

Name: Solution

ChBE 2130 Thermodynamics I
Fall 2015
Exam 3

Remember

- Write down relevant relationships needed to solve each problem
- Provide details, intermediate steps, and units
- Note any assumptions
- Show your work
- Where indicated, place your final answer on the _____
- **Submit your crib sheet with your exam.**

Problem	Possible Points	Score
1	20	
2	20	
3	20	
4	40	
Total		

1. Concept Questions [20 pts: 5 points each, no partial credit within sub-problem]

- For an ideal gas, simplify the equation, $V - T \left(\frac{\partial V}{\partial T} \right)_P$

- a. V
- b. R/P
- c. RT/P
- d. 0

for ideal gas $V = \frac{RT}{P}$ $\left(\frac{\partial V}{\partial T} \right)_P = \frac{R}{P}$
by substitution

$$\frac{RT}{P} - T \left(\frac{R}{P} \right) = 0$$

- Using the differential form of enthalpy as a basis, simplify $\left(\frac{\partial H}{\partial S} \right)_P$

- a. P
- b. T
- c. U
- d. 0

$$dH = TdS + VdP$$

$$\text{constant } P \Rightarrow VdP = 0$$

$$dH = TdS \quad \left(\frac{\partial H}{\partial S} \right)_P = T$$

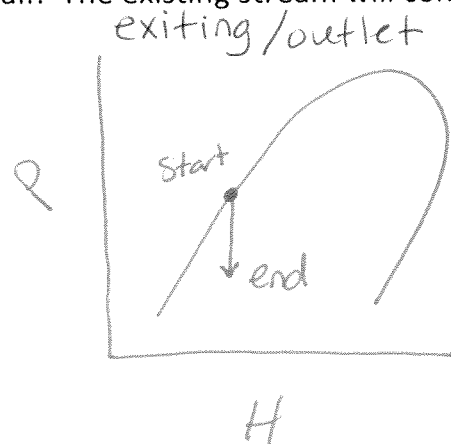
- A gas is flowing in an insulated duct of constant cross sectional area at stable conditions. Compared to the inlet conditions, the outlet will have

- a. Lower pressure and higher velocity
- b. Higher pressure and higher velocity
- c. Lower pressure and lower velocity
- d. Higher pressure and lower velocity

Gas \rightarrow Compressible
Insulated \rightarrow Adiabatic
Stable \rightarrow Subsonic

- A saturated liquid at moderate temperature and pressure enter a throttle valve where the pressure is reduced by half. The ~~existing~~ exiting stream will consist of which of the following

- a. Subcooled liquid
- b. Saturated liquid
- c. Mixed phase – gas and liquid
- d. Saturated gas



Throttle $\Rightarrow \Delta H = 0$
Pressure is reduced

2. [20 pts] The behavior of a fluid is described by the following equation of state:

$V = B + (RT/p)$ where B is a constant. Develop a simplified expression for the residual enthalpy, H^R .

$$Z = \frac{PV}{RT} \text{ by substitution of } V = B + \left(\frac{RT}{P}\right)$$

$$Z = \frac{BP}{RT} + \frac{RT P}{RTP} = \frac{BP}{RT} + 1 \quad (b)$$

$$Z = f(P, T)$$

$$\frac{H^R}{RT} = - \int_0^P T \left(\frac{\partial Z}{\partial T} \right)_P \frac{dP}{P} \quad (b)$$

$$\left(\frac{\partial Z}{\partial T} \right)_P = - \frac{BP}{RT^2} \quad (4)$$

$$\frac{H^R}{RT} = - \int_0^P T \left(- \frac{BP}{RT^2} \right) \frac{dP}{P} = \int_0^P \frac{B}{RT} dP \quad (2)$$

$$\frac{H^R}{RT} = \left. \frac{BP}{RT} \right|_0^P$$

$$H^R = BP \quad (2)$$

3. [20 pts] In terms of experimentally measurable properties, $\left(\frac{\partial T}{\partial V}\right)_S$ equals which of the following? Show your calculations and substitutions for full credit. Hint: Start with the cyclic relationship for $\left(\frac{\partial T}{\partial V}\right)_S$.

- a. $-(T/C_V)(\beta/\kappa)$
- b. $-(T/C_V)(1/\beta V)$
- c. $(C_P/T)(1/\beta V)$
- d. $(C_P/T)(\beta/\kappa)$

$$(5) \quad \left(\frac{\partial T}{\partial V}\right)_S = -\left(\frac{\partial T}{\partial S}\right)_V \left(\frac{\partial S}{\partial V}\right)_T$$

$$(5) \quad -\left(\frac{\partial T}{\partial S}\right)_V = -\frac{T}{C_V} \quad \text{by definition}$$

$$(5) \quad \left(\frac{\partial S}{\partial V}\right)_T = \left(\frac{\partial P}{\partial T}\right)_V \quad \text{by Maxwell Eq}$$

$$(5) \quad \left(\frac{\partial P}{\partial T}\right)_V = \frac{\beta}{\kappa} \quad \text{by definition}$$

$$\left(\frac{\partial T}{\partial V}\right)_S = -\frac{T}{C_V} \left(\frac{\beta}{\kappa}\right)$$

8pts each


4. [40 pts] Steam enters a turbine at 2000 kPa and 500°C. The exhaust is at 20 kPa.

a. [16] If the turbine operates isentropically, what is the outlet phase and/or quality and the enthalpy?

$$x = \frac{s^v - s}{s^v - s^l} = \frac{7.9094 - 7.4323}{7.9094 - 0.8321}$$

$P_1 = 2000 \text{ kPa}$
 $T_1 = 500^\circ\text{C}$
 $H_1 = 3467.3 \text{ kJ/kg}$
 $S_1 = 7.4323 \text{ kJ/kgK}$

$P_2 = 20 \text{ kPa}$



Phase/Quality Mixed $x = 0.067$
 $q = 0.933$

$$H = xH^v + (1-x)H^l$$

$$H = 0.067(251.453) + (1-0.067)2609.9$$

Outlet Enthalpy 2451.9 kJ/kg

OR $H = 0.0674(251.453) + (1-0.0674)2609.9 = 2450.9$

b. [24] If work is produced at a rate of 2500 kW and the exhaust is saturated vapor, what is the outlet enthalpy, mass flow rate, and turbine efficiency?
 Note that a kW = kJ/s

20 kPa, sat. vapor \rightarrow From Steam table

Outlet Enthalpy 2609.9 kJ/kg

$$\dot{W} = \dot{m} \Delta H \quad \dot{m} = \frac{\dot{W}}{\Delta H}$$

$$\frac{-2500 \text{ kJ/s}}{(2609.9 - 3467.3) \text{ kJ}} \text{ kg}$$

Mass Flow Rate (in kg/s) 2.92 kg/s

$$\eta = \frac{\Delta H}{\Delta H_{\text{isen}}} = \frac{2609.9 - 3467.3}{2451.9 - 3467.3}$$

OR $\eta = \frac{\Delta H}{\Delta H_{\text{isen}}} = \frac{2609.9 - 3467.3}{2450.9 - 3467.3} = 84.36$

Efficiency 84.4%

Table F.2. Superheated Steam, SI Units (Continued)

TEMPERATURE: $t^{\circ}\text{C}$
(TEMPERATURE: T kelvins)

P/kPa ($t^{\text{sat}}/^{\circ}\text{C}$)	sat. liq.	sat. vap.	300 (573.15)	350 (623.15)	400 (673.15)	450 (723.15)	500 (773.15)	550 (823.15)	600 (873.15)	650 (923.15)
V	1.000	129200.	264500.	287580.	310660.	333730.	356810.	379880.	402960.	426040.
U	29.334	2385.2	2812.3	2889.9	2969.1	3049.9	3132.4	3216.7	3302.6	3390.3
H	29.335	2514.4	3076.8	3177.5	3279.7	3383.6	3489.2	3596.5	3705.6	3816.4
S	0.1060	8.9767	10.3450	10.5133	10.6711	10.8200	10.9612	11.0957	11.2243	11.3476
V	1.010	14670.	26440.	28750.	31060.	33370.	35670.	37980.	40290.	42600.
U	191.822	2438.0	2812.2	2889.8	2969.0	3049.8	3132.3	3216.6	3302.6	3390.3
H	191.832	2584.8	3076.6	3177.3	3279.6	3383.5	3489.1	3596.5	3705.5	3816.3
S	0.6493	8.1511	9.2820	9.4504	9.6083	9.7572	9.8984	10.0329	10.1616	10.2849
V	1.017	7649.8	13210.	14370.	15520.	16680.	17830.	18990.	20140.	21300.
U	251.432	2456.9	2812.0	2889.6	2968.9	3049.7	3132.3	3216.5	3302.5	3390.2
H	251.453	2609.9	3076.4	3177.1	3279.4	3383.4	3489.0	3596.4	3705.4	3816.2
S	0.8321	7.9094	8.9618	9.1303	9.2882	9.4372	9.5784	9.7130	9.8416	9.9650
V	1.022	5229.3	8810.8	9581.2	10350.	11120.	11890.	12660.	13430.	14190.
U	289.271	2468.6	2811.8	2889.5	2968.7	3049.6	3132.2	3216.5	3302.5	3390.2
H	289.302	2625.4	3076.1	3176.9	3279.3	3383.3	3488.9	3596.3	3705.4	3816.2
S	0.9441	7.7695	8.7744	8.9430	9.1010	9.2499	9.3912	9.5257	9.6544	9.7778
V	1.027	3993.4	6606.5	7184.6	7762.5	8340.1	8917.6	9494.9	10070.	10640.
U	317.609	2477.1	2811.6	2889.4	2968.6	3049.5	3132.1	3216.4	3302.4	3390.1
H	317.650	2636.9	3075.9	3176.8	3279.1	3383.1	3488.8	3596.2	3705.3	3816.1
S	1.0261	7.6709	8.6413	8.8100	8.9680	9.1170	9.2583	9.3929	9.5216	9.6450
V	1.030	3240.2	5283.9	5746.7	6209.1	6671.4	7133.5	7595.5	8057.4	8519.2
U	340.513	2484.0	2811.5	2889.2	2968.5	3049.4	3132.0	3216.3	3302.3	3390.1
H	340.564	2646.0	3075.7	3176.6	3279.0	3383.0	3488.7	3596.1	3705.2	3816.0
S	1.0912	7.5947	8.5380	8.7068	8.8649	9.0139	9.1552	9.2898	9.4185	9.5419
V	1.037	2216.9	3520.5	3829.4	4138.0	4446.4	4754.7	5062.8	5370.9	5678.9
U	384.374	2496.7	2811.0	2888.9	2968.2	3049.2	3131.8	3216.1	3302.2	3389.9
H	384.451	2663.0	3075.1	3176.1	3278.6	3382.7	3488.4	3595.8	3705.0	3815.9
S	1.2131	7.4570	8.3502	8.5191	8.6773	8.8265	8.9678	9.1025	9.2312	9.3546
V	1.043	1693.7	2638.7	2870.8	3102.5	3334.0	3565.3	3796.5	4027.7	4258.8
U	417.406	2506.1	2810.6	2888.6	2968.0	3049.0	3131.6	3216.0	3302.0	3389.8
H	417.511	2675.4	3074.5	3175.6	3278.2	3382.4	3488.1	3595.6	3704.8	3815.7
S	1.3027	7.3598	8.2166	8.3858	8.5442	8.6934	8.8348	8.9695	9.0982	9.2217

V = SPECIFIC VOLUME $\text{cm}^3 \text{g}^{-1}$
 U = SPECIFIC INTERNAL ENERGY kJ kg^{-1}
 H = SPECIFIC ENTHALPY kJ kg^{-1}
 S = SPECIFIC ENTROPY $\text{kJ kg}^{-1} \text{K}^{-1}$

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		Temperature °C									
		400	425	450	475	500	550	600	650		
	Sat. liq.									Sat. vap.	
1750 (205.72)	V 1.166 U 876.234 H 878.274 S 2.3846	173.32 2949.3 3252.7 7.1955	180.32 2991.3 3306.9 7.2746	187.26 3033.4 3361.1 7.3509	194.17 3075.7 3415.5 7.4248	201.04 3118.2 3470.0 7.4965	214.71 3204.3 3580.0 7.6344	228.28 3291.8 3691.3 7.7656	241.80 3380.8 3803.9 7.8910		
1800 (207.11)	V 1.168 U 882.472 H 884.574 S 2.3976	168.39 2948.8 3251.9 7.1816	175.20 2990.8 3306.1 7.2608	181.97 3032.9 3360.4 7.3372	188.69 3075.2 3414.9 7.4112	195.38 3117.8 3469.5 7.4830	208.68 3203.9 3579.5 7.6209	221.89 3291.5 3690.9 7.7522	235.03 3380.5 3803.6 7.8777		
1850 (208.47)	V 1.170 U 888.585 H 890.750 S 2.4103	163.73 2948.2 3251.1 7.1681	170.37 2990.3 3305.4 7.2474	176.96 3032.4 3359.8 7.3239	183.50 3074.8 3414.3 7.3980	190.02 3117.4 3468.9 7.4698	202.97 3203.6 3579.1 7.6079	215.84 3291.1 3690.4 7.7392	228.64 3380.2 3803.2 7.8648		
1900 (209.80)	V 1.172 U 894.580 H 896.807 S 2.4228	159.30 2947.6 3250.3 7.1550	165.78 2989.7 3304.7 7.2344	172.21 3031.9 3359.1 7.3109	178.59 3074.3 3413.7 7.3851	184.94 3117.0 3468.4 7.4570	197.57 3203.2 3578.6 7.5951	210.11 3290.8 3690.0 7.7265	222.58 3380.0 3802.8 7.8522		
1950 (211.10)	V 1.174 U 900.461 H 902.752 S 2.4349	155.11 2947.0 3249.5 7.1421	161.43 2989.2 3304.0 7.2216	167.70 3031.5 3358.5 7.2983	173.93 3073.9 3413.1 7.3725	180.13 3116.6 3467.8 7.4445	192.44 3202.9 3578.1 7.5827	204.67 3290.5 3689.6 7.7142	216.83 3379.7 3802.5 7.8399		
2000 (212.37)	V 1.177 U 906.236 H 908.589 S 2.4469	151.13 2946.4 3248.7 7.1296	157.30 2988.7 3303.3 7.2092	163.42 3031.0 3357.8 7.2859	169.51 3073.5 3412.5 7.3602	175.55 3116.2 3467.3 7.4323	187.57 3202.5 3577.6 7.5706	199.50 3290.2 3689.2 7.7022	211.36 3379.4 3802.1 7.8279		
2100 (214.85)	V 1.181 U 917.479 H 919.959 S 2.4700	143.73 2945.3 3247.1 7.1053	149.63 2987.6 3301.8 7.1851	155.48 3030.0 3356.5 7.2621	161.28 3072.6 3411.3 7.3365	167.06 3115.3 3466.2 7.4087	178.53 3201.8 3576.7 7.5472	189.91 3289.6 3688.4 7.6789	201.22 3378.9 3801.4 7.8048		
2200 (217.24)	V 1.185 U 928.346 H 930.953 S 2.4922	137.00 2944.1 3245.5 7.0821	142.65 2986.6 3300.4 7.1621	148.25 3029.1 3355.2 7.2393	153.81 3071.7 3410.1 7.3139	159.34 3114.5 3465.1 7.3862	170.30 3201.1 3575.7 7.5249	181.19 3289.0 3687.6 7.6568	192.00 3378.3 3800.7 7.7827		
2300 (219.55)	V 1.189 U 938.866 H 941.601 S 2.5136	130.85 2942.9 3243.9 7.0598	136.28 2985.5 3299.0 7.1401	141.65 3028.1 3353.9 7.2174	146.99 3070.8 3408.9 7.2922	152.28 3113.7 3464.0 7.3646	162.80 3200.4 3574.8 7.5035	173.22 3288.3 3686.7 7.6355	183.58 3377.8 3800.0 7.7616		