## Numerical Methods in Mechanics and Environmental Flows

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## Outline for Environmental Flows

#### Oct 13

- Introduction
- Delft3D Introduction and Assignment 1: Spin-up?

#### Oct 20

- Box models and solution methods
- Delft3D Assignment 2 Boundary conditions; initial conditions

#### Oct 27

- Solution methods: vertical layers / transport processes
- Delft3D Assignment 3 stratification (wind-driven flows)

#### Nov 3

- Transport processes in flows (2)
- Delft3d Project

#### **Nov 10**

- Transport processes in flows (3)
- Delft3d assignment 4 model extents (estuarine stratification as an example)

#### **Nov 17**

- Presentation of term assignment (4 groups)
- Revision / past exam questions

## Last week

#### Looked at numerical solutions:

- What are needed for numerical solutions?
  - The Mathematical model
  - Discretization method
  - Solver
- Expected errors
  - Modeling
  - Discretization
  - Iteration
- Why? Due to the components which affect the solutions:
  - Grids
  - Approximations
  - Solution methods

## Reservoir with pollutant

### Reservoir with following conditions:

• V =  $2x10^5$ m³; Q<sub>u/s</sub> =  $9x10^4$ m³/yr; Q<sub>evap</sub> =  $1x10^4$ m³/yr; Assume steady state; upstream c = 6 mg/l; c decays at K = 0.12/year

#### Find c



- What is budget?
- What is then c?

What if now upstream c = 0 due to changes in management?

- What is budget?
- How long does it take to drop to 50%
- How long does it take to reach 0.1 mg/l?
- What is the main cause of the improvement in c?

## Basic Flow Transport Processes

#### Basic transport processes:

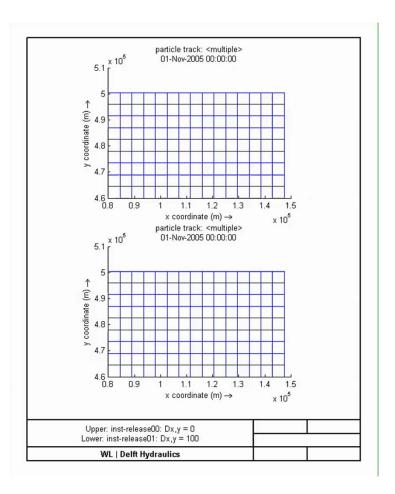
• A, D, S

## Main difference between advection and diffusion:

• 1-D Example 
$$\frac{\partial c}{\partial t} + u \frac{\partial c}{\partial x} = D_x \left( \frac{\partial^2 c}{\partial x^2} \right)$$

#### Diffusion occurs because of:

- · 1.
- · 2.



## The use of the 1-D Diffusion Equation

Derivation from Fick's Law and conservation of

**mass:** 
$$\frac{\partial c}{\partial t} = \frac{\partial}{\partial x} \left( D \frac{\partial c}{\partial x} \right)$$

• Important because it is a linear equation!

Started with solution for infinite domain.

Obtained solutions for finite domains

• Concentration 
$$c(x,t) = \frac{M}{\sqrt{4\pi Dt}} \left[ \exp\left(-\frac{\left(x - 2mL - a\right)^2}{4Dt}\right) + \exp\left(-\frac{\left(x - 2mL + a\right)^2}{4Dt}\right) \right]$$

• Mixing time: 
$$T = 0.134 \frac{L^2}{D}$$
 or  $T = 0.536 \frac{L^2}{D}$ 

Extended to include Decay and Source

## Scales & the issue of physical diffusion (1)

Different D (or K, k) values depending on scale.

$$\frac{\partial c}{\partial t} + u \frac{\partial c}{\partial x} + v \frac{\partial c}{\partial y} + w \frac{\partial c}{\partial z} = D \left( \frac{\partial^2 c}{\partial x^2} + \frac{\partial^2 c}{\partial y^2} + \frac{\partial^2 c}{\partial z^2} \right)$$

- All scales solved: true  $D \sim O(10^{-9}) \frac{m^2}{s} < v$
- Averaged 3D:  $K_z \sim O() \frac{m^2}{s}$
- 2D Depth-average:  $k_{x,y} \sim O() \frac{m^2}{s}$
- $\circ$  1D: $K \sim O() \frac{m^2}{s}$

## Scales & the issue of physical diffusion (2)

### How does averaging create this difference?

• What is averaging?

$$u = \bar{u} + u'$$
 (etc.)

All scales solved: 
$$\frac{\partial c}{\partial t} + u \frac{\partial c}{\partial x} + v \frac{\partial c}{\partial y} + w \frac{\partial c}{\partial z} = D \left( \frac{\partial^2 c}{\partial x^2} + \frac{\partial^2 c}{\partial y^2} + \frac{\partial^2 c}{\partial z^2} \right)$$

Averaged 3D:

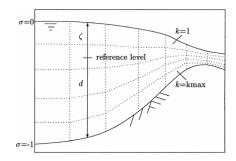
$$\frac{\partial c}{\partial t} + \frac{\partial (uc)}{\partial x} + \frac{\partial (vc)}{\partial y} + \frac{\partial (wc)}{\partial z} = \frac{\partial}{\partial x} \left( K_x \frac{\partial c}{\partial x} \right) + \frac{\partial}{\partial y} \left( K_y \frac{\partial c}{\partial y} \right) + \frac{\partial}{\partial z} \left( K_z \frac{\partial c}{\partial z} \right)$$

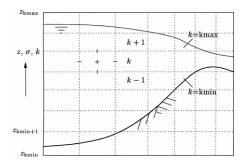
• 2D Depth-average:  $\frac{\partial(ch)}{\partial t} + \frac{\partial(uhc)}{\partial x} + \frac{\partial(vhc)}{\partial y} = \frac{\partial}{\partial x} \left( hk_x \frac{\partial c}{\partial x} \right) + \frac{\partial}{\partial y} \left( hk_y \frac{\partial c}{\partial y} \right)$ 

• 1D: 
$$\frac{\partial(cA)}{\partial t} + \frac{\partial(uAc)}{\partial x} = \frac{\partial}{\partial x} \left( AK \frac{\partial c}{\partial x} \right)$$

# Artificial diffusion due to vertical grid coordinate system

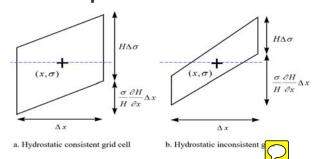
#### Two main types:





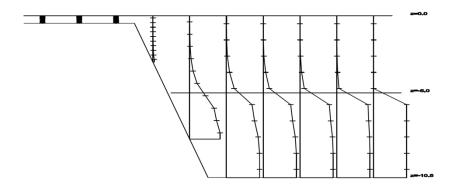
### Issue arises due to transformation of equations

- Results in hydrostatic inconsistency
- Essentially  $\rightarrow \frac{\sigma}{H} \frac{\partial H}{\partial x} \Delta x < \Delta \sigma$

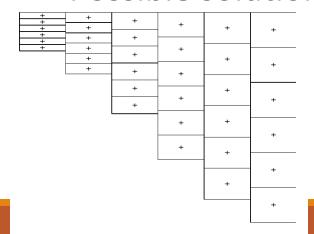


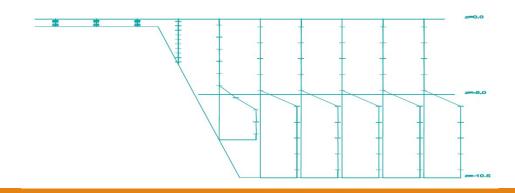
## Discussed the Effect and solution

## Artificial diffusion and mixing



## Possible solution:





## Turbulent mixing and models

## Introduced the issue of turbulent mixing and its importance on mixing

$$\frac{\partial u}{\partial t} + \frac{u}{\sqrt{G_{\xi\xi}}} \frac{\partial u}{\partial \xi} + \frac{v}{\sqrt{G_{\eta\eta}}} \frac{\partial u}{\partial \eta} + \frac{\omega}{d+\zeta} \frac{\partial u}{\partial \sigma} - \frac{v^2}{\sqrt{G_{\xi\xi}}} \frac{\partial \sqrt{G_{\eta\eta}}}{\partial \xi} + + \frac{uv}{\sqrt{G_{\xi\xi}}\sqrt{G_{\eta\eta}}} \frac{\partial \sqrt{G_{\xi\xi}}}{\partial \eta} - fv = -\frac{1}{\rho_0 \sqrt{G_{\xi\xi}}} P_{\xi} + F_{\xi} + + \frac{1}{(d+\zeta)^2} \frac{\partial}{\partial \sigma} \left( \nu V \frac{\partial u}{\partial \sigma} \right) + M_{\xi}, \quad (9.6)$$

$$\frac{\partial v}{\partial t} + \frac{u}{\sqrt{G_{\xi\xi}}} \frac{\partial v}{\partial \xi} + \frac{v}{\sqrt{G_{\eta\eta}}} \frac{\partial v}{\partial \eta} + \frac{\omega}{d+\zeta} \frac{\partial v}{\partial \sigma} + \frac{uv}{\sqrt{G_{\xi\xi}}} \frac{\partial \sqrt{G_{\eta\eta}}}{\partial \xi} + \frac{v}{\partial \xi} + \frac{v}{\sqrt{G_{\xi\xi}}} \frac{\partial v}{\partial \eta} + \frac{v}{\partial \xi} + \frac{v}{\sqrt{G_{\eta\eta}}} \frac{\partial v}{\partial \xi} + fu = -\frac{1}{\rho_0 \sqrt{G_{\eta\eta}}} P_1 + F_{\eta} + \frac{1}{(d+\zeta)^2} \frac{\partial}{\partial \sigma} \left( \nu_V \frac{\partial v}{\partial \sigma} \right) + M_{\eta}. \quad (9.7)$$

$$\begin{split} \frac{\partial \left(d+\zeta\right)c}{\partial t} + \frac{1}{\sqrt{G_{\xi\xi}}\sqrt{G_{\eta\eta}}} \left\{ \frac{\partial \left[\sqrt{G_{\eta\eta}}\left(d+\zeta\right)uc\right]}{\partial \xi} + \frac{\partial \left[\sqrt{G_{\xi\xi}}\left(d+\zeta\right)vc\right]}{\partial \eta} \right\} + \frac{\partial \omega c}{\partial \sigma} = \\ \frac{d+\zeta}{\sqrt{G_{\xi\xi}}\sqrt{G_{\eta\eta}}} \left\{ \frac{\partial}{\partial \xi} \left[ \frac{D_H}{\sigma_{c0}} \frac{\sqrt{G_{\eta\eta}}}{\sqrt{G_{\xi\xi}}} \frac{\partial c}{\partial \xi} \right] + \frac{\partial}{\partial \eta} \left[ \frac{D_H}{\sigma_{c0}} \frac{\sqrt{G_{\xi\xi}}}{\sqrt{G_{\eta\eta}}} \frac{\partial c}{\partial \eta} \right] \right\} + \\ + \frac{1}{d+\zeta} \frac{\partial}{\partial \sigma} \left[ \frac{\nu_{mol}}{\sigma_{mol}} + \max \left( \frac{\nu_{3D}}{\sigma_c}, D_V^{back} \right) \frac{\partial c}{\partial \sigma} \right] - \lambda_d \left(d+\zeta\right) c + S, \quad (9.29) \end{split}$$

## Mixing and Turbulence

WHAT, WHY, HOW

## What, where is turbulence?

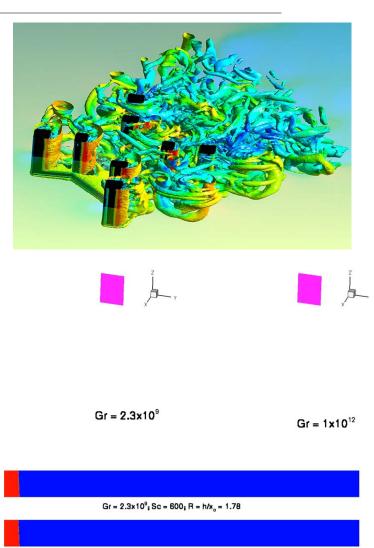
#### Everywhere

 Essentially all environmental flows are turbulent

#### Characteristics of turbulent flows:

- Highly unsteady
- 3-Dimensional
- High vorticity
- Fluctuates over a large range of length and time scales.
- Unpredictable (inherent instability)

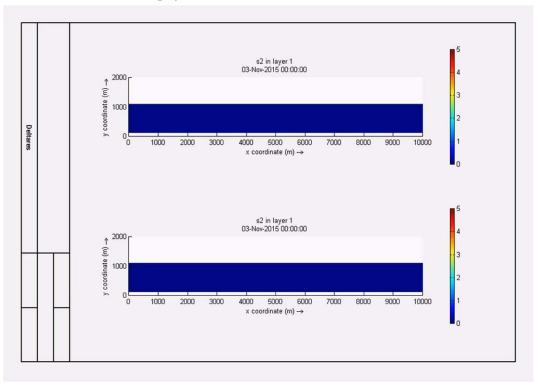
Great → More effective mixing as there is more agitation → Larger D!



 $Gr = 1 \times 10^{12}$ ; Sc = 600;  $R = h/x_0 = 1.78$ 

# Are there simple ways to analyze turbulent mixing?

Shear flows are a starting point:

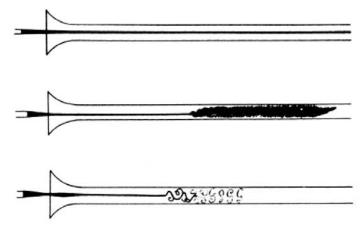


But fully turbulent flows have an unsteady velocity profile which restricts this method of analysis

- So how do we proceed?

## Let's classify turbulence

3 observations by Reynolds about dye streaks



### Types of flows and its implications:

- Laminar → dependent on molecular diffusion
- Turbulent 

  creation of eddies which are unstable and grow

What does turbulence do then?

## Describing turbulence

Simplest turbulent flow concept: Homogeneous turbulence..?

Statistically steady (no spatial gradients)

#### Energy cascade

- Eddies going from large → small
- HOWEVER very little loss of energy until viscosity takes over → very small scales!
- Conversion of KE to heat at small scales ightarrow DISSIPATION:  $\epsilon = \frac{\text{dissipated kinetic energy}}{\text{times}}$
- → P must equal E in a homogeneous turbulent flow

Which leads to the  $2^{nd}$  concept - length scale,  $L_K$ 

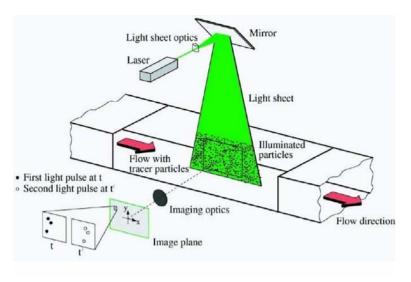
- How Large?
- Dimensional Analysis and physical understanding
- L<sub>K</sub> must depend on rate of dissipation and viscosity

Can we prove it?

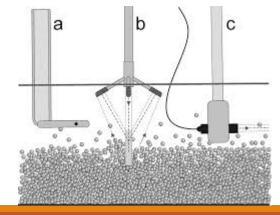
## Can we measure eddies?

#### A few different methods exist to measure

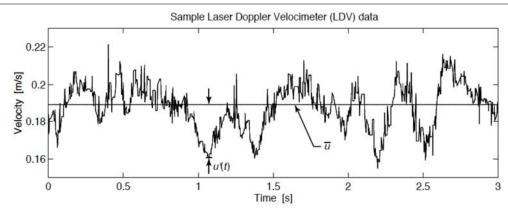
Lagrangian view (PTV / PIV)



Eulerian view (velocity probes)



## A sample result and its implication



#### What do we see?

- A large spectrum of velocity fluctuations
- Different periods which we can correlate to eddy sizes
  - Large Eddy → Long Period; Small Eddy → Short Period
- Short time → High correlation;
- Longer time → lower correlation → eventually seemingly random
  - Integral Time Scale → Characteristic Length, Velocity

This allows us to use Reynolds Decomposition!

## Remember Reynolds Decomposition?

Decompose velocity into mean and fluctuating components  $\rightarrow u_i(x_i,t) = \overline{u_i}(x_i) + u_i'(x_i,t)$ .

 $^{\circ}$  Essentially Integral Time Scale becomes the time for  $\overline{u}_i$  to become steady/constant

Which allows us to have a 3<sup>rd</sup> important quantity

$$u_{rms} = \sqrt{\overline{u'u'}}$$

• Why is this important?



## Turbulence and mixing (1)

## Let's investigate using the 1D transport equation and introduce 2 concepts

Reynolds Decomposition

$$C(x,t) = \overline{C}(x) + C'(x,t)$$

And Time Averaging

$$q_{x} = \overline{uC}$$

$$\overline{uC} = \frac{1}{t_{I}} \int_{t_{I}}^{t+t_{I}} uC \, d\tau = \overline{\left(\overline{u}_{i} + u'_{i}\right)\left(\overline{C} + C'\right)} = \overline{u'C'} + \overline{uC}$$

## Turbulence and mixing (2)

The advective-diffusive equation  $\frac{\partial c}{\partial t} + \frac{\partial uc}{\partial x} = \frac{\partial}{\partial x} \left( D \frac{\partial c}{\partial x} \right)$ 

Transforms into 
$$\frac{\partial \overline{c}}{\partial t} + \overline{u} \frac{\partial \overline{c}}{\partial x} = -\frac{\partial \overline{u'c'}}{\partial x} + \frac{\partial}{\partial x} \left( D \frac{\partial \overline{c}}{\partial x} \right)$$

A Fickian Relationship can be derived  $\rightarrow$ 

$$\overline{u'c'} = D_t \frac{\partial \overline{c}}{\partial x}; \text{ where } D_t = u_I L_I$$

Allowing us to re-write the equation as

$$\frac{\partial \overline{c}}{\partial t} + \overline{u} \frac{\partial \overline{c}}{\partial x} = \frac{\partial}{\partial x} \left( D_t \frac{\partial \overline{c}}{\partial x} \right) + \frac{\partial}{\partial x} \left( D \frac{\partial \overline{c}}{\partial x} \right)$$

## Can we estimate D<sub>t</sub> for Open Channel Flows?

### Take a river as an example

#### Assume

- Wide (W >> H); how does this help?
- Turbulence is generated in high shear zones →
  - Where would you expect this to occur?
  - $\circ$  How does this help us ightarrow Shear velocity :  $u_* = \sqrt{rac{ au_0}{
    ho}}$
- Uniform flow  $\rightarrow$  so?

A logical deduction is that  $D_t$  will not be isotropic.  $\square$  Why?

## Then how do we estimate $D_t$ ?

## In the vertical (z)

• From log-law  $\overline{u_t}(z) = \overline{u} + \frac{u_*}{\kappa} (1 + \ln(z/h))$   $\rightarrow$   $D_t = 0.067 u_* d$ 

## In the transverse (y)

- From measurements
  - Lab  $D_t = 0.15u_*d$



• Field  $D_t = 0.60u_*d$ 

## And in the longitudinal (x)

- One can expect this to be the same as transverse dispersion. If we assume...
- However, since the vertical profile is not uniform → there
  is another process that dominates mixing in the
  longitudinal direction, which allows us to ignore this

## So let's compare

$$B = 10m$$
,  $d = 0.3m$ ,  $Q = 1 m^3/s$ ,  $S = 0.0005$ 

#### From these formulas

$$D_t = 0.067 u_* d$$

$$D_t = 0.60u_*d$$

## TURBULENCE CLOSURE MODELS

MODELING TURBULENT MIXING

## The need for turbulence closure models

If we could solve all flow scales practically, then there is no need for turbulence closure models.

The issue arises from the need to achieve a solution practically

 Without averaging, one has to carry out Direct Numerical Simluation (DNS) which requires MASSIVE computational resources (memory, CPUs and CPU speed) due to this approximate relationship

$$N^3 \ge \text{Re}^{2.25}$$

However averaging over space (LES) or time (RANS) results in the problem of an additional tensor term.

## The additional tensor term:

LES:

$$\frac{\partial \bar{u_i}}{\partial t} + \frac{\partial}{\partial x_j} \left( \overline{u_i u_j} \right) = -\frac{1}{\rho} \frac{\partial \overline{p}}{\partial x_i} + \nu \frac{\partial}{\partial x_j} \left( \frac{\partial \bar{u_i}}{\partial x_j} + \frac{\partial \bar{u_j}}{\partial x_i} \right) = -\frac{1}{\rho} \frac{\partial \overline{p}}{\partial x_i} + 2\nu \frac{\partial}{\partial x_j} S_{ij},$$

**RANS:** 

• Momentum 
$$\frac{\partial \overline{u}_{i}}{\partial t} + \overline{u}_{j} \frac{\partial \overline{u}_{i}}{\partial x_{j}} = -\frac{\partial P}{\partial x_{i}} + v \frac{\partial^{2} \overline{u_{i}}}{\partial x_{i} x_{j}} - \underbrace{\partial u_{i}' u_{j}}{\partial x_{j}}$$
• Scalar 
$$\frac{\partial \overline{c}}{\partial t} + \overline{u}_{j} \frac{\partial \overline{c}}{\partial x_{j}} = \frac{\partial}{\partial x_{j}} \left( D \frac{\partial \overline{c}}{\partial x_{j}} \right) - \underbrace{\partial u_{i}' u_{j}}{\partial x_{j}}$$

Focusing on RANS, how do we solve this?

## What is the issue of the additional term?

$$\frac{\partial \overline{u}_{i}}{\partial t} + \overline{u}_{j} \frac{\partial \overline{u}_{i}}{\partial x_{j}} = -\frac{\partial P}{\partial x_{i}} + v \frac{\partial^{2} \overline{u}_{i}}{\partial x_{i} x_{i}} \left( \frac{\partial \overline{u}_{i}' \overline{u}_{j}}{\partial x_{j}} \right)$$

We now have more unknowns than equations!

- These terms didn't exist originally and arise due to the averaging process.
- Called the Reynolds Stress Tensor!

To solve this problem one needs either ext parameters or equations (called closure models) to have a unique solution

Simplest models use the Boussinesq hypothesis

$$\overline{u_i'u_j'} = \frac{2}{3}k\delta_{ij} + v_i \left(\frac{\partial \overline{u_i}}{\partial x_j} + \frac{\partial \overline{u_j}}{\partial x_i}\right)$$
where  $k = \frac{1}{2}\overline{u_i'u_i}$  EDDY VISCOSITY

## Where is this term?

### In Delft3D for example:

$$\frac{\partial u}{\partial t} + \frac{u}{\sqrt{G_{\xi\xi}}} \frac{\partial u}{\partial \xi} + \frac{v}{\sqrt{G_{\eta\eta}}} \frac{\partial u}{\partial \eta} + \frac{\omega}{d+\zeta} \frac{\partial u}{\partial \sigma} - \frac{v^2}{\sqrt{G_{\xi\xi}}} \frac{\partial \sqrt{G_{\eta\eta}}}{\partial \xi} + + \frac{uv}{\sqrt{G_{\xi\xi}}\sqrt{G_{\eta\eta}}} \frac{\partial \sqrt{G_{\xi\xi}}}{\partial \eta} - fv = -\frac{1}{\rho_0 \sqrt{G_{\xi\xi}}} P_{\xi} + F_{\xi} + + \frac{1}{(d+\zeta)^2} \frac{\partial}{\partial \sigma} \left( vv \frac{\partial u}{\partial \sigma} \right) + M_{\xi}, \quad (9.6)$$

$$\frac{\partial v}{\partial t} + \frac{u}{\sqrt{G_{\xi\xi}}} \frac{\partial v}{\partial \xi} + \frac{v}{\sqrt{G_{\eta\eta}}} \frac{\partial v}{\partial \eta} + \frac{\omega}{d+\zeta} \frac{\partial v}{\partial \sigma} + \frac{uv}{\sqrt{G_{\xi\xi}}} \frac{\partial \sqrt{G_{\eta\eta}}}{\partial \eta} \frac{\partial \sqrt{G_{\eta\eta}}}{\partial \xi} + \frac{v}{\sqrt{G_{\xi\xi}}} \frac{\partial v}{\partial \eta} + fu = -\frac{1}{\rho_0 \sqrt{G_{\eta\eta}}} P_{\eta} + F_{\eta} + \frac{1}{(d+\zeta)^2} \frac{\partial}{\partial \sigma} \left( \nu_V \frac{\partial v}{\partial \sigma} \right) + M_{\eta}. \quad (9.7)$$

## Closing the Eddy Viscosity Relation

4 ways that have been proposed to calculate the eddy viscosity include

- Constant coefficient [Not strictly a Boussinesq assumption]
- Zero Equation
- One Equation
- Two Equation

Boussinesq assumption of eddy viscosity assumes that there is a relationship between characteristic length and velocity scales

$$v_{t} = C_{\mu} \sqrt{kL}$$

## Zero Equation Models

The zero-equation is the simplest turbulence closure model, we only need to use analytical formulas to obtain k and L to solve the relationship:

$$v_{t} = C_{u} \sqrt{k} L$$

 $v_{t} = C_{\mu} \sqrt{k} L$  • Typically k is calculated directly from the flow; in Delft3D

$$k = \frac{1}{\sqrt{c_{\mu}}} L^2 \left[ \left( \frac{\partial u}{\partial z} \right)^2 + \left( \frac{\partial v}{\partial z} \right)^2 \right] \quad \text{or} \quad k = \frac{1}{\sqrt{c_{\mu}}} \left[ \left( u_*^b \right)^2 \left( 1 - \frac{z + d}{H} \right) + u_{*8}^2 \frac{z + d}{H} \right]$$

And a relationship to obtain L is derived from either

$$L = \kappa(z+d)\sqrt{1 - \frac{z+d}{H}}f(Ri) \quad \text{or} \quad \sqrt{k} = L\frac{\partial u}{\partial y}$$

Where 
$$Ri = \frac{-g \frac{\partial \rho}{\partial z}}{\rho \left[ \left( \frac{\partial u}{\partial z} \right)^2 + \left( \frac{\partial v}{\partial z} \right)^2 \right]}$$

## One Equation Model

#### One equation models

solve for k through a transport equation e.g.

$$\frac{\partial k}{\partial t} + \overline{u}_j \frac{\partial k}{\partial x_j} = +D_t \frac{\partial^2 k}{\partial x_i x_j} + P_k + P_{kw} + B_k - \varepsilon$$

• And prescribe L (typically using the previous relationships shown for the zero-equation model) and then using the same viscosity relationship to calculate viscosity  $\rightarrow v_t = C_u \sqrt{k} L$ 

### In Delft3D the transport equation for k is:

$$\frac{\partial k}{\partial t} + \frac{u}{\sqrt{G_{\xi\xi}}} \frac{\partial k}{\partial \xi} + \frac{v}{\sqrt{G_{\eta\eta}}} \frac{\partial k}{\partial \eta} + \frac{\omega}{d + \zeta} \frac{\partial k}{\partial \sigma} = P_k = \nu_V \frac{1}{(d + \zeta)^2} \left[ \left( \frac{\partial u}{\partial \sigma} \right)^2 + \left( \frac{\partial v}{\partial \sigma} \right)^2 \right] + \frac{1}{(d + \zeta)^2} \frac{\partial}{\partial \sigma} \left[ \left( \nu_{mol} + \frac{\nu_{3D}}{\sigma_k} \right) \frac{\partial k}{\partial \sigma} \right] + P_k + P_{kw} + B_k - \varepsilon$$

$$B_k = \frac{\nu_{3D}}{\rho \sigma_\rho} \frac{g}{H} \frac{\partial \rho}{\partial \sigma}$$

$$\varepsilon = c_D \frac{k\sqrt{k}}{L}$$

$$c_D = c_\mu^{3/4} \approx 0.1925.$$

## **Two Equation Models**

#### Two-equation models solve for

- Transport of k and the dissipation rate of k,  $\varepsilon$ .
- From k and  $\varepsilon$  the mixing length and viscosity are calculated.

For hydrostatic models, the equations can be written as:

$$\frac{\partial k}{\partial t} + \frac{u}{\sqrt{G_{\xi\xi}}} \frac{\partial k}{\partial \xi} + \frac{v}{\sqrt{G_{\eta\eta}}} \frac{\partial k}{\partial \eta} + \frac{\omega}{d + \zeta} \frac{\partial k}{\partial \sigma} = + \\
+ \frac{1}{(d + \zeta)^2} \frac{\partial}{\partial \sigma} \left[ \left( \nu_{mol} + \frac{\nu_{3D}}{\sigma_k} \right) \frac{\partial k}{\partial \sigma} \right] + P_k + P_{kw} + B_k - \varepsilon, \\
\frac{\partial \varepsilon}{\partial t} + \frac{u}{\sqrt{G_{\xi\xi}}} \frac{\partial \varepsilon}{\partial \xi} + \frac{v}{\sqrt{G_{\eta\eta}}} \frac{\partial \varepsilon}{\partial \eta} + \frac{\omega}{d + \zeta} \frac{\partial \varepsilon}{\partial \sigma} = \\
\frac{1}{(d + \zeta)^2} \frac{\partial}{\partial \sigma} \left[ \left( \nu_{mol} + \frac{\nu_{3D}}{\sigma_{\varepsilon}} \right) \frac{\partial \varepsilon}{\partial \sigma} \right] + P_{\varepsilon} + P_{\varepsilon w} + B_{\varepsilon} - c_{2\varepsilon} \frac{\varepsilon^2}{k}.$$

## After all that...

## This is how it is taken care of in your model

#### Note that there are

3 horizontal components

$$\nu_H = \nu_{SGS} + \nu_V + \nu_H^{back}.$$

• And 2 vertical components:

$$\nu_V = \nu_{mol} + \max(\nu_{3D}, \nu_V^{back})$$

model description	$\nu_{SGS}$	$\nu_H^{back}$ (represents)	$ u_{3D}$	$ u_V^{back}$
2D, no HLES	-	2D-turbulence + dispersion coefficient	-	-
2D, with HLES	computed by HLES	3D-turbulence + dispersion coefficient	-	-
3D, no HLES	-	2D-turbulence	computed by vertical turbulence model.	background ver- tical viscosity
3D, with HLES	computed by HLES		computed by vertical turbulence model.	background ver- tical viscosity

## What about the turbulent diffusivity?

That solved the turbulent viscosity and taken care of turbulence in our momentum equation.

What about mixing processes for our transport equations?

Simplest model assumes a gradient-diffusion hypothesis that is

$$\overline{u_j'c'} = -D_t \frac{\partial \overline{c'}}{\partial x_j}$$

where

$$D_t = \frac{V_t}{Sc_t}$$

## What about Diffusion?

Where is Diffusion and what is it affected by?

$$\frac{\partial (d+\zeta) c}{\partial t} + \frac{1}{\sqrt{G_{\xi\xi}} \sqrt{G_{\eta\eta}}} \left\{ \frac{\partial \left[ \sqrt{G_{\eta\eta}} (d+\zeta) uc \right]}{\partial \xi} + \frac{\partial \left[ \sqrt{G_{\xi\xi}} (d+\zeta) vc \right]}{\partial \eta} \right\} + \frac{\partial \omega c}{\partial \sigma} = \frac{d+\zeta}{\sqrt{G_{\xi\xi}} \sqrt{G_{\eta\eta}}} \left\{ \frac{\partial}{\partial \xi} \left[ \frac{D_H}{\sigma_{c0}} \sqrt{\frac{G_{\eta\eta}}{G_{\xi\xi}}} \frac{\partial c}{\partial \xi} \right] + \frac{\partial}{\partial \eta} \left[ \frac{D_H}{\sigma_{c0}} \sqrt{\frac{G_{\xi\xi}}{G_{\eta\eta}}} \frac{\partial c}{\partial \eta} \right] \right\} + \frac{1}{d+\zeta} \frac{\partial}{\partial \sigma} \left[ \frac{\nu_{mol}}{\sigma_{mol}} + \max \left( \frac{\nu_{3D}}{\sigma_c}, D_V^{back} \right) \frac{\partial c}{\partial \sigma} \right] - \lambda_d (d+\zeta) c + S, \quad (9.29)$$

## Diffusion in Delft3D

### Similar to the momentum equation:

- Horizontal  $D_H = D_{SGS} + D_{3D} + D_H^{back}$
- Vertical  $D_V = rac{
  u_{mol}}{\sigma_{mol}} + \max(D_V^{back}, D_{3D})$

But what is  $D_{3D}$ ?

$$D_{3D} = \frac{\nu_{3D}}{\sigma_c}$$

- What is  $\sigma_c$ ?  $\sigma_c = \sigma_{c_0} F_{\sigma}(Ri)$ .
- What is actually put in the code?  $D_{3D} = \max \left( D_{3D}, 0.2 L_{oz}^2 \sqrt{-\frac{g}{\rho} \frac{\partial \rho}{\partial z}} \right)$

# The issue of stratification, the Prandtl-Schmidt Number, $\sigma_c$

This is a dimensionless number that relates viscous diffusion to

- Molecular diffusion v/D (Schmidt); or
- Turbulent diffusion  $v_t/D_t$ ; or
- Thermal diffusion:  $v/\alpha$  (Prandtl)

In the Delft3D formulation:

$$\sigma_c = \sigma_{c_0} F_{\sigma}(Ri).$$

- $\sigma_{c0}$  is a constant (0.7; 1.0[k]; 1.3[ $\epsilon$ ])
- $^{\rm o}$   ${\it F}_{\sigma}$  is a damping function for the ALM when the flow is strongly stratified

## In summary:

model description	$D_{2D}$	$D_H^{back}$	$D_{3D}$	$D_V^{back}$
2D, no HLES	0	2D-turbulence +	-	-
		dispersion coefficient		
2D, with HLES	computed	3D-turbulence +	-	-
	by HLES	dispersion coefficient		
3D, no HLES	0	2D-turbulence	maximum of value	background
			of turbulence	eddy diffusivity
			model and the	
			Ozmidov length	
			scale. *2	
3D, with HLES	computed		maximum of value	background
	by HLES	\	of turbulence	eddy
			model and the	diffusivity
			Ozmidov length	
			scale. *2	

#### Diffusion is accounted for as:

$$D_{3D}=\max\left(D_{3D},0.2L_{oz}^2\sqrt{-rac{g}{
ho}rac{\partial
ho}{\partial z}}
ight).$$
 where  $D_{3D}=rac{
u_{3D}}{\sigma_c}$ 

• HOWEVER by default  $L_{OZ} = 0$ ; why?