FORMERLY-Math

Constrained Form-Finding through Membrane Equilibrium Analysis in Mathematica

https://github.com/colivieri89/FORMERLY-Math

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What is FORMERLY-Math?

FORMERLY-Math: Constrained Form-Finding through Membrane Equilibrium Analysis in Mathematica

It is a package designed for the form-finding of compressed or tensile membranes in Mathematica.

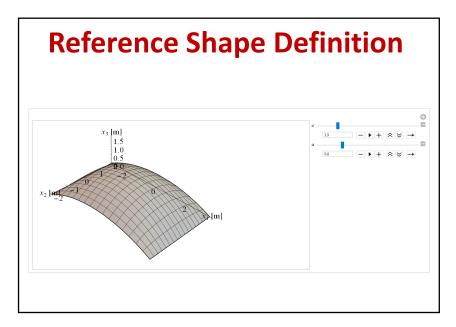
Mission:

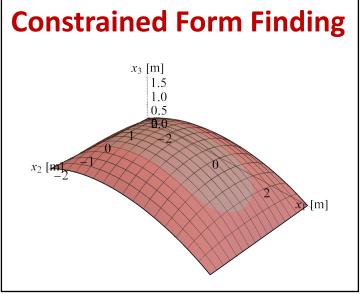
- static assessment method for masonry vaults within the framework of Limit Analysis;
- searching for efficient new forms in structural design.

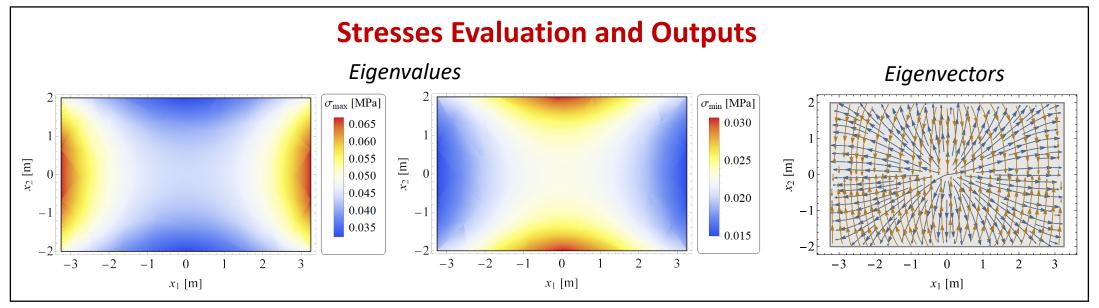
FORMERLY-Math Modules

- Pre-processing
- Reference Shape Definition
- Constrained Form Finding
- Stresses Evaluation and Outputs

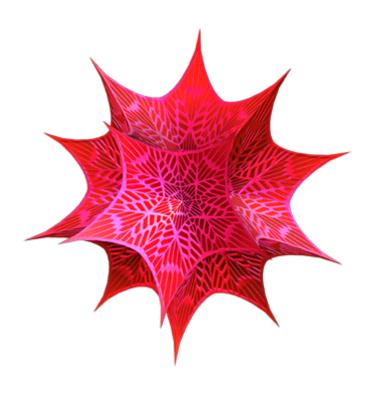
Pre-processing Data Input Airy Stress Function Boundary conditions







Software



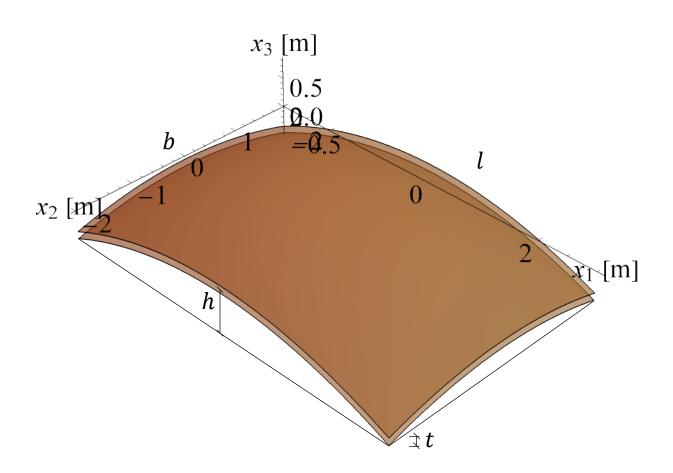
FORMERLY-Math is compatible with the proprietary software Wolfram Mathematica[©] (run successfully on versions 11.2, 12.1, and 13.2)

Pre-Processing

Initializing

Load the required packages (NDSolve 'FEM' and Notation) by running the sub-module.

Data



In this sub-module is possible to assign the geometrical parameters b, l, h, and t, as well as the shell material density ρ_a , and the mesh size size.

These parameters will define the initial sell weight w, the shell domain Ω , and the boundary conditions Γ_{S^0} , Γ_{S^1} , and Γ_{S^2} .

Domain and Airy Stress Function Definition

Airy Stress Function

```
\mathcal{A}[2][\sigma_{}, \alpha_{}] := \sigma (1/8 (1 (b^2 - 4x2^2) + \alpha (1^2 - 4x1^2))) (*Quadratic Airy Stress Function*) 
\mathcal{A}[4][\sigma_{}, \alpha_{}] := \sigma / 2 ((b^2 - x2^2)^2 + \alpha (1^2 - x1^2)^2) (*Quartic Airy Stress Function*)
```

To select an Airy stress function for the equation, please use one of the following notations: i=2 for the quadratic function, or i=4 for the quartic function. Note that different Airy stress functions can be used to analyze different solution fields, but it's important to ensure that the function you choose is either completely concave or convex.

```
i = 2;
A[\sigma_{-}, \alpha_{-}] := \{\{x1, x2, \mathcal{F}[i][\sigma, \alpha]\}, \{x1, x2\} \in \Omega\}
```

Conditions Γ_{S^i}

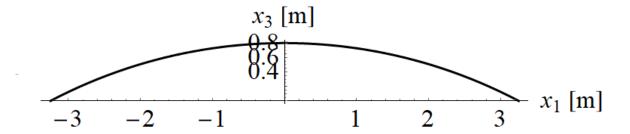
In this sub-module three Dirichlet boundary conditions are defined:

Domain boundary ($\Gamma_{\mathcal{S}^0}$)

$$\Gamma_{S^0} = 0;$$

Arches in x_1 direction (Γ_{S^1})

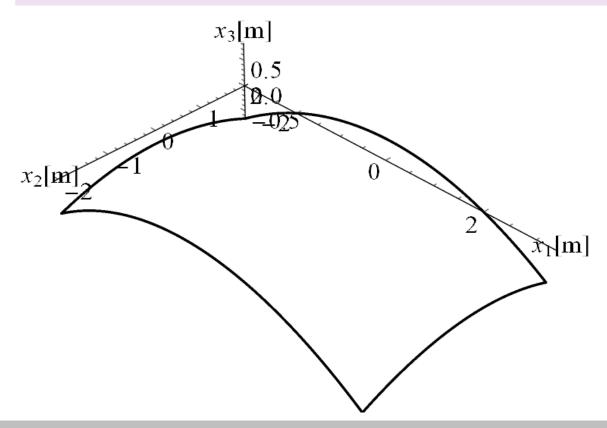
$$\Gamma_{S^1} = x3 /. soll[[2]];$$



Conditions Γ_{S^i}

Arches in x_1 and x_2 direction (Γ_{S^2})

$$\Gamma_{s^2} = h + (-(2h/b^2)(x1^2 + x2^2));$$



Reference Shape Definition

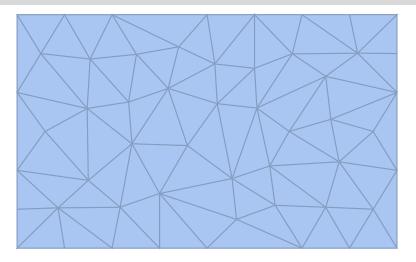
Transversal Equilibrium Equation Solution

The second-order partial differential equation $F_{,22}$ $f_{,11} + F_{,11}$ $f_{,22} - 2$ $F_{,12}$ $f_{,12} - w = 0$ is set up in this module and solved parametrically with the chose Dirichlet boundary conditions bcs through a FEM routine.

Equilibrium solution

```
\begin{split} &\textbf{h}_{\mathcal{S}^1} = \textbf{f}_{\mathcal{S}^1} \text{ /. ParametricNDSolve} \big[ \\ & & \{ \text{EqDiff}[\sigma, \alpha] = \emptyset, \, \text{bcs}[\mathbf{i}] \, \}, \, \textbf{f}_{\mathcal{S}^1}, \, \{ \text{x1, x2} \} \in \Omega, \, \{ \sigma, \, \alpha \}, \\ & \text{Method} \rightarrow \big\{ \text{"FiniteElement",} \\ & \text{"MeshOptions"} \rightarrow \big\{ \text{"MeshOrder"} \rightarrow 2, \, \text{MaxCellMeasure} \rightarrow \text{size} \big\}, \\ & \text{"IntegrationOrder"} \rightarrow 4, \\ & \text{"InterpolationOrder"} \rightarrow \big\{ \textbf{f}_{\mathcal{S}^1} \rightarrow 2 \big\} \big\} \big]; \end{split}
```

Transversal Equilibrium Equation Solution



Equilibrium solution

In this implementation, standard 6-node quadratic triangle mesh elements, with 3 nodes at vertices and 3 nodes at the edges' midpoints, are employed. The Gauss quadrature is employed for numerical integration and is capable of exactly integrating 4th-order polynomials with an interpolation of order 2. These choices generally yield satisfactory convergence for smooth problems, which are the target of this software tool. For a good balance between computational time and results, setting the size of MaxCellMeasure to be about 10 percent of the plan dimension of the structure is suggested.

```
\begin{split} & h_{\mathcal{S}^1} = f_{\mathcal{S}^1} \ / \ . \ \text{ParametricNDSolve} \big[ \\ & \left\{ \text{EqDiff}[\sigma, \, \alpha] = 0 \, , \, \text{bcs}[\mathbf{i}] \, \right\}, \, f_{\mathcal{S}^1}, \, \left\{ \text{x1, x2} \right\} \in \Omega, \, \left\{ \sigma, \, \alpha \right\}, \\ & \text{Method} \rightarrow \left\{ \text{"FiniteElement",} \right. \\ & \text{"MeshOptions"} \rightarrow \left\{ \text{"MeshOrder"} \rightarrow 2 \, , \, \text{MaxCellMeasure} \rightarrow \text{size} \right\}, \\ & \text{"IntegrationOrder"} \rightarrow 4 \, , \\ & \text{"InterpolationOrder"} \rightarrow \left\{ f_{\mathcal{S}^1} \rightarrow 2 \right\} \big\} \big] \, ; \end{split}
```

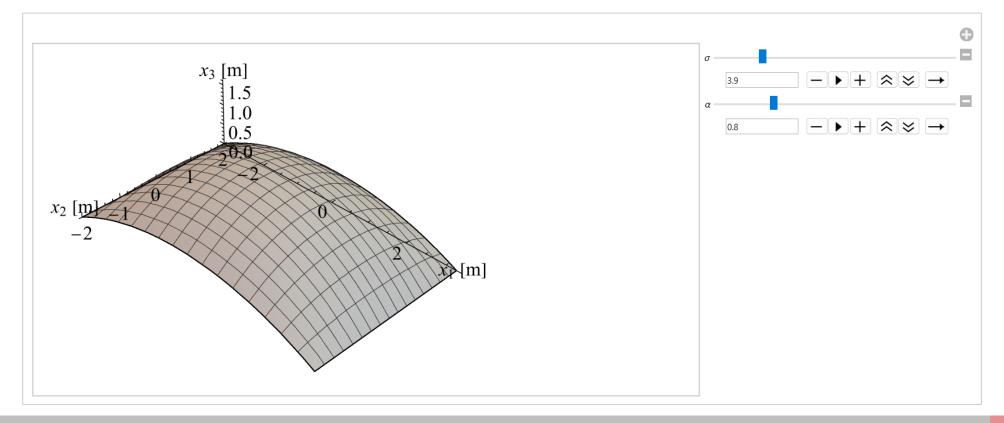
Transversal Equilibrium Equation Solution

The value of i should be selected based on the desired boundary condition for solving the transverse equilibrium equation. For example, setting i=1 implies the use of boundary condition bcs[1], which corresponds to the boundary condition Γ_{S^1} .

```
i = 1;
Equilibrium solution
 h_{S^1} = f_{S^1} /. ParametricNDSolve
       \{\text{EqDiff}[\sigma, \alpha] = 0, \text{bcs}[i]\}, f_{s^1}, \{x_1, x_2\} \in \Omega, \{\sigma, \alpha\},
       Method → {"FiniteElement",
          "MeshOptions" → { "MeshOrder" → 2, MaxCellMeasure → size},
          "IntegrationOrder" → 4,
          "InterpolationOrder" \rightarrow \{f_{c1} \rightarrow 2\}\}];
```

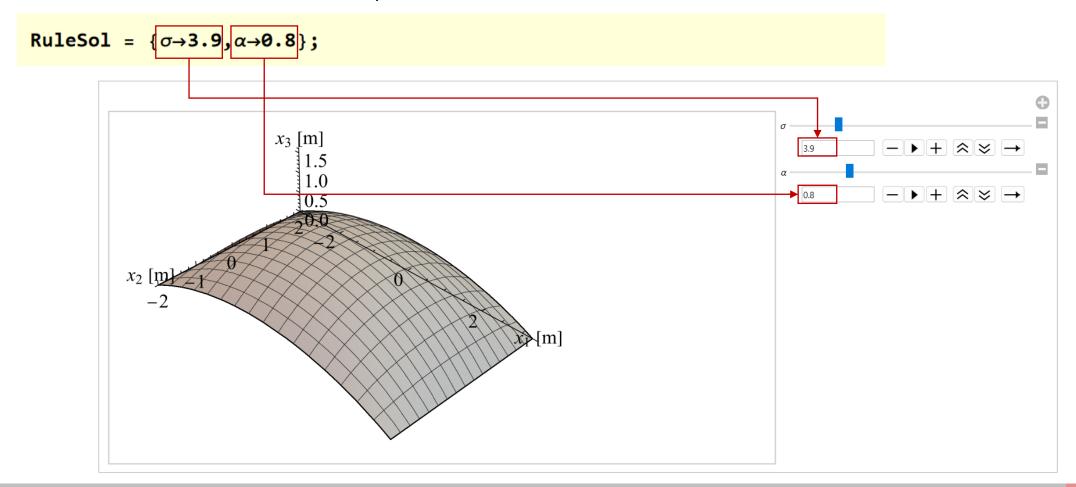
Reference surface selection

Through the Manipulate function, it is then possible to see how as the value of the parameters changes, the shape of the membrane changes, thus choosing the one that best reflects the user's architectural needs. The value of the parameters chosen will then go on to define what is the reference surface for constrained form-finding



Reference surface selection

Assign the values of σ and α that visualize on the right panel of the previous output once the shape that meets the user's architectural requirements are chosen.

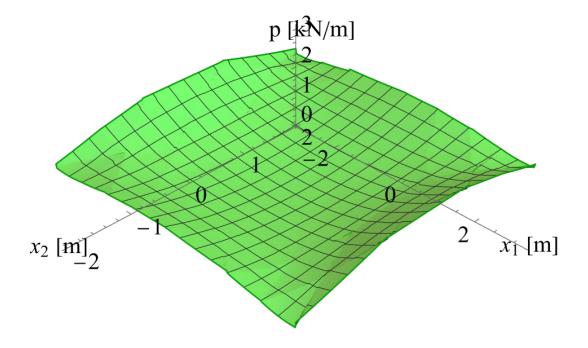


Constrained Form-Finding

The module calculates the load associated with the reference surface and incorporates it into the parametric solution of the second-order partial differential equation using the same FEM routine as demonstrated previously. The primary objective of this module is to minimize the mean distance between the shell and the reference surface by selecting appropriate parameters through the minimization of an objective function.

Real Load Evaluation

With this sub-module, you can easily calculate and visualize the load p on the designated reference surface. The sub-module achieves this by computing the prime derivatives of the shape, which are then used to establish the natural bases ($a_1 = \hat{e}_1 f_{,1}$ and $a_1 = \hat{e}_2 f_{,2}$). Subsequently, the Jacobian is defined as $J = |a_1 \times a_2|$. By leveraging the Jacobian, the load p can be estimated as the product of the uniformely distributed load p and the Jacobian p.



The second-order partial differential equation $F_{,22}$ $f_{,11} + F_{,11}$ $f_{,22} - 2$ $F_{,12}$ $f_{,12} - p = 0$ is set up in this module and solved parametrically with the chose Dirichlet boundary conditions dbs through a FEM routine as previously shown. The same bcs assigned above is used to solve the pde.

Equilibrium solution

```
\begin{split} & h_{\mathcal{S}^2} = f_{\mathcal{S}^2} \ / . \ \text{ParametricNDSolve} \big[ \\ & \left\{ \text{EqDiff2} \big[ \sigma, \, \alpha \big] = \emptyset, \, \text{bcs2} \big[ i \big] \right\}, \, f_{\mathcal{S}^2}, \, \left\{ \text{x1, x2} \right\} \in \Omega, \, \left\{ \sigma, \, \alpha \right\}, \\ & \text{Method} \rightarrow \left\{ \text{"FiniteElement"}, \right. \\ & \text{"MeshOptions"} \rightarrow \left\{ \text{"MeshOrder"} \rightarrow 2, \, \text{MaxCellMeasure} \rightarrow \text{size} \right\}, \\ & \text{"IntegrationOrder"} \rightarrow 4, \\ & \text{"InterpolationOrder"} \rightarrow \left\{ f_{\mathcal{S}^2} \rightarrow 2 \right\} \big\} \big]; \end{split}
```

To define the mean square deviation between the reference shape h and the optimized shape f_m , we use the objective function, Obj Fun, which depends on two numerical variables: σ and α .

```
ObjFun[\sigma_?NumericQ, \alpha_?NumericQ] := Module[\{h, \omega\},
h = h_{s^2}[\sigma, \alpha][x1, x2];
\omega = \Omega;
NIntegrate[((h/fr) - 1)^2, \{x1, x2\} \in \omega]
] // Simplify // Chop // N
```

Finally, the function ObjFun is minimized over the specified range of values for σ and α , with a working precision of two decimal places. It is suggested to assign solution range values of σ and α close to those assigned in RuleSol. The result of tThe robustness and efficiency of the FEM approach can be

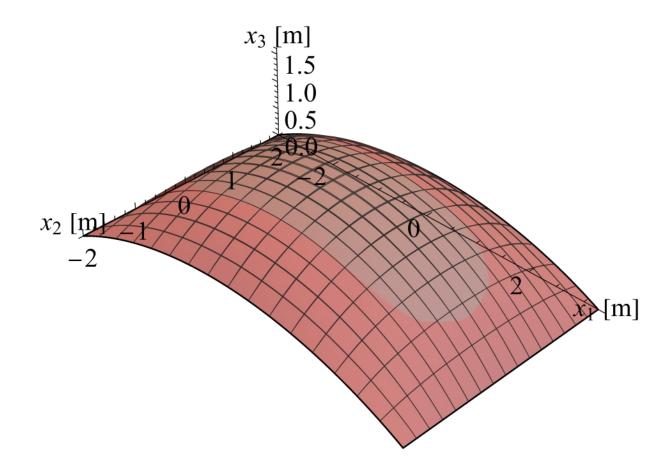
```
sol = Assuming [\alpha > 0, NMinimize [ ObjFun [\sigma, \alpha], \{\{\sigma, 5.2, 5.3\}, \{\alpha, 0.5, 0.6\}\}, WorkingPrecision \rightarrow 2]] // Quiet;
```

The robustness and efficiency of the FEM approach can be measured by the error norm, which is the average distance between the approximate and exact solutions. If the error norm is below a predetermined tolerance, such as 5%, it indicates a highly accurate, robust, and efficient solution. This value can be read from the output of the function DistPerc.

```
DistPerc = sol[[1]] 100 "%"
```

0.17 %

Is then possible to plot the form-finded final surface f_m (red) compared with the reference one (grey).

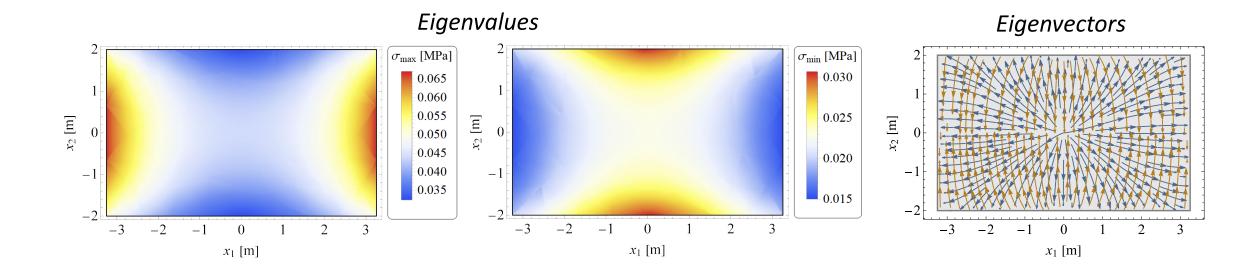


Stresses evaluation and outputs

Stresses evaluation and outputs
The last module of this code allows us to calculate the tensor of the stresses acting on the membrane and to evaluate and plot its eigenvalues and eigenvectors. This part provides fundamental data for the final design of the structure.

Stress tensor, eigenvalues, and eigenvectors

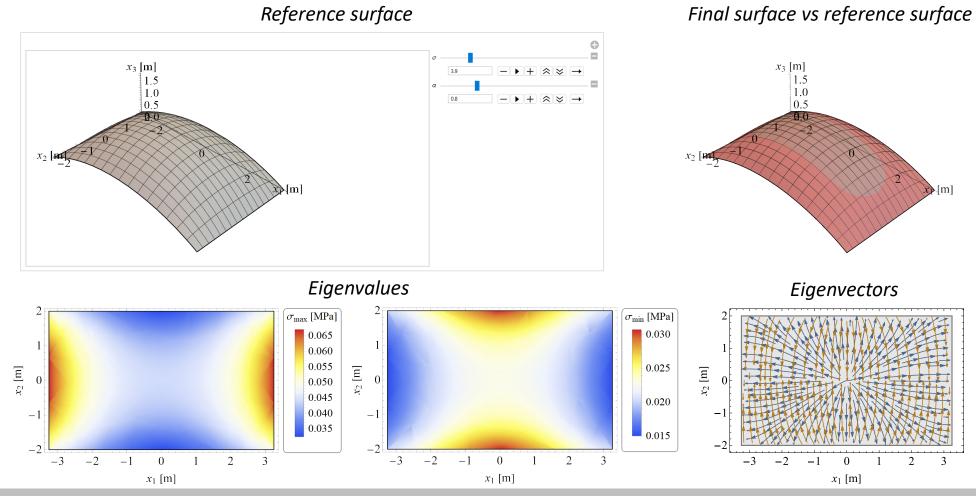
After obtaining the final surface, it is possible to evaluate its stress tensor and generate DensityPlot and StreamPlot graphs of the surface's eigenvalues and eigenvectors, respectively.



Examples

Example 1

Data: b=4 m, l=6.5 m, h=0.8 m, t=0.12 m, $\rho_a=18$ kN/m³, size=0.5, $\mathcal{A}[2]$, Γ_{S^1} , $\sigma=3.9$, $\alpha=0.8$



Example 2

Data: $b=10 \text{ m}, l=10 \text{ m}, h=0.0 \text{ m}, t=0.20 \text{ m}, \rho_a=24 \text{ kN/m}^3, \text{ size}=0.4, \mathcal{A}[4], \Gamma_{S^0}, \sigma=0.09, \alpha=1.0$

