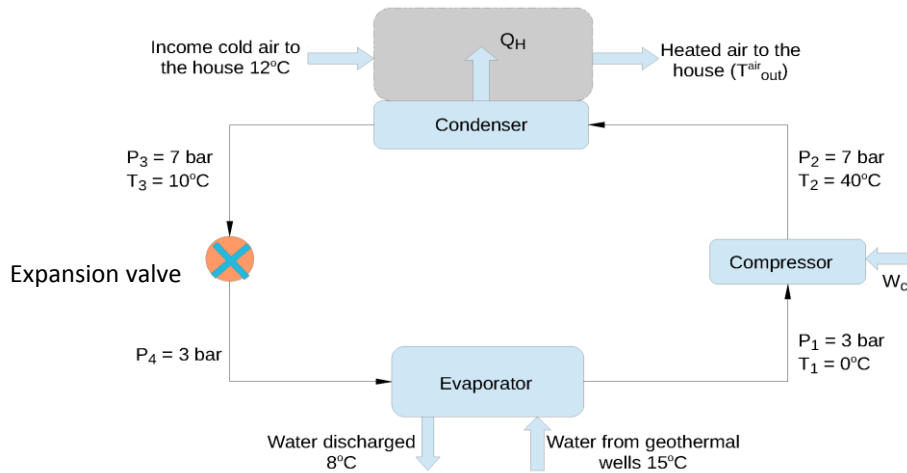


RESIT PAPER

PROBLEM 1

An engineer decided to use a geothermal source from the yard to keep her Scottish house warm during the winter. The designed heat pump extracts water from the well at 15°C and discharges at 8°C. As working fluid, she decided to use propane (C_3) - see Figure below, that will transfer heat into a constant stream of cold air ($m_{\text{air}} = 2 \text{ kg/s}$, $C_{p,\text{air}} = 1.004 \text{ kJ/(kg.K)}$) at 12°C.



- Calculate enthalpies and entropies of streams 1-4. **[8 Marks]**
- For a mass flow rate of C_3 (m_{C_3}) of 0.1 kg/s, calculate the required water flow rate (m_W). The heat capacity (C_p) of water is 4.1813 kJ/(kg.K). **[2 Marks]**
- Assuming that all heat extracted in the condenser is transferred to the air stream ($m_{\text{air}} = 2 \text{ kg/s}$), calculate the temperature of this heated stream ($T^{\text{air}}_{\text{out}}$). **[5 Marks]**
- Nearby the engineer's house, a geothermal reservoir was mapped and the following data was gathered,

Temperature (°C)	25	40	63	100	155	245
Depth (m)	0	200	400	600	800	1000

- Calculate the temperature gradient of this reservoir. **[2 Marks]**
- Binary cycle geothermal power plants operate at a temperature above 100°C. Assuming there is no heat losses in the production well, what is the ideal depth for a source geothermal fluid of 143°C? **[3 Marks]**
- Describe how binary cycle geothermal power plants operate. **[5 Marks]**

To solve this problem, you should assume that the saturated liquid streams are incompressible, and therefore $dh = v dP$ (where h , v and P are specific enthalpy, specific volume and pressure, respectively). Quality of the vapour stream is expressed as,

$$x_j = \frac{\psi_i - \psi_f}{\psi_g - \psi_f} \quad \text{with } \psi = h, s$$

where s is the specific entropy.

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Solution: (a)

Stage 1: is at $P_1 = 3 \text{ bar}$ and $T_1 = 0^\circ\text{C} \Rightarrow T_{sat} = -14.16^\circ\text{C} < T_1$, thus C_3 is at
 [1/8] superheated state (SHF). Thus, from SHF table, $h_1 = 477.1 \text{ kJ.kg}^{-1}$ and
 [1/8] $s_1 = 1.851 \text{ kJ.(kg.K)}^{-1}$

Stage 2: At $P_2 = 7 \text{ bar}$ and $T_2 = 40^\circ\text{C} \Rightarrow T_{sat} = 13.41^\circ\text{C} < T_2$, thus C_3 is at
 [1/8] superheated state (SHF). Thus, from SHF table $h_2 = 534.8 \text{ kJ.kg}^{-1}$ and
 [1/8] $s_2 = 1.901 \text{ kJ.(kg.K)}^{-1}$.

Stage 3: At $P_3 = 7 \text{ bar}$ and $T_3 = 10^\circ\text{C} \Rightarrow T_{sat} = 13.41^\circ\text{C} > T_3$, thus C_3 is
 [1/8] a sub-cooled fluid. Thus, from saturated table, $h_3 = 129.6 \text{ kJ.kg}^{-1}$ and
 [1/8] $s_3 = 0.495 \text{ kJ(kg.K)}^{-1}$.

[1/8] **Stage 4:** At $P_4 = 3 \text{ bar} \Rightarrow$ Isenthalpic expansion, thus $h_4 = h_3 = 129.6 \text{ kJ.kg}^{-1}$.

In order to calculate the entropy, we first need to calculate the quality of the fluid,

$$x_4 = \frac{h_4 - h_f}{h_g - h_f} = \frac{129.6 - 60.3}{453.6 - 60.3} = 0.1762$$

$$x_4 = 0.1762 = \frac{s_4 - s_f}{s_g - s_f} = \frac{s_4 - 0.244}{1.762 - 0.244} \Rightarrow s_4 = 0.5115 \frac{\text{kJ}}{\text{kg.K}}$$

[1/8]

Solution: (b)

The heat exchange in the evaporator can be expressed as:

$$\begin{aligned} -\dot{m}_{C3}(h_1 - h_4) &= (\dot{h}_w^{out} - \dot{h}_w^{in}) = \dot{m}_w C_{p,w} (T_w^{out} - T_w^{in}) \\ -10^{-1}(477.1 - 129.6) &= \dot{m}_w \times 4.1813 \times (8 - 15) \Rightarrow \dot{m}_w = 1.1873 \frac{\text{kg}}{\text{s}} \end{aligned}$$

[2/2]

Solution: (c)

The heat exchange in the condenser is expressed as

$$Q_H = \dot{m}_{C3}(h_2 - h_3) = 40.52 \frac{\text{kJ}}{\text{kg}}$$

[2/5] Assuming that there is no heat loss,

$$Q_4 = 40.52 = \dot{m}_{air} C_{p,air} (T_{out}^{air} - T_{in}^{air}) = 2 \times 1.004 \times (T_{out}^{air} - 12) \Rightarrow T_{out}^{air} = 32.18^\circ\text{C}$$

[3/5]

Solution: (d - i)

$$\nabla T = \frac{\partial T}{\partial z} = \frac{T_n - T_1}{z_n - z_1} = \frac{245 - 25}{1000 - 0} = 0.22 \frac{^\circ\text{C}}{\text{m}}$$

[2/2]

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Solution: (d – ii)

The depth can be obtained via linear interpolation at $100 \leq T \leq 155^{\circ}\text{C}$. At 143°C the
[3/3] depth is **756.36m**

Solution: (d – iii)

- Binary cycle geothermal power plants are often operated between $107 \leq T \leq 182^{\circ}\text{C}$;

[1/5]

- The produced hot brine vaporises the working fluid and is isentropically expanded in the turbine;

[2/5]

- Working fluids often used in geothermal plants are organic chemical species with low boiling temperature.

[2/5]