Introduction Analog and Digital formulations Implementation Examples PID Tuning

## Chapter 13 - PID Controllers

August 6, 2015

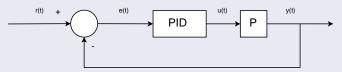
### Outline

- Introduction
- 2 Analog and Digital formulations
- Implementation Examples
- 4 PID Tuning

### What is a PID controller?

#### Definition

A Proportional Integral Deriviative controller is a control loop feedback mechanism (controller) widely used in process industry.



Continuous-time text book equation:

$$u(t) = \underbrace{K_p e(t)}_{\text{Proportional Action}} + \underbrace{K_i \int_0^t e(\tau) d\tau}_{\text{Integral Action}} + \underbrace{K_d \frac{de(t)}{dt}}_{\text{Derivative Action}}$$

**Note:** More than 90% of all closed loop controllers are PID.

### What is a PID controller?

- Proportional action  $u_p(t) = K_p e(t)$ : Action depends on the instaneous value of the control error.
  - + Reduces rise time
  - Reduces but **does not eliminate steady-state error**: Only when  $K \to \infty$ , error  $\to 0$  (unless plant has pole(s) at s = 0)
- Integral action  $u_i(t) = K_i \int_0^t e(\tau) d\tau$ : Gives a controller output that is proportional to the accumulated error. Reacts on constant errors
  - + Can eliminate steady state error in some cases
  - Makes transient response slower
- **D**erivative action  $u_d(t) = K_d \frac{de(t)}{dt}$ : Acts on the rate of change of the control error.
  - + Damping effect: reduces overshoot, improves transient response
  - Sensitive for noise, amplifies it if present

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## **Proportional Control**

The continuous-time and discrete implementation are identical Continuous:

$$u_p(t) = K_p e(t)$$
  $\rightarrow$   $\frac{U_p(s)}{E(s)} = K_p$ 

Discrete:

$$u_p[k] = K_p e[k] \rightarrow \frac{U_p(z)}{E(z)} = K_p$$

where e(t) or e[k] is the error signal.

### Derrivative Control

Continuous:

$$u_d(t) = K_d \frac{de(t)}{dt} \quad o \quad \frac{U_d(s)}{E(s)} = K_d s$$

Discrete (using backward Euler):

$$u_d[k] = K_d \frac{e[k] - e[k-1]}{T_s} \rightarrow \frac{U_d(z)}{E(z)} = K_d \frac{z-1}{T_s z}$$

with  $T_s$  the sampling time.

## Integral Control

The continuous equation is:

$$u_i(t) = K_i \int_0^t e(\tau) d\tau \quad \rightarrow \quad \frac{U_i(s)}{E(s)} = \frac{K_i}{s}$$

Differentiating this gives :

$$\dot{u}_i = K_i e(t)$$

Then applying backward Euler:

$$u_i[k] = u[k-1] + K_i Te[k] \rightarrow \frac{U_i(z)}{E(z)} = K_i \frac{zT_s}{z-1}$$

with  $T_s$  the sampling time.



# Digital formulation (conventional version)

#### Digital PID controller (conventional version)

$$u[k] = K_p e[k] + \frac{K_d}{T_s} (e[k] - e[k-1]) + u_i[k]$$
  
with  $u_i[k] = u_i[k-1] + K_i T_s e[k]$ 

In z-domain:

$$\frac{U(z)}{E(z)} = K_p + \frac{K_d}{T_s} \frac{z-1}{z} + K_i T_s \frac{z}{z-1}$$

where  $\frac{K_d}{T_s}$  and  $K_i T_s$  are the new derivative and gains.

#### Digital PI controller

$$\frac{U(z)}{E(z)} = K_p + K_i T \frac{z}{z - 1}$$

#### Digital PD controller

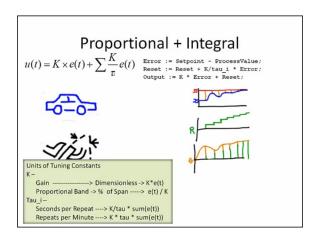
$$\frac{U(z)}{E(z)} = K_p + \frac{K_d}{T} \frac{z - 1}{z}$$

## Alternative Digital PID controller

We can also discretize using the bilinear transformation:

$$\begin{aligned} \frac{U(z)}{E(z)} &= K_p + \frac{K_i}{s} + K_d s \bigg|_{s = \frac{2}{T} \left(\frac{z-1}{z+1}\right)} \\ &= K_p + \frac{K_i T(z+1)}{2(z-1)} + \frac{2K_d(z-1)}{T(z+1)} \\ &= \frac{\alpha_2 z^2 + \alpha_1 z + \alpha_0}{(z-1)(z+1)} \end{aligned}$$

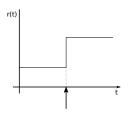
where  $\alpha_2, \alpha_1, \alpha_0$  are design parameters.



https://www.youtube.com/watch?v=JEpWlTl95Tw

# Alterantive Derivative Action(Continous time)

Imagine a jumping set point or rapidly changing signal. This results in a theoretically infinite, practically very large response of the derivative term.



 $\Rightarrow$  Add a low-pass filter to the derivative term:

$$\frac{U_d(s)}{E(s)} = \frac{K_d s}{1 + s\tau}$$

With  $s=j\omega$ , breakpoint at  $\omega=1/\tau$ . This prevents amplification of high frequencies.



# Alterantive Derivative Action(Continous time)

$$\frac{U_d(s)}{E(s)} = \frac{K_d s}{1 + s\tau}$$

Further e(t) is replaced by  $c \cdot r(t) - y(t)$  with c the set point weighting, which is often set to zero to further reduce immediate influence of a sudden set point jump.

In time domain:

$$u_d(t) = -\tau \frac{du_d}{dt} + K_d(c \cdot r(t) - y(t))$$

This can also be discretized, but the bilinear method then introduces *ringing*, i.e. large oscillations in transient response.

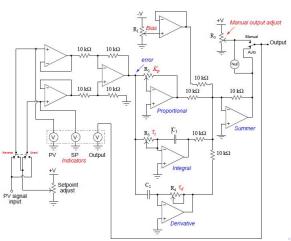


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## **Analog Implementation**

The key building block is the operational amplifier (op-amp).



- PV Process Variable y(t)
- SP Set Point r(t)
- Output Control action u(t)

# Analog Implementation

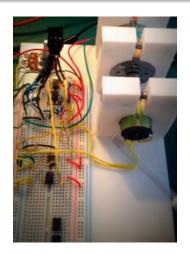






FOXBORO 62H-4E-OH M/62H

# Analog PI Motor Speed Control



https://youtu.be/6W3PLiVIcmE



## Digital Implementation

The difference equations are typically implemented in a micro controller or FPGA (field-programmable gate array):

$$u[k] = K_p e[k] + \frac{K_d}{T} (e[k] - e[k-1]) + u_i[k]$$
  
with  $u_i[k] = u_i[k-1] + K_i Te[k]$ 

Steps to be implemented:

```
previous_error = 0
integral = 0
Start:
    error = setpoint - measured_value
    proportional = K_P * error
    integral = integral + K_i*sampling_time*error
    derivative = K_d*(error-previous_error)/sampling_time
    output = proportional + integral + derivative
    previous_error = error
    wait(sampling_time)
    goto Start
```

## Digital Implementation Example

PLC with a digital PID module:



Digital PID's:





## **PLC**

Programmable Logic Controller is a digital computer used for automation in the process industry.





## What is a PLC? Basics of PLCs



https://youtu.be/iWgHqqunsyE

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# Manual Tuning

The controller can be tuned while connected to the plant. Following routine can be used:

- Set  $K_i$  and  $K_d$  equal to 0
- ② Increase  $K_p$  until you observe that the step response is fast enough and the steady-state error in small
- **3** Start adding some integral action in order to get rid of the steady state error. Keep in mind that too much  $K_i$  can cause instability!
- 4 Add some derivative action in order to quickly react to disturbance and/or dampen the response

# Manual Tuning

PID gains	Rise Time	Overshoot	Setlling time	Steady-State error
$K_p \uparrow$	Decrease	Increase	Small Change	Decrease
$K_i \uparrow$	Decrease	Increase	Increase	Eliminate
$K_d \uparrow$	Small change	Decrease	Decrease	No change

#### Important note

Changing one parameter can influence the effect of the other two. Use this table only as an indication.

## Heuristic Methods

Manual tuning is used on a plant from which we know the mathematical model.

Sometimes the **mathematical model** of the plant is **not known**. In these cases we will uses the **heuristic methods**:

- Ziegler-Nichols tuning rule based on step response (First method)
- Ziegler-Nichols tuning rule based on critical gain and critical period (Second method)

# Heuristic Methods: Ziegler-Nichols tuning rule

This method relies on empirically determining two parameters of the system:

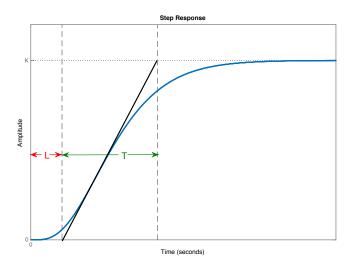
- Set the integral and derivative gains to 0
- ② Increase the proportional gain  $K_p$  until the output of the control loop starts oscillating with at constant amplitude. The value of  $K_p$  at this point is referred to as ultimate gain  $K_u \triangleq K_p$
- **3** Measure the period of the oscillations  $T_u$  at the output
- Adjust the controller parameters according the table on the next slide.

### First Method

In this method, the response of the plant to a unit-step input is obtained experimentally (because we do not know the mathematical model). If the plant involves neither integrators nor dominant complex-conjugate poles, then such a unit-step response curve may look S-shaped as shown in the figure on the next slide. This S-shaped curve may be characterized by two constants: delay time L and time constant T. The delay time and the time constant are determined by drawing a tangent line at the inflection point of the S-shaped curve and determining the intersections of the tangent line with the time axis and line Amplitude = K.

If the response to a step input does not exhibit an S-shaped curve, the first method does not apply.

## First Method



### First Method

Control Type	$K_P$	$K_{I}$	$K_D$
P	$\frac{T}{L}$	0	0
PI	$0.9\frac{T}{L}$	$K_p \frac{0.3}{L}$	0
PID	$1.2\frac{\bar{T}}{L}$	$\frac{K_p}{2L}$	0.5 <i>LK<sub>P</sub></i>

The PID controller tuned by the first method of Ziegler-Nichols rules gives:

$$\frac{U(s)}{E(s)} = K_P + \frac{K_I}{s} + K_D s$$

$$= 1.2 \frac{T}{L} \left( 1 + \frac{1}{2Ls} + 0.5 Ls \right)$$

$$= 0.6 T \frac{\left( s + \frac{1}{L} \right)^2}{s}$$

with a pole at the origin and a double zero at  $s=-\frac{1}{I}$ .

### Second Method

With  $K_u$  and  $T_u$  defined as before, a starting point for the parameters can be determined:

Control Type	$K_p$	$K_i$	$K_d$
Р	$0.5K_{u}$	-	-
PI	$0.45K_{u}$	$1.2K_p/T_u$	-
PD	$0.8K_{u}$	-	$K_pT_u/8$
PID	$0.6K_{u}$	$2K_p/T_u$	$K_pT_u/8$
Pessen Integral Rule	$0.7K_{u}$	$2.5K_p/T_u$	$3K_{p}T_{u}/20$
Some overshoot	$0.33K_{u}$	$2K_p/T_u$	$K_pT_u/3$
No overshoot	$0.2K_{u}$	$2K_p/T_u$	$K_pT_u/3$

If the output does not exhibit sustained oscillations for whatever value  $K_P$  may take, then the second method does not apply.

## Ziegler-Nichols tuning rule(example)

#### Example

Consider a plant with a given model:

$$P(s) = \frac{1}{(s+1)^3}$$

• We compute the critical gain  $K_c$ . This is the value of  $K_p$  for which  $\angle(K_pP(s)) = -180^\circ$ . On the Nyquist plot this the value of  $K_p$  for which  $K_pP(s)$  passes through (-1,0).

$$K_c P(j\omega_c) = -1$$

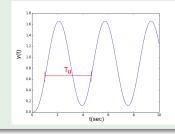
$$\Leftrightarrow K_c = -(j\omega_c + 1)^3$$

$$= (3\omega_c^2 - 1) + j(\omega_c^3 - 3\omega_c)$$

$$\omega_c^3 - 3\omega_c = 0 \Rightarrow \omega_c = \sqrt{3}$$

$$K_c = 8, T_u = \frac{2\pi}{\omega} = 3.628$$

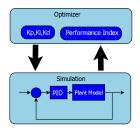
$$K_p = 4.8, K_l = 0.551K_p, K_d = 0.45K_p$$



## Numerical Optimization Methods

The tuning of a PID controller is posed as a constrained optimization problem.

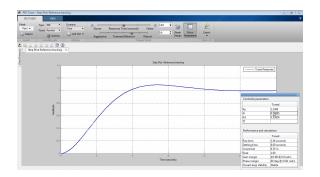
- For a given set of parameters K<sub>p</sub>, K<sub>i</sub> and K<sub>d</sub> run a simulation of the closed-loop system, and compute some performance parameters (e.g. setting time, rise time, etc.) and a performance index.
- Optimize the performance index over the three PID gains.



## Some Software Tools

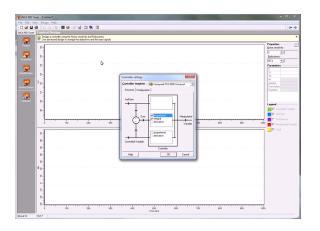
Software Tool	Brief Description
pidtool / pidTuner	It is a Matlab tool to interactively design a SISO
	PID controller in the feed-forward path of single-
	loop, unity-feedback control configuration
Pidpy	It is a modular PID control library for python
	that supports PID auto tuning. https://pypi.
	python.org/pypi/pypid/
INCA PID Tuner	It is a commercial tuning tool developed by
	IPCOS. It has a vast library of PID structures
	for DCS and PLC Systems including Siemens,
	ABB, Honeywell, Emerson, etc. http://www.
	ipcos.com/advancedprocesscontrol/
	advanced-process-control/
	pid-tuning-software/inca-pid-tuning/

# pidtool /pidTuner - Demo



https://www.youtube.com/watch?v=2tKeOcaUv1I

## INCA PID Tuner Demo



https://www.youtube.com/watch?v=XH2bkq1URSg