# **IIX Plasticity**

### 19 Limits & Failure Criteria

Shear flow strength:  $k=Y/\sqrt{3}$ 

#### Invariants

1. Mean normal pressure.

$$\bar{p} = -\frac{1}{3} \operatorname{tr} \sigma = -\frac{1}{3} (\sigma_{kk})$$

$$ar{p}=-rac{1}{3}\left(\sigma_{11}+\sigma_{22}+\sigma_{33}
ight)$$

2. Equivalent shear/tensile stress.

$$ar{ au} = rac{1}{\sqrt{2}} \left| \sigma' 
ight| = \sqrt{rac{1}{2} \operatorname{tr} \left( \sigma'^2 
ight)} = \sqrt{rac{1}{2} \sigma'_{ij} \sigma'_{ij}}$$

$$ar{ au} = \left[rac{1}{6}\left(\left(\sigma_{11} - \sigma_{22}
ight)^2 + \left(\sigma_{22} - \sigma_{33}
ight)^2 + \left(\sigma_{33} - \sigma_{11}
ight)^2
ight) + \left(\sigma_{12}^2 + \sigma_{23}^2 + \sigma_{31}^2
ight)
ight]^{1/2}$$

$$ar{\sigma} = \sqrt{rac{3}{2}} \left| \sigma' 
ight| = \sqrt{rac{3}{2} \operatorname{tr} \left( \sigma'^2 
ight)} = \sqrt{rac{3}{2} \sigma'_{ij} \sigma'_{ij}}$$

$$\bar{\sigma} = \left[\frac{1}{2}\left(\left(\sigma_{11} - \sigma_{22}\right)^2 + \left(\sigma_{22} - \sigma_{33}\right)^2 + \left(\sigma_{33} - \sigma_{11}\right)^2\right) + 3\left(\sigma_{12}^2 + \sigma_{23}^2 + \sigma_{31}^2\right)\right]^{1/2}$$

$$\bar{\sigma}=\sqrt{3}\bar{\tau}$$

3. Third stress invariant.

$$ar{r} = \left(rac{9}{2}\operatorname{tr}\left(\sigma'^3
ight)
ight)^{rac{1}{3}} = \left(rac{9}{2}\sigma'_{ik}\sigma'_{kj}\sigma'_{jk}
ight)^{rac{1}{3}}$$

Failure Criteria

1. Von Mises

$$f=ar{\sigma}-Y\leq 0$$

2. Tresca

$$f = (\sigma_1 - \sigma_3) \leq Y$$

$$f = au_{ ext{max}} - au_{y,\, ext{Tresca}} \, \leq 0$$

$$au_{ ext{max}} \stackrel{ ext{def}}{=} rac{1}{2} \left( \sigma_1 - \sigma_3 
ight) \geq 0 \ au_{y,\, ext{Tresca}} \stackrel{ ext{def}}{=} rac{1}{2} Y$$

3. Mohr-Coulomb

$$f=rac{1}{2}\left(\sigma_{1}-\sigma_{3}
ight)+rac{1}{2}\left(\sigma_{1}+\sigma_{3}
ight)\sin\phi-c\cos\phi\leq0$$

$$\mu = \tan \phi$$

4. Drucker-Prager

$$f(ar{ au},ar{p},S)=ar{ au}-(S+lphaar{p})\leq 0$$

# 20 One-Dimensional Plasticity

## 20.1 Elastic-Plastic Response

### Isotropic hardening

#### Kinematic hardening

$$egin{aligned} \sigma_b &= rac{1}{2} \left( \sigma_f + \sigma_r 
ight) \ \left| \sigma - \sigma_b 
ight| \leq Y_0 \end{aligned}$$

20.2 Isotropic rate-independent theory

1. Kinematic decomposition

$$\epsilon = \epsilon^e + \epsilon^p$$

2. Constitutive relation

$$\sigma=E\epsilon^{e}=E\left(\epsilon-\epsilon^{p}
ight)$$

3. Yield condition

$$f=\left|\sigma
ight|-Y\left(ar{\epsilon}^{\mathrm{p}}
ight)\leq0$$

4. Flow rule

$$\begin{split} \dot{\epsilon}^{\mathrm{p}} &= \chi \beta \dot{\epsilon} \\ \beta &= \frac{E}{E + H\left(\bar{\epsilon}^{\mathrm{p}}\right)} > 0 \quad \text{(by hypothesis)} \end{split}$$

$$H\left(\bar{\epsilon}^{\mathrm{P}}
ight) = rac{dY\left(ar{\epsilon}^{\mathrm{P}}
ight)}{dar{\epsilon}^{\mathrm{P}}}$$

5. Kuhn-Tucker condition

$$\chi=0 \quad \text{ if } f<0, \text{ or if } f=0 \text{ and } n^p \dot{\epsilon}<0, \quad \text{ where } \quad n^p=\operatorname{sign}(\sigma)$$

6. Consistency condition

$$\chi=1 \quad \text{if } f=0 \quad \text{ and } \quad n^p \dot{\epsilon}>0$$

$$\dot{\sigma} = E[1 - \chi \beta] \dot{\epsilon}$$

 $\dot{\sigma} = E_{ an} \dot{\epsilon}$ 

$$E_{\rm tan} = \left\{ \begin{array}{ll} E & \mbox{if } \chi = 0 \\ \frac{EH\left(\overline{\epsilon}^{\rm P}\right)}{E+H\left(\overline{\epsilon}^{\rm P}\right)} & \mbox{if } \chi = 1 \end{array} \right.$$

## 21 3D plasticity with isotropic hardening

21.1 Introduction

21.2 Basic equations

21.3 Kinematical assumptions

21.4 Separability hypothesis

21.5 Constitutive characterization of elastic response

21.6 Constitutive equations for plastic response

21.7 Summary of Mises-Hill Theory

1. Strain decomposition, (21.7.1)

$$\epsilon = \epsilon^{\mathrm{e}} + \epsilon^{\mathrm{P}}$$

2. Constitutive relation (21.7.2)

$$\sigma = 2\mu \left(\epsilon - \epsilon^{\rm p}\right) + (\kappa - (2/3)\mu)(\operatorname{tr}\epsilon)\mathbf{1}$$

with  $\mu$  > 0 and  $\kappa$  > 0 the elastic shear and bulk moduli.

3. Yield condition (21.7.3)

$$f = ar{\sigma} - Y\left(ar{\epsilon}^{ ext{P}}
ight) \leq 0$$

4. Evolution equations, (21.7.5)

$$\begin{split} \dot{\epsilon}^{\mathrm{p}} &= \chi \beta \left( \bar{\epsilon}^{\mathrm{p}} \right) \left( \mathbf{n}^{\mathrm{p}} : \dot{\epsilon} \right) \mathbf{n}^{\mathrm{p}}, \quad \mathbf{n}^{\mathrm{p}} &= \sqrt{3/2} \frac{\sigma'}{\bar{\sigma}} \\ \dot{\epsilon}^{\mathrm{p}} &= \sqrt{2/3} \left| \dot{\epsilon}^{\mathrm{p}} \right| \end{split}$$

with

Stiffness ratio:

$$eta\left(ar{\epsilon}^{\mathrm{p}}
ight)=rac{3\mu}{3\mu+H\left(ar{\epsilon}^{\mathrm{p}}
ight)}>0$$

Hardening modulus (21.7.6):

$$H\left(ar{\epsilon}^{\mathrm{p}}
ight)=rac{dY\left(ar{\epsilon}^{\mathrm{p}}
ight)}{dar{\epsilon}^{\mathrm{p}}}$$

Switching parameter (21.7.7)

$$\chi = \left\{ \begin{array}{ll} 0 & \text{ if } f < 0, \text{ or if } f = 0 \text{ and } n^p : \dot{\epsilon} \leq 0 \\ 1 & \text{ if } f = 0 \text{ and } n^p : \dot{\epsilon} > 0 \end{array} \right.$$

and typical initial conditions:

$$\epsilon(\mathbf{x},0)=\epsilon^{\mathbf{p}}(\mathbf{x},0)=0, \quad \text{ and } \quad \bar{\epsilon}^{\mathbf{p}}(\mathbf{x},0)=0$$

Also note:

$$d\epsilon_{ij} = \underbrace{\frac{1+
u}{E} d\sigma_{ij} - rac{
u}{E} \left( d\sigma_{kk} 
ight) \delta_{ij}}_{d\epsilon_{ij}^e} + \underbrace{\left( 3/2 
ight) dar{\epsilon}^p rac{\sigma_{ij}'}{ar{\sigma}}}_{d\epsilon_{ij}^p}$$

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Partly-plastic spherical shell 400

Fully-plastic spherical shell 403

Residual stresses upon unloading

25 Rigid-perfectly-plastic materials. Two extremum principles

## 25.1 Mixed boundary value problem for a rigid-perfectly-plastic solid

- 25.3 Limit Analysis
- 25.4 Lower bound theorem

Statically admissible stress field, with respect to to traction field  $\boldsymbol{t}^{\ast}$  satisfies:

- 1. Equilibrium,  $\operatorname{div} oldsymbol{\sigma}^* + \mathbf{b}^* = \mathbf{0}$  on  $\mathcal B$
- 2. Traction B.Cs,  $\sigma^*\mathbf{n}=\mathbf{t}^*$  on  $\partial\mathcal{B}$

3. Yield condition: 
$$f\left(\sigma^{*},Y\right)=\sqrt{3/2}\left|\sigma^{*\prime}\right|-Y\leq0$$
 in  ${\cal B}$ 

 $Q=Q_{pl}$ 

#### 25.5 Upper bound theorem

Kinematically admissible velocity field,  $\boldsymbol{v}^*$ , satisfies:

- 1. Stretching-velocity relation,  $\dot{\boldsymbol{\epsilon}}^* = \frac{1}{2} \left( (\nabla \mathbf{v}^*) + (\nabla \mathbf{v}^*)^\top \right);$
- 2. gives no volume change,  ${
  m tr}\, \dot{\epsilon}^*=0$
- 3. Satisfies velocity B.Cs,  $\mathbf{v}^* = \hat{\mathbf{v}}$  on  $\mathcal{S}_1$

#### Upper bound:

$$\begin{split} \Phi \left\{ \mathbf{v}^* \right\} &= \mathcal{D}_{\text{int}} \left\{ \mathbf{v}^* \right\} - \mathcal{W}_{\text{ext}} \left\{ \mathbf{v}^* \right\} \geq \underbrace{\Phi \left\{ \mathbf{v} \right\}}_{=0} \\ \beta_U &= \frac{\sum_{\mathcal{S}_{\hat{d}}} A_{\mathcal{S}_{\hat{d}}^*} k \left[ v^* \right]}{\int_{\mathcal{B}} \tilde{\mathbf{b}} \cdot v^* dv + \int_{\partial \mathcal{B}} \tilde{\mathbf{t}} \cdot v^* da} = \frac{\sum_{\mathcal{S}_{\hat{d}}} A_{\mathcal{S}_{\hat{d}}} \cdot k \left[ v^* \right]}{\int_{\mathcal{B}} \tilde{\mathbf{b}} \cdot v^* dv + \int_{\partial \mathcal{B}} \tilde{\mathbf{t}} \cdot v^* da} \end{split}$$

## 25.5.1 Block-sliding velocity fields

### 25.6.2 Hodograph

### 25.6.3 Plane strain frictionless extrusion

25.7 Plane-strain indentation of a semi-infinite solid with a flat punch