

## Turbojet Engine Cycle Analysis

Recall, from the discussion of isentropic flow with work that the compressor power is given by  $\dot{\phi}_{sc} = \dot{m}c_p(T_{01} - T_{02})$ , where subscripts 1 and 2 represent the compressor entry and exit stations. For an ideal compressor the temperature and pressure ratios are related by the isentropic expression:

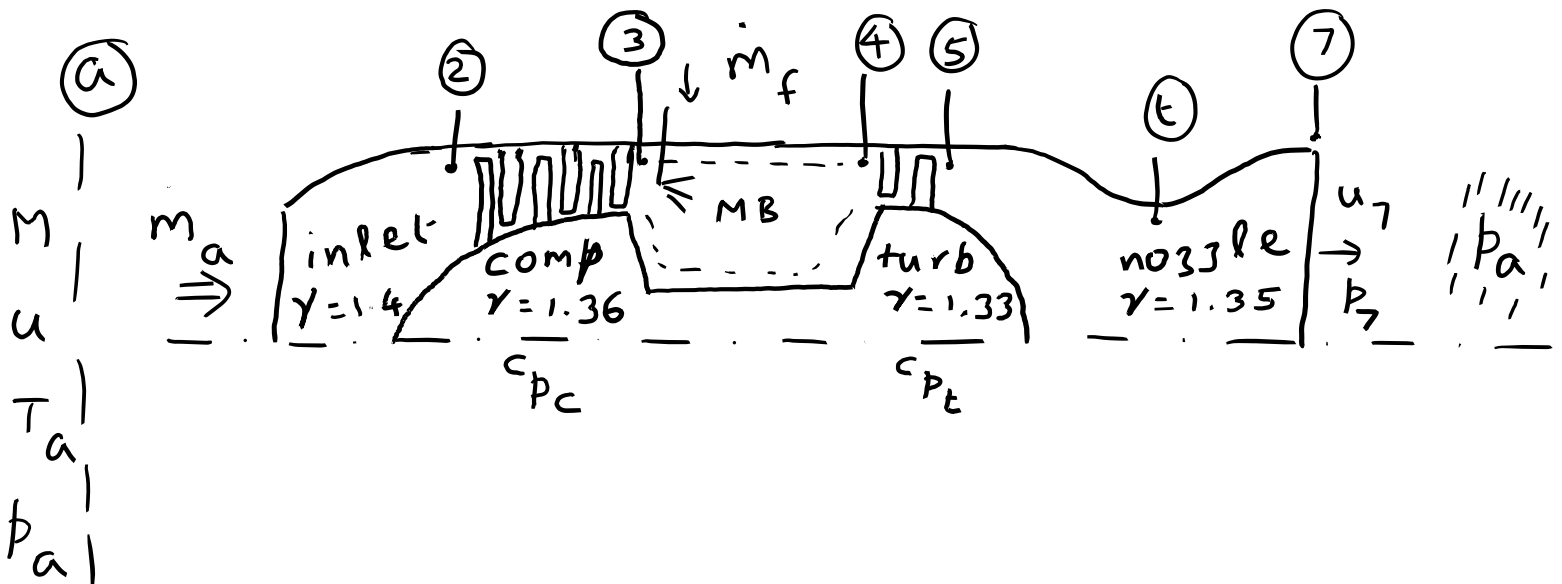
$$\left(\frac{T_{02}}{T_{01}}\right) = \left(\frac{p_{02}}{p_{01}}\right)^{\frac{\gamma-1}{\gamma}} = \pi_c^{\frac{\gamma-1}{\gamma}}, \text{ where } \pi_c \text{ is the compression ratio.}$$

The turbine power is given by  $\dot{\phi}_{st} = \dot{m}c_p(T_{01} - T_{02})$ , and for the ideal turbine, the temperature and pressure ratios are related by the isentropic expression:

$$\left(\frac{p_{02}}{p_{01}}\right) = \left(\frac{T_{02}}{T_{01}}\right)^{\frac{\gamma}{\gamma-1}}.$$

Consider the burner in the gas turbine engine. Fuel, with heating value  $Q_R$ , is added at the rate of  $\dot{m}_f$ . The rate of energy release ( $\dot{Q}$ ) due to combustion of this fuel with air is  $\dot{m}_f Q_R \eta_b$ , where  $\eta_b$  is the burner (or combustion) efficiency. The mass flow rate through the burner is  $(\dot{m}_a + \dot{m}_f)$ , and the heat added per unit mass is  $q = \frac{\dot{Q}}{\dot{m}_a + \dot{m}_f} = \frac{\dot{m}_f Q_R \eta_b}{\dot{m}_a + \dot{m}_f} = \frac{f Q_R \eta_b}{1+f}$ . As mentioned earlier,  $f$  is typically small (around 0.02 or 0.03, so it can be ignored in comparison to 1; that is  $(1+f) \simeq 1$ ).

Therefore,  $q = f Q_R \eta_b$ .



Example:

An ideal turbojet has the following characteristics:  $M = 2$ ,  $T_a = 217$  K,  $p_a = 19.4$  kPa,  $\pi_c = 22$ ,  $T_{04} = 1600$  K, fully expanded exhaust jet,  $Q_R = 43500$  kJ/kg.

- (i) Perform cycle analysis and find the engine exit stagnation temperature and stagnation pressure. The values of  $\gamma$  across the various components are noted in the Figure.
- (ii) Find the specific thrust and TSFC of the engine, and the exhaust nozzle area ratio.

Solution:

(i)

Station (a):  $M = 2$ ,  $u = M\sqrt{\gamma RT} = 590.6$  m/s

$$T_{0a} = T_a \left( 1 + \frac{\gamma-1}{2} M^2 \right) = 390.6 \text{ K}$$

$$p_{0a} = p_a \left( 1 + \frac{\gamma-1}{2} M^2 \right)^{\frac{\gamma}{\gamma-1}} = 151.8 \text{ kPa}$$

(a)  $\rightarrow$  (2), inlet diffuser:

$$T_{02} = T_{0a} = 390.6 \text{ K}$$

$$p_{02} = p_{0a} = 151.8 \text{ kPa}$$

(2)  $\rightarrow$  (3), compressor:

$$\pi_c = 22, \text{ or } \left( \frac{p_{03}}{p_{02}} \right) = 22 \rightarrow p_{03} = 22p_{02} = 3340 \text{ kPa}$$

$$\left( \frac{T_{03}}{T_{02}} \right) = \left( \frac{p_{03}}{p_{02}} \right)^{\frac{\gamma-1}{\gamma}} = \pi_c^{\frac{\gamma-1}{\gamma}} = 22^{\frac{1.36-1}{1.36}} = 2.2665 \rightarrow T_{03} = 2.2286T_{02} = 885.3 \text{ K}$$

$\phi_{sc} = \dot{m}_a c_{pc} (T_{02} - T_{03}) = -5.363(10^5) \dot{m}_a$  {negative value indicates work done on air passing through the compressor}

$$\{ c_{pc} = \frac{\gamma R}{\gamma-1} = \frac{1.36}{1.36-1} 287 = 1084 \text{ J/kg-K} \}$$

(3) → (4), main burner:

$$\text{I law: } (h_{04} - h_{03}) = q - w = f Q_R \eta_b - 0 \rightarrow f = \frac{(h_{04} - h_{03})}{Q_R \eta_b} = \frac{(c_{pt} T_{04} - c_{pc} T_{03})}{Q_R \eta_b}, \text{ or}$$

$$f = \frac{(1157)(1600) - (1084)(885.3)}{43500(1000)(1)} = 0.0205$$

$$\{c_{pt} = \frac{\gamma R}{\gamma - 1} = \frac{1.33}{1.33 - 1} 287 = 1157 \text{ J/kg-K}\}$$

$p_{04} = p_{03} = 3340 \text{ kPa}$  {From Rayleigh line analysis (AE 308), recall that heat addition at zero Mach number entails no loss in stagnation pressure}

(4) → (5), turbine:

Turbine generates power to drive compressor. This observation constitutes the turbine-compressor power compatibility condition.

$$|\dot{\phi}_{st}| = |\dot{\phi}_{sc}| = 5.363(10^5) \dot{m}_a, \text{ or}$$

$$\dot{m}_a(1 + f)c_{pt}(T_{04} - T_{05}) = 5.363(10^5) \dot{m}_a \rightarrow T_{05} = 1137 \text{ K}$$

$$\left(\frac{p_{05}}{p_{04}}\right) = \left(\frac{T_{05}}{T_{04}}\right)^{\frac{\gamma}{\gamma-1}} = 0.2524 \rightarrow p_{05} = (0.2524)(3340) = 843 \text{ kPa}$$

(5) → (7), nozzle:

$$T_{07} = T_{05} = 1137 \text{ K}$$

$$p_{07} = p_{05} = 843 \text{ kPa}$$

↑ \_\_\_\_\_ end cycle analysis \_\_\_\_\_ ↑

(ii)

Fully expanded exhaust jet:  $p_7 = p_a = 19.4 \text{ kPa}$

$$\left(\frac{p_{07}}{p_7}\right) = \frac{843}{19.4} = 4.3454$$

$$\left(1 + \frac{\gamma-1}{2} M_7^2\right)^{\frac{\gamma}{\gamma-1}} = 4.3454 \rightarrow M_7 = 3.079 \{ \gamma = 1.35 \}$$

$$T_7 = \frac{T_{07}}{\left(1 + \frac{\gamma-1}{2} M_7^2\right)} = 427.6 \text{ K}$$

$$u_7 = M_7 \sqrt{\gamma R T_7} = 1253 \text{ m/s}$$

$$\tau = \dot{m}_a(1+f)u_7 - \dot{m}_a u + (p_7 - p_a)A_7 \simeq \dot{m}_a(1253 - 590.6) = 662.4\dot{m}_a$$

$$\text{Specific thrust } \frac{\tau}{\dot{m}_a} = 662.4 \text{ N/(kg/s)}$$

$$TSFC = \frac{\dot{m}_f}{\tau} = \frac{f\dot{m}_a}{\tau} = \frac{f}{\left(\frac{\tau}{\dot{m}_a}\right)} = (0.0205/662.4) = 3.095(10^{-5}) \text{ (kg/s)/N}$$

$$(TSFC)_{wb} = (TSFC)g_e = 3.095(10^{-5})(9.81) = 13.036(10^{-4}) \text{ s}^{-1} \xrightarrow{x3600} 1.093 \text{ h}^{-1}$$

$$\text{Nozzle area ratio is } (A_7/A_t) = (A_7/A_7^*) = \frac{1}{M_7} \left[ \frac{2}{\gamma+1} \left( 1 + \frac{\gamma-1}{2} M_7^2 \right) \right]^{\frac{\gamma+1}{2(\gamma-1)}} = 5.039$$

$$\{\gamma = 1.35\}$$

If the propellant (air) mass flow rate is 100 kg/s,

$$\tau = \left(\frac{\tau}{\dot{m}_a}\right) \dot{m}_a = (662.4)(100) = 66240 \text{ N} \simeq 14892 \text{ lb}_f$$

$$\dot{m}_f = f\dot{m}_a = (0.0205)(100) = 2.05 \text{ kg/s} \xrightarrow{x3600} 7380 \text{ kg/h} \simeq 16236 \text{ lb}_m/\text{h}$$

$$A_7 = \frac{\dot{m}_7}{\rho_7 u_7} = \frac{\dot{m}_a(1+f)}{\rho_7 u_7} \simeq \frac{\dot{m}_a}{\rho_7 u_7}$$

$$\dot{m}_a = 100 \text{ kg/s}, \rho_7 = \frac{p_7}{RT_7} = \frac{(19.4)(1000)}{(287)(427.6)} = 0.1581 \text{ kg/m}^3, u_7 = 1253 \text{ m/s}$$

$$A_7 = \frac{100}{(0.1581)(1253)} = 0.5048 \text{ m}^2 \{D_7 = \sqrt{\frac{4A_7}{\pi}} = 0.80 \text{ m} \simeq 31.5 \text{ in.}\}$$

$$A_t = \frac{A_7}{\left(\frac{A_7}{A_t}\right)} = \frac{0.5048}{5.039} = 0.1002 \text{ m}^2 \{D_t = \sqrt{\frac{4A_t}{\pi}} = 0.36 \text{ m} \simeq 14.2 \text{ in.}\}$$