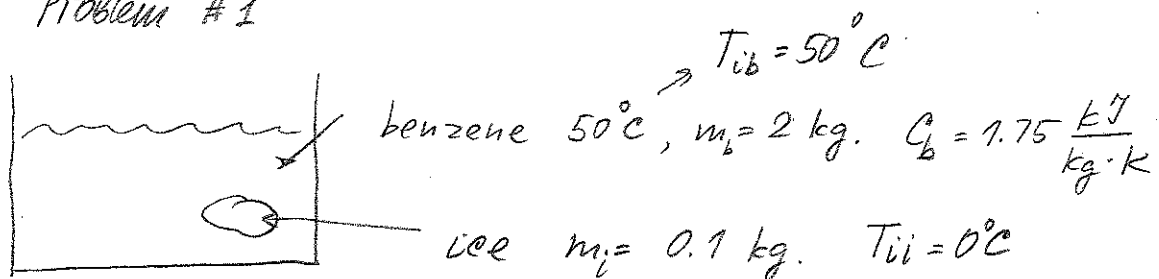


## Homework #5

3760

## Problem #1



- 1) benzene and water do not mix  $\rightarrow$  there is no contribution to entropy from mixing.
- 2) Final temperature of the solution.

$$L \cdot m_i + C_w m_i (T_f - T_{ii}) + C_b m_b (T_f - T_{ib}) = 0$$

(heat from benzene is transferred to ice and water)

$$T_f = \frac{L \cdot m_i - C_w m_i T_{ii} - C_b m_b T_{ib}}{-(C_w m_i + C_b m_b)} =$$

$$= \frac{333 \cdot 0.1 - 4.2 \cdot 0.1 \cdot 273 - 1.75 \cdot 2 \cdot (50 + 273)}{-(4.2 \cdot 0.1 + 1.75 \cdot 2)} \approx 309 \text{ K}$$

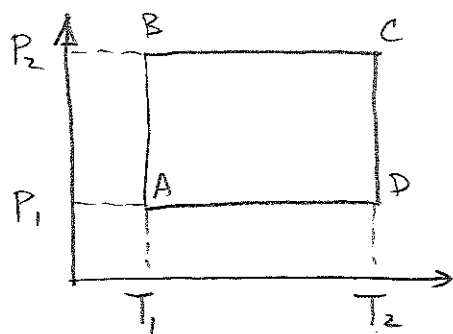
$$dS = \frac{dQ}{T}$$

$$\Delta S = \frac{L m_i}{273 \text{ K}} + \int_{T_{ii}}^{T_f} C_w m_i \cdot \frac{dT}{T} + \int_{T_{ib}}^{T_f} C_b m_b \cdot \frac{dT}{T} =$$

$$= \frac{333 \cdot 0.1}{273} + 4.2 \cdot 0.1 \cdot \ln\left(\frac{309}{273}\right) + 1.75 \cdot 2 \cdot \ln\left(\frac{309}{323}\right) =$$

$$= 0.12 + 0.05 - 0.155 \approx 0.015 \text{ J/K}$$

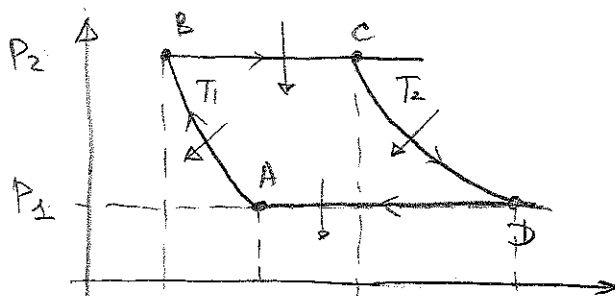
## Problem #2



$$\eta = \frac{W}{Q_{in}}$$

$$W = \int P dV$$

We need to draw our cycle in P-V coordinates.



$$PV = nRT$$

$$V = \frac{nRT}{P}$$

$$V_C = \frac{nRT_2}{P_2}$$

$$V_D = \frac{nRT_2}{P_1}$$

Heat goes into the system at B-C and C-D parts of the cycle.

$$dQ = dU - PdV$$

heat  $\Delta Q_{BC} = \frac{3}{2} nR \cdot (T_2 - T_1) - \int_{V_C}^{V_B} P dV = \frac{3}{2} nR (T_2 - T_1) - (P_2 V_C - P_2 V_B) =$

$$= \frac{3}{2} nR (T_2 - T_1) - nR (T_2 - T_1)$$

$$\Delta Q_{CD} = \int_{V_C}^{V_D} P dV = nRT_2 \cdot \int_{V_C}^{V_D} \frac{dV}{V} = nRT_2 \ln\left(\frac{V_D}{V_C}\right) = nRT_2 \ln\left(\frac{P_2}{P_1}\right)$$

WORK.  $W = \oint P dV = \oint V dp$  - we may choose any way to compute the area of the cycle.

$$W = \int_{CD} V dp + \int_{AB} V dp = \int_{P_2}^{P_1} \frac{nRT_2}{P} dp + \int_{P_2}^{P_1} \frac{nRT_1}{P} dp =$$

$$= nR (T_2 - T_1) \ln\left(\frac{P_2}{P_1}\right)$$

$$\eta = \frac{(T_2 - T_1) \ln\left(\frac{P_2}{P_1}\right)}{T_2 \ln\left(\frac{P_2}{P_1}\right) + \frac{5}{2} (T_2 - T_1)}$$

# Homework #5

## Problem #3

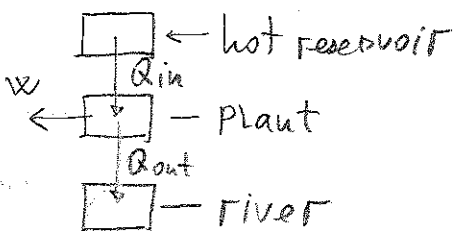
$g(u) = C u^{3N/2}$  Show that  $U = \frac{3}{2} k_B N T$

$$S_B = \frac{1}{B} \ln g = k_B \ln C u^{3N/2} = k_B \ln C + \frac{3N}{2} k_B \ln u$$

$$\frac{1}{T} = \left( \frac{\partial S}{\partial U} \right) = \frac{3N}{2} k_B \cdot \frac{1}{U} \quad U = \frac{3}{2} k_B N T$$

$$\frac{\partial^2 S}{\partial U^2} = \frac{3N}{2} k_B \left( -\frac{1}{U^2} \right) < 0.$$

## Problem #4.



$$\eta = 1 - \frac{T_c}{T_H} = \frac{W}{Q_{in}} = \frac{W}{W + Q_{out}}$$

We may introduce power as  $A = \frac{W}{t}$  and heat flow in and out as

$$F_{in} = \frac{Q_{in}}{t} \quad F_{out} = \frac{Q_{out}}{t}, \text{ where}$$

$t$  is time of plant operation.

We may then re-write the plant efficiency as

$$\eta = \left( 1 - \frac{T_c}{T_H} \right) = \frac{A}{A + F_{out}}$$

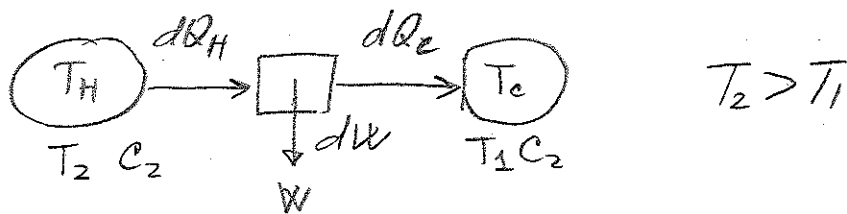
$$\frac{T_H}{T_H - T_c} = \frac{A + F_{out}}{A} = 1 + \frac{F_{out}}{A}$$

$$A = F_{out} \times \left( \frac{T_H}{T_H - T_c} - 1 \right) = F_{out} \cdot \left( \frac{T_H - T_c}{T_c} \right) =$$

$$A = 1500 \text{ MW} \cdot \left( \frac{273+500 - (273+20)}{273+20} \right) = 2457 \text{ kW}.$$

$$T_H = 600^\circ \text{C} \quad A = 2970 \text{ kW}.$$

Problem #5 extra credit.



$T_1$  is initial temperature of the hot body. It does change it time.  $T_c$  also changes in time and its initial temperature is  $T_1$ .

The maximum work will be done if at any stage the engine operates as an ideal heat engine that it does not produce any entropy.

$$\frac{dQ_H}{T_H} + \frac{dQ_C}{T_C} = 0 \Rightarrow \frac{C_1 dT_H}{T_H} + \frac{C_2 dT_C}{T_C} = 0 \quad (1)$$

The engine stops to operate when  $T_H = T_C = T_f$ .

$$C_1 \ln\left(\frac{T_f}{T_1}\right) + C_2 \ln\left(\frac{T_f}{T_2}\right) = 0 \quad \ln\left[\left(\frac{T_f}{T_1}\right)^{C_1}\right] = \ln\left[\left(\frac{T_2}{T_f}\right)^{C_2}\right]$$

$$\left(\frac{T_f}{T_1}\right)^{C_1} = \left(\frac{T_2}{T_f}\right)^{C_2} \quad T_f = \left(T_2^{C_2} T_1^{C_1}\right)^{\frac{1}{C_1+C_2}} \quad (2)$$

eq. (1) also allows us to find relation between  $T_H$  and  $T_C$

$$\left(\frac{T_C}{T_1}\right)^{C_1} = \left(\frac{T_2}{T_H}\right)^{C_2} \quad T_C = T_H^{C_1/C_2} \cdot T_2^{C_2/C_1} \cdot T_1$$

$$dW = -\left(1 - \frac{T_C}{T_H}\right) \cdot C_2 \cdot dT_H \quad (- \text{ is because } dT_H \text{ is negative})$$

$$W = \int_{T_2}^{T_f} \left(\frac{T_C}{T_H} - 1\right) \cdot C_2 dT_H = \int_{T_2}^{T_f} \frac{C_2}{T_2^{C_2/C_1} T_1} \cdot T_H^{C_1/C_2 - 1} \cdot C_2 dT_H -$$

$$- \int_{T_2}^{T_f} C_2 dT_H = \frac{C_2}{T_2^{C_2/C_1} T_1} \cdot \frac{C_2}{C_1} \cdot \left(T_f^{C_1/C_2} - T_2^{C_1/C_2}\right) +$$

$$+ C_2 \cdot (T_2 - T_f)$$