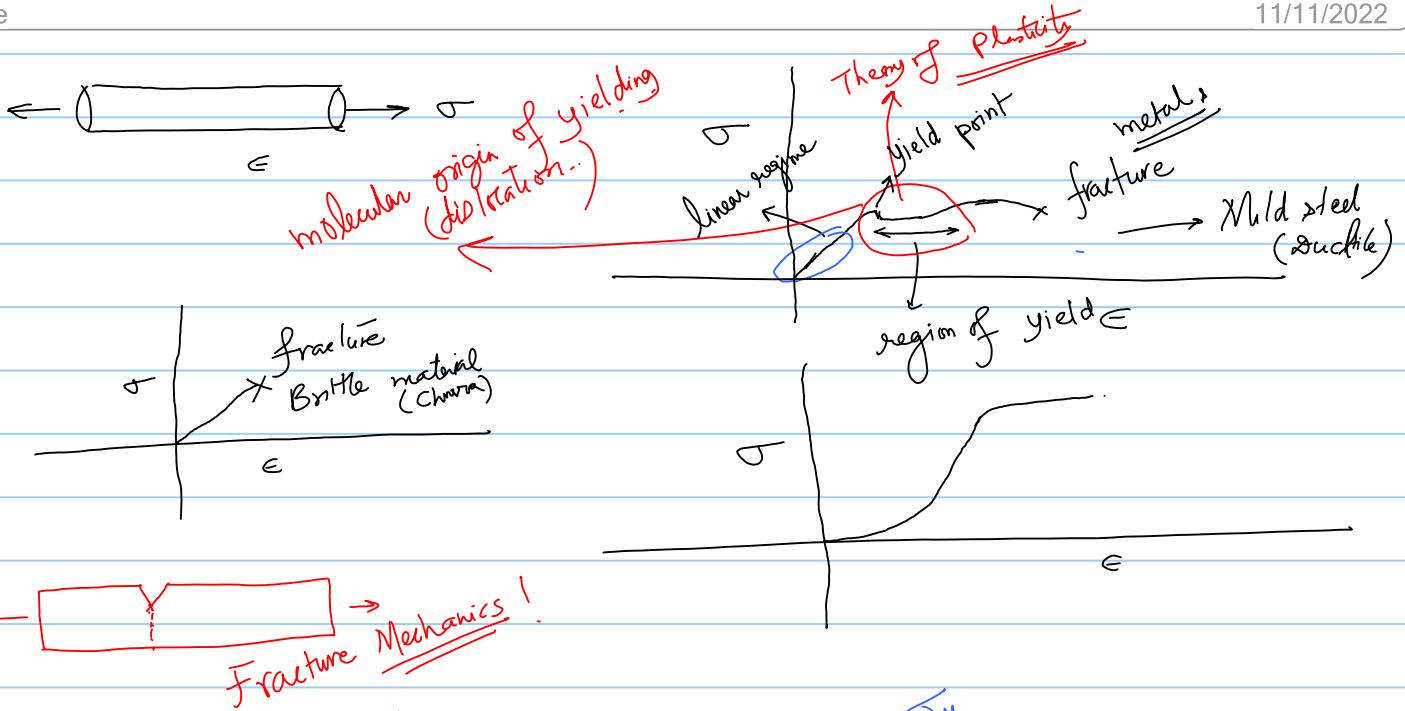


Lec 37 (Theories of Failure)

Note Title

11/11/2022



- 1) Maximum principal stress theory σ_y
- 2) Maximum shear stress theory: τ_y
- 3) Maximum normal strain theory ϵ_y
- 4) Maximum shear strain theory γ_y
- 5) Distortional energy theory
- 6) Octahedral shear stress theory

Diagram of a cylinder under stress σ_{11} and strain ϵ_1 .

$$[\underline{\underline{\sigma}}] = \begin{bmatrix} \sigma_{11} & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \end{bmatrix}, \quad [\underline{\underline{\epsilon}}] = \begin{bmatrix} \epsilon_{11} & 0 & 0 \\ 0 & \epsilon_{22} & 0 \\ 0 & 0 & \epsilon_{33} \end{bmatrix}$$

- 1) σ_{11}^* : the load value at which the body failed
 $\sigma_{11}^* = \sigma_y$ (yield stress!)
 $\max_x \lambda_1(x) \leq \sigma_y$
 $[\underline{\underline{\sigma}}] = \begin{bmatrix} x & x & x \\ x & x & x \\ x & x & x \end{bmatrix}$
- 2) $\frac{\sigma_{11}^*}{2} = \tau_y$
 $\max_x \frac{\lambda_1(x) - \lambda_3(x)}{2} \leq \tau_y$
- 5) Distortional energy theory:-
 $E = \sum_{i,j} \frac{1}{2} \sigma_{ij} \epsilon_{ij}$

In the coordinate system of principal direction

$$[\underline{\sigma}] = \begin{bmatrix} \sigma_1 & 0 & 0 \\ 0 & \sigma_2 & 0 \\ 0 & 0 & \sigma_3 \end{bmatrix}, \quad [\underline{\epsilon}] = \begin{bmatrix} \epsilon_1 & 0 & 0 \\ 0 & \epsilon_2 & 0 \\ 0 & 0 & \epsilon_3 \end{bmatrix}$$

$$E = \frac{1}{2} \sum_i \sigma_i \epsilon_i \quad \checkmark \epsilon_1 = \frac{1}{E} [\sigma_1 - \nu(\sigma_2 + \sigma_3)]$$

$$= \frac{1}{2E} \left[\sigma_1 (\sigma_1 - \nu(\sigma_2 + \sigma_3)) + \sigma_2 (\sigma_2 - \nu(\sigma_1 + \sigma_3)) + \sigma_3 (\sigma_3 - \nu(\sigma_1 + \sigma_2)) \right]$$

$$= \frac{1}{2E} \left[\sigma_1^2 + \sigma_2^2 + \sigma_3^2 - 2\nu(\sigma_1\sigma_2 + \sigma_2\sigma_3 + \sigma_3\sigma_1) \right]$$

Vol. strain energy density = $\frac{1}{2} \underline{\sigma}^T \underline{\epsilon}$

$$= \frac{1}{2} \left(\frac{\sigma_1 + \sigma_2 + \sigma_3}{3} \right) (\epsilon_1 + \epsilon_2 + \epsilon_3)$$

$$= \frac{(1-2\nu)}{6E} (\sigma_1 + \sigma_2 + \sigma_3)^2$$

$$\epsilon_1 = \frac{1}{E} [\sigma_1 - \nu(\sigma_2 + \sigma_3)]$$

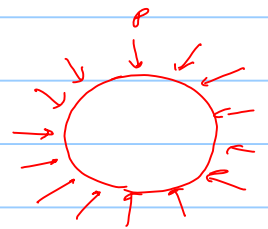
ϵ_2
 ϵ_3

$$\Rightarrow \epsilon_1 + \epsilon_2 + \epsilon_3 = \frac{1}{E} \left[(\sigma_1 + \sigma_2 + \sigma_3) - 2\nu(\sigma_1 + \sigma_2 + \sigma_3) \right]$$

$$= \frac{1-2\nu}{E} (\sigma_1 + \sigma_2 + \sigma_3)$$

$$\underline{\sigma} = p \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$\underline{\epsilon} = \begin{bmatrix} \epsilon & & \\ & \epsilon & \\ & & \epsilon \end{bmatrix}$$



$\underline{\sigma} = -p \underline{I}$
→ only volume change is happening without any distortion (shear is absent..)

$$\underline{\sigma}^T \underline{\epsilon} = \frac{1}{2} \sigma_{ij} \epsilon_{ij} = \frac{1}{2} p \cdot 3\epsilon$$

$$= \frac{1}{2} p \epsilon_{vol}$$

Distortional energy density = $\frac{1}{2E} \left[\sigma_1^2 + \sigma_2^2 + \sigma_3^2 - 2\nu(\sigma_1\sigma_2 + \sigma_2\sigma_3 + \sigma_3\sigma_1) - \frac{(1-2\nu)}{3} (\sigma_1 + \sigma_2 + \sigma_3)^2 \right]$

$$= \frac{1+\nu}{3E} (\sigma_1^2 + \sigma_2^2 + \sigma_3^2 - \sigma_1\sigma_2 - \sigma_2\sigma_3 - \sigma_3\sigma_1)$$

simple tension test

$$\frac{1+\nu}{3E} \sigma_1^{*2} = \frac{1+\nu}{3E} \sigma_y^2$$

for isotropic materials

$$\max_x \frac{1+\nu}{3E} (\sigma_1^2 + \sigma_2^2 + \sigma_3^2 - \sigma_1\sigma_2 - \sigma_2\sigma_3 - \sigma_3\sigma_1) < \frac{1+\nu}{3E} \sigma_y^2$$

$$\max_x (\sigma_1^2 + \sigma_2^2 + \sigma_3^2 - \sigma_1\sigma_2 - \sigma_2\sigma_3 - \sigma_3\sigma_1) < \sigma_y^2$$

K factor, therefore, we also call it octahedral shear stress theory!

Last class on Tuesday

Quiz 2

4:530
IV-254