# DYNAMCIS OF STEEL RAILWAY BRIDGES(please note: this is not the final title!)

 $study\ of\ dynamic\ behaviour\ of\ railway\ bridges\ and\ applicability\ assessment\ of\ simplified\ dynamic\\ analysing\ method.$ 

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## INTRODUCTION

Dear Prof.Frans Bijlaard

This is a sample thesis created by Sijian Deng.

General remarks by Sijian: Firstly I want to apologize for flaws in the whole document, I know for some part it's not that pleasant to read but I'll fix them in future versions. Since everything written here is not finally decided, please read Section1.3 first for my primarily thought about the objectives. In addition: I don't want to narrow down on one particular topic before the first kick-off meeting.

Since I'm still open for different interesting topics with regard to railway bridge dynamics, please let me know if you know any possible topic besides from those listed in Section1.3 for me.

The table of contents will give you a grasp of my thesis structure. Although there is nothing besides from literature research for now, I already have a vague thought about following thesis contents. Please take a look at the sections after literature research.

#### 1.1 Problem statement

#### 1.2 Motivation of the thesis

## 1.3 Objectives and research question

Within the study field of steel railway bridge dynamics, I found several potential research topics as my literature research goes on. These topics are seldom(or never) discussed according to my knowledge. I want to ask your opinion on these topics and tell me whether it's feasible for my graduation thesis.

- 1. Criteria for bridge lateral behaviour. 1.2Hz sufficient or not? Or is it too conservative? Any other criteria can be found, to be used together with 1.2Hz? Please see Section2.8.9.2 for detailed information about this criteria.
- 2. Long span bridges usually okay with dynamic but always have frequency below 1.2Hz-¿can't meet Eurocode requirements. Any possible situation to avoid dynamic check?
- 3. Upgrade train dynamic signature method for dynamic analyses expand application from vertical direction only to 3-dimensional. Please see Section 2.9.5 for detailed information.
- 4. Background of the verification checks
- 5. Study the future usage of high-strength steel on railway bridges less mass, same stiffness higher possibility of resonance under train excitation

- 1.4 Research methodology
- 1.5 Outline of the report

## LITERATURE REVIEW

Eurocode 1990 and Eurocode 1991-2 and their corresponding National Annex are primary codes to be fulfilled through out the whole process of conducting a railway bridge in Netherlands. It is of great importance to study dynamic effect on railway bridges due to increasing usage of public train service. What's more, as the need on capacity of railway service increase, high-speed train dynamic loading becomes a more general issue in the design of railway bridges. It is common knowledge that bridge structures loaded by high-speed trains have bigger chance of resonance, as well as of being required to be dynamic analysed in designing process.

Unfortunately in Chapter 6.4 of Eurocode NEN-EN 1991-2-2003, the description of various subjects is vague including general procedures of conducting dynamic analyses and methods of additional dynamic analysing calculating, etc. The following paragraphs aim to summarize Chapter 6.4 of Eurocode 1991-2 [7], in order to give a better interpretation.

This literature research will be done by reviewing both physics knowledge and engineering standards.

## 2.1 Introduction to the dynamics of railway bridges

[8] Dynamics of railway bridges is concerned with the study of deflections and stresses in railway bridges. The loads are represented by the moving wheel and axle forces, by means of which railway vehicles transmit their load and inertia actions to railway bridges. A survey of the dynamic effects of vehicles on railway bridges is given in Figure 2.1

#### 2.1.1 Factors influencing dynamic behaviour

As stated in [7, p. 6.4.2] there are 11 factors influencing dynamic behaviour of a railway bridge. The principal factors which influence dynamic behaviour are:

- the speed of traffic across the bridge
- the span L of the element and the influence line length for deflection of the element being considered
- the mass of the structure
- the natural frequencies of the whole structure and relevant elements of the structure and the associated mode shapes (eigenforms) along the line of the track
- the number of axles, axle loads and the spacing of axles
- the damping of the structure

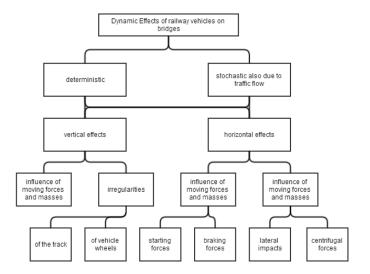


Figure 2.1: Dynamic effects of railway vehicles on bridges. Extracted from [8, p. 1.1]

- vertical irregularities in the track
- the unsprung/sprung mass and suspension characteristics of the vehicle
- the presence of regularly spaced supports of the deck slab and/or track (cross girders, sleepers etc.)
- vehicle imperfections (wheel flats, out of round wheels, suspension defects etc.)
- the dynamic characteristics of the track (ballast, sleepers, track components etc.)

## 2.2 Theoretical bridge models

According to [8, Chapter.2], theoretical models of railway bridges are of two types

- with continuously distributed mass
- with mass concentrated in material points(lumped mass)
- their combinations

#### 2.2.1 Mass beams

The most common simplified model for bridge is a simply supported Euler-Bernoulli beam model (see Figure 2.2). The equation of motion of the beam expresses the equilibrium of forces per unit length:

$$EI\frac{\partial^4 v(x,t)}{\partial x^4} + \mu \frac{\partial^2 v(x,t)}{\partial t^2} + 2\mu \omega_b \frac{\partial v(x,t)}{\partial t} = f(x,t)$$
 (2.1)

where:

v(x,t):vertical deflection of the beam at the point x and at time t

E:modulus of elasticity of the beam

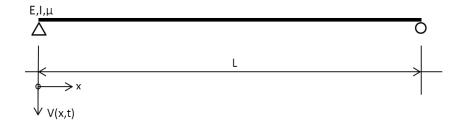


Figure 2.2: Mass beam model of span  ${\bf L}$ 

I:moment of inertia of beam cross section

 $\mu$ :mass per unit length fo the beam

 $\omega_b$ :circular frequency of viscous damping

f(x,t):load at point x and time t per unit length of the beam

#### 2.2.2 Continuous beam

The continuous beam model is suitable for multi-span bridges in general. The equation of motion is the same as simple beam model in Equation 2.1.

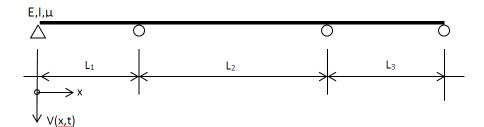


Figure 2.3: Continuous beam model

#### 2.2.3 Complex systems

- 2.2.3.1 Trusses
- 2.2.3.2 Frames
- 2.2.3.3 Curved bars

## 2.3 Modelling of railway vehicles

According to Newton's law, 2 basic load effects are produced by moving train: vertical forces due to vehicle weight, and inertia effects caused by vehicle acceleration. The loads on a railway bridge are very complex problems thus in engineering practice, loads are often simplified. But, the simplification depends on the purpose of the analysis.

#### 2.3.1 Moving vertical forces model

If the inertia effects are neglected, loads of the moving trains can be modelled as moving vertical forces. For example, load diagram for type TALGO trains is shown in Figure 2.4 proposed by [12].

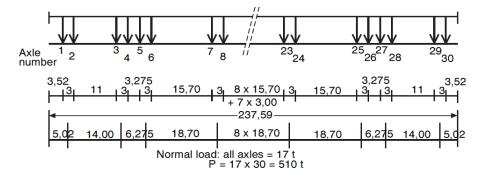


Figure 2.4: Moving vertical force model for TALGO trains

#### 2.3.2 Advanced models

Nowadays more and more models have been proposed to meet different requirements of railway bridge dynamic analysis. The complexity of these models differs from each other but they are all more complicated than moving vertical forces models. For example, vehicle-bridge interaction model takes vehicle suspension system into account, which gives an alternative for discovering resonance effects between bridge and the vehicle suspension systems.

See Figure.2.5 for an example of advanced model.

#### 2.3.3 Models proposed in Eurocodes

See Chapter 2.8.4

#### 2.4 Track model

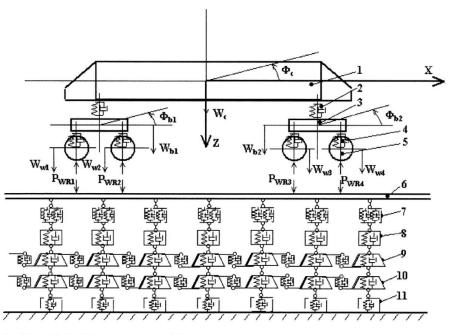
Proposed in [12, A.6.1.3], the track is represented by Timoshenko beam elements for the rails and takes account of the rail/sleeper fastening characteristics as well as the ballast(if one exists).

"A sleeper is generally represented by two beam elements, with two covering the rail and one used for the deck. Sleepers and ballast are modelled as concentrated masses. They are linked to the nodes of the rail and the bridge by a parallel spring and damper system. The track can be modelled to any length on both side of the bridges, where the stiffening effect of the bridge has to be taken into account. The effects of track distribution are not considered. Each vehicle is able to absorb the kinetic energy of the bridge and it is for this reason that, at resonance, the deflections and accelerations of the bridge obtained with this model are lower than those obtained with a live load diagram."

The most complete model for analysing train/track/bridge interaction is shown in Figure

## 2.5 Vertical Dynamic effects

As stated in EN 1991-2[7], the static stress and deformations (and associated bridge deck acceleration) induced in a bridge are increased and decreased under the effects of moving traffic by the following:



- 1 Wagon Body (M<sub>c</sub> mass, J<sub>c</sub> inertial moment),
- 2 Secondary Suspension (K<sub>sc</sub> stiffness coefficient, C<sub>sc</sub> damping coefficient),
- $3 Bogie (M_b mass, J_b inertial moment),$
- $4-Primary\ Suspension\ (K_{pr}-stiffness\ coefficient,\ C_{pr}-damping\ coefficient),$
- 5 Wheelset (M<sub>w</sub> mass),
- 6 Rail (Timoshenko Beam)
- 7 Fastener and Pad ( $K_f$ ,  $K_p$  stiffness coefficients,  $C_f$ ,  $C_p$  damping coefficients), 8 Sleeper ( $M_s$  mass,  $K_{sl}$  stiffness coefficient,  $C_{sl}$  damping coefficient),
- 9 Ballast (M<sub>bl</sub> mass, K<sub>bl</sub> stiffness coefficient, C<sub>bl</sub> damping coefficient),
- $10-Subballast \, (M_{sb}-mass,\, K_{sb}-stiffness \, coefficient,\, C_{sb}-damping \, coefficient),$
- $11 Subgrade (K_{sg} stiffness coefficient, C_{sg} damping coefficient).$

Figure 2.5: A dynamic model for the vertical interaction of the rail track and wagon system. Proposed in [14]

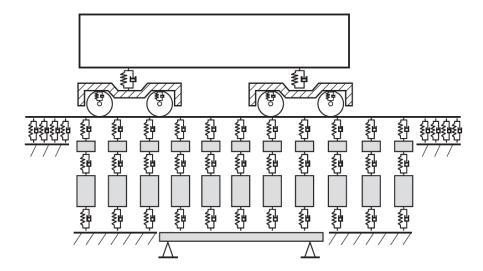


Figure 2.6: Diagram of the dynamic train-track-bridge model. Extracted from [12, Fig. 15]

- the rapid rate of loading due to the speed of traffic crossing the structure and the inertial response (impact) of the structure,
- the passage of successive loads with approximately uniform spacing which can excite the structure and under certain circumstances create resonance (where the frequency excitation(or a multiple there of) matches a natural frequency of the structure (or a multiple there of), there is a possibility that the vibrations caused by successive axles running onto the structure will be excessive),
- variations in wheel loads resulting from track or vehicle imperfections (including wheel irregulations).

For determining the effects (stresses, deflections, bridge deck acceleration etc.) of rail traffic actions the above effects shall be taken into account.

## 2.6 Horizontal transverse dynamic effects

There's only one criterion in the Eurocodes mentioned that the bridge's first lateral natural frequency should no lower that 1.2 Hz. Dynamic analyses are required if this criterion is not met.

However, as more and more long-span bridges are built nowadays, this requirement is not valid for more bridges. This is because, in general, the lateral natural frequency of a bridge decreases when span increases. For bridges with span longer than 100m, there's few bridge can have a lateral frequency higher than 1.2Hz, according to senior engineers' designing experience.

So it is vital to discuss horizontal dynamic effects for the sake of longer span bridges. In additional, study the requirements for horizontal vibration of railway bridges to make the results of dynamic analysis usable.

#### 2.6.1 Sources induce transverse reactions

According to [13][8][7], following sources are identified:

- Horizontal track irregularities
- Sinusoidal motion of conical wheels along cylindrical rail heads

- Centrifugal forces on curved tracks

#### 2.6.2 Horizontal vibration of a beam

Fryba[8] described the first two sources in mathematical terms: Consider a simply supported beam loaded by moving train loading, horizontal vibrations of the beam in a transverse direction w(x,t) are generated by lateral random forces  $H_n(t)$  due to random irregularities and sinusoidal motion. The differential equation for the deformation is:

$$EI_{y}w^{IV}(w,t) + \mu \ddot{w}(x,t) = \sum_{n=1}^{N} \varepsilon_{n}\delta(x+d_{n}-ct)H_{n}(t)$$
(2.2)

where  $H_n(t)$  stands for forces due to random irregularities and sinusoidal motion.  $H_n(t)$  is of a typically random character and can be replaced by horizontal transverse forces with zero mean values. See Figure 2.7 for example.

[8, Table 9.1] gives sample of natural frequencies of steel truss bridges with open deck. Shown as follow Table 2.1. Please note that truss bridges have high stiffness.

Vibration Type	Symbol	i	Bridge with span				
vibration Type	Symbol	j	l = 25.85m	l = 48.4m			
		1	8.7	5.4			
Vertical	$f_{j}$	2	34.7	21.5			
		3	78.1	48.3			
		1	15.3	4.7			
	$f_{hj}$	2	61.1	19.0			
	3		137.5	42.7			
		1	14.8	4.7			
Horizontal	$f_{yj}$	$^2$	53.9	l = 48.4m $5.4$ $21.5$ $48.3$ $4.7$ $19.0$ $42.7$ $4.7$ $18.1$ $38.8$ $4.6$ $17.5$ $36.4$ $19.2$ $38.4$ $57.7$ $16.7$ $35.4$ $56.9$ $19.7$ $41.9$			
		3	107.4	18.1 38.8 4.6 17.5			
		1	14.7	4.6			
	$f'_{yj}$	2	52.5	38.8 4.6 17.5 36.4			
	00	3	101.7	36.4			
		1	35.7	19.2			
	$f_{\xi j}$	2	71.3	38.4			
		3 107.0		57.7			
		1	34.9	16.7			
Torsional	$f_{arphi j}$	$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	73.5	35.4			
		3	117.4	56.9			
		1	36.5	19.7			
	$f'_{\varphi j}$		77.6	41.9			
	, ,	3	126.4	67.5			

Table 2.1: Spatial vibrations of steel truss bridges with open bridge deck

#### 2.6.2.1 Centrifugal forces

In [7, p. 6.5.1] specifies following principles about centrifugal forces act on railway bridges:

Where the track on a bridge is curved over the whole or part of the length of the bridge, the centrifugal force and the track cant shall be taken into account.

The centrifugal forces should be taken to act outwards in a horizontal direction at a height of 1.80m above the running surface (see [7, Figure 1.1]). For some traffic types, e.g. double stacked containers, an increased value of  $h_t$  should be specified.

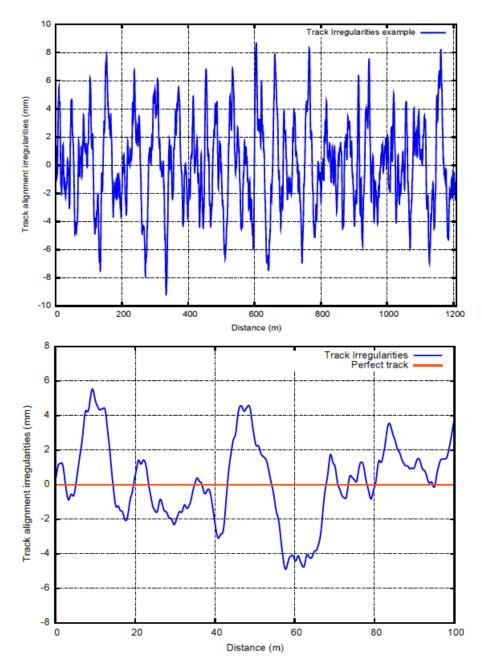


Figure 2.7: Example of a track lateral alignment irregularities profile for a track with low irregularities in a total length of 1209m. Extracted from [13]

The centrifugal force shall always be combined with the vertical load. The centrifugal force shall not be multiplied by the dynamic factor  $\Phi_1$  or  $\Phi_3$ .

The characteristic value of the centrifugal force shall be determined according to the following equations:

$$Q_t k = \frac{v^2}{g \cdot r} (f \cdot Q_{vk}) = \frac{V^2}{127r} (f \cdot Q_{vk})$$
 (2.3)

$$q_{tk} = \frac{v^2}{q \cdot r} (f \cdot q_{vk}) = \frac{V^2}{127r} (f \cdot q_{vk})$$
 (2.4)

where:

 $Q_{tk}, q_{tk}$  Characteristic values of the centrifugal forces

 $Q_{vk}, q_{vk}$  Characteristic values of the vertical loads specified in [7, p. 6.3]

f Reduction factor, see below

v Maximum speed in accordance with [7, 6.5.1(5)][m/s]

V Maximum speed in accordance with [7, 6.5.1(5)][km/h]

g Acceleration due to gravity  $[9.81m/s^2]$ 

r Radius of curvature [m].

For Load Model 71 (and where required Load Model SW/0) the reduction factor f is given by:

$$f = \left[1 - \frac{V - 120}{1000} \left(\frac{814}{V} + 1.75\right) \left(1 - \sqrt{\frac{2.88}{L_f}}\right)\right]$$
 (2.5)

#### 2.6.3 Requirements for traffic safety(horizontal)

Requirements other than bridge first lateral frequency higher than 1.2Hz. Since there's no further requirements mentioned by Eurocode, following requirements are gathered from other European codes, eg. British standards, UIC leaflet, etc.

- Requirements regarding traffic safety for vehicles
  - (a) Guiding Force: [2], [5] and [3] propose safty limitations against railway vehicle overturning. From [5] the maximum guiding force for a vehicle with a load per axle of 170kN (AVE) is 66kN per axle and 48kN per axle for a vehicle with a load per axle of 112kN (ICE2). For the R1 freight wagon (load per axle of 245kN), the maximum guiding force per axle is 78kN.
  - (b) Maximum lateral acceleration of the railway vehicle: proposed by [16]
- Requirements regarding safety for bridge [9] A2.4.4.1(2): Horizontal transverse deflection(to ensure acceptable horizontal track radii) and horizontal rotation of a deck about a vertical axis at ends of a deck(to ensure acceptable acceptable horizontal track geometry and passenger comfort)

# 2.6.4 Requirements for traffic safety on derailment: Railway vehicle derailment mechanism and safety criteria

The four types of derailment: flange climb derailment, derailment caused by guage widening and rail rollover, derailment caused by track panel shift, derailment cause by vehicle lateral instability have a common cause of high lateral force at the wheel-rail interface. According to [10, Chapter 8, IV] any conditions that lead to high lateral forces or lead to lower the ability of the system to sustain the force should be corrected.

#### 2.6.4.1 Flange limb derailment

Wheel flange climb derailments are caused by wheels climbing onto the top of the railhead then further running over the rail. Wheel climb derailments generally occur in situations where the wheel experiences a high lateral force combined with circumstances where the vertical force is reduced on the flanging wheel. The high lateral force is usually induced by a large wheelset angle-of-attack. The vertical force on the flanging wheel can be reduced significantly on bogies having poor vertical wheel load equalisations, such as when negotiating rough track, large track twist, or when the car is experiencing roll resonances.

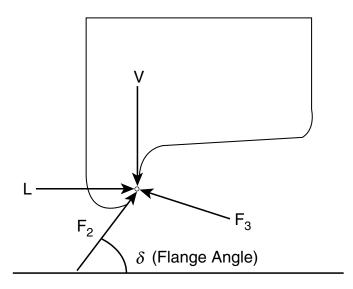


Figure 2.8: Forces at flange contact location. Extracted from [10, Figure 8.4]

The criterion L/V ratio can be expressed as:

$$\frac{L}{V} = \frac{\tan \delta - \frac{F_2}{F_3}}{1 + \frac{F_2}{F_2} \tan \delta} \tag{2.6}$$

Nadal's famous L/V ratio limiting criterion, given by Equation.2.7, was proposed for the saturated condition  $F_2/F_3 = \mu$ 

$$\frac{L}{V} = \frac{\tan \delta - \mu}{1 + \mu \tan \delta} \tag{2.7}$$

#### 2.6.4.2 Derailment caused by guage widening and rail rollover

Derailments caused by guage widening usually involve a combination of wide gauges and large lateral rail defections (rail roll), as shown in Figure 2.9. Large lateral forces from the wheels act to spread the rails in curves. Both rails may experience significant lateral translation and/or railhead roll, which often cause the nonflanging wheel to drop between rails.

**AAR Chapter XI rail roll criterion** The AAR Chapter XI rail roll criterion is established by using the L/V ratio. The roll moment about the pivot point is given by,

$$M = Vd - Lh \tag{2.8}$$

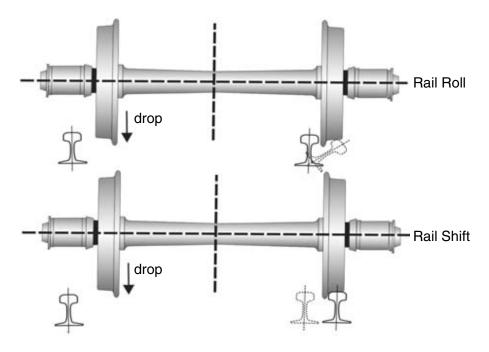


Figure 2.9: Gauge widening derailment. Extracted from [10, Figure 8.18]

under an equilibrium condition, just before the rail starts to roll, M approaches to zero, then,

$$\frac{L}{V} = \frac{d}{h} \tag{2.9}$$

This L/V ratio is considered as the critical value to evaluate the risk of rail roll. When the L/V ratio is larger than the ratio of d/h, the risk of rail roll becomes high. The critical L/V ratio for rail roll can vary from above 0.6 for contact at the gauge side to approximately 0.2 when the contact position is at the far-field side based on the dimension of the rails. This is because the distance d is reduced. Note that this L/V ratio is calculate assuming that neither the rail fasteners nor the torsional stiffness of the rail section provide any restraint.

#### 2.6.4.3 Derailment caused by track panel shift

Track panel shift is the cumulative lateral displacement of the track panel, including rails, tie plates and ties, over the ballast, as shown in Figure 2.10. A small shift of these components may not immediately cause the loss of guidance to bogies. However, as the situation gradually depreciates to a certain level, wheels could lose guidance and drop to the ground at some speed. The derailments caused by track panel usually result in one wheel falling between the rails and the other falling outside of the track.

**Panel shift criterion** Researched by the French National Railways suggested that the limiting lateral axle load can be defined in a general expression for preventing excessive track panel shift:

$$L_c = aV + b (2.10)$$

where  $L_c$  is the critical lateral load and V is the vertical axle load. [10, Table 8.2] lists two groups of suggested valued of a and b. It is possible that different values for a and b can be specified in different area.

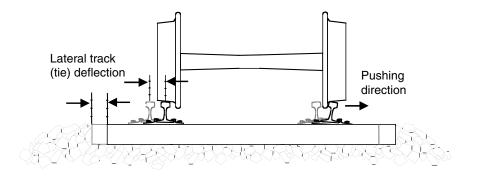


Figure 2.10: Lateral track panel shift. Extracted from [10, Figure 8.27]

#### 2.6.4.4 Derailment caused by vehicle lateral instability

On tangent track, the wheelset generally oscillates around the track centre due to any vehicle and track irregularities, as shown in Figure 2.11. This movement occurs because vehicle and track are never absolutely smooth and symmetric. This self-centring capability of a wheelset is induced by the coned shape of the wheel tread. However, as speed is increased, if the whelset conicity is high, the lateral movement of wheelset, as well as the associated bogic and car body motion, can cause oscillations with large amplitude and a well-defined wavelength. The lateral movements are limited only by the contact of the wheel flanges with the rail. This vehicle dynamic response is also termed as vehicle hunting, and can produce high lateral forces to damage track to cause derailments.

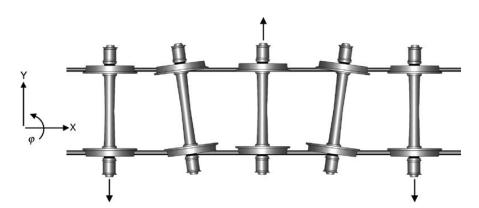


Figure 2.11: Wheelset oscillates around the track centre. Extracted from [10, Figure 8.28]

Derailment cause by vehicle hunting can have derailment mechanisms of all four types discussed in the previous sections. The high lateral force induced from hunting may cause wheel flange climbing on the rail, guage widening, rail rollover, track panel shift, or combinations of these. The safety concerns for this type of derailment, usually occuring at higher speeds, make it an important area of study.

Hunting predominantly occurs in empty or lightweight vehicles. The critical hunting speed is highly dependent on the vehicle/track characteristics. Investigation of the critical speed for such a system with nonlinearities is to examine the vehicle response to a disturbance using a numerical solution of the equations of motion.

#### 2.6.5 Requirements for traffic serviceability(horizontal)

The criteria Comfort Indexes for assessing ride comfort in railway vehicles proposed in [15]. This standard describes a methodology for assessing ride comfort as a function of longitudinal, vertical and transverse accelerations.

Comfort Index indicates the percentage of passengers experimenting discomfort in a specific situation. These indexes can be computed via empiric formula given in the standard, which depend on variables such as lateral acceleration, rate of change of acceleration and rolling velocity. All these values are filtered with a moving average filter that eliminates small wavelength components. Using this methodology for the computed worst-case situations, the comfort indexes have been found excellent, therefore no passenger should feel uncomfortable.

#### 2.7 Torsional vibration

According to [8, p. 9.1.3], the horizontal lateral forces of railway vehicles act on the rial top level, i.e. outside the cross section centroid of the bridge in the majority of cases. Let the difference of elevations be h. Consequently, they affect the bridge by twisting moments  $hH_n(t)$ . The differential equation of a beam due to simple torsion is

$$-GI_{\xi}\xi''(x,t) + \mu \ddot{\xi}(x,t) = \sum_{n=1}^{N} \varepsilon_n \delta(x + d_n - ct) h H_n(t)$$
(2.11)

where  $\xi(x,t)$  is the rotation about the longitudinal beam axis x, G is the modulus of elasticity in shear,  $GI_{\xi}$  is the moment of torsional rigidity per unit length,  $\mu_{\xi}$  is the mass polar moment of inertia with regard to axis x per unit length.

## 2.8 General design principles and procedures concerning railway bridge dynamics proposed by Eurocodes

#### 2.8.1 Requirements for railway bridge verification

[9] propose following requirements

- 1. Checks on bridge deformations shall be performed for traffic safety purposes for the following items:
  - vertical accelerations of the deck
  - vertical deflection of the deck throughout each span
  - unrestrained uplift at the bearings (to avoid premature bearing failure)
  - vertical deflection of the end of the deck beyond bearings(to avoid destabilising the track, limit uplift forces on rail fastening systems and limit additional rail stresses)
  - twist of the deck measured along the centre line of each track on the approaches to a bridge and across a bridge(to minimise the risk of train derailment)
  - rotation of the ends of each deck about a transverse axis or the relative total rotation between adjacent deck ends(to limit additional rail stresses, limit uplift forces on rail fastening systems and limit angular discontinuity at expansion devices and switch blades)
  - longitudinal displacement of the end of the upper surface of the deck due to longitudinal displacement and rotation of the deck end(to limit additional rail stresses and minimise disturbance to track ballast and adjacent track formation)

- horizontal transverse deflection (to ensure acceptable horizontal track radii)
- horizontal rotation of a deck about a vertical axis at ends of a deck (to ensure acceptable horizontal track geometry and passenger comfort)
- limits on the first natural frequency of lateral vibration of the span to avoid the occurrence of resonance between the lateral motion of vehicles on their suspension and the bridge
- 2. Checks on bridge deformations should be performed for passenger comfort, i.e. vertical deflection of the deck to limit coach body acceleration in accordance with A2.4.4.3[9]
- 3. The limits given in A2.4.4.2 and A2.4.4.3[9] take into account the mitigating effects of track maintenance (for example to overcome the effects of the settlement of foundations, creep, etc.)

#### 2.8.2 Conceptual check

The conceptual check is to help designers avoid unsafe designs in conceptual stage. Once the bridge type and rough geometry is sketched, designers can easily know whether the bridge would have dynamic problem in the future.

For example, in [1, p. cl.8.7.4], it is stated that if the bridge meets criteria 2.12, no dynamic analysis in necessary.

$$V > 200km/h$$
  $\delta_{dyn} \le \text{ value given by the dynamic study, but } \delta_{stat} \le L/2600$  (2.12)

On the other hand, no other conceptual check criterion is given for train speed under 200km/h. Since there is higher possibility that, for bridges with span larger than 100m, they will have resonance with the normal trains.

#### 2.8.3 Logic diagram

This logic diagram is used to determine whether a dynamic analysis is required, as shown in Figure 2.12a, is represented in [7] Chapter 6.4 Dynamic Effects, where V[Km/h] is the Maximum Line Speed at the site, L[m], the span length,  $n_0[Hz]$ , the first nature frequency under permanent loads,  $n_T$ , the first natural torsional frequency for the same load, v[m/s] the maximum nominal speed and finally  $(v/n_0)_{lim}$ , as given in Annex F of EN1991-2. The frequency first of vibration,  $n_0$ , must be within the limits established in figure 2.12b.

Checking through the logic diagram is regarded as first step is because designers can try to avoid designs that will be required to be dynamic analysed in the very beginning of the designing phase.

The upper limit (1) is defined as

$$n_0 = 94,76L^{-0.748} (2.13)$$

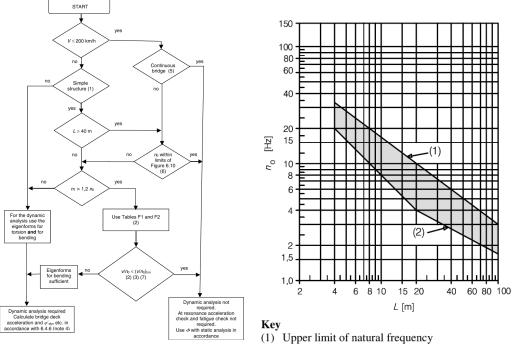
and the lower limit (2) as:

$$n_0 = \begin{cases} 80/L & \text{for } 4m \le L \le 20m \\ 94,76L^{-0.748} & \text{for } 20m \le L \le 100m \end{cases}$$
 (2.14)

#### 2.8.4 Train models

According to NEN 1991-2[7] and Designers Guide[1] Rail traffic actions are defined by means of load models, Four models of railway loading are given:

• LM71 and LM SW/0(for continuous bridges) to represent normal rail traffic on mainline railways (passenger and heavy freight traffic)



- (a) Flow chart for determining whether a dynamic analysis is required. Extracted from EN1991-2[7]
- (2) Lower limit of natural frequency
- Extracted from (b) Limits for bridge natural frequencies,  $n_0[Hz]$ , as a function L[m]. Extracted from EN1991-2[7].

Figure 2.12: Logic diagram for determining whether dynamic analyses are necessary, extracted from [7, p. 6.4.4]

- SW/2 to represent abnormal loads or waggons
- LM 'unloaded train' to represent the effect of an unloaded train
- LM HSLM (comprising HSLM-A and HSLM-B) to represent the loading from passenger trains at speeds exceeding 200km/h.

#### 2.8.4.1 Load Model 71

LM71 represents the static effect of vertical loading due to normal rail traffic.

The load arrangement and the characteristic values for vertical loads have to be taken as shown in Figure. 2.13

The actions listed below, associated with LM71, have to be multiplied by factor  $\alpha$ :

- equivalent vertical loading for earthworks and earth pressure effects
- centrifugal forces
- nosing force (multiplied by  $\alpha$  for  $\alpha \geq 1$  only)
- traction and braking forces
- derailment actions for accidental design situations
- Load Model SW/0 for continuous span bridges

For international lines, it is recommended that a value of  $\alpha \geq 1.0$  is adopted. But this freedom of choice of the factor  $\alpha$  could lead to a non-uniform railway network in Europe. Therefore in UIC Code  $702^4$   $\alpha = 1.33$  is generally recommended for all new bridges constructed for the international freight network, but not compulsory.

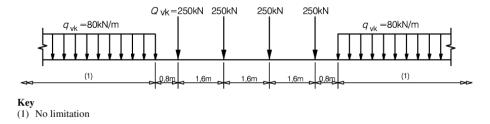


Figure 2.13: Load Model 71 and characteristic values for vertical loads. Extracted from EN1991-2[7]

#### 2.8.4.2 Load Models SW/0 and SW/2

Load Models SW/0 represents the static effect of vertical loading due to normal rail traffic on continuous beams.

Load Model SW/2 represents the static effect of vertical loading due to heavy abnormal rail traffic.

The load arrangement is as shown in Figure. 2.14, with the characteristic values of the vertical loads according to Table.2.2

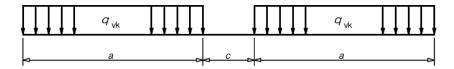


Figure 2.14: Load Models SW/0 and SW/2.. Extracted from EN1991-2[7]

Load model	$q_{vk}(kN/m)$	a(m)	c(m)
SW/0	133	15.0	5.3
SW/2	150	25.0	7.0

Table 2.2: Characteristic values for vertical loads for Load Models SW/0 and SW/2

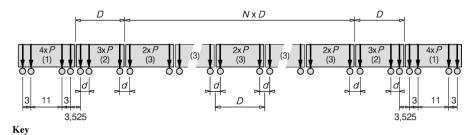
#### 2.8.4.3 Load Model 'unloaded train'

From some specific verification purposes a specific load model is used, called 'unloaded train'. The Load Model 'unloaded train' consists of a vertical uniformly distributed load with a characteristic value of  $10.0 \mathrm{kN/m}$ .

#### 2.8.4.4 Load Models SHLM

Load Models HSLM comprises two separate universal *high-speed* trains with variable coach lengths. In order to ensure that they deliver dynamic behaviour with regards to current and future train traffic, bridges should be calculated using the Universal Dynamic Train(HSLM) consisting of HSLM-A and/or HSLM-B. These are defined as follows:

- For the definition of train HSLM-A, a set of ten reference trains A1 to A10: see Figure. 2.15 and Table. 2.3 below.
- For the definition of train HSLM-B: See Figure. 2.16 and '2.17 below



- (1) Power car (leading and trailing power cars identical)
- (2) End coach (leading and trailing end coaches identical)
- (3) Intermediate coach

Figure 2.15: Diagram of Universal Dynamic Train HSLM-A. Extracted from EN 1991-2[7]

Universal train	Number of	Coach	length	Bogie	axle	Point	force
	intermediate	D(m)		spacing of	d(m)	P(kN)	
	coaches, $N$						
A1	18	18		2.0		170	
A2	17	19		3.5		200	
A3	16	20		2.0		180	
A4	15	21		3.0		190	
A5	14	22		2.0		170	
A6	13	23		2.0		180	
A7	13	24		2.0		190	
A8	12	25		2.5		190	
A9	11	26		2.0		210	
A10	11	27		2.0		210	

Table 2.3: HSLM-A, definition of ten trains. Extracted from EN1991-2[7]

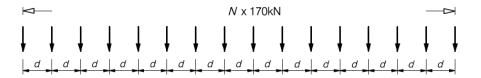


Figure 6.13 - HSLM-B

Figure 2.16: Diagram of Universal Dynamic Train HSLM-B. Extracted from EN 1991-2[7]

This Load Model comprises N number of point forces of 170kN at regular spacing d(m) (2.16) where N and d are defined in Figure. 2.17

Table. 2.4 illustrates how HSLM-A and HSLM-B are applied and indicates the train to be used for dynamic bridge calculations.

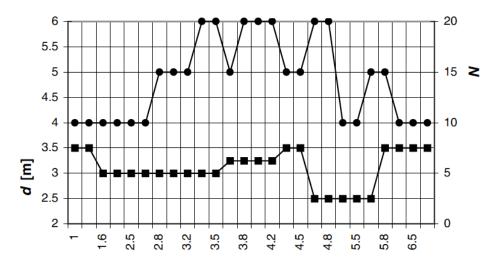


Figure 2.17: Universal Dynamic Train HSLM-B. Extracted from EN 1991-2[7]

Structural configuration	Span			
	L < 7m	$L \geq 7m$		
Simply supported span	HSLM-B	HSLM-A		
Continuous structure or Complex Structure	Trains A1 to A10 inclusive	Trains A1 to A10 inclusive		

Table 2.4: Application of HSLM-A and HSLM-B. Data extracted from EN 1991-2[7]

#### 2.8.5 Dynamic factor

The dynamic factor  $\Phi$  takes account of the dynamic magnification of stress and vibration effects in the structure but does not take account of resonance effects.

Generally the dynamic factor  $\Phi$  is taken as either  $\Phi_2$  or  $\Phi_3$  according to the quality of track maintenance as follows:

(a) For Carefully maintained track:

$$\Phi_2 = \frac{1.44}{\sqrt{L_{\Phi}} - 0.2} + 0.82 \quad with 1.00 \le \Phi_2 \le 1.67 \tag{2.15}$$

(b) For track with standard maintenance:

$$\Phi_3 = \frac{2.16}{\sqrt{L_{\Phi}} - 0.2} + 0.73 \quad with 1.00 \le \Phi_3 \le 2.0$$
(2.16)

where  $L_{\Phi}$  is the 'determinant' length (length associated with  $\Phi$ ) in metres as defined in Table 6.2, EN1991-2[7].

#### 2.8.6 Static analysis

A static analysis shall be carried out with the load models defined in Section Load Models(LM71 and where required Load Models SW/0 and SW/2). The result shall be multiplied by the dynamic factor  $\Phi$  (and if required multiplied by  $\alpha$ )

#### 2.8.7 Bridge parameters

In designers' guide[1], bridge parameters on dynamic effects are discussed.

#### 2.8.7.1 Structural damping

Structural damping is a key parameter in dynamic analysis. The magnitude of the vibrations depends heavily on structural damping, especially in proximity to resonance.

#### 2.8.7.2 Mass of the bridge

Maximum dynamic effects occur at resonance peaks, where a multiple of the load frequency coincides with the natural frequency of the structure. Underrating the mass will lead to over-estimation of the natural frequency of the structure and of the speed at which resonance occurs.

#### 2.8.7.3 Stiffness of the bridge

Maximum dynamic load effects are likely to occur at resonant peaks when a multiple of the frequency of loading and a natural frequency of the structure coincide. Any overestimation of bridge stiffness will overestimate the natural frequency of the structure and speed at which resonance occurs; it provides conservative results.

#### 2.8.8 Dynamic analysis

EN 1991-2[7] doesn't specify the method of dynamic analysing, but UIC Leaflet 776<sup>2</sup>[12] indicates that for normal bridges there are 4 methods of analysing(an approximate method and three simplified method). However, 776<sup>2</sup>[12] also indicates for special structures (bridges with long spans such as bowstring bridges), have to be solved using generic finite element (FEM) programs.

Analysing using FEM programs can be very time and money consuming providing a complicated structure. Fortunately UIC Leaflet 776<sup>2</sup>[12] is an optional reference so FEM analysing is not required in EN 1991-2 [7]. Thus finding a simplified way of analysing complicated bridge structures is vital. The development of dynamic analysing methods will be discussed in Chapter.2.9

#### 2.8.9 Verification of the Limit States

[7, p. 6.4.6.5] proposes following principles to be followed during design: To ensure traffic safety:

- 1. The verification of maximum peak deck acceleration shall be regarded as a traffic safety requirement checked at the serviceability limit state for the prevention of track instability
- 2. The dynamic enhancement of load effects shall be allowed for by multiplying the static loading by the dynamic factor  $\Phi$  defined in [7, p. 6.4.5]. If a dynamic analysis is necessary, the results of the dynamic analysis shall be compared with the results of the static analysis enhanced by  $\Phi$  (and if required multiplied by  $\alpha$  in accordance with [7, p. 6.3.2]) and the most unfavourable load effects shall be used for the bridge design.
- 3. If a dynamic analysis is necessary, a check shall be carried out according to [7, p. 6.4.6.6] to establish whether the additional fatigue loading at high speeds and at resonance is covered by consideration of the stresses due to load effects from  $\Phi \times LM71$  (and if required  $\Phi \times LoadModelSW/0$  for continuous structures and classified vertical load in accordance with [7, 6.3.2(3)] where required). The most adverse fatigue loading shall be used in the design.

#### 2.8.9.1 Ultimate limit states

#### 2.8.9.2 Serviceability limit states - traffic safety

**Vertical accelerations of the deck** The maximum peak values for bridge deck accelerations, calculated along each track, shall not exceed the following design values:

- 1.  $3.5m/s^2$  for ballasted track
- 2.  $5m/s^2$  for direct fastened track with tracks and structural elements designed for high speed traffic

Also, the range of frequencies to take into account in the determination of the dynamic response in terms of accelerations, shall not exceed the maximum of the following values:

- 1. 30Hz
- 2.  $1.5 \times n_0$
- 3. the frequency of the third mode of vibration of the member in study

**Deck twist** The twist of the bridge shall be calculated taking into account the characteristic values of Load Model 71 as well as SW/0 or SW/2 as appropriated multiplied by  $\phi$  and  $\alpha$  and Load Model HSLM including centrifugal effects, all in accordance with EN1991-2, 6.

Twist shall be checked on the approach to the bridge, across the bridge and for the departure from the bridge (see A2.4.4.1(2)P).

The maximum twist t [mm/3m] of a track gauge s [m] of 1.435m measured over a length of 3m(Figure 2.18) should not exceed the values given in Table: 2.5

Speed range $V$ (km/h)	Maximum twist $t \text{ (mm/3m)}$
$v \le 120$	$t \le t_1$
$120 \le v \le 200$	$t \le t_2$
v > 200	$t \le t_3$

Note The values for t may be defined in the National Annex.

The recommended values for the set of t are:

 $t_1 = 4.5$ 

 $t_2 = 3.0$ 

 $t_3 = 1.5$ 

Values for a track with a different gauge may be defined in the National Annex

Table 2.5: Limiting Values of deck twist

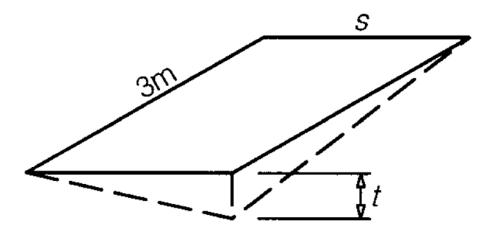


Figure 2.18: Definition of deck twist. Extracted from [6, Figure A2.1]

**Vertical deformations** For all the structures configurations, loaded with the classified characteristic vertical LM 71, or with the models SW/0 and SW/2 if required, the maximum total vertical deflection measured along any track due to railway traffic actions should not exceed L/600. The angular rotations at the end of decks, represented in Figure.2.19, in the vicinity of expansion devices, switches and crossings, should be verified.

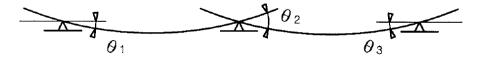


Figure 2.19: Definition of anular rotations at the end of the decks. Extracted from [6, Figure A2.2]

Transverse deformations and vibrations [6, A2.4.4.2.4] proposed that transverse deformation and vibration of the deck shall be checked for characteristic combinations of Load Model 71 and SW/0 as appropriate multiplied by the dynamic factor  $\phi$  and  $\alpha$  (or real train with the relevant dynamic factor if appropriate), wind loads, nosing force, centrifugal forces in accordance with [7, p. 6] and the effect of a transverse temperature differential across the bridge.

The transverse deflection  $\delta_h$  at the top of the deck should be limited to ensure:

- 1. a horizontal angle of rotation of the end of a deck about a vertical axis not greater than the values given in Table. 2.6, or
- 2. the change of radius of the track across a deck is not greater than the values in Table. 2.6, or
- 3. at the end of a deck the differential transverse deflection between the deck and adjacent track formation or between adjacent decks does not exceed the specified value

Speed range V(km/h)	Maximum	Maximum chai	nge of radius of curvature		
	horizon-				
	tal rota-				
tion(radian)					
		Single deck	Multi-deck bridge		
$V \le 120$	$\alpha_1$	$r_1$	$r_4$		
$120 \le V \le 200$	$lpha_2$	$r_2$	$r_5$		
V > 200	$\alpha_3$	$r_3$	$r_6$		
NOTE 1 The change	NOTE 1 The change of the radius of curvature may be determined using:				

$$r = \frac{L^2}{8\delta_h}$$

NOTE 2 The transverse deformation includes the deformation of the bridge deck and the substructure(including piers, piles and foundations).

NOTE 3 The values for the set of  $\alpha_i$  and  $r_i$  may be defined in the National Annex. The recommended values are:

$$\begin{array}{l} \alpha_1=0.0035;\ \alpha_2=0.0020;\ \alpha_3=0.0015;\\ r_1=1700;\ r_2=6000;\ r_3=14000;\\ r_4=3500;\ r_5=9500;\ r_6=17500 \end{array}$$

Table 2.6: Maxiumum horizontal rotation and maximum change of radius of curvature

The first natural frequency of lateral vibration of a span should not be less than  $f_{h0}$ . The value for  $f_{h0}$  may be defined in the National Annex. The recommended value is:  $f_{h0} = 1.2Hz$ 

**Longitudinal displacements** For rails on the bridge and on the adjacent abutment, the permissible additional rail stresses due to the combined response of the structure and the track, due to variable actions, should be limited to the following design values:

1. Compression:  $72KN/mm^2$ 

2. Tension:  $92KN/mm^2$ 

When determining the combined response of track and structure to traction and braking forces, these forces should not be applied on the adjacent embankment unless a complete analysis is carried out considering the approach, passage over and departure from the bridge of rail traffic on the adjacent embankments to evaluate the most adverse load effects.

For the determination of load effects in the combined track/structure system a model based upon Figure 2.20 may be used where the longitudinal load/displacement behaviour of the track or rail supports may be represented by the relationship shown in Figure 2.21.

Model in Figure 2.20 is very important to evaluate the security of the track structure and not the structural security. High track deformations can lead to unfavourable effects for the structure and for vehicles when these are crossing the bridge.

#### 2.8.9.3 Serviceability limit states - passenger comfort

For these type of verifications [6] defines the limiting values for the maximum vertical deflection for passenger comfort, as following:

- 1. Comfort criteria
- 2. Deflection criteria for checking passenger comfort
- 3. Requirements for a dynamic vehicle/bridge interaction analysis for checking passenger comfort

Passenger comfort depends on the vertical accelerations,  $b_v$ , inside the coach. These levels of comfort limiting values for the vertical accelerations are presented in Table.2.7

Very good	$1.0m/s^{2}$
$\operatorname{Good}$	$1.3m/s^{2}$
Acceptable	$2.0m/s^{2}$

Table 2.7: Recommended accelerations values to ensure the respective levels of comfort. Extracted from [6, Table A 2.9].

In order to limit vertical vehicle acceleration, being the limits defined in Table 2.7, vertical displacements should be less than the maximum permissible vertical deflection,  $\delta$ , obtained from Figure 2.22. These values are expressed in function of the span length L[m], and train speed V[km/h], which is valid only for railway bridges with three or more successive simply supported spans. Alternatively these accelerations can be determined considering the vehicle-structure interaction dynamic analysis.

Additionally, the limiting values of  $L/\delta$ , defined in Figure 2.22 are given for  $b_v = 1.0 m/s^2$ .

Vertical deflections should be determined with the LM 71 model multiplied by the factor  $\phi$  and adopting  $\alpha = 1$ , being only one track loaded for the case of bridges with two or more tracks.

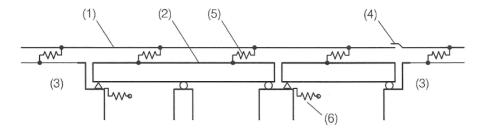
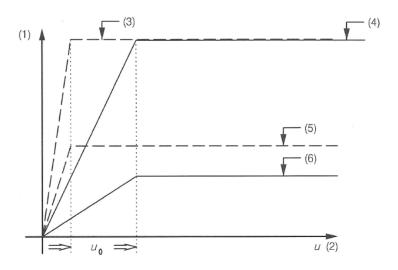


Figure 2.20: Model of a track/structure system. Extracted from [7, Figure 6.19]



#### Key

- (1) Longitudinal shear force in the track per unit length
- (2) Displacement of the rail relative to the top of the supporting deck
- Resistance of the rail in sleeper (loaded track)
   (frozen ballast or track without ballast with conventional fastenings)
- (4) Resistance of sleeper in ballast (loaded track)
- (5) Resistance of the rail in sleeper (unloaded track) (frozen ballast or track without ballast with conventional fastenings)
- (6) Resistance of sleeper in ballast (unloaded track)

Figure 2.21: Variation of longitudinal shear force with longitudinal track displacement for one track. Extracted from [7, Figure 6.20]

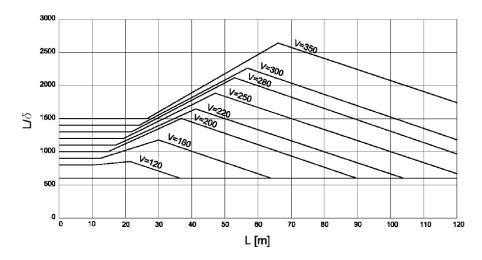


Figure 2.22: Maximum permissible vertical deflection  $\delta$  for railway bridges with 3 or more successive simply supported spans corresponding to a permissible vertical acceleration of  $b_v = 1m/s^2$  in a coach for speed V[km/h]. Extracted from [6, Figure A2.3]

#### 2.8.10 Principal supplementary checks

- 2.8.10.1 Verification of maximum peak deck acceleration along each track
- 2.8.10.2 Verification of whether the calculated load effects from high-speed rail traffic, including HSLM on high-speed interoperable routes, are greater than those of normal rail traffic loading(LM71"+"SW/0)
- 2.8.10.3 Additional verification for fatigue where dynamic analysis is required
- 2.8.10.4 Verification of limiting values for the maximum vertical deflection for passenger comfort

#### 2.8.11 Diagram of general procedures

By summarizing Eurocode several steps of calculation are extracted as following, arranged in chronological order.

- 1. Follow the conceptual check to avoid unsafe designs
- 2. Follow the logic diagram in Figure 2.12a to check whether dynamic analyses are required
- 3. Find the appropriate train models, including
  - (a) -Hypotheses relating to rolling stock
  - (b) -Rolling stock for interoperability
  - (c) -Load models HSLM
  - (d) -Load distribution
  - (e) -Load combinations and partial factors
  - (f) -Train speeds to be considered
- 4. Perform the static analyses

- 5. Examine bridge parameters, including
  - (a) -Structural damping
  - (b) -Mass of the bridge
  - (c) -Stiffness of the bridge
- 6. Perform the dynamic analyses
- 7. Principal supplementary design checks, including
  - (a) -Verification of maximum peak deck acceleration along each track
  - (b) -Verification of whether the calculated load effects from high-speed rail traffic, including HSLM on high-speed interoperable routes, are greater than those of normal rail traffic loading (LM71"+"SW/0)
  - (c) -Additional verification for fatigue where dynamic analysis is required
  - (d) -Verification of limiting values for the maximum vertical deflection for passenger comfort
- 8. The results of the dynamic analysis shall be compared with the results of the static analysis multiplied by the dynamic factor  $\Phi$  in 6.4.5 The most unfavourable values of the load effects shall be used.

### 2.9 Dynamic analysing methods

There are several dynamic analysing methods developed in Europe over time since Eurocodes don't specify what method to be used during dynamic analysing. These methods differ from each other in the level of calculation complexity. For example, Dynamic train signature method is a good solution for simple structures with well-known train types since it cost less time and effort. On the other hand, Train-vehicle method is an inevitable process for some complicated bridge structures thanks to its wide applicability. However, generally, Train-vehicle method is much more time and money consuming. In following paragraphs, some state-of-art dynamic analysing methods will be reviewed.

#### 2.9.1 Method based on impact factor

#### 2.9.2 Method based on dynamic train signature

As mentioned in [12, A.4.3], the dynamic signature of a train is obtained by breaking down the load diagram of a train in Fourier series and by extrapolating it to the natural modes. It represents the dynamic excitation features of the train and is independent of the characteristics of the structure. The signature depends on axle spacing and loads only. However, newly developed Train dynamic signature method like LIR method takes structure characteristics into account, giving more applicable solutions for specific bridge projects.

Train dynamic signature is a useful method of producing quick analyzes on resonance characteristics of the vehicle-bridge systems. The method is especially effective for simple bridge structures.

#### 2.9.2.1 DER method

DER = Decomposition of the Resonance Excitation

The development used in DER method begins from the analysis of the frequency of excitation produced by a train of moving loads. This method is based in the following assumptions:

- Applicable only on statistically determined bridges

- For the analysis of statistically determined bridges, is considered that the dynamic response is significantly represented by the first mode of vibration

The development of the method is summarised as follows:

- 1. Reduce the response of a statistically determinate beam to a single degree of freedom system
- 2. Decomposition of the dynamic response of the bridge, in Fourier series;
- 3. Consideration of the term which corresponds to the condition of resonance frequencies.

The maximum accelerations at the midspan of the beam for a certain speed, is given as follows:

$$\ddot{y}(t) \le C_t \cdot A(L/\lambda) \cdot G(\lambda) \tag{2.17}$$

where the first factor is a constant that depends from bridge characteristics:

$$C_t = \frac{8\pi f_0^2}{K} = \frac{4}{\rho \pi L} \tag{2.18}$$

The second factor is a function called dynamic influence line:

$$A(L/\lambda) = \left| \frac{\cos(\pi L/\lambda)}{(2L/\lambda)^2 - 1} \right| \tag{2.19}$$

and the third factor represents the train dynamic signature, defined as follows:

$$G(\lambda) = \sqrt{\left[\sum_{k=1}^{N} F_k \cos(\frac{2\pi x_k}{\lambda})\right]^2 + \left[\sum_{k=1}^{N} F_k \sin(\frac{2\pi x_k}{\lambda})\right]^2} \cdot (1 - e^{-2\pi \xi} \frac{x_N}{\lambda}) \cdot \frac{L}{\xi x_N}$$
(2.20)

#### 2.9.2.2 LIR method

LIR method is based on residual influence line. It is applicable only on statistically determined bridges, too. The solution of the displacements and accelerations in the midspan, of the simply supported beam, is developed by Domínguez Barbero. The solution is given as follows:

$$y_{max} = C_{desp} \cdot A(r) \cdot G(\lambda) \tag{2.21}$$

$$\ddot{y}_{max} = C_{acel} \cdot A(r) \cdot G(\lambda) \tag{2.22}$$

with,

$$C_{desp} = \frac{1}{M\omega_0^2}, C_{acel} = \frac{1}{M}$$
 (2.23)

The factor A(r) is the dynamic influence line, give as:

$$A(r) = \frac{r}{1 - r^2} \sqrt{e^{-2\xi \frac{\pi}{r}} + 1 + 2\cos(\frac{\pi}{r})e^{-2\xi \frac{\pi}{r}}}$$
 (2.24)

with  $r = \lambda/2L$ .

 $G(\lambda)$  is named train dynamic signature, depending from train characteristics and from the damping coefficient of the structure, give as follows:

$$G(\lambda) = \sqrt{\left[\sum_{i} F_{i} \cos\left(2\frac{\pi x_{i}}{\lambda}\right) e^{-2\frac{\pi \xi x_{i}}{\lambda}}\right]^{2} + \left[\sum_{i} F_{i} \sin\left(2\frac{\pi x_{i}}{\lambda}\right) e^{-2\frac{\pi \xi x_{i}}{\lambda}}\right]^{2}}$$
(2.25)

To make  $G(\lambda)$  representative for the maximum response, maximum value of  $G(\lambda)$  for each different subtrain is considered the value of  $G(\lambda)$ 

$$G(\lambda) = \max_{i=1}^{N} \sqrt{\left[\sum_{x_1}^{x_i} F_i \cos(2\pi\delta_i) e^{-2\pi\xi\delta_i}\right]^2 + \left[\sum_{x_1}^{x_i} F_i \sin(2\pi\delta_i) e^{-2\pi\xi\delta_i}\right]^2}$$
(2.26)

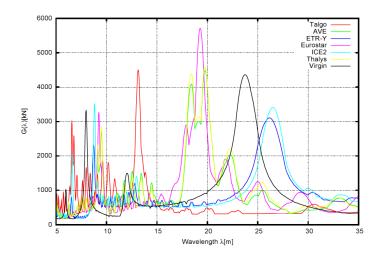


Figure 2.23: Train dynamic signatures for the seven real trains, considering a damping value of  $\xi = 0.00$ . Extracted from [13, Figure 2.20]

#### 2.9.3 Methods based on finite element models

Finite element methods are the most applicable methods available. The methods are based on direct time integration.

#### 2.9.3.1 Direct time integration methods

General equation of motion for a SDOF system can be given in following form:

$$M\ddot{d} + C\dot{d} + Kd = f(t) \tag{2.27}$$

where M is the mass matrix, c is the damping matrix, K is the stiffness matrix, f(t) is the vector of external loads and d the unknown vector of nodal displacements.

Direct time integration method is the principle of finite element models. Due to the expected huge amount of calculation, computers are used to process time integration calculations. Thanks to the reliability and efficiency of computers, there are more and more project done with the help of computer FEM software nowadays.

#### 2.9.3.2 Modelling a train of moving loads

The method of modelling spatial moving loads in FEM software is applying load histories in each convenient node. At a certain time-step, a load F, whose magnitude depending linearly on the distance from the axle to the node, is assigned to each node if the load axis is above an element that contains the node. The procedure is outlined in Figure 2.24.

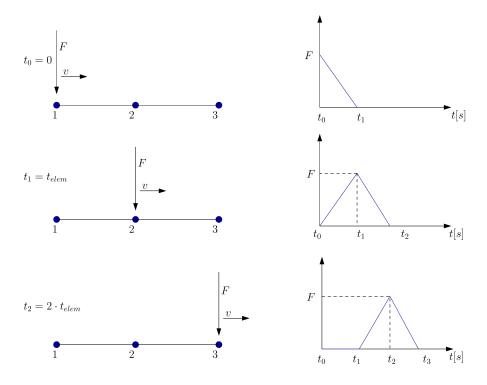


Figure 2.24: Nodal force time history definition for a single moving load F, with speed v. Extracted from [13, Figure 2.15]

#### 2.9.4 Analytical methods based on modal analysis

Modal analysis is the study of the dynamic properties of structures under vibrational excitation. Applied on railway bridge structures, the analysis can be simple if the bridge is modelled as a simply supported beam.

#### 2.9.4.1 Modal analysis of a simply supported beam

blahblahblahblah stuff

Hello, here is some text without a meaning. This text should show what a printed text will look like at this place. If you read this text, you will get no information. Really? Is there no information? Is there a difference between this text and some nonsense like "Huardest gefburn"? Kjift – not at all! A blind text like this gives you information about the selected font, how the letters are written and an impression of the look. This text should contain all letters of the alphabet and it should be written in of the original language. There is no need for special content, but the length of words should match the language.

Contents to be added

#### 2.9.4.2 Modes of vibration

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written in of the original language. There is no need for special content, but the length of words should match the language.

#### 2.9.4.3 Number of modes of vibration to consider in the analysis

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#### 2.9.5 Method based on vehicle-Structure interaction dynamic analysis

Different from point load models, vehicle-structure models takes suspension systems of train vehicles into account, providing associations between train carriage and structure. Impact of suspension system vibration on bridge structure can be neglected when the span of the bridge is comparably small since there's little chance for suspension system and bridge to resonant. As the span of bridges increase, the first vertical/transverse natural frequency of bridge can decrease into the natural frequency range of train suspension system. This means long-span bridges can resonant with train suspension system. To study the effects related to train suspension system, vehicle-structure interaction method is developed.

A general model for a conventional coach on two bogies are shown in Figure 2.25, including the stiffness and damping  $(K_P, c_P)$  of the primary suspension of each axle, the secondary suspension of the bogies  $(K_s, c_s)$ , the unsprung mass of the wheels  $(M_w)$ , the bogies  $(M_b, J_b)$ , and the vehicle body (M, J). Similar models may be developed for articulated or regular trains.

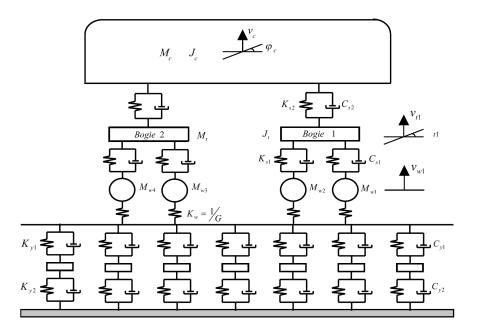


Figure 2.25: Model for analysis of vehicle-structure system. Extracted from [11, Figure 12]

Sometimes more simplified model, represented by one mass, one spring and one damper per

bogie can be considered, depending on the purpose of the analysis. See Figure 2.26 for example.

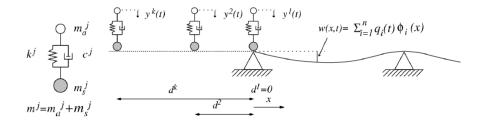


Figure 2.26: Load train with vehicle-bridge interaction: simplified interaction model and variables definition. Extracted from [11, Figure 15]

# POSSIBLE THEORETICAL RESEARCH

- 3.1 Horizontal Train dynamic signature method for bridge lateral frequency under 1.2Hz
- 3.2 3-d model Train dynamic signature method for bridge lateral frequency under 1.2Hz
- 3.3 Study for bridge horizontal vibration criteria
- 3.4 Study for sufficiency of criterion 'Lateral first natural frequency udner 1.2Hz'

# Case study

A case study to assess the feasibility of the methods/criteria developed in previous paragraphs by using both FEW software and methods/criteria.

# Conclusion

Compare the result from different calculation method. Give conclusion based on the comparison result.

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