

*- Alex Jaskins***Mechanical Energy**

1. A river flowing steadily at a rate of $175 \text{ m}^3/\text{s}$ is being considered for hydroelectric power generation. It is determined that a dam can be built to collect water and release it from an elevation difference of 80 m to generate power. Determine how much power can be generated from this river water after the dam is filled. Is the answer you get realistic? [15]

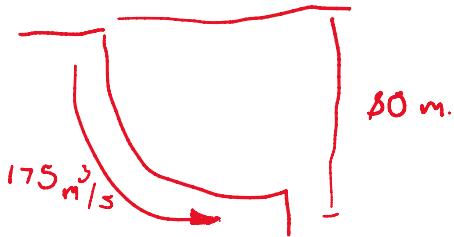
$$\dot{m} = \rho \dot{V} = 1000 (175) = 175000$$

$$\rho = \dot{m} g h \quad v_1 = v_2; \Delta Y = 0$$

$$P = 175000 (9.81) (80)$$

$$P = 137.34 \text{ MW}$$

\therefore Realistic



2. A typical 50 cm diameter room fan is desired to generate an airflow of 7.5 m/s. How much power does this fan require? Is this answer realistic? Why or why not? [15]

$$\rho = 1.225 \text{ kg/m}^3$$

$$A = \frac{1}{4} (\pi d)^2$$

$$\dot{m} = \rho A v = 1.225 \left(\frac{1}{4} (\pi (0.5))^2 \right) (7.5)$$

$$P = \dot{m} v (Y) = 1.225 \left(\frac{1}{4} (\pi (0.5))^2 \right) (7.5) (7.5) (7.5)$$

$$P = 13.5297 (7.5)$$

$$P = 101.478 \text{ W}$$

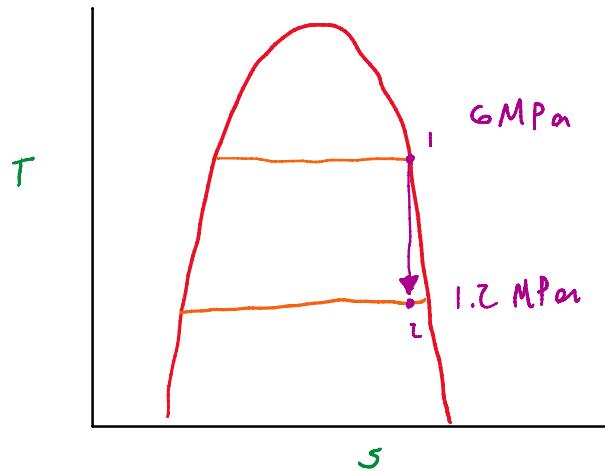
\therefore Realistic because energy is lost as heat and due to air resistance

Entropy

3. Steam enters a steady-flow adiabatic nozzle with a low inlet velocity (assume ~ 0 m/s) as a saturated vapor at 6 MPa and expands to 1.2 MPa. [15]

- a. Under what conditions is the velocity from this nozzle maximized? Sketch the T-s diagram for this process.

For the maximum exit velocity, $h_i - h_e$ should also be maximized, requiring an isentropic process.



b. Determine the maximum possible exit velocity in m/s.

$$@ 6 \text{ MPa} \quad h_i = h_g = 2784.6 \text{ kJ/kg} \quad s_i = s_g = 5.8902 \text{ kJ/kgK}$$

$$@ 1.2 \text{ MPa} \quad h_{f2} = 798.33 \text{ kJ/kg} \quad h_{fg2} = 1985.4 \text{ kJ/kg}$$

$$s_{f2} = 2.2159 \text{ kJ/kgK} \quad s_{fg2} = 4.3058 \text{ kJ/kgK}$$

$$s_i = s_2$$

$$s_{f2} + x s_{fg2} = s_i$$

$$2.2159 + x(4.3058) = 5.8902$$

$$x = \frac{5.8902 - 2.2159}{4.3058}$$

$$x = .853$$

$$h_2 = h_{f2} + x(h_{fg2})$$

$$h_2 = 798.33 + .8533(1985.4)$$

$$h_2 = 2492.47 \text{ kJ/kg}$$

$$h_i + \frac{V_i^2}{2} = h_2 + \frac{V_2^2}{2}$$

$$\therefore \therefore \therefore \therefore - V_2^2$$

$$h_1 - h_2 + 0 = \frac{V^2}{2}$$

$$(2784.6(10^3)) - (2492.47(10^3)) = \frac{V^2}{2}$$

$$V_2 = 764.37 \text{ m/s}$$

4. Helium gas is compressed from 90 kPa and 30°C to 450 kPa in a reversible, adiabatic process. Determine the final temperature and the work done, assuming the process takes place (a) in a piston-cylinder device and (b) in a steadyflow compressor. [15]

$$T_2 = 303 \left(\frac{450}{90} \right) = 576.9 \text{ K}$$

$$E_i - E_o = \Delta E = W_{in} = m(u_2 - u_1)$$

$$W_{in} = c_v(T_2 - T_1)$$

$$W_{in} = 3.1156(576.9 - 303)$$

$$W_{in} = 853.4 \text{ nJ/kg}$$

$$\dot{W} + \dot{m}h_1 - \dot{m}h_2 = 0$$

$$\dot{W} = \dot{m}(h_2 - h_1) = c_p(T_2 - T_1)$$

$$\dot{W} = 5.1926(576.9 - 303)$$

$$\dot{W} = 1422.3 \text{ nJ/kg}$$

5. Argon gas enters an adiabatic compressor at 14 psia and 75°F with a velocity of 60 ft/s, and it exits at 200 psia and 240 ft/s. If the isentropic efficiency of the compressor is 87 percent, determine (a) the exit temperature of the argon and (b) the work input to the compressor. [20]

$$\tau = \tau \left(\frac{P_2}{P_1} \right)^{\frac{n-1}{n}} = 5.26 \left(\frac{200}{14} \right)^{\frac{1.667-1}{1.667}} = 1553 \text{ R}$$

$$T_2 = T_1 \left(\frac{P_2}{P_1} \right)^{\frac{n-1}{n}} = 536 \left(\frac{200}{14} \right)^{\frac{1.667-1}{1.667}} = 1553 \text{ R}$$

$$ME = \frac{V_2^2 - V_1^2}{2} = \frac{(240)^2 - (60)^2}{2} = \frac{27000}{25037} = 1.078 \frac{\text{Btu}}{\text{lbm}}$$

$$\eta = \frac{h_s - h_1 + ME}{h_2 - h_1 + ME} = \frac{c_p(T_s - T_1) + ME}{c_p(T_2 - T_1) + ME}$$

$$n(c_p T_2 - c_p T_1) + n(ME) = c_p(T_s - T_1) + ME$$

$$n c_p T_2 = c_p T_s - c_p T_1 + ME + c_p T_1 - n ME$$

$$T_2 = \left(1 - \frac{1}{n}\right)(T_1 - \frac{ME}{c_p}) + \frac{T_s}{n}$$

$$T_2 = \left(1 - \frac{1}{.87}\right)\left(536 - \frac{1.078}{.1253}\right) + \frac{1553}{.87}$$

$$T_2 = 1706 \text{ R}$$

$$W + \dot{m} \left(h_1 + \frac{V_1^2}{2} \right) = \dot{m} \left(h_2 + \frac{V_2^2}{2} \right)$$

$$W + c_p T_1 + \frac{V_1^2}{2} = c_p T_2 + \frac{V_2^2}{2}$$

$$W = c_p(T_2 - T_1) + ME = .1253(1706 - 536) + 1.078$$

$$W = 147.68 \frac{\text{Btu}}{\text{lbm}}$$