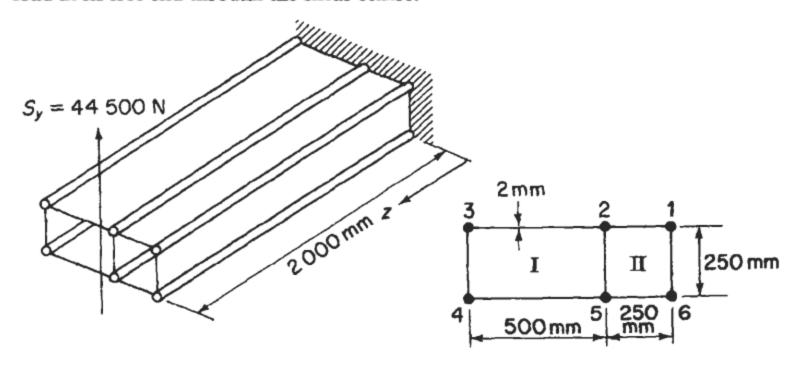
Calculate the deflection at the free end of the two-cell beam shown in Fig. 10.39 allowing for both bending and shear effects. The booms carry all the direct stresses while the skin panels, of constant thickness throughout, are effective only in shear.

Take 
$$E = 69\,000\,\text{N/mm}^2$$
 and  $G = 25\,900\,\text{N/mm}^2$   
Boom areas:  $-B_1 = B_3 = B_4 = B_6 = 650\,\text{mm}^2$ ,  $B_2 = B_5 = 1300\,\text{mm}^2$ 

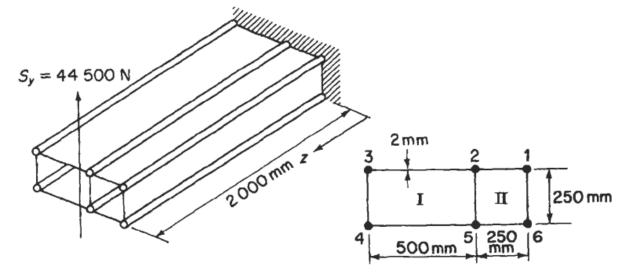
The beam cross-section is symmetrical about a horizontal axis and carries a vertical load at its free end through the shear centre.



Calculate the deflection at the free end of the two-cell beam shown in Fig. 10.39 allowing for both bending and shear effects. The booms carry all the direct stresses while the skin panels, of constant thickness throughout, are effective only in shear.

Take 
$$E = 69\,000\,\text{N/mm}^2$$
 and  $G = 25\,900\,\text{N/mm}^2$   
Boom areas:  $-B_1 = B_3 = B_4 = B_6 = 650\,\text{mm}^2$ ,  $B_2 = B_5 = 1300\,\text{mm}^2$ 

The beam cross-section is symmetrical about a horizontal axis and carries a vertical load at its free end through the shear centre.



Equação de deslocamento (Princípio das Forças Virtuais)

$$\Delta = \Delta_T + \Delta_M + \Delta_S$$

$$\Delta_T = \int \frac{T\delta T}{GJ} dz \quad \Delta_S = \int \left[ \int \frac{q\delta q}{Gt} ds \right] dz$$

$$\Delta_M = \frac{1}{E(I_{xx}I_{yy} - I_{xy}^2)^2} \left[ \Delta_{xx} + \Delta_{yy} + \Delta_{xy_1} + \Delta_{xy_2} \right]$$

$$\Delta_{xx} = \int (\delta M_x I_{yy} - \delta M_y I_{xy}) (M_x I_{yy} - M_y I_{xy}) I_{xx} dz$$

$$\Delta_{yy} = \int (\delta M_y I_{xx} - \delta M_x I_{xy}) (M_y I_{xx} - M_x I_{xy}) I_{yy} dz$$

$$\Delta_{xy_1} = \int (\delta M_y I_{xx} - \delta M_x I_{xy}) (M_x I_{yy} - M_y I_{xy}) I_{xy} dz$$

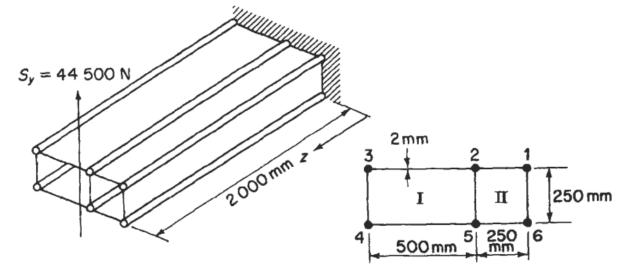
$$\Delta_{xy_2} = \int (\delta M_x I_{yy} - \delta M_y I_{xy}) (M_y I_{xx} - M_x I_{xy}) I_{xy} dz$$

Calculate the deflection at the free end of the two-cell beam shown in Fig. 10.39 allowing for both bending and shear effects. The booms carry all the direct stresses while the skin panels, of constant thickness throughout, are effective only in shear.

Take 
$$E = 69\,000\,\text{N/mm}^2$$
 and  $G = 25\,900\,\text{N/mm}^2$ 

Boom areas: 
$$-B_1 = B_3 = B_4 = B_6 = 650 \,\mathrm{mm}^2$$
,  $B_2 = B_5 = 1300 \,\mathrm{mm}^2$ 

The beam cross-section is symmetrical about a horizontal axis and carries a vertical load at its free end through the shear centre.



Equação de deslocamento (Princípio das Forças Virtuais)

$$\Delta = \Delta_M + \Delta_S$$

$$\Delta_S = \int \left[ \int \frac{q \delta q}{G t} ds \right] dz$$

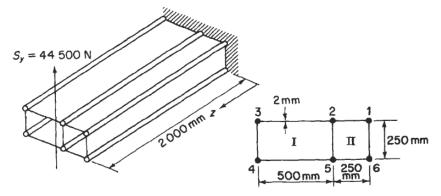
$$\Delta_{M} = \int \left[ \frac{M_{x} \delta M_{x}}{E I_{xx}} + \frac{M_{y} \delta M_{y}}{E I_{yy}} \right] dz$$

Calculate the deflection at the free end of the two-cell beam shown in Fig. 10.39 allowing for both bending and shear effects. The booms carry all the direct stresses while the skin panels, of constant thickness throughout, are effective only in shear.

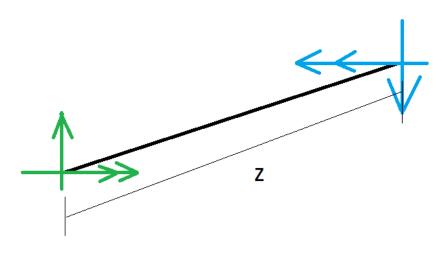
Take 
$$E = 69\,000\,\text{N/mm}^2$$
 and  $G = 25\,900\,\text{N/mm}^2$ 

Boom areas: 
$$-B_1 = B_3 = B_4 = B_6 = 650 \,\mathrm{mm}^2$$
,  $B_2 = B_5 = 1300 \,\mathrm{mm}^2$ 

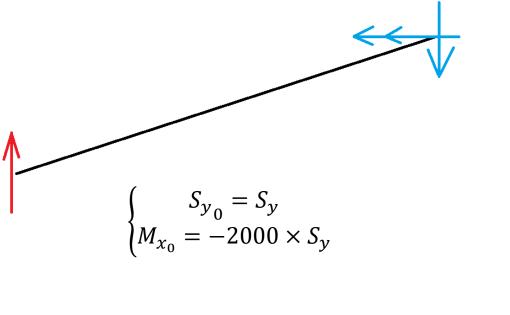
The beam cross-section is symmetrical about a horizontal axis and carries a vertical load at its free end through the shear centre.



Passo 2: Análise da seção



### Passo 1: Diagrama de corpo livre



$$\begin{cases} S_y(z) = S_{y_0} = S_y \\ M_x(z) = M_{x_0} + S_{y_0} \times z = -2000 \times S_y + S_y \times z = -S_y(2000 - z) \end{cases}$$

$$\begin{cases} S_y(z) = S_y \\ M_x(z) = -S_y(2000 - z) \end{cases}$$

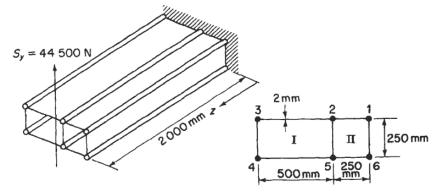
$$\Delta = \int \left\{ \left[ \int \frac{q \delta q}{G t} ds \right] + \frac{M_x \delta M_x}{E I_{xx}} \right\} dz$$

Calculate the deflection at the free end of the two-cell beam shown in Fig. 10.39 allowing for both bending and shear effects. The booms carry all the direct stresses while the skin panels, of constant thickness throughout, are effective only in shear.

Take 
$$E = 69\,000 \,\text{N/mm}^2$$
 and  $G = 25\,900 \,\text{N/mm}^2$ 

Boom areas: 
$$-B_1 = B_3 = B_4 = B_6 = 650 \,\mathrm{mm}^2$$
,  $B_2 = B_5 = 1300 \,\mathrm{mm}^2$ 

The beam cross-section is symmetrical about a horizontal axis and carries a vertical load at its free end through the shear centre.



Passo 3: Análise do Carregamento

$$\begin{cases} S_{y}(z) = S_{y} \\ M_{x}(z) = -S_{y}(2000 - z) \end{cases} \qquad \Delta = \int \left\{ \left[ \int \frac{q \delta q}{G t} ds \right] + \frac{M_{x} \delta M_{x}}{E I_{xx}} \right\} dz$$

## Passo 3.1: Contribuição do Momento

a) Propriedades geométricas

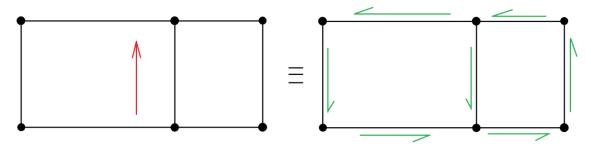
$$I_{xx} = 4 \times 650 \times 125^2 + 2 \times 1300 \times 125^2 \rightarrow$$
  
  $\rightarrow I_{xx} = 81.25 \times 10^6 \text{ mm}^4$ 

$$\begin{split} \Delta_{\mathrm{M}} &= \int \frac{M_{x} \delta M_{x}}{E I_{xx}} dz = \int_{0}^{2000} \frac{\left[ -S_{y}(2000-z) \right] \left[ -\delta S_{y}(2000-z) \right]}{E I_{xx}} dz = S_{y} \delta S_{y} \int_{0}^{2000} \frac{(2000-z)^{2}}{69 \times 10^{3} \times 81.25 \times 10^{6}} dz \rightarrow \\ \Delta_{\mathrm{M}} &= \frac{S_{y} \delta S_{y}}{69 \times 10^{3} \times 81.25 \times 10^{6}} \int_{0}^{2000} (2000-z)^{2} dz = \frac{S_{y} \delta S_{y}}{69 \times 10^{3} \times 81.25 \times 10^{6}} \left[ -\frac{1}{3} (2000-z)^{3} \right] \Big|_{0}^{2000} = \\ \Delta_{\mathrm{M}} &= \frac{S_{y} \delta S_{y} \times 8}{3 \times 69 \times 81.25} = 4.76 \times 10^{-4} \times S_{y} \delta S_{y} = 4.76 \times 10^{-4} \times 44500 = \\ \Delta_{\mathrm{M}} &= 21.17 \; \mathrm{mm} \end{split}$$

## Passo 3: Análise do Carregamento

$$\begin{cases} S_{y}(z) = S_{y} \\ M_{x}(z) = -S_{y}(2000 - z) \end{cases} \qquad \Delta = \int \left\{ \left[ \int \frac{q \delta q}{G t} ds \right] + \frac{M_{x} \delta M_{x}}{E I_{xx}} \right\} dz$$

## Passo 3.2: Contribuição do Cisalhamento



Dado que os booms não se encontram inclinados, nota-se que toda a carga cisalhante será suportada apenas pela casca

## a) Cálculo do Fluxo

$$q_{s} = -\frac{S_{x_{cis}}.I_{xx} - S_{y_{cis}}I_{xy}}{I_{xx}I_{yy} - I_{xy}^{2}}.\left(\int_{0}^{s}t_{d}xds + \sum_{r=1}^{n}B_{r}x_{r}\right) - \frac{S_{y_{cis}}.I_{yy} - S_{x_{cis}}I_{xy}}{I_{xx}I_{yy} - I_{xy}^{2}}.\left(\int_{0}^{s}t_{d}yds + \sum_{r=1}^{n}B_{r}y_{r}\right) + q_{s,0}$$

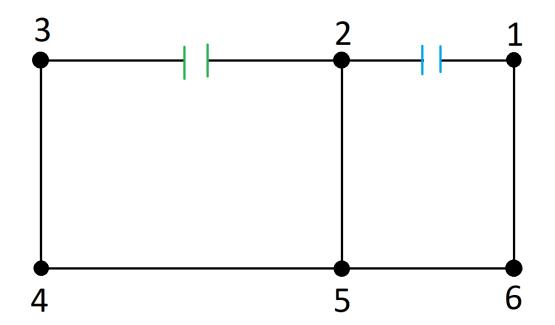
$$q_s = -\frac{S_{y_{cis}}}{I_{xx}} \cdot \left(\sum_{r=1}^n B_r y_r\right) + q_{s,0} = -\frac{S_y}{I_{xx}} \cdot \left(\sum_{r=1}^n B_r y_r\right) + q_{s,0}$$

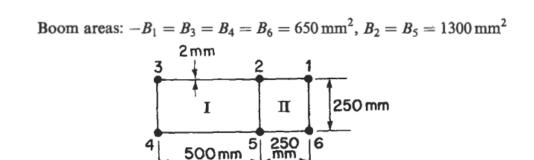
Passo 3: Análise do Carregamento

$$\begin{cases} S_{y}(z) = S_{y} \\ M_{x}(z) = -S_{y}(2000 - z) \end{cases} \qquad \Delta = \int \left\{ \left[ \int \frac{q \delta q}{G t} ds \right] + \frac{M_{x} \delta M_{x}}{E I_{xx}} \right\} dz$$

a) Cálculo do Fluxo

$$q_s = -\frac{S_{y_{cis}}}{I_{xx}} \cdot \left(\sum_{r=1}^n B_r y_r\right) + q_{s,0} = -\frac{S_y}{I_{xx}} \cdot \left(\sum_{r=1}^n B_r y_r\right) + q_{s,0}$$

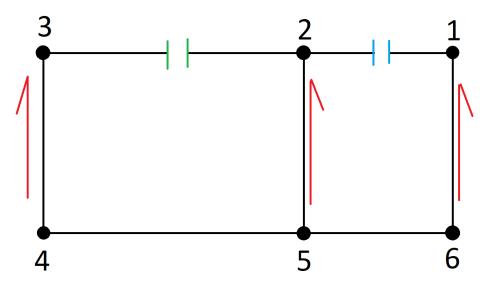




Segmento	$q_b = -\frac{S_y}{81.25e6} \cdot \left(\sum_{r=1}^n B_r y_r\right)$	$q_{s,0}$
1 - 2	0	
2 - 3	0	
2 - 5	$-0.002 \times Sy$	
3 - 4	$-0.001 \times Sy$	
4 - 5	0	
5 - 6	0	
6 – 1	$0.001 \times S_y$	

Passo 3.2: Contribuição do Cisalhamento

a) Cálculo do Fluxo



Segmento	$q_b = -\frac{S_y}{81.25e6} \cdot \left(\sum_{r=1}^n B_r y_r\right)$	$q_{s,0}$
1 – 2	0	
2 - 3	0	
2 – 5	$-0.002 \times Sy$	
3 - 4	$-0.001 \times Sy$	
4 - 5	0	
5 – 6	0	
6 – 1	$0.001 \times S_y$	

Cálculo dos 
$$q_{s,0,i}$$

Análise da Célula I

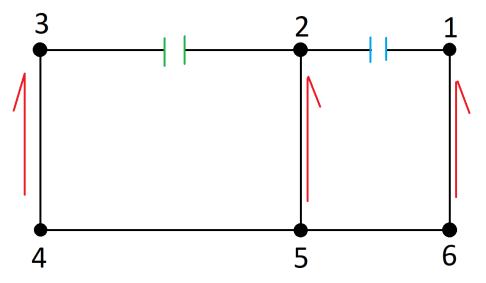
$$\frac{d\theta}{dz} = \frac{1}{2A_r} \oint q \frac{ds}{Gt} = \frac{1}{2A_r Gt} \oint q ds =$$

$$= \frac{1}{2A_rGt} \left[ q_{s,0,1} + \left( q_{s,0,1} - 0.002 \times Sy - q_{s,0,2} \right) + q_{s,0,1} + q_{s,0,1} + 0.001 \times S_y \right] \times 250 =$$

$$= \frac{1}{2 \times 250 \times 25900 \times 2} \left[ 4q_{s,0,1} - q_{s,0,2} - 0.001 \times S_y \right] =$$

$$\frac{d\theta}{dz} = 3.86 \times 10^{-8} (4q_{s,0,1} - q_{s,0,2} - 0.001 \times S_y)$$

a) Cálculo do Fluxo



Segmento	$q_b = -\frac{S_y}{81.25e6} \cdot \left(\sum_{r=1}^n B_r y_r\right)$	$q_{s,0}$
1 – 2	0	
2 - 3	0	
2 - 5	$-0.002 \times Sy$	
3 - 4	$-0.001 \times Sy$	
4 – 5	0	
5 - 6	0	
6 – 1	$0.001 \times S_y$	

Cálculo dos 
$$q_{s,0,i}$$

Análise da Célula II

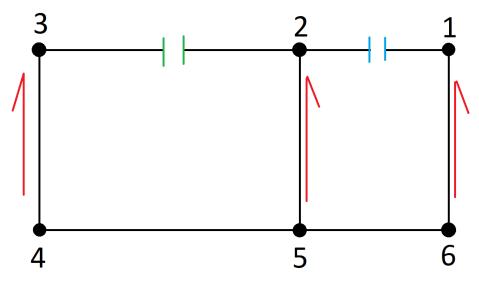
$$\frac{d\theta}{dz} = \frac{1}{2A_r} \oint q \frac{ds}{Gt} = \frac{1}{2A_r Gt} \oint q ds =$$

$$= \frac{1}{2A_rGt} \left[ 2q_{s,0,2} + \left( q_{s,0,2} - 0.001 \times Sy \right) + 2q_{s,0,2} + \left( q_{s,0,2} + 0.002 \times Sy - q_{s,0,1} \right) \right] \times 250 =$$

$$= \frac{1}{2 \times 500 \times 25900 \times 2} \left[ 6q_{s,0,2} - q_{s,0,1} + 0.001 \times S_y \right] =$$

$$\frac{d\theta}{dz} = 1.93 \times 10^{-8} \left( 6q_{s,0,2} - q_{s,0,1} + 0.001 \times S_y \right)$$

## a) Cálculo do Fluxo



Cálculo dos  $q_{s,0,i}$ 

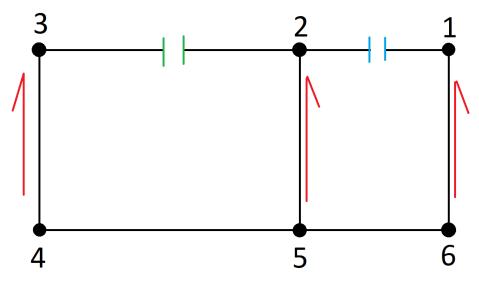
$$\frac{d\theta}{dz} = 1.93 \times 10^{-8} (6q_{s,0,2} - q_{s,0,1} + 0.001 \times S_y) = 0$$

$$\frac{d\theta}{dz} = 3.86 \times 10^{-8} (4q_{s,0,1} - q_{s,0,2} - 0.001 \times S_y) = 0$$

$$\begin{cases} -q_{s,0,1} + 6q_{s,0,2} = -0.001 \times S_y \\ 4q_{s,0,1} - q_{s,0,2} = 0.001 \times S_y \end{cases} \rightarrow \begin{cases} q_{s,0,1} = 0.00022 \times S_y \\ q_{s,0,2} = -0.00013 \times S_y \end{cases}$$

Segmento	$q_b = -\frac{S_y}{81.25e6} \cdot \left(\sum_{r=1}^n B_r y_r\right)$	$q_{s,0}$
1 – 2	0	
2 - 3	0	
2 - 5	$-0.002 \times Sy$	
3 - 4	$-0.001 \times Sy$	
4 - 5	0	
5 – 6	0	
6 – 1	$0.001 \times S_y$	

## a) Cálculo do Fluxo



Cálculo dos  $q_{s,0,i}$ 

$$\frac{d\theta}{dz} = 1.93 \times 10^{-8} (6q_{s,0,2} - q_{s,0,1} + 0.001 \times S_y) = 0$$

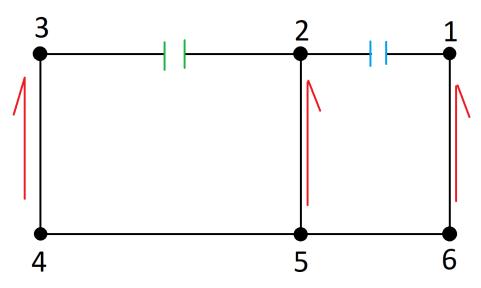
$$\frac{d\theta}{dz} = 3.86 \times 10^{-8} (4q_{s,0,1} - q_{s,0,2} - 0.001 \times S_y) = 0$$

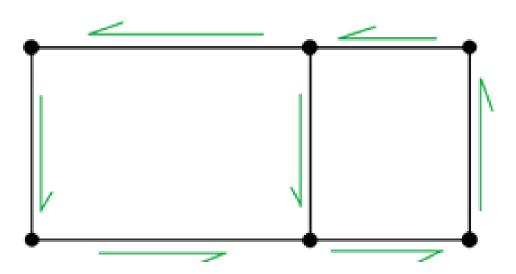
$$\begin{cases} -q_{s,0,1} + 6q_{s,0,2} = -0.001 \times S_y \\ 4q_{s,0,1} - q_{s,0,2} = 0.001 \times S_y \end{cases} \rightarrow \begin{cases} q_{s,0,1} = 0.00022 \times S_y \\ q_{s,0,2} = -0.00013 \times S_y \end{cases}$$

Segmento	$q_b = -\frac{S_y}{81.25e6} \cdot \left(\sum_{r=1}^n B_r y_r\right)$	$q_{s,0}$
1 – 2	0	
2 - 3	0	
2 - 5	$-0.002 \times Sy$	
3 - 4	$-0.001 \times Sy$	
4 - 5	0	
5 – 6	0	
6 – 1	$0.001 \times S_y$	

Passo 3.2: Contribuição do Cisalhamento

# b) Cálculo do Fluxo Total



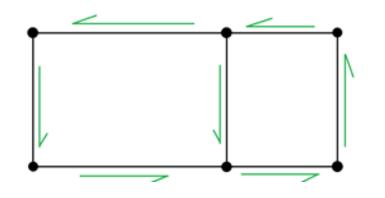


Segmento	$q_b$
1 – 2	0
2 - 3	0
2 - 5	$-0.002 \times Sy$
3 - 4	$-0.001 \times Sy$
4 - 5	0
5 – 6	0
6 – 1	$0.001 \times S_y$

$(q_{s,0,1})$	$= 0.00022 \times S_y$
$\left\{q_{s,0,2}\right\} =$	$= -0.00013 \times S_y$

Segmento	q
1 – 2	$0.00022 \times S_y$
2 - 3	$-0.00013 \times S_y$
2 - 5	$-0.00165 \times S_y$
3 - 4	$-0.00113 \times S_y$
4 - 5	$-0.00013 \times S_{y}$
5 — 6	$0.00022 \times S_y$
6 – 1	$0.00122 \times S_y$

Passo 3.2: Contribuição do Cisalhamento c) Cálculo do Deslocamento



Segmento	q
1 – 2	$0.00022 \times S_y$
2 - 3	$-0.00013 \times S_y$
2 – 5	$-0.00165 \times S_y$
3 - 4	$-0.00113 \times S_y$
4 – 5	$-0.00013 \times S_y$
5 – 6	$0.00022 \times S_y$
6 – 1	$0.00122 \times S_y$

$$\Delta = \int \left[ \int \frac{q\delta q}{Gt} ds \right] dz = 2000 \int \frac{q\delta q}{Gt} ds = \frac{2000}{2 \times 25900} \int q\delta q \, ds =$$

$$= \frac{2000}{2 \times 25900} \left[ 0.00022^2 + 0.00013^2 \times 2 + 0.00165^2 + 0.00113^2 + 0.00013^2 \times 2 + 0.00022^2 + 0.00122^2 \right] \times S_y \times 250 =$$

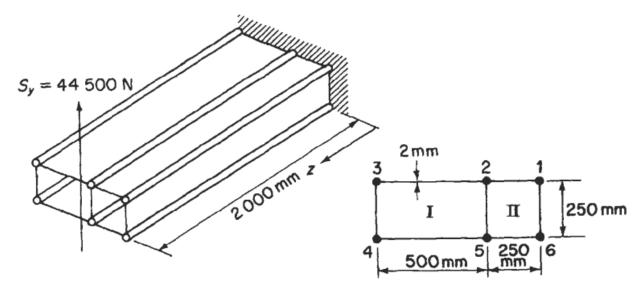
$$= \frac{2000}{2 \times 25900} \left[ 2 \times 0.00022^2 + 0.00013^2 \times 4 + 0.00165^2 + 0.00113^2 + 0.00122^2 \right] \times 44500 \times 250 =$$

$$\Delta_S = 2.43 \text{ mm}$$

Calculate the deflection at the free end of the two-cell beam shown in Fig. 10.39 allowing for both bending and shear effects. The booms carry all the direct stresses while the skin panels, of constant thickness throughout, are effective only in shear.

Take 
$$E = 69\,000\,\text{N/mm}^2$$
 and  $G = 25\,900\,\text{N/mm}^2$   
Boom areas:  $-B_1 = B_3 = B_4 = B_6 = 650\,\text{mm}^2$ ,  $B_2 = B_5 = 1300\,\text{mm}^2$ 

The beam cross-section is symmetrical about a horizontal axis and carries a vertical load at its free end through the shear centre.



Passo 4: Resultado Final

$$\Delta = 21.17 + 2.43 = 23.6 \text{ mm}$$

**P.10.8** Determine the torsional stiffness of the four-cell wing section shown in Fig. P.10.8.

Data:

Wall 12 23 34 78 67 56 45° 45° 36 27 18 Peripheral length (mm) 762 812 812 1525 356 406 356 254 Thickness (mm) 0.915 0.915 0.915 0.711 1.220 1.625 1.220 0.915 Cell areas (mm²) 
$$A_{III} = 161\,500, \quad A_{II} = 291\,000$$
  $A_{III} = 291\,000, \quad A_{IV} = 226\,000$ 

Ans.  $522.5 \times 10^6 G \,\mathrm{N}\,\mathrm{mm}^2/\mathrm{rad}$ .

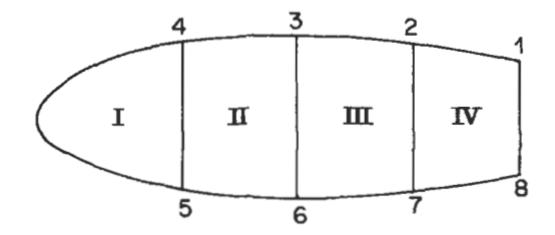


Fig. P.10.8

P.10.8 Determine the torsional stiffness of the four-cell wing section shown in Fig. P.10.8.

Data:

Wall 12 23 34 78 67 56 45° 45<sup>i</sup> 36 27 18 Peripheral length (mm) 762 812 812 1525 356 406 356 254 Thickness (mm) 0.915 0.915 0.915 0.711 1.220 1.625 1.220 0.915 Cell areas (mm<sup>2</sup>) 
$$A_{\text{III}} = 161\,500, \quad A_{\text{II}} = 291\,000$$
  $A_{\text{III}} = 291\,000, \quad A_{\text{IV}} = 226\,000$ 

Ans.  $522.5 \times 10^6 G \,\mathrm{N} \,\mathrm{mm}^2/\mathrm{rad}$ .

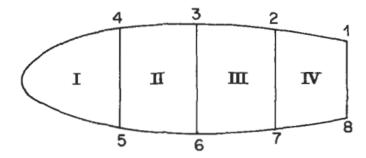


Fig. P.10.8

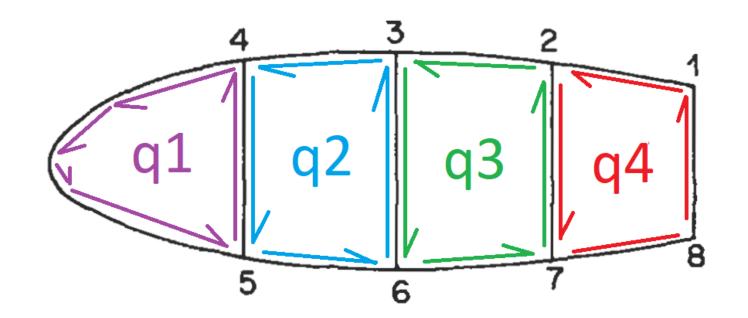
$$T = GJ. \frac{d\theta}{dz} \rightarrow \frac{T}{\frac{d\theta}{dz}} = GJ \text{ (rigidez torsional)}$$

Passo 1: Análise da Carga

$$T = T_1 + T_2 + T_3 + T_4 =$$

$$= q_1.2A_1 + q_2.2A_2 + q_3.2A_3 + q_4.2A_4 \rightarrow$$

$$\rightarrow T = 2 \times (q_1.A_1 + q_2.A_2 + q_3.A_3 + q_4.A_4)$$



Passo 2: Análise de compatibilidade de deslocamento

Célula I

$$\frac{d\theta}{dz} = \frac{1}{2A_r} \oint q \frac{ds}{Gt} = \frac{1}{2A_r G} \oint \frac{q}{t} ds = \frac{1}{2A_1 G} \left[ q_1 \times \frac{1525}{0.711} + (q_1 - q_2) \times \frac{356}{1.22} \right] =$$

$$= \frac{1}{2 \times 161500 \times G} \left[ q_1 \times \left( \frac{1525}{0.711} + \frac{356}{1.22} \right) - q_2 \times \frac{356}{1.22} \right] =$$

$$\frac{d\theta}{dz} = \left[ 75.4 \ q_1 - 9.03 \ q_2 \right] \times \frac{10^{-4}}{G}$$

Data: Wall 12 23 34 78 67 56 45° 45
$$^{\rm i}$$
 36 27 18 Peripheral length (mm) 762 812 812 1525 356 406 356 254 Thickness (mm) 0.915 0.915 0.915 0.711 1.220 1.625 1.220 0.915 Cell areas (mm²)  $A_{\rm III} = 291\,000$ ,  $A_{\rm III} = 291\,000$ ,  $A_{\rm III} = 291\,000$ 

Passo 2: Análise de compatibilidade de deslocamento

Célula II

$$\begin{split} \frac{d\theta}{dz} &= \frac{1}{2A_r} \oint q \frac{ds}{Gt} = \frac{1}{2A_r G} \oint \frac{q}{t} \, ds = \frac{1}{2A_2 G} \left[ q_2 \left( 2 \times \frac{812}{0.915} + \frac{356}{1.22} + \frac{406}{1.625} \right) - q_1 \frac{356}{1.22} - q_3 \frac{406}{1.625} \right] = \\ &= \frac{1}{2 \times 291000 \times G} \left[ q_2 \left( 2 \times \frac{812}{0.915} + \frac{356}{1.22} + \frac{406}{1.625} \right) - q_1 \frac{356}{1.22} - q_3 \frac{406}{1.625} \right] = \\ &\frac{d\theta}{dz} = \left[ -5.01 \, q_1 + 39.8 \, q_2 - 4.29 \, q_3 \right] \times \frac{10^{-4}}{G} \end{split}$$

Data: Wall 12 23 34 78 67 56 45° 45¹ 36 27 18 Peripheral length (mm) 762 812 812 1525 356 406 356 254 Thickness (mm) 0.915 0.915 0.915 0.711 1.220 1.625 1.220 0.915 
$$A_{\rm III} = 291\,000$$
,  $A_{\rm III} = 291\,000$ ,  $A_{\rm III} = 291\,000$ 

Passo 2: Análise de compatibilidade de deslocamento

Célula III

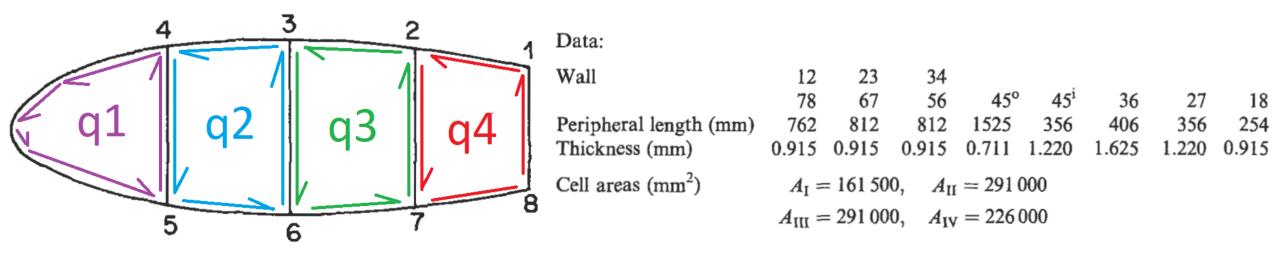
$$\begin{split} \frac{d\theta}{dz} &= \frac{1}{2A_r} \oint q \frac{ds}{Gt} = \frac{1}{2A_r G} \oint \frac{q}{t} \, ds = \frac{1}{2A_3 G} \left[ q_3 \left( 2 \times \frac{812}{0.915} + \frac{406}{1.625} + \frac{326}{1.22} \right) - q_2 \frac{406}{1.625} - q_4 \frac{356}{1.22} \right] = \\ &= \frac{1}{2 \times 291000 \times G} \left[ q_3 \left( 2 \times \frac{812}{0.915} + \frac{406}{1.625} + \frac{326}{1.22} \right) - q_2 \frac{406}{1.625} - q_4 \frac{356}{1.22} \right] = \\ &\frac{d\theta}{dz} = \left[ -4.29 \ q_2 + 39.8 \ q_3 - 5.01 \ q_4 \right] \times \frac{10^{-4}}{G} \end{split}$$

Data: Wall 12 23 34 
Wall 78 67 56 45° 45
$$^{\rm i}$$
 36 27 18 
Peripheral length (mm) 762 812 812 1525 356 406 356 254 
Thickness (mm) 0.915 0.915 0.915 0.711 1.220 1.625 1.220 0.915 
Cell areas (mm²)  $A_{\rm III} = 291\,000$ 
 $A_{\rm III} = 291\,000$ ,  $A_{\rm IV} = 226\,000$ 

Passo 2: Análise de compatibilidade de deslocamento

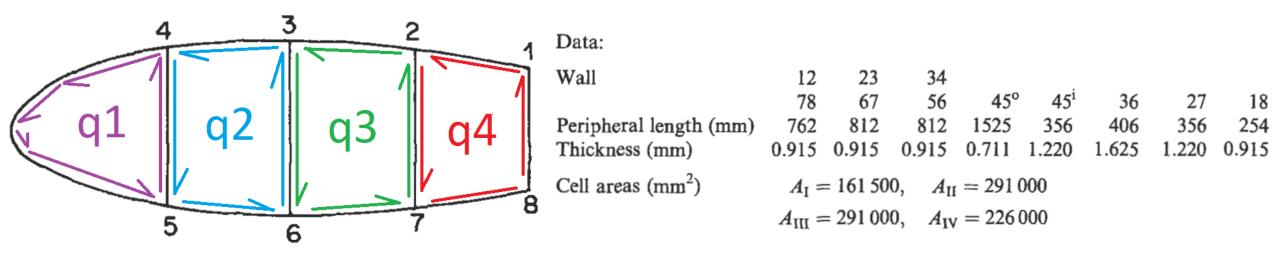
Célula IV

$$\begin{split} \frac{d\theta}{dz} &= \frac{1}{2A_r} \oint q \frac{ds}{Gt} = \frac{1}{2A_r G} \oint \frac{q}{t} \, ds = \frac{1}{2A_4 G} \left[ q_4 \left( 2 \times \frac{762}{0.915} + \frac{254}{0.915} + \frac{326}{1.22} \right) - q_3 \frac{356}{1.22} \right] = \\ &= \frac{1}{2 \times 226000 \times G} \left[ q_4 \left( 2 \times \frac{762}{0.915} + \frac{254}{0.915} + \frac{326}{1.22} \right) - q_3 \frac{356}{1.22} \right] = \\ &\frac{d\theta}{dz} = \left[ -6.45 \; q_3 + 49.4 \; q_4 \right] \times \frac{10^{-4}}{G} \end{split}$$



Passo 3: Sistema de Equações

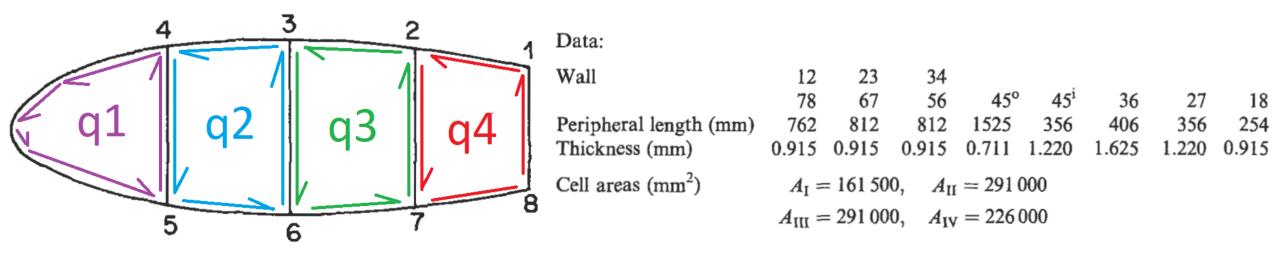
$$\begin{cases} T = 2 \times (q_1.A_1 + q_2.A_2 + q_3.A_3 + q_4.A_4) \\ \frac{d\theta}{dz} = [75.4 \ q_1 - 9.03 \ q_2] \times \frac{10^{-4}}{G} \\ \frac{d\theta}{dz} = [-5.01 \ q_1 + 39.8 \ q_2 - 4.29 \ q_3] \times \frac{10^{-4}}{G} \\ \frac{d\theta}{dz} = [-4.29 \ q_2 + 39.8 \ q_3 - 5.01 \ q_4] \times \frac{10^{-4}}{G} \\ \frac{d\theta}{dz} = [-6.45 \ q_3 + 49.4 \ q_4] \times \frac{10^{-4}}{G} \end{cases}$$



Passo 3: Sistema de Equações

$$\begin{cases} 75.4 \ q_1 - 9.03 \ q_2 = -5.01 \ q_1 + 39.8 \ q_2 - 4.29 \ q_3 \\ 75.4 \ q_1 - 9.03 \ q_2 = -4.29 \ q_2 + 39.8 \ q_3 - 5.01 \ q_4 \rightarrow \\ 75.4 \ q_1 - 9.03 \ q_2 = -6.45 \ q_3 + 49.4 \ q_4 \end{cases} \begin{cases} 80.41 \ q_1 - 48.83 \ q_2 + 4.29 \ q_3 = 0 \\ 75.4 \ q_1 - 4.74 \ q_2 - 39.8 \ q_3 = -5.01 \ q_4 \rightarrow \\ 75.4 \ q_1 - 9.03 \ q_2 + 6.45 \ q_3 = 49.4 \ q_4 \end{cases} \\ \rightarrow \begin{cases} 80.41 \ q_1 - 48.83 \ q_2 + 4.29 \ q_3 = 0 \\ 75.4 \ q_1 - 4.74 \ q_2 - 39.8 \ q_3 = -5.01 \ q_4 \rightarrow \\ 75.4 \ q_1 - 9.03 \ q_2 + 6.45 \ q_3 = 49.4 \ q_4 \end{cases} \begin{cases} q_1 = 0.695 \ q_4 \\ q_2 = 1.260 \ q_4 \\ q_3 = 1.290 \ q_4 \end{cases}$$

$$\begin{cases} T = 2 \times (q_1.A_1 + q_2.A_2 + q_3.A_3 + q_4.A_4) \\ \frac{d\theta}{dz} = [-6.45 \ q_3 + 49.4 \ q_4] \times \frac{10^{-4}}{G} \end{cases} \rightarrow \begin{cases} T = 2 \times [0.695 \times 161500 + (1.26 + 1.29) \times 291000 + 226000] q_4 \\ \frac{d\theta}{dz} = \frac{41.07 \times 10^{-4}}{G} q_4 \end{cases}$$



Passo 3: Sistema de Equações

$$\begin{cases} T = 2 \times (q_1.A_1 + q_2.A_2 + q_3.A_3 + q_4.A_4) \\ \frac{d\theta}{dz} = [-6.45 \ q_3 + 49.4 \ q_4] \times \frac{10^{-4}}{G} \end{cases} \rightarrow \begin{cases} T = 2 \times [0.695 \times 161500 + (1.26 + 1.29) \times 291000 + 226000] q_4 \\ \frac{d\theta}{dz} = \frac{41.07 \times 10^{-4}}{G} q_4 \end{cases}$$

$$\begin{cases} T = 2160585 q_4 \\ \frac{d\theta}{dz} = \frac{41.07 \times 10^{-4}}{G} q_4 \end{cases} \rightarrow \frac{T}{\frac{d\theta}{dz}} = \frac{2160585}{41.07 \times 10^{-4}} G = 526.08 \times 10^6 \ \text{G N} \frac{\text{mm}^2}{\text{rad}} \end{cases} \rightarrow GJ = 526.08 \times 10^6 \ \text{G N} \frac{\text{mm}^2}{\text{rad}}$$