

Abstract

This work presents a physics-infused reduced-order modeling (PIROM) framework towards the design, analysis, and optimization of ablating hypersonic thermal protection systems (TPS).

1 Introduction

At hypersonic speeds, aerospace vehicles experience extreme aero-thermal environments that requires specialized thermal protection systems (TPS) to shield internal sub-structures, electronics, and possibly crew members from the intense aerodynamic heating. The TPS is often composed of ablating materials – a high-temperature capable fibrous material injected with a resin that fills the pore network and strengthens the composite [Amar2016]. This design promotes the exchange of mass through thermal and chemical reactions (i.e., pyrolysis), effectively mitigating heat transfer to the sub-structures.

Accurate modeling of ablating TPS under extreme hypersonic heating becomes fundamental to ensuring the survivability, performance, and safety of hypersonic vehicles. Particularly, not only is it necessary to assess the performance of the thermal management systems, but also the shape changes of the vehicle’s outer surface induced by the ablating material, and its impact on the aerodynamics, structural integrity, and controllability. Nonetheless, despite these pressing requirements for highly performant hypersonic vehicles, high-fidelity simulations of ablating TPS remains a formidable challenge both theoretically and computationally.

On the theoretical side, the thermo-chemical reactions, coupled with the irregular pore network structure, translate into simplifying assumptions to reduce the equation set to be more amenable for engineering application and design analysis [x]. For instance, one of the most notable codes is the one-dimensional CMA code that was developed by Aerotherm Corporation in the 1960s and is still in use today [Howard2015]. Despite its practical use in...

Another example is the CHarring Ablator Response (CHAR) ablation code, which ignores elemental decompositions of the pyrolyzing gases, assumes the gases to be a mixture of perfect gases in thermal equilibrium, and assumes no reaction or condensation with the porous network [?].

theoretically:

computationally:

2 Modeling of Ablating Thermal Protection Systems

This section presents the ablation problem for a non-decomposing generic TPS geometry as a parametrized system of non-linear PDEs. These non-linear PDEs govern the energy of heat conduction and the pseudo-elastic material deformation of the mesh motion. Two different but mathematically-connected numerical solution strategies are provided: (1) a high-fidelity full-order model (FOM) based on a discontinuous Galerkin FEM, and (2) a thermo-elastic RPM based on a one-dimensional approximation to the energy and pseudo-elasticity equations.

2.1 Governing Equations

Consider a generic domain $\Omega \subset \mathbb{R}^d$, $d = 2$ or 3 , illustrated in Fig. 1. A heat flux $q_b(x, t)$ is prescribed on the boundary Γ_q (i.e., Neumann boundary condition), and the temperature $T_b(x, t)$ is prescribed on boundary Γ_T (i.e., Dirichlet boundary condition), where $\Gamma_q \cup \Gamma_T = \partial\Omega$ and $\Gamma_q \cap \Gamma_T = \emptyset$. The ablation occurs only on the heated boundary Γ_q , and its effects are included into the energy equation using an Arbitrary Lagrangian-Eulerian (ALE) description. The ALE assumes the computational mesh moves with a velocity $\mathbf{v}(x, t)$, which is different to the material velocity, which is set to zero in this work.



Figure 1: General domain Ω with prescribed heat flux $q_b(x, t)$ and temperature $T_b(x, t)$ on the boundaries Γ_q and Γ_T , respectively. The mesh moves with a velocity $\mathbf{v}(x, t)$, while the material velocity is $\mathbf{w}(x, t)$. draw mesh next to arbitrary domain with moving boundaries.

The transient heat conduction with receding boundary is coupled with the steady pseudo-elastic mesh motion equation into the following system of PDEs,

$$\rho c_p \frac{\partial T}{\partial t} - \nabla \cdot (\mathbf{k}(T) \nabla T) - \rho c_p \mathbf{v}(x, t) \cdot \nabla T - \mathcal{Q}(x, t) = 0, \quad x \in \Omega \quad (1a)$$

$$\nabla \cdot \boldsymbol{\sigma}(\mathbf{w}) = 0 \quad (1b)$$

The density ρ is constant, while the heat capacity $c_p = c_p(T)$ and thermal conductivity $\mathbf{k} = \mathbf{k}(T) \in \mathbb{R}^{d \times d}$ may vary with temperature. The boundary and initial conditions for the heat equation are,

$$-\mathbf{k} \nabla T \cdot \mathbf{n} = q_b(x, t), \quad x \in \Gamma_q \quad (2a)$$

$$T(x, t) = T_b(x, t), \quad x \in \Gamma_T \quad (2b)$$

$$T(x, 0) = T_0(x), \quad x \in \Omega \quad (2c)$$

while for the pseudo-elastic mesh motion, the boundary and initial conditions are,

$$\mathbf{w}(x, t) = \mathbf{w}_q(x, t), \quad x \in \Gamma_q \quad (3a)$$

$$\mathbf{w}(x, t) = 0, \quad x \notin \Gamma_q \quad (3b)$$

$$\mathbf{w}(x, 0) = 0, \quad x \in \Omega \quad (3c)$$

The terms in the thermo-elastic equations eq. (1), along with the boundary conditions in eqs. (2) and (3) are described in detail next. The terms in eq. (1a), in the order they appear, correspond to the unsteady energy storage, heat conduction, temperature advection due to mesh motion, and heat source terms. The eq. (1b) implies that the divergence of the stress tensor $\boldsymbol{\sigma}(\mathbf{w})$ is zero. The stress tensor is related to the strain tensor $\boldsymbol{\epsilon}(\mathbf{w})$ through Hooke's law,

$$\boldsymbol{\sigma}(\mathbf{w}) = \mathbb{D} : \boldsymbol{\epsilon}(\mathbf{w})$$

where \mathbb{D} is the constitutive operator, “:” is the double contraction of tensors, and ϵ is the symmetric strain tensor given by,

$$\epsilon(\mathbf{w}) = \frac{1}{2} (\nabla \mathbf{w} + \nabla \mathbf{w}^T)$$

For instance, an isotropic material assumption results in,

$$\boldsymbol{\sigma} = \lambda (\nabla \cdot \mathbf{w}) \mathbf{I} + 2\mu \epsilon(\mathbf{w})$$

where λ and μ are Lamé constants that are arbitrarily selected to model the mesh motion. The “material” properties λ and μ can be chosen to tailor the mesh deformation and need not present the actual material being modeled [Amar2016](#).

The boundary conditions for the energy equation includes a heated surface (eq. (2a)) and a constant-temperature surface (eq. (2b)). The boundary conditions for the pseudo-elasticity equation are a function of the surface temperature $T_q(x, t)$ for $x \in \Gamma_q$ using a B’ table. The B’ table...

$$\mathbf{w}(x, t) = \int_0^t \mathbf{v}(x, \tau) d\tau = \int_0^t \mathbf{f}(T_q(x, \tau)) d\tau \quad (4)$$

2.2 Full-Order Model: Finite Element Method

2.2.1 Numerical Solution

To obtain the full-order numerical solution, the governing equation is spatially discretized using variational principles of Discontinuous Galerkin (DG) to result in a high-dimensional system of ordinary differential equations (ODEs).

2.2.2 Usage Within an Ablation Simulation

2.3 Formulation of Reduced-Physics Model

This section presents the derivation of the thermo-elastic RPM to model the one-dimensional temperature distribution and surface recessions for the ablating TPS. The RPM is composed of three coupled components: (1) a thermal solver based on FEM, (2) a pseudo-elastic solver for the FEM mesh, and (3) a lumped capacitance model (LCM) to model the volumetric energy source (i.e., the $\mathcal{Q}(x, t)$ in eq. (1a)) interactions between the inter-connected components.

To start the RPM formulation, consider a partitioning of the general domain in Fig. 1 into smaller one-dimensional inter-connected material components as illustrated in Fig. [x](#). Consider the n -th component with length $\ell^{(n)}$ illustrated in Fig. [x](#), and for simplicity, assume that the material properties are independent of temperature.

For the n -th component, the left surface is exposed to the hypersonic flow (Neumann boundary condition), while the right surface is perfectly insulated (adiabatic boundary condition). The left surface is receding with a velocity $v(t)$ that is a function of the surface temperature $T_w(t) = T(0, t)$. This velocity is obtained from a B’-table lookup table,

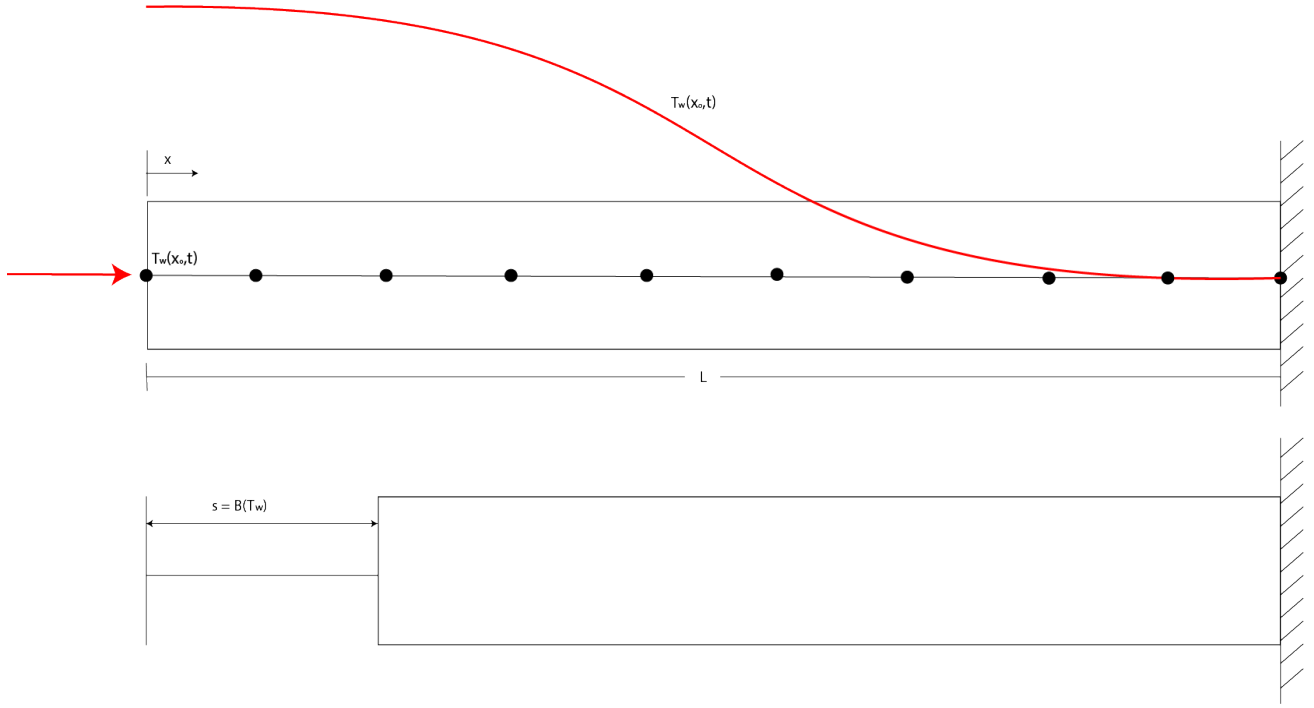
$$v(t) = f(T_w(t))$$

where $f(\cdot)$ is a cubic spline interpolation of B’ and enthalpy tabulated data.

The governing energy and elasticity equations are defined over the domain $\Omega = [0, \ell]$,

$$\rho c_p \left(\frac{\partial T}{\partial t} - v(x, t) \frac{\partial T}{\partial x} \right) - \frac{\partial}{\partial x} \left(k \frac{\partial T}{\partial x} \right) = 0 \quad (5a)$$

$$\frac{\partial}{\partial x} \left(\frac{\partial u}{\partial x} \right) = 0 \quad (5b)$$



with boundary conditions for the energy equation,

$$-k \frac{\partial T}{\partial x} \Big|_{x=0} = q_b(t) \quad (6a)$$

$$-k \frac{\partial T}{\partial x} \Big|_{x=\ell} = 0 \quad (6b)$$

and for the elasticity equation,

$$u(0, t) = \int_{t_0}^t v(\tau) d\tau = \int_0^t f(T_w(\tau)) d\tau \quad (7a)$$

$$u(\ell, t) = 0 \quad (7b)$$

2.4 Numerical Solution

Along the one-dimensional domain, a numerical solution based on FEM is adopted for the energy equation, while an analytical solution is adopted for the pseudo-elastic mesh motion. The quasi-steady boundary conditions for the mesh motion are employed via a spline fit to the B' and enthalpy tabulated data as a function of surface temperature. The main results of the numerical approach are presented here and the reader is directed to Sec. A for details.

2.4.1 Thermal Solver

The result of the FEM discretization is a system of ODEs for the nodal temperature values given by,

$$\mathbf{M} \frac{d\mathbf{T}}{dt} + (\mathbf{K} - \mathbf{C}(t)) \mathbf{T} = \mathbf{f}(t) \quad (8)$$

2.4.2 Pseudo-Elastic Solver

Note that eq. (5b) is steady. Under the assumption that the mesh deformation is quasi-steady, it can be applied at each time step within an ablation simulation. For instance, for a known value of the wall

temperature $T_w(t)$ the surface displacement

Assuming the Young's modulus is constant, the PDE simplifies to,

$$\frac{\partial^2 u}{\partial x^2} = 0 \quad (9)$$

which has the analytical solution,

$$u(x, t) = a(t)x + b(t) \quad (10)$$

Using the boundary conditions leads to,

$$u(x, t) = u(0, t) * \left(1 - \frac{x}{\ell}\right) \quad (11)$$

The mesh velocity is the time derivative of the displacement,

$$v(x, t) = \frac{\partial u(x, t)}{\partial t} = v(t) \left(1 - \frac{x}{\ell}\right) \quad (12)$$

2.4.3 Coupling Scheme

2.4.4 Reduced-Physics Ablation Simulation

A Numerical Implementation

A.1 Full-Order Model

A.2 Reduced-Physics Model

Let $\phi_i^{(e)}(x)$ with $i = 1, 2$ be two linear shape defined over the element $e_i = [x_i, x_{i+1}]$ with length $h_e = x_{i+1} - x_i$,

$$\phi_1^{(e)}(x) = \begin{cases} \frac{x_{i+1}-x}{h_e}, & x \in [x_i, x_{i+1}] \\ 0, & \text{otherwise} \end{cases}, \quad \phi_2^{(e)}(x) = \begin{cases} \frac{x-x_i}{h_e}, & x \in [x_i, x_{i+1}] \\ 0, & \text{otherwise} \end{cases} \quad (13)$$

Letting,

$$x(\xi) = \frac{1-\xi}{2}x_i + \frac{1+\xi}{2}x_{i+1}$$

for $\xi \in [-1, 1]$,

$$\hat{\phi}_1^{(e)}(\xi) = \frac{1-\xi}{2}, \quad \hat{\phi}_2^{(e)}(\xi) = \frac{1+\xi}{2} \quad (14)$$

Multiply through by the test function,

$$\int_{\Omega} \left[\rho c_p \frac{\partial T}{\partial t} - \rho c_p v(x, t) \frac{\partial T}{\partial x} - \frac{\partial}{\partial x} \left(k \frac{\partial T}{\partial x} \right) \right] \phi_i(x) dx = 0 \quad (15)$$

$$\int_{\Omega} \rho c_p \frac{\partial T}{\partial t} \phi_i(x) dx - \int_{\Omega} \rho c_p v(x, t) \frac{\partial T}{\partial x} \phi_i(x) dx + \int_{\Omega} k \frac{\partial T}{\partial x} \frac{\partial \phi_i(x)}{\partial x} dx = k \frac{\partial T}{\partial x} \phi_i(x) \Big|_{\partial \Omega} \quad (16)$$

Perform the finite-element approximation,

$$T(x, t) \approx \sum_j T_j(t) \phi_j(x) \quad (17)$$

and define the matrix elements,

$$M_{ij} = \int_{\Omega} \rho c_p \phi_i(x) \phi_j(x) dx \quad (18)$$

$$C_{ij}(t) = \int_{\Omega} \rho c_p v(x, t) \frac{\partial \phi_j}{\partial x} \phi_i(x) dx \quad (19)$$

$$K_{ij} = \int_{\Omega} k \frac{\partial \phi_i(x)}{\partial x} \frac{\partial \phi_j(x)}{\partial x} dx \quad (20)$$

$$f_i(t) = k \frac{\partial T}{\partial x} \phi_i(x) \Big|_{\partial \Omega} \quad (21)$$

The time-dependent finite-dimensional ODE system for nodal temperatures $\mathbf{T}(t)$, including the ALE-induced advection effect from mesh motion, is given as,

$$\mathbf{M} \frac{d\mathbf{T}}{dt} + (\mathbf{K} - \mathbf{C}(t)) \mathbf{T} = \mathbf{f}(t) \quad (22)$$

The element-level expressions for the mass, stiffness, advection, and forcing vectors are given as,

$$M_{mn}^{(e)} = \int_{x_i}^{x_{i+1}} \rho c_p \phi_m(x) \phi_n(x) dx = \rho c_p \frac{h_e}{6} \begin{pmatrix} 2 & 1 \\ 1 & 2 \end{pmatrix} \quad (23)$$

$$K_{mn}^{(e)} = \int_{x_i}^{x_{i+1}} k \frac{\partial \phi_m}{\partial x} \frac{\partial \phi_n}{\partial x} dx = \frac{k}{h_e} \begin{pmatrix} 1 & -1 \\ -1 & 1 \end{pmatrix} \quad (24)$$

$$C_{mn}^{(e)}(t) = \int_{x_i}^{x_{i+1}} \rho c_p v(x, t) \frac{\partial \phi_n(x)}{\partial x} \phi_m(x) dx \quad (25)$$

$$f_1^{(e)}(t) = (q(t), 0)^T \quad (26)$$