

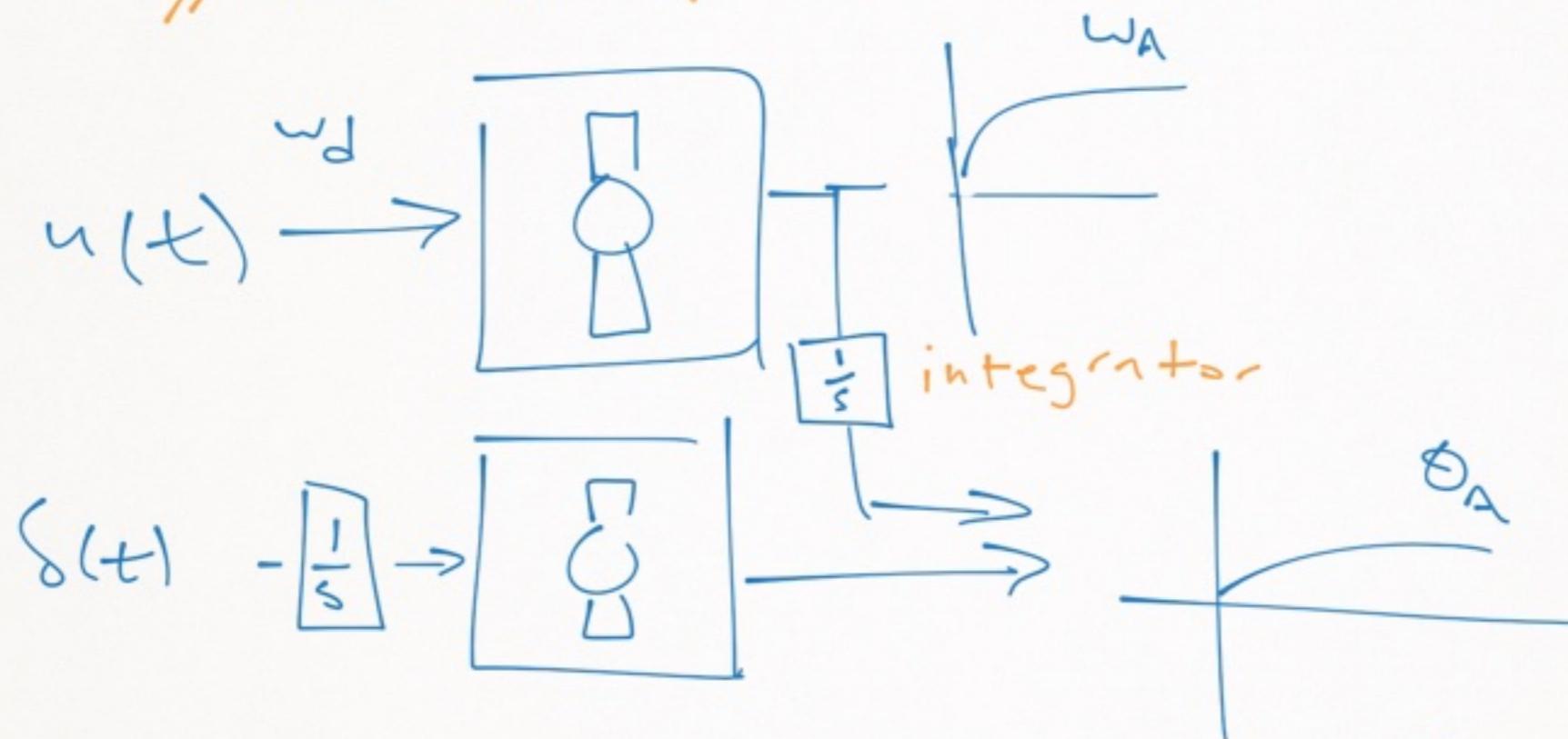
$$\frac{k_m}{R_B + k_m K_B}$$

IMPULSE

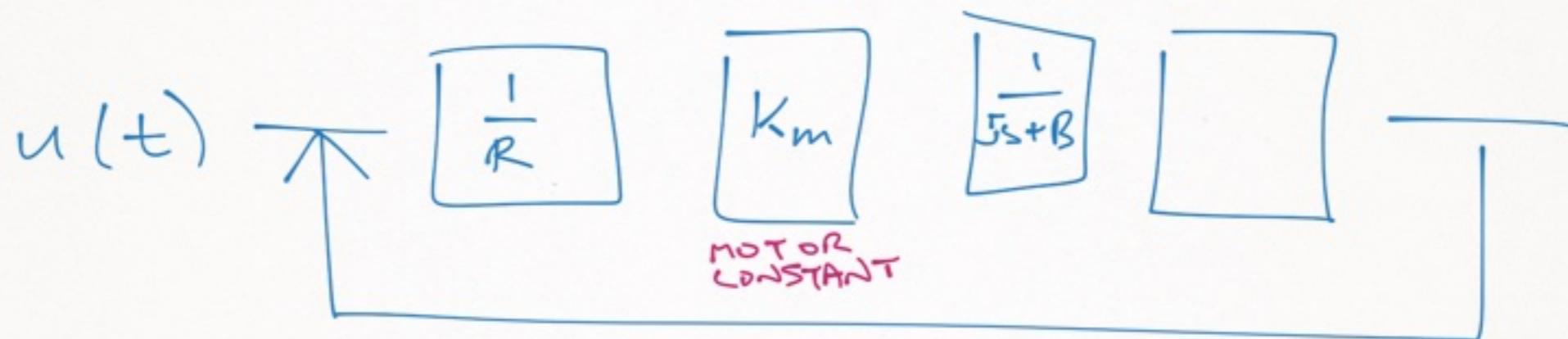


$$w(t) = \frac{k_m}{R_B + k_m K_B} \left(u(t) - e^{-\frac{(R_B + k_m K_B)t}{R_S}} \right)$$

//calculus in laplace is commutative

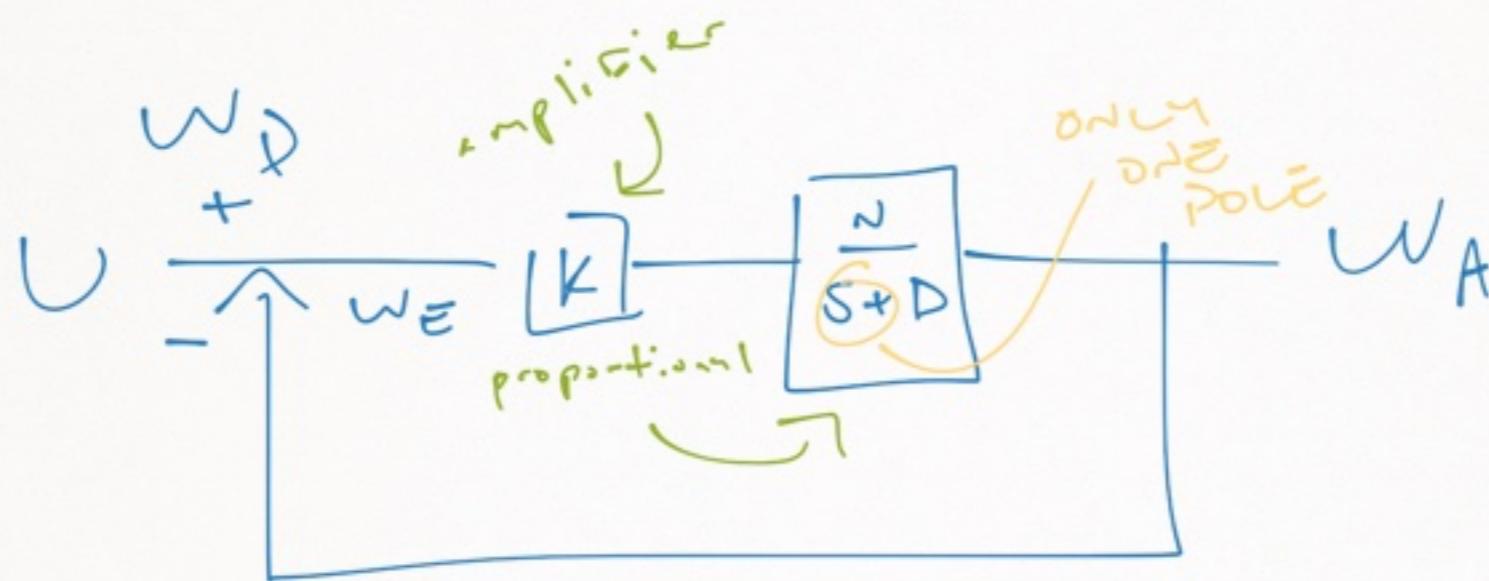


MOTOR MODEL // yline



TAKE motor & control
(BRUIN OR COURSEWORK)

MOST FUNDAMENTAL CONTROL SYS.



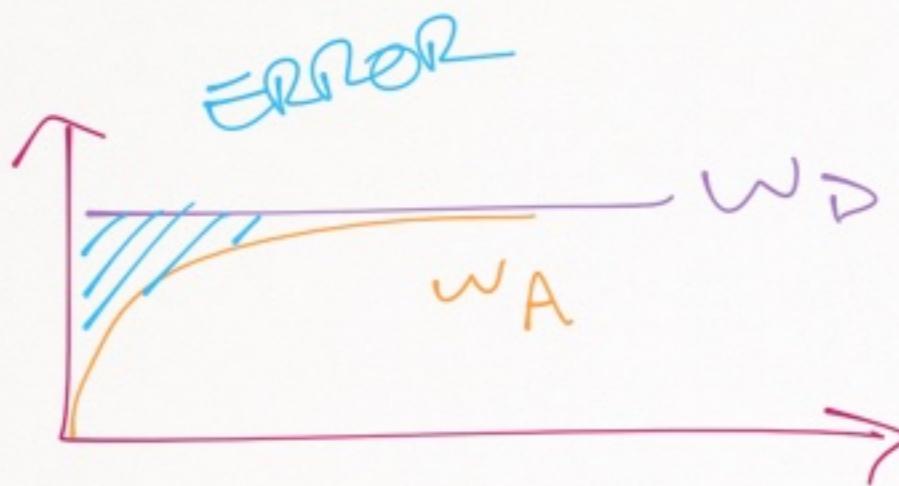
$$TF = \frac{\frac{K_N}{S+D}}{1 + \frac{K_N}{S+D}} = \frac{K_N}{S + K_N + D}$$

Separate DC Gain

you pick K

$$= \frac{K_N}{K_N + D} \frac{K_N + D}{(S + K_N + D)} \quad K_{DC} = \frac{K_N}{K_N + D}$$

\uparrow \uparrow
 K_{DC} $\frac{a}{S+a}$

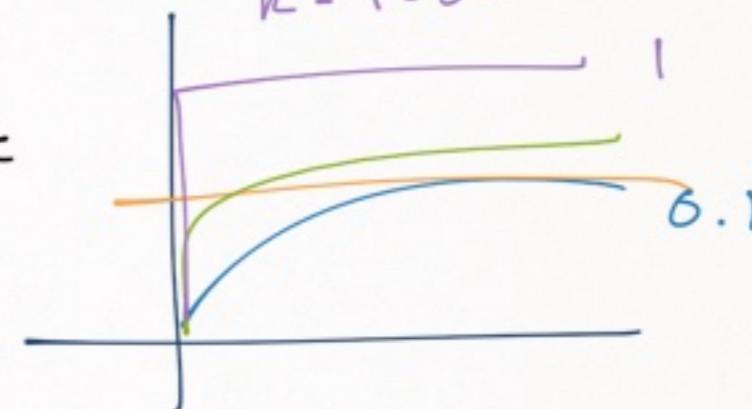


VALUE OF K_D Shows
YOU VALUE, $e^{-(K_D + D)t}$
Shows ROUTE TO
GET THERE.

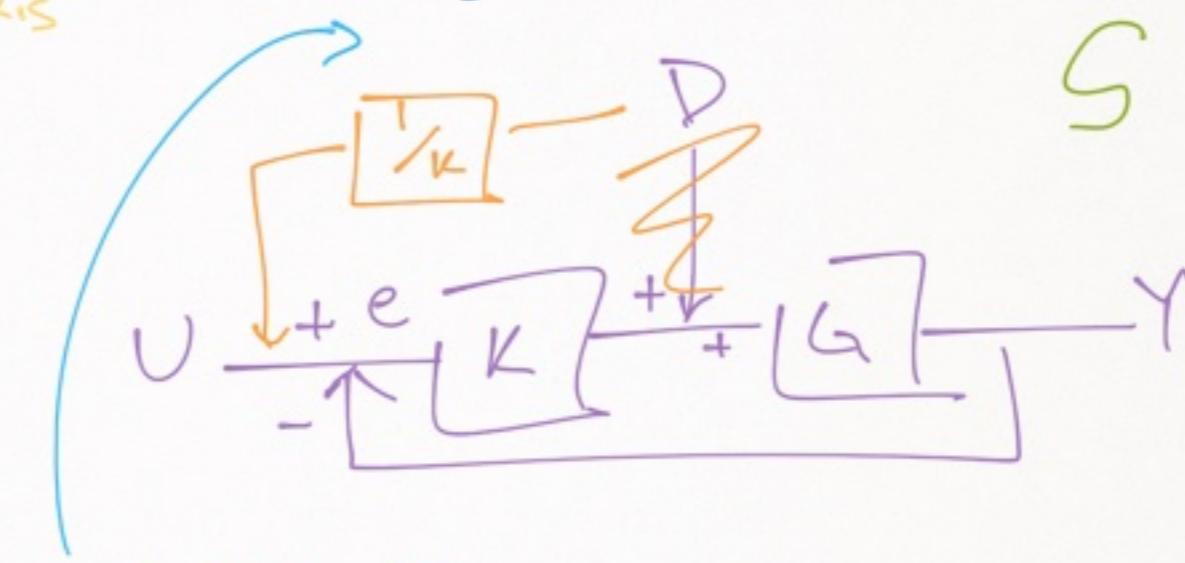
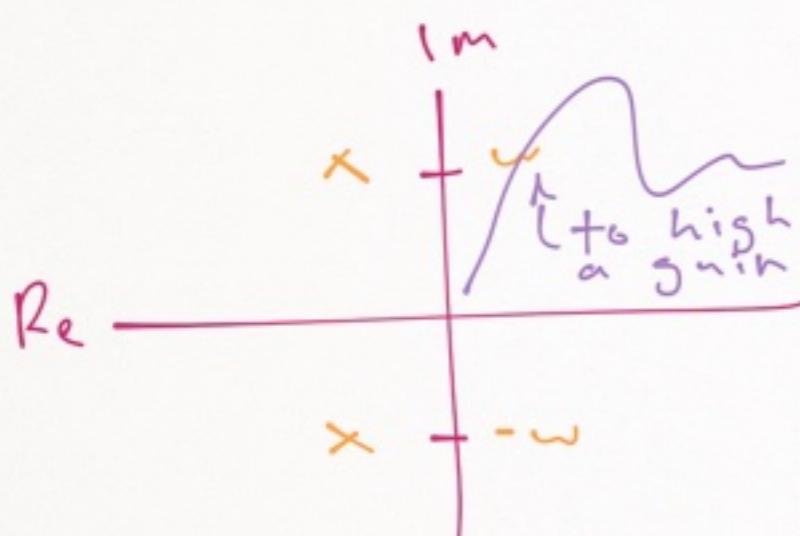
$$N=1 \quad D=3 \quad K=1$$

$$\frac{1}{4} (v(t) - e^{-4t})$$

$$N=1 \quad D=3 \quad K=3$$



THE BIGGER
THE K THE
BETTER



reduces the disturbances based on how big you pick K

$$R = 2 \\ OTHERS = 1$$

MOTOR

$$TF = \frac{K_m}{RJs + RB + K_m K_B}$$

$$= \frac{1}{2s+3} = \frac{1/2}{s + 3/2} \rightarrow \text{pole } @ -\frac{3}{2}$$

ADD F.B.

$$TF = \frac{KK_m}{RJs + RB + K_m K_B + KK_m}$$

$G_D H_D$

$K K_m$

$$= \frac{G_D H_D}{2s+2+1+K}$$

$K/2$

$$= \frac{K/2}{s + \frac{3}{2} + \frac{K}{2}}$$

→ Poles @ $-\left(\frac{3}{2} + \frac{K}{2}\right)$

NO FB:
→ pole @ $-\frac{3}{2}$

FB
→ pole @ $-\frac{3}{2} - \frac{K}{2}$

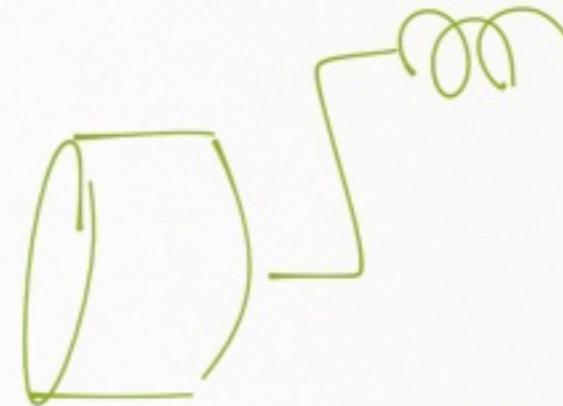
To make it sharper
feedback can move
pole to left!

SECOND ORDER SYS.

first order systems // you can model
never oscillate gravity as spring

→ mass introduces derivative term

→ spring introduces integral term

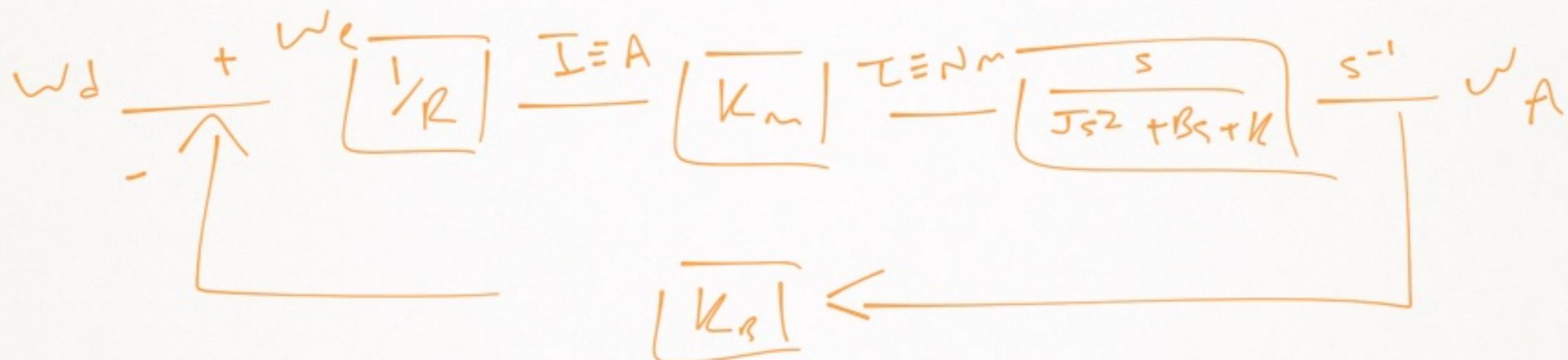


$$\bar{\tau} = J_s \omega = B \omega \\ = \underbrace{\text{SPRING}}_K \cdot \frac{1}{s} \omega$$

$$\bar{\tau} = (J_s + B + \frac{K}{s}) \omega$$

M+T+R MODEL

$$\bar{\tau} = \frac{s}{Js^2 + Bs + K}$$



TRANSFER
FUNCTION
MOTOR
MODEL

$$G(s) = \frac{\frac{K_m s}{RJ s^2 + RBs + RK}}{1}$$

$$= \frac{K_m s}{RJ s^2 + (RB + K_A K_B) s + RK}$$

$$H(s) = \frac{K_p}{1}$$

$$R = 2 \Omega$$

$$K_m = 6 \frac{\text{Nm}}{\text{A}}$$

$$J = 3 \text{ Nm}^2$$

$$B = 9 \text{ Nms}$$

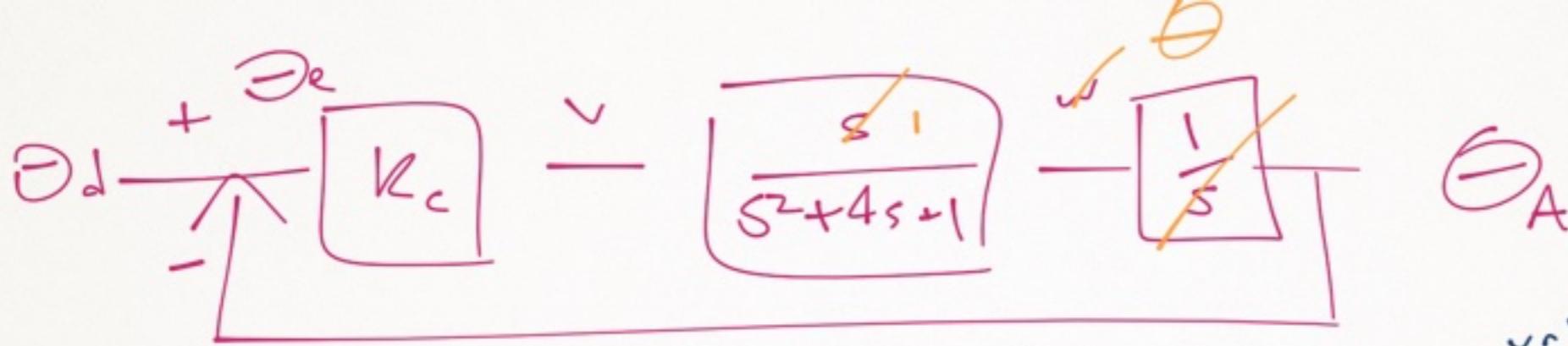
$$K = 3 \text{ Nm}$$

$$K_B = 1 \sqrt{s}$$

$$\left. \begin{aligned} \text{TF} : & \frac{6s}{6s^2 + 24s + 6} \\ & = \frac{s}{s^2 + 4s + 1} \end{aligned} \right\}$$

\hookrightarrow poles at

$$\frac{-4 \pm \sqrt{16 - 4}}{2} = -2 \pm \sqrt{3}$$



$$TF = \frac{K_c}{s^2 + 4s + 1 + K_c}$$

$$\omega_n = ? \quad (\text{NAT. f}) = \sqrt{1 + K_c}$$

$$2\zeta\sqrt{1+K_c} = 4 \quad \zeta = \frac{2}{\sqrt{1+K_c}}$$

$$TF = \frac{K_c}{1+K_c} \frac{1+K_c}{s^2 + 4s + 1 + K_c}$$

↑
controller

\therefore if $K_c = 1$, underdamped

\therefore if $\zeta > 1$, overdamped

critical damping: $\zeta = 1$

natural f increases w/ K_c

K_c compensates for friction
 \therefore reduces damping

$$\frac{2}{\sqrt{1+K_c}} = 1 \quad \therefore K_c = 3 \quad \begin{array}{l} \text{(critically} \\ \text{damped)} \end{array}$$

ie. as fast
 as possible w/o
 overshooting

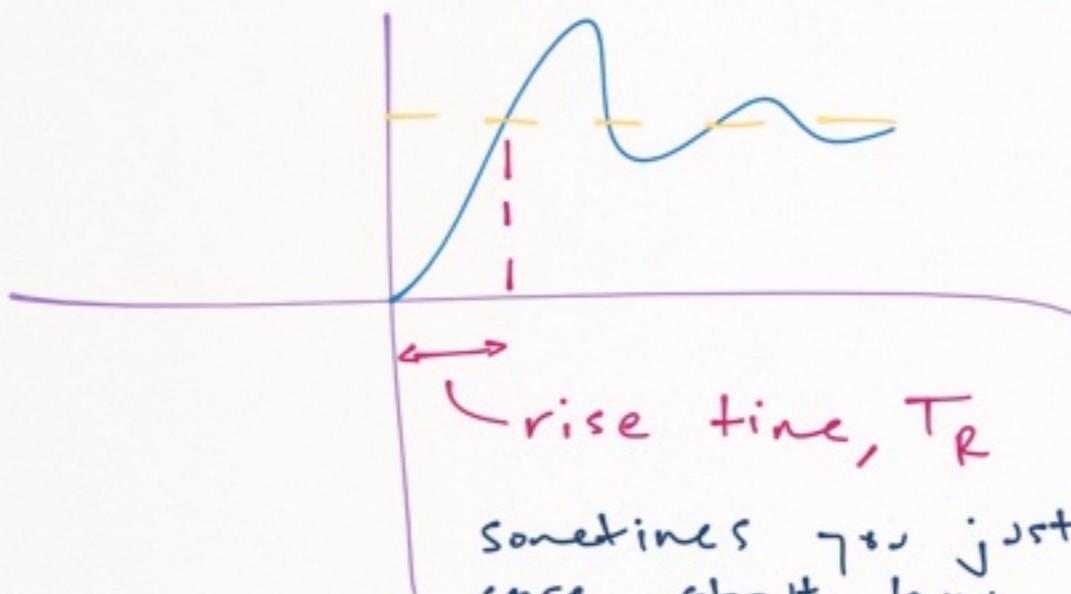


Note: get simulink
get MATLAB

$$\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

$$\frac{\omega_n^2}{s(s^2 + 2\zeta\omega_n s + \omega_n^2)}$$

be started
these laplace
functions



Sometimes just
care about how
long it takes to
reach target

ex. racing

for rise time, we're interested
in step response

$$\frac{1}{s} \cdot \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} = T(s)$$

@ $t=?$ such
that $f(t) = 1$

T_R is the time $f(t) = 1$

$$f(t) = 1 - \frac{1}{\beta} e^{-\beta \omega_n t} \sin(\beta \omega_n t + \cos^{-1} \zeta)$$

$$\zeta^2 + \beta^2 = 1$$

$$\begin{aligned} \sin \theta &= \beta & \theta &= 0 \\ \cos \theta &= \zeta & \theta &= \pi, 2\pi \end{aligned}$$

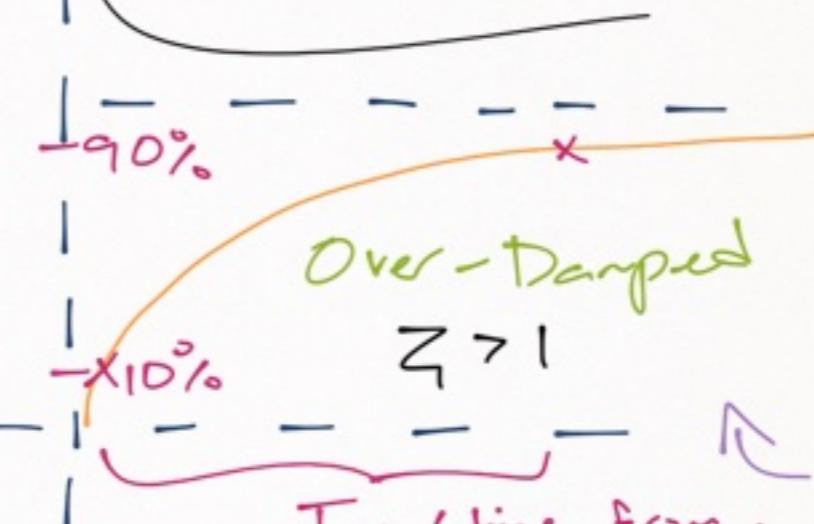
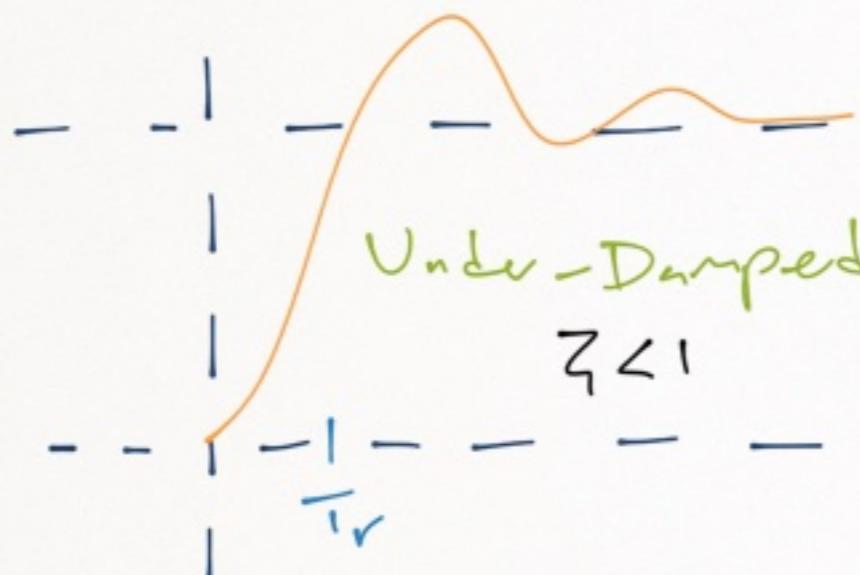
$$\phi: t = \frac{-\cos^{-1} \zeta}{\beta \omega_n}$$

~~$t < 0$~~ doesn't satisfy our conditions

$$\pi: t = \frac{\pi - \cos^{-1} \zeta}{\beta \omega_n}$$

$$= \frac{\pi - \tan^{-1} \frac{-\beta}{\zeta}}{\beta \omega_n}$$

b7 trig identity



Tr_1 (time from 10% \rightarrow 90%)

$$\beta = j \# \quad \text{ie. never reach } Tr$$

$$Tr = \frac{1}{\beta \omega_n} \left[\pi - \tan^{-1} \frac{\beta}{\zeta} \right]$$

$$\beta = \sqrt{1 - \zeta^2}$$

↑ only parameters are $\omega_n \# \zeta$, can determine rise-time

Oct 6/2014

Computing T_{r1} (T_r for overdamped sys)

Take step response

$$y(t) = 1 - \frac{1}{\beta} e^{-\zeta \omega_n t} \sin(\beta \omega_n t - \cos^{-1} \zeta)$$

$$0.9/0.1$$

β is still imaginary, doesn't make sense

this is only the function
for underdamped.

↳ "gets ugly when overdamped"

don't do mathy stuff, just plot & read

as a last resort:

when $\|a\| < \|10.b\|$

just ignore b-term

Actual \neq Approx. Rise Time (underdamped)

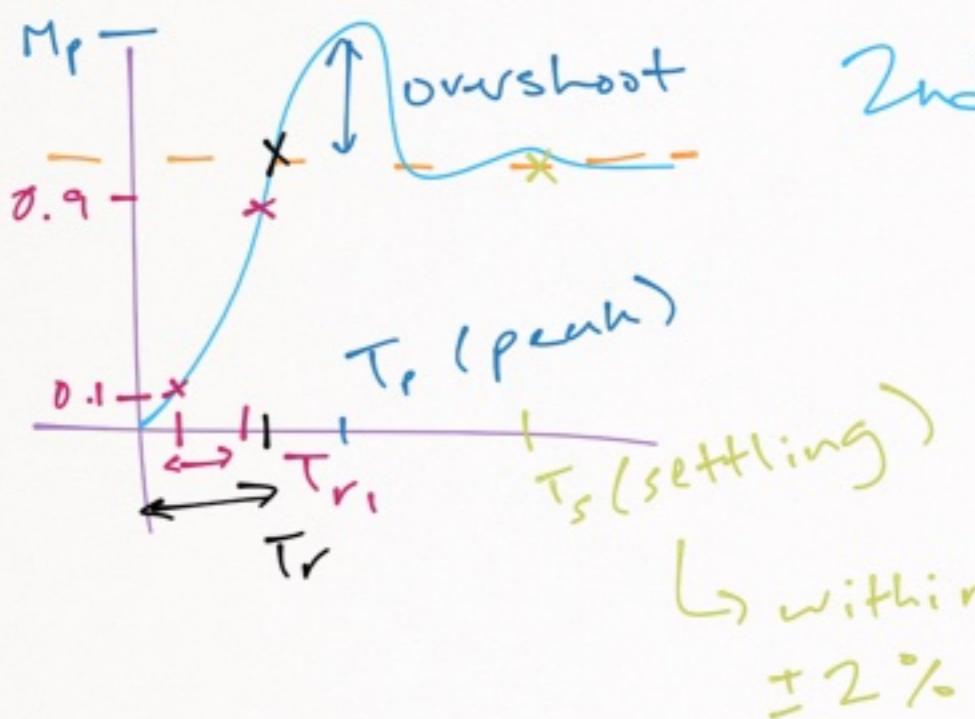
When $0.3 < \zeta < 0.8$

$$T_{r1} \approx \frac{2.16\zeta + 0.60}{\omega_n}$$

When $\zeta > 0.8$

$$T_{r1} \approx \frac{4.65\zeta - 1.3}{\omega_n}$$

} just reasonable estimates



2nd order underdamped

$$Y(s) = \frac{1}{s} \cdot \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$$

} within
 $\pm 2\%$

Peak time, T_p

Oct. 15, 2014

Same as 1st year calc., take derivative and find zeros

$$\cancel{s} \cdot \cancel{\frac{1}{s}} T(s) \rightarrow t(t) = \phi$$

↑
derivative

$$\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \rightarrow \frac{\omega_n}{\beta} e^{-\zeta\omega_n t} \sin(\beta\omega_n t) = \phi$$

$$= \phi, \pi, 2\pi$$

flat
slope
at $t = \phi$
(trivial)

$$\beta\omega_n t = \pi$$

$$T_p = \frac{\pi}{\beta\omega_n}$$

$$\frac{1}{s} T(s) \rightarrow t(t) = \phi$$

sub T_p

$$t(t) = 1 - \frac{1}{\beta} e^{-\zeta\omega_n t} \sin[\beta\omega_n t + \cos^{-1} \zeta]$$

% overshoot

$$\hookrightarrow = \frac{1}{\beta} e^{-\zeta\pi/\beta} \sin(\pi + \cos^{-1} \zeta)$$

find val. @ T_p

$$\sin(\pi + \#) = -\sin(\#)$$

$$\therefore -\sin(\cos^{-1} \zeta) = \beta$$

$$\frac{1}{\beta} e^{-\zeta\pi/\beta} \cdot (-\beta) = -e^{-\zeta\pi/\beta}$$

$$t(t) = 1 + \underbrace{e^{-\zeta\pi/\beta}}_{\% \text{ overshoot}}$$

Settle time, T_S

time to get 98% of destination ($\pm 2\%$)

$$y(t) = 1 - \frac{1}{\beta} e^{-\zeta \omega_n t} \sin(\beta \omega_n t + \cos^{-1} \zeta)$$

$\underbrace{\phantom{e^{-\zeta \omega_n t}}}_{\text{envelope}} \quad \text{of oscillation}$

T_S , when magnitude of envelope < 0.2

$$\frac{1}{\beta} e^{-\zeta \omega_n t} = 0.2$$

$$e^{-\zeta \omega_n t} = \beta \cdot 0.2$$

$$-\zeta \omega_n t = \ln(0.2 \cdot \beta)$$

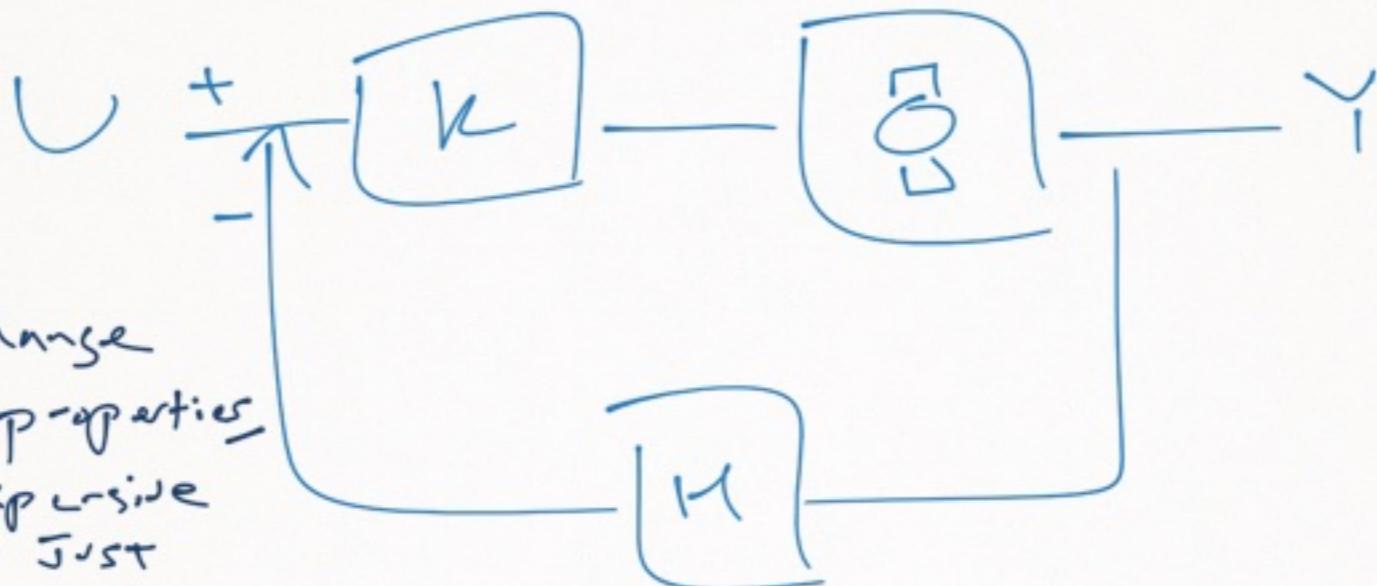
$$t = \frac{\ln(50/\beta)}{\zeta \omega_n} \xrightarrow{\text{sum p to cancel (-)ive}} \frac{1}{50}$$

$$\approx 4 \xrightarrow{\zeta \omega_n}$$

≈ 0 when ζ is small

$$t = \frac{\ln 50 - \ln \beta}{\zeta \omega_n}$$

$$T_S = t \approx \frac{4}{\zeta \omega_n}$$



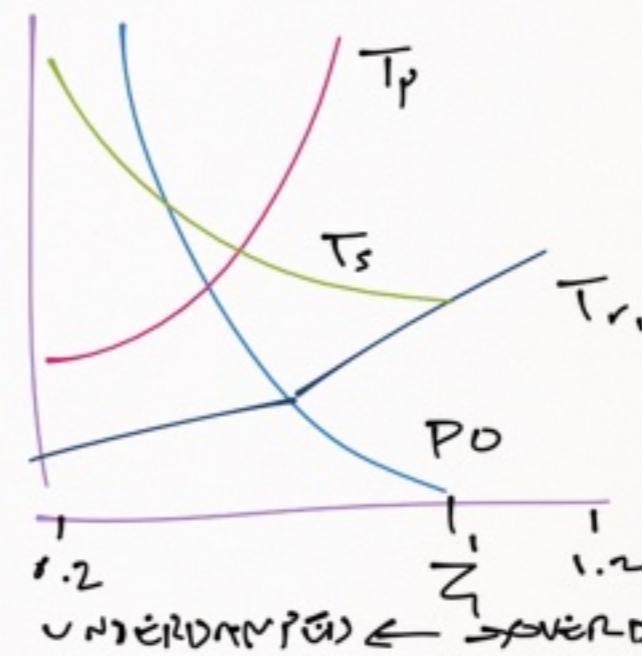
Don't change physical properties that's expensive and hard. Just change using control.

$$\hookrightarrow T(s) = \frac{N(u,s)}{D(u,s)}$$

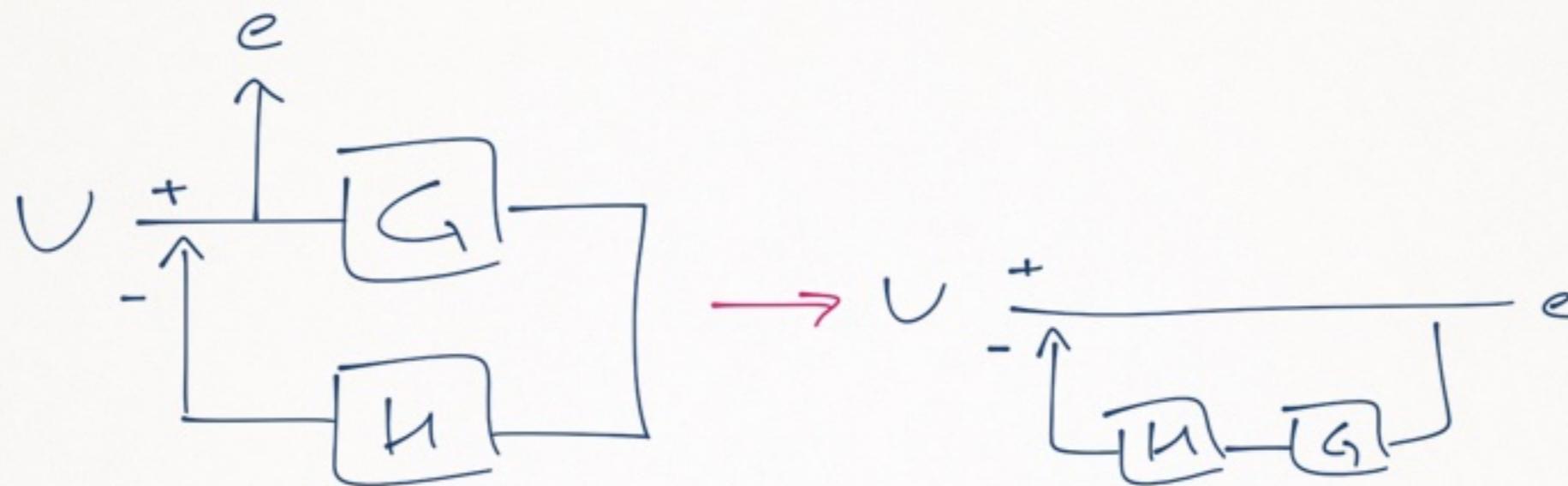
ex. can change a number (K) in C code and increase damping or whatever without changing bearings or physical motor.

$$D(u,s) = s^2 + 2\zeta(u)\omega_n(u)s + \omega_n(u)^2$$

2nd ORDER STEP RESP.

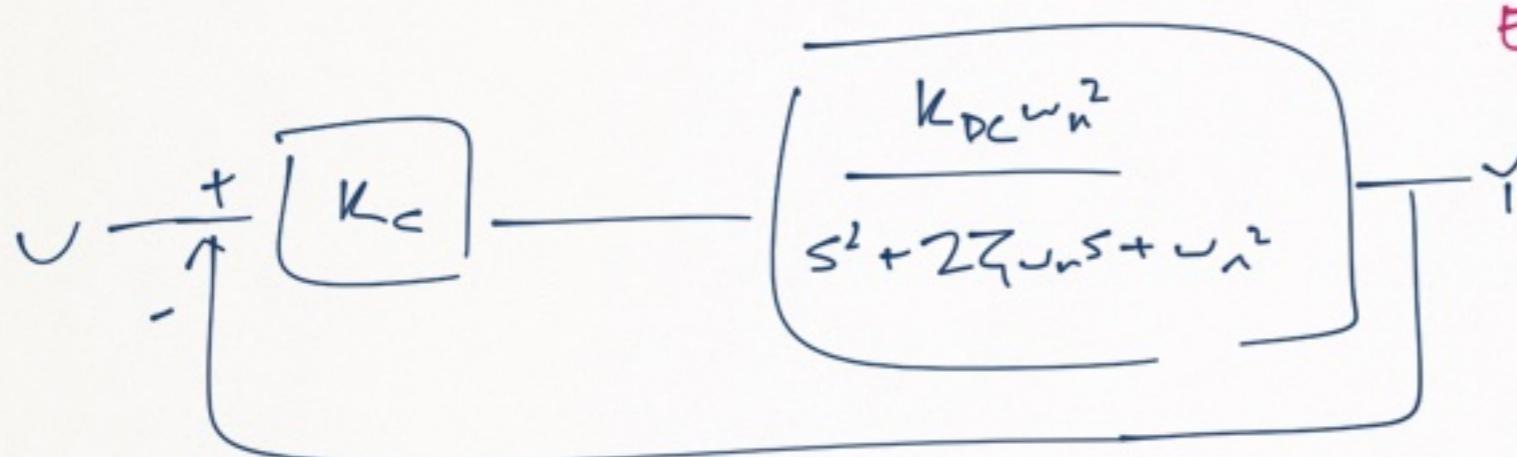


OCT 8,
2014



STEADY-STATE ERROR

= DC Gain of error



$$\frac{e}{U} = \frac{1}{1+GH}$$

$$E_{ss} = \left. \frac{e}{U} \right|_{s=0} = \frac{1}{1+G(s=0)H(r=0)}$$

D-L DC GAIN

$$E_{ss} = \frac{1}{1 + \frac{K_c K_o C \omega_n^2}{\omega_n^2}} = \frac{1}{1 + K_c K_{dc}}$$

If you increase K_c , E_{ss} decreases

↪ pre-tensioning error is bigger ∵ compensates faster

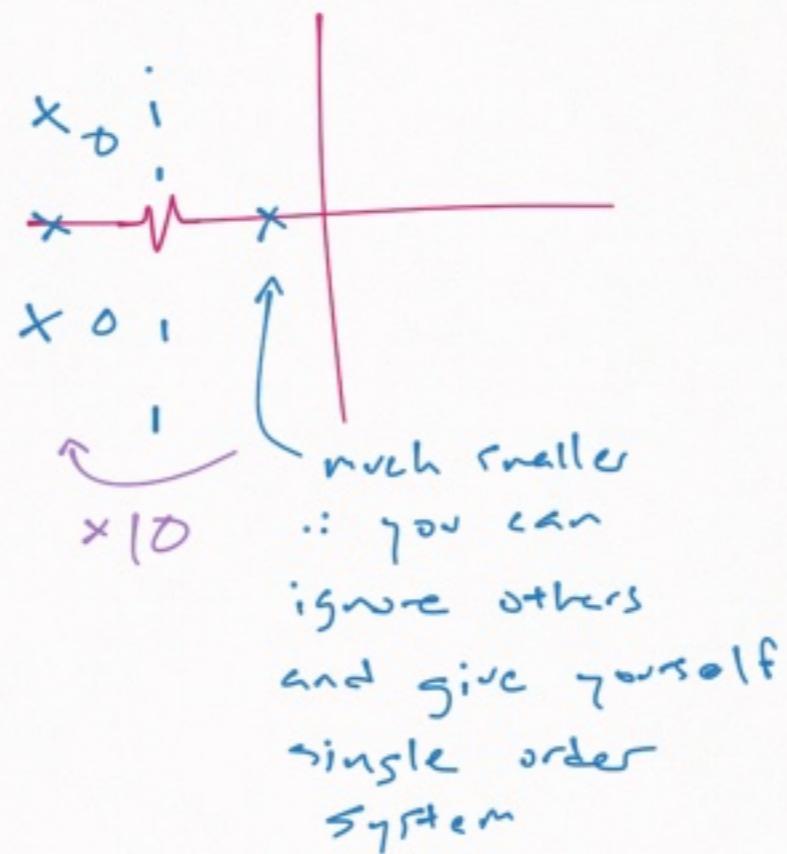
ζ^{rd} + order $s_1 s$

$\zeta = ?$ $u_n = ?$

$$\frac{N}{s^3 + As^2 + Bs + C}$$

$$\frac{D}{s+a} \quad \frac{E}{s+b} \quad \frac{F}{s+c}$$

DOMINANT
ROOTS
SLIDES



Design controller taking into account changes in system we have no control over. (Differences in motors for instance)
ex. different friction, heat, worn bearings, etc.

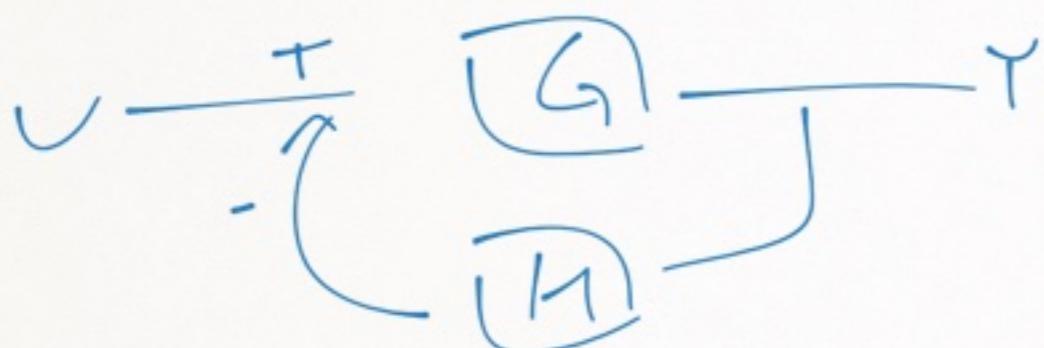
Sensitivity to S_{sys} , Δs

$$S = \frac{\% \Delta \text{ in } T(s)}{\% \Delta \text{ in } G(s)}$$

Smaller S , the better
ie. if big change in $G(s)$
only causes small change in $T(s)$

$$\lim_{\Delta \rightarrow 0} \frac{\Delta T}{\Delta G} = \frac{\delta T}{\delta G}$$

$$S = \frac{G}{T} \cdot \frac{\delta T}{\delta G}$$



$$T = \frac{G}{1+GH} = G(1+GH)^{-1}$$

$$S_G^T = \frac{G}{T} \frac{\delta T}{\delta G}$$

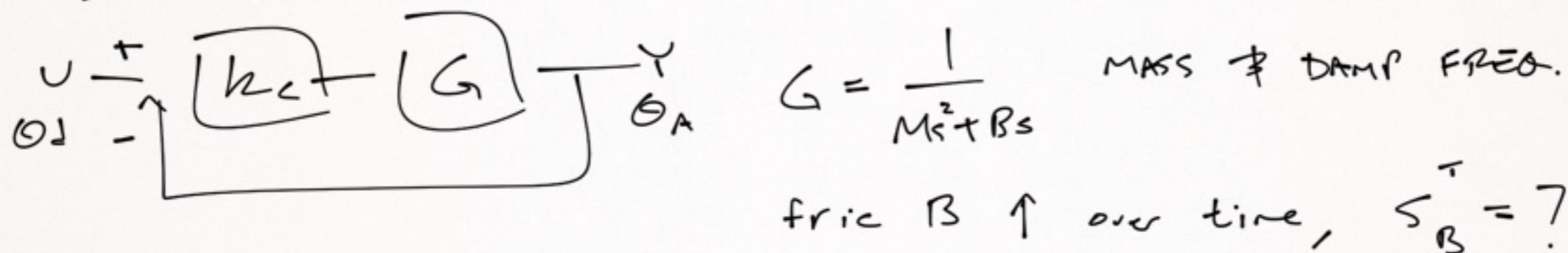
$$\frac{\partial T}{\partial G} = (1+GH)^{-1} - G(1+GH)^{-2}H = \frac{1+GH}{(1+GH)^2} - \frac{GH}{(1+GH)^2} = \frac{1}{(1+GH)^2}$$

Cont'd

$$S_G^T = \frac{G(1+GN)}{G} \frac{\partial T}{\partial G} = \frac{1}{1+GN}$$

Sensitivity of T v.r.t. G

Ex)



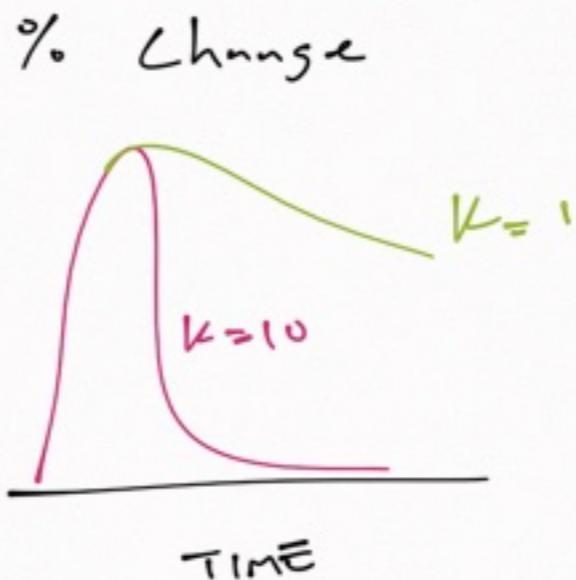
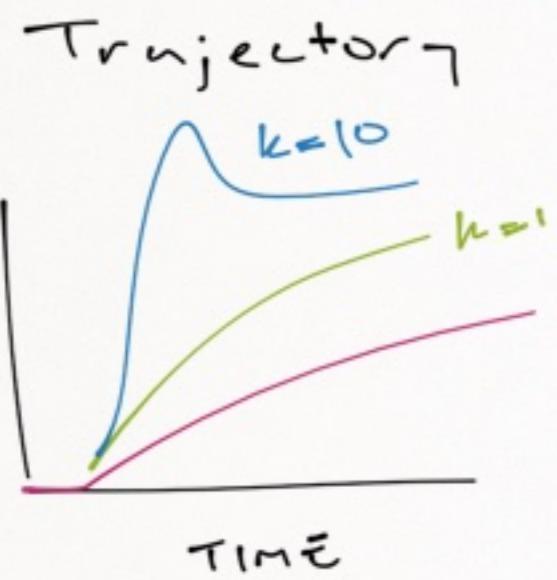
$$T(s) = \frac{\frac{k_c}{Ms^2 + Bs}}{1 + \frac{k_c}{Ms^2 + Bs}} = \frac{k_c}{Ms^2 + Bs + k_c} = k_c (Ms^2 + Bs + k_c)^{-1}$$

$$\frac{\partial T}{\partial B} = -k_c (Ms^2 + Bs + k_c)^{-2} s = \frac{-k_c s}{(Ms^2 + Bs + k_c)^2}$$

$$S_B^T = \frac{B(Ms^2 + Bs + k_c)}{k_c} \cdot \frac{-k_c s}{(Ms^2 + Bs + k_c)^2} = \frac{-Bs}{Ms^2 + Bs + k_c}$$

$\frac{G(1+GN)}{G}$ $\frac{\partial T}{\partial G}$ freq dependency
freq proportional to k_c

OUTPUT



RNP ZEROS OK

RNP POLES = INSTABILITY

$$CE: (s+a)(s+b)(s+c) = \emptyset$$

$$s^3 + \underbrace{(abc+ac)}_A s^2 + \underbrace{(ab+ac+bc)s}_B + abc \underbrace{c}_C$$

if any coeff $< \emptyset$ then a, b, c must $< \emptyset$

\uparrow
 A, B, C

ex. CE: $s^5 + 4s^4 + s^2 - 9s^2 + 2s + 8$

UNSTABLE

C if negative anywhere, \rightarrow
know a pole in
RNP \therefore UNSTABLE

All coeffs need to be $> \emptyset$

(but that's not enough to know
if it's stable)

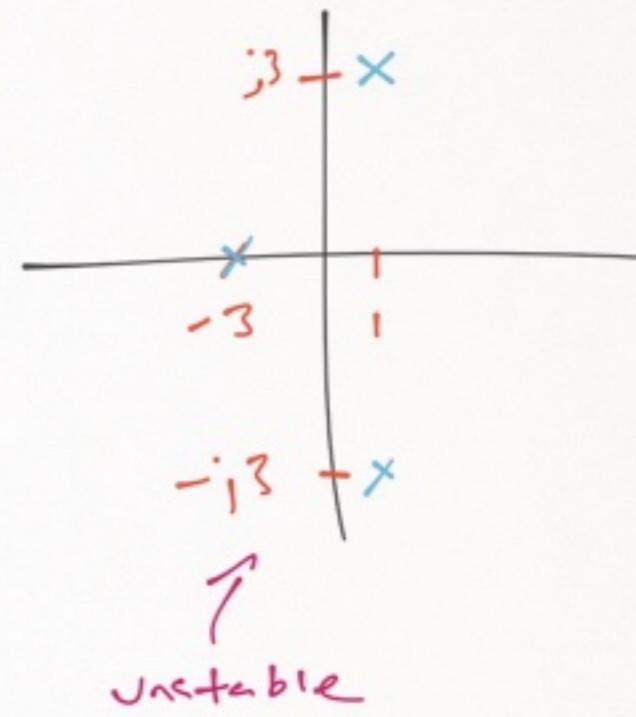
$$\text{ex. } (s+3)(s-1+j3)(s-1-j3)$$

$$(s+3)(s^2 + 2s + 10)$$

$$s^3 - 2s^2 + 10s + 3s^2 - 6s + 30$$

$$s^3 + s^2 + 4s + 30 = 0$$

but doesn't
look it.
unstable



You can't determine
stability this way.

RAO TH HURWITZ CRITERIA (1895)

$$CE: as^7 + bs^6 + cs^5 + ds^4 + es^3 + fs^2 + gs + h$$

orders

7	a	c	e	g	ϕ
6	b	d	f	h	ϕ
5	m_1	m_2	m_3	ϕ	
4	n_1	n_2	n_3		
3	o_1	o_2			
2	p_1	p_2			
1	q_1				
0	r_1				

$$m_1 = \frac{-1}{b} \begin{vmatrix} a & c \\ d & f \end{vmatrix}$$

$$m_2 = \frac{-1}{d} \begin{vmatrix} a & e \\ b & f \end{vmatrix}$$

$$m_3 = \frac{-1}{b} \begin{vmatrix} a & g \\ d & h \end{vmatrix}$$

$$n_1 = \frac{-1}{m_1} \begin{vmatrix} b & d \\ m_1 & m_2 \end{vmatrix}$$

$$n_2 = \frac{-1}{m_1} \begin{vmatrix} b & f \\ m_1 & m_3 \end{vmatrix}$$

$$n_3 = \frac{-1}{m_1} \begin{vmatrix} b & h \\ m_1 & \phi \end{vmatrix}$$

if the sign
changes between
any of these ∴ UNSTABLE

$$7s^2 + 6s^4 + 5s^3 + 4s^2 + s + 1 \quad 7 \quad 5 \quad 3 \quad 1$$

$$-\frac{1}{6}(28 - 30) = \frac{1}{3}$$

$$-\frac{1}{6}(14 - 18) = \frac{2}{3}$$

$$-3\left(\frac{18}{3} - \frac{4}{3}\right) = -\frac{3 \cdot 14}{3} = -14$$

UNSTABLE
B/C SIGN

$$6 \quad 4 \quad 2 \quad 1$$

$$\frac{1}{3} \quad \frac{2}{3}$$

$$-14$$

R-11 CRITERIA EXCEPTION

→ WHEN C.E. IS UNFACTORABLE \nRightarrow NO COEFFICIENT SIGNS CHANGE

1. Check if numerator order < denominator order (if not what?)
2. Check if sign Δ in denominator

$$\frac{+}{-} \boxed{K_c} \rightarrow \boxed{\frac{6}{(s+1)(s+2)(s+7)}} \quad \boxed{-}$$

$$\frac{Y}{U} = \frac{1}{6K_c + s^3 + 6s^2 + 11s + 6}$$

$$= \frac{1}{s^3 + 6s^2 + 11s + 6(K_c + 1)}$$

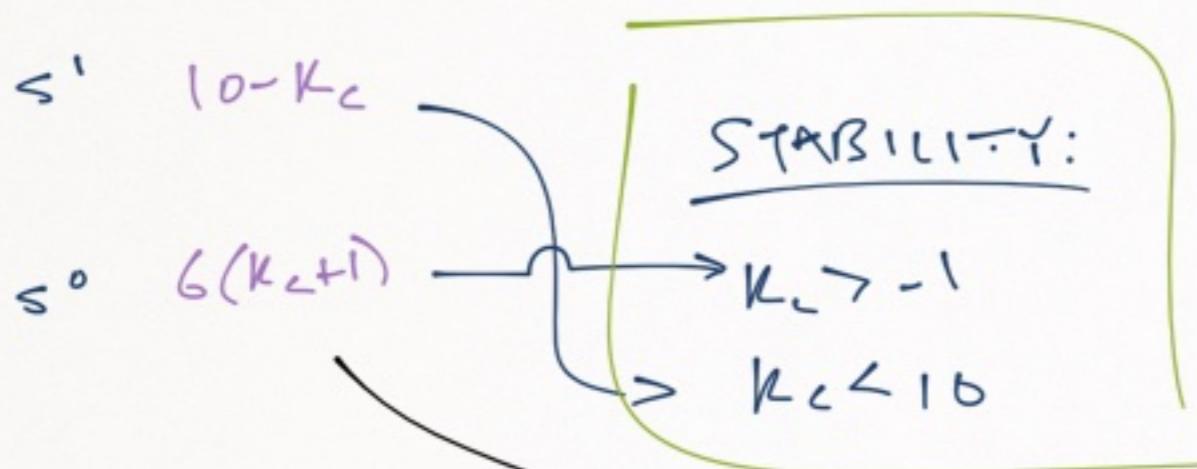
what's value of K_c
such that system is stable?

↪ can change \overline{Y} by changing K_c

K_c at least needs
to be ≥ -1

$$\begin{array}{ccc} s^3 & | & 11 \\ s^2 & 6 & 6(K_c+1) \end{array} \quad -\frac{1}{6}(6(K_c+1) - 66) = 11 - (K_c+1) > 0$$

$$K_c < 10$$



$$K_c = -1 \quad \frac{1}{s} \frac{6K_c}{(s^2 + 6s + 11)} \quad \text{if either of these } = 0 \text{ then MARGINAL STABILITY}$$

=

CE:

0 in engineering
is lim in
 $\lim_{x \rightarrow 0}$
mathematics

zero is not nothing,
its insignificance,
but you can amplify
it to being finite.

$$D(s) = s^3 + s^2 + 7s + 7$$

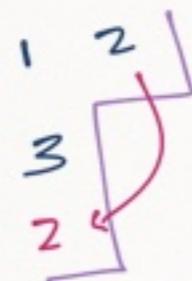
$$\begin{array}{ccc} s^3 & 1 & 7 \\ s^2 & 1 & 7 \end{array} \quad -\frac{1}{6}(7-7)$$

$$\begin{array}{ccc} s^1 & \cancel{\infty} & \emptyset \\ s^0 & \cancel{\frac{\phi}{\phi}} & 7 \end{array} \quad -\frac{1}{6}(\phi - \phi) = 7$$

can't push a rope
two fingers to scratch nose
 $\frac{\phi}{\phi} = 7$ (or something finite)

$O = \epsilon$ (small insignificant value that could multiply to something)

$$s^4 + 3s^3 + 2s^2 \longrightarrow s^2(s^2 + 3s^2 + 2)$$



✓ looks stable.

$$\frac{1}{s} \cdot \frac{N(s)}{s^2 + 3s + 2}$$

$$\hookrightarrow (s+2)(s+1) (?)$$

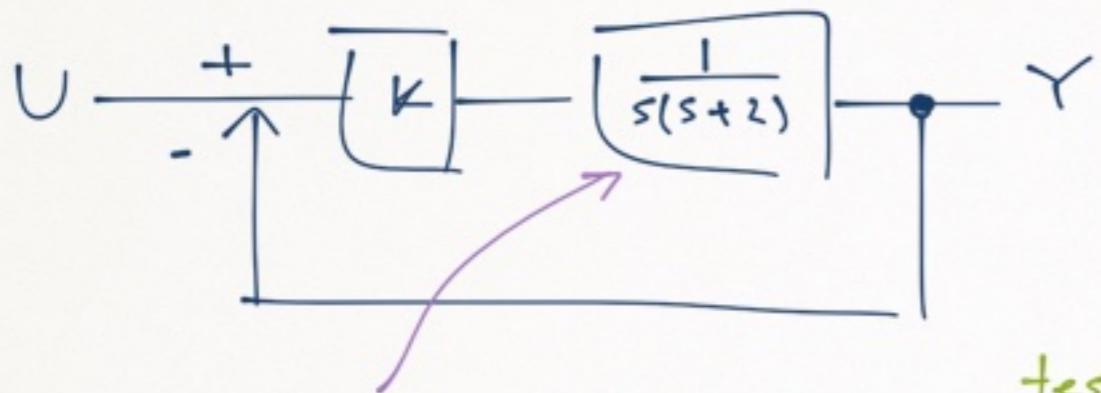
HEAVYSIDE



have to do

$$s^4 + 3s^3 + 2s^2 + Es + E$$

$$\begin{matrix} 1 & 2 & \phi \\ 3 & \phi \\ 2 \\ -\frac{E}{2} \end{matrix}$$



poles of open loop transfer function = poles of closed loop transfer function

$$\frac{Y}{U} = \frac{k}{s^2 + 2s + k}$$

$$CE: s^2 + 2s + k = 0$$

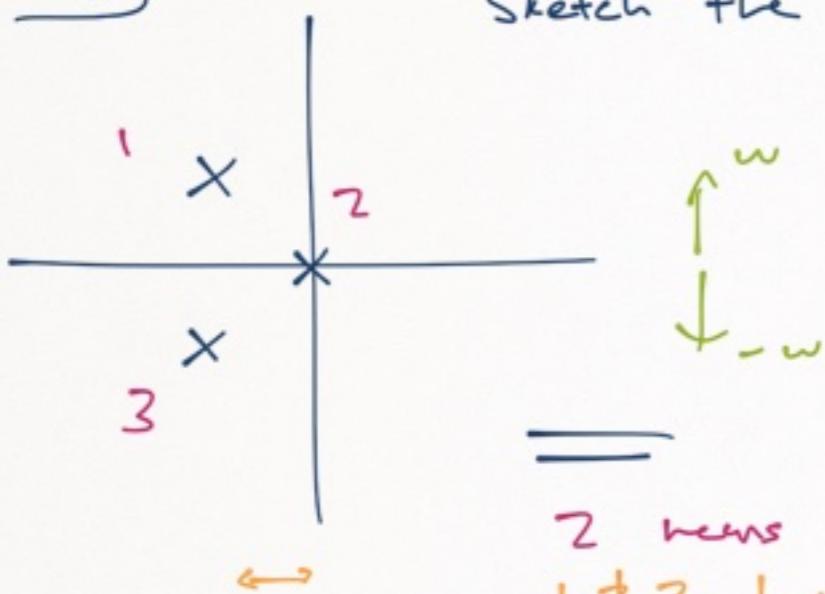
↑ values of s that makes this true
defines our poles, depends on k
test $k=0$ (not mathematical \emptyset , a negligible #)
 $\hookrightarrow s=0, -2 \rightarrow$ poles @ $k=0$



if γ_{out} have a pole like ω mag 10x bigger than the inside pole, γ_{out} can ignore it.
(not w/ \emptyset though, so not now)

Ex)

Sketch the impulse response.

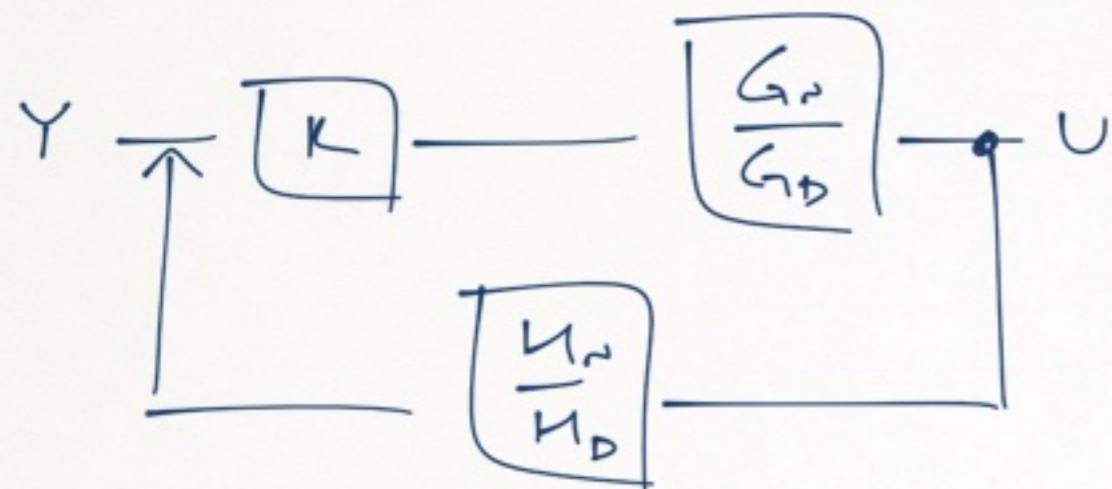


2 means step response

1 + 3 being on left means decaying envelope
being on top & bottom gives us w so there's a sign



OCT 17 2014



KGH = open loop transfer function

$$TF = \frac{KG_nH_D}{KG_nH_n + G_DH_D}$$

When $K = \emptyset$ (small)

$$TF = \frac{1}{G_DH_D}$$

G_DH_D = open loop characteristic equation

closed loop poles = open loop poles

When $K \rightarrow \infty$

TF

closed loop poles = open loop zeros

$$CE: s^2 + 2s + k = 0$$

$$s = \frac{-2 \pm \sqrt{4 - 4k}}{2} = -1 \pm \sqrt{1-k}$$

σ

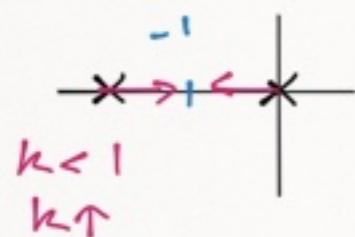
from $k=0 \rightarrow 1$, poles are real

$k > 1$, imaginary component

↳ corresponds to frequency

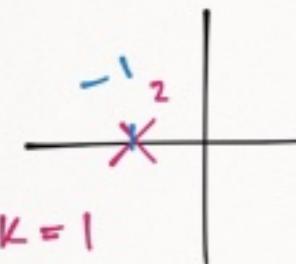
$k < 0$ (not useful, just unstable)

①



$$k < 1$$

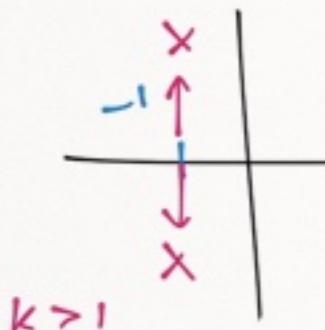
②



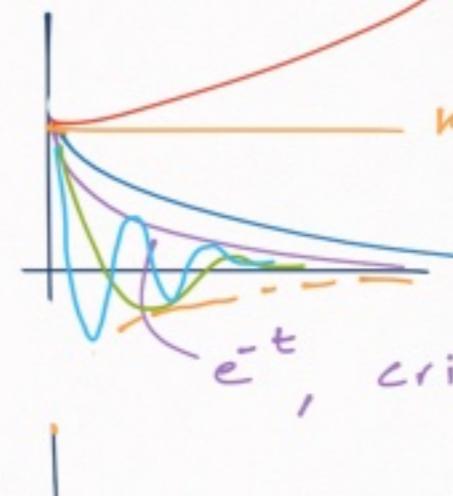
$$k = 1$$

CRITICAL
DAMPING

③

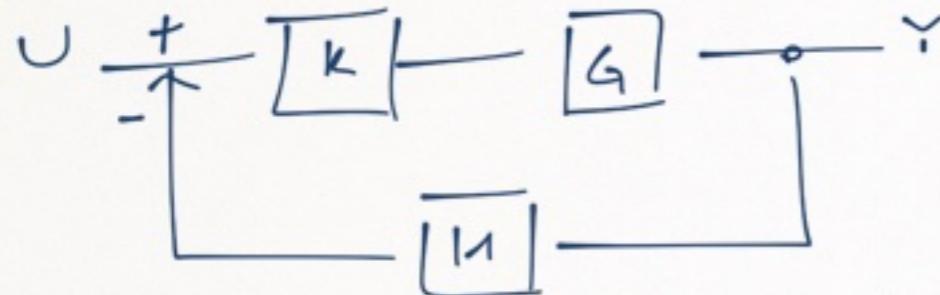


$$k > 1$$



WHEN YOU HAVE MANY POLES

OCT 20 2014



$$TF = \frac{KG}{1+KGH} \quad \begin{matrix} \leftarrow \text{function of } s \\ \text{which is a complex number,} \\ s = \sigma + j\omega \end{matrix}$$

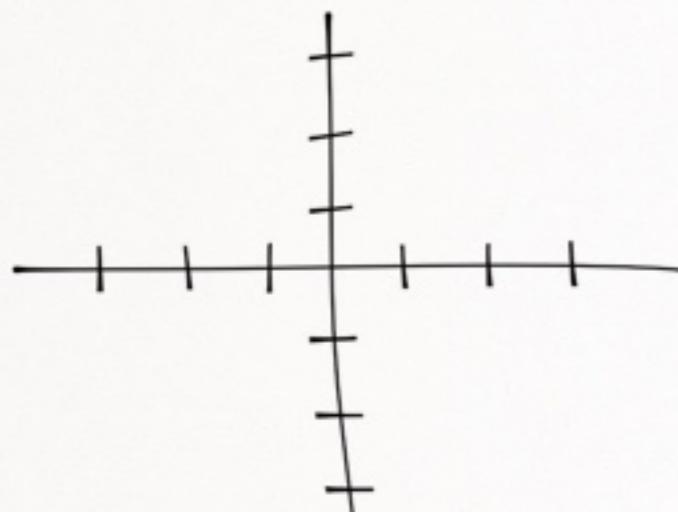
CE: $\underbrace{1+KG(s)H(s)}_{=0} = 0$

this is just a complex number we need to be equal to -1
ie. $= 1 < 180^\circ$

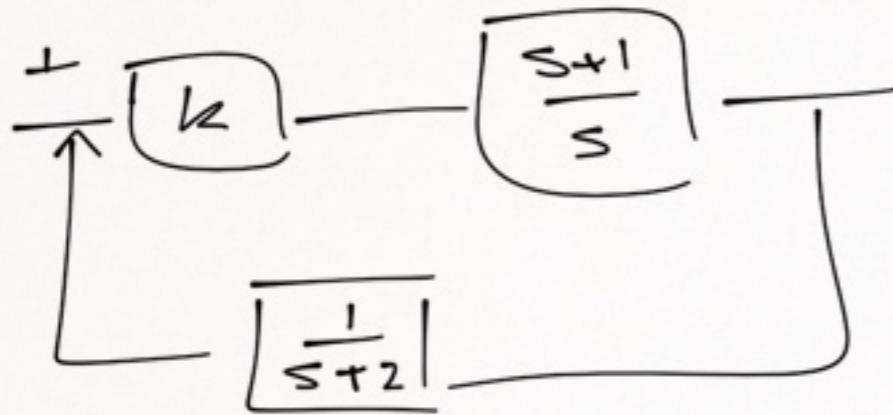
$$|KGH| = 1$$

$$\angle KGH = 180^\circ \pm n360^\circ$$

we can extract many rules from this criteria



$$\angle \frac{C_1}{C_2 C_3} = \angle C_1 - \angle C_2 - \angle C_3$$

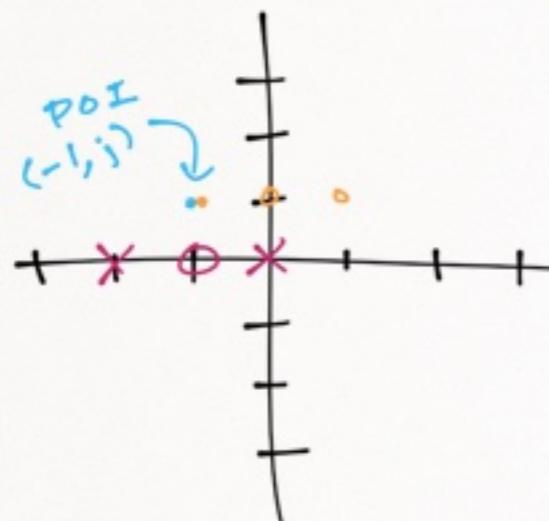


$$KG_H = \frac{k(s+1)}{s(s+2)}$$

CE: $1 + KG_H = \phi$

$$KG_H \Big|_{s=POI} = -1 = 1 < 180^\circ$$

i.e. $|KG_H| = 1, \angle KG_H = 180^\circ$

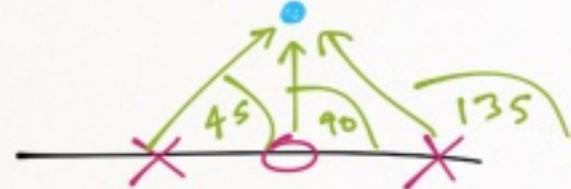


POI

$$\text{ZERO: } s+1 \Big|_{\text{POI}} = (-1+j+1) = j$$

$$\text{POLES: } s \Big|_{\text{POI}} = -1+j$$

$$(s+2) \Big|_{\text{POI}} = -1+j+2 = 1+j$$

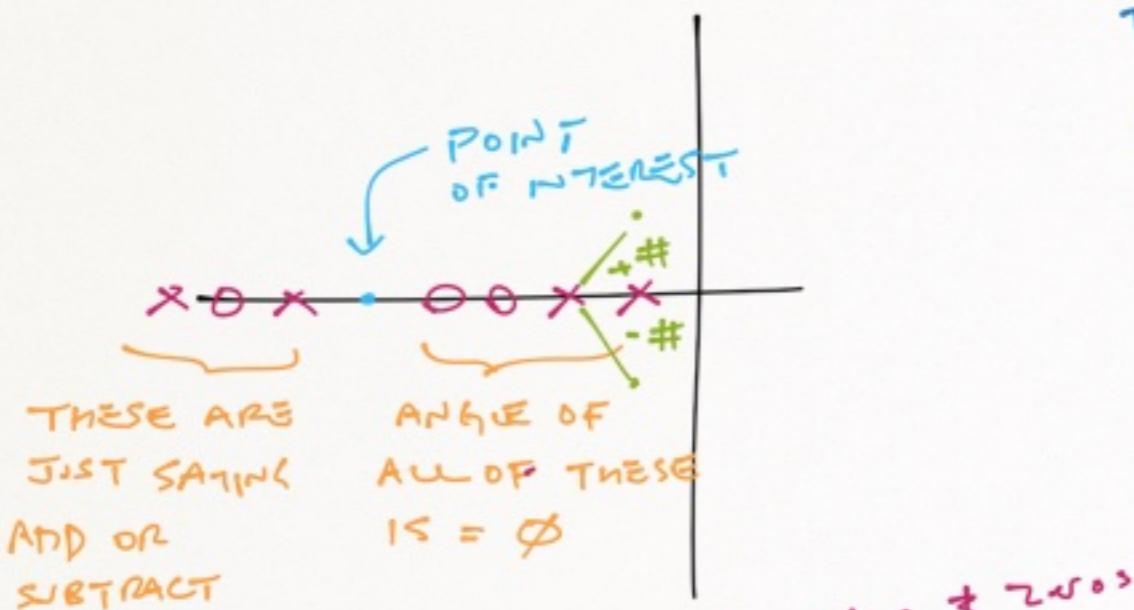


$$\sum \angle \text{ZEROS} = \sum \angle \text{POLES} = 90^\circ - 45^\circ - 135^\circ = -90^\circ \neq 180^\circ$$

\therefore NO, POINT IS NOT ON ROOT LOCUS

how the roots of
system change w/
variation in k

HOW ABOUT REAL AXIS?



root locus
RL exists to left of all odd #'s of poles and/or zeros

ANTITIME COMPLEX CONJUGATES, THEIR ANGLES WILL CANCEL.

→ of poles $\neq 2\pi^\circ$

IF even # right of poi, $\angle = \phi^\circ \rightarrow$ RL doesn't exist
IF odd # right of poi, $\angle = 180^\circ \rightarrow$ RL does exist



More on root locus

Oct 22 2014

$$KGH = \frac{1}{(s+1)(s+2)(s+3)}$$

As $k \rightarrow \infty$, $\alpha_1 = \alpha_2 = \alpha_3$

$$(\#Z - \#P) \alpha = -180$$

$$(\#P - \#Z) \alpha = 180$$

$$\alpha = \frac{180}{xs}$$

$$\frac{180}{xs} = \frac{180}{3} = 60^\circ$$

$$\frac{180 + n360^\circ}{2} = 60 + n120^\circ$$

$60, 180, 300$
we can already see this here

VALUES SATISFYING

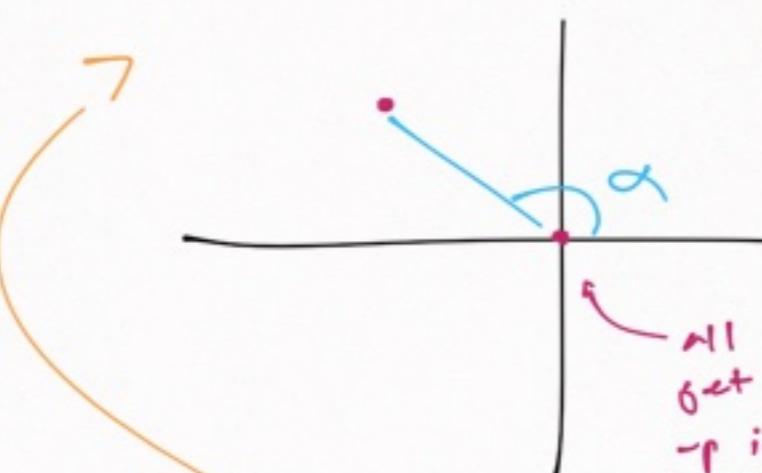
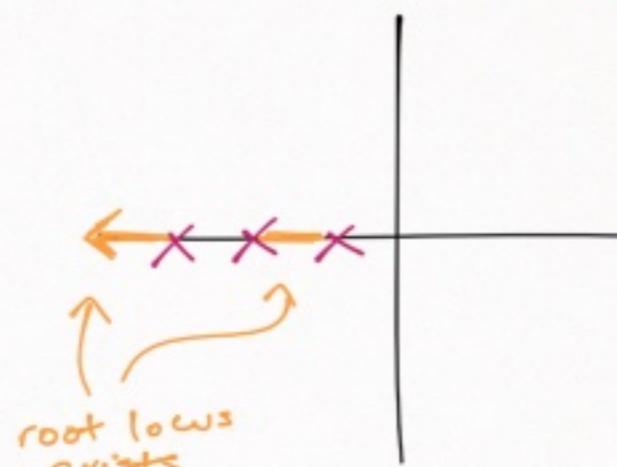
ANGLE CRITERIA

↪ "ASYMPTOTES"

↪ SYMMETRIC $\rightarrow 60 \neq 300$ have to be complex conjugates
 ↪ EVENLY SPACED

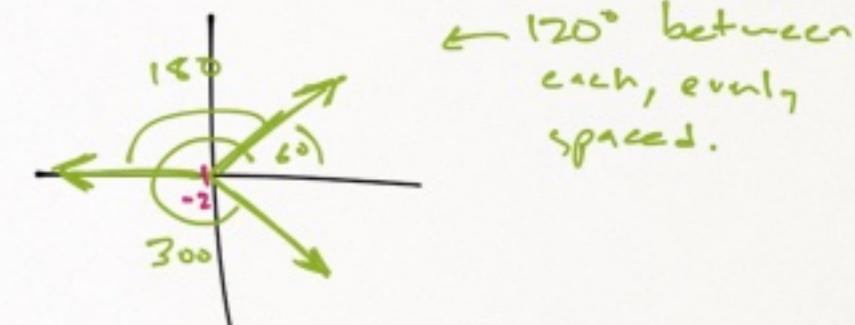
$$\sigma_A = \frac{\sum P - \sum Z}{xs} = \frac{-1 + (-2) + (-3)}{3} = -2$$

R #P - #Z



$k = 10000$

all poles get bunched up into one point when we zoom out of a big value of k



$$CE: 1 + KG_n = \phi$$

$$K = \frac{-1}{GH}$$

SLOPE OF ROOT LOCUS = ∞ @ CORNER

$$\therefore \frac{\delta s}{\delta K} = \infty, \quad \frac{\delta K}{\delta s} = \phi$$

$$\frac{\delta}{\delta s} \left(\frac{1}{G_n} \right) = \phi$$

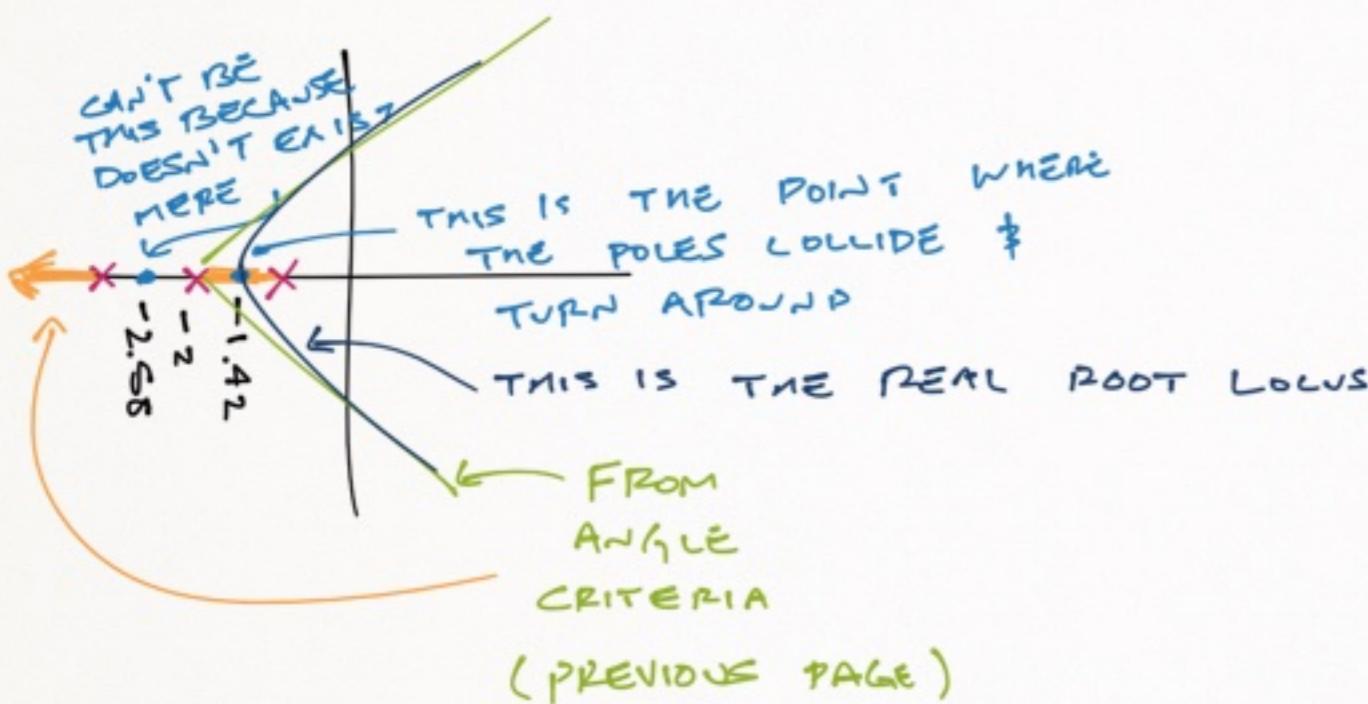
(?)

Ex)

$$GH = \frac{1}{(s+1)(s+2)(s+3)} = \frac{1}{s^3 + 6s^2 + 11s + 6}$$

$$\frac{\delta}{\delta s} \frac{1}{GH} = 3s^2 + 12s + 11 = \phi \quad \rightarrow \quad s = \frac{-12 \pm \sqrt{144 - 132}}{6} = -2 \pm 0.58$$

$$= \begin{cases} -2.58 \\ -1.42 \end{cases}$$

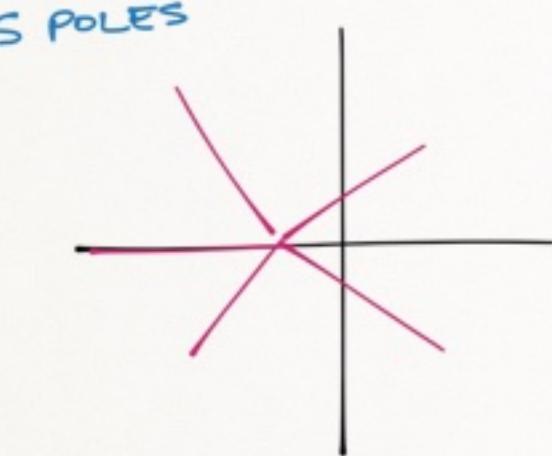
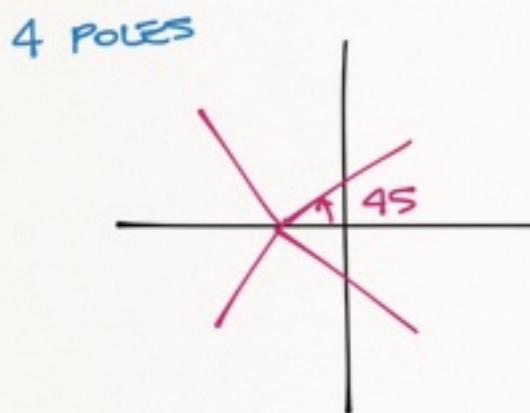
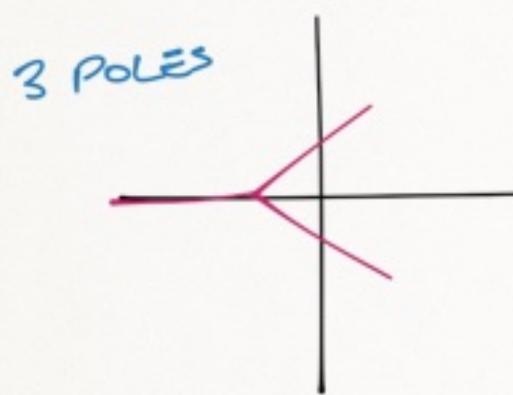


WE KNOW
FROM PREVIOUS
ANGLE CRITERIA
THIS IS WRONG

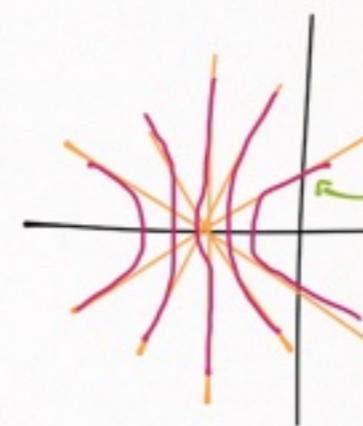
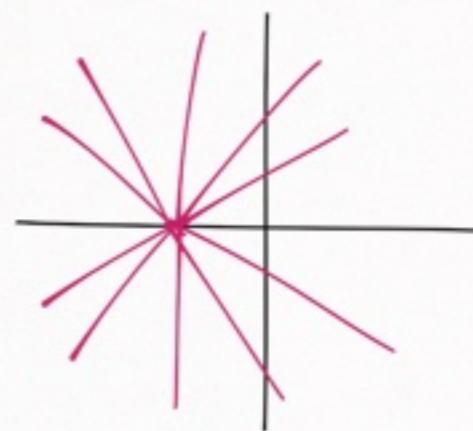
(NOT FALSE, JUST
CORRESPONDS
TO -IVE K,
WHICH WE'RE
NOT CONCERNED
WITH)

$$\phi_A = \frac{2q+1}{xs} \times 180^\circ, \quad q = \text{index } 0, 1, 2 \dots$$

P → k
I +
D



10 POLES



The system goes unstable when poles pass over to right. So it's important to understand @ what k this happens.

BREAK POINT @ -1.42 , FIND K

PLUG INTO C.E. (closed loop)

$$(-1.42)^3 + (-1.42)^2 + 11(-1.42) + 6 + k = 0$$

$$\hookrightarrow k = 0.4$$

WHAT K DOES THIS GO UNSTABLE AT?
R.H. $s^3 + 6s^2 + 11s + 6 + k$

~~$s=0$?~~ No, that's only for finding DC gain @ $\omega=0$

USE ROUTH-HURWITZ TO FIND K THAT MAKES SYSTEM UNSTABLE.

$$1 \quad 11$$

$$6 \quad 6+k$$

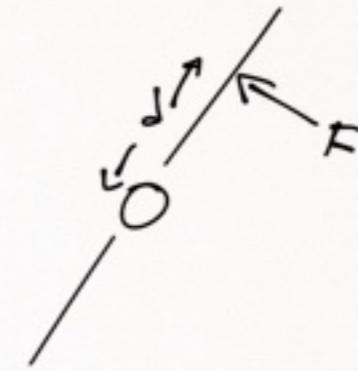
$$\frac{60-k}{6}$$

SYSTEM UNSTABLE
 $\Leftrightarrow k = 60$ or $k = -6$

ONLY CARE ABOUT POSITIVE K VALUES

PROJECT HINTS (Nov. 3rd)

- making sense of control
- remove diode
- figure out units
- assume wind resistance is $F_{\text{point}} @ \frac{\text{propeller length}}{2}$
- start soon

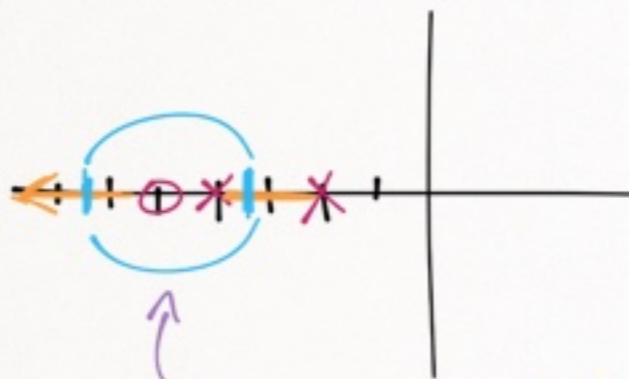


DETERMINE TIME CONSTANT EXPERIMENTALLY

DESIRED ALTITUDE m

$$G_H = \frac{s+5}{s^2 + 6s + 8} = \frac{s+5}{(s+2)(s+4)}$$

DRAW ROOT LOCUS



AND WE JUST
SORT OF CONNECT
THE POINTS. WE
COULD PROVE IT
DOING THE ANGLE
CRITERIA, BUT IT'S
A BUNCH OF WORK.

POLES: -2, -4

ZERO: -5

BY ODD # TEST

BP

$$\frac{d}{ds} \frac{1}{G_H} = \phi \quad \frac{d}{ds} (s+5)^{-1} (s+6s+8) = \phi$$

....

$$s^2 + 10s + 22 = \phi$$

$$s = -6.7, -3.3$$

BOTH VALID B/C
AGREE w/ ODD # TEST

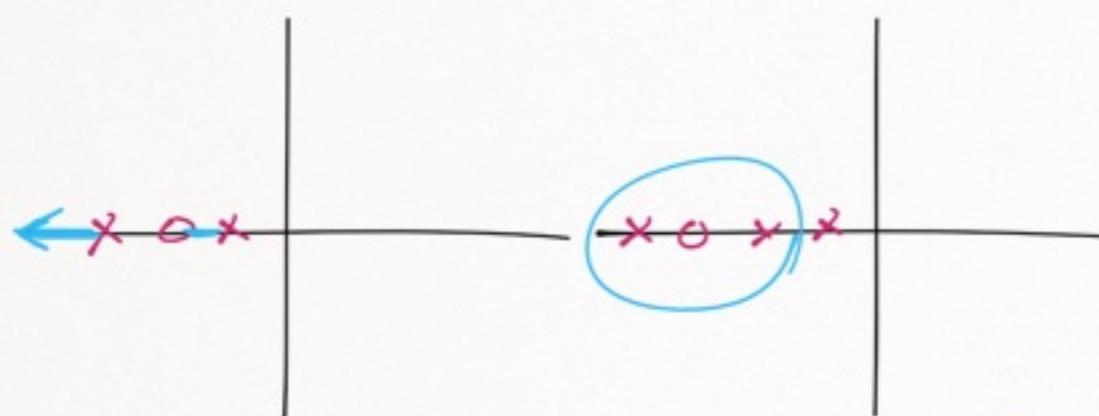
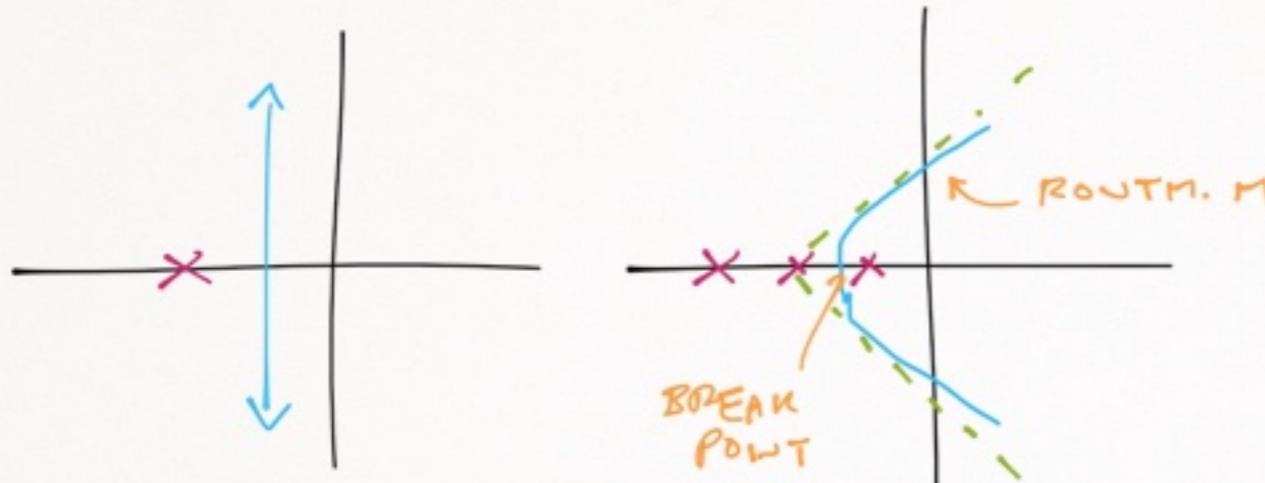
$$xs = \#P - \#Z = 1$$

OF
ASYMPTOTES

Anyway, that's the
root locus.

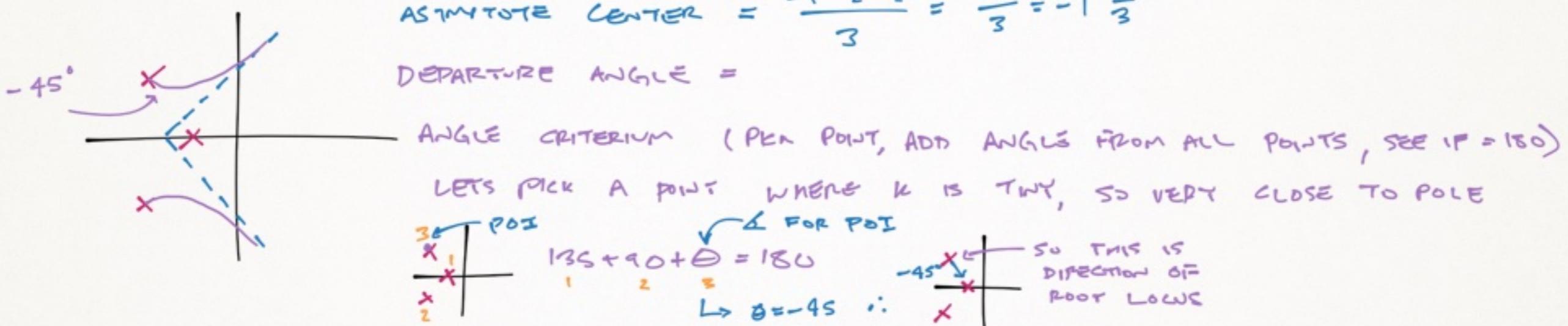
ROOT LOCUS SUMMARY SO FAR

OCT 24 2014



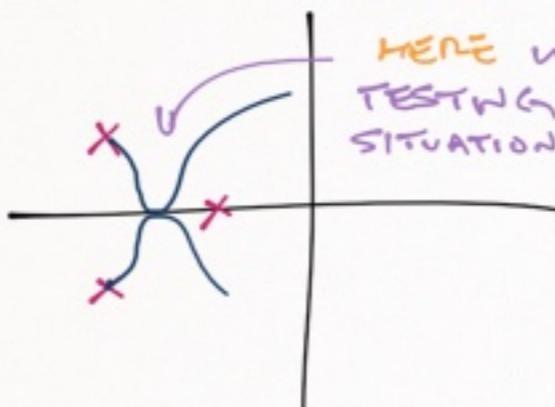
OKAY, WELL WHAT ABOUT COMPLEX POLES?

$$\text{Ex) } \frac{1}{(s+1)(s+2+j)(s+2-j)} = \frac{1}{(s+1)(s^2+4s+5)} \quad \text{Root Locus?}$$



Ex cont'D

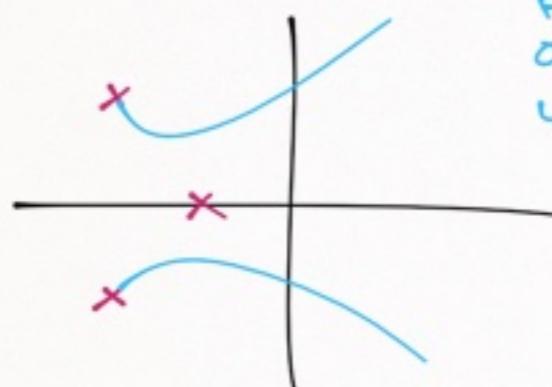
FIND BREAK POINTS $\frac{d}{ds} \frac{1}{Gm} = 3s^2 + 10s + 9 = 0$



HERE WE WERE
TESTING FOR A
SITUATION LIKE THIS,
BUT WE'RE OKAY.
IT DIDN'T HAPPEN.

$$\frac{-10 \pm \sqrt{100-108}}{6}$$

NO BREAK POINTS
ON REAL AXIS

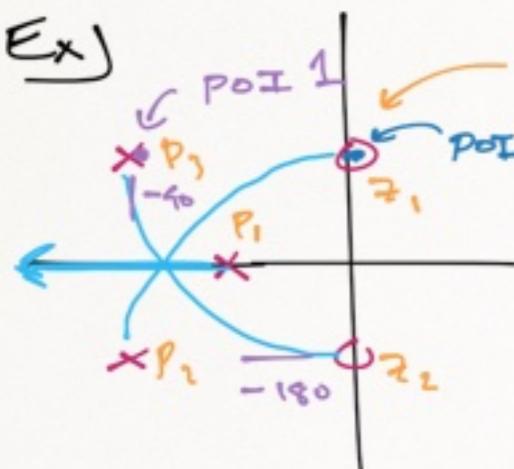


FINAL ANSWER
OF WHAT ROOT
LOCUS LOOKS LIKE.

WHAT ABOUT COMPLEX ZEROS?

$$\frac{k(s^2 + 1)}{(s+1)(s+2+j)(s+2-j)} = kG_H \quad \text{ROOT LOCUS?}$$

Ex)



ZEROS PULL POLES TOWARDS THEM
 → SO WE'RE INTERESTED IN ARRIVAL ANGLE (OF ZERO)
 → THE ZEROS ALSO AFFECT DEPARTURE ANGLE (OF POLE)

POI ANGLE CRITERIA DEPARTURE

$$180 + 135 - 135 - 90 - \theta = 180 \quad (\text{SUBTRACTING POLES, ADDING ZEROS})$$

$$z_1 \quad z_2 \quad p_1 \quad p_2 \quad \begin{cases} \theta = -90 \\ p_1 \end{cases} \quad \hookrightarrow \text{ARBITRARY}$$

POI ANGLE ARRIVAL

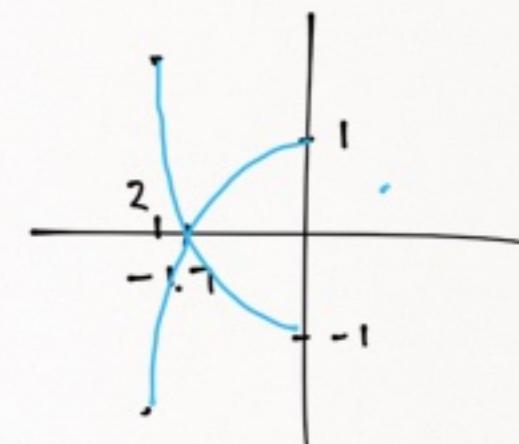
$$\theta + 90 - 45 - 45 - 0 = 180$$

$$z_1 \quad z_2 \quad p_1 \quad p_2 \quad \begin{cases} \theta = 150 \\ p_2 \end{cases}$$

BREAK POINTS

$$\frac{1}{2s} \frac{1}{G_H} = \frac{1}{2s} (s^2 + 1)^{-1} (s^2 + 5s^2 + 9s + 5) = \phi$$

$$\hookrightarrow (s^2 - 3)^2 = \phi \quad s = \pm\sqrt{3} \approx \pm 1.7$$



Root locus exact answer

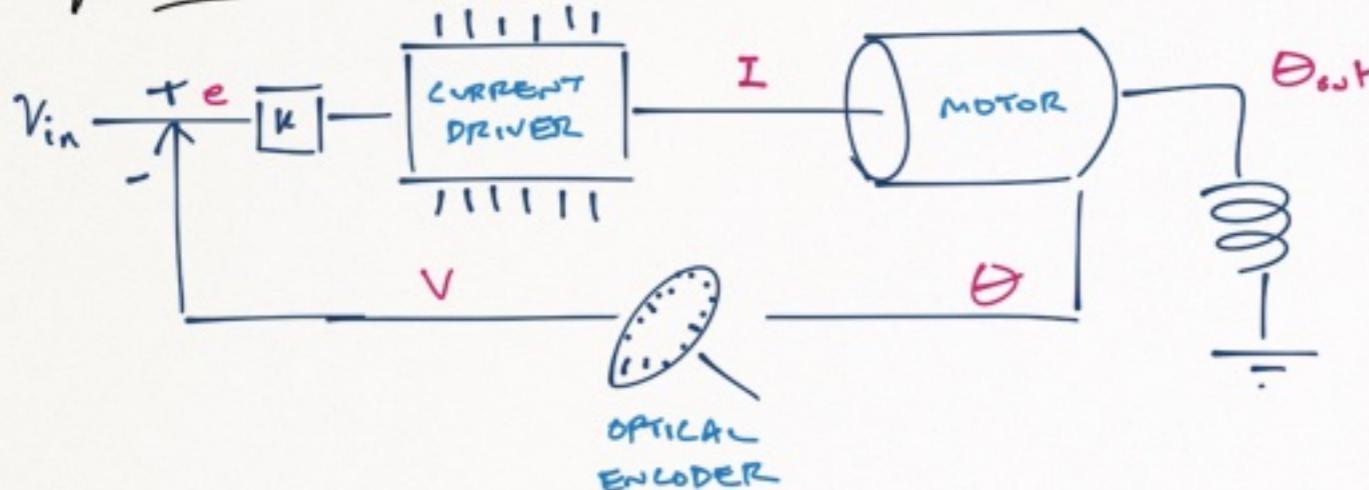
~ CAN'T BE +1.7 BECAUSE
 DOESN'T EXIST

ONLY ONE BREAK
 POINT

∴ BREAK P = -1.7

Ex] FOR THE FOLLOWING SYSTEM, DRAW ROOT LOCUS

OCT 27 2014



$$\text{CURRENT DRIVER: } \frac{1}{s+1}$$

$$\text{MOTOR: } J=1, B=5$$

$$k = 6$$

=

$$\tau = (J_s^2 + Bs + k) \theta$$

$$\frac{\dot{\theta}}{\tau} = \frac{1}{(s+2)(s+3)}$$

MOTOR \rightarrow $\frac{1}{(s+2)(s+3)}$

$$\frac{1}{k - \frac{1}{s+1} \left[\frac{1}{(s+2)(s+3)} \right]}$$

$$KG_{IN} = \frac{k}{(s+1)(s+2)(s+3)} = \frac{1}{s^3 + 6s^2 + 11s + 6}, \quad \frac{Y}{U} = \frac{1}{s^3 + 6s^2 + 11s + 6 + k}$$

$$CE: s^3 + 6s^2 + 11s + 6 + k$$

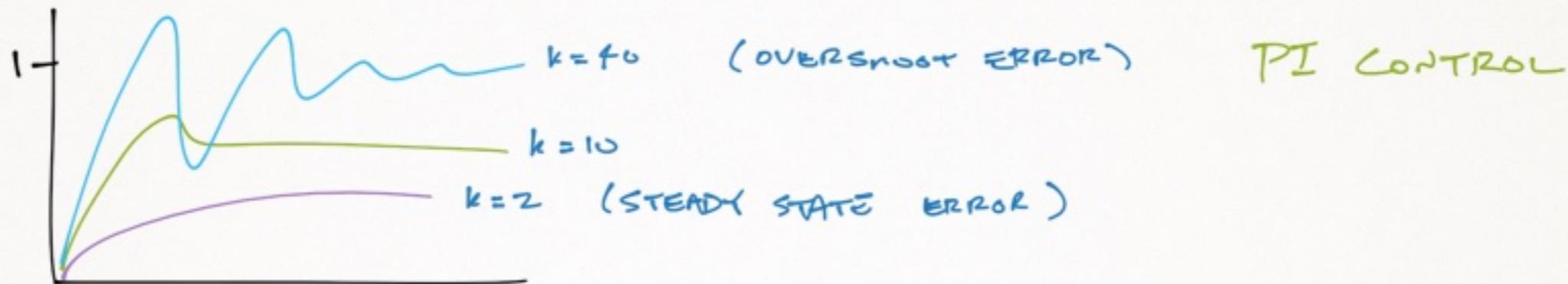
$$DC \text{ GAIN} = K_{DS} = \left. \frac{V}{U} \right|_{s=0} = \frac{k}{k+6}$$

$$\text{STEADY STATE ERROR} = e_{ss} = 1 - K_{DS} = \frac{k+6}{k+6} - \frac{k}{k+6} = \frac{6}{k+6}$$

MAKE k BIG TO
REDUCE ERROR

→ TRADEOFF
B/W OVERSHOOT
& STEADY
STATE ERROR

HOW TO GET RID OF STEADY STATE ERROR → FUNCTION LANDING IN WRONG PLACE

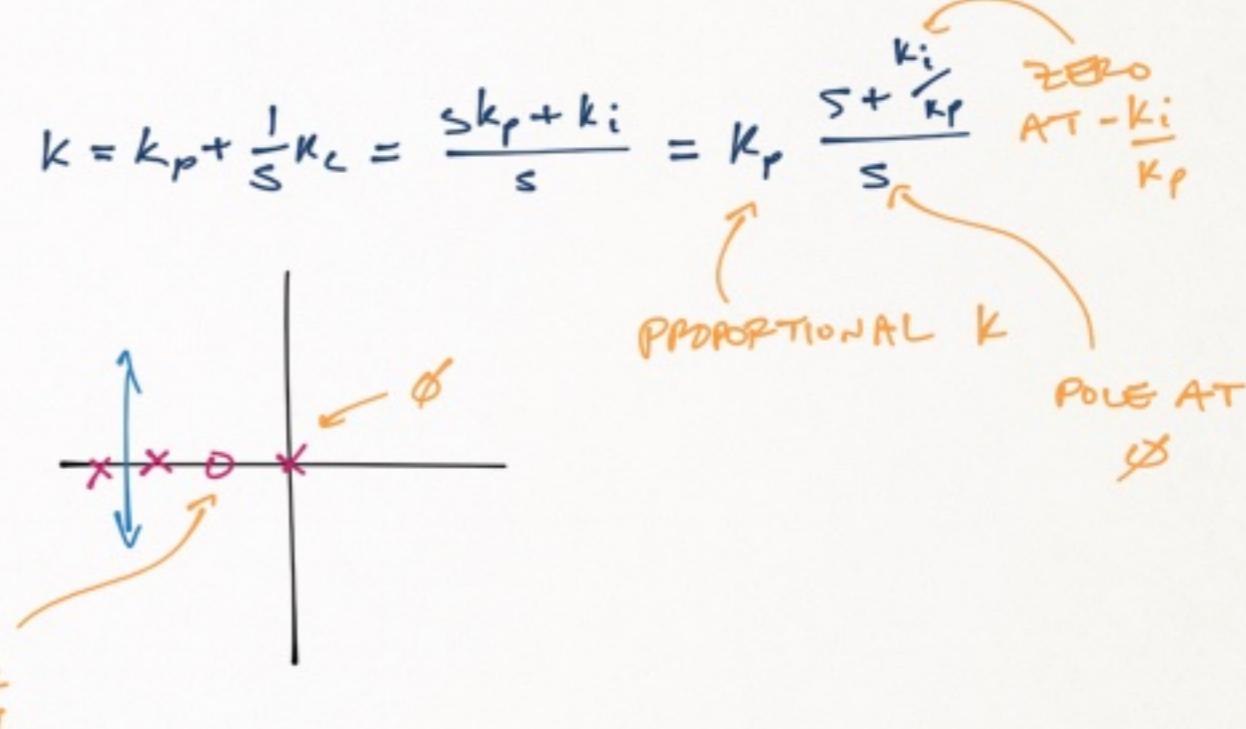
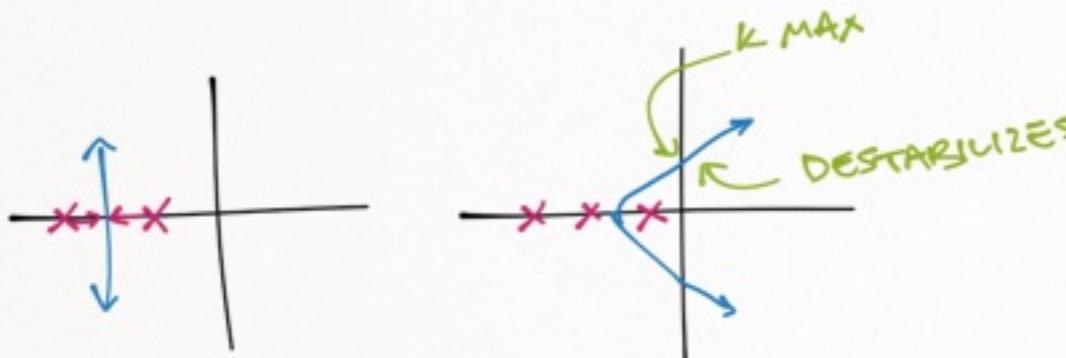


$$KGH = \frac{N}{D} \quad e_{ss} = \frac{1}{1 + \frac{N}{D}} \Big|_{s=\phi} = \frac{D}{D+N} \Big|_{s=\phi} \quad \leftarrow \text{CAN MAKE } D \text{ SMALL OR MAKE } N \text{ LARGE TO REDUCE.}$$

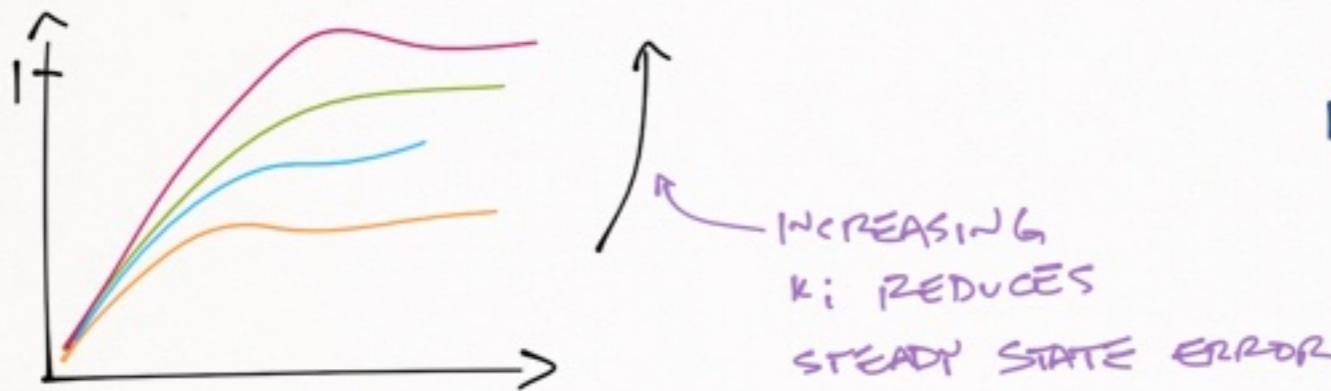
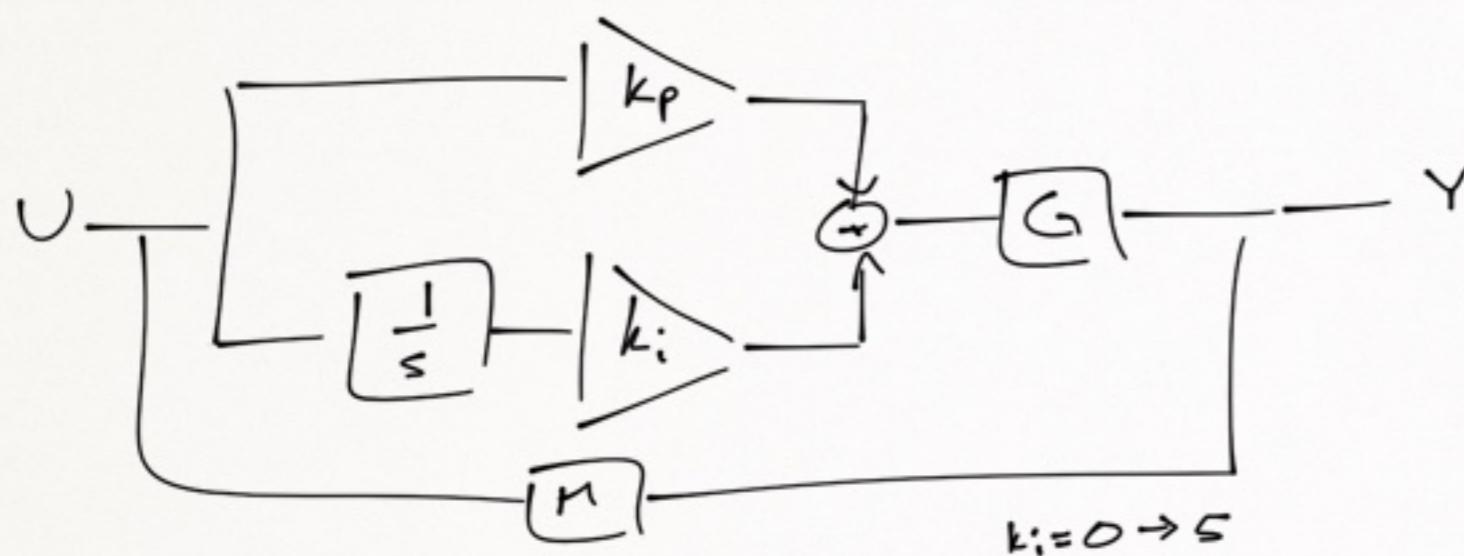
OR SET LOW POLE AT ϕ

$$KGH = \left(\frac{k}{s} \right) \cdot \frac{G_p}{G_D} \cdot \frac{H_N}{H_D}$$

→ sends $e_{ss} \rightarrow \phi$



PI CONTROL



$$k = k_p \frac{s + \frac{k_i}{k_p}}{s}$$

$$k = k_p \frac{s + z}{s} G_H$$

$$k = k_p + \frac{k}{s}$$

Z-N METHOD (ZIEGLER - NICHOLS)

$K_u \rightarrow \max K$ for stability

\therefore proportional control ($k_i = 0$)

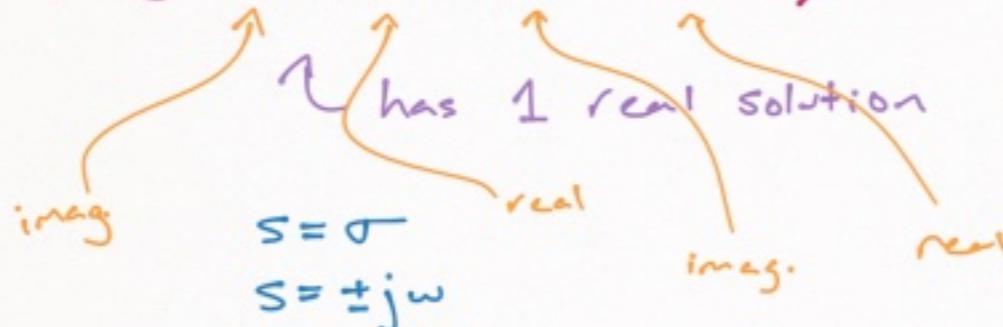
ultimate gain, gain you can have b/f you need an integrator

by R-H, $K_u = 6\phi$

$$T_u = \frac{2\pi}{\omega} @ k = K_u$$

$$CE: s^3 + 6s^2 + 11s + 6 + k_r = 0$$

$$@ K_u CE: s^3 + 6s^2 + 11s + 66 = 0$$



$$6(j\omega)^2 + 6\omega = 0$$

$$-6\omega^2 + 6 = 0$$

$$\omega = \sqrt{11}$$

$$(j\omega)^2 + 11j\omega = 0$$

$$-\omega^2 + 11 = 0$$

$$\omega = \sqrt{11}$$

BIG k_i REDUCES STEADY STATE ERROR FASTER

\hookrightarrow OFFSET IN OSCILLATING

BUT DESTABILIZES

\hookrightarrow MAKES IT OVERSHOOT AND FLUCTUATE

and 2 purely imaginary ones

Z-N FORMULAS

$$k_p = 0.45 K_u$$

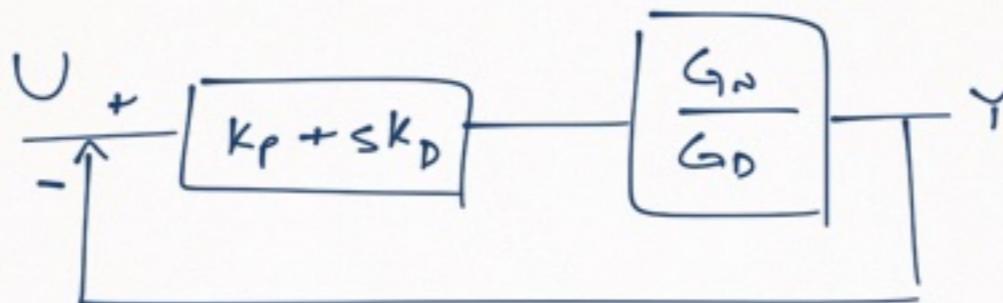
$$k_I = 0.54 \frac{k_u}{T_u}$$

$$T_u = \frac{2\pi}{\sqrt{11}} = 1.9 \text{ s}$$

\hookrightarrow period at frequency where marginal stability (ie. "sustained oscillation")

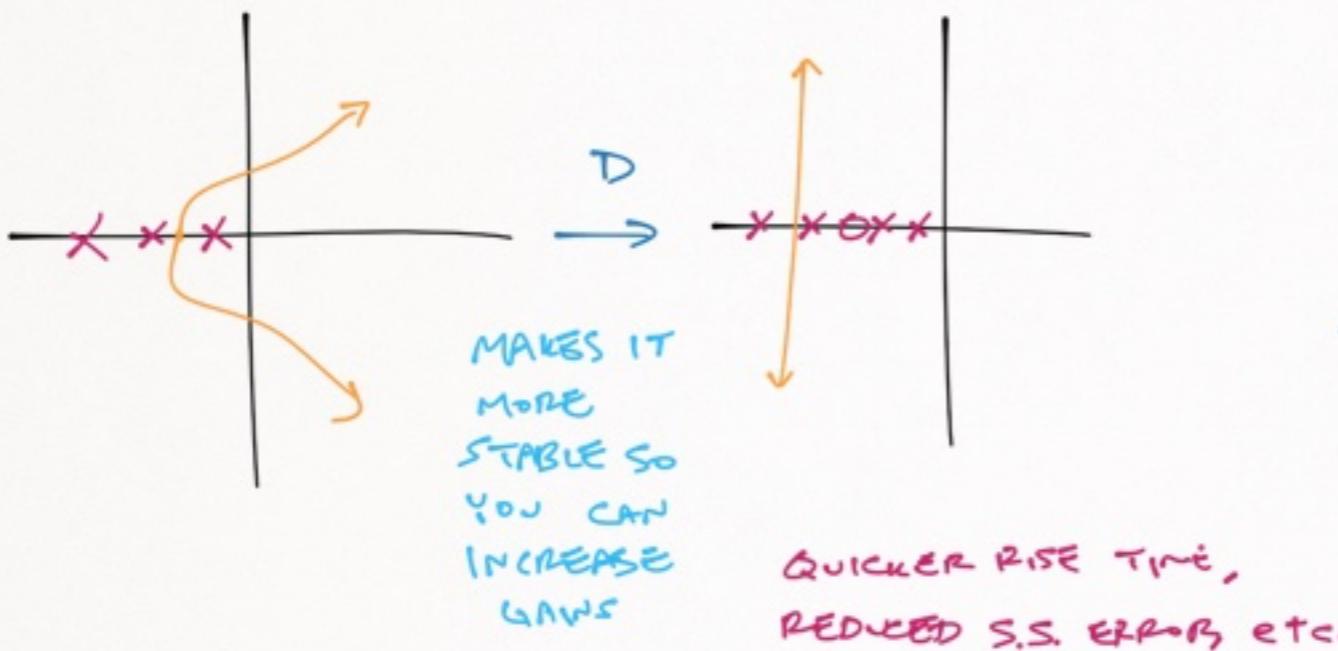
PD CONTROL

$$K = K_p + sK_D$$



$$\frac{Y}{U} = \frac{(K_p + sK_D)G_r}{(K_p + sK_D)G_r + G_D} = \frac{K_D(s + \frac{K_p}{K_D})G_r}{\sim}$$

↪ ADDED A NEW ZERO
 ↪ POLES MOVED



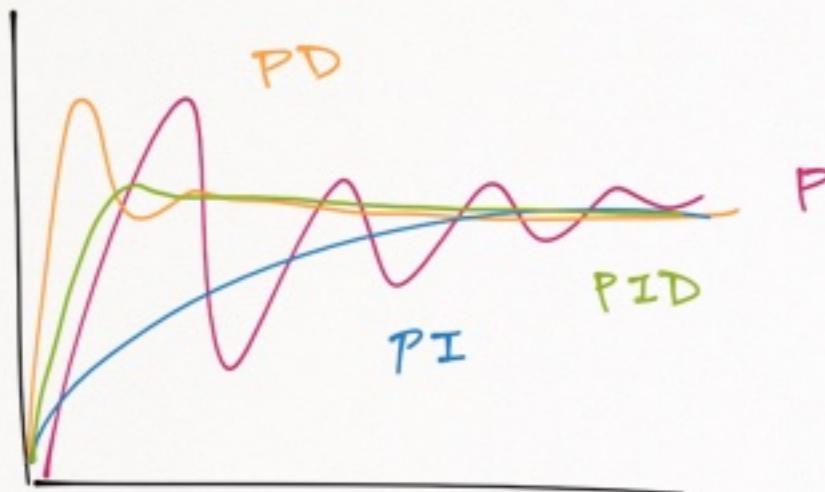
K_D IS A STABILIZING GAIN

$$e_{ss} = \left. \frac{1}{1 + KG_H} \right|_{s=0} = \frac{1}{1 + (K_p + sK_D)G_H} \Big|_{s=0}$$

↑
KD DOES NOT DIRECTLY EFFECT SS ERROR



PI VS. PD



NONE ARE PERFECT,
BUT LET'S COMBINE THEM ALL

Gain	T_R	overshoot	settling time	ss error	
P	↓	↑	↑	↓	
I	↓	↑	↑	∅	
D	↑	↓	↓	n/a	

want everything to be ↓

PID CONTROL

3 POLE SYSTEM EXAMPLE

MOTOR

CURRENT DRIVER

SPRING ← constant force needing to
be readjusted

↳ creates steady state error

integrator tells system to push back until at right level

↳ for example, if gravity is pulling down on robot arm

PID CONTROL

$$G = \frac{1}{(s+1)(s+2)(s+3)} = \frac{1}{s^3 + 6s^2 + 11s + 6}$$

$$\frac{Y}{U} = \frac{k}{s^3 + 6s^2 + 11s + k} \quad \left. \right\} P \text{ CONTROL}$$

$$k = k_p + s k_D \quad \frac{Y}{U} = \frac{k_p + s k_D}{s^3 + 6s^2 + (11+k_b)s + 6 + k_p} \quad \text{closed loop poles}$$

$$k = k_p + \frac{1}{s} k_i + s k_D = \frac{s^2 k_D + s k_p + k_i}{s} = k_D \frac{s^2 + s \frac{k_p}{k_D} + \frac{k_i}{k_D}}{s} = k_D \frac{(s+z_1)(s+z_2)}{s}$$



$\tau = \infty$

GET K_u
GET T_u } set by when all non-proportional gain set to zero

$$K_u = 60$$

$$T_u = 1.9$$

$$K_p = 0.6 K_u$$

$$K_I = 1.2 \frac{K_u}{T_u} \leftarrow \text{differentiator allows you to become more aggressive w/ integrator}$$

$$K_D = 0.75 K_u T_u$$

kind of like adding fake friction

QUIZ #3 REVIEW

E7.1

$$1 + \frac{ks(s+4)}{s^2 + 2s + 2} = \phi$$

CHARACTERISTIC EQU
IF NEGATIVE FEEDBACK



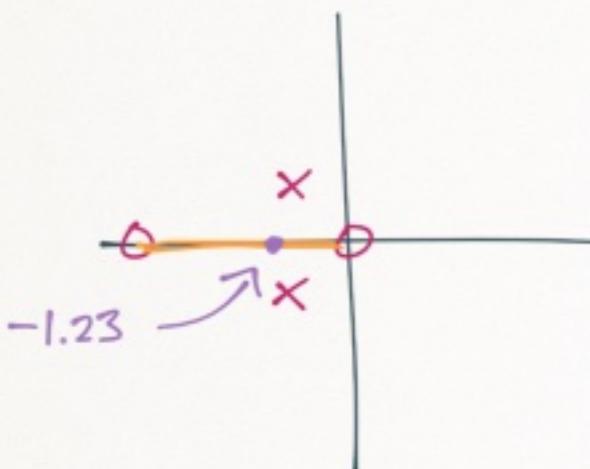
- a) SKETCH ROOT LOCUS
- b) FIND THE GAIN WHEN ROOTS ARE EQUAL
- c) FIND THE TWO EQUAL ROOTS
- d) FIND THE SETTLING TIME WHEN ROOTS EQUAL

CE: $\underbrace{1 + kG(s)H(s)}$

ZEROS @ $s = \emptyset, s = -4$

POLES @ $s = -1 \pm i$

$$= 1 < 180^\circ$$



R.L. EXISTS TO LEFT OF ALL ODD # Z.P

$$\frac{s}{ds} \frac{s^2 + 2s + 2}{s(s+4)} = \phi$$

$$\frac{2s^2 - 4s - 8}{s^4 + 8s^3 + 16s^2} = \phi$$

$$s = 1 - \sqrt{5} \cup s = -1.23(j)$$

b) $1 + \frac{k \cdot s(s+4)}{s^2 + 2s + 2} = \phi$

$$s = -\frac{2k + 1 \pm \sqrt{4k^2 + 2k - 1}}{k+1}$$

WHEN ROOTS ARE EQUAL

$$\sqrt{4k^2 + 2k - 1} = \phi$$

$$\therefore k = \frac{-1 \pm \sqrt{5}}{4} = [0.309 \text{ or } -0.809]$$

Why not?



c) when $k=0.309$

$$\text{roots } s_{1,2} = -\frac{2k+1}{k+1} = \boxed{-1.2361}$$

d) $T_s = \frac{\ln 50 - \ln \beta}{\zeta \omega_n} = \frac{\ln(50) - \ln \sqrt{1-\zeta^2}}{\zeta \omega_n}$

@ $k=0.309$

$$CE: 1 + \frac{0.309s(s+4)}{s^2 + 2s + 2} = \phi$$

$$s^2 + 2s + 2 + 0.309s^2 + 4 \cdot 0.309s = \phi$$

$$1.309s^2 + 3.236s + 2 = \phi$$

$$(s + 1.2361)^2 = \phi$$

$$\therefore \omega_n = 1.2361$$

$\zeta = 1$ CRITICALLY DAMPED

$$T_s \approx \frac{\ln(50)}{1.2361} = 3.2s$$

E7.2

$$H(s) = 1$$

$$G_c(s)G(s) = \frac{k}{s(s+2)(s^2+4s+5)}$$

a) sketch root locus & show dominant roots are $s = -0.35 \pm j0.80$ when $k=6.5$

b) For dominant roots, calc. T_s & overshoot for step input

=====

$$TF = \frac{KG}{1+KGH}$$

$$CE: 1 + KG_H = \infty = 1 + \frac{6.5}{s(s+2)(s^2+4s+5)}$$

$$s(s+2)(s^2+4s+5) = -6.5$$

$$\begin{aligned} s &= (-0.35, \pm 0.8) && \leftarrow \text{this is dominant b/c much closer to } \gamma\text{-axis (8x)} \\ s &= (-2.65, \pm 1.228) \end{aligned}$$

$$T_s = \frac{\ln \zeta \omega_n - \ln \sqrt{1-\zeta^2}}{\zeta \omega_n}$$

$$\text{take } (s+\zeta)(s+\bar{\zeta}) = 0$$

$$(s + 0.35 + j0.8)(s + 0.35 - j0.8) = 0$$

$$s^2 + 0.7s + 0.7625 = 0$$

$$\omega_n = |(0.35, 0.8)| = 0.873$$

$$\zeta = \frac{0.7}{2(0.873)} = 0.4$$

$$T_s \approx \frac{4}{\zeta \omega_n} = \boxed{11.46 \text{ s}}$$

$$\% \text{ overshoot} = e^{\frac{-\zeta \pi}{\beta}} = e^{\frac{-0.4\pi}{\sqrt{1-0.4^2}}} = \boxed{25.38\%}$$

E7.3 $G_c(s)G(s) = \frac{k(s^2 + 4s + 8)}{s^2(s+4)}$

HAS NEGATIVE UNITY FEEDBACK
 SHOW $K=7.35$ FOR
 $s = -1.3 \pm j 2.2$, $\zeta = 0.5$

$\underline{\underline{}}$

$H = 1$

$$1 + KG_H = \emptyset = 1 + 7.35 \frac{s^2 + 4s + 8}{s^2(s+4)}$$

\swarrow

$s = (-1.3, \pm 2.244)$
 $s = -8.7$

E7.4 $G_cG_H = \frac{K(s+1)}{s^2+4s+5}$

FIND χ OF DEPARTURE
 FIND ENTRY POINT OF R.L. UNITY
 ON REAL AXIS

$\underline{\underline{}}$

sum of angles to POI from poles $\neq 2\pi = 180^\circ$

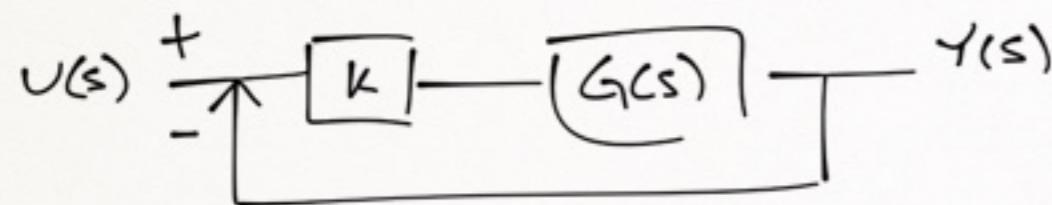
$$1 + KG_H = \emptyset = 1 + \frac{K(s+1)}{s^2+4s+5}$$

$$\frac{d}{ds} \frac{1}{G_H} = \emptyset = \frac{d}{ds} \frac{s^2+4s+5}{s+1} = \frac{s^2+2s-1}{(s+1)^2}$$

$$\rightarrow s = -1 \pm \sqrt{2}$$

$$\rightarrow s = \boxed{-2.4}$$

EXTRA PROBLEMS



$$CE: s^3 + (k-4)s^2 + 10ks + 29k = 0$$

find k for which system is stable
 draw root locus

ROUTIN HURWITZ

$$1 \quad 10k$$

$$k-4 \quad 29k$$

$$m_1$$

$$m_1 = -\frac{1}{b} \begin{vmatrix} a & c \\ b & d \end{vmatrix} = \frac{10k^2 - 69k}{k-4} > 0$$

$$\hookrightarrow k > 0 \neq k < 4 \quad || \quad k > 6.9$$

$$E7.5 \quad G_c(s)G(s) = \frac{s^2 + 2s + 10}{s^4 + 38s^3 + 515s^2 + 2950s + 6000}$$

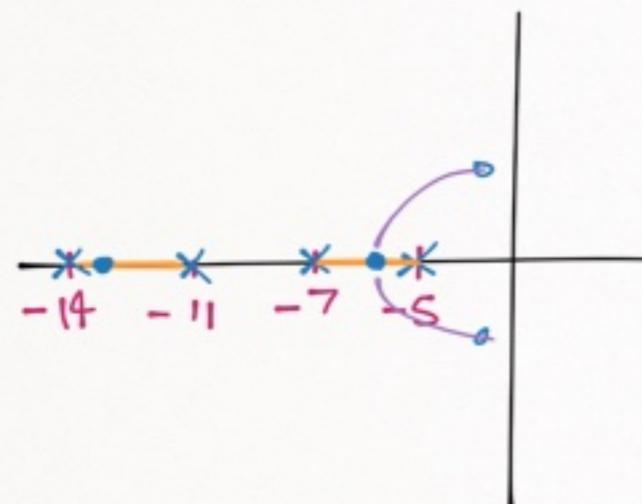
FIND BREAKAWAY POINTS ON REAL AXIS
 FIND ASYMPTOTE CENTROID
 FWD VALUES OF K AT BREAKAWAY POINTS

=

→ ZEROS @ $(-1 \pm 3j)$

CE: $1 + KG_H = \emptyset \rightarrow$ POLES @ $-5.2, -7.05, -14.25, -11.5$

$\frac{d}{ds} \frac{1}{G_H} = \emptyset \rightarrow$ BREAK POINTS @ $-5.89, -8.73, -13.0$



→ DOESN'T
FIT ODD/EVEN
REQUIREMENTS

$$E7.6 \quad G_c G = \frac{20K}{s(s^2 + 20s + 100)}$$

SKETCH ROOT LOCUS
FIND VALUE RESULTING IN UNSTABLE SYSTEM

=

$$1 + KG_H = \phi$$

$$\frac{d}{ds} \frac{1}{G_H} = \phi \rightarrow s = -3.3, -10$$

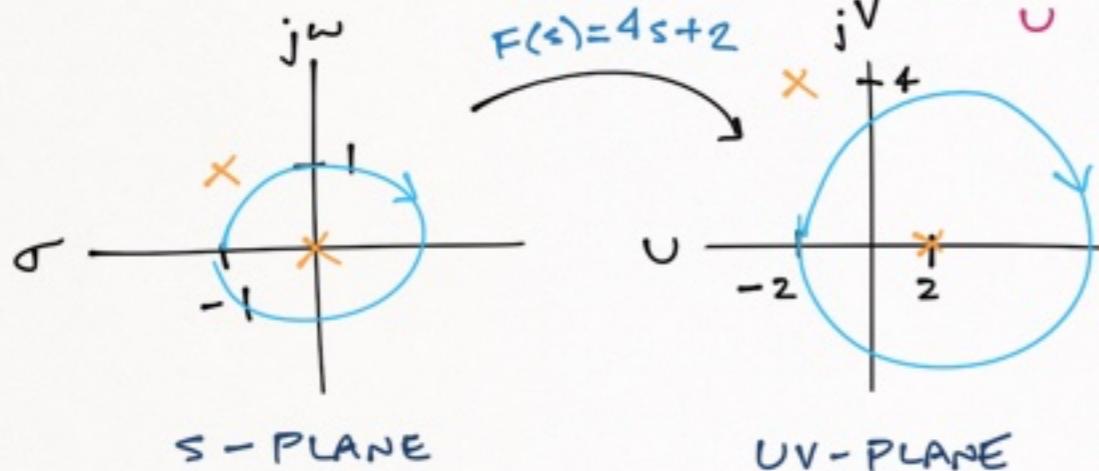
NYQUIST CRITERIA

↳ MEASURE OF INSTABILITY

MATHEMATICAL BACKGROUND

Anytime you have function of $s = (\sigma + j\omega)$, $F(s)$ is also a complex # $(u + jv)$

$$F(s) = 4s + 2 = 4(\sigma + j\omega) + 2 = 4\sigma + 2 + j4\omega$$



scales by 4
≠ shifts right by 2

NOV 7, 2014

EXAMPLE OUT OF TEXTBOOK

$$B = 1 + j\phi$$

$$A = 1 + jl$$

$$H = \phi + jl$$

$$G = -1 + jl$$

$$F = -1 + j\phi$$

$$F(s) = \frac{s}{s+2}$$

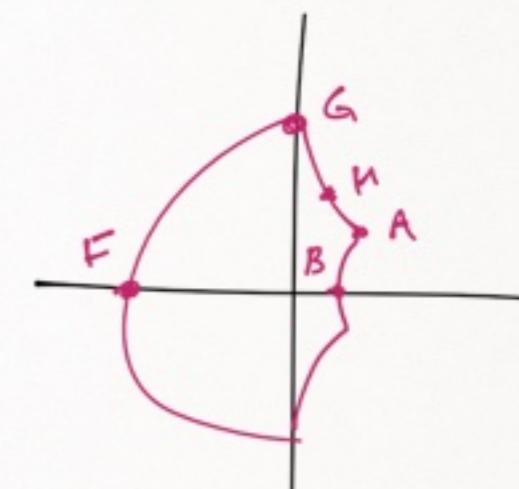
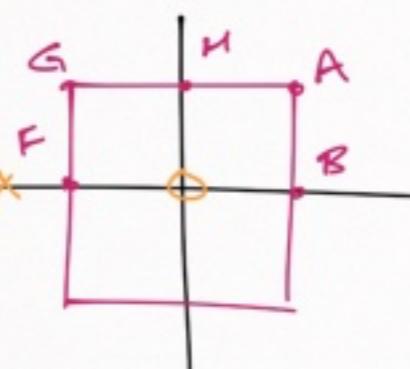
$$B \rightarrow \frac{1}{3}$$

$$A \rightarrow 0.4 + j0.2$$

$$H \rightarrow 0.2 + j0.4$$

$$G \rightarrow j$$

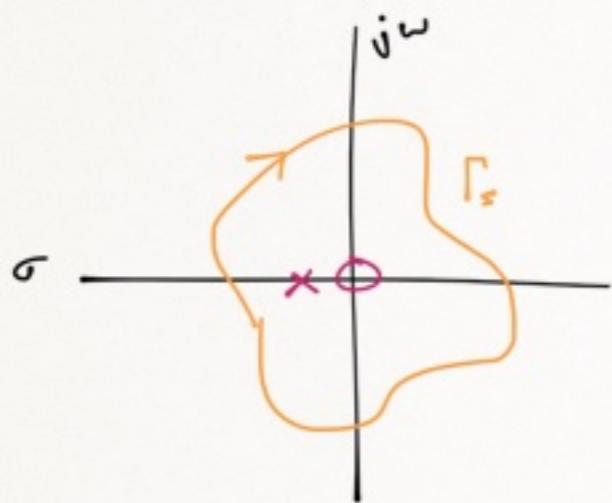
$$F \rightarrow -1$$



pick a few points, transform the points, then connect the dots

CAUCHY'S THEOREM

S-PLANE

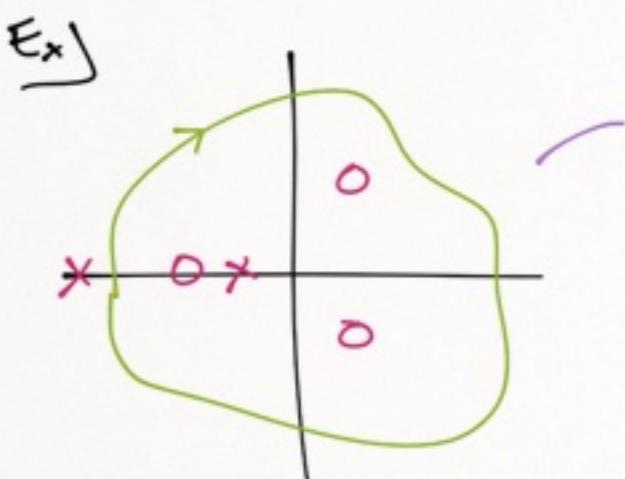


IF S-PLANE CONTOUR

- ① CLOSED
- ② CLOCKWISE
- ③ DOES NOT INTERSECT $+P/z$ OF $F(s)$

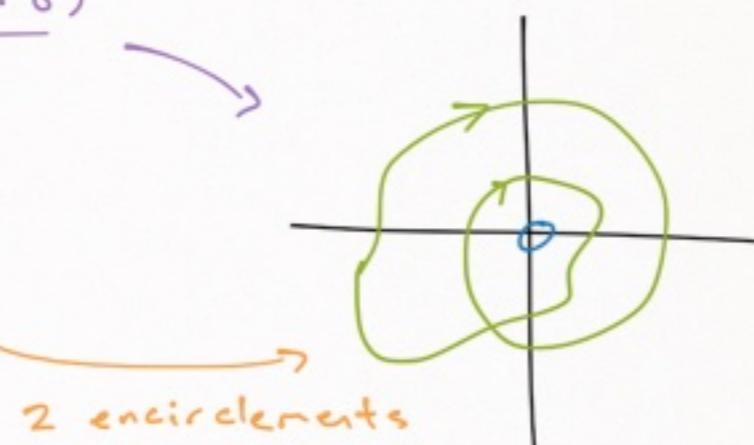
THEN U-V PLANE CONTOUR, $\Gamma_s \leftarrow$ ^{uppercase} _{gamma}

- ① $z = \# \text{ZEROS}$ INSIDE Γ_s
- ② $P = \# \text{POLES}$ INSIDE Γ_s
- ③ $N = z - P = \# \text{CLOCKWISE ENCIRCLEMENTS}$
OF ORIGIN



$$\frac{(s+2)(s^2+4s+8)}{(s+1)(s+5)}$$

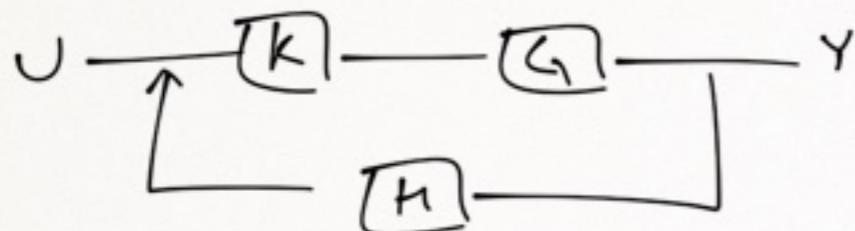
$$N = 3 - 2 = 1$$



2 encirclements

INSTABILITY HAPPENS WHEN
CLOSED-LOOP POLES IN RNP
→ OPEN-LOOP POLES ARE
CLOSED-LOOP POLES WHEN $K = \emptyset$

THIS THEOREM LETS US SEE IF THERE ARE ANY P/z
IN THE ENTIRE RNP IF WE CHOOSE A GIANT CONTOUR.



IF YOU HAVE POLES IN THE RHP, IT'S GOING TO BE PRETTY UNRESOLVABLE.

IF ONLY ONE, YOU MIGHT BE ABLE TO ↑K UNTIL IT'S IN THE LHP.

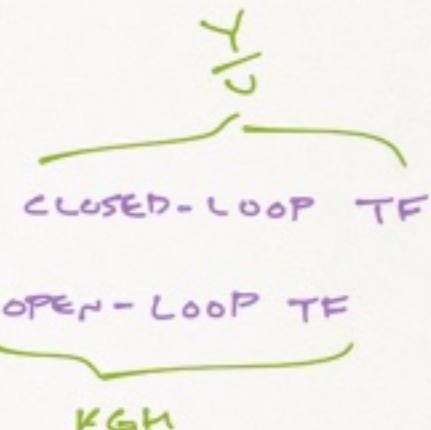
IF ALL O-L POLES IN LHP:

$$N = \# C-L \text{ IN RHP}$$

$$\frac{Y}{U} = \frac{KG}{1+KGH} = \frac{KG_N H_D}{KG_N H_N + G_0 H_D}$$

$$KGH = \frac{KG_N H_N}{G_0 H_D}$$

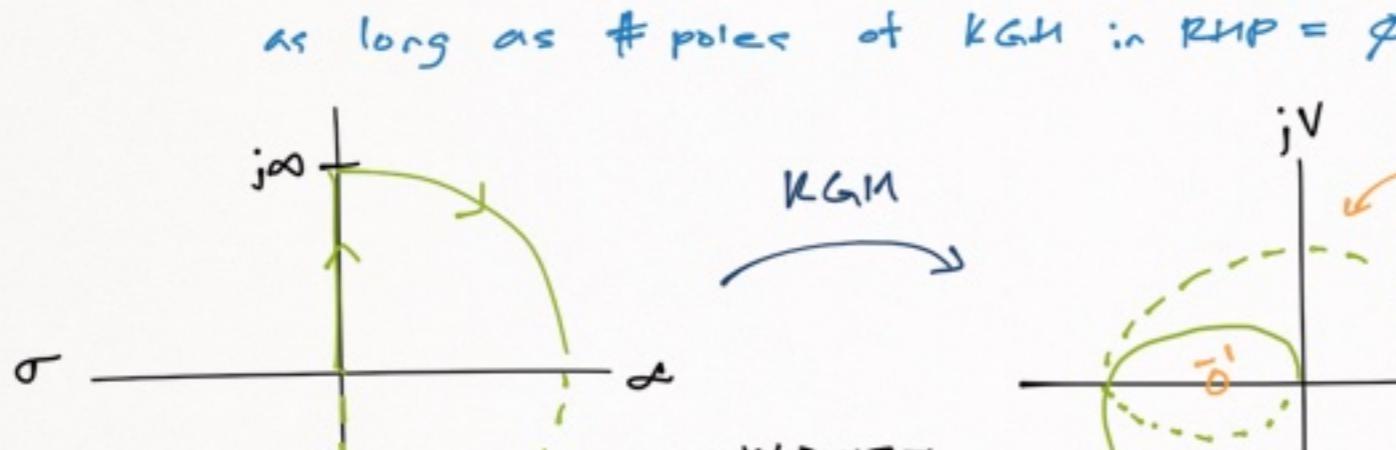
$$1+KGH = \frac{G_0 H_D + KG_N H_N}{G_0 H_D}$$



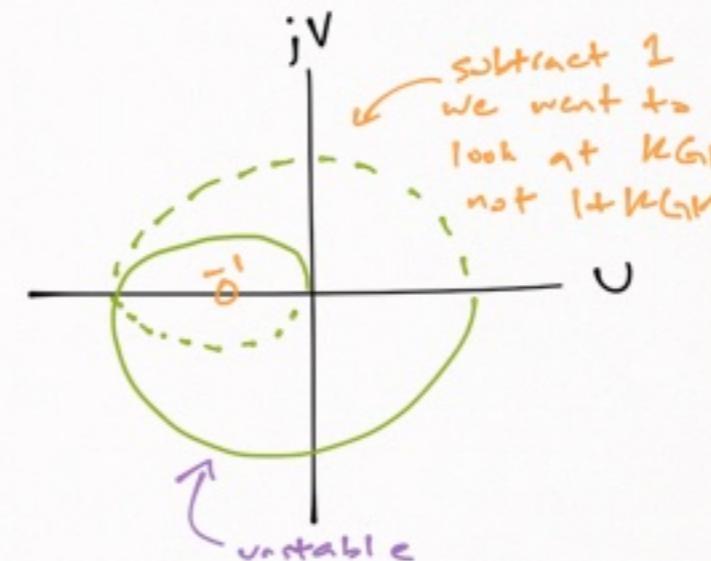
ZEROS OF $1+KGH$ = POLES OF CLOSED-LOOP TF

POLES OF $1+KGH$ = POLES OF OPEN-LOOP TF

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don't actually have to do this half b/c mirror image

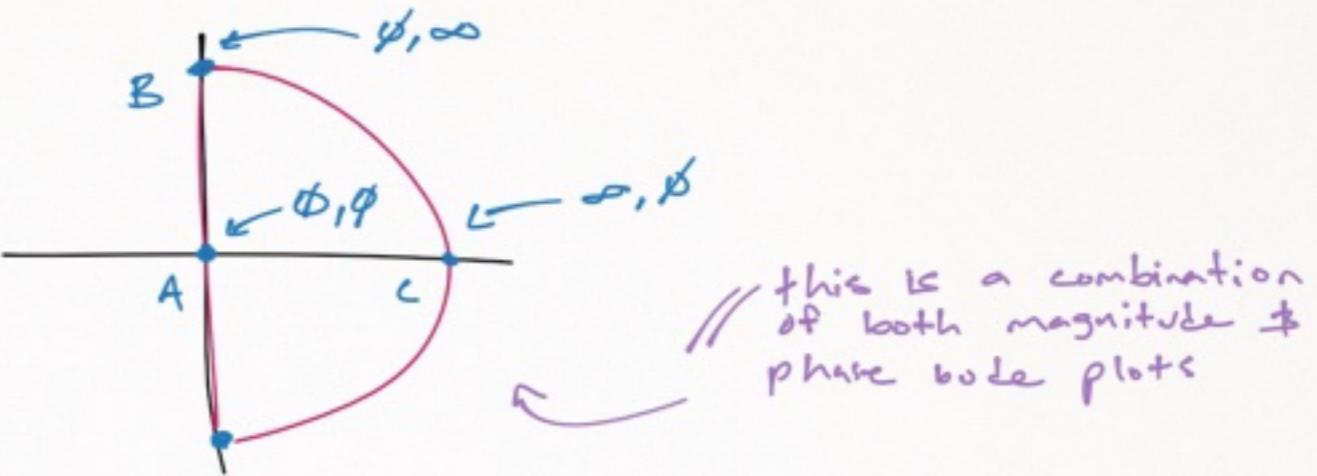


// SPRING VS. GRAVITY
↳ GRAVITY FORCE DOES NOT PUSH HARDER WHEN STRETCHED FURTHER

Pi GAW COUNTERS SPRING, BUT OUR HELICOPTER DOESN'T HAVE SPRING

$$\text{Ex}) \quad K_{GM} = \frac{1000}{(s+1)(s+2)}$$

DO NYQUIST CRITERIA



s	A	B	C
real	100	-△	△
imag.	∅	∅	∅
polar co-ords	mag.	100	△
phase	∅	180°	∅

$$A = \frac{1000}{1(10)} = 100$$

$$B = \frac{1000}{(1+j\omega)(10+j\omega)} = \frac{1000}{-\infty^2} = -\Delta$$

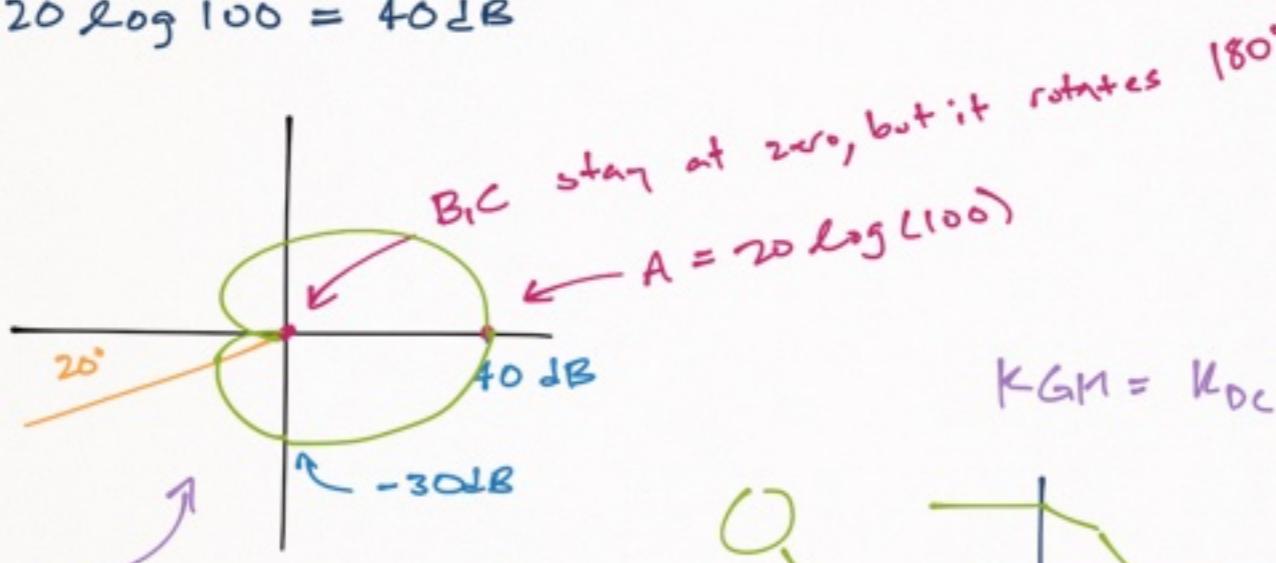
$$C = \frac{1000}{\infty^2} = \Delta$$

negative tiny

NOTE:
tiny not zero
b/c tiny has phase

You want to plot in dB $\xrightarrow{\text{Bell}}$ power ratio

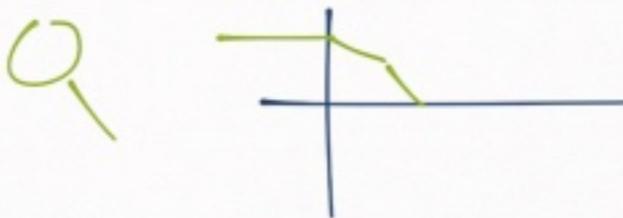
$$20 \log 100 = 40 \text{ dB}$$

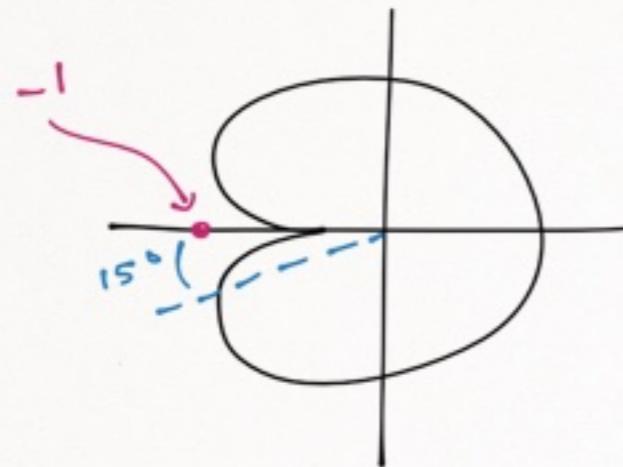


FROM A to B $s=j\omega$
no σ \nwarrow envelope \nearrow oscillation

$$K_{GM} = K_{DC} \frac{(1 + \frac{s}{z_1})(1 + \frac{s}{z_2}) \dots}{(1 + \frac{s}{P_1})(1 + \frac{s}{P_2}) \dots} = 100 \frac{1}{(1 + \frac{s}{z_1})(1 + \frac{s}{z_2})}$$

-20 @ $\phi \neq -10$





root locus
gain you can add before instability

NOV 12 2014

$$KG_m = \frac{100}{(1+s)(1+\frac{s}{10})}$$

INFORMATION FROM NYQUIST CRITERIA

- gain margin ← this is also what the root locus gives you
- phase margin
- crossover margin

GAIN MARGIN

↪ gain at phase = 180°

PHASE MARGIN

↪ phase at gain 20 dB

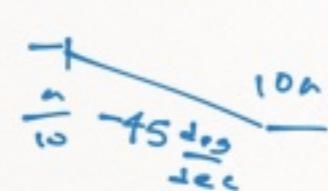
IF MAG comes in AT $-20 \frac{\text{dB}}{\text{dec}}$

PHASE comes in AT 90°

" " $40 \frac{\text{dB}}{\text{dec}} \rightarrow 180^\circ$

" " $-20 \frac{\text{dB}}{\text{dec}} \rightarrow -90^\circ$

" " $-40 \frac{\text{dB}}{\text{dec}} \rightarrow -180^\circ$



PHASE MARGIN

→ how much added phase before instability
↪ rotation of Nyquist

GAIN MARGIN

→ how much added gain before instability

CROSSOVER MARGIN

→ frequency at which the magnitude goes to one

YOU CAN USUALLY GET THIS INFORMATION FROM BODE PLOT

phase @ 1° ← phase margin
↪ ASYMPTOTICALLY SO MAYBE ∞
gain @ phase 180° ← how far below x-axis gain is

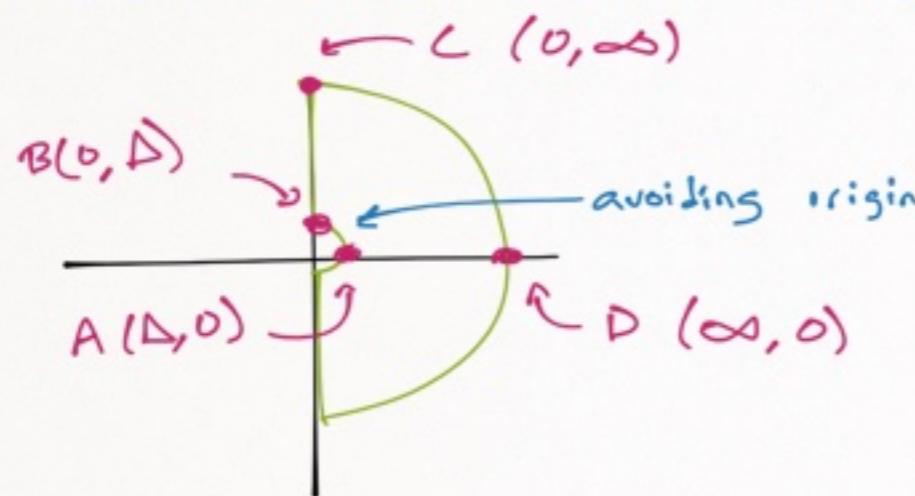


WHAT IF POLE AT ZERO?

$$\text{Ex) } K_{GH} = \frac{100}{s(1 + \frac{s}{100})}$$

One criteria of Nyquist was that it doesn't touch any poles or zeros.

so define a new curve that detours around pole at zero

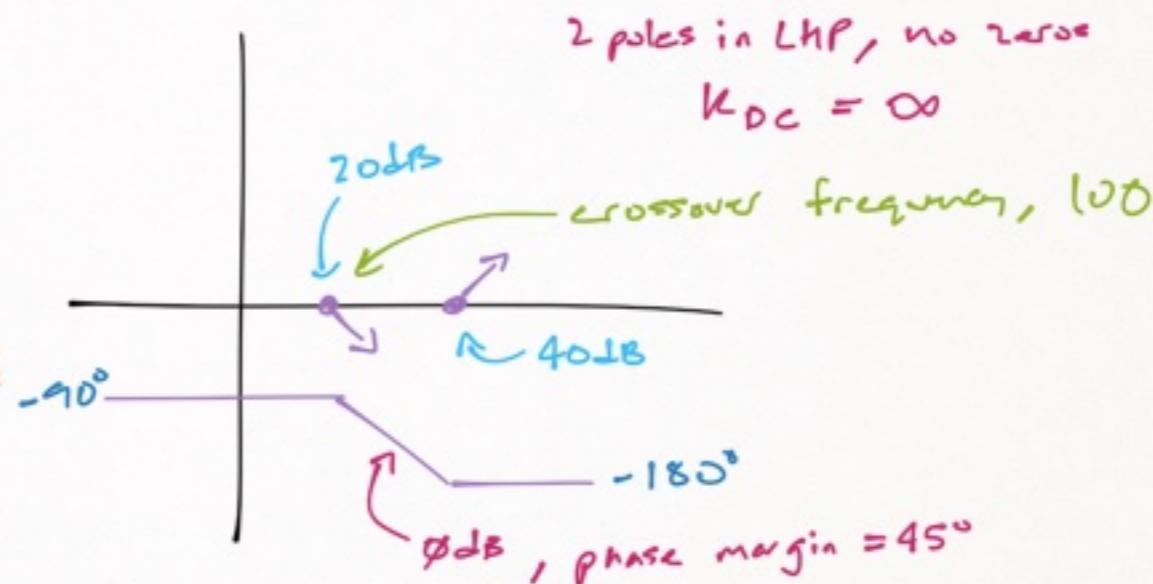
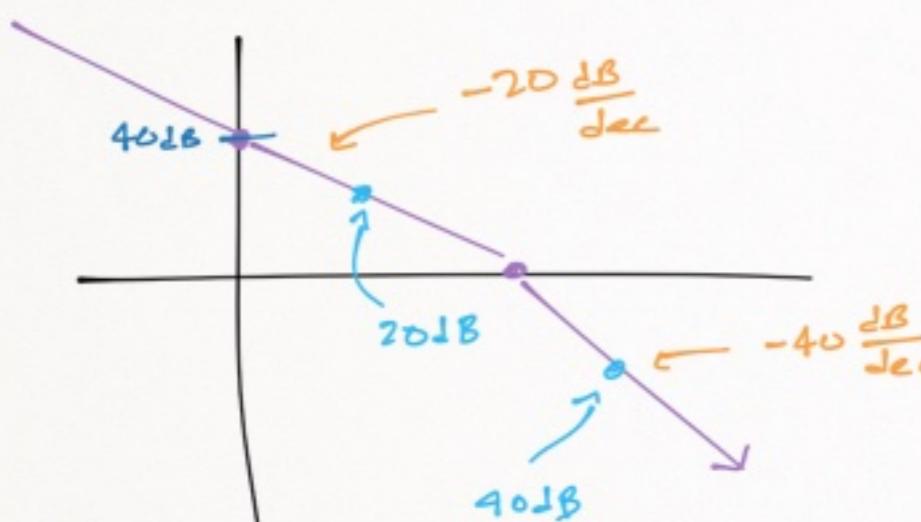


s	K_{GH}	$ K_{GH} $	$\angle K_{GH}$
A	Δ	$+\infty$	∞
B	$j\Delta$	$\frac{100}{j\Delta} = \frac{100}{\Delta e^{j90^\circ}} = \infty e^{-j90^\circ}$	-90°
C	$j\infty$	$\frac{100}{-\infty^2} = -\Delta$	Δ
D	∞	Δ	0°

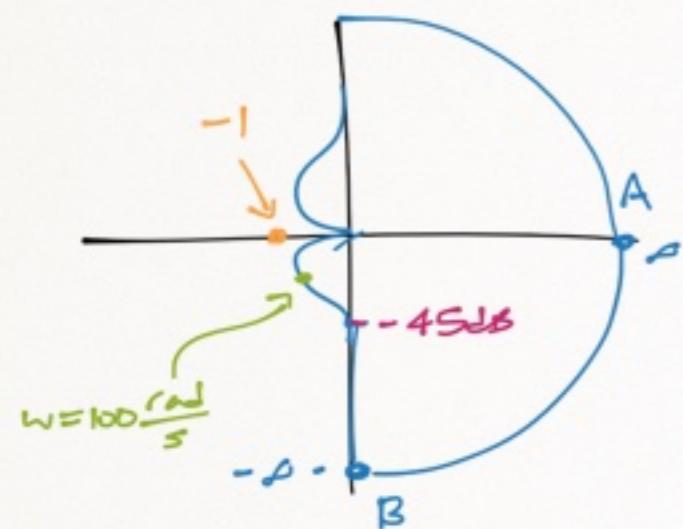
$$\text{DC GAIN} = K_{GH}(s=0) = \frac{1}{\phi} = \infty$$

~~K_{DC}~~ $K_{\omega} \approx 1 = 10^\circ$

GET BODE PLOTS



Ex) CONT'D DRAW NYQUIST



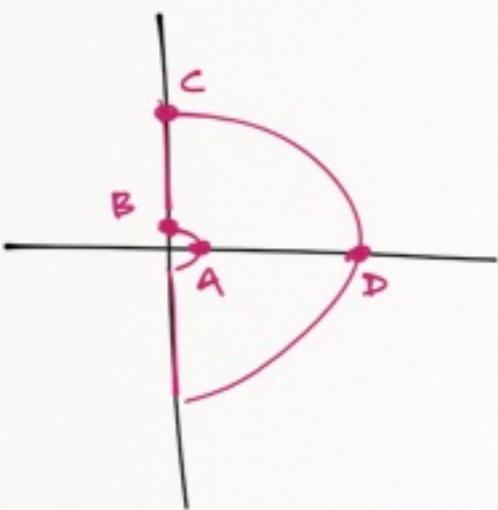
$20 \text{ dB} @ -90^\circ$

$-40 \text{ dB} @ \approx -180^\circ$

$$\omega = 100 \frac{\text{rad}}{\text{s}}$$

Ex) $K_{GN} = \frac{1000}{s(s+1)(s+100)}$ NYQUIST PLOT

EXCESS OF THREE POLES \therefore THERE WILL BE A GAIN MARGIN

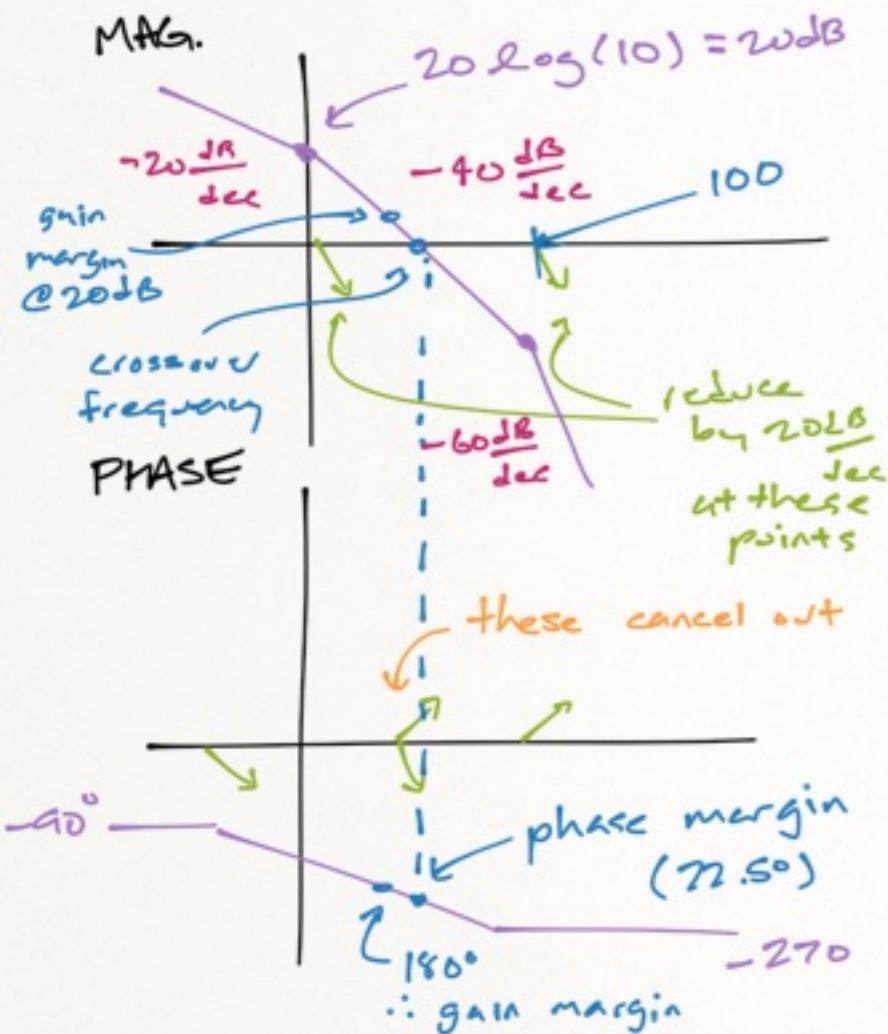


$$K_{GN} = 10 \frac{1}{s(1+\frac{s}{1})(1+\frac{s}{100})}$$

$K_{\omega=1}$ MAGNITUDE OF 10 @ $\omega=1$

A: $\frac{10}{\Delta(X)} = \frac{10}{\Delta} = \infty \times 0^\circ$

D: $\frac{10}{\infty \times 180^\circ} = \frac{10}{\infty^3} = \Delta \times 0^\circ$

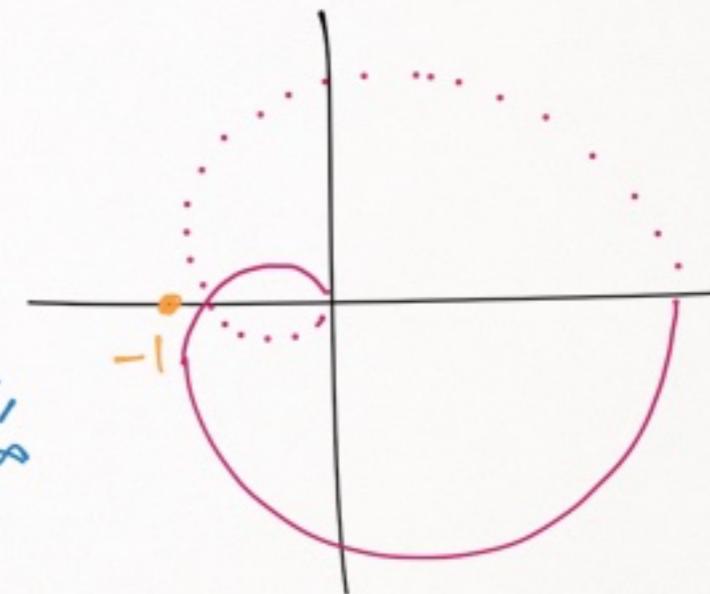


B: don't bother doing this
you can just read this off the bode plot which you're going to need to get anyway

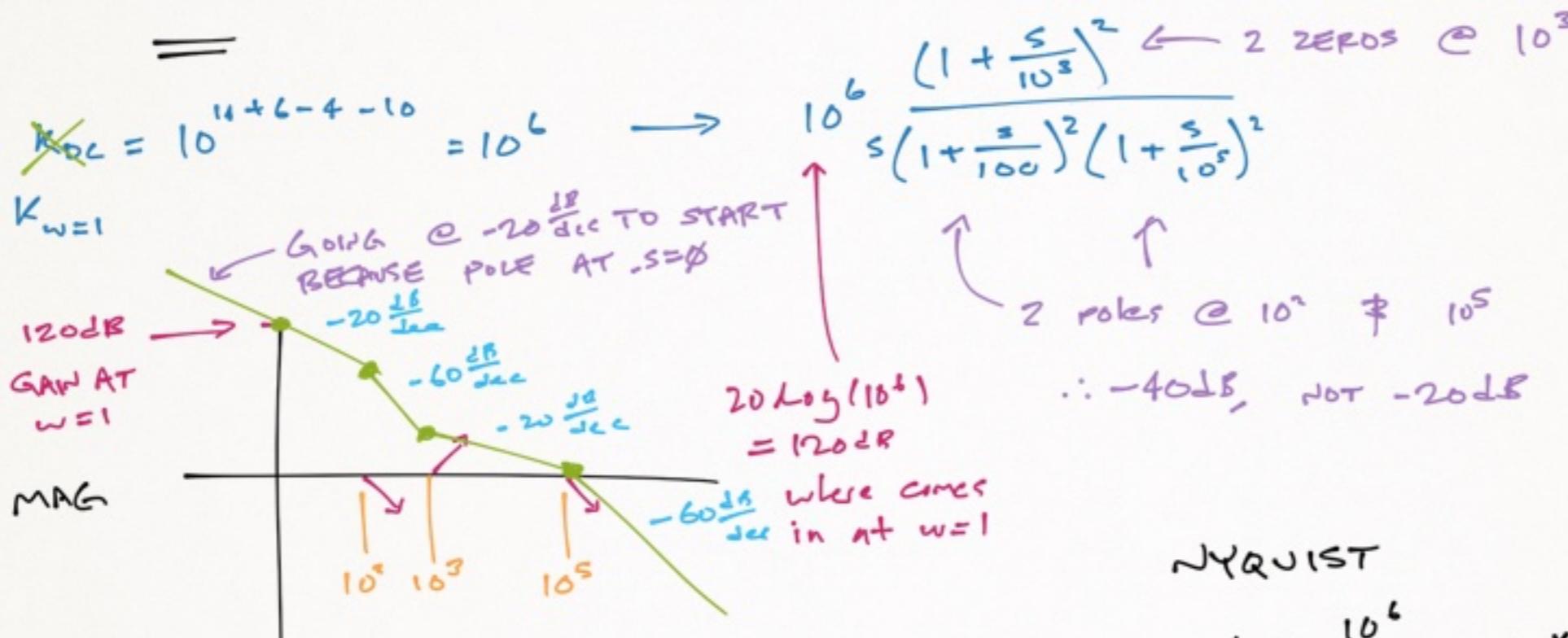
$B = \infty \times -90^\circ$

$C = \Delta \times 90^\circ$

an extremely small number, not negative ∞

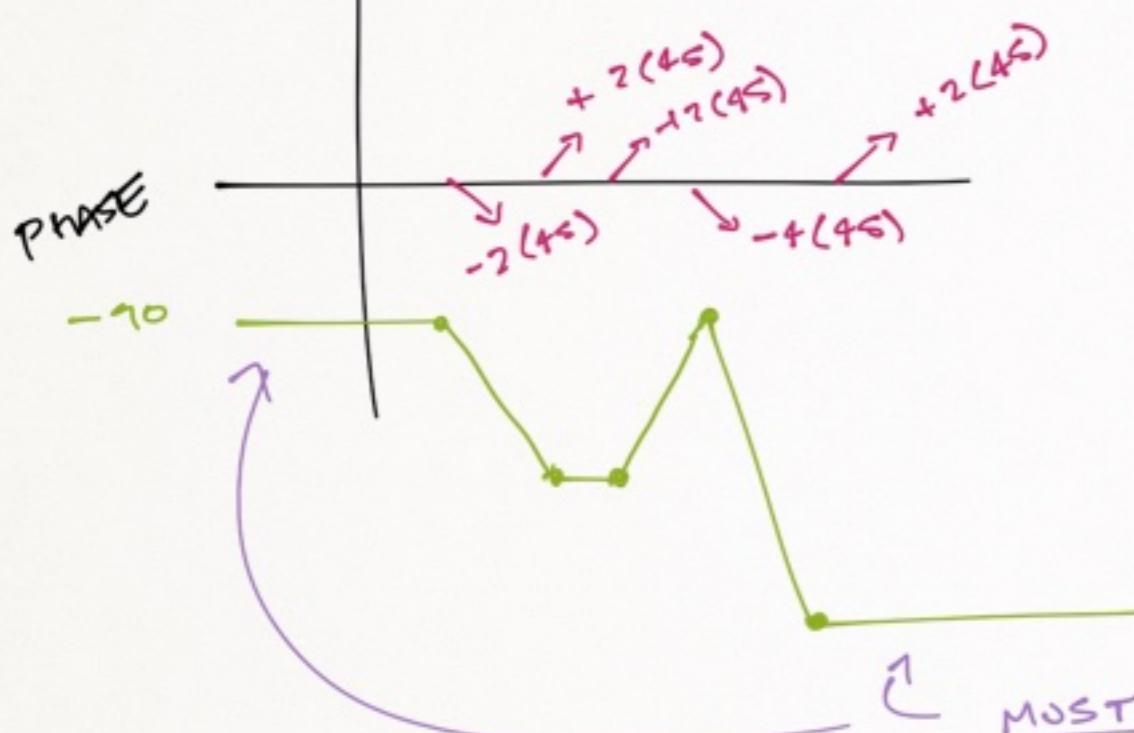


$$Ex] \frac{10^{14} (s+10^3)^2}{s(s+10^2)^2(s+10^5)^2}$$

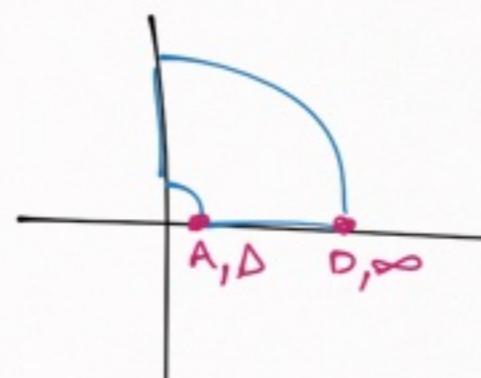


A POINT $\Delta: \frac{10^6}{\Delta} = \infty \times \phi^\circ$

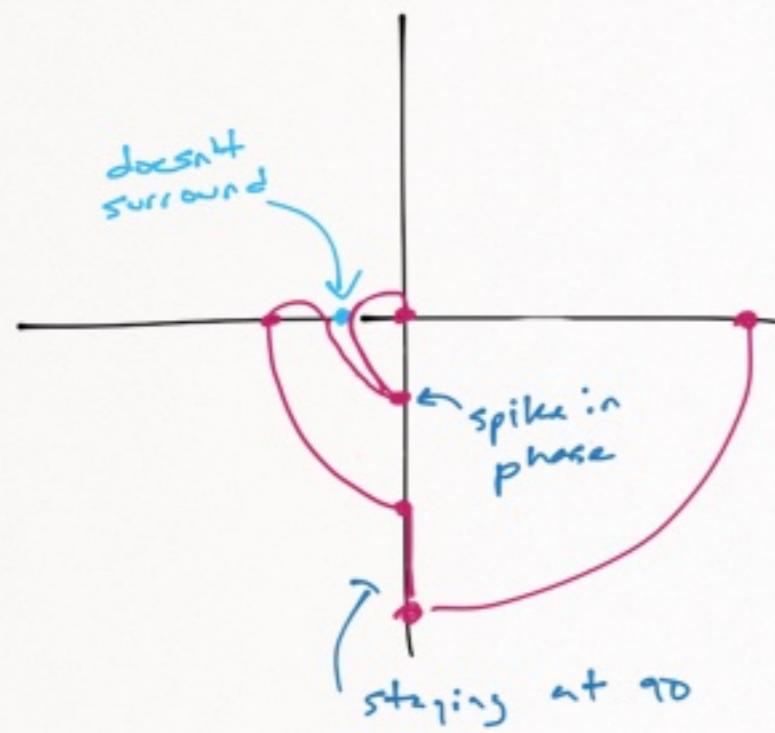
D POINT $\infty: \frac{1}{\infty} = \phi \times \phi^\circ$



MOST BEGIN AND END LEVEL. IF IT DOESN'T
YOU SCREWED UP.



Ex] Cont'd



IMPORTANT POINTS

@ 180 IT GOES FROM
0 TO 20dB

IF YOU AMPLIFY OR ATTENUATE BY 20dB,
YOU WILL SURROUND THE ZERO.

gain margin: $\pm \sim 20\text{dB}$

phase margin: 70°

$$\omega_c = 10^4 \frac{\text{rad}}{\text{s}}$$

$$\text{Ex] } KGH = \frac{10^4}{(s+100)(s^2+14s+98)}$$

=

$$\rightarrow 10^2 \left(\frac{1}{1 + \frac{s}{100}} \right) (s + 7 + j7)(s + 7 - j7)$$

$$(s + 10 \angle 45^\circ)(s + 10 \angle -45^\circ)$$

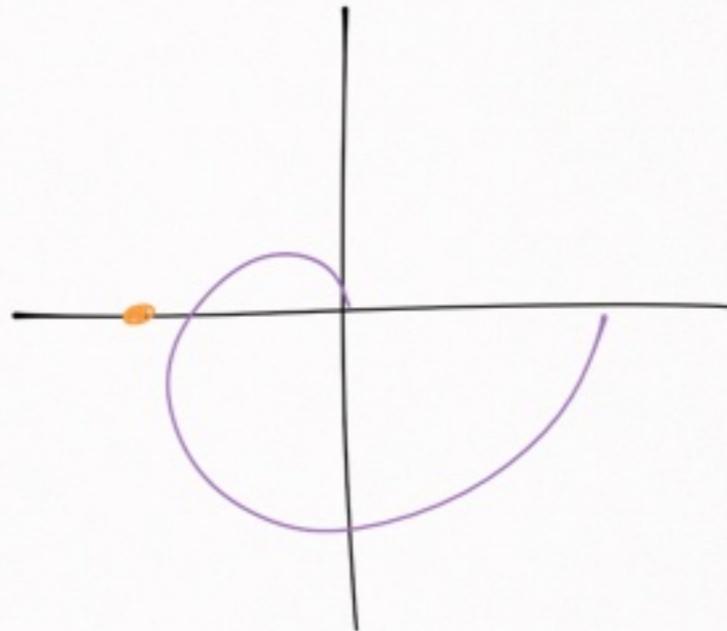
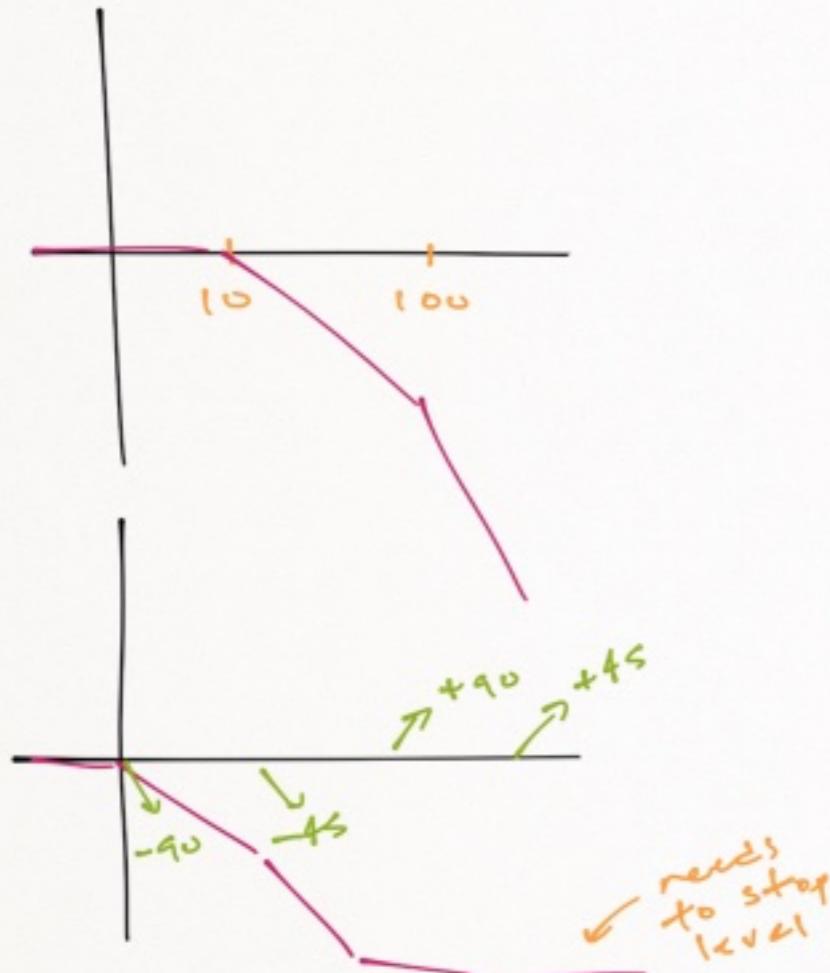
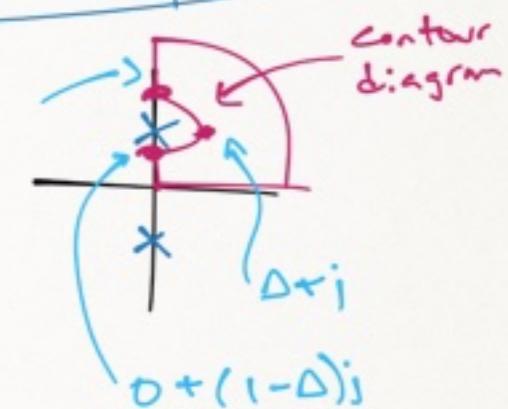
for bode plot, you can treat this as $(s+10)^2$

$$\rightarrow \left(\frac{1}{1 + \frac{s}{100}} \right) \left(1 + \frac{s}{10 \angle 45^\circ} \right) \left(1 + \frac{s}{10 \angle -45^\circ} \right)$$

NOTE POSSIBILITY

$$\frac{1}{s^2 + 1}$$

purely imaginary



GAIN MARGIN $\sim 30 \text{ dB}$
PHASE MARGIN $\sim 180^\circ$

QUIZ # 4 REVIEW - NYQUIST

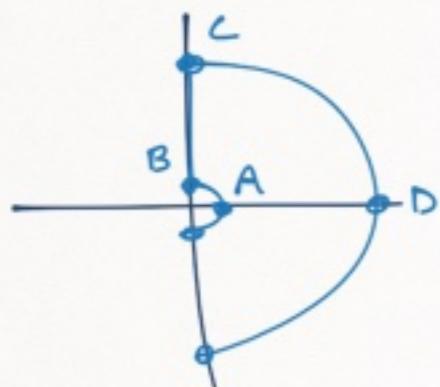
Ex) $KGH = \frac{1000}{s(s+1)(s+100)}$

DRAW IN CORRECT FORM FOR BODE PLOT

$$\frac{\frac{1000}{100}}{\frac{s}{100} \left(\frac{s}{100} + 1 \right) \left(s + 1 \right)} = 10 \frac{1}{s(s+1)\left(\frac{s}{100} + 1\right)}$$

\uparrow
 $k @ \omega=1$

DRAW THE ASSOCIATED NYQUIST CONTOUR



$A = (\Delta, \phi)$

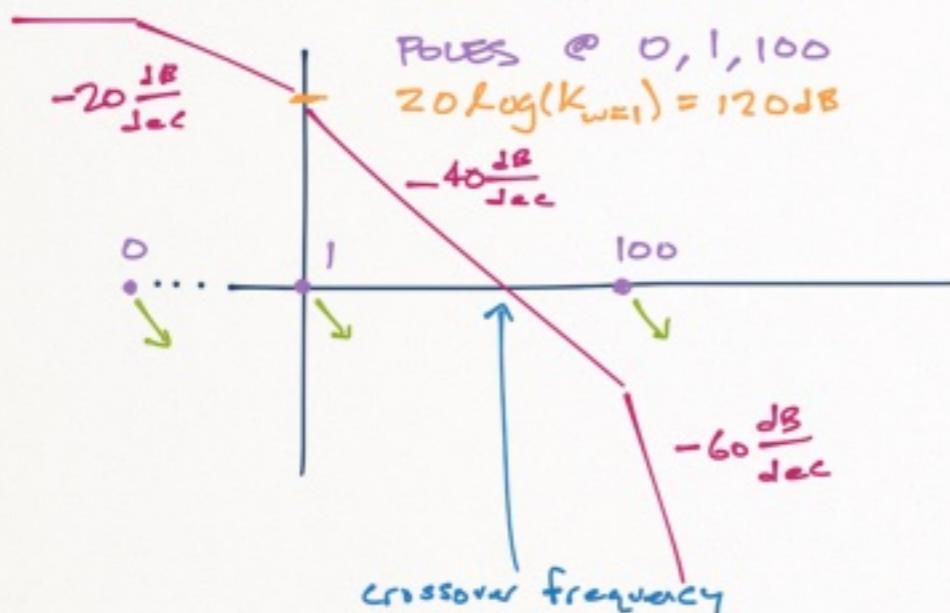
$B = (\phi, \Delta)$

$C = (\phi, \infty)$

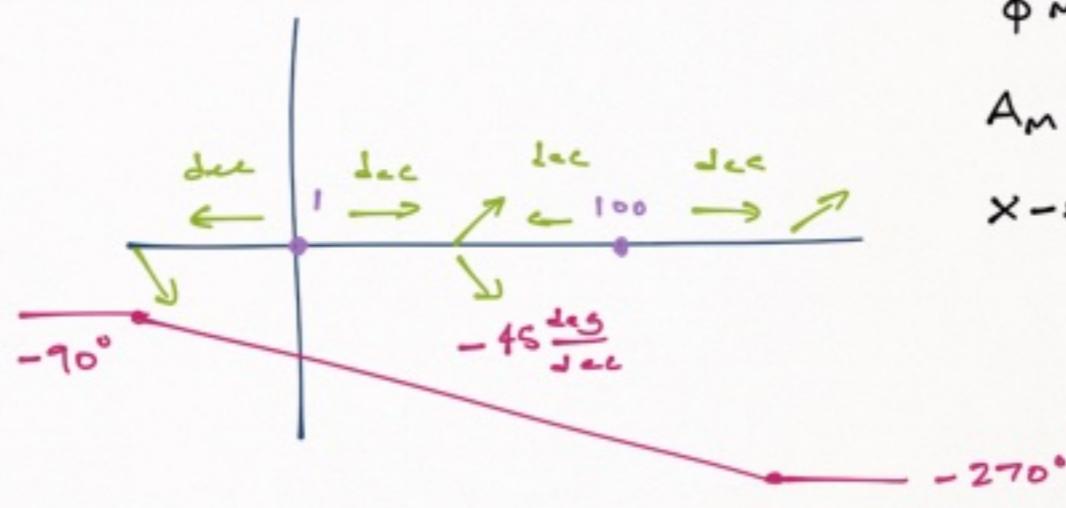
$D = (\infty, \phi)$

s	$ KGH $	$\angle KGH$
$A = (\Delta, \phi)$	∞	ϕ
$B = (\phi, \Delta)$	∞	-90°
$C = (\phi, \infty)$	Δ	90°
$D = (\infty, \phi)$	Δ	ϕ

DRAW THE MAGNITUDE PLOT

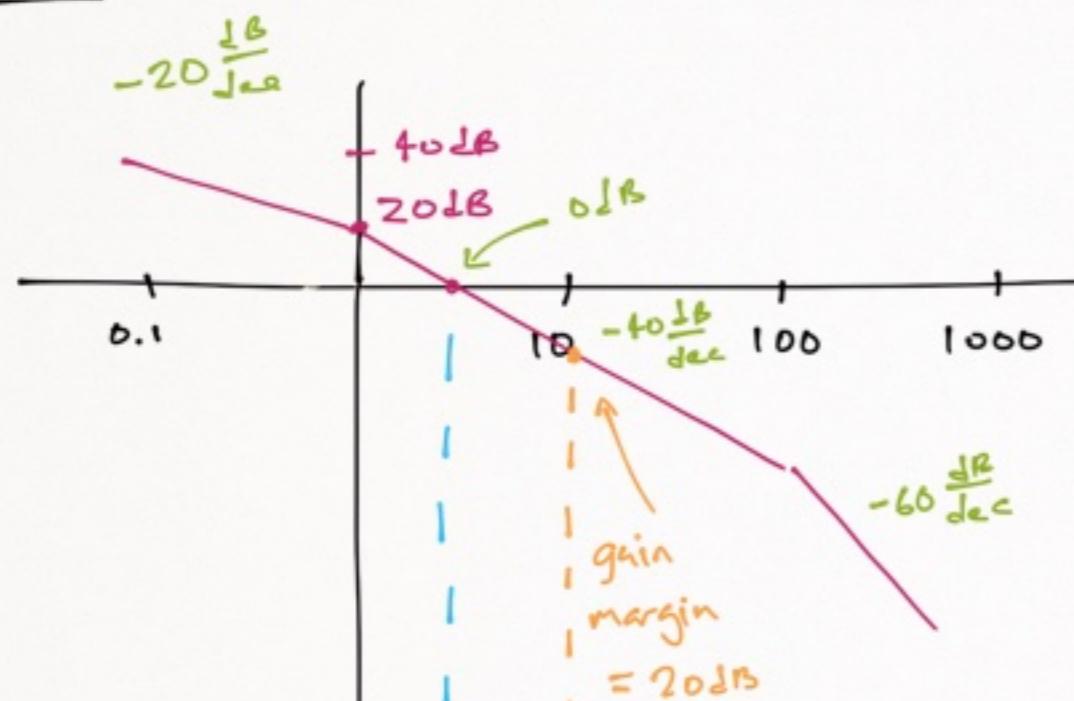


DRAW THE PHASE PLOT

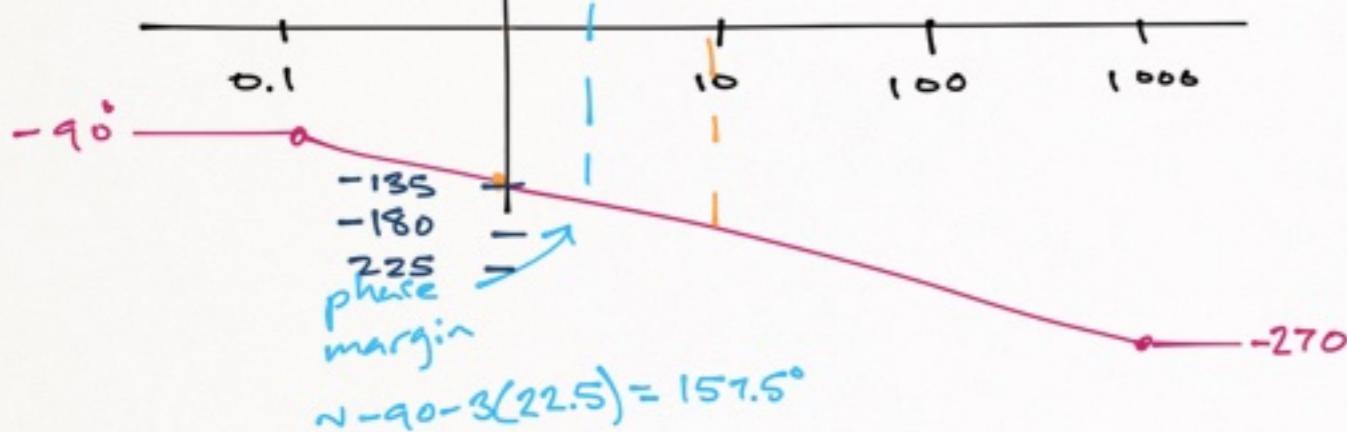


ϕ MARGIN: 122.5°
 A_m MARGIN: 20 dB
 X-FREQUENCY: $2-3 \text{ Hz}$

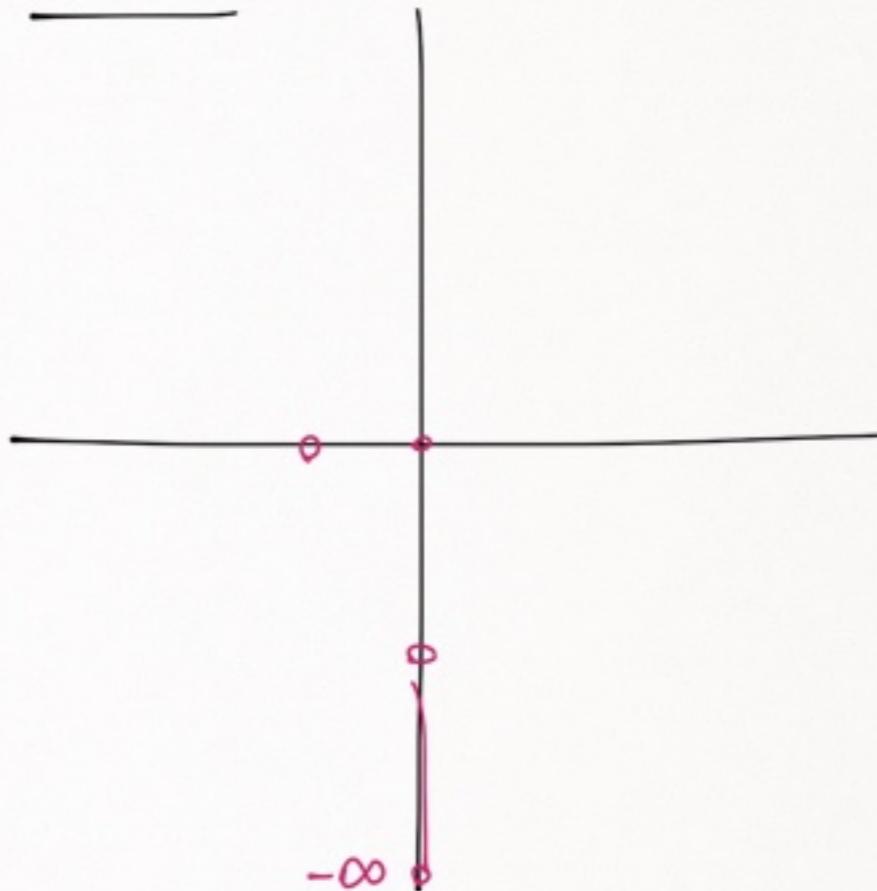
GAIN



PHASE



NYQUIST



$$\text{Ex)} KGH = \frac{10^4 (s+10^3)^2}{s(s+10^2)^2(s+10^5)^2}$$

BODE PLOT FORM

$$10^6 \frac{\left(1 + \frac{s}{10^2}\right)^2}{s\left(1 + \frac{s}{10^2}\right)\left(1 + \frac{s}{10^5}\right)^2}$$

$$10^{14+2(3)-2(2)-2(5)} = 10^6 = K_{w=1}$$

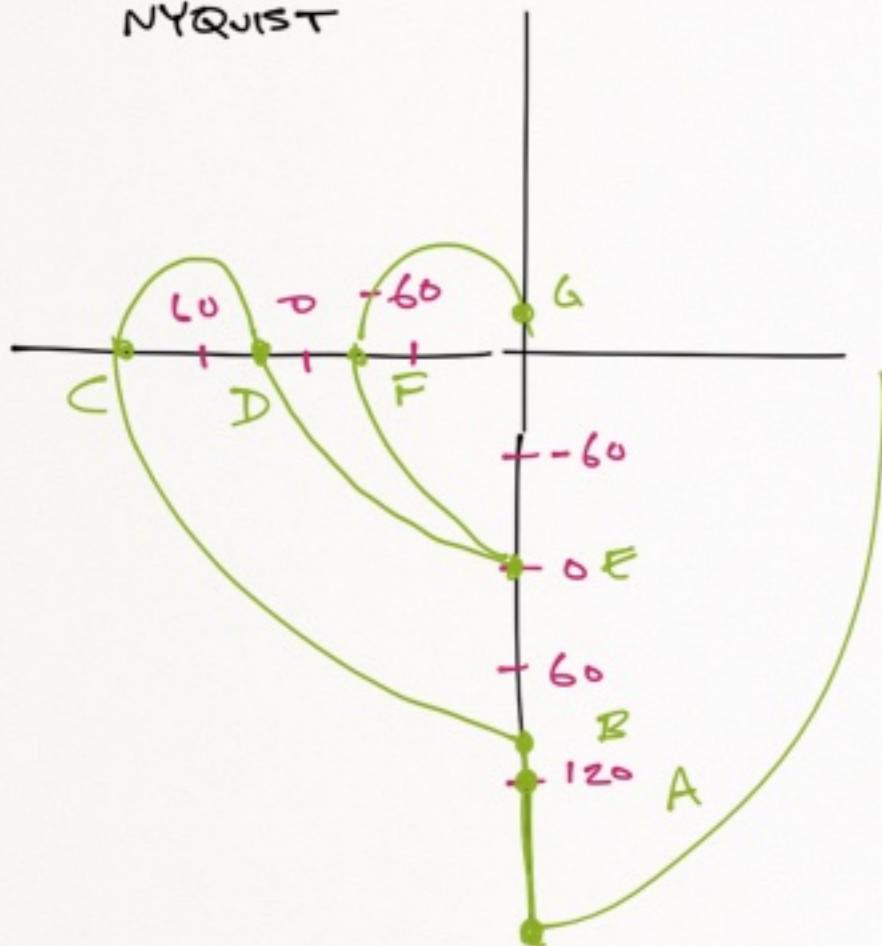
2 POLES @ $10^2 \pm 10^5$, 2 ZEROS @ 10^3
1 POLE @ 0

($40 \frac{d\theta}{dec}$ changes w/ 2)

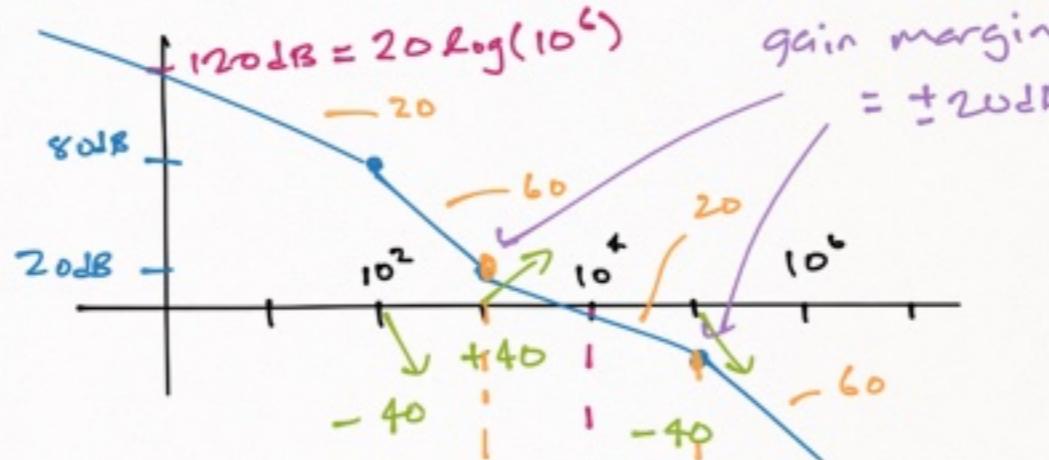
ASSOCIATED CONTOUR

- A $(\Delta, \phi) \rightarrow \infty \times 40^\circ$
- B $(\phi, \Delta) \rightarrow \infty \times -90^\circ$
- C $(\phi, \infty) \rightarrow 0 \times 90^\circ$
- D $(\infty, \phi) \rightarrow 0 \times 0^\circ$

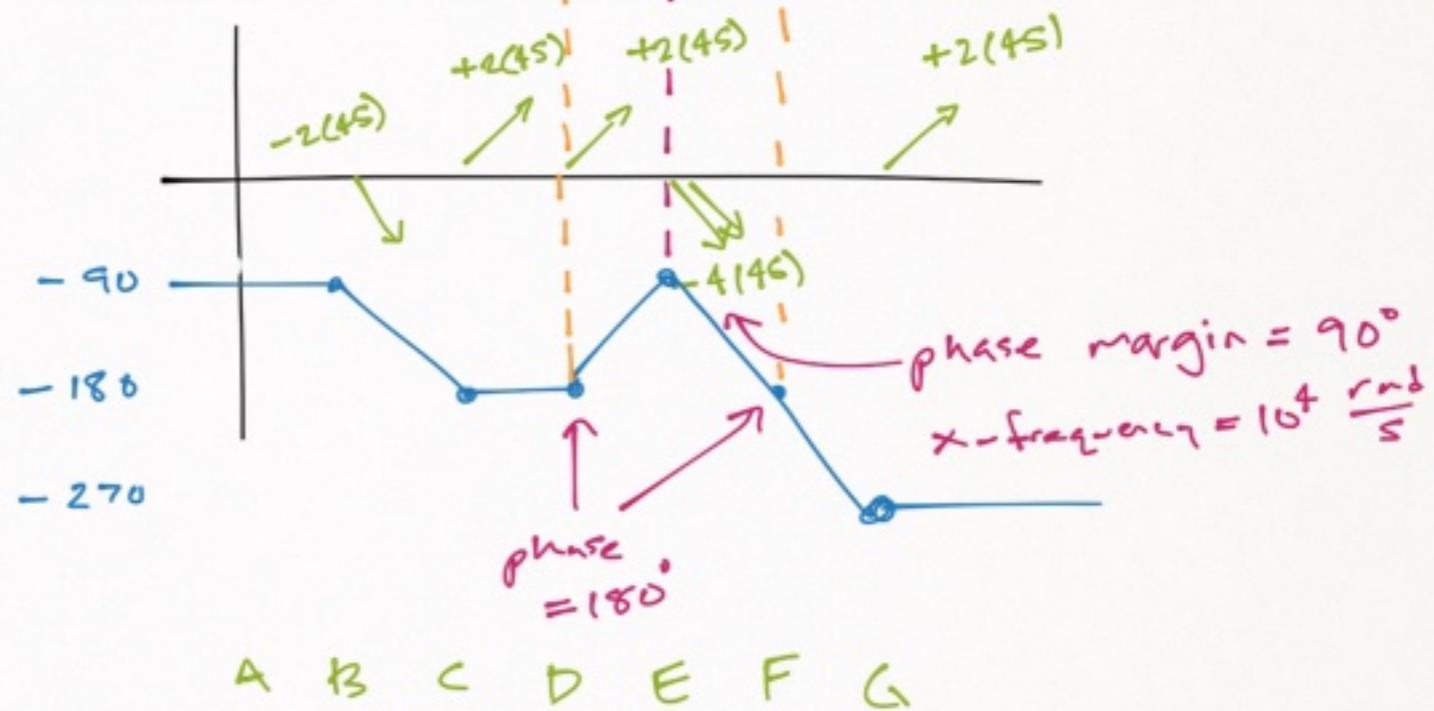
NYQUIST



MAGNITUDE PLOT



PHASE PLOT



$$\text{Ex) } \frac{10^9(s+10^3)}{(s+100)(s^2+4s+9)}$$

$\approx (s+10\angle 45^\circ)(s+10\angle -45^\circ)$

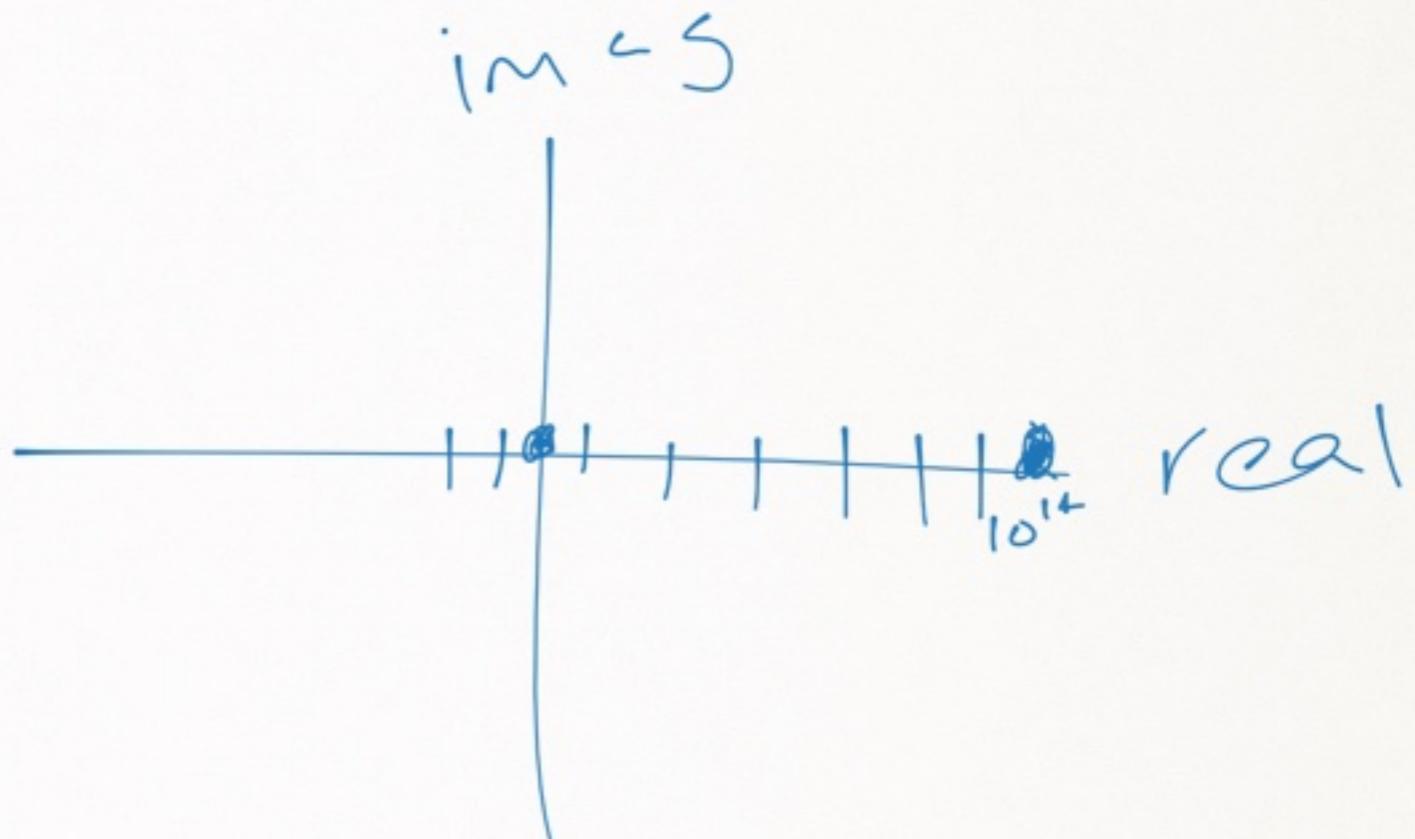
$$10^{s+3-2-2} = 10^8$$

$$10^8 \frac{\left(\frac{s}{10^3} + 1\right)}{\left(\frac{s}{10^2} + 1\right)\left(\frac{s}{10\angle 45^\circ} + 1\right)\left(\frac{s}{10\angle -45^\circ} + 1\right)}$$

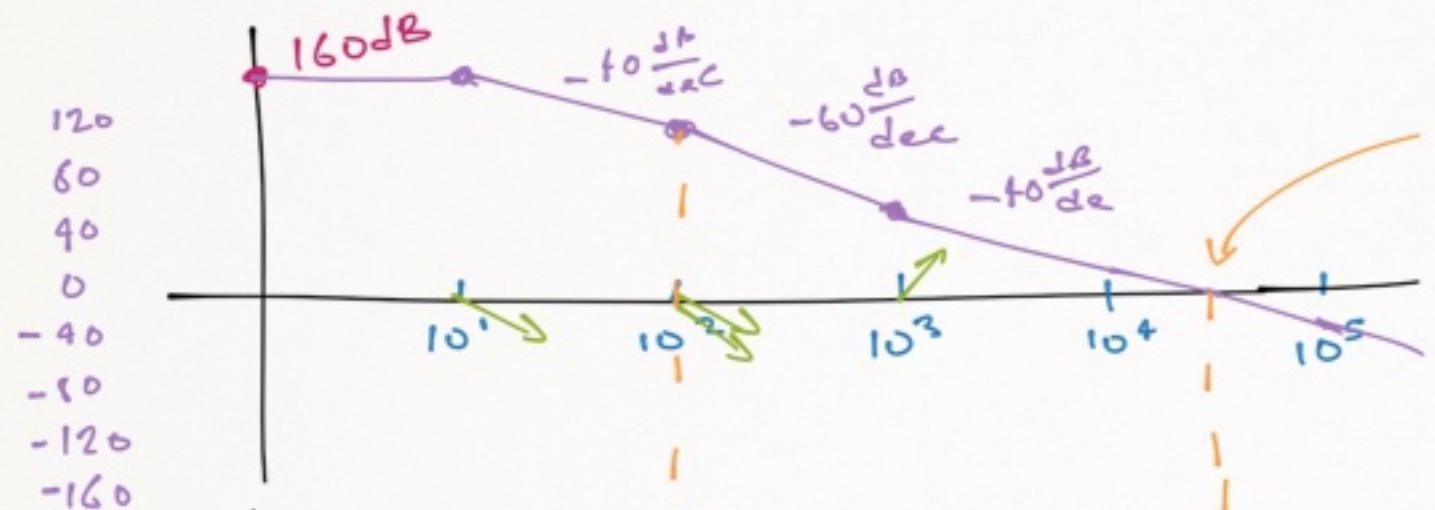
$$K_{w=1} = 20 \log(10^8) = 160 \text{ dB}$$

DRAW CONTOUR + DO ABC

s	$ k_{GN} $	$\angle k_{GN}$
A $(0, 0)$	10^{14}	$\neq \phi$
B $(0, \infty)$	Δ	$\neq 180^\circ$
C $(\infty, 0)$	Δ	ϕ

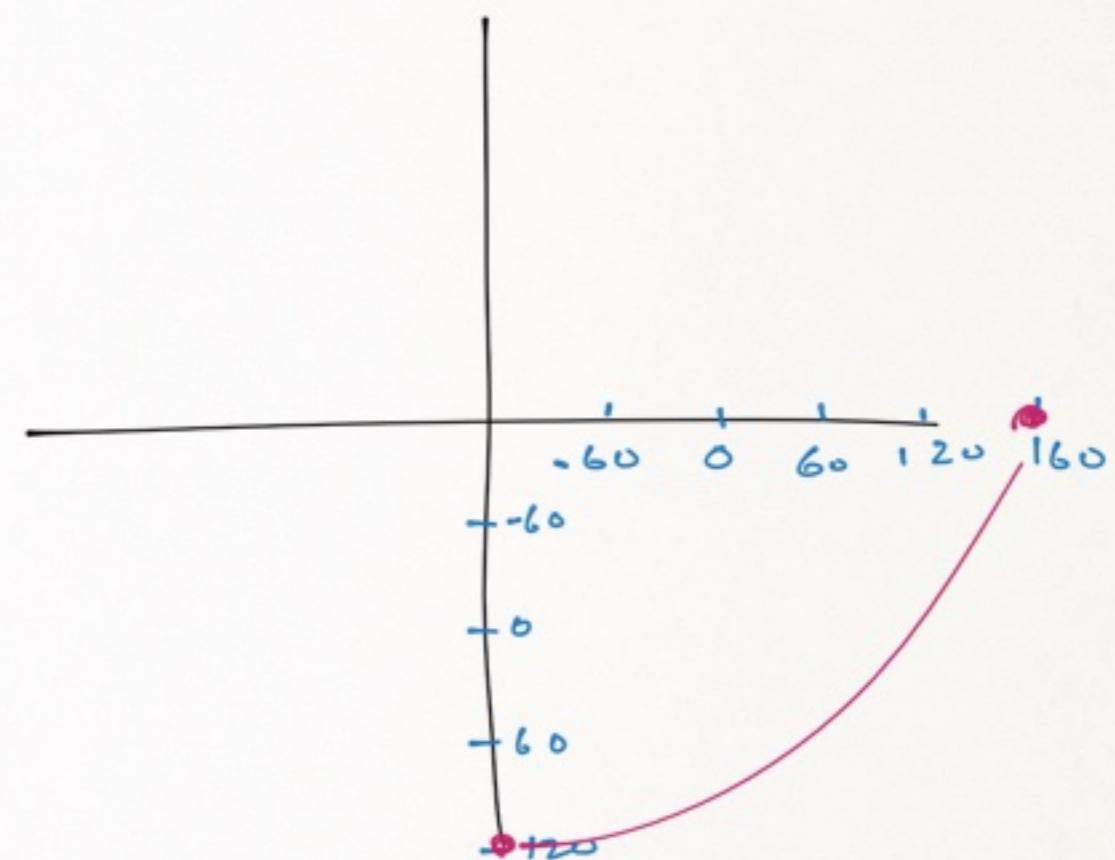
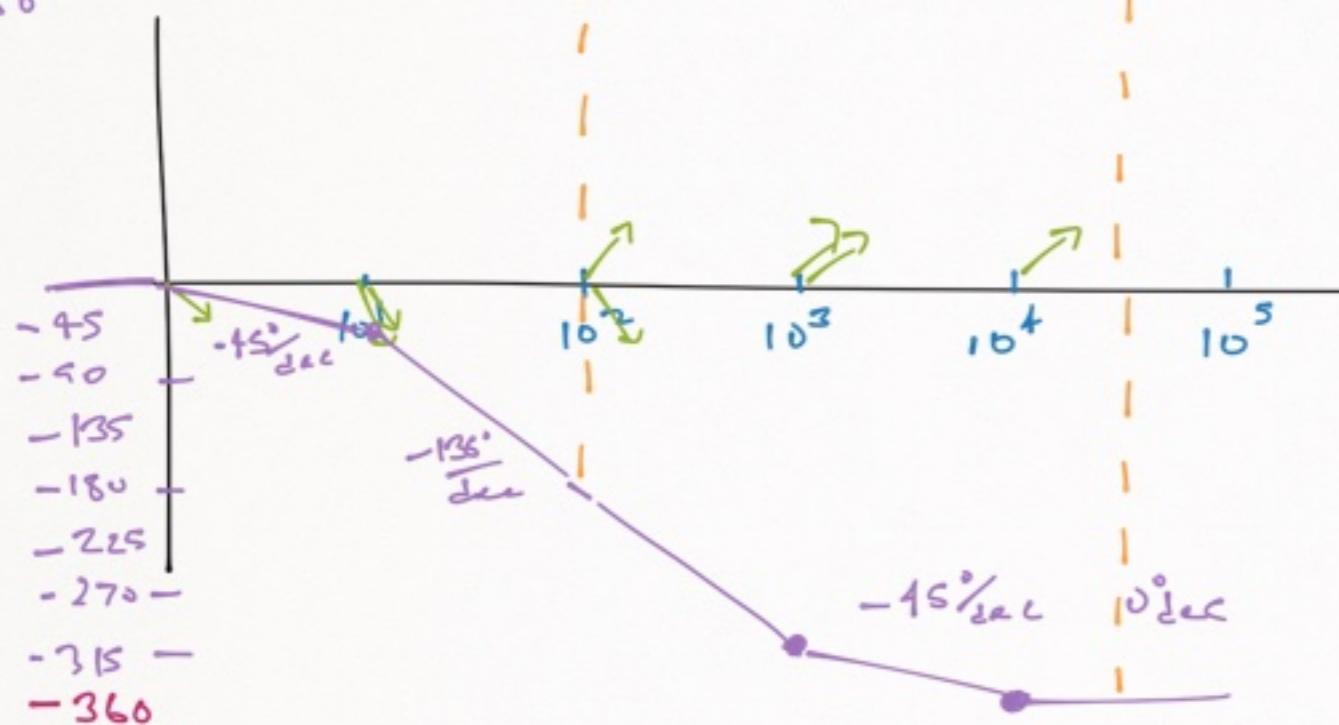


MAG. PLOT



cutoff frequency
is $\sim 10^{4.5}$

GAIN MARGIN $120 dB$
PHASE MARGIN $360^\circ = 0^\circ ?$



STATE SPACE

↳ "SUPER DUPER CIRCUIT ANALYSIS"

↳ SOLVING EVERYTHING AT ONCE

NOV 17 / 2014



$$\text{Cap: } \frac{1}{2}CV^2$$

$$\text{Ind: } \frac{1}{2}LI^2$$

Work: Fd

$$\text{Mass: } Mda = M \int v dt \quad \frac{d}{dt} v = \frac{1}{2}mv^2$$

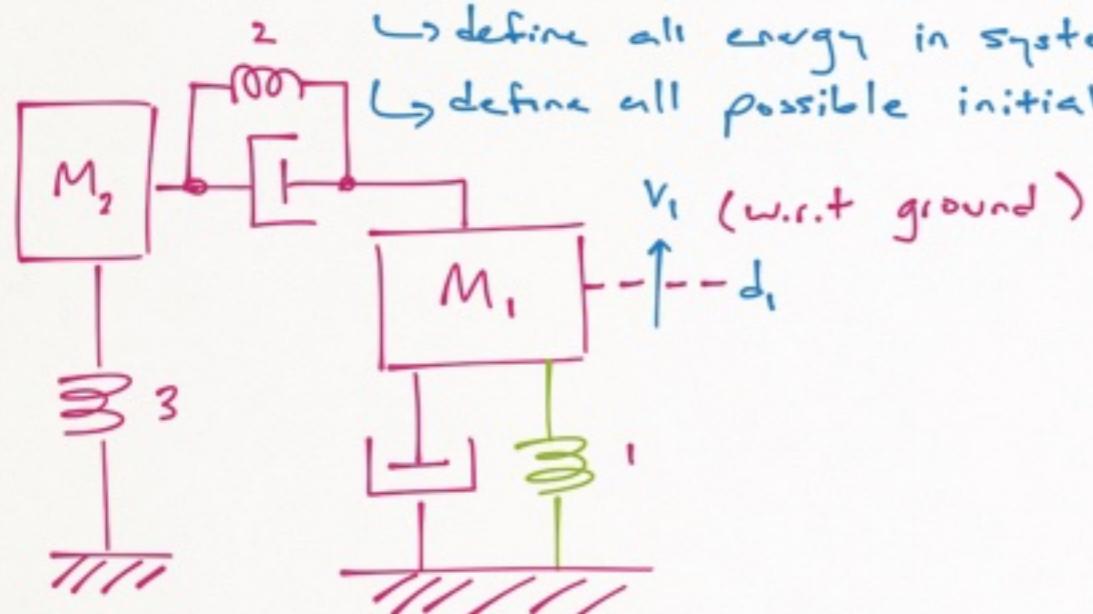
$$\text{Spring: } Kdd = k \int d dt = \frac{1}{2}kd^2$$

SYSTEM STATES → DEFINES HOW MUCH ENERGY A SYSTEM HAS AT A TIME

↳ all must be independent values

↳ define all energy in system

↳ define all possible initial conditions



$$E_{m1} = \frac{1}{2}M_1v_1^2$$

$$E_{k1} = \frac{1}{2}k_1d_1^2$$

$$E_{m2} = \frac{1}{2}M_2v_2^2$$

$$E_{k2} = \frac{1}{2}k_2(d_2 - d_1)^2$$

STATES

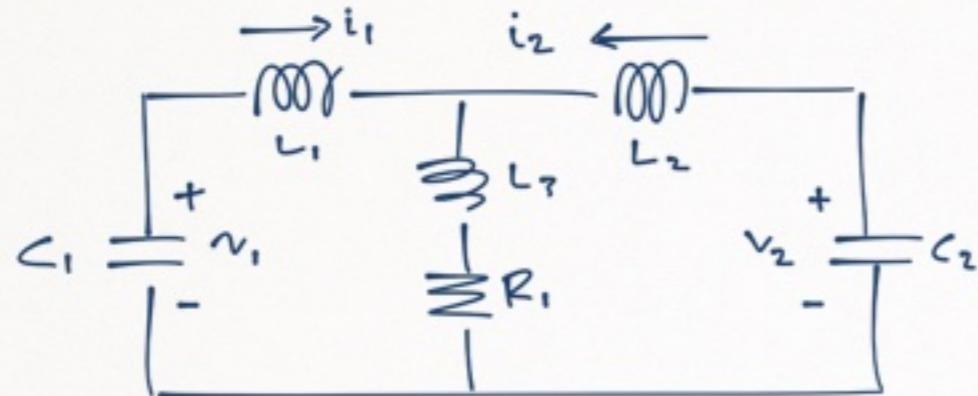
v_1, d_1 → defines energy in system

both independent, cannot calculate one from the other

v_2, d_2
4 states

$$E_{k3} = \frac{1}{2}k_3d_3^2 \leftarrow \text{not it's own state.}$$

ELECTRIC EXAMPLE STATE SPACE



$$\begin{aligned} E_{C_1} &= \frac{1}{2} C_1 v_1^2 && \text{STATES } v_1, i_1 \\ E_{L_1} &= \frac{1}{2} L_1 i_1^2 && \\ E_{C_2} &= \frac{1}{2} C_2 v_2^2 && \text{STATES } v_2, i_2 \\ E_{L_2} &= \frac{1}{2} L_2 i_2^2 \end{aligned}$$

STATE EQUATION

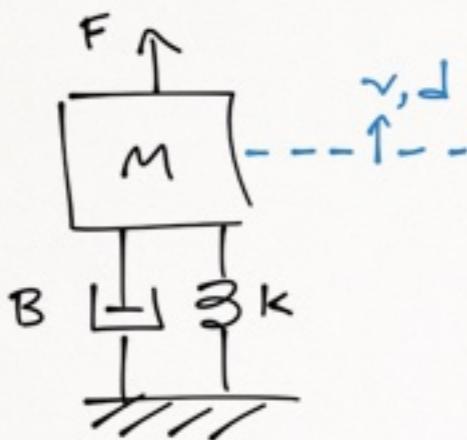
1. IDENTIFY ALL STATES
2. TREAT ALL STATES LIKE OUTPUT
3. WRITE EQUATION FOR EACH.
4. COMBINE IN MATRIX FORM
5. REARRANGE INTO:

$$\dot{\bar{x}} = \bar{A}\bar{x} + \bar{B}\bar{u}$$

vector of state derivatives Matrix vector of states vector of inputs

$i_{L_3} = i_1 + i_2 \rightarrow i_1 + i_2$ is not its own state

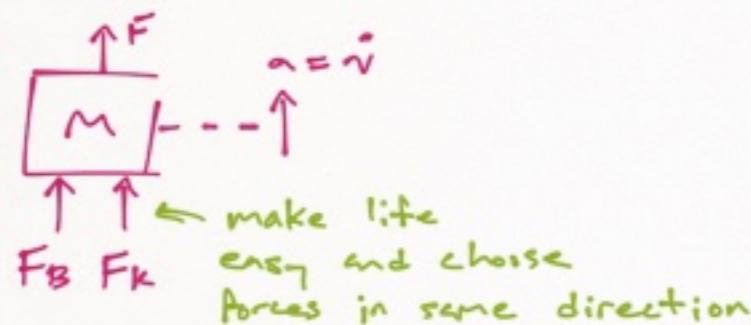
Ex]



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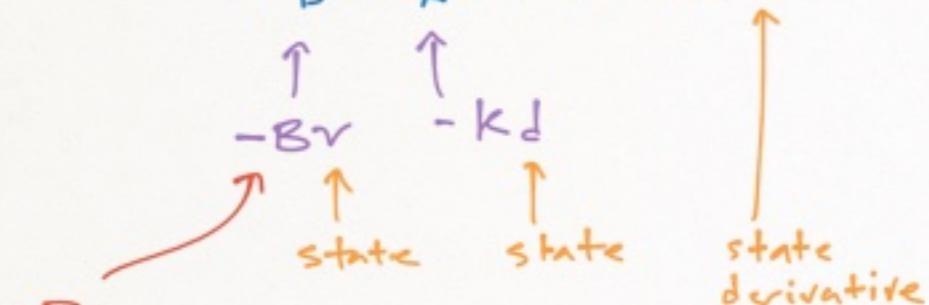
STATES v, d

DO F.B.D.



$$F_B + F_K = M_a = M\ddot{v}$$

F is input



$$\dot{\bar{x}} = \begin{bmatrix} d \\ \dot{v} \end{bmatrix}$$

$$\bar{x} = \begin{bmatrix} d \\ v \end{bmatrix}$$

$$\dot{\bar{x}} = \bar{A}\bar{x} + \bar{B}u$$

$$\begin{bmatrix} \dot{d} \\ \dot{v} \end{bmatrix} = \begin{bmatrix} ? & ? \end{bmatrix} \begin{bmatrix} d \\ v \end{bmatrix} + \bar{B}u$$

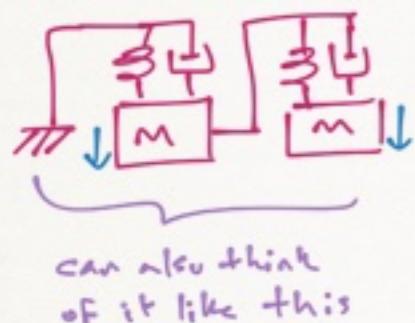
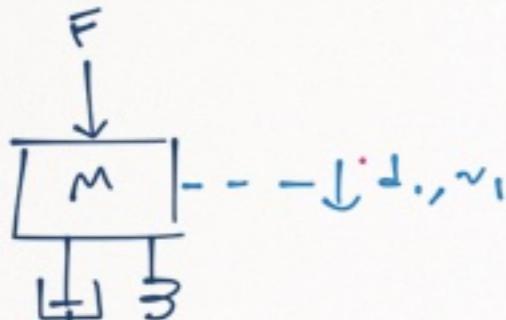
$$\begin{bmatrix} \dot{d} \\ \dot{v} \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -\frac{K}{M} & \frac{-B}{M} \end{bmatrix} \begin{bmatrix} d \\ v \end{bmatrix} + \begin{bmatrix} 0 \\ \frac{1}{M} \end{bmatrix} F$$

$$\dot{v} = \frac{-K}{M}d - \frac{B}{M}v$$

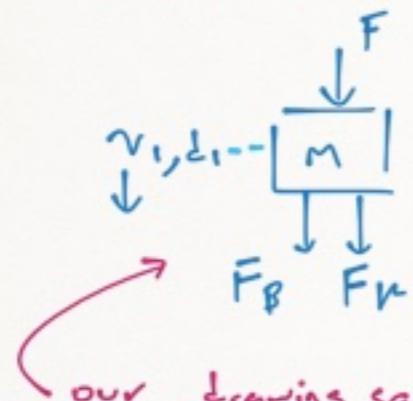
$$\dot{d} = \cancel{0} + 1 \cdot v$$

usually get this one for free

$$\ddot{\mathbf{x}} = \bar{A}\ddot{\mathbf{x}} + \bar{B}\dot{\mathbf{u}}$$



KCL in mechanical system



$$F + F_B + F_K = M\ddot{v}_1$$

watch sign

define vectors

$$\ddot{\mathbf{x}} = \begin{bmatrix} d_1 \\ d_2 \\ v_1 \\ v_2 \end{bmatrix}$$

states

$$\dot{\mathbf{x}} = \begin{bmatrix} \dot{d}_1 \\ \dot{d}_2 \\ \dot{v}_1 \\ \dot{v}_2 \end{bmatrix}$$

$$\dot{\mathbf{u}} = \mathbf{F}$$

inputs

define matrices

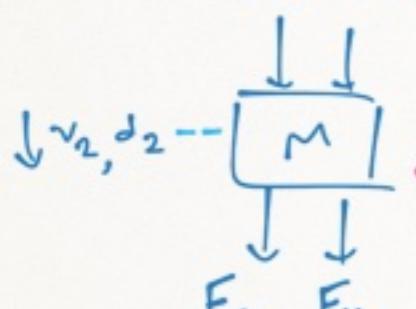
$$\begin{bmatrix} d_1 \\ d_2 \\ v_1 \\ v_2 \end{bmatrix} = \begin{bmatrix} \emptyset & \emptyset & 1 & \emptyset \\ \emptyset & \emptyset & \emptyset & 1 \\ -\frac{k}{m} & \frac{k}{m} & -\frac{B}{m} & \frac{B}{m} \\ \frac{k}{m} & -\frac{2k}{m} & \frac{B}{m} & -\frac{2B}{m} \end{bmatrix} \begin{bmatrix} d_1 \\ d_2 \\ v_1 \\ v_2 \end{bmatrix} + \begin{bmatrix} \emptyset \\ \emptyset \\ \emptyset \\ \emptyset \end{bmatrix} \mathbf{F}$$

$$\frac{F}{m} + \frac{B}{m}(v_2 - v_1) + k(d_2 - d_1) = \ddot{v}_1$$

$$F_K = K(d_2 - d_1)$$

our drawing says it's stretching $\therefore v_2 > v_1 \therefore$ use $v_2 - v_1$ to keep positive

$$F_B = F(v_2 - v_1)$$



our drawing says it's compressing $\therefore v_1 > v_2 \therefore$ use $v_1 - v_2$ to keep positive

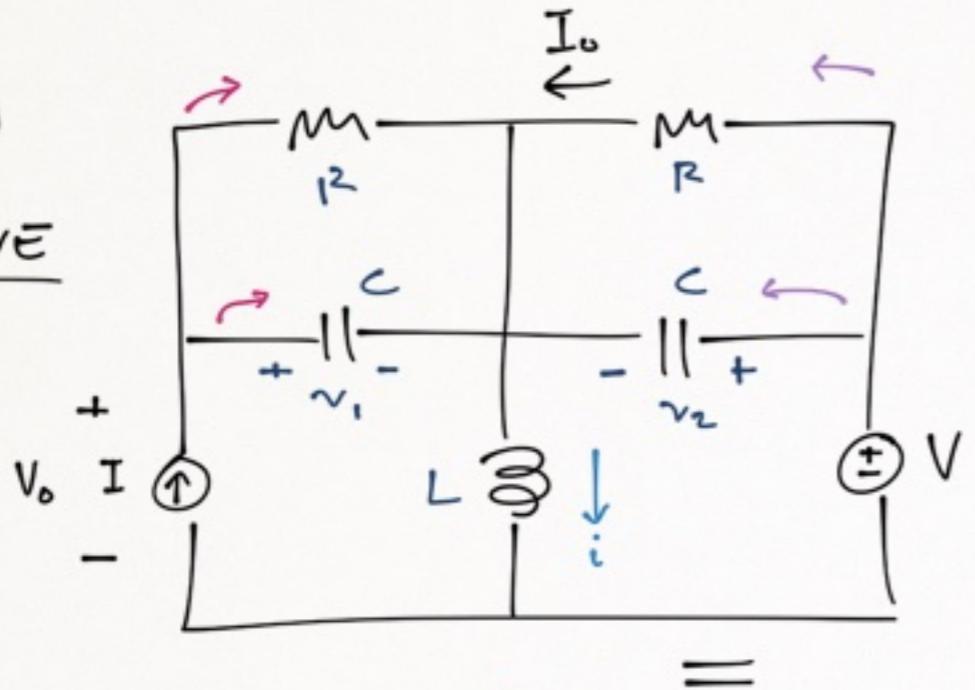
$$F_{B1} + F_{K1} + F_{B2} + F_{K2} = \ddot{v}_2 m$$

$$B(v_1 - v_2) + K(d_1 - d_2) - Bv_2 - kd_1 = \ddot{v}_2 m$$

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Ex]

SOLVE

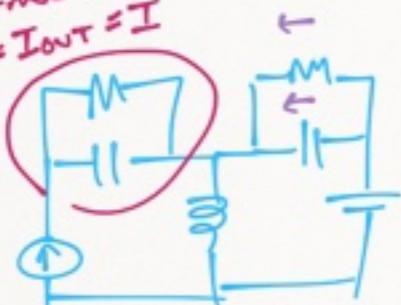


$$I = C \dot{v}_1 + \frac{1}{R} v_1$$

$$\rightarrow \dot{v}_1 = \frac{1}{C} I - \frac{1}{RC} v_1$$

$$= -\frac{1}{RC} v_1 + \frac{1}{C} I$$

Supernode:
 $I_{IN} = I_{OUT} = I$



$$i = I + C \dot{v}_2 + \frac{1}{R} v_2$$

$$\rightarrow \dot{v}_2 = \frac{1}{RC} v_2 + \frac{1}{C} i - \frac{1}{C} I$$

OUTPUT EQUATION (V_o, I_o)

$$\bar{y} = \bar{C} \bar{x} + \bar{D} \bar{u}$$

$2 \times 1 \quad 2 \times 3 \quad 3 \times 1 \quad 2 \times 2 \quad 2 \times 1$

$$-v_1 + v_2 - V + V_o = \phi$$

$$\rightarrow V_o = v_1 - v_2 + V$$

$$I_o = \frac{1}{R} v_2$$

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$$\text{HWT: state vect} = \begin{vmatrix} v_1 \\ v_2 \\ i \end{vmatrix} \quad \bar{x} = \begin{vmatrix} v_1 \\ v_2 \\ i \end{vmatrix} \quad \bar{u} = \begin{vmatrix} V \\ I \end{vmatrix}$$

$$i = C \dot{v}$$

$$v = L \dot{i}$$

$$\bar{x} = \bar{A} \bar{x} + \bar{B} \bar{u}$$

$3 \times 1 \quad 3 \times 3 \quad 3 \times 1 \quad 3 \times 2 \quad 2 \times 1$

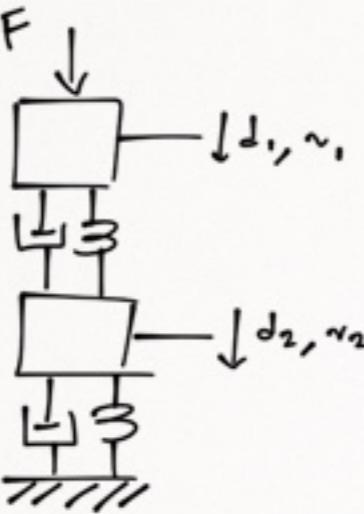
$$\begin{bmatrix} \dot{v}_1 \\ \dot{v}_2 \\ i \end{bmatrix} = \begin{bmatrix} -\frac{1}{RC} & \phi & \phi \\ \phi & -\frac{1}{RC} & \frac{1}{C} \\ \phi & \frac{1}{L} & \phi \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \\ i \end{bmatrix} + \begin{bmatrix} \phi & \frac{1}{C} \\ \phi & -\frac{1}{C} \\ \frac{1}{L} & \phi \end{bmatrix} \begin{bmatrix} V \\ I \end{bmatrix}$$

$$V = v_2 + L \dot{i}$$

$$\rightarrow \dot{i} = -\frac{1}{L} v_2 + \frac{1}{L} V$$

$$\begin{bmatrix} V_o \\ I_o \end{bmatrix} = \begin{bmatrix} 1 & -1 & 0 \\ 0 & \frac{1}{R} & 0 \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \\ i \end{bmatrix} + \begin{bmatrix} 1 & 0 \\ 0 & 0 \end{bmatrix} \begin{bmatrix} V \\ I \end{bmatrix}$$

Ex]



=

$$\Delta d = d_2 - d_1$$

$$\Delta v = v_2 - v_1$$

$$\bar{\gamma} = \bar{C}\bar{x} + D\bar{u}$$

① set up matrices

$$\begin{vmatrix} \Delta d \\ \Delta v \end{vmatrix} = \begin{vmatrix} -1 & 1 & 0 & 0 \\ 0 & 0 & -1 & 1 \end{vmatrix} \begin{vmatrix} d_1 \\ d_2 \\ v_1 \\ v_2 \end{vmatrix} + \begin{vmatrix} \phi \\ \phi \end{vmatrix} F$$

② transform
into s -domain
using laplace

$$\dot{\bar{x}} = \bar{A}\bar{x} + \bar{B}\bar{u} \xrightarrow{\mathcal{L}} s\bar{x} - \bar{x}_0 = \bar{A}\bar{x} + \bar{B}\bar{u}$$

subtracting initial conditions

$$s\bar{x} - \bar{A}\bar{x} = \bar{x}_0 + \bar{B}\bar{u}$$

$$s\bar{I}\bar{x} - \bar{A}\bar{x} = \bar{x}_0 + \bar{B}\bar{u}$$

$$(s\bar{I} - \bar{A})\bar{x} = \bar{x}_0 + \bar{B}\bar{u}$$

$$\bar{x} = (\underbrace{s\bar{I} - \bar{A}}_{= \bar{\phi}})^{-1} \bar{x}_0 + (\underbrace{s\bar{I} - \bar{A}}_{= \bar{\phi}})^{-1} \bar{B}\bar{u}$$

(natural
resp.)

$$\bar{x} = \underbrace{\bar{\phi}\bar{x}_0}_{\text{RESPONSE TO INITIAL CONDITIONS}} + \underbrace{\bar{\phi}\bar{B}\bar{u}}_{\text{RESPONSE TO FORCING FUNCTION}}$$

total response (forced resp.)

Ex] cont'd

Nov 24/2014

OUTPUT EQN.

$$\bar{y} = \bar{C}\bar{x} + \bar{D}\bar{u}$$

$$\bar{y} = \bar{C}(\bar{\phi}\bar{x}_0 + \bar{\phi}\bar{B}\bar{j}) + \bar{D}\bar{u}$$

$$\bar{y} = \bar{C}\bar{\phi}\bar{x}_0 + (\bar{C}\bar{\phi}\bar{B} + \bar{D})\bar{u}$$

natural
resp.

forced
resp.

$$TF: \bar{C}\bar{\phi}\bar{B} + \bar{D}$$

wait for
natural
response
to go away

$$\textcircled{1} \quad \bar{x} = \bar{A}\bar{x} + \bar{B}\bar{u}$$

$$\textcircled{2} \quad \bar{y} = \bar{C}\bar{x} + \bar{D}\bar{u}$$

$$\textcircled{3} \quad \bar{\phi} = [s\bar{I} - \bar{A}]^{-1}$$

$$\textcircled{4} \quad \bar{x} = \bar{\phi}\bar{x}_0 + \bar{\phi}\bar{B}\bar{u}$$

$$\textcircled{5} \quad \bar{y} = \bar{C}\bar{\phi}\bar{x}_0 + (\bar{C}\bar{\phi}\bar{B} + \bar{D})\bar{u}$$

TF

\bar{E}_x



$$\gamma = i_R$$

$$v = I$$

\Leftarrow

① identify engn

$$v_c, i_L$$

② validate, are any of these calculated by dependent on each other

$$\bar{x} = \begin{bmatrix} v \\ i \end{bmatrix} \quad \bar{x} = \begin{bmatrix} \frac{1}{RC} & \frac{1}{C} \\ \frac{1}{L} & \phi \end{bmatrix} \bar{x} + \begin{bmatrix} \frac{1}{C} \\ \phi \end{bmatrix} I$$

\bar{A} \bar{B}

$$v_c = i_c \Rightarrow v_c = i_c \frac{1}{sC}$$

$$i_c = sCv = Cs\bar{v}$$

$$v_L = i_L \Rightarrow v_L = i_L sL = Ls\bar{i}$$

$$s\bar{i} = \frac{1}{L}v$$

$$I = i_R + i_L + C s \bar{v}$$

$$I = \frac{1}{R}v + i_L + C s \bar{v}$$

$$s\bar{v} = \frac{-1}{RC}v - \frac{1}{C}\bar{i} + \frac{1}{C}I$$

$$\gamma = \frac{1}{R}v$$

$$\bar{q} = \bar{C}\bar{x} + \bar{D}\bar{v}$$

$$\bar{q} = \underbrace{\begin{bmatrix} \frac{1}{R} & \phi \end{bmatrix}}_{\bar{C}} \bar{x} + \underbrace{\begin{bmatrix} \phi \end{bmatrix}}_{\bar{D}} I$$

$$\textcircled{3} \quad \bar{\phi} = [sI - \bar{A}]^{-1}$$

$$\bar{\phi} = \left(\begin{bmatrix} s & \phi \\ \phi & s \end{bmatrix} - \begin{bmatrix} \frac{-1}{RC} & \frac{-1}{C} \\ \frac{1}{L} & 0 \end{bmatrix} \right)^{-1} = \begin{bmatrix} s + \frac{1}{RC} & \frac{1}{C} \\ -\frac{1}{L} & s \end{bmatrix}^{-1}$$

$$\bar{C}\bar{\phi} = \frac{1}{\det} \begin{bmatrix} \frac{1}{R} & \phi \end{bmatrix} \begin{bmatrix} s & \frac{-1}{C} \\ \frac{1}{L} & s + \frac{1}{RC} \end{bmatrix} = \frac{1}{\det} \begin{bmatrix} s & -\frac{1}{C} \\ \frac{1}{R} & \frac{-1}{C} \end{bmatrix}$$

$$= \frac{1}{s^2 + \frac{s}{RC} - \frac{1}{LC}} \begin{bmatrix} s & \frac{-1}{C} \\ \frac{1}{L} & s + \frac{1}{RC} \end{bmatrix}$$

$$\bar{C}\bar{\phi}\bar{B} + \bar{D} = \frac{1}{\det} \begin{bmatrix} \frac{s}{R} & \frac{-1}{RC} \\ \phi & \phi \end{bmatrix} \begin{bmatrix} \frac{1}{C} \\ \phi \end{bmatrix} = \frac{\frac{s}{RC}}{s^2 + \frac{1}{RC}s + \frac{1}{LC}}$$

TF

Notice $\bar{T}\bar{F} = \bar{\bar{C}}\bar{\bar{\phi}}\bar{\bar{B}} + \bar{\bar{D}}$

$$= \bar{\bar{C}}(\bar{s}\bar{\bar{I}} - \bar{\bar{A}})^{-1}\bar{\bar{B}} + \bar{\bar{D}}$$

$$= \bar{\bar{C}}(\bar{s}\bar{\bar{I}} - \bar{\bar{A}})^{-1}\bar{\bar{B}} + \underbrace{(\bar{s}\bar{\bar{I}} - \bar{\bar{A}})^{-1}(\bar{s}\bar{\bar{I}} - \bar{\bar{A}})}_{\bar{\bar{I}}} \bar{\bar{D}}$$

$$\frac{1}{\det(s\bar{\bar{I}} - \bar{\bar{A}})} \begin{bmatrix} \bar{\bar{B}} \\ \bar{\bar{D}} \end{bmatrix}$$

$$CE: \det(s\bar{\bar{I}} - \bar{\bar{A}}) = \emptyset$$

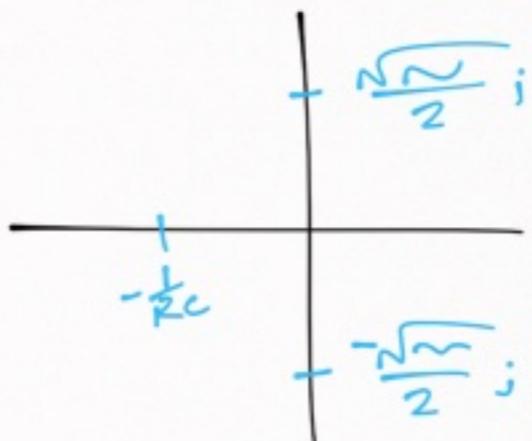
$$\det(\bar{\bar{\phi}}^{-1}) = \emptyset$$

REMEMBER EIGENVALUES roots of CE : eigenvalues ($\bar{\bar{A}}$)

$$\det(\lambda\bar{\bar{I}} - \bar{\bar{A}}) = \emptyset$$

$$CE: s^2 + \frac{1}{RC}s + \frac{1}{LC} = \emptyset$$

$$\frac{-\frac{1}{RC} \pm \sqrt{\frac{1}{RC}c^2 - \frac{4}{LC}}}{2}$$



$$\begin{aligned}\dot{\bar{x}} &= \bar{A}\bar{x} + \bar{B}\bar{u} \\ \bar{q} &= \bar{C}\bar{x} + \bar{D}\bar{u}\end{aligned}$$

\xrightarrow{f}

$$\begin{aligned}s\bar{x} &= \bar{A}\bar{x} + \bar{B}\bar{u} \\ \bar{q} &= \bar{C}\bar{x} + \bar{D}\bar{u}\end{aligned}$$

ABOUT EXAM

NOV 26 / 2014

SIGNAL FLOW

what val. crit damped
nat. frequency?
what's zeta?
what's settle time?

"DO YOUR WORK RIGHT"

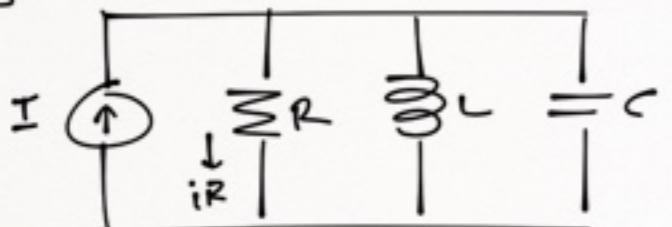
"I'M TRAINING YOU TO
BE AN ENGINEER &
ENGINEERS DO WORK
RIGHT"

"HALF-RIGHT ISN'T RIGHT"

GRAPHICAL METHOD

- ① DRAW INPUT NODES
- ② DRAW ALL STATES & STATE DERIVS. AS NODES
- ③ DRAW ALL OUTPUT NODES
- ④ ADD CONNECTIONS (GAINS) → matrix values
- ⑤ REDRAW

Ex]

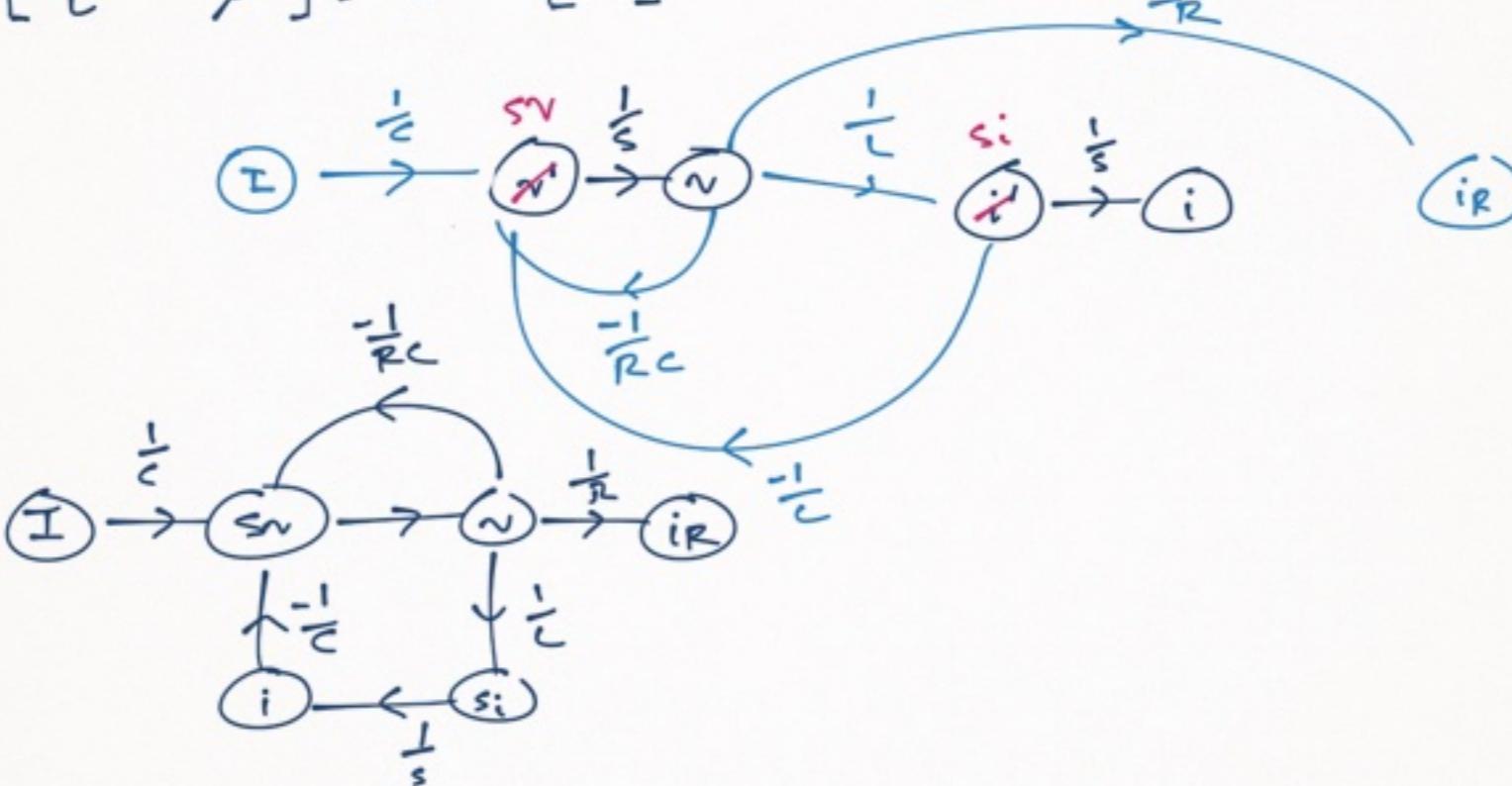


DRAW SIGNAL FLOW
DIAGRAM

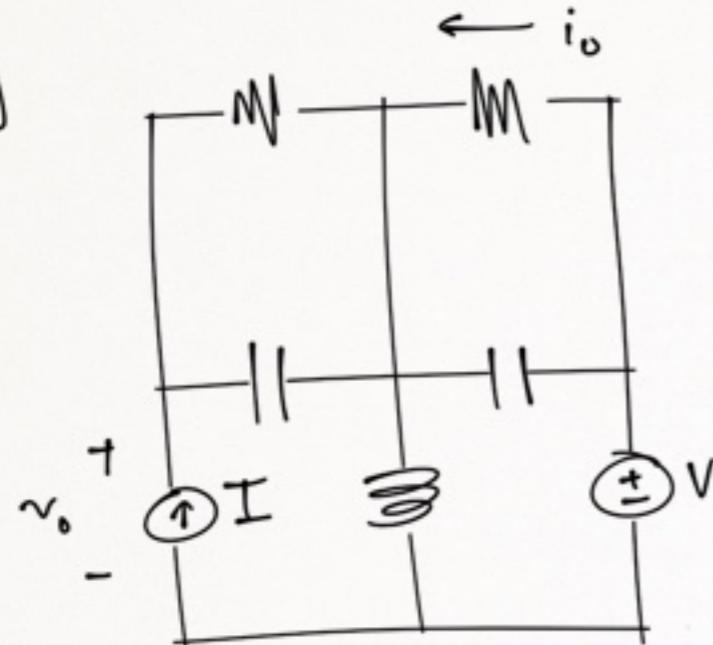
$$\begin{bmatrix} \dot{v} \\ \dot{i} \end{bmatrix} = \begin{bmatrix} -\frac{1}{RL} & \frac{-1}{C} \\ \frac{1}{L} & \emptyset \end{bmatrix} \begin{bmatrix} v \\ i \end{bmatrix} + \begin{bmatrix} \frac{1}{C} \\ \emptyset \end{bmatrix} I$$

using primes
for derivatives
because this is
shit is ridiculous.

$$i_R = \begin{bmatrix} \frac{1}{R} & \emptyset \end{bmatrix} \begin{bmatrix} v \\ i \end{bmatrix} + \emptyset I$$



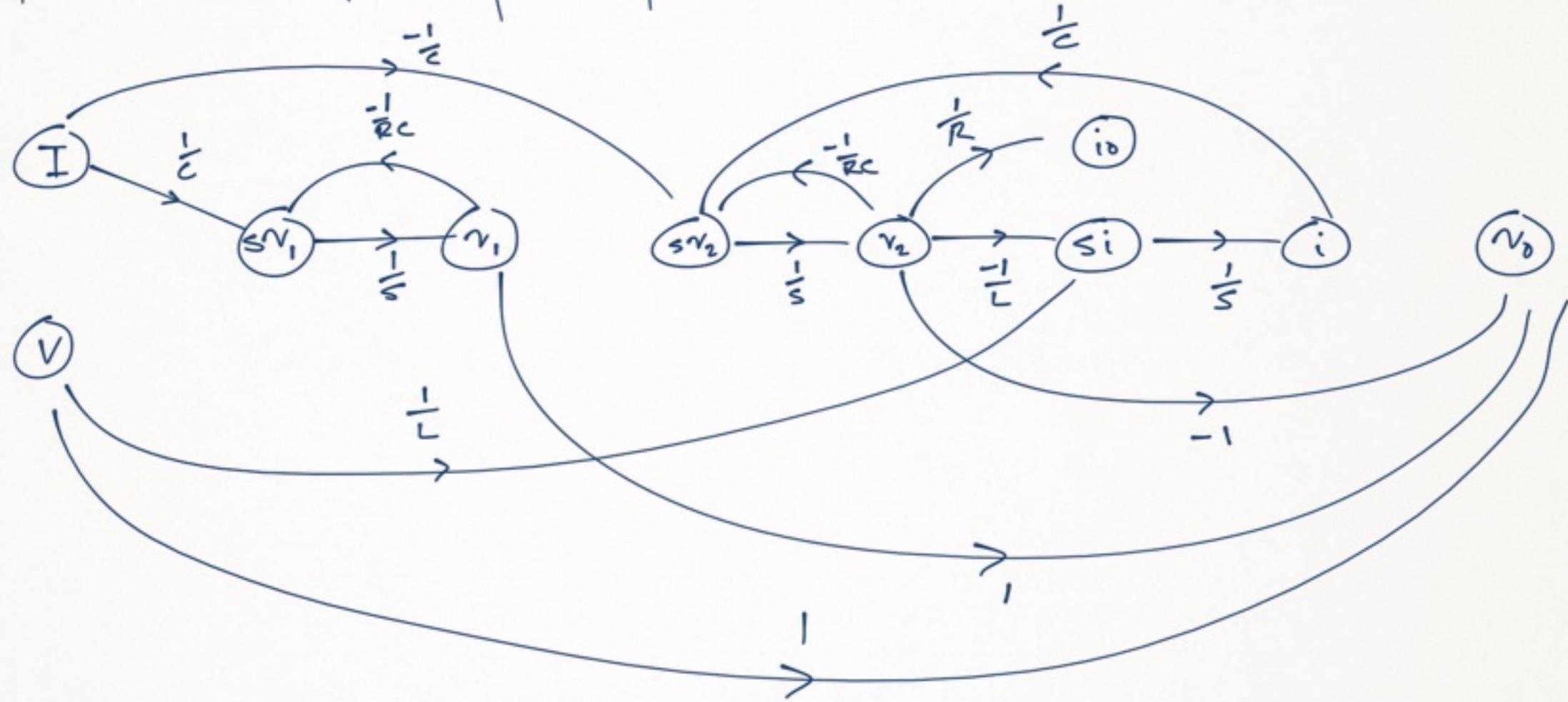
Ex]



DRAW SIGNAL FLOW DIAGRAM
DETERMINE CE.

inputs V, I
outputs v_o, i_o

$$\begin{bmatrix} v_1 \\ v_2 \\ i \end{bmatrix} = \begin{bmatrix} -\frac{1}{Rc} & \phi & \phi \\ \phi & -\frac{1}{Rc} & \frac{1}{C} \\ \phi & -\frac{1}{L} & \phi \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \\ v_3 \end{bmatrix} + \begin{bmatrix} \frac{1}{C} & \phi \\ -\frac{1}{C} & \phi \\ \phi & \frac{1}{L} \end{bmatrix} \begin{bmatrix} I \\ V \end{bmatrix}, \quad \begin{bmatrix} v_o \\ i_o \end{bmatrix} = \begin{bmatrix} 1 & -1 & \phi \\ \phi & \frac{1}{R} & \phi \end{bmatrix} \begin{bmatrix} v_1 \\ v_2 \\ v_3 \end{bmatrix} + \begin{bmatrix} \phi & 1 \\ \phi & \phi \end{bmatrix} \begin{bmatrix} I \\ V \end{bmatrix}$$



Ex) cont'd

CE is form eigs of \bar{A}

$$\det(s\bar{I} - \bar{A}) = \emptyset$$

$$\det \begin{vmatrix} s + \frac{1}{RC} & \emptyset & \emptyset \\ \emptyset & s + \frac{1}{RC} & -\frac{1}{C} \\ \emptyset & -\frac{1}{C} & s \end{vmatrix} = (s + \frac{1}{RC})(s^2 + \frac{1}{RC} + \frac{1}{C}) = s^3 + s^2 \frac{2}{RC} + s(\frac{1}{RC} + \frac{1}{RC^2}) + \frac{1}{RC^2} = \emptyset$$

↑
third order b/c 3 stored
energy systems

Rautr - Nullwerte

$$\begin{matrix} a & c & e & g & h \\ b & d & f & i & j \end{matrix} \emptyset$$

$$m_1, m_2, m_3 \emptyset$$

$$n_1, n_2, n_3 \emptyset$$

$$\sigma_1, \sigma_2 \emptyset$$

$$p_1, p_2 \emptyset$$

$$q_1 \emptyset$$

$$r_1 \emptyset$$

$$m_1 = -\frac{1}{b}(ad - cb)$$

$$m_2 = -\frac{1}{b}(cf - be)$$

$$m_3 = -\frac{1}{b}(ah - bg)$$

$$\sigma_1 = -\frac{1}{n_1}(m_1 n_2 - n_1 m_2)$$

$$\sigma_2 = -\frac{1}{n_1}(m_1 n_3 - n_1 m_3)$$

$$n_1 = -\frac{1}{m_1}(bm_2 - dm_1)$$

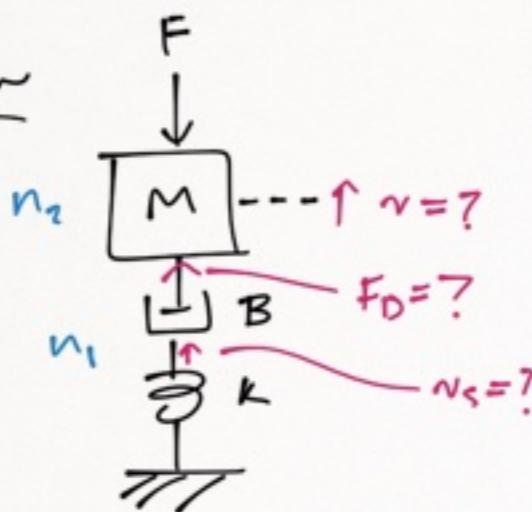
$$n_2 = -\frac{1}{m_1}(bm_3 - fm_1)$$

$$n_3 = -\frac{1}{m_1}(b\emptyset - hm_1)$$

$$p_1 = -\frac{1}{\sigma_1}(n_1 \sigma_2 - \sigma_1 n_2)$$

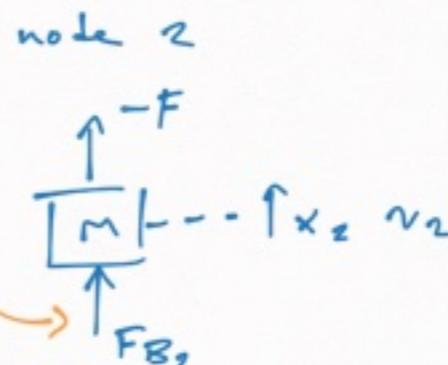
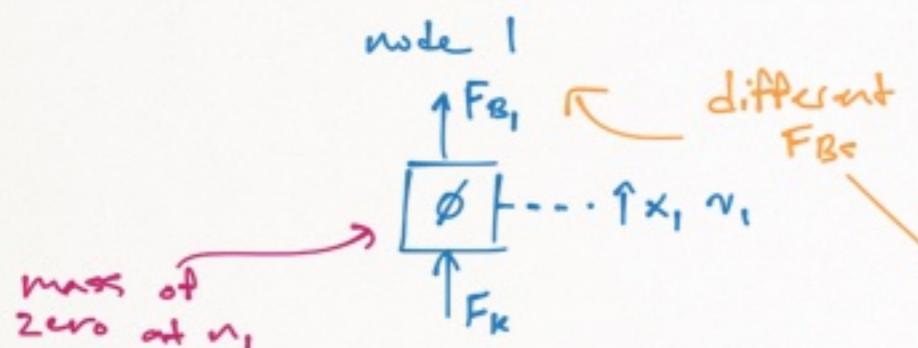
$$p_2 = -\frac{1}{\sigma_2}(n_1 \emptyset - \sigma_1 n_3)$$

REVIEW



everything defined as upward even though force downward

1. F.B.D. at each node



$$2. \sum F = ma \quad F_{B1} + F_K = \emptyset \quad \text{mass of zero at } n_1$$

$$B(n_2 - n_1) + K(-x_1) = \emptyset$$

$$\underline{Bn_2 - Bn_1 - Kx_1 = \emptyset} \quad \rightarrow Bn_2 - B\dot{x}_1 - Kx_1 = \emptyset$$

$$\begin{aligned} 3. \quad \dot{x}_1 &= n_1 \\ \dot{x}_2 &= n_2 \end{aligned}$$

4. determine states

$$F_{B1} - F = M\ddot{n}_2$$

$$\underline{B(n_1 - n_2) - F = M\ddot{n}_2}$$

$$F_{B1} = -F_{B2}$$

$$\rightarrow B(\dot{x}_1 - \dot{x}_2) - F = M\ddot{n}_2$$

$$\dot{n}_2 = \frac{B}{M}\dot{x}_1 - \frac{B}{M}n_2 - \frac{1}{M}F$$

$$\begin{pmatrix} x_1 \\ n_2 \end{pmatrix} \quad \begin{cases} \dot{x}_1 = n_2 - \frac{B}{M}x_1 \\ \dot{n}_2 = \frac{B}{M}n_2 - \frac{B}{M}x_1 - \frac{1}{M}F \end{cases}$$

for next page

5. get rid of non-states

$$6. \ddot{\bar{x}} = \bar{A}\bar{x} + \bar{B}\bar{u}$$

$$\begin{vmatrix} \dot{x}_1 \\ \dot{x}_2 \end{vmatrix} = \begin{vmatrix} -\frac{k}{B} & 1 \\ -\frac{k}{m} & 0 \end{vmatrix} \begin{vmatrix} x_1 \\ x_2 \end{vmatrix} + \begin{vmatrix} 0 \\ -\frac{1}{m} \end{vmatrix} F$$

7. output equation

$$\begin{vmatrix} v_m \\ v_s \\ F_{B_2} \end{vmatrix} = \begin{vmatrix} 0 & 1 \\ -\frac{k}{B} & 0 \\ -k & 0 \end{vmatrix} \begin{vmatrix} x_1 \\ x_2 \end{vmatrix} + \begin{vmatrix} 0 \\ 0 \\ 0 \end{vmatrix} F$$

$$TF = ?$$

$$CE = ? \leftarrow \det(s\bar{I} - \bar{A})$$

do states
calc state matrix
c.e.?
poles etc.?

$$\begin{aligned} v_s &= \dot{x}_1 \\ F_B &= M\dot{x}_2 + F = -kx_1 \end{aligned}$$

?

$$\bar{\Phi} = [s\bar{I} - \bar{A}]^{-1} = \begin{bmatrix} s + \frac{k}{B} & -1 \\ \frac{k}{m} & s \end{bmatrix}^{-1} = \frac{1}{s^2 + \frac{k}{B}s + \frac{k}{m}} \begin{bmatrix} s & 1 \\ -\frac{k}{m} & s + \frac{k}{B} \end{bmatrix}$$

$$CE: s^2 + \frac{k}{B}s + \frac{k}{m} = 0$$

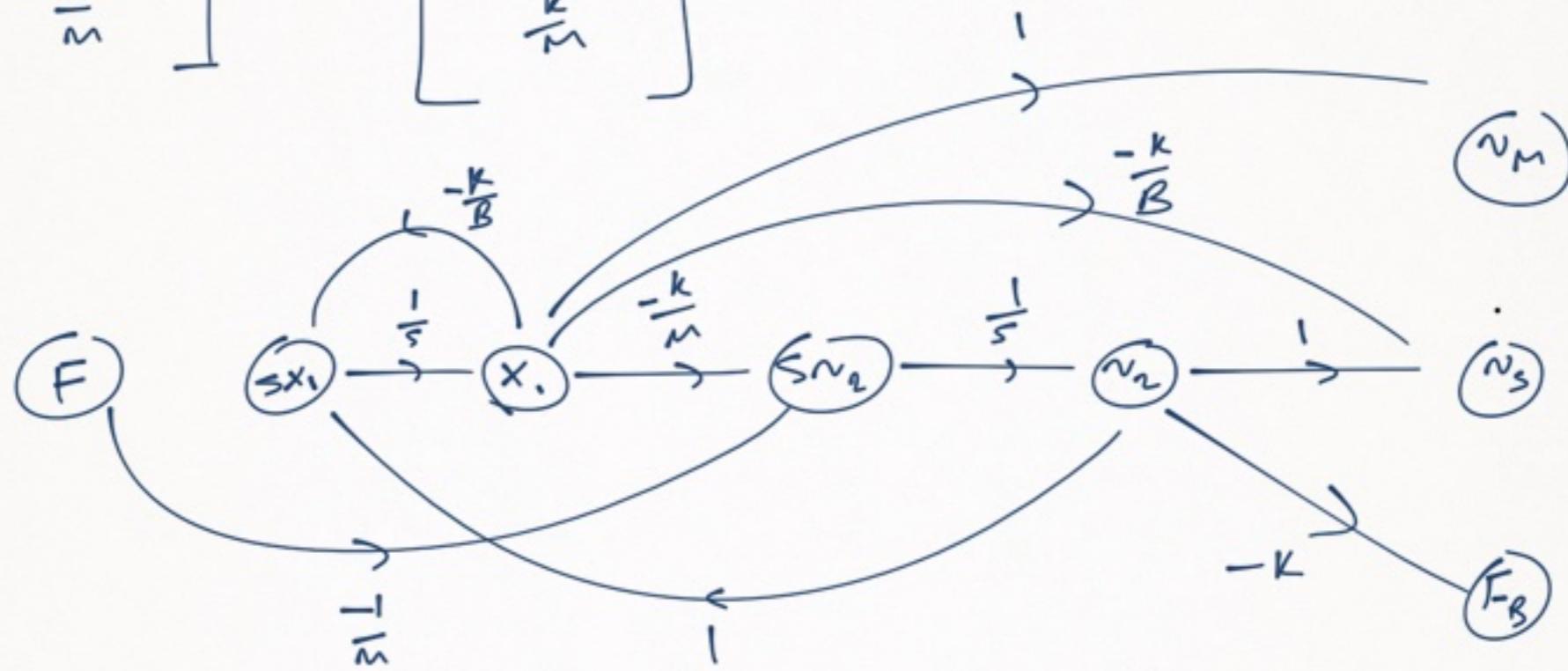
$$\omega_n = \sqrt{\frac{k}{m}}$$

2 ω_n

$$TF \quad C\Phi B + D = \frac{1}{\det} \begin{bmatrix} 0 & 1 \\ -\frac{k}{B} & 1 \\ -k & 0 \end{bmatrix} \begin{bmatrix} s & 1 \\ -\frac{k}{m} & s + \frac{k}{B} \end{bmatrix} \begin{bmatrix} 0 \\ -\frac{1}{m} \end{bmatrix} = \frac{1}{\det} \begin{bmatrix} 0 & 1 & 0 \\ -\frac{k}{B} & 1 & -\frac{1}{m} \\ -k & 0 & \frac{s}{m} - \frac{k}{Bm} \end{bmatrix}$$

$$\begin{aligned} \gamma &= \frac{k}{B(2\omega_n)} = \frac{k}{2B} \frac{1}{\sqrt{\frac{k}{m}}} = \frac{k}{2B} \frac{\sqrt{m}}{\sqrt{k}} \\ &= \frac{k}{2B} \frac{\sqrt{m}}{\sqrt{k}} \end{aligned}$$

$$= \frac{1}{J_{eff}} \begin{bmatrix} -\frac{s}{m} - \frac{k}{BM} \\ \frac{k}{BM} - \frac{s}{m} - \frac{k}{BM} \\ -\frac{k}{m} \end{bmatrix} = \frac{1}{J_{eff}} \begin{bmatrix} -\frac{s}{m} - \frac{k}{BM} \\ -\frac{s}{m} \\ \frac{k}{m} \end{bmatrix}$$



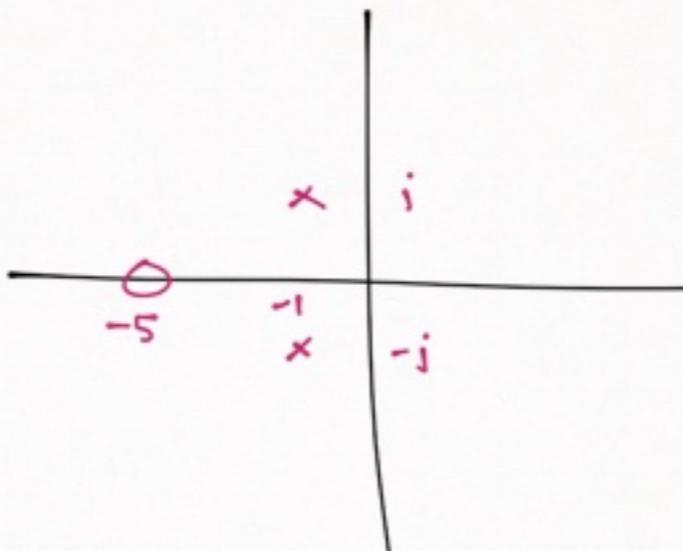
NOV 28 2014

Review #2

$$\frac{k(s+5)(s^2+2s+2)}{1 + \frac{k(s+5)}{(s+1)(s^2+2s+2)}}$$

$$\frac{-2 \pm \sqrt{4-8}}{2}$$

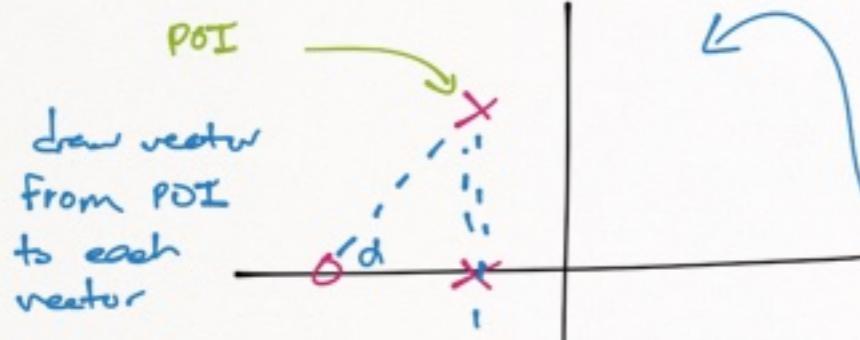
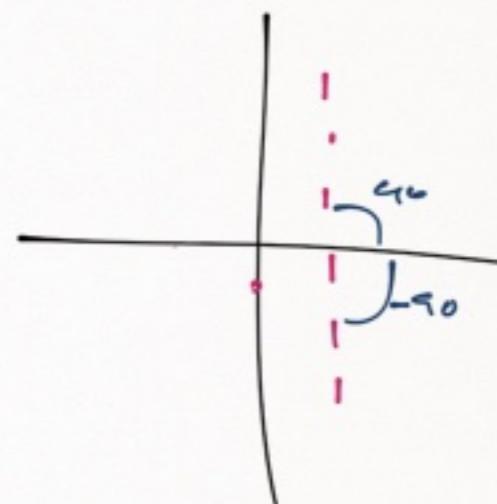
$$xs = 3 - 1 = 2$$



odd or even?

real axis
is asymptote

not a real axis
& you can see 90°



$$\sum P - \sum Z = 180$$

$$90 + 90 + DA - \alpha = 180$$

$$\alpha = \tan^{-1}(\frac{1}{4})$$

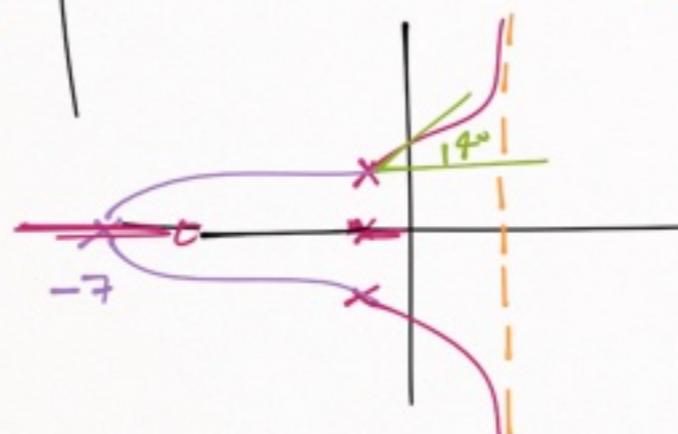
$$\therefore DA = 14^\circ$$

Assym: $\pm 90^\circ$

$$AC = \frac{\sum P - \sum Z}{RS} = \frac{-3+5}{2} = 1$$

departure angle?

~~arrival angle?~~ N/A need complex zeros



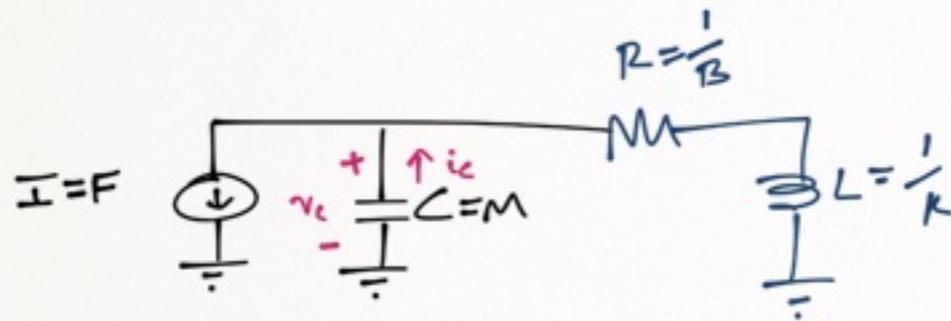
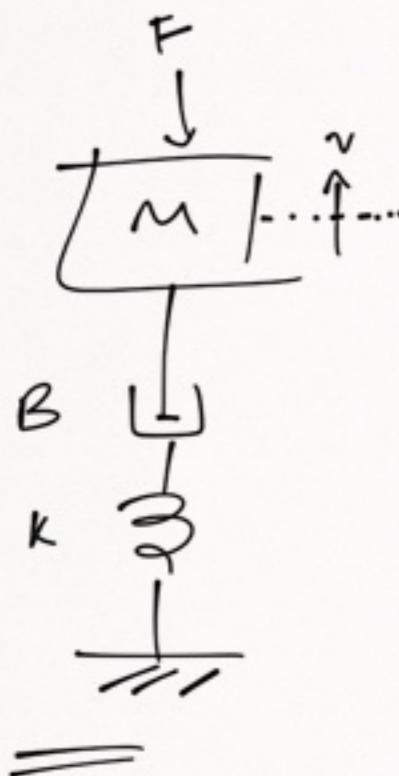
break points

$$\frac{1}{ds} \frac{1}{G_H} = \infty$$

we only care about
breakpoints on real axis

$$(s+7)(s+cmplx)$$

$$\frac{v}{F} = ?$$



velocity it rises corresponds to voltage on capacitor

apply force, increase velocity down
∴ reduce velocity up

∴ current down ↘

current divider

$$Z_C = \frac{1}{sC}$$

$$Z_{RL} = R + sL$$

$$Y_C = sC$$

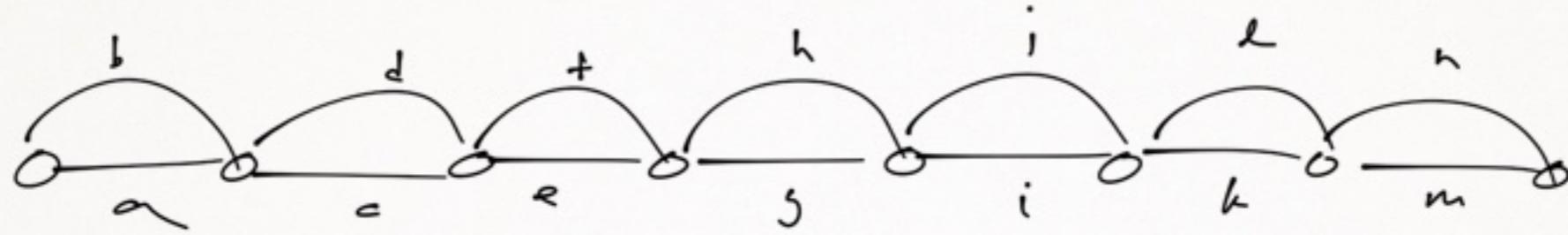
$$Y_{RL} = \frac{1}{R + sL}$$

$$i_C = \frac{Y_C}{Y_{RL} + Y_C} I = \frac{sC}{sC + R + sL} I = \frac{sC(R + sL)}{s^2LC + sRC + 1} I$$

$$v_C = i_C Z_C = \frac{1}{sC} \cdot i_C = \frac{-(R + sL)}{s^2LC + sRC + 1} I$$

$$\frac{v_C}{I} = \frac{\frac{1}{C}s - \frac{R}{LC}}{s^2 + s\frac{R}{L} + \frac{1}{LC}} = \frac{-\frac{1}{M}s - \frac{K}{BM}}{s^2 + \frac{K}{B}s + \frac{K}{M}}$$

✓ // $\frac{i_1}{I} = \frac{R_2}{R_1 + R_2}$
 $= \frac{G_1}{G_1 + G_2}$



$P = a \rightarrow c \rightarrow e \rightarrow g \rightarrow i \rightarrow k \rightarrow m$

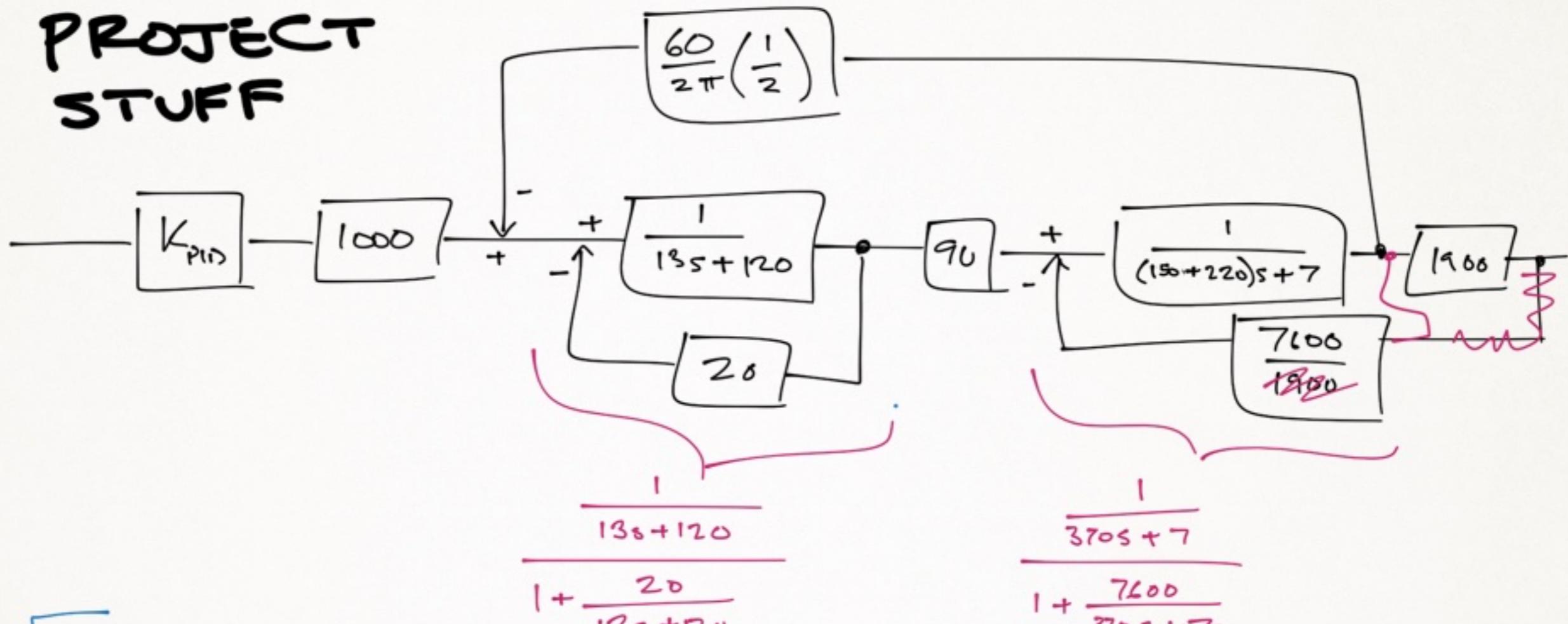
$$L_1 = ab \quad L_2 = cd \quad L_3 = ef \quad \dots$$

$$\sum \pi_1 = L_1 + L_2 + L_3 + \dots + L_7$$

$$\begin{aligned} \sum \pi_2 = & L_1 L_2 + L_1 L_3 + L_1 L_5 + L_1 L_6 + L_1 L_7 \\ & + L_2 L_4 + L_2 L_5 + \dots + L_2 L_7 \\ & + L_3 L_5 + L_3 L_6 + L_3 L_7 \\ & + L_4 L_6 + L_4 L_7 + L_5 L_7 \end{aligned}$$

$$\begin{aligned} \sum \pi_3 = & L_1 L_3 L_5 + L_1 L_3 L_6 + L_1 L_3 L_7 \\ & + L_1 L_4 L_6 + L_1 L_4 L_7 + L_1 L_5 L_7 \end{aligned}$$

PROJECT STUFF



$$\frac{60}{2\pi} \left(\frac{1}{2}\right)$$

$$\frac{\frac{1}{6700s^2}}{1 + \frac{1}{13s+120} \cdot \left(1 + \frac{20}{13s+120} \right) (90) \left(\frac{1}{370s+7} + \frac{7600}{370s+7} \right)} \cdot 1900 - (6700 \cdot 9.81)$$

OPEN LOOP TRANSFER
FUNCTION

OPEN LOOP TRANSFER FUNCTION SIMPLIFIED

$$\frac{-9.81s^2 - 307.33s - 21(7.598)}{s^4 + 31.33s^3 + 221.5s^2} = G(s)$$

6RA4: (6.8)

CLOSED LOOP

$$\frac{G(s)}{1 + G(s)} \leftarrow \text{characteristic eqn.}$$

CHARACTERISTIC EQN

$$1 + KGH = K_{PID} \cdot G(s) + 1$$

722 5228 624.4

6 630 1010

MAW 721 522 624

TAIL 0 620 1012

6.57

ZEROS @ $s = -10.605, s = -10.723$

ZEROS @ $s = -3.1257, s = 3.1245$

$s = -10.78346, s = -20.549$

POLES @ $s = 0, s = 0, -10.77836, -20.5503256627$

poles @ $s = 0, s = 0$

$s = -10.77836, s = -20.55$

ROUTIN-HURWITZ

$$1s^4 + 31.33s^3 + 221.5s^2 + \phi s + k_c + 1$$

1 221.5 $(k_c + 1)$

6.55

31.33 ϕ

6.5514

MAW 724 518 624

221.47 $(k_c + 1)$

721 522 624
0 622 1014

TAIL 100 500 750

1 1414 35092.87 -450120.6 k_c+1
 62.6573 13263.87 -135982.25 \emptyset
 726 525 627
 300 660 930
 6.4901

65438
 721 522 624
 90 491 964

721 522 624
 90 491.5 964

6.5340

65433
 721 522 624
 90 493 964

721 522 624
 90 501.5 986

6.5299

6.5405
 721 522 624
 90 493 964

725 524 126
 0 ~~650~~ 1015
 63216

6.5400
 721 522 624
 90 495.5 964

726 525 627
 6 ~~625~~ 1039
 6.3191 240 746 937

726 525 627
 250 750 950
 6.5633 6.5037

726 525 627
 450 860 900
 6.47

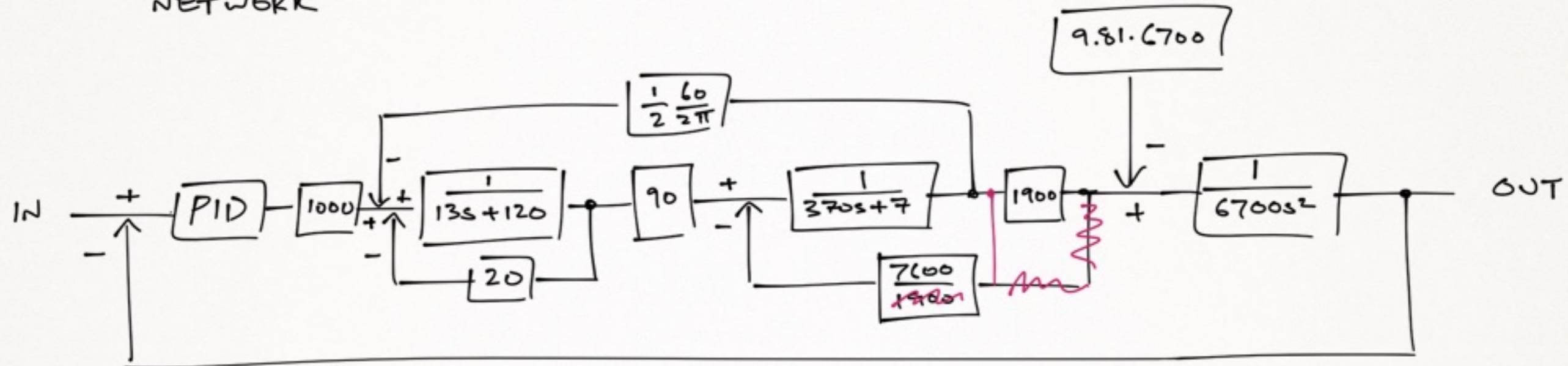
457 860 875
 6.478

726 525 627
 465 865 871
 625 6.4619

86 510 995
 511
 6.5196
 65174.

only main fuck.

NETWORK



$$G_1(s) = 90 \cdot \frac{1}{13s+120} \cdot \frac{1}{370s+7} = \frac{90}{4810s^2 + 150691s + 1064980}$$

$$1000 \cdot \frac{G(s)}{1 + \frac{60}{4\pi} G(s)} \cdot 1900 = \frac{171 \cdot 10^6 \pi}{(4810s^2 + 150691s + 1064980)\pi + 1350}$$

① OPEN LOOP TF EXCLUDING PLANT AND DISTURBANCES

$$\left[\frac{171 \cdot 10^6 \pi}{(4810s^2 + 150691s + 1064980)\pi + 1350} - 9.81(6700) \right] \frac{1}{6700s^2} \approx \frac{-9.81s^2 - 307.33s + 2167.6}{s^4 + 31.32s^3 + 221.4989s^2}$$

② OPEN LOOP TF INCLUDING DISTURBANCES

$$\frac{171 \cdot 10^4 \pi}{(322270s^4 + 10096297s^3 + 71353660s^2)\pi + 90450s^2}$$

③ OPEN LOOP TF EXCLUDING DISTURBANCES

MATCHES MODEL

D625
NOT MATCH MODEL

ROUTH HURWITZ

OF CLOSED LOOP CE OF ①

$$4810\pi \quad 1064980\pi + 1350 + k_v + 1$$

$$150691\pi \quad \emptyset$$

$$\emptyset < \left[\emptyset - 150691\pi(106 + 180\pi + 1350 + k_v + 1) \right] \frac{-1}{150691\pi}$$

$$\emptyset < 1064980\pi + 1350 + k_v + 1$$

$$k_v > -3347084.34422$$

$$\therefore K_p = 0.45(K_v) \approx -1.5062 \cdot 10^6 \quad \underline{\text{UNACCEPTABLE}}$$

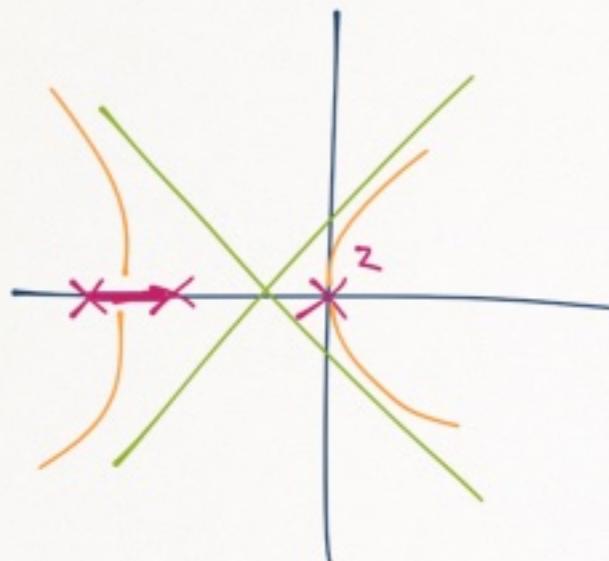
OF CLOSED LOOP CE OF ③

$$322270\pi \quad 71353660\pi + 90450 \quad 1 + k_v$$

$$10096297\pi \quad \emptyset$$

NO VALUE OF k_v CAN PREVENT SIGN CHANGE

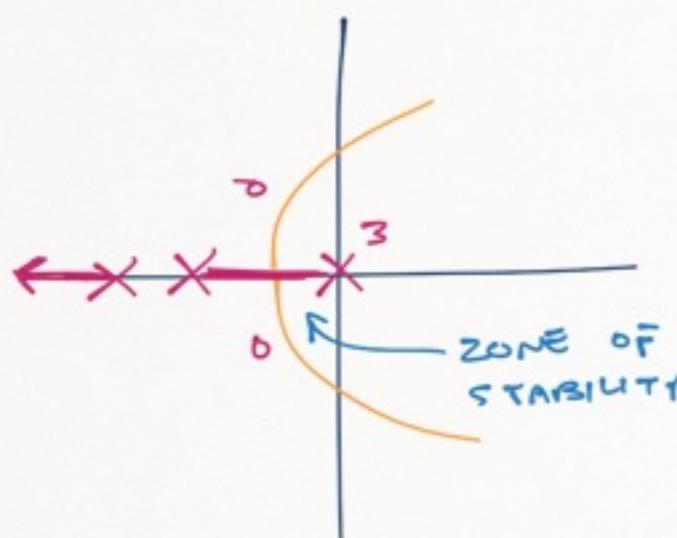
ZEIGLER METHODS IS NOT GOING TO WORK



ROOT LOCUS OF NON-PID SYSTEM

NO AMOUNT OF K_p WILL MAKE SYSTEM STABLE

$$xs = 4(?)$$



PID ADDS POLE @ \emptyset
≠ 2 ZEROS WHERE WE DEFINE

$xs = -$ which changes asymptotes (?)

FIND ZEROS THAT MAKE STABLE BY REVERSE
ENGINEERING PID FROM $k_p = 725, k_i = 623, k_d = 525$

$$s^2 + s \frac{k_p}{k_d} + \frac{k_i}{k_d} = \emptyset \quad \text{TOP SECTION}$$

$$s \approx -0.690 \pm j 0.842$$

