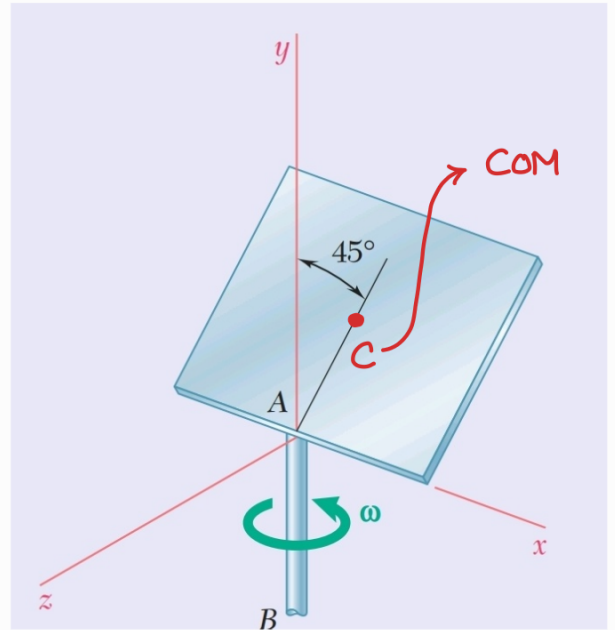


Part A solutions

1> Find kinetic energy of plate

Recall that computation of KE requires careful selection of a point on the body for which the calculation becomes easy.

Two choices for computing KE



$T|_F$ \rightarrow pt A has zero linear velocity, $\underline{v}_{A|F} = \underline{0}$

$$\Rightarrow \text{can use } T|_F = \frac{1}{2} \underline{\omega}_{m|F} \underline{I}^A \underline{\omega}_{m|F}$$

Calculation of \underline{I}^A needs the use of parallel axes thm to transfer from C to A

\rightarrow pt C \equiv COM

$$\Rightarrow \text{use } T|_F = \frac{1}{2} m \underline{v}_{C|F} \cdot \underline{v}_{C|F}$$

$$+ \frac{1}{2} \underline{\omega}_{m|F} \underline{I}^C \underline{\omega}_{m|F}$$

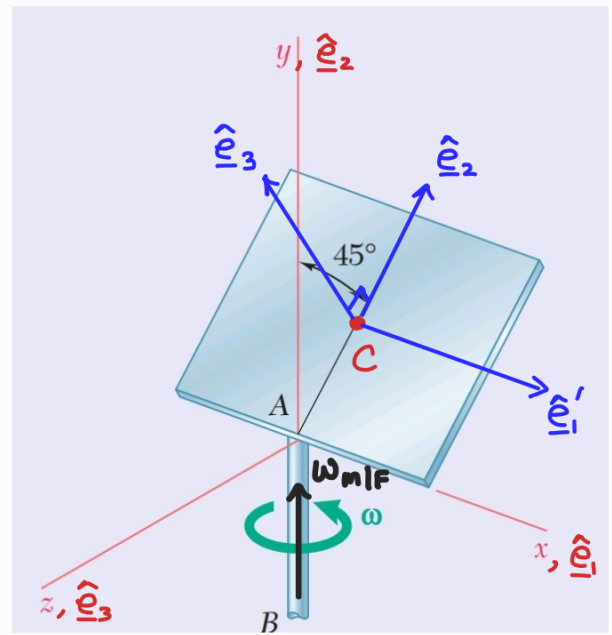
\underline{I}^C can be easily compute but $\underline{v}_{C|F}$ needs to be calculated as well

For the csys given in the figure,

$[\underline{I}^C] \begin{pmatrix} \hat{e}_1 \\ \hat{e}_2 \\ \hat{e}_3 \end{pmatrix}$ is not easily calculated.

as they don't coincide with the principal directions of the plate

$\hat{e}'_1 - \hat{e}'_2 - \hat{e}'_3$



T_{IF} computation using C as base point

in $\hat{e}_1 - \hat{e}_2 - \hat{e}_3$ csys

$$[\underline{\omega}_{m|F}] = \begin{bmatrix} 0 \\ \omega \\ 0 \end{bmatrix}$$

$$[\underline{I}^C] = \begin{bmatrix} ? & ? & ? \\ ? & ? & ? \\ ? & ? & ? \end{bmatrix}$$

Get $[\underline{I}^C] \begin{pmatrix} \hat{e}'_1 \\ \hat{e}'_2 \\ \hat{e}'_3 \end{pmatrix}$ in prin. csys $\hat{e}'_1 - \hat{e}'_2 - \hat{e}'_3$

and then use transformation rule

$$[\underline{I}^C] \begin{pmatrix} \hat{e}_1 \\ \hat{e}_2 \\ \hat{e}_3 \end{pmatrix} = [\underline{A}] [\underline{I}^C] \begin{pmatrix} \hat{e}'_1 \\ \hat{e}'_2 \\ \hat{e}'_3 \end{pmatrix} [\underline{A}]^T$$

need to calculate $[\underline{A}]$

in principal csys $\hat{e}'_1 - \hat{e}'_2 - \hat{e}'_3$

$$[\underline{I}^C] = \begin{bmatrix} \frac{ma^2}{12} & 0 & 0 \\ 0 & \frac{ma^2}{12} & 0 \\ 0 & 0 & \frac{ma^2}{6} \end{bmatrix}$$

$$[\underline{\omega}_{m|F}] = \begin{bmatrix} 0 \\ \omega \cos 45^\circ \\ \omega \sin 45^\circ \end{bmatrix}$$

Working in the principal csys at COM is easier!

Compute $\underline{v}_{C|F}$ using velocity transfer relations

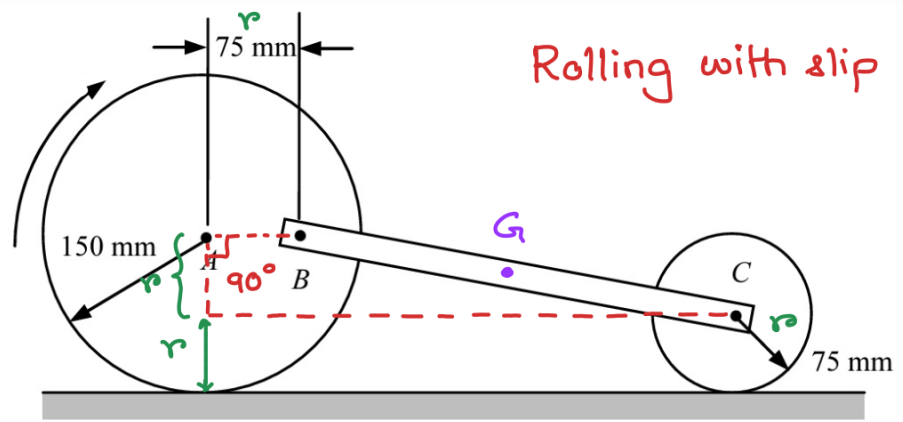
$$\begin{aligned}\underline{v}_{C|F} &= \cancel{\underline{v}_{A|F}}^0 + \underline{\omega}_{m|F} \times \underline{r}_{CA} \\ &= \left(\omega/\sqrt{2} \hat{e}'_2 + \omega/\sqrt{2} \hat{e}'_3 \right) \times a/2 \hat{e}'_2 \\ &= -a\omega/2\sqrt{2} \hat{e}'_1\end{aligned}$$

Finally compute KE using C as base pt (using $\hat{e}'_1 - \hat{e}'_2 - \hat{e}'_3$)

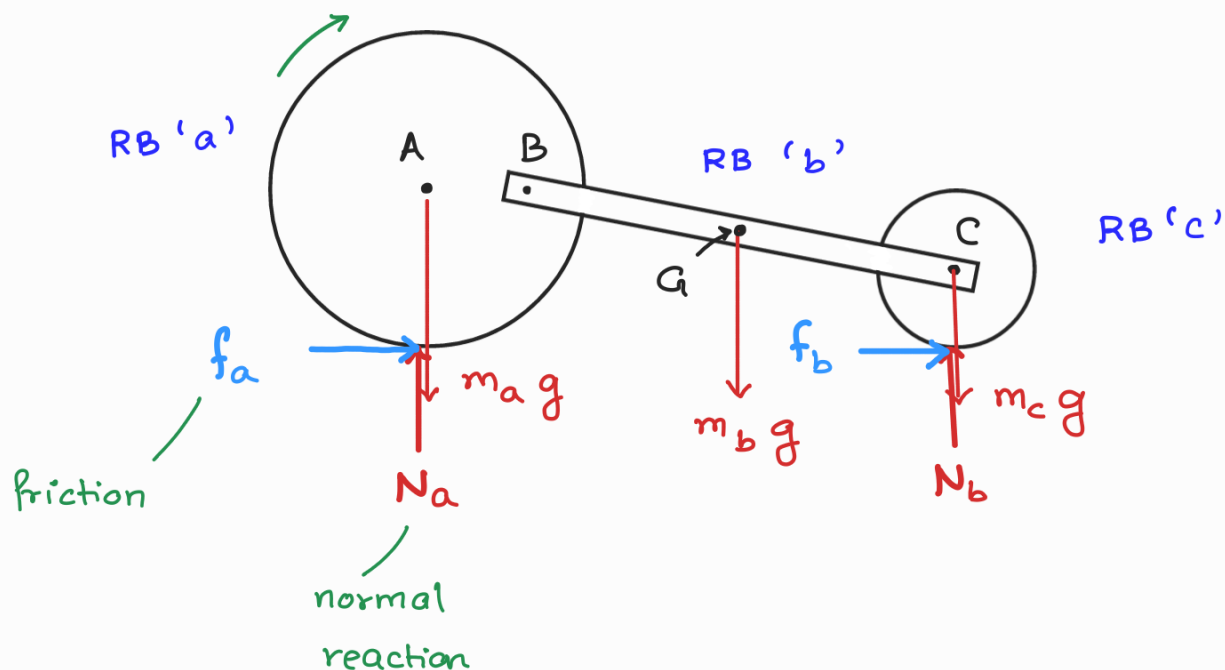
$$\begin{aligned}T|_F &= \frac{1}{2} m \underline{v}_{C|F} \cdot \underline{v}_{C|F} + \frac{1}{2} \underline{\omega}_{m|F} \cdot \underline{I}^C \underline{\omega}_{m|F} \\ &= \frac{1}{2} m [\underline{v}_{C|F}]^T [\underline{v}_{C|F}] + \frac{1}{2} [\underline{\omega}_{m|F}] [\underline{I}^C] [\underline{\omega}_{m|F}] \\ &= \frac{m}{2} \cdot \frac{a^2 \omega^2}{8} + \frac{1}{2} \begin{bmatrix} 0 \\ \omega/\sqrt{2} \\ \omega/\sqrt{2} \end{bmatrix}^T \begin{bmatrix} \frac{ma^2}{12} & 0 & 0 \\ 0 & \frac{ma^2}{12} & 0 \\ 0 & 0 & \frac{ma^2}{6} \end{bmatrix} \begin{bmatrix} 0 \\ \omega/\sqrt{2} \\ \omega/\sqrt{2} \end{bmatrix} \\ &= \frac{ma^2 \omega^2}{16} + \frac{1}{2} \left[\frac{\omega^2}{2} \cdot \frac{ma^2}{12} + \frac{\omega^2}{2} \cdot \frac{ma^2}{6} \right] \\ &= \frac{ma^2 \omega^2}{16} + \frac{3ma^2 \omega^2}{48} \\ &= \frac{ma^2 \omega^2}{8}\end{aligned}$$

$$T|_F = \frac{ma^2 \omega^2}{8}$$

2) Determine velocity of rod BC after disk has rotated by 90°



- Since there are **Two positions** of rod BC : initial-1 and final-2, and we are interested in knowing the velocity of rod, we use the work-energy principle.
- Consider system = rod + two disks, and draw FBD for the system. The work done by internal forces are equal and opposite and hence cancel out. We therefore consider only forces/moments external to the "system"

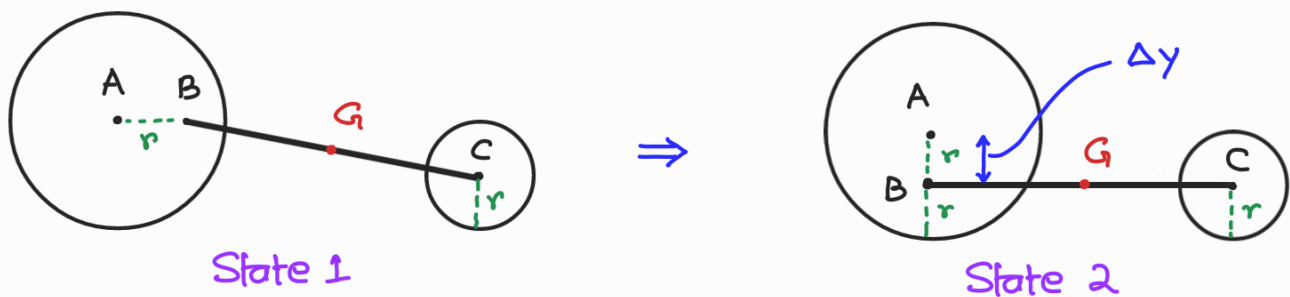


Work energy principle $\Rightarrow \underline{W_{1 \rightarrow 2}} = \Delta T = T_{2/F} - T_{1/F}$

Work done by ext. forces
in moving the body from state 1 \rightarrow 2

- 1> Frictional forces f_a and f_b do zero work as velocities at the no-slip contact points are zero
- 2> The normal reactions N_a and N_b are a part of reaction force system, and reaction force systems do not any work as they arise due to constraints in motion.
- 3> The forces that do non-zero work in this problem are the weights due to gravity (which are constant forces, hence also conservative forces): $m_a g$, $m_b g$, and $m_c g$

Work done by weights due to gravity



Work done = $m_b g \cdot \Delta y$ ← vertical disp of G going from state 1 to state 2

$W_{1 \rightarrow 2}$

= $m_b g \cdot r$

According to work-energy principle,

$$W_{1 \rightarrow 2} = \Delta T = \overset{\text{KE at state 2}}{T_2} - \overset{\text{KE at state 1}}{T_1}$$

$$\Rightarrow m_b g \cdot r = (T^a + T^b + T^c)_2 - \cancel{(T^a + T^b + T^c)_1}^0$$

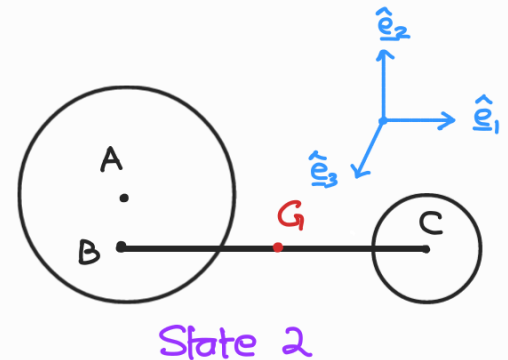
("system" was
at rest initially)

$$\Rightarrow (T^a + T^b + T^c)_2 = m_b g \cdot r - (**)$$

Kinetic energy of the system at state 2

$$(T^a)_2 = ? \quad (T^b)_2 = ? \quad (T^c)_2 = ?$$

Using $\hat{e}_1 - \hat{e}_2 - \hat{e}_3$ csys for calculation



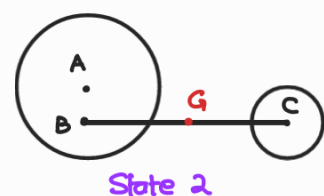
$$(T^a)_2 = \frac{1}{2} m_a [\underline{v}_{a|I}]^T [\underline{v}_{a|I}] + \frac{1}{2} [\underline{\omega}_{a|I}]^T [\underline{I}^a] [\underline{\omega}_{a|I}]$$

$$(T^b)_2 = \frac{1}{2} m_b [\underline{v}_{b|I}]^T [\underline{v}_{b|I}] + \frac{1}{2} [\underline{\omega}_{b|I}]^T [\underline{I}^b] [\underline{\omega}_{b|I}]$$

$$(T^c)_2 = \frac{1}{2} m [\underline{v}_{c|I}]^T [\underline{v}_{c|I}] + \frac{1}{2} [\underline{\omega}_{c|I}]^T [\underline{I}^c] [\underline{\omega}_{c|I}]$$

Use kinematics to relate $\underline{v}_{a|I}$, $\underline{v}_{b|I}$, and $\underline{v}_{c|I}$

In state 2, $\underline{v}_{a|I} = \underline{v}_{b|I} = \underline{v}_{c|I}$ (*)



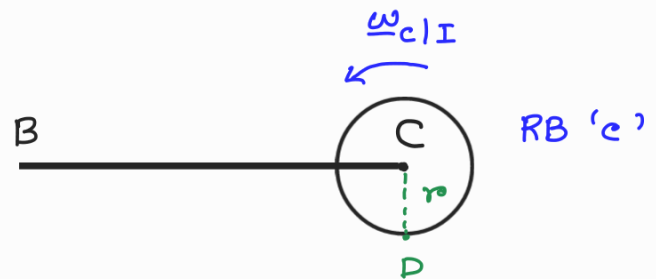
Therefore, at the instant of state 2, $\underline{\omega}_{b|I} = \underline{0}$

\Rightarrow every point on rod BC has identical velocity

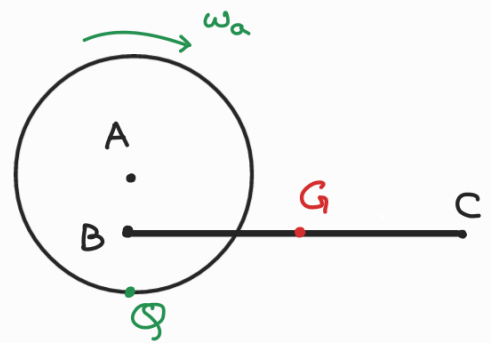
$$\underline{v}_{c|I} = \underline{v}_{o|I} + \underline{\omega}_{c|I} \times \underline{r}_{co}$$

$$\Rightarrow v_c \hat{e}_1 = \omega_c r \hat{e}_1$$

$$\Rightarrow v_c = \omega_c r$$



$$\begin{aligned} \underline{v}_{A|I} &= \underline{v}_{o|I} + \underline{\omega}_{a|I} \times \underline{r}_{Ao} \\ &= 2\omega_a r \hat{e}_1 \end{aligned}$$



$$\underline{v}_{o|I} = \underline{v}_{o|I} + \underline{\omega}_{a|I} \times \underline{r}_{Ao}$$

$$= \omega_a r \hat{e}_1 = \underline{v}_{G|I} = \underline{v}_{c|I} \Rightarrow \omega_a r \hat{e}_1 = \omega_c r \hat{e}_1$$

$$\Rightarrow \boxed{\omega_a = \omega_c}$$

$$(T_{II}^a)_2 = \frac{1}{2} m_a [\underline{v}_{A|I}]^T [\underline{v}_{A|I}] + \frac{1}{2} [\underline{\omega}_{a|I}]^T [\underline{I}^A] [\underline{\omega}_{a|I}]$$

$$= \frac{1}{2} m_a \begin{bmatrix} 2r\omega_a \\ 0 \\ 0 \end{bmatrix}^T \begin{bmatrix} 2r\omega_a \\ 0 \\ 0 \end{bmatrix} + \frac{1}{2} \begin{bmatrix} 0 \\ 0 \\ -\omega_a \end{bmatrix}^T \begin{bmatrix} \checkmark & 0 & 0 \\ 0 & \checkmark & 0 \\ 0 & 0 & \frac{m_a R^2}{2} \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ -\omega_a \end{bmatrix}$$

$$= \frac{1}{2} m_a \cdot (2r\omega_a)^2 + \frac{1}{2} (\omega_a)^2 \cdot \left(\frac{m_a (2r)^2}{2} \right) \quad [R = 2r]$$

$$= \frac{m_a 4r^2 \omega_a^2}{2} + \frac{m_a 4r^2 \omega_a^2}{4}$$

$$= 3 m_a r^2 \omega_a^2$$

$$(T_{II}^b)_2 = \frac{1}{2} m_b [\underline{v}_{G|I}]^T [\underline{v}_{G|I}] + \frac{1}{2} [\underline{\omega}_{b|I}]^T [\underline{I}^G] [\underline{\omega}_{b|I}]$$

$$= \frac{1}{2} m_b \begin{bmatrix} r\omega_a \\ 0 \\ 0 \end{bmatrix}^T \begin{bmatrix} r\omega_a \\ 0 \\ 0 \end{bmatrix} + \frac{1}{2} \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}^T \begin{bmatrix} \checkmark & 0 & 0 \\ 0 & \checkmark & 0 \\ 0 & 0 & \checkmark \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ 0 \end{bmatrix}$$

$$= \frac{1}{2} m_b r^2 \omega_a^2$$

$$(T_{II}^c)_2 = \frac{1}{2} m_c [\underline{v}_{c|I}]^T [\underline{v}_{c|I}] + \frac{1}{2} [\underline{\omega}_{c|I}]^T [\underline{I}] [\underline{\omega}_{c|I}]$$

$$= \frac{1}{2} m_c \begin{bmatrix} r\omega_a \\ 0 \\ 0 \end{bmatrix}^T \begin{bmatrix} r\omega_a \\ 0 \\ 0 \end{bmatrix} + \frac{1}{2} \begin{bmatrix} 0 \\ 0 \\ \omega_c \end{bmatrix}^T \begin{bmatrix} \checkmark & 0 & 0 \\ 0 & \checkmark & 0 \\ 0 & 0 & \frac{m_c r^2}{2} \end{bmatrix} \begin{bmatrix} 0 \\ 0 \\ \omega_c \end{bmatrix}$$

$$= \frac{1}{2} m_c r^2 \omega_a^2 + \frac{1}{2} \omega_a^2 \frac{m_c r^2}{2} \quad [\omega_c = \omega_a]$$

$$= \frac{3}{4} m_c r^2 \omega_a^2$$

Adding $(T_{II}^a)_2$, $(T_{II}^b)_2$, and $(T_{II}^c)_2$

$$(T_{1I}^a + T_{1I}^b + T_{1I}^c)_2 = 3 \overset{6}{m_a} \overset{6}{r^2 \omega_a^2} + \frac{1}{2} \overset{5}{m_b} \overset{5}{r^2 \omega_a^2} + \frac{3}{4} \overset{1.5}{m_c} \overset{1.5}{r^2 \omega_a^2}$$

[$v_B = \omega_a r$]

$$= 18 v_B^2 + \frac{5}{2} v_B^2 + \frac{9}{8} v_B^2$$

$$= 21.6 v_B^2$$

Using work-energy relation **(**)**

$$21.6 v_B^2 = \overset{5}{m_b} \overset{0.075}{g} r$$

$$g = 9.81$$

$$\Rightarrow v_B = 0.412 \text{ m/s}$$