

Tutorial 3 solution

- ▷ In the system shown (Figure 1), disk A is free to rotate about the horizontal rod OA. Assuming that shaft OC and disk B rotate with constant angular velocities ω_1 and ω_2 , respectively, both counterclockwise. Determine:

- (a) The angular velocity of disk A w.r.t ground
- (b) The angular acceleration of disk A w.r.t ground

Notice this is a Category II problem where angular velocity & acceleration of the disk A is unknown and we will find the velocity of point of contact D from two sides to arrive at required equations.

① Identify the ref. frames

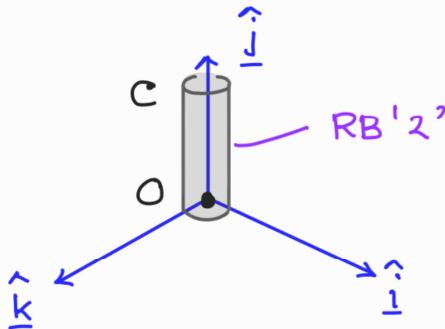
'F' → ground

'1' → disk B

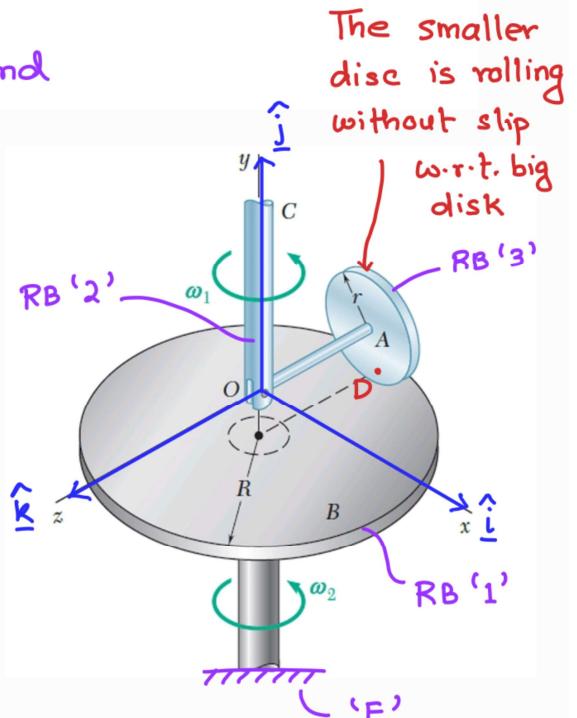
'2' → shaft OCA

'3' → disk A

② Select a coordinate sys so that we can express our answers in terms of components



This csys is fixed to RB'2'



③ Use "rolling without slip" condition at point of contact D from two sides

From side COA (I)

$$\begin{aligned} \underline{\nu}_{D/F} &= \underline{\nu}_{D/3} + \underline{\nu}_{A/F} \\ &\quad + \omega_{3/F} ? \times \underline{\tau}_{DA} \end{aligned}$$

$$\underline{\tau}_{DA} = - r \underline{j}$$

$$\begin{aligned} \omega_{3/F} &= \omega_{3/2} + \omega_{2/F} \\ &= \omega_k + \omega_1 j \end{aligned}$$

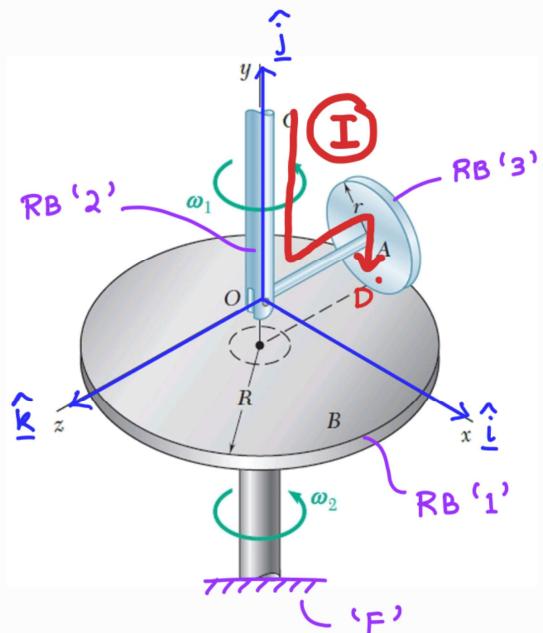
unknown scalar

You can think of point A as lying at the end of shaft OCA:

$$\begin{aligned}\underline{\nu}_{AIF} &= \underline{\nu}_{A1z} + \underline{\nu}_{O1F} + \underline{\omega}_{21F} \times \underline{\tau}_{AO} \\ &= \underline{\omega} + \underline{\omega} + \omega_1 \hat{j} \times (-R \hat{k}) \\ &= -\omega R \hat{i}\end{aligned}$$

$$\underline{\nu}_{DIF} = \underline{\nu}_{AIF} + \underline{\omega}_{31F} \times \underline{\tau}_{DA}$$

$$\begin{aligned}&= -\omega_1 R \hat{i} + (\omega \hat{k} + \omega_1 \hat{j}) \times (-r \hat{j}) \\ &= -\omega_1 R \hat{i} + \omega r \hat{i} \\ &= (-\omega_1 R + \omega r) \hat{i}\end{aligned}$$



From side (I)

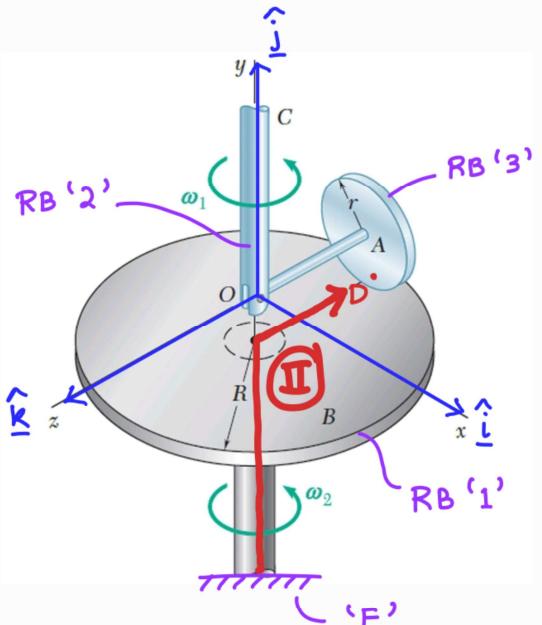
$$\begin{aligned}\underline{\nu}_{DIF} &= \underline{\nu}_{D1z} + \underline{\nu}_{O1F} + \underline{\omega}_{11F} \times \underline{\tau}_{DO} \\ &= \omega_2 \hat{j} \times (-R \hat{k}) \\ &= -\omega_2 R \hat{i}\end{aligned}$$

Matching $\underline{\nu}_{DIF}$ from RB'(3) and that from RB'(1), we get:

$$\underline{\nu}_{DIF}^{(I)} = \underline{\nu}_{DIF}^{(II)}$$

$$\Rightarrow (-\omega_1 R + \omega r) \hat{i} = -\omega_2 R \hat{i}$$

$$\Rightarrow \omega = (\omega_1 - \omega_2) \frac{R}{r}$$



\therefore Angular velocity of disk A rel. to ground:

$$\underline{\omega}_{31F} = \omega_1 \hat{j} + (\omega_1 - \omega_2) \frac{R}{r} \hat{k}$$

We already know: $\underline{\omega}_{3|F} = \underline{\omega}_{3|2} + \underline{\omega}_{2|F}$

We can take time-derivative directly w.r.t F

$$\begin{aligned}\dot{\underline{\omega}}_{3|F} &= \frac{d}{dt} \left(\underline{\omega}_{3|2} + \underline{\omega}_{2|F} \right) \Big|_F \\ &= \dot{\underline{\omega}}_{3|2} \Big|_F + \dot{\underline{\omega}}_{2|F} \Big|_F \\ &= \dot{\underline{\omega}}_{3|2} \Big|_2 + \underline{\omega}_{2|F} \times \underline{\omega}_{3|2} + \dot{\underline{\omega}}_{2|F} \\ &= \dot{\underline{\omega}}_{3|2} + \underline{\omega}_{2|F} \times \underline{\omega}_{3|2} + \dot{\underline{\omega}}_{2|F}\end{aligned}$$

this represents

the angular acc. of

RB '3' as seen by

an observer attached

to RB '1' (and is 0)

is given as

zero

$$\dot{\underline{\omega}}_{2|F} = 0$$

$$\dot{\underline{\omega}}_{3|2} = 0$$

$$\therefore \dot{\underline{\omega}}_{3|F} = \underline{\omega}_{2|F} \times \underline{\omega}_{3|2}$$

$$= \omega_1 \hat{j} \times (\omega_1 - \omega_2) \frac{R}{r} \hat{k}$$

$$= \omega_1 (\omega_1 - \omega_2) \frac{R}{r} \hat{i}$$

- 2) A wheel rolls without slipping on a fixed cylinder. Knowing that at the instant shown (Figure 2) the angular velocity of the wheel is 10 rad/s clockwise and its angular acceleration is 30 rad/s² counterclockwise, determine the acceleration of:

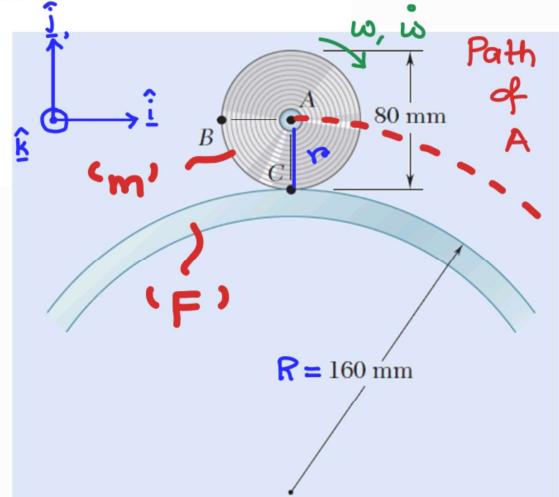
(a) Point A, (assumed +)

(b) Point B,

(c) Point C.

$$r = 40 \text{ mm} \quad \omega_m|F = -\omega_0 \hat{k} = -10 \hat{k}$$

$$R = 160 \text{ mm} \quad \dot{\omega}_m|F = \alpha_0 \hat{k} = 30 \hat{k}$$



(i) Note that the path of point A follows a circle of radius = $R + r$

Problem amenable to use of path csys.

This problem is best solved using path csys because :

- (a) the given data has info about the radii of curvature,
- (b) the path of point A is known.

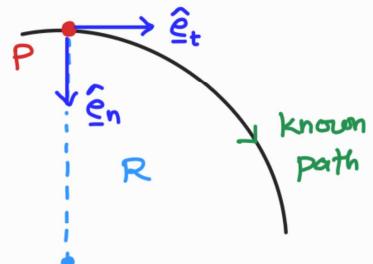
(ii) Rolling without slip $\Rightarrow v_{C/F} = 0$ [pt C on the wheel in contact has zero velocity relative to 'F', and instantaneous motion of the wheel is just a rotation around contact pt. C]

Acceleration of a point P relative to frame F in path csys

$$\ddot{a}_{P/F} = \ddot{s} \hat{e}_t + \frac{\dot{s}^2}{R} \hat{e}_n$$

Velocity
normal
acceleration

tangential
acceleration



$\dot{s} \rightarrow$ velocity of the point of interest

For velocity of pt A, we use the known velocity at C:

$$\begin{aligned}\underline{v}_{A/F} &= \cancel{\underline{v}_{C/F}} + \underline{\omega}_{m/F} \times \underline{r}_{AC} && \text{at that instant relative to 'F'} \\ &= \underline{\omega} + (-\omega_o) \hat{k} \times (-r \hat{j}) \\ &= -\omega_o r \hat{i} = \dot{s} \hat{e}_t\end{aligned}$$

Since pt A travels along a circle of radius $(R+r)$,

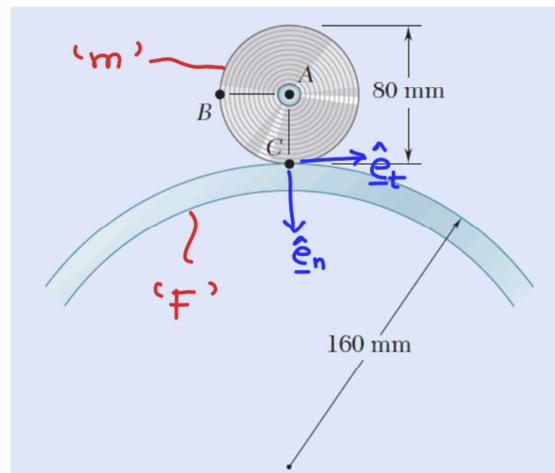
we may also write $\underline{v}_{A/F} = \dot{s} \hat{e}_t$ using path csys.

Note that at the given instant \hat{e}_t coincides with \hat{i}

$$\therefore \dot{s} = -\omega_o r$$

Acceleration of pt A,

$$\begin{aligned}\underline{a}_{A/F} &= \ddot{s} \hat{e}_t + \frac{\dot{s}^2}{R+r} \hat{e}_n \\ &= \ddot{s} \hat{e}_t + \frac{(\omega_o r)^2}{R+r} \hat{e}_n\end{aligned}$$



To find \ddot{s} , we can use the acceleration at C and use

the fact that the tangential component of acc. at C = 0

$$\Rightarrow \underline{a}_{C/F} \cdot \hat{e}_t = 0$$

$$\begin{aligned}
 \underline{\alpha}_{c/F} &= \cancel{\underline{\alpha}_{clm}}^{\textcircled{O}} + \underline{\alpha}_{A/F} ? + \dot{\underline{\omega}}_m/F \times \underline{r}_{CA} + \underline{\omega}_{m/F} \times (\underline{\omega}_{m/F} \times \underline{r}_{CA}) \\
 &= \ddot{s} \hat{\underline{e}}_t + \frac{(\omega_o r)^2}{R+r} \hat{\underline{e}}_n + \alpha_o \hat{\underline{e}}_b \times -r \hat{\underline{e}}_n \\
 &\quad + (-\omega_o \hat{\underline{e}}_b) \times [(-\omega_o \hat{\underline{e}}_b) \times (-r \hat{\underline{e}}_n)] \\
 &= \ddot{s} \hat{\underline{e}}_t + \frac{(\omega_o r)^2}{R+r} \hat{\underline{e}}_n + \alpha_o r \hat{\underline{e}}_t - \omega_o^2 r \hat{\underline{e}}_n
 \end{aligned}$$

$$\underline{\alpha}_{c/F} \cdot \hat{\underline{e}}_t = 0$$

$$\Rightarrow (\ddot{s} + \alpha_o r) = 0 \Rightarrow \ddot{s} = -\alpha_o r$$

$$\therefore \underline{\alpha}_{A/F} = -\alpha_o r \hat{\underline{e}}_t + \frac{(\omega_o r)^2}{R+r} \hat{\underline{e}}_n$$

\therefore Acceleration of point C :

$$\underline{\alpha}_{c/F} = \frac{(\omega_o r)^2}{R+r} \hat{\underline{e}}_n - \omega_o^2 r \hat{\underline{e}}_n \quad (\because \text{tangential comp. is zero})$$

$$= \frac{\omega_o^2 r^2 - \omega_o^2 r^2 - \omega_o^2 r R}{R+r} \hat{\underline{e}}_n$$

$$= -\frac{\omega_o^2}{\left(\frac{1}{R} + \frac{1}{r}\right)} \hat{\underline{e}}_n$$

Acceleration of point B:

$$v_{B/F} = \cancel{v_{B/m}}^{\alpha} + v_{A/F} + \dot{\omega}_{m/F} \times r_{BA} + \omega_{m/F} \times (\omega_{m/F} \times r_{BA})$$

$\rightarrow -r \hat{e}_t$

$$= -\alpha_r \hat{e}_t + \frac{(\omega_0 r)^2}{R+r} \hat{e}_n + \alpha_r \hat{e}_b \times -r \hat{e}_t$$

$$+ (-\omega_0 \hat{e}_b) \times [(-\omega_0 \hat{e}_b) \times (-r \hat{e}_r)]$$

$$= -\alpha_r \hat{e}_t + \frac{(\omega_0 r)^2}{R+r} \hat{e}_n - \alpha_r r \hat{e}_n + \omega_0^2 r \hat{e}_t$$

$$= (\omega_0^2 r - \alpha_r r) \hat{e}_t + \left[\frac{(\omega_0 r)^2}{R+r} - \alpha_r r \right] \hat{e}_n$$

Note: $\hat{e}_t \rightarrow \hat{i}$
 $\hat{e}_n \rightarrow -\hat{j}$

Plug in the numerical values, and you should get the answers provided.