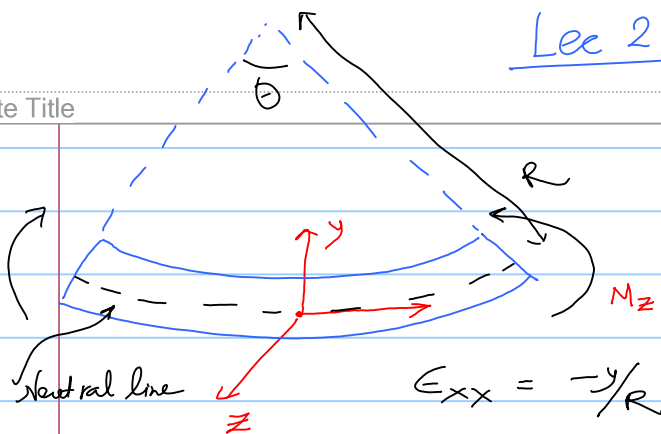


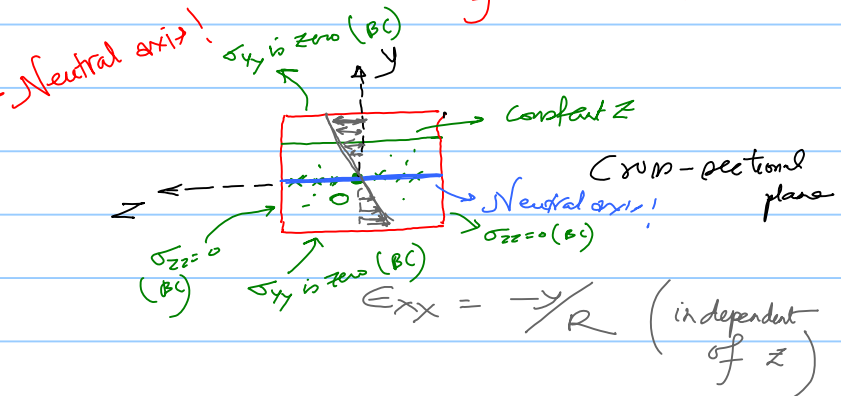
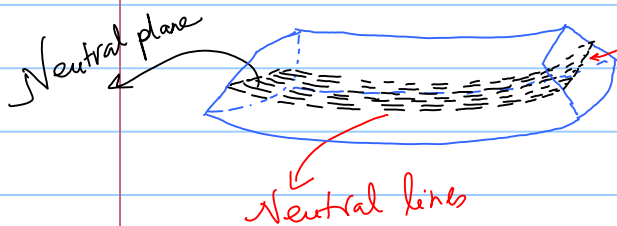
Lee 27

(pure bending of beams)

* Only bending moment is present in any internal section (no force!)



* unknowns are R and also the location of neutral line



What is σ_{xx} ?

$$\epsilon_{xx} = \frac{1}{E} \left(\sigma_{xx} - \nu(\sigma_{yy} + \sigma_{zz}) \right)$$

Approximation:

$\sigma_{yy} = \sigma_{zz} = 0$ everywhere in the cross-section!

$$\sigma_{xx} = E \epsilon_{xx} = -E y/R$$

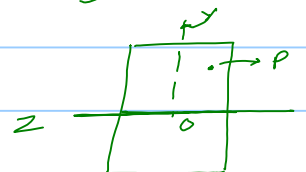
$$F_x = \iint_{\text{cross-sectional plane}} \sigma_{xx} dA = -\frac{E}{R} \iint_{\text{cross-section}} y dA = -\frac{E}{R} y_c A = 0$$

$$\Rightarrow y_c = 0!$$

\Rightarrow Neutral axis passes through the centroid of the cross-section!

$$\vec{M}_O = \iint_{\text{cross-section}} \vec{r}_{P/O} \times \vec{F}(P) dA$$

Centre of cross-section



$$= \iint_{\text{cross-section}} (y\hat{j} + z\hat{k}) \times (\sigma_{xx}\hat{i} + \tau_{yx}\hat{j} + \tau_{zx}\hat{k}) dA$$

$$M_z = \vec{M}_O \cdot \vec{k} = \left[\iint_{\text{cross-section}} (y\hat{j} + z\hat{k}) \times (\sigma_{xx}\hat{i} + \tau_{yx}\hat{j} + \tau_{zx}\hat{k}) dA \right] \cdot \hat{k}$$

do not contribute to final result -

$$= \left[\iint [y \hat{j} \times (\sigma_{xx} \hat{i} + \tau_{yx} \hat{j})] \cdot \hat{k} dA \right]$$

$$= - \iint y \sigma_{xx} dA$$

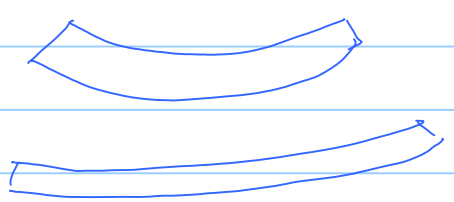
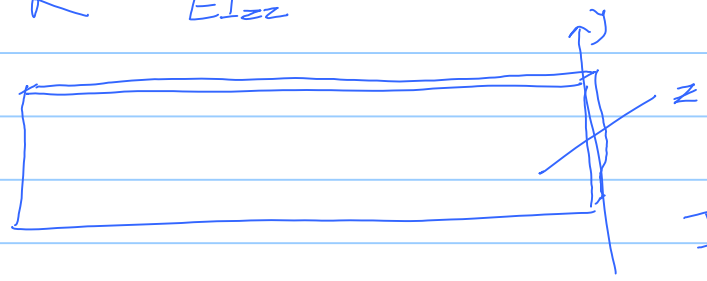
$$M_z = \frac{E}{R} \iint y^2 dA = \frac{E I_{zz}}{R}$$

cross-section

$$\Rightarrow R = \frac{E I_{zz}}{M_z}$$

bending curvature

$$\kappa = \frac{1}{R} = \frac{M_z}{E I_{zz}}$$



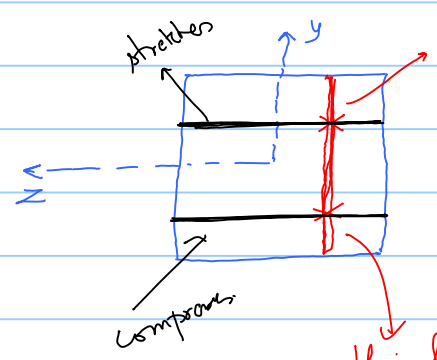
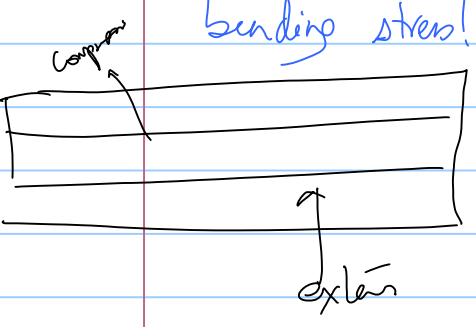
$$R = \frac{E I_{yy}}{M_y}$$

$$I_{zz} = \iint y^2 dA$$

$$I_{yy} = \iint z^2 dA$$

$$\sigma_{xx} = -\frac{E y}{R} = -\frac{M_z y}{I_{zz}}$$

bending stress!



$\epsilon_{yy} = ?$

$$\epsilon_{yy} = \frac{1}{E} (\sigma_{yy} - \nu (\sigma_{xx} + \sigma_{zz}))$$

$$= -\nu \frac{\sigma_{xx}}{E} = -\nu \epsilon_{xx}$$

$$\epsilon_{yy} = -\nu \epsilon_{xx} = \nu y / R$$

$$\epsilon_{zz} = -\nu \epsilon_{xx} = \nu y / R$$

this line does not change its length

