

\mathcal{H}_∞ synthesis: Weighting functions design

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1 Introduction

In the framework of \mathcal{H}_∞ optimization for designing a control system, **weighting functions** are used in order to approximate by using rational functions some constraints in the frequency domain on the sensitivity function $S(s)$ and the complementary sensitivity function $T(s)$. Such constraints are derived by suitably traducing given specifications for the feedback control system. Roughly speaking at the end of the design (if we want to obtain robust stability and nominal performances) we want to impose:

$$\begin{Bmatrix} W_1 S_n \\ W_2 T_n \end{Bmatrix}_\infty < 1, \quad W_1 = W_S, \quad W_2 = \max(W_T, W_U) \quad (1)$$

Where S_n, T_n are related to the *nominal model*.

2 Weighting function $W_S^{-1}(s)$ ($\nu + p = 1$)

We can explicit the constraints on the sensitivity function $S(s)$ using the following inequality:

$$\|W_S(j\omega)S(j\omega)\|_\infty < 1 \quad (2)$$

This is the same to state the for all frequencies ω , $S(s)$ must lie below $W_S^{-1}(s)$. This weighting function is in general the most complicate to design since, when the requirements are translated, different types of constraints comes up:

1. A constraint on the shape of $s^{\nu+p}S^*(0)$ for $\omega \rightarrow 0$
2. A constraint on the level of attenuation for *low frequencies*, what we call M_S^{LF} ;
3. A constraint in the *high frequency* range due to the peak of $S(s)$ of a prototype second order system;

Clearly, it could be that some of this constraints are not present for example the attenuation M_S^{LF} due to a sinusoidal disturbance $d_p(t)$ on the output of the plant. Beyond this constraints, it is useful at this stage to keep in mind the shape of the prototype second order system $S^{II}(s)$:

$$S^{II}(s) = \frac{s(\frac{s}{\omega_n^2} + \frac{2\zeta}{\omega_n})}{(1 + \frac{2\zeta}{\omega_n}s + \frac{s^2}{\omega_n^2})} \quad (3)$$

It is remarkable that the transient requirements are mapped directly on such a type of prototype, then in the mid-range frequencies could be effective tempting to assume that shape.

The introduction we have just done holds in general, then there are some critical aspects related to the design of the weighting function which is related to the system type $\nu+p$. In the following we show some significant examples for weighting functions embedding the constraints on $S(s)$.

2.1 Second order polynomials

Here we want to imitate the shape of $S^{II}(s)$ only in the neighborhood of ω_n . An example of such a type of weighting function is reported in Figure (2.1). The general form for such a type

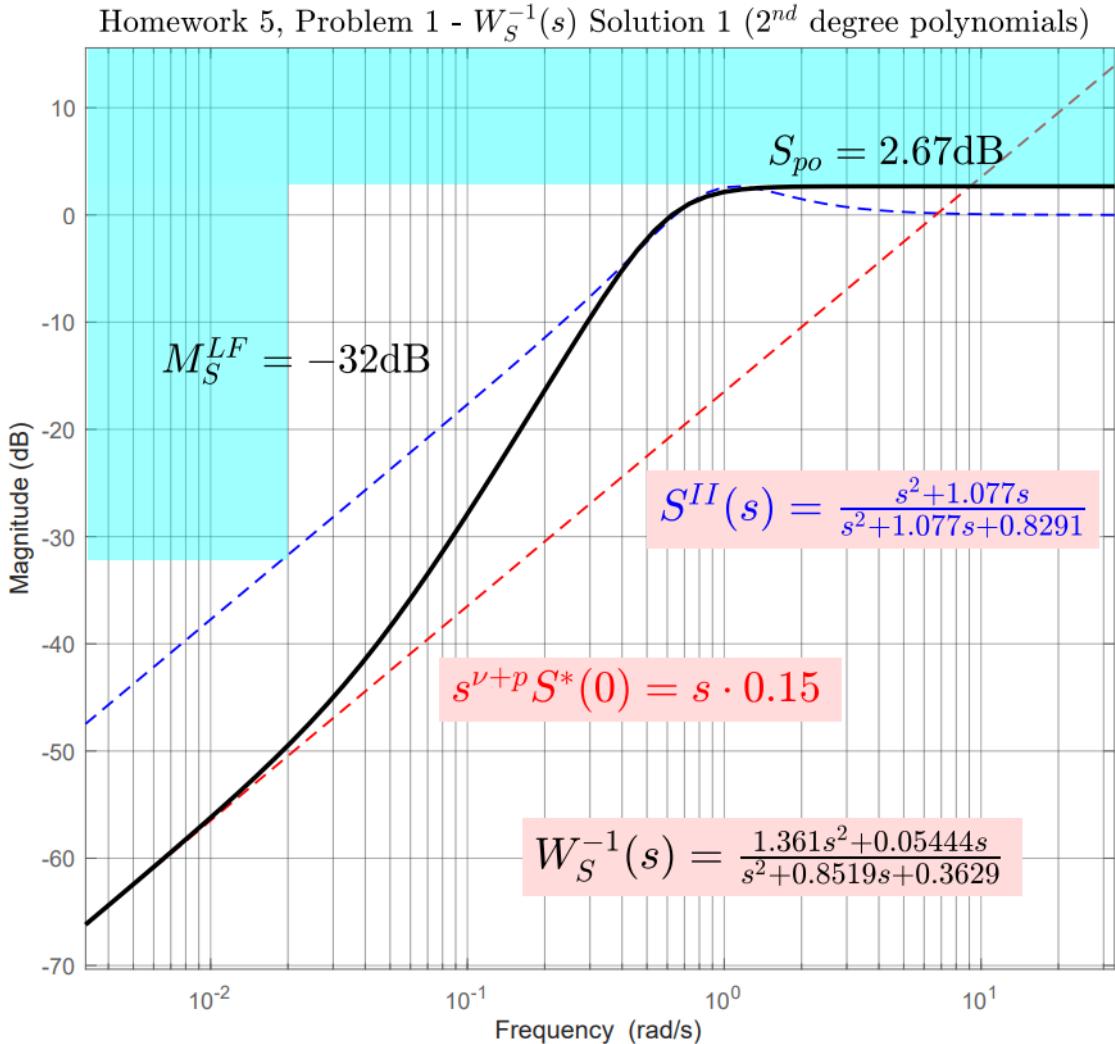


Figure 1: $W_S^{-1}(s)$, system-type 1, second order polynomials

of function is:

$$W_S^{-1}(s) = a s^1 \frac{1 + \frac{s}{\omega_1}}{(1 + \frac{2\zeta}{\omega_2}s + \frac{s^2}{\omega_2^2})} \quad (4)$$

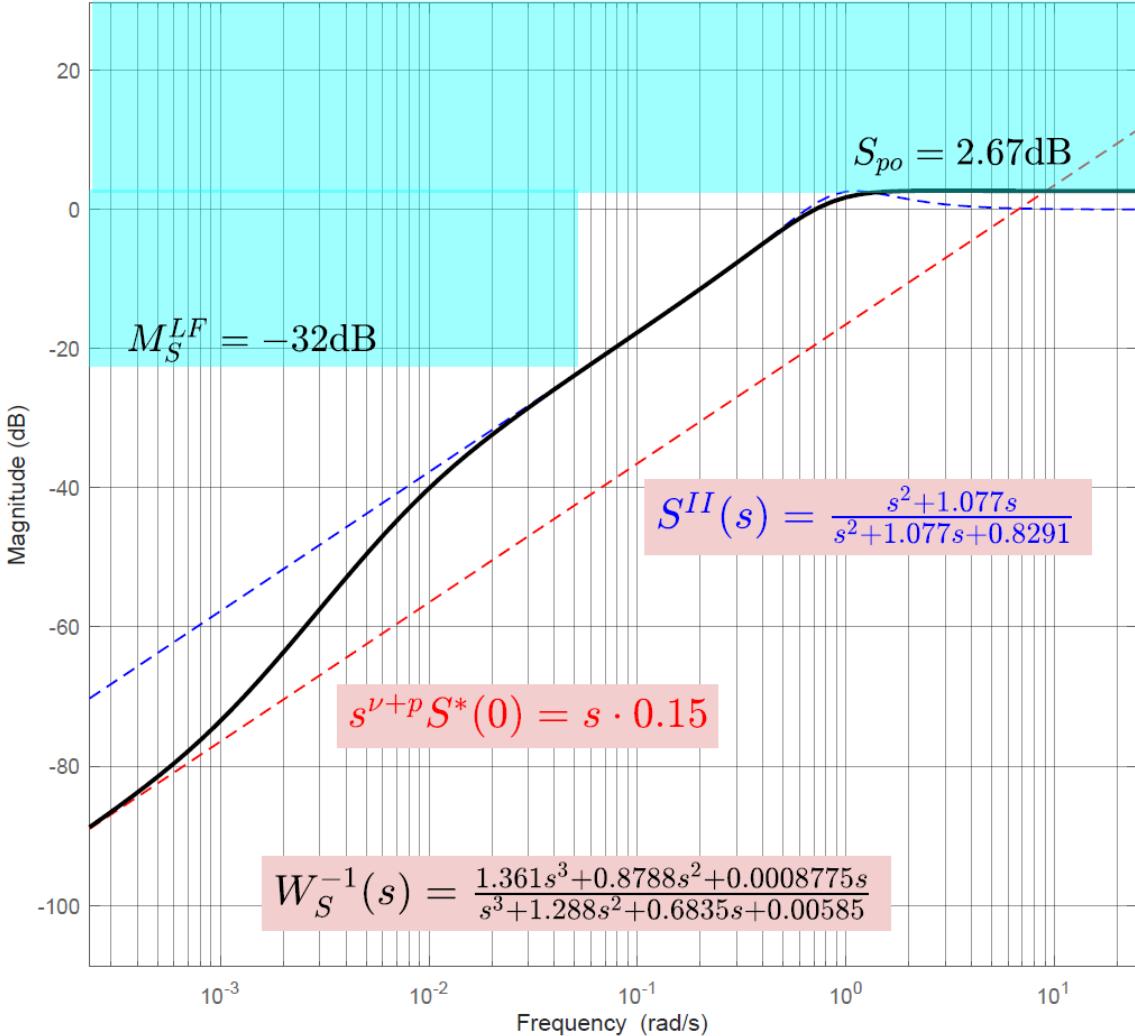
About the parameters:

- The gain a is chosen by computing

$$\lim_{s \rightarrow 0} \frac{1}{s} W_S^{-1}(s) = S^*(0) \quad (5)$$

in this way for $s \rightarrow 0$ the function will be exactly the red dashed line;

- for the zero at ω_1 there is not a precise choice if M_S^{LF} is present a frequency around ω_p can be chosen, otherwise by a trial and error procedure you can chose a frequency such that at a certain point W_S could reach $S^{II}(s)$;


 Figure 2: $W_S^{-1}(s)$, system-type 1, third order polynomial

- In order to imitate the shape of $S^{II}(s)$ around ω_n a pair of complex conjugate poles are used. The frequency ω_2 can be chosen by solving the following equation:

$$S_{po} = \lim_{s \rightarrow \infty} W_S^{-1}(s) \quad (6)$$

The damping factor initially could be $\zeta = 0.707$ in order to use a *Butterworth polynomial*, but it is only a guideline, you can increase/decrease this value so that the functions in the range of the "bump" could be as similar as possible.

2.2 Third order polynomials

In order to obtain better results from the optimizer solving the \mathcal{H}_∞ optimization, higher order polynomial can be used in order to better approximate the behaviour of the prototype second order system. As we have done before, an example of this is reported in Figure (2.2)

The shape of such a function is the following:

$$W_S^{-1}(s) = as^1 \frac{(1 + \frac{s}{z_1})(1 + \frac{s}{z_2})}{(1 + \frac{2\zeta}{\omega}s + \frac{s^2}{\omega^2})(1 + \frac{s}{p})} \quad (7)$$

Here a pair zero-pole ($z1, p$) is used in order to reach the prototype second order sooner as possible, a pair of complex conjugate poles is used to model W_S^{-1} around ω_n , an ω similar to ω_n could be a good choice. The pole p is chosen as in (6) in order to reach the peak S_{po} in the *high-frequency range*.

3 Weighting function $W_S^{-1}(s)$ ($\nu + p = 2$)

Whether from the requirement translation comes up that $\nu + p = 2$, then the red dashed line and S^{II} at LF¹ are not parallel anymore.

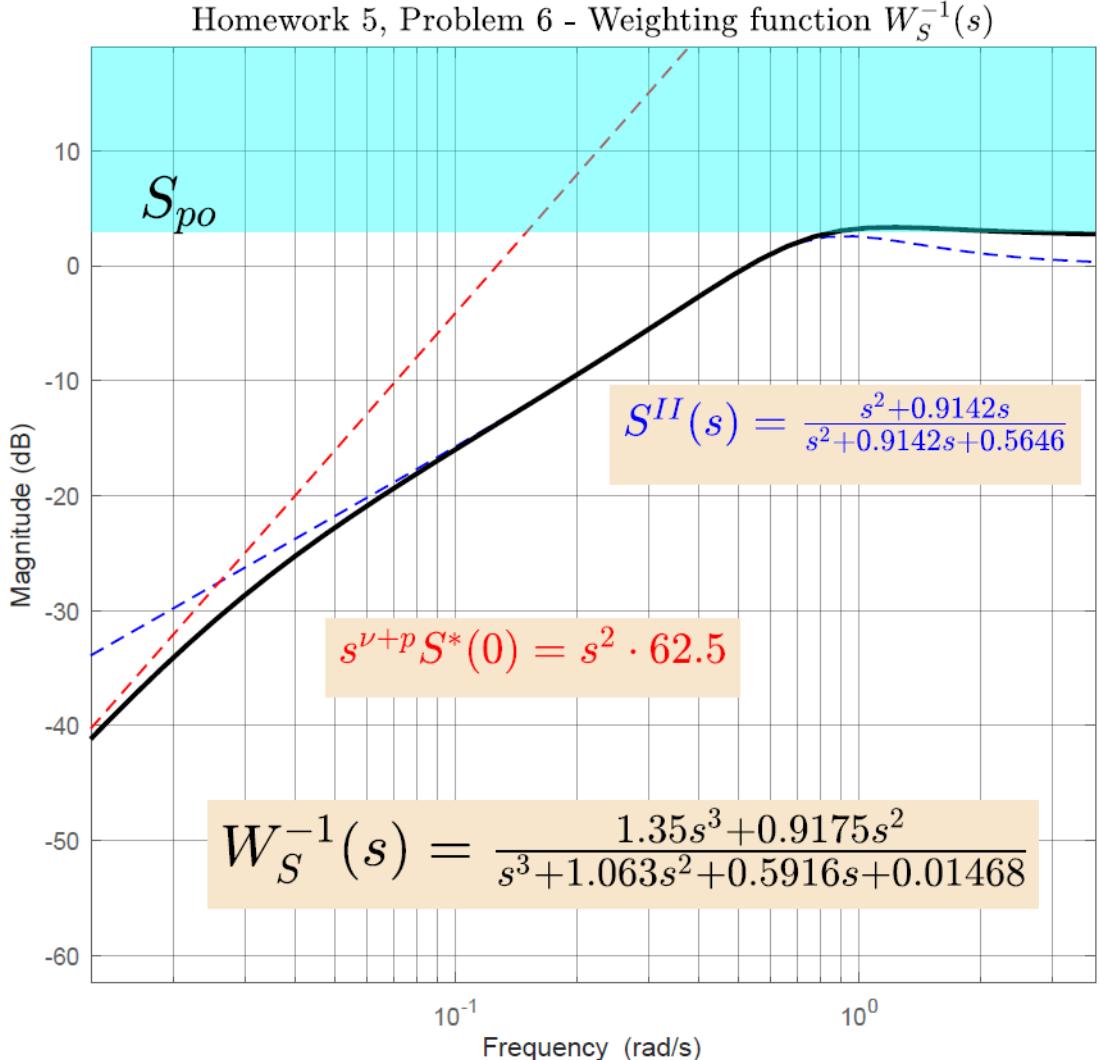


Figure 3: $W_S^{-1}(s)$, system-type 2

The shape for $W_S^{-1}(s)$ in this case is:

$$W_S^{-1}(s) = as^2 \frac{\left(1 + \frac{s}{z}\right)}{\left(1 + \frac{2\zeta}{\omega}s + \frac{s^2}{\omega^2}\right)\left(1 + \frac{s}{p}\right)} \quad (8)$$

The parameter a is chosen according to the same principle as before. Moreover, around the intersection point between $s^2S^*(0)$ we place a pole at the frequency p , then a complex conjugate

¹From now on LF → Low Frequency

pair is added in order to model the 'bump' around ω_n , in order to obtain a slope of 0 dB/dec a zero at the frequency z is needed. The zero is chosen imposing that the condition (6) is met.

4 Weighting function $W_T(s)$

Good news: the design of the weighting function on $T(s)$ is simpler. We recall that in the phase of requirements translation we derive (at most) two constraints: (i) the first due to the sinusoidal disturbance at HF on the sensor, (ii) the second related to the maximum resonance peak T_{po} derived from the requirement on the overshoot \hat{s} . The shape of such a rational function is always the same:

$$W_T^{-1}(s) = \frac{T_{po}}{\left(1 + \frac{2\zeta}{\omega_T} s + \frac{s^2}{\omega_T^2}\right)} \quad (9)$$

The parameter ω_T is chosen by using the following equation:

$$\omega_T = \omega_s 10^{\frac{M_{T,dB}^{HF} - T_{po,dB}}{40}} \quad (10)$$

so that the $W_T^{-1}(s)$ could perfectly pass for the corner imposed by the pair (ω_s, M_T^{HF}) . An example is reported in the following figure.

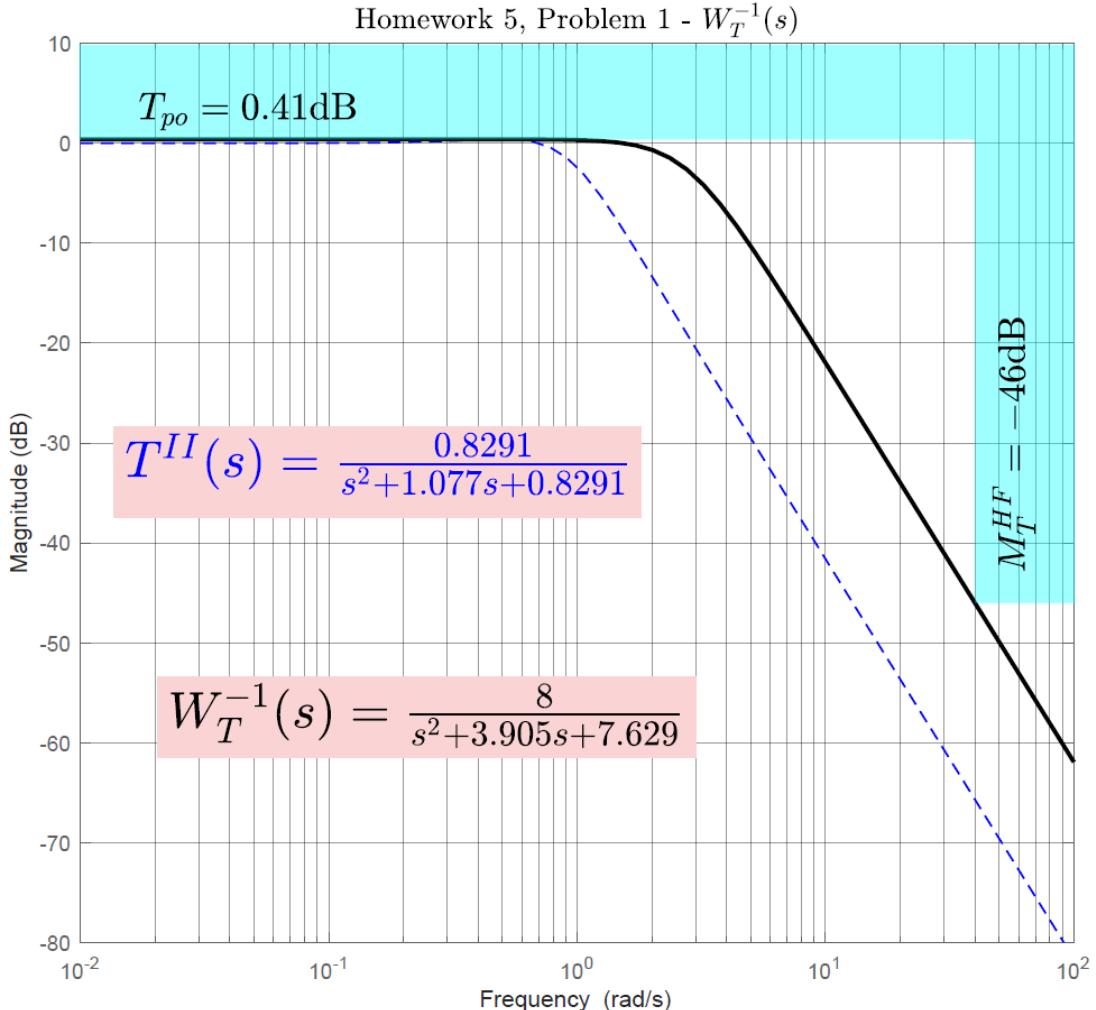


Figure 4: Weighting function $W_T^{-1}(s)$ on $T(s)$

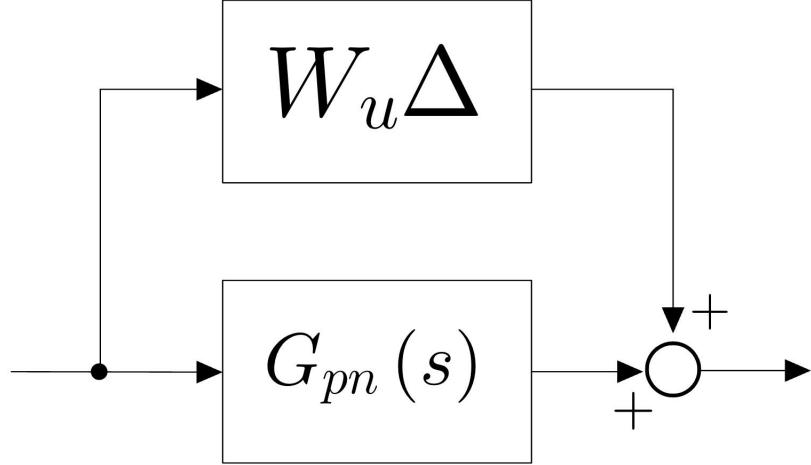


Figure 5: Additive uncertainty model (Block diagram)

5 Weighting function $W_U(s)$

Despite our best effort to approximate real dynamical systems, there is always uncertainty which could be either dynamical (this is due to the neglection of some parts of the system), or *parametric* (this is due to the uncertainty on the parameters entering in the mathematical description of a certain system for example by using a transfer function). Different approaches for describing the uncertainty can be considered.

5.1 Additive uncertainty model

In this case the **real plant** is considered as a member of a family of systems which can be described by the set

$$M_a = \{G_p(s) = G_{pn}(s) + W_U(s)\Delta(s), \quad \|\Delta(s)\|_\infty \leq 1\} \quad (11)$$

where $G_p(s)$ is a member of the family, $G_{pn}(s)$ is a given **nominal plant**, $W_U(s)$ is a **weighting function** taking into account (frequency-by-frequency) the maximum uncertainty entering into the problem, finally $\Delta(s)$ is any transfer function whose \mathcal{H}_∞ norm is less than one.

If we go deeper and retrieve the expression of $\Delta(s)$ we can write that

$$|G_p(j\omega) - G_{pn}(j\omega)| \leq |W_U(j\omega)| \quad \forall \omega \iff |W_U(j\omega)| \geq |G_p(j\omega) - G_{pn}(j\omega)| \quad (12)$$

This gives us useful information on the weighting function $W_U(s)$ that must lie above the *transfer functions cloud* obtained by gridding the parameter space and drawing the function $|G_p(j\omega) - G_{pn}(j\omega)|$ for each combination of the parameters entering into the description of the **real uncertain plant**. (see Homework 6, Problem A). By using the (11) we can represent such a model of uncertainty by using the following block diagram:

5.2 Multiplicative uncertainty model

Here the real uncertain plant is considered as part of a set of a dynamical system M_m described by:

$$M_m = \{G_p(s) = G_{pn}(s)(1 + W_U(s)\Delta(s)), \quad \|\Delta(s)\| \leq 1\} \quad (13)$$

Where $G_p(s)$, $G_{pn}(s)$, $\Delta(s)$, $W_U(s)$ are defined as before. What is different in this case is the relationship between $\Delta(s)$ and the weighting function $W_U(s)$. By doing simple calculation and

applying the definition of \mathcal{H}_∞ norm² we obtain:

$$|W_U(j\omega)| \geq \left| \frac{G_p(j\omega)}{G_{pn}(j\omega)} - 1 \right| \quad \forall \omega \quad (14)$$

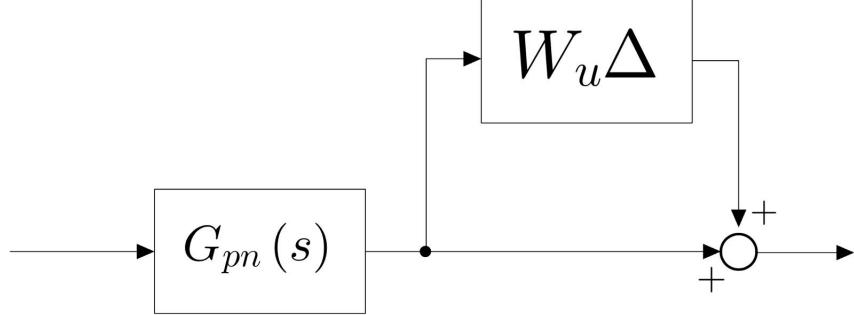


Figure 6: Multiplicative Uncertainty Interval (block diagram)

5.3 Obtaining a nominal plant given the PUI

Before going on the description of the method by which a $W_U(s)$ is obtained, we have to deal with an issue which is directly linked to our problems: given a plant described by a transfer function in which some parameters enter and are described by the Parameter Uncertainty Intervals (PUI), **how can we obtain a nominal model for the plant under analysis?** The statistics says us that peeking the average of the extrema for each parameter we obtain the minimum uncertainty. That is *the center of each PUI guarantees the minimum uncertainty*. This is not always true, but for the cases of our interest is a good technique in order to retrieve the nominal model. Let us give an example:

$$G_p(s) = K \frac{(1 + \frac{1}{z_1})}{(1 + \frac{1}{p_1})}, \quad K \in [\underline{K}, \bar{K}], \quad z_1 \in [\underline{z}_1, \bar{z}_1], \quad p_1 \in [\underline{p}_1, \bar{p}_1] \quad (15)$$

The nominal model is:

$$G_{pn}(s) = K_n \frac{(1 + \frac{1}{z_{1n}})}{(1 + \frac{1}{p_{1n}})}, \quad K_n = \frac{\underline{K} + \bar{K}}{2} \quad z_{1n} = \frac{\underline{z}_1 + \bar{z}_1}{2} \quad p_{1n} = \frac{\underline{p}_1 + \bar{p}_1}{2} \quad (16)$$

Note that this is the unstructured way to describe structured uncertainty, other techniques provide us with the possibility to dealing with structured uncertainty without conservativeness (μ -analysis).

5.4 On the design of $W_U(s)$

From the previous paragraphs we have understood that the weighting function $W_U(s)$ must lie above the cloud generated by (12) or (14). There are mainly two ways to design it:

²The \mathcal{H}_∞ norm is defined as:

$$\|H(s)\|_\infty = \max_\omega |H(j\omega)|$$

where $H : \mathbb{C} \rightarrow \mathbb{C}$ is a complex function of the complex variable s .

1. **By Hand**, in the sense that a transfer function $W_U(s)$ is implemented by adding zeros and poles and considering the following constraints:

$$\lim_{s \rightarrow 0} W_U(s) = W_U^0 \quad (17)$$

$$\lim_{s \rightarrow \infty} W_U(s) = W_U^\infty \quad (18)$$

where W_U^0 and W_U^∞ can be found empirically by analyzing the graph.

2. **By using the fitmag command**, in this case a proper choice is to pick frequency-by-frequency the maximum gain of the function $\Delta_a(s)$ or $\Delta_m(s)$ (additive or multiplicative).

In the following the family of transfer functions and the weight $W_U(s)$ is showed in the case the second method is used.

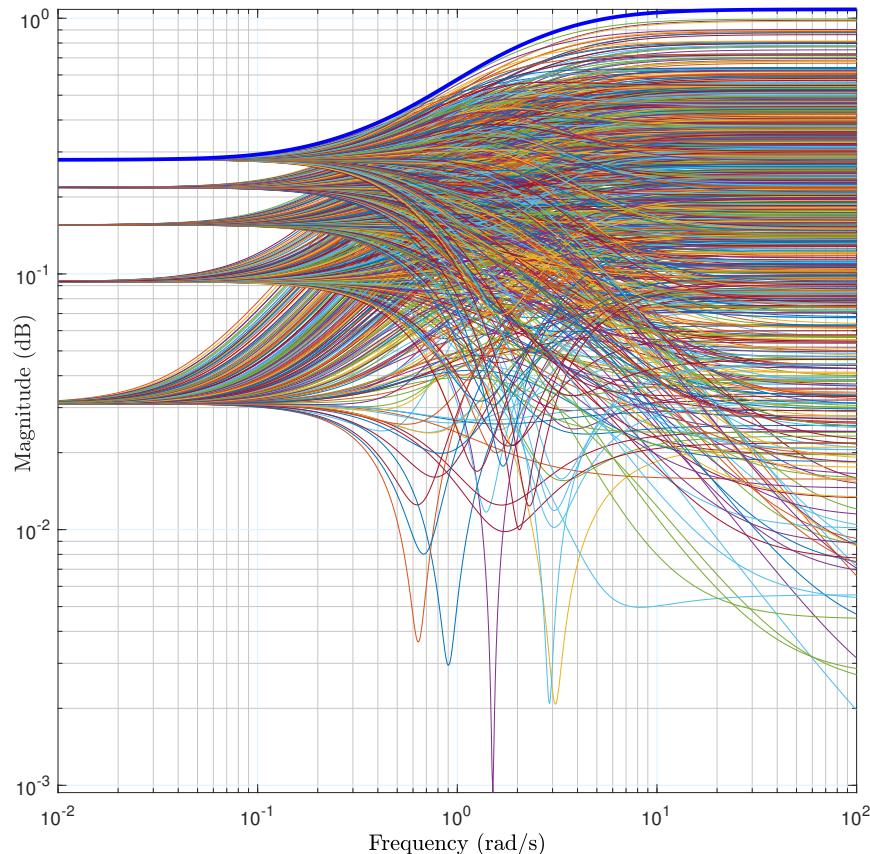


Figure 7: Transfer functions cloud and $W_U(s)$