

## Application of Computers in Process Control

Purdue University, Calumet

# [LABORATORY 8]

**Tuning of a PID controller using Ziegler-Nichols Method**

## Tuning of a PID controller using Ziegler-Nichols Method

### LAB 8

#### Objective:

- 1) To demonstrate the use of PID controller using LabVIEW.
- 2) Finding the values of  $K_P$  and  $T_I$  and  $T_D$  using Ziegler Nichols Method for Tuning the PID controller

**Software required:** LabVIEW

#### Background:

PID controllers are probably the most commonly used controller structures in industry. They do, however, present some challenges to control and instrumentation engineers in the aspect of tuning of the gains required for stability and good transient performance. There are several prescriptive rules used in PID tuning. An example is that proposed by Ziegler and Nichols in the 1940's and described in this note.

The PID controller encapsulates three of the most important controller structures in a single package. The parallel form of a PID controller has transfer function:

$$C(s) = K_p + \frac{K_I}{s} + K_d s$$

$$= K_p \left( 1 + \frac{1}{T_I s} + T_d s \right)$$

where:

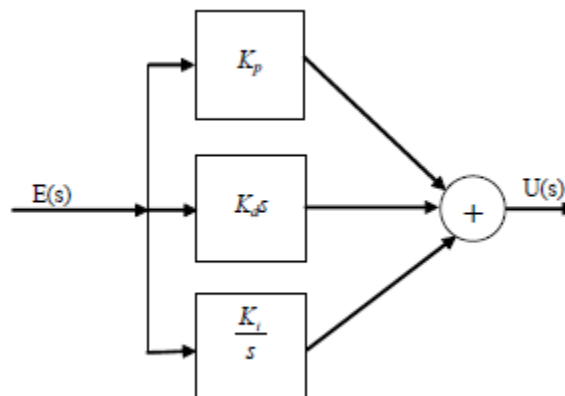
$K_p$  := Proportional Gain

$K_I$  := Integral Gain

$K_d$  := Derivative gain

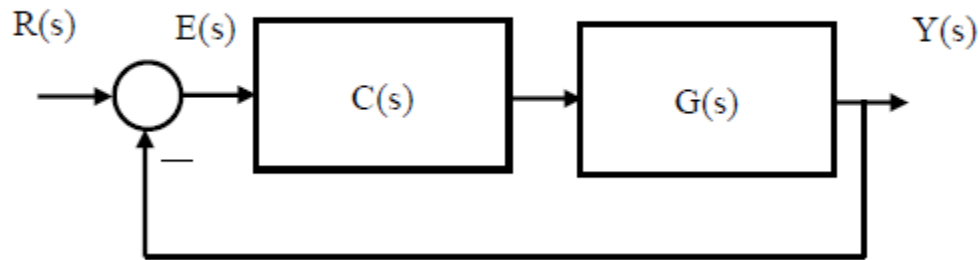
$T_I$  := Reset Time =  $K_p/K_i$

$T_d$  := Rate time or derivative time



**Parallel Form of the PID Compensator**

The proportional term in the controller generally helps in establishing system stability and improving the transient response while the derivative term is often used when it is necessary to improve the closed loop response speed even further. Conceptually the effect of the derivative term is to feed information on the rate of change of the measured variable into the controller action. The most important term in the controller is the integrator term that introduces a pole at  $s = 0$  in the forward loop of the process. This makes the compensated open loop system (i.e. original system plus PID controller) a type 1 system at least; our knowledge of steady state errors tells us that such systems are required for perfect steady state setpoint tracking.



### *Ziegler-Nichols Tuning*

In 1942 Ziegler and Nichols, both employees of Taylor Instruments, described simple mathematical procedures for tuning PID controllers. These procedures are now accepted as standard in control systems practice. Ziegler-Nichols formulae for specifying the controllers are based on plant step responses.

#### **Steps to determine PID controller parameters:**

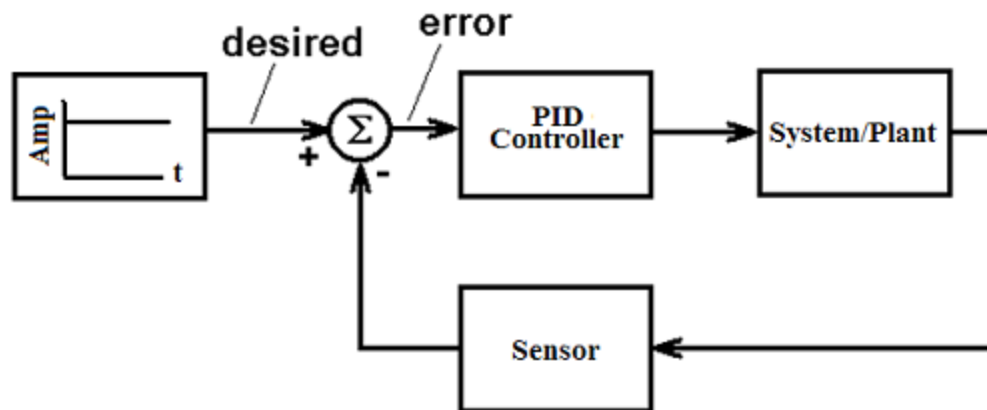
1. Reduce the integrator and derivative gains to 0.
2. Increase  $K_p$  from 0 to some critical value  $K_p = K_c$  at which sustained oscillations occur
3. Note the value  $K_c$  and the corresponding period of sustained oscillation,  $T_c$
4. The controller gains are now specified as follows:

PID Type	$K_P$	$T_I$	$T_d$
<b>P</b>	$0.5K_c$	Inf	0
<b>PI</b>	$0.45K_c$	$T_c/1.2$	0
<b>PID</b>	$0.6K_c$	$T_c/2$	$T_c/8$

Consider a process with transfer function:

$$G(s) = \frac{1}{(s+1)(s+3)(s+5)} = \frac{1}{s^3 + 9s^2 + 23s + 15}$$

Let us consider that the overall system has a unity feedback.



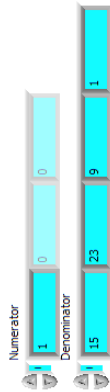
Changes in system's closed loop response because of the changes in PID parameters with respect to a step input can be best described using the following chart:

CL RESPONSE	RISE TIME	OVERSHOOT	SETTLING TIME	S-S ERROR
<b>Kp</b>	Decrease	Increase	Small Change	Decrease
<b>Ki</b>	Decrease	Increase	Increase	Eliminate
<b>Kd</b>	Small Change	Decrease	Decrease	Small Change

In LabVIEW there are several PID controller modules. We shall use the Academic PID controller. The difference between different PID controllers is shown below:

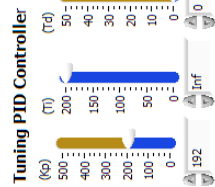
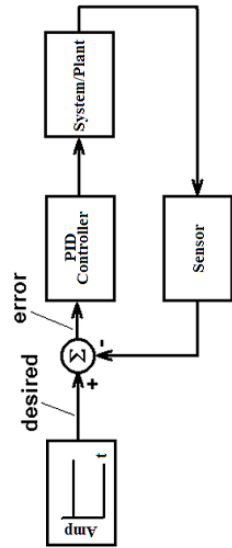
PID Controller Form	Equation
PID Academic	$\frac{U(s)}{E(s)} = K_c \left( 1 + \frac{1}{T_i s} + \frac{T_d s}{\alpha T_d s + 1} \right)$
PID Parallel	$\frac{U(s)}{E(s)} = K_c + \frac{K_i}{s} + \frac{K_d s}{\alpha K_d s + 1}$
PID Parallel Discrete	$D(z) = \frac{(2K_p T + K_i T^2 + 2K_d)z^2 + (K_i T^2 - 2K_p T - 4K_d)z + 2K_d}{2Tz(z-1)}$
PID Series	$\frac{U(s)}{E(s)} = K_c \left( 1 + \frac{1}{T_i s} \right) \left( \frac{T_d s + 1}{\alpha T_d s + 1} \right)$

**Program:**



## Plant Transfer Function

$$\frac{1}{s^3 + 9s^2 + 23s + 15}$$



## PID Controller transfer function

192

marginally stable

## Sustained Oscillation at

[192 inf 0]

So,

So,  
 $K_C = 192$  and  $T_C = 1.31s$

For Nominal response accoring to Ziegler Nichols:

For quarter-amplitude response:

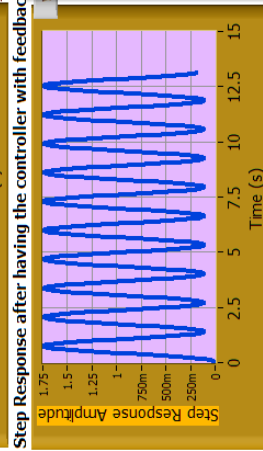
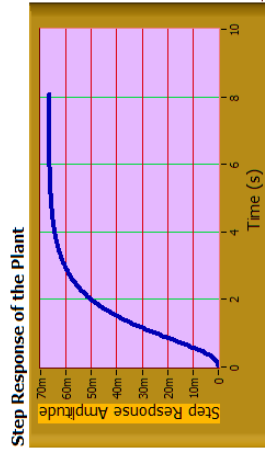
$$K_p = 0.6 K_c$$
$$T_I = T_C/2$$
$$T_D = T_C/8$$

Adjust the gain until the dynamic response pattern to a step change in setpoint obeys the quarter amplitude criterion.

**Note** In some applications, you specify the gain in the PID Academic transfer function in terms of a proportional band (PB).

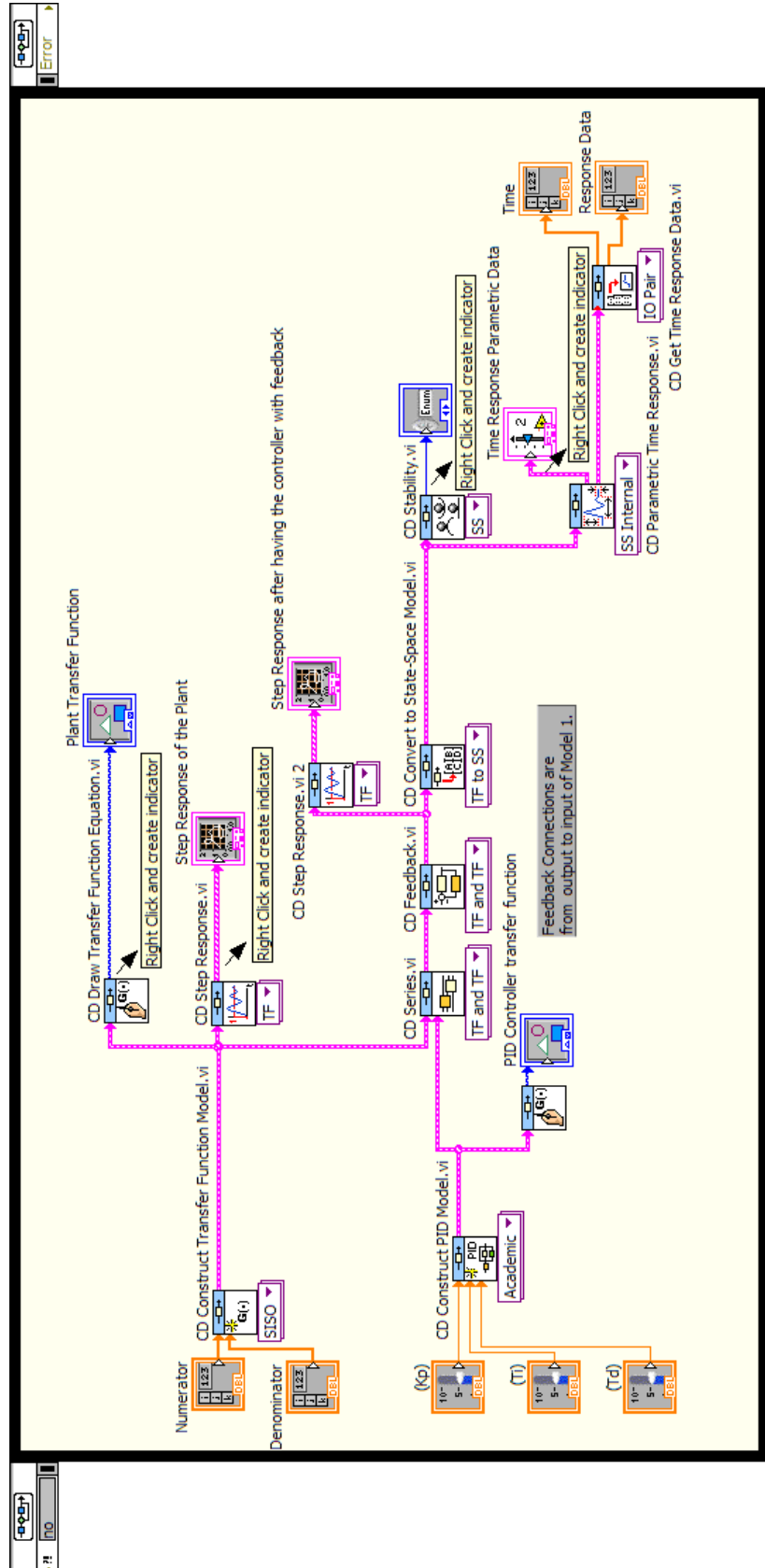
$$PB = \frac{1}{K_c} \times 100\%$$

A proportional band, defined by the previous equation, is the percentage of the input range of the controller that causes a change equal to the maximum range of the output.



CL RESPONSE	RISE TIME	OVERSHOOT	SETTLING TIME	S-S ERROR
Kp	Decrease	Increase	Small Change	Decrease
Ki	Decrease	Increase	Increase	Eliminate
Kd	Small Change	Decrease	Decrease	Small Change

Figure 1 displays two rows of control panels for a system. The top row is labeled "Time" and the bottom row is labeled "Response Data". Each panel contains three digital displays with up/down arrows and a "0" value. The top row panels are labeled "Rise Time (s)", "Peak Time (s)", and "Setting Time (s)". The bottom row panels are labeled "Overshoot (%)", "Steady-State Gain", and "Peak Value". The "Steady-State Gain" panel has a green background. The "Response Data" row has a yellow background.



**Conclusion:**

- 1) Follow the steps describe above to tune the PID controller.
- 2) Find the values of  $K_P$ ,  $T_I$ ,  $T_D$ .
- 3) What is the effect of changing the PID controller parameters in the step response of the overall system?
- 4) Note the Time response parametric data after the controller is tuned.

**Ref:**

*The Design of PID Controllers using Ziegler Nichols Tuning* --Brian R Copeland; March, 2008