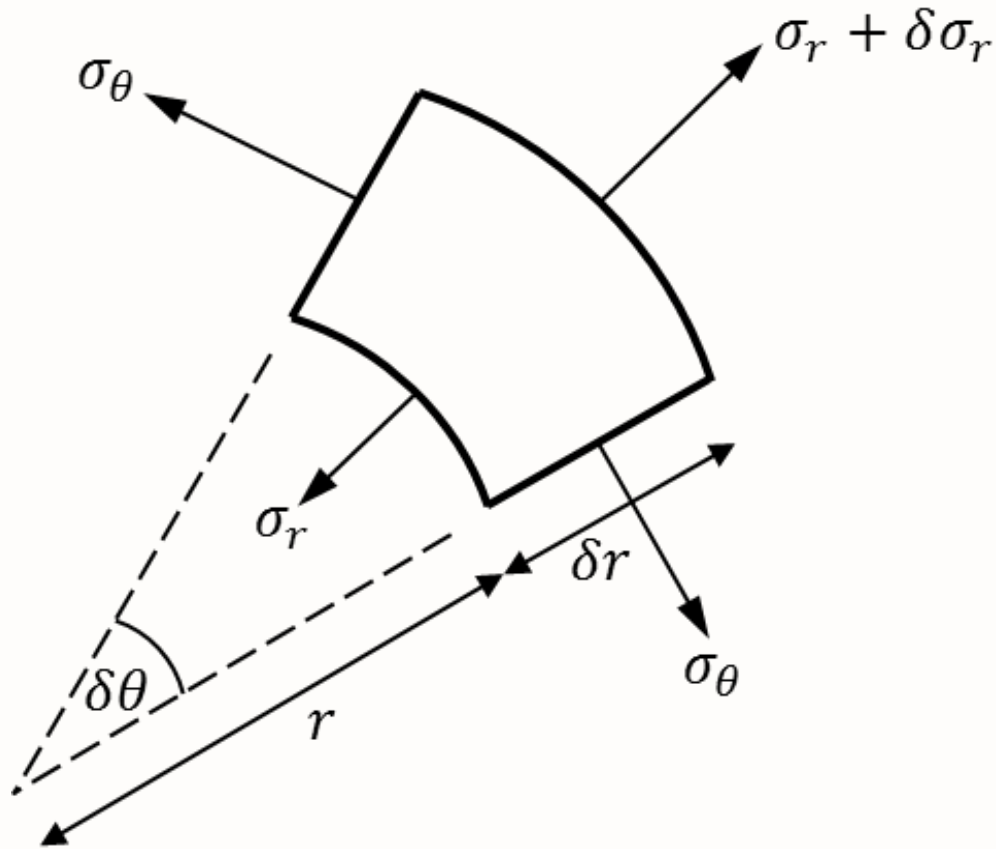
- Thin rotating rings or cylinder
- Rotating discs
- Solid circular disc of uniform thickness
- Circular disc of uniform thickness with a central hole
- Maximum stresses
- Effect of turbine blades on a rotating disc

# Thin Rotating Ring or Cylinder



$$\sigma_{\theta} = \rho \omega^2 r^2$$

# Rotating Disc



# Rotating Solid Disc

$$\sigma_r = A - \frac{B}{r^2} - \frac{(3 + \nu)}{8} \rho \omega^2 r^2$$

$$\sigma_\theta = A + \frac{B}{r^2} - \frac{(1 + 3\nu)}{8} \rho \omega^2 r^2$$

# Solid Circular Disc

$$\sigma_r = \frac{(3 + \nu)}{8} \rho \omega^2 (R_o^2 - r^2)$$

$$\sigma_\theta = \frac{\rho \omega^2}{8} [(3 + \nu)R_o^2 - (1 + 3\nu)r^2]$$

# Circular Disc with Central Hole

$$\sigma_r = (3 + \nu) \frac{\rho \omega^2}{8} \left[ R_i^2 + R_o^2 - \frac{R_i^2 R_o^2}{r^2} - r^2 \right]$$

$$\sigma_\theta = \frac{\rho \omega^2}{8} \left[ (3 + \nu) \left[ R_i^2 + R_o^2 + \frac{R_i^2 R_o^2}{r^2} \right] - (1 + 3\nu) r^2 \right]$$

# Example 1:

A steel ring of outer diameter 300mm and internal diameter 200mm is shrunk onto a solid steel shaft.

The interface is such that the radial pressure between the mating surfaces remains above  $30\text{MN/m}^2$  at all times whilst the assembly rotates.

The circumferential stress on the inside surface of the ring must not exceed  $240\text{MN/m}^2$

Determine the maximum speed at which the assembly can rotate.

You may assume that  $\rho = 7500\text{kg/m}^3$ ,  $\nu = 0.3$ ,  $E = 210\text{GPa}$ .



# Maximum Hoop Stress

$$\sigma_{\theta} = \frac{\rho\omega^2}{8} \left[ (3 + \nu) \left[ R_i^2 + R_o^2 + \frac{R_i^2 R_o^2}{r^2} \right] - (1 + 3\nu)r^2 \right]$$

Maximum when  $r = R_i$ :

$$\hat{\sigma}_{\theta} = \frac{\rho\omega^2}{4} \left[ (3 + \nu)R_o^2 + (1 - \nu)R_i^2 \right]$$

As  $R_i \rightarrow 0$ :

$$\hat{\sigma}_{\theta} = \frac{\rho\omega^2}{4} (3 + \nu)R_o^2$$

# Comparison:

Solid disc:

$$\sigma_{\theta} = \frac{\rho\omega^2}{8} [(3 + \nu)R_o^2 - (1 + 3\nu)r^2]$$

At centre of when  $r = 0$ :

$$\sigma_{\theta} = \frac{\rho\omega^2}{8} (3 + \nu)R_o^2$$

Disc with central hole:

$$\sigma_{\theta} = \frac{\rho\omega^2}{4} (3 + \nu)R_o^2$$

# Maximum Radial Stress

$$\sigma_r = (3 + \nu) \frac{\rho \omega^2}{8} \left[ R_i^2 + R_o^2 - \frac{R_i^2 R_o^2}{r^2} - r^2 \right]$$

Max when:

$$\frac{d\sigma_r}{dr} = 0 \quad \rightarrow \quad r = \sqrt{R_i R_o}$$

$$\hat{\sigma}_r = (3 + \nu) \frac{\rho \omega^2}{8} (R_o - R_i)^2$$